

Illinois Statewide Technical Reference Manual for Energy Efficiency Version 6.0

Volume 1: Overview and User Guide

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1 Purpose of the TRM

The purpose of the Illinois Statewide Technical Reference Manual (TRM) is to provide a transparent and consistent basis for calculating energy (electric kilowatt-hours (kWh) and natural gas therms) and capacity (electric kilowatts (kW)) savings generated by the State of Illinois' energy efficiency programs¹ which are administered by the state's largest electric and gas Utilities² (collectively, Program Administrators or the Utilities).

The TRM is a technical document that is filed with the Illinois Commerce Commission (Commission or ICC) and is intended to fulfill a series of objectives, including:

- "Serve as a common reference document for all... stakeholders, [Program Administrators], and the Commission, so as to provide transparency to all parties regarding savings assumptions and calculations and the underlying sources of those assumptions and calculations.
- Support the calculation of the Illinois Total Resource Cost test^[3] ("TRC"), as well as other cost-benefit tests in support of program design, evaluation and regulatory compliance. Actual cost-benefit calculations and the calculation of avoided costs will not be part of this TRM.
- Identify gaps in robust, primary data for Illinois, that can be addressed via evaluation efforts and/or other targeted end-use studies.
- [Provide] a process for periodically updating and maintaining records, and preserve a clear record of what deemed parameters are/were in effect at what times to facilitate evaluation and data accuracy reviews.
- ...[S]upport coincident peak capacity (for electric) savings estimates and calculations for electric utilities in a manner consistent with the methodologies employed by the utility's Regional Transmission Organization ("RTO"), as well as those necessary for statewide Illinois tracking of coincident peak capacity impacts."⁴

¹ 220 ILCS 5/8-103B and 220 ILCS 5/8-104.

² The Program Administrators include: Ameren Illinois, ComEd, Peoples Gas, North Shore Gas, and Nicor Gas (collectively, the Utilities).

³ The Illinois TRC test is defined in 220 ILCS 5/8-104(b) and 20 ILCS 3855/1-10.

⁴ Illinois Statewide Technical Reference Manual Request for Proposals, August 22, 2011, pages 3-4, http://ilsag.org/yahoo_site_admin/assets/docs/TRM_RFP_Final_part_1.230214520.pdf

1.1 Acknowledgements

This document was created through collaboration amongst the members of the Illinois Energy Efficiency Stakeholder Advisory Group (SAG). The SAG is an open forum where interested parties may participate in the evolution of Illinois' energy efficiency programs. Parties wishing to participate in the SAG process may do so by visiting <http://www.ilsag.info/questions.html> and contacting the Independent Facilitator at Annette.Beitel@FutEE.biz. Parties wishing to participate in the Technical Advisory Committee (TAC), a subcommittee of the SAG, may do so by contacting the TRM Administrator at iltrmadministrator@veic.org.

SAG Stakeholders ⁵
Ameren Illinois Company (Ameren)
Citizen's Utility Board (CUB)
City of Chicago
Commonwealth Edison Company (ComEd)
Elevate Energy
Energy Resources Center at the University of Illinois, Chicago (ERC)
Environment IL
Environmental Law and Policy Center (ELPC)
Future Energy Enterprises LLC
Illinois Attorney General's Office (AG)
Illinois Commerce Commission Staff (ICC Staff)
Illinois Department of Commerce and Economic Opportunity (DCEO)
Independent Evaluators (ADM, Cadmus, Itron, Navigant)
Metropolitan Mayor's Caucus (MMC)
Midwest Energy Efficiency Association (MEEA)
Natural Resources Defense Council (NRDC)
Nicor Gas
Peoples Gas and North Shore Gas

⁵ Being an open forum, this list of SAG stakeholders and participants may change at any time.

Table 1.1: Document Revision History

Document Title	Applicable to PY Beginning
Illinois_Statewide_TRM_Effective_060112_Version_1.0_091412_Clean.doc	6/1/12
Illinois_Statewide_TRM_Effective_060113_Version_2.0_060713_Clean.docx	6/1/13
Illinois_Statewide_TRM_Effective_060114_Version_3.0_022414_Clean.docx	6/1/14
Illinois_Statewide_TRM_Effective_060115_Final_022415_Clean.docx	6/1/15
IL-TRM_Effective_060116_v5.0_Vol_1_Overview_021116_Final IL-TRM_Effective_060116_v5.0_Vol_2_C_and_I_021116_Final IL-TRM_Effective_060116_v5.0_Vol_3_Res_021116_Final IL-TRM_Effective_060116_v5.0_Vol_4_X-Cutting_Measures_and_Attach._021116_Final	6/1/16
IL-TRM_Effective_010118_v6.0_Vol_1_Overview_020817_Final IL-TRM_Effective_010118_v6.0_Vol_2_C_and_I_020817_Final IL-TRM_Effective_010118_v6.0_Vol_3_Res_020817_Final IL-TRM_Effective_010118_v6.0_Vol_4_X-Cutting_Measures_and_Attach_020817_Final	1/1/18

1.2 Summary of Measure Revisions

The following tables summarize the evolution of measures that are new, revised or errata. This version of the TRM contains 86 measure-level changes as described in the following table.

Table 1.2: Summary of Measure Level Changes

Change Type	# Changes
Errata	5
Revision	68
New Measure	13
Total Changes	86

The 'Change Type' column indicates what kind of change each measure has gone through. Specifically, when a measure error was identified and the TAC process resulted in a consensus, the measure is identified here as an 'Errata'. In these instances the measure code indicates that a new version of the measure has been published, and that the effective date of the measure dates back to June 1st, 2016. Measures that are identified as 'Revised' were included in the fifth edition of the TRM, and have been updated for this edition of the TRM. Both 'Revised' and 'New Measure(s)' have an effective date of January 1st, 2018.

The following table provides an overview of the 86 measure-level changes that are included in this version of the TRM.

Table 1.3: Summary of Measure Revisions

Volume	End Use	Measure Name	Measure Code	Change Type	Explanation	Impact on Savings
Volume 1: Overview	N/A	N/A	N/A	Revision	Removal of DCEO as program administrator. Update of Program Year dates. Addition of Review Deadline text. Edits to "Energy Efficiency" in Glossary. Edits to Section 3.9 to remove WACC as discount rate and define Nominal and Real societal discount rates. Timing of retiring T12 as a viable baseline pushed back to 1/1/2019.	N/A
Volume 2: C&I	Food Service Equipment	4.2.4 Conveyor Oven	CI-FSE-CVOV-V02-180101	Revision	Update to deemed gas savings	Increase
		4.2.5 ENERGY STAR Convection Oven	CI-FSE-ESCV-V02-180101	Revision	Update of ENERGY STAR specification	Increase
		4.2.6 ENERGY STAR Dishwasher	CI-FSE-ESDW-V03-180101	Revision	Addition of CF to peak demand algorithm	Decrease
		4.2.11 High Efficiency Pre-Rinse Spray Valve	CI-FSE-SPRY-V04-180101	Revision	Clarification of direct install costs being full installation cost (including labor)	None
		4.2.12 Infrared Charbroiler	CI-FSE-IRCB-V02-180101	Revision	Change to algorithm methodology. Revised source for costs and loadshape.	Dependent on inputs
		4.2.13 Infrared Rotisserie Oven	CI-FSE-IROV-V02-180101	Revision	Change to algorithm methodology. Revised source for costs and loadshape.	Dependent on inputs
		4.2.14 Infrared Salamander Broiler	CI-FSE-IRBL-V02-180101	Revision	Change to algorithm methodology. Revised source for costs and loadshape.	Dependent on inputs
		4.2.15 Infrared Upright Broiler	CI-FSE-IRUB-V02-180101	Revision	Change to algorithm methodology. Revised source for costs and loadshape.	Dependent on inputs
		4.2.16 Kitchen Demand Ventilation Controls	CI-FSE-VENT-V03-160601	Errata	Fixed cost and savings assumptions to be per HP of fan, not per fan.	Dependent on inputs
		4.2.17 Pasta Cooker	CI-FSE-PCOK-V02-180101	Revision	Revised source for costs and loadshape.	Dependent on inputs
		4.2.18 Rack Oven – Double Oven	CI-FSE-RKOV-V02-180101	Revision	Change to algorithm methodology. Revised source for costs and loadshape.	Dependent on inputs
	Hot Water	4.3.1 Storage Water Heater	CI-HWE-STWH-V03-180101	Revision	Update of source for hot water consumption estimates	Dependent on inputs
		4.3.2 Low Flow Faucet Aerators	CI-HWE-LFFA-V07-180101	Revision	Clarification of direct install costs being full installation cost (including labor). Addition of Laminar devices for healthcare applications. Additional clarity of source of assumptions added.	None
		4.3.3 Low Flow	CI-HWE-LFSH-V04-180101	Revision	Clarification of direct install costs being full installation	None

Volume	End Use	Measure Name	Measure Code	Change Type	Explanation	Impact on Savings
		Showerheads			cost (including labor)	
		4.3.8 Controls for Central Domestic Hot Water	CI-HWE-CDHW-V02-180101	Revision	Updated cost assumption. Added loadshape. Updated deemed kWh assumption. Replaced deemed gas with algorithm methodology with defaults. Added distinction of defaults for dormitories v multi-family buildings.	Dependent on inputs
		4.3.10 DHW Boiler Tune-up	CI-HWE-DBTU-V01-180101	New	New Measure	N/A
	HVAC	4.4 HVAC End Use	N/A	Revision	Decision not to add new or split existing building types. Added language to allow custom entry for installation within a building or application that does not fit with any of the defined building types.	None
		4.4.1 Air Conditioner Tune-Up	CI-HVC-ACTU-V04-180101	Revision	Addition of deemed methodology based on eQuest modeling.	Dependent on inputs
		4.4.6 Electric Chiller	CI-HVC-CHIL-V05-180101	Revision	Fix of peak savings example.	None
		4.4.9 Heat Pump Systems	CI-HVC-HPSY-V05-180101	Revision	Update to GSHP measure life assumption. Savings example edited to reflect new code and appropriate efficiency ratings.	None
		4.4.11 High Efficiency Furnace	CI-HVC-FRNC-V07-180101	Revision	Clarity on discount rate for future cost.	None
		4.4.13 Package Terminal Air Conditioner (PTAC) and Package Terminal Heat Pump (PTHP)	CI-HVC-PTAC-V08-180101	Revision	Clarity on discount rate for future cost.	None
		4.4.15 Single-Package and Split System Unitary Air Conditioners	CI-HVC-SPUA-V05-180101	Revision	Addition of Early Replacement requirements and assumptions. Update of measure costs. Update of efficiency rating (EER to IEER) for units greater than 65 kBtu/hr. Clarity on discount rate for future cost.	None
		4.4.16 Steam Trap Replacement or Repair	CI-HVC-STRE-V05-180101	Revision	Removal of redundant cost assumptions.	None
		4.4.17 Variable Speed Drives for HVAC Pumps and Cooling	CI-HVC-VSDHP-V04-180101	Revision	Clarity of measure applicability. Removal of 4 fan applications and addition of Cooling Tower Fan assumptions.	None

Volume	End Use	Measure Name	Measure Code	Change Type	Explanation	Impact on Savings
		Tower Fans				
		4.4.19 Demand Controlled Ventilation	CI-HVC-DCV-V04-180101	Revision	Fixing the label of one of the savings tables.	None
		4.4.30 Notched V Belts and Synchronous Belts for HVAC Systems	CI-HVC-NVBE-V03-180101	Revision	Addition of assumptions for Synchronous Belt applications. Clarity added on cost assumption source.	None
		4.4.32 Combined Heat and Power	CI-HVC-CHAP-V02-180101	Revision	Updated Heat Rate values based upon 2014 eGrid data.	Dependent on inputs
		4.4.34 Destratification Fan	CI-HVC-DSFN-V02-180101	Revision	Fix of example to appropriately use wall area.	None
		4.4.35 Economizer Repair and Optimization	CI-HVC-ECRP-V02-160601	Errata	Fixing alignment of algorithms to building types. Clarity on applicability of algorithms.	Dependent on inputs
		4.4.38 Covers and Gap Sealers for Room Air Conditioners	CI-HVC-CRAC-V01-180101	New	New Measure	N/A
		4.4.39 High Temperature Heating and Ventilation (HTHV) Direct Fired Heater	CI-HVC-HTHV-V01-180101	New	New Measure	N/A
	Lighting	4.5 Lighting End Use Table	N/A	Revision	Decision not to add new or split existing building types. Added language to allow custom entry for installation within a building or application that does not fit with any of the defined building types. Addition of method to estimate exterior hours based on business operating hours.	None
		4.5.1 Commercial Compact Fluorescent Lamp	CI-LTG-CCFL-V07-180101	Revision	Removed ENERGY STAR specification. Updated deemed RES v C&I split. Updated lamp and O&M costs. Edits to measure life to account for PY change. Miscellaneous hours of use updated – and corresponding lifetime calculations.	None
		4.5.3 High Performance and Reduced Wattage T8 Fixtures and Lamps	CI-LTG-T8FX-V06-160601	Errata	Fixing multiple fixture wattage assumptions. Timing of retiring T12 as a viable baseline pushed back to 1/1/2019.	Dependent on inputs
		4.5.4 LED Bulbs and Fixtures	CI-LTG-LEDB-V06-180101	Revision	Updated deemed LED wattage based on ENERGY STAR V2.0 specification.	Increase

Volume	End Use	Measure Name	Measure Code	Change Type	Explanation	Impact on Savings
					Updated deemed RES v C&I split. Updated lamp and O&M costs, using new real discount rate. Miscellaneous hours of use updated – and corresponding lifetime and O&M calculations. Lamp life for LED Agricultural Fixtures updated to reflect mix of T8 and EISA lamps.	
		4.5.12 T5 Fixtures and Lamps	CI-LTG-T5FX-V05-180101	Revision	Update of wattage assumption for 4 and 6 lamp T5 High-Bays. Timing of retiring T12 as a viable baseline pushed back to 1/1/2019.	Dependent on inputs
		4.5.14 Commercial Specialty Compact Fluorescent Lamp	CI-LTG-SCFL-V03-180101	Revision	Removed ENERGY STAR specification. Updated deemed RES v C&I split. Miscellaneous hours of use updated – and corresponding lifetime calculations.	None
		4.5.15 LED Open Sign	CI-LTG-OPEN-V01-180101	New	New Measure	N/A
	Refrigeration	4.6.4 Electronically Commutated Motors (ECM) for Walk-in and Reach-in Coolers / Freezers	CI-RFG-ECMF-V02-180101	Revision	Updated source of savings assumptions. Removal of restriction to combine ECM with Fan Control measures.	Dependent on inputs
		4.6.6 Evaporator Fan Control for Electrically Commutated Motors	CI-RFG-EVPF-V03-180101	Revision	Updated source of savings assumptions. Limited application to Fan Controls on ECM motors to allow savings to be claimed for both 4.6.4 and 4.6.6. Additional reference added.	Dependent on inputs
		4.6.7 Strip Curtain for Walk in Coolers and Freezers	CI-RFG-CRTN-V04-180101	Revision	Update savings to be based upon square footage of cooler / freezer. Update of efficient equipment definition. Update to lifetime assumption. Update of cost to be per sq ft of door opening.	Dependent on inputs
		4.6.10 High Speed Roll Up Doors	CI-RFG-HSRD-V01-180101	New	New Measure	N/A
	Miscellaneous	4.8.5 High Speed Clothes Washer	CI-MSC-HSCW-V01-180101	New	New Measure	N/A
		4.8.6 ENERGY STAR Computers	CI-MSC-COMP-V01-180101	New	New Measure	N/A
		4.8.7 Advance Power Strip – Tier 1 Commercial	CI-MSC-APSC-V01-180101	New	New Measure	N/A

Volume	End Use	Measure Name	Measure Code	Change Type	Explanation	Impact on Savings
Volume 3: Residential		4.8.8 High Efficiency Transformer	CI-MSC-TRNS-V01-180101	New	New Measure	N/A
		4.8.9 High Efficiency Battery Charger	CI-MSC-BACH-V01-180101	New	New Measure	N/A
	Appliances	5.1.6 ENERGY STAR and CEE Tier 2 Refrigerator	RS-APL-ESRE-V05-180101	Revision	Clarity on discount rate for future cost.	None
		5.1.7 ENERGY STAR Room Air Conditioner	RS-APL-ESRA-V06-180101	Revision	Clarity on discount rate for future cost.	None
		5.1.11 ENERGY STAR Water Coolers	RS-APL-WTCL-V01-180101	New	New Measure	N/A
	Consumer Electronics	5.2.1 Advanced Power Strip Tier 1	RS-CEL-SSTR-V03-180101	Revision	Update to lifetime and cost assumption. Addition of Efficiency Kit assumptions. Addition of unknown # of plug assumptions.	None
		5.2.2 Tier 2 Advanced Power Strip	RS-CEL-APS2-V02-180101	Revision	Update to Baseline AV Energy assumption and resulting savings values. Removal of In Service Rate as single assumption in TRM to be additional vendor specific assumption in Classification document. Fixing column heading for BaselineEnergy variable.	Decrease
		Attachment Advanced Power Strip Tier 2 Product Classification	N/A	Revision	Addition of product specific In Service Rate assumption.	N/A
	HVAC	5.3.1 Air Source Heat Pump	RS-HVC-ASHP-V07-180101	Revision	Cost assumption update.	None
		5.3.3 Central Air Conditioning	RS-HVC-CAC1-V07-180101	Revision	Cost assumption update.	None
		5.3.6 Gas High Efficiency Boiler	RS-HVC-GHEB-V06-180101	Revision	Clarity on discount rate for future cost.	None
		5.3.7 Gas High Efficiency Furnace	RS-HVC-GHEF-V07-180101	Revision	Addition of Verified Quality Installation assumptions. Clarity on discount rate for future cost.	None
		5.3.8 Ground Source Heat Pump	RS-HVC-GSHP-V07-180101	Revision	Correction of typos in natural gas algorithms. Addition of per ton Central AC cost assumption. Clarity on discount rate for future cost.	None
		5.3.11 Programmable Thermostat	RS-HVC-PROG-V04-180101	Revision	Update to unknown electric heat percentage assumption.	None
		5.3.12 Ductless Heat Pumps	RS-HVC-DHP-V05-180101	Revision	Allowance of whole house application. Addition of TOS, NC and Early Replacement applications.	Dependent on inputs

Volume	End Use	Measure Name	Measure Code	Change Type	Explanation	Impact on Savings
					Addition of fuel switch algorithms and 'Cost effectiveness screening and load reduction forecasting when fuel switching' section. CF based on whole house v zonal application. Updated Heat Rate values based upon 2014 eGrid data. Clarity on discount rate for future cost.	
		5.3.13 Residential Furnace Tune Up	RS-HVC-FTUN-V03-180101	Revision	Addition of Verified Quality Maintenance assumptions. Addition of AFUE variable definition.	None
		5.3.15 ENERGY STAR Ceiling Fan	RS-HVC-CFAN-V02-180101	Revision	Edits to measure life to account for PY change.	None
		5.3.16 Advanced Thermostats	RS-HVC-ADTH-V02-180101	Revision	Addition of footnote explaining collaborative evaluation effort for future Addition of %AC assumption in demand calculation. Update to unknown electric heat percentage assumption.	None
	Hot Water	5.4.2 Gas Water Heater	RS-HWE-GWHT-V07-180101	Revision	Clarity on discount rate for future cost.	None
		5.4.3 Heat Pump Water Heaters	RS-HWE-HPWH-V06-180101	Revision	Update to COP estimates for electric heat to account for duct losses.	Dependent on inputs
		5.4.4 Low Flow Faucet Aerators	RS-HWE-LFFA-V06-180101	Revision	Clarification of direct install costs being full installation cost (including labor).	None
		5.4.5 Low Flow Showerheads	RS-HWE-LFSH-V05-180101	Revision	Clarification of direct install costs being full installation cost (including labor).	None
		5.4.6 Water Heater Temperature Setback	RS-HWE-TMPS-V05-160601	Errata	Addition of In Service Rate for Efficiency Kits	Decrease
		5.4.8 Thermostatic Restrictor Shower Valve	RS-HWE-TRVA-V03-180101	Revision	Clarification of direct install costs being full installation cost (including labor).	None
		5.4.9 Shower Timer	RS-DHW-SHTM-V01-180101	New	New Measure	N/A
	Lighting	5.5.1 Compact Fluorescent Lamp	RS-LTG-ESCF-V06-180101	Revision	Removed ENERGY STAR specification. Updated deemed RES v C&I split. Update In Service Rate assumptions. Clarification that KITS leakage should be through evaluation. Add Leakage deemed values for TOS measures. Updated lamp and O&M costs. Edits to measure life to account for PY change. Update to COP estimates for electric heat to account for duct losses.	Dependent on inputs

Volume	End Use	Measure Name	Measure Code	Change Type	Explanation	Impact on Savings
		5.5.2 Specialty Compact Fluorescent Lamp	RS-LTG-ESCC-V05-180101	Revision	Removed ENERGY STAR specification. Updated deemed RES v C&I split. Update In Service Rate assumptions. Clarification that KITS leakage should be through evaluation. Add Leakage deemed values for TOS measures. Edits to measure life to account for PY change. Update to COP estimates for electric heat to account for duct losses.	Dependent on inputs
		5.5.3 ENERGY STAR Torchiere	RS-LTG-ESTO-V04-180101	Revision	Addition of leakage deemed values for TOS measures. Update to COP estimates for electric heat to account for duct losses.	Dependent on inputs
		5.5.4 Exterior Hardwired Compact Fluorescent Lamp (CFL)	RS-LRG-EFOX-V06-180101	Revision	Addition of leakage deemed values for TOS measures. As per TAC decision, these are estimated as 50% of the lamp assumptions. Edits to measure life to account for PY change. Replacement of NPV calc for O&M with lifetime/cost since no baseline shift.	Dependent on inputs
		5.5.5 Interior Hardwired Compact Fluorescent Lamp (CFL)	RS-LTG-IFIX-V06-180101	Revision	Addition of leakage deemed values for TOS measures. As per TAC decision, these are estimated as 50% of the lamp assumptions. Update to COP estimates for electric heat to account for duct losses. Edits to measure life to account for PY change. Replacement of NPV calc for O&M with lifetime/cost since no baseline shift.	Dependent on inputs
		5.5.6 LED Specialty Lamps	RS-LTG-LEDD-V07-180101	Revision	Updated deemed RES v C&I split. Update In Service Rate assumptions. Clarification that KITS leakage should be through evaluation. Add Leakage deemed values for TOS measures. Updated lamp and O&M costs. Addition of NPV calc for EISA compliant lamps. Added Early Replacement assumptions. Update to COP estimates for electric heat to account for duct losses.	Dependent on inputs
		5.5.7 LED Exit Sign	RS-LTG-LEDE-V02-180101	Revision	Update to COP estimates for electric heat to account for duct losses	Dependent on inputs
		5.5.8 LED Screw Based Omnidirectional Bulbs	RS-LTG-LEDA-V05-180101	Revision	Updated deemed LED wattage based on ENERGY STAR V2.0 specification. Updated deemed RES v C&I split.	Dependent on inputs

Volume	End Use	Measure Name	Measure Code	Change Type	Explanation	Impact on Savings
					Update In Service Rate assumptions. Clarification that KITS leakage should be through evaluation. Add Leakage deemed values for TOS measures. Updated lamp and O&M costs, using new real discount rate. Added Early Replacement assumptions. Update to COP estimates for electric heat to account for duct losses.	
	Shell	5.6.1 Air Sealing	RS-SHL-AIRS-V06-180101	Revision	Update to COP estimate for '2015 on' electric heat to account for duct losses.	Dependent on inputs
		5.6.2 Basement Sidewall Insulation	RS-SHL-BINS-V08-180101	Revision	Update to COP estimate for '2015 on' electric heat to account for duct losses.	Dependent on inputs
		5.6.3 Floor Insulation Above Crawlspace	RS-SHL-FINS-V08-180101	Revision	Update to COP estimate for '2015 on' electric heat to account for duct losses.	Dependent on inputs
		5.6.4 Wall and Ceiling/Attic Insulation	RS-SHL-AINS-V07-180101	Revision	Update to COP estimate for '2015 on' electric heat to account for duct losses.	Dependent on inputs
	Miscellaneous	5.7.1 Residential High Efficiency Pool Pumps	RS-MSC-RPLP-V01-180101	New	New Measure	N/A
Volume 4: Cross Cutting Measures	Behavior	6.1.1 Behavior	CC-BEH-BEHP-V02-160601	Errata	Addition of text to clarify persistence calculations; update of electric persistence factors. Text referring to weather adjustments clarified to give examples rather than a standardized approach for the adjustment. Updated persistence values to reflect a non-linear decay curve for electric savings persistence based on Navigant's ComEd studies; updated examples and tables to include this change. Updated text to change the effective date of behavioral persistence measure applicability from 6-1-17 to 1-1-18, specifying that all residential HERs-type programs implemented prior to January 1, 2018 will assume a one-year measure life. Clarified that the persistence values in this version of the protocol are specific to residential home energy reports (HERs)-type programs and provided a definition of Residential HERs-type programs.	Dependent on inputs

Table 1.4: Summary of Attachment A: IL-NTG Methods Revisions

IL-TRM Volume	Sectors	Protocol Name	Change Type	Explanation
Vol. 4	All Sectors	Residential Cross-Cutting Approaches Residential Nonparticipant Spillover Residential Upstream Lighting Residential New Construction Core Non-Residential Spillover Protocol Core Participant Spillover Protocol	Revision	Lowered spillover and nonparticipant spillover threshold to >5 (on a 0-10 scale)
Vol. 4	All Sectors	Residential Nonparticipant Spillover Prescriptive Rebates (with no audit) Single Family Home Energy Audit Multifamily Residential New Construction Core Non-Residential Spillover Protocol Core Participant Spillover Protocol	Revision	Updated all 0-10 scales using “not at all” and “extremely” anchor definitions
Vol. 4	Commercial, Industrial, and Public Sectors	Core Non-Residential Free Ridership Protocol	Revision	Added Quality Control Review language. Eliminated the Program Components 2 score variant under which a ‘normalized’ score is calculated. Eliminated the Core Free Ridership Option 3 algorithm specification. Added language regarding adjusting survey wording and adding consistency checks. Deleted Public Sector Planning language noting continued Working Group deliberation on Public Sector approach and documentation.
Vol. 4	Commercial, Industrial, and Public Sectors	Core Nonparticipant Spillover Protocol – Measured from End Users	New	Added new Nonparticipant Spillover Protocol. This protocol incorporates the same spillover threshold of >5 as is used in the Participant Spillover Protocol.
Vol. 4	Commercial, Industrial, and Public Sectors	Changed section name from Training and Technical Assistance Protocol to Technical Assistance Protocol	Revision	Revised to use Core Nonresidential Free Ridership Scoring Algorithm with program and non-program components adapted to specific features relevant to Technical Assistance Programs. Impacts of Training courses are to be counted as Spillover
Vol. 4	Residential and Low Income Sectors	Nonparticipant Spillover Measured from Customers	Revision	Updated methodology for collecting nonparticipant spillover data Removed previous program experience as a potential driver

IL-TRM Volume	Sectors	Protocol Name	Change Type	Explanation
				of program influence
Vol. 4	Residential and Low Income Sectors	Residential Upstream Lighting	Revision	Added more detailed spillover and nonparticipant spillover instructions and scoring algorithms to the protocol
Vol. 4	Residential and Low Income Sectors	Prescriptive Rebates (with no audit)	Revision	Added contractor recommendation as a potential driver of program influence Updated phrasing of “prior plans” component of the Program Influence Score and its adjustment on the rebate influence Removed early replacement text and scoring algorithm
Vol. 4	Residential and Low Income Sectors	Prescriptive Rebates (with no audit) Single Family Home Energy Audit Multifamily Energy Saving Kits and Elementary Education	Revision	Updated the scoring algorithm to use the minimum of the timing, efficiency, and quantity scores to calculate the Non-Program Score
Vol. 4	Cross-Sector	Behavioral Protocol, Randomized Controlled Trials	Revision	Added language to address negative uplift methodology issue to ensure consistency across evaluations

1.3 Enabling ICC Policy

This Illinois Statewide Technical Reference Manual (TRM) was developed to comply with the Illinois Commerce Commission (ICC or Commission) Final Orders from the electric and gas Utilities⁶ Energy Efficiency Plan dockets. In the Final Orders, the ICC required the utilities to work with the Illinois Department of Commerce and Economic Opportunity (DCEO) and the Illinois Energy Efficiency Stakeholder Advisory Group (SAG) to develop a statewide TRM. See, e.g., ComEd's Final Order (*Docket No. 10-0570, Final Order*⁷ at 59-60, December 21, 2010); Ameren's Final Order (*Docket No. 10-0568, Order on Rehearing*⁸ at 19, May 24, 2011); Peoples Gas/North Shore Gas' Final Order (*Docket No. 10-0564, Final Order*⁹ at 76, May 24, 2011), and Nicor's Final Order (*Docket No. 10-0562, Final Order*¹⁰ at 30, May 24, 2011).

As directed in the Utilities' Efficiency Plan Orders, the SAG had the opportunity to, and also participated in, every aspect of the development of the TRM. Interested members of the SAG participated in weekly teleconferences to review, comment, and participate in the development of the TRM. The active participants in the TRM were designated as the "Technical Advisory Committee" (TAC). The TAC participants include representatives from the following organizations:

- the Utilities (ComEd, Ameren IL, Nicor Gas, Peoples Gas/North Shore Gas),
- DCEO, Implementation contractors (Applied Proactive Technologies (APT), CLEAResult, Conservation Services Group, Elevate Energy, Franklin Energy, GDS Associates, PECL, 360 Energy Group),
- Illinois Department of Commerce and Economic Opportunity (DCEO),
- the independent evaluators (ADM Associates, The Cadmus Group, Itron, Navigant Consulting, Michael's Engineering, Opinion Dynamics Corporation),
- ICC Staff,
- the Illinois Attorney General's Office (AG),
- Natural Resources Defense Council (NRDC),
- the Environmental Law and Policy Center (ELPC),
- the Citizen's Utility Board (CUB),
- The University of Illinois at Chicago,
- Future Energy Enterprises,
- Selective participants including; Geothermal Alliance of Illinois, the Geothermal Exchange Organization, Embertec and TrickleStar.

1.4 Development Process

The first edition of the IL-TRM was approved by the Commission in ICC Docket No. 12-0528¹¹. The second edition of the IL-TRM was approved by the Commission in ICC Docket No. 13-0437¹². The policies surrounding the applicability and use of the IL-TRM in planning, implementation, and evaluation were established by the Commission in ICC

⁶ The Illinois Utilities subject to this TRM include: Ameren Illinois Company d/b/a Ameren Illinois (Ameren), Commonwealth Edison Company (ComEd), The Peoples Gas Light and Coke Company and North Shore Gas Company, and Northern Illinois Gas Company d/b/a Nicor Gas.

⁷ <http://www.icc.illinois.gov/docket/files.aspx?no=10-0570&docId=159809>

⁸ <http://www.icc.illinois.gov/docket/files.aspx?no=10-0568&docId=167031>

⁹ <http://www.icc.illinois.gov/docket/files.aspx?no=10-0564&docId=167023>

¹⁰ <http://www.icc.illinois.gov/docket/files.aspx?no=10-0562&docId=167027>

¹¹ <http://www.icc.illinois.gov/docket/files.aspx?no=12-0528&docId=187554>

¹² <http://www.icc.illinois.gov/docket/files.aspx?no=13-0437&docId=200492>

Docket No. 13-0077¹³. The Commission extended these policies, including the applicability of the IL-TRM, to the Section 16-111.5B energy efficiency programs in ICC Docket No. 14-0588¹⁴ and most recently in ICC Docket No. 15-0541¹⁵, in order to increase certainty for all parties. The third edition of the IL-TRM was approved by the Commission in ICC Docket No. 14-0189¹⁶. The fourth edition of the IL-TRM was approved by the Commission in ICC Docket No. 15-0187¹⁷. The fifth edition of the IL-TRM was approved by the Commission in ICC Docket No. 16-0171¹⁸. This document represents the sixth edition of the IL-TRM and it applies to Section 8-103B and Section 8-104 energy efficiency programs. It contains a series of new measures, as well as a series of errata items¹⁹ and updates to existing measures that were already present in the first five editions. Like the previous editions, it is a result of an ongoing review process involving the Illinois Commerce Commission (ICC) Staff (Staff or ICC Staff), the Utilities, DCEO, the Evaluators, the SAG TAC, and the SAG. VEIC meets with the SAG and/or the TRM TAC at least once each month to create a high level of transparency and vetting in the development of this TRM.

Measure requests that are submitted by interested parties are ranked based on the following criteria to determine the approximate priority level for order of inclusion in the TRM:

1. High Priority
 - a. For those existing measures that make up a significant portion of a utilities' portfolio and/or where the impact of the requested change is high
 - b. For new measures where plans are in place to implement in the next program year
2. Medium Priority
 - a. For existing measures that are a less significant percent of a utilities' portfolio and value change will not have a significant impact
 - b. For new measures where a savings value is estimated but implementation plans not yet developed
3. Low Priority
 - a. For existing measures that represent a very small percent of a utilities' portfolio
 - b. For new measures that are just beginning to be explored and will not be implemented in the next program year

These rankings are used to align budget and schedule constraints with desired updates from the TRM.

As measure requests are finalized leading up to the next update of the TRM, weekly TAC meetings are often scheduled to maximize the level of collaboration and visibility into the measure characterization process. Where consensus does not emerge on specific measures or issues, those items are identified in a memo, and are not included in the TRM. As a result, this TRM represents a broad consensus amongst the SAG and TAC participants. In

¹³<http://www.icc.illinois.gov/docket/files.aspx?no=13-0077&docId=203903>;

<http://www.icc.illinois.gov/docket/files.aspx?no=13-0077&docId=195913>;

<http://www.icc.illinois.gov/downloads/public/edocket/339744.pdf>

¹⁴ ICC Docket No. 14-0588, [Final Order](#) at 227, December 17, 2014.

The adopted [consensus language](#) concerning the IL-TRM and its applicability to future Section 16-111.5B energy efficiency programs can be accessed from the following link:

<http://www.icc.illinois.gov/downloads/public/June%2018%202014%20Consensus%20Language%20for%20Section%2016-111.5B%20Oversight%20and%20Evaluation%20Responsibility%20Energy%20Efficiency%20Issues.pdf>

¹⁵ ICC Docket No. 15-0541, [Final Order](#) at 36, 82-83, December 16, 2015.

¹⁶ <http://www.icc.illinois.gov/docket/files.aspx?no=14-0189&docId=210478>

[http://www.icc.illinois.gov/downloads/public/Illinois Statewide TRM Effective 060114 Version 3.0 022414 Clean.pdf](http://www.icc.illinois.gov/downloads/public/Illinois%20Statewide%20TRM%20Effective%20060114%20Version%203.0%2022414%20Clean.pdf)

¹⁷ <http://www.icc.illinois.gov/docket/files.aspx?no=15-0187&docId=226161>

[http://www.icc.illinois.gov/downloads/public/Illinois Statewide TRM Effective 060115 Final 022415 Clean.pdf](http://www.icc.illinois.gov/downloads/public/Illinois%20Statewide%20TRM%20Effective%20060115%20Final%2022415%20Clean.pdf)

¹⁸ <https://www.icc.illinois.gov/docket/files.aspx?no=16-0171&docId=239985> <https://www.icc.illinois.gov/downloads/public/IL-TRM%20Version%205.0%20dated%20February%202011.%202016%20Final%20-%20Compiled%20Volumes%201-4.pdf>

¹⁹ Errata as well as links to the official IL-TRM documents, dockets, and policy documents are available on the following ICC webpage: <http://www.icc.illinois.gov/Electricity/programs/TRM.aspx>

keeping with the goal of transparency, all of the comments and their status to-date are available through the TAC SharePoint web site, <https://portal.veic.org>.

For each measure characterization, this TRM includes engineering algorithm(s) and a value(s) for each parameter in the equation(s). These parameters have values that fall into one of three categories: a single deemed value, a lookup table of deemed values or an actual value such as the capacity of the equipment. The TRM makes extensive use of lookup tables because they allow for an appropriate level of measure streamlining and customization within the context of an otherwise prescriptive measure.

Accuracy is the overarching principle that governs what value to use for each parameter. When it is explicitly allowed within the text of the measure characterization, the preferred value is the actual or on-site value for the individual measure being implemented. The *deemed values*²⁰ in the lookup tables are the next most accurate choice, and in the absence of either an actual value or an appropriate value in a lookup table, the single, *deemed value* should be used. As a result, this single, *deemed value* can be thought of as a default value for that particular input to the algorithm.

A single *deemed savings estimate* is produced by any given combination of an algorithm and the allowable input values for each of its parameters. In cases where lookup tables are provided, there is a range of deemed savings estimates that are possible, depending on site-specific factors such as equipment capacity, location and building type.

Algorithms and their parameter values are included for calculating estimated:

- Gross annual electric energy savings (kWh)
- Gross annual natural gas energy savings (therms)
- Gross electric summer coincident peak demand savings (kW)

To support cost-effectiveness calculations, parameter values are also included for:

- Incremental costs (\$)
- Measure life (years)
- Operation and maintenance costs (\$)
- Water (gal) and other resource savings where appropriate.

1.4.1 Reliability Review

The process of incorporating new and better information into the TRM occurs annually as new measures and errors are identified, program designs change, old measures are dropped from programs, or other external events (such as code and standard changes or new evaluations and other data) warrant a review of assumptions. However, not all measures have updates triggered by such events, and some measures continue to appear in the TRM without ongoing review. Short of proactively identified issues that would trigger an update to a TRM characterization, a regular reliability review should be undertaken to assess that the information in older measures is still relevant and reliable. This review will include a general appraisal of reasonableness and continued program relevancy and an update of any assumptions to reflect new information.

To ensure that measures initially developed in the past and not recently revisited are updated and retired as needed, each measure is given a Review Deadline – a date that triggers a reliability review. This Review Deadline is established for each measure based on factors such as expected revisions to energy codes or federal standards; knowledge of upcoming evaluation or research efforts; knowledge of rapidly changing technology, cost, baselines, or other factors; or expected shifts in current customer practices. No Review Deadline is longer than six years from the date of the

²⁰ Emphasis has been added to denote the difference between a “deemed value” and a “deemed savings estimate”. A deemed value refers to a single input value to an algorithm, while a deemed savings estimate is the result of calculating the end result of all of the values in the savings algorithm.

initial characterization or last update of a measure. The TRM Administrator will propose Review Deadlines for each measure, and they are reviewed and approved by the TAC. The Review Deadline for each measure is indicated in the measure characterization within the TRM. For example, a Review Deadline specified as 1/1/2019 means that the measure will be reviewed no later than the annual IL-TRM update process that occurs in 2018, in advance of the 1/1/2019 Review Deadline. Following a review and/or update, a new Review Deadline will be assigned to that measure.

2 Organizational Structure

The organization of this document follows a three-level format. These levels are designed to define and clarify what the measure is and where it is applied.

1. Market Sectors Volumes²¹

- This level of organization specifies the type of customer the measures apply to, either Commercial and Industrial (provided in Volume 2), Residential (provided in Volume 3) or cross-cutting measures, such as Behavior Persistence (provided in Volume 4, together with Attachments including the documentation of Illinois Statewide Net-to-Gross methodologies).
- Answers the question, “What category best describes the customer?”

2. End-use Category

- This level of organization represents most of the major end-use categories for which an efficient alternative exists. The following table lists all of the end-use categories in this version of the TRM.
- Answers the question, “To what end-use category does the measure apply?”

Table 2.1: End-Use Categories in the TRM²²

Volume 2: Commercial and Industrial Market Sector	Volume 3: Residential Market Sector	Volume 4: Cross-Cutting Measures and Attachments
Agricultural Equipment	Appliances	Behavior
Food Service Equipment	Consumer Electronics	
Hot Water	Hot Water	
HVAC	HVAC	
Lighting	Lighting	
Refrigeration	Shell	
Compressed Air	Miscellaneous	
Miscellaneous		

3. Measure & Technology

- This level of organization represents individual efficient measures such as CFL lighting and LED lighting, both of which are individual technologies within the Lighting end-use category.
- Answers the question, “What technology defines the measure?”

This organizational structure is silent on which fuel the measure is designed to save; electricity or natural gas. By organizing the TRM this way, measures that save on both fuels do not need to be repeated. As a result, the TRM will be easier to use and to maintain.

2.1 Measure Code Specification

In order to uniquely identify each measure in the TRM, abbreviations for the major organizational elements of the TRM have been established. When these abbreviations are combined and delimited by a dash ('-') a unique, 18-character alphanumeric code is formed that can be used for tracking the measures and their associated savings estimates. Measure codes appear at the end of each measure and are structured using five parts.

Code Structure = Market + End-use Category + Measure + Version # + Effective Date

²¹ Note that the Public sector buildings and low income measures are not listed as a separate Market Sector. The Public building type is one of a series of building types that are included in the appropriate measures in the Commercial and Industrial Sector.

²² Please note that this is not an exhaustive list of end-uses and that others may be included in future versions of the TRM.

For example, the commercial boiler measure is coded: “CI-HVC-BLR_-V01-120601”

Table 2.2: Measure Code Specification Key

Market (@@)	End-use (@@@)	Measure (@@@@)	Version (V##)	Effective Date
CI (C&I)	AGE (Agricultural Equipment)	BLR_	V01	YYMMDD
RS (Residential)	APL (Appliances)	T5FX	V02	YYMMDD
CC (Cross-Cutting)	BEH (Behavior)	T8FX	V03	YYMMDD
	CEL (Consumer Electronics)
	CPA (Compressed Air)			
	FSE (Food Service Equipment)			
	HVC (HVAC)			
	HWE (Hot Water)			
	LTG (Lighting)			
	MSC (Miscellaneous)			
	RFG (Refrigeration)			
	SHL (Shell)			

2.2 Components of TRM Measure Characterizations

Each measure characterization uses a standardized format that includes at least the following components. Measures that have a higher level of complexity may have additional components, but also follow the same format, flow and function.

DESCRIPTION

Brief description of measure stating how it saves energy, the markets it serves and any limitations to its applicability.

DEFINITION OF EFFICIENT EQUIPMENT

Clear definition of the criteria for the efficient equipment used to determine delta savings. Including any standards or ratings if appropriate.

DEFINITION OF BASELINE EQUIPMENT

Clear definition of the efficiency level of the baseline equipment used to determine delta savings including any standards or ratings if appropriate. If a Time of Sale measure the baseline will be new base level equipment (to replace existing equipment at the end of its useful life or for a new building). For Early Replacement or Early Retirement measures the baseline is the existing working piece of equipment that is being removed.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected duration in years (or hours) of the savings. If an early replacement measure, the assumed life of the existing unit is also provided.

DEEMED MEASURE COST

For time of sale measures, incremental cost from baseline to efficient is provided. Installation costs should only be included if there is a difference between each efficiency level. For Early Replacement the full equipment and install cost of the efficient installation is provided in addition to the full deferred hypothetical baseline replacement cost.

LOADSHAPE

The appropriate loadshape to apply to electric savings is provided.

COINCIDENCE FACTOR

The summer coincidence factor is provided to estimate the impact of the measure on the utility's system peak – defined as 1PM to hour ending 5PM on non-holiday weekdays, June through August.

Algorithm

CALCULATION OF ENERGY SAVINGS

Algorithms are provided followed by list of assumptions with their definition.

If there are no Input Variables, there will be a finite number of Output values. These will be identified and listed in a table. Where there are custom inputs, an example calculation is often provided to illustrate the algorithm and provide context.

ELECTRIC ENERGY SAVINGS

SUMMER COINCIDENT PEAK DEMAND SAVINGS

NATURAL GAS SAVINGS

WATER IMPACT DESCRIPTIONS AND CALCULATION

DEEMED O&M COST ADJUSTMENT CALCULATION

Only required if the operation and maintenance cost for the efficient case is different to the baseline.

MEASURE CODE

REVIEW DEADLINE

If not otherwise updated as part of an identified new TRM issue request before this Review Deadline, the measure will undergo a reliability review for reasonableness, continued program relevancy, and update of material assumptions during the update cycle prior to this deadline.

2.3 Variable Input Tables

Many of the measures in this TRM require the user to select the appropriate input value from a list of inputs for a given parameter in the savings algorithm. Where the TRM asks the user to select the input, look-up tables of allowable values are provided. For example, a set of input parameters may depend on building type; while a range of values may be given for each parameter, only one value is appropriate for any specific building type. If no table of alternative inputs is provided for a particular parameter, then the single deemed value will be used, unless the measure has a custom allowable input.

2.3.1 C&I Custom Value Use in Measure Implementation

This section defines the requirements for capturing Custom variables that can be used in place of defaults for select assumptions within the prescriptive measures defined in this statewide TRM. This approach is to be used when a variable in a measure formula can be replaced by a verifiable and documented value that is not presented in the TRM. This approach assumes that the algorithms presented in the measure are used as stated and only allows changes to certain variable values and is not a replacement algorithm for the measure. A custom variable is when customer input is provided to define the number or the value is measured at the site. Custom values can also be supplied from product data of the measure installed. In certain cases the custom data can be provided from a documented study or report that is applicable to the measure. Custom variables and potential sources are clearly

defined in the specific measures where “Actual” or “Custom” is noted.

In exceptional cases where the participant, program administrator, and independent evaluator all agree that the TRM algorithm for a particular energy efficiency measure does not accurately characterize the energy efficiency measure within a project due to the complexity in the design and configuration of the particular energy efficiency project, a more comprehensive custom engineering and financial analysis may be used that more accurately incorporates the attributes of the measure in the complex energy efficiency project. In such cases and consistent with Commission policy adopted in ICC Docket No. 13-0077, Program Administrators are subject to retrospective evaluation risk (retroactive adjustments to savings based on ex post evaluation findings) for such projects utilizing customized savings calculations.

2.4 Program Delivery & Baseline Definitions

The measure characterizations in this TRM are not grouped by program delivery type. As a result, the measure characterizations provided include information and assumptions to support savings calculations for the range of program delivery options commonly used for the measure. The organizational significance of this approach is that multiple baselines, incremental costs, O&M costs, measure lives and in-service rates are included in the measure characterization(s) that are delivered under two or more different program designs. Values appropriate for each given program delivery type are clearly specified in the algorithms or in look-up tables within the characterization.

Care has been taken to clearly define in the measure’s description the types of program delivery that the measure characterization is designed to support. However, there are no universally accepted definitions for a particular program type, and the description of the program type(s) may differ by measure. Nevertheless, program delivery types can be generally defined according to the following table. These are the definitions used in the measure descriptions, and, when necessary, individual measure descriptions may further refine and clarify these definitions of program delivery type.

Table 2.3: Program Delivery Types

Program	Attributes
Time of Sale (TOS)	Definition: A program in which the customer is incented to purchase or install higher efficiency equipment than if the program had not existed. This may include retail rebate (coupon) programs, upstream buydown programs, online store programs or contractor based programs as examples. Baseline = New equipment. Efficient Case = New, premium efficiency equipment above federal and state codes and standard industry practice. Example: CFL rebate
New Construction (NC)	Definition: A program that intervenes during building design to support the use of more-efficient equipment and construction practices. Baseline = Building code or federal standards. Efficient Case = The program’s level of building specification Example: Building shell and mechanical measures
Retrofit (RF)	Definition: A program that upgrades existing equipment before the end of its useful life. Baseline = Existing equipment or the existing condition of the building or equipment. A single baseline applies over the measure’s life. Efficient Case = New, premium efficiency equipment above federal and state codes and standard industry practice. Example: Air sealing and insulation
Early Replacement (EREP)	Definition: A program that replaces existing equipment before the end of its expected life. Baseline = Dual; it begins as the existing equipment and shifts to new baseline equipment after the expected life of the existing equipment is over. Efficient Case = New, premium efficiency equipment above federal and state codes and standard industry practice. Example: Refrigerators, freezers

Program	Attributes
Early Retirement (ERET)	Definition: A program that retires duplicative equipment before its expected life is over. Baseline = The existing equipment, which is retired and not replaced. Efficient Case = Zero because the unit is retired. Example: Appliance recycling
Direct Install (DI)	Definition: A program where measures are installed during a site visit. Baseline = Existing equipment. Efficient Case = New, premium efficiency equipment above federal and state codes and standard industry practice. Example: Lighting and low-flow hot water measures
Efficiency Kits (KITS)	Definition: A program where measures are provided free of charge to a customer in an Efficiency Kit. Baseline = Existing equipment. Efficient Case = New, premium efficiency equipment above federal and state codes and standard industry practice. Example: Lighting and low-flow hot water measures

The concept and definition of the baseline is a key element of every measure characterization and is directly related to the program delivery type. Without a clear definition of the baseline, the savings algorithms cannot be adequately specified and subsequent evaluation efforts would be hampered. As a result, each measure has a detailed description (and in many cases, specification) of the specific baseline that should be used to calculate savings. Baselines in this TRM fall into one of the following four categories, and are organized within each measure characterization by the program delivery type to which it applies.

1. **Building Code:** As defined by the minimum specifications required under state energy code or applicable federal standards.
2. **Existing Equipment:** As determined by the most representative (or average) example of equipment that is in the existing stock. Existing equipment baselines apply over the equipment's remaining useful life.
3. **New Equipment:** As determined by the equipment that represents standard practice in the current market environment. New equipment baselines apply over the effective useful life of the measure.
4. **Dual Baseline:** A baseline that begins as the existing equipment and shifts to new equipment after the expected life of the existing equipment is over

3 Assumptions

The information contained in this TRM contains VEIC's recommendations for the content of the Illinois TRM. Sources that are cited within the TRM have been chosen based on two priorities, geography and age. Whenever possible and appropriate, VEIC has incorporated Illinois-specific information into each measure characterization. The Business TRM documents from Ameren and ComEd were reviewed, as well as program and measure specific data from evaluations, efficiency plans, and working documents.

The assumptions for these characterizations rest on our understanding of the information available. In each case, the available Illinois and Midwest-specific information was reviewed, including evaluations and support material provided by the Illinois Utilities.

When Illinois or region-specific evaluations or data were not available, best practice research and data from other jurisdictions was used, often from west and east-coast states that have allocated large amounts of funding to evaluation work and to refining their measure characterization parameters. As a result, much of the most-defensible information originates from these regions. In every case, VEIC used the most recent, well-designed, and best-supported studies and only if it was appropriate to generalize their conclusions to the Illinois programs.

3.1 Footnotes & Documentation of Sources

Each new and updated measure characterization is supported by a work paper, which is posted to the SharePoint web site (<https://portal.veic.org>).²³ Both the work paper and the measure characterizations themselves use footnotes to document the references that have been used to characterize the technology. The reference documents are too numerous to include in an Appendix and have instead been posted to the TRM's Sharepoint website. These files can be found in the 'Sources and Reference Documents' folder in the main directory, and are also posted to the SAG's public web site (<http://www.ilsag.info/technical-reference-manual.html>).

3.2 General Savings Assumptions

The TRM savings estimates are expected to serve as average, representative values, or ways to calculate savings based on program-specific information. All information is presented on a per-measure basis. In using the measure-specific information in the TRM, it is helpful to keep the following notes in mind.

- All estimates of energy (kWh or therms) and peak (kW) savings are for first-year savings, not lifetime savings.
- Unless otherwise noted, measure life is defined to be the life of an energy consuming measure, including its equipment life and measure persistence.
- Where deemed values for savings are provided, they represent the average energy (kWh or therms) or peak (kW) savings that could be expected from the average of all measures that might be installed in Illinois in the program year.
- In general, the baselines included in the TRM are intended to represent average conditions in Illinois. Some are based on data from the state, such as household consumption characteristics provided by the Energy Information Administration. Some are extrapolated from other areas, when Illinois data are not available.

²³ To gain access to the SharePoint web site, please contact the TRM Administrator at iltrmadministrator@veic.org.

3.3 Shifting Baseline Assumptions

The TRM anticipates the effects of changes in efficiency codes and standards on affected measures. When these changes take effect, a shift in the baseline is usually required. This complicates the measure savings estimation somewhat, and will be handled in future versions of the TRM by describing the choice of and reasoning behind a shifting baseline assumption. In this version of the TRM, this applies to CFLs and T5/T8 Linear Fluorescents, Furnaces and Early Replacement Measures.

3.3.1 CFL and T5/T8 Linear Fluorescents Baseline Assumptions

Specific reductions in savings have been incorporated for CFL measures that relate to the shift in appropriate baseline due to changes in Federal Standards for lighting products. Federal legislation (stemming from the Energy Independence and Security Act of 2007) mandates a phase-in process beginning in 2012 for all general-purpose light bulbs between 40W and 100W to be approximately 30% more energy efficient than current incandescent bulbs, in essence beginning the phase-out of the current style, or “standard”, incandescent bulbs. In 2012, standard 100W incandescent bulbs will no longer be manufactured, followed by restrictions on standard 75W bulbs in 2013 and 60W and 40W bulbs in 2014. The baseline for the CFL measure in the corresponding program years starting June 1 each year will therefore become bulbs (improved or “efficient” incandescent, or halogen) that meet the new standard and have the same lumen equivalency. Those products can take several different forms we can envision now and perhaps others we do not yet know about. Halogens are one of those possibilities and have been chosen to represent a baseline at that time. To account for this shifting baseline, annual savings are reduced within the lifetime of the measure. Other lighting measures will also have baseline shifts (for example screw based LED and CFL fixtures) that will result in significant impacts to annual estimated savings in later years.

In July 14, 2012, Federal Standards were enacted that were expected to eliminate T-12s as an option for linear fluorescent fixtures. Through v3.0 of the TRM, it was assumed that the T-12 would no longer be baseline for retrofits from 1/1/2016. However, due to significant loopholes in the legislation, T-12 compliant product is still freely available and in Illinois T-12s continue to hold a significant share of the existing and replacement lamp market. Therefore the timing of the sunset of T-12s as a viable baseline was pushed back in v6.0 to 1/1/2019, and will be revisited in future update sessions.

3.3.2 Early Replacement Baseline Assumptions

A series of measures have an option to choose an Early Replacement Baseline if the following conditions are met:

Early Replacement determination will be based on meeting the following conditions:

- The existing unit is operational when replaced, or
- The existing unit requires minor repairs (see table below) ²⁴.

Existing System	Maximum repair cost
Air Source Heat Pump	\$918
Central Air Conditioner	\$734
Boiler	\$709
Furnace	\$528
Ground Source Heat Pump	<\$249 per ton

- All other conditions will be considered Time of Sale.

The Baseline efficiency of the existing unit replaced:

²⁴ The Technical Advisory Committee agreed that if the cost of repair is less than 20% of the new baseline replacement cost it can be considered early replacement.

- If the efficiency of the existing unit is less than the maximum shown below, the Baseline efficiency is the actual efficiency value of the unit replaced. If the efficiency is greater than the maximum, the Baseline efficiency is shown in the “New Baseline” column below:

Existing System	Maximum efficiency for Actual	New Baseline
Air Source Heat Pump	10 SEER	14 SEER
Central Air Conditioner	10 SEER	13 SEER
Boiler	75% AFUE	82% AFUE
Furnace	75% AFUE	80% AFUE
Ground Source Heat Pump	10 SEER	13 SEER

- If the operational status, repair cost or efficiency of the existing unit is unknown, the Baseline efficiency is the “New Baseline” column above.

3.3.3 Furnace Baseline

The prior national standard for residential oil and gas furnaces was 78% AFUE. DOE raised the standard in 2007 to 80% AFUE, effective 2015. However, virtually all furnaces on the market have an AFUE of 80% or better, which prompted states and environmental and consumer groups to sue DOE over its 2007 decision. In April 2009, DOE accepted a “voluntary remand” in that litigation. In October 2009, manufacturers and efficiency advocates negotiated an agreement that, for the first time, included different standard levels in three climate regions: the North, South, and Southwest. DOE issued a direct final rule (DFR) in June 2011 reflecting the standard levels in the consensus agreement. The DFR became effective on October 25, 2011 establishing new standards: In the North, most furnaces will be required to have an AFUE of 90%. The 80% AFUE standard for the South and Southwest will remain unchanged at 80%. Oil furnaces will be required to have an AFUE of 83% in all three regions. The amended standards will become effective in May 2013 for non-weatherized furnaces and in January 2015 for weatherized furnaces. DOE estimates that the standards will save about 3.3 quads (quadrillion Btu) of energy over 30 years and yield a net present value of about \$14 billion at a 3 percent discount rate.

Update: On January 14th 2013, the U.S. Department of Energy (DOE) proposed to settle a lawsuit brought by the American Public Gas Association (APGA) that seeks to roll back gas furnace efficiency standards. As a result, the new standards, completed in 2011 and slated to take effect in May 2013, would be eliminated in favor of yet another round of DOE hearings and studies. Even if DOE completes a new rulemaking in two years, it's unlikely to take effect before 2020.²⁵

As a result, each of the furnace measures contains the following language describing the baseline assumption:

“Although the current Federal Standard for gas furnaces is an AFUE rating of 78%, based upon review of available product in the AHRI database, the baseline efficiency for this characterization is assumed to be 80%. The baseline will be adjusted when the Federal Standard is updated.”

3.4 Glossary

Baseline Efficiency: The assumed standard efficiency of equipment, absent an efficiency program.

Building Types²⁶:

Note where a measure installation is within a building or application that does not fit with any of the defined building

²⁵ Appliance Standards Awareness Project, <http://www.appliance-standards.org/product/furnaces>

²⁶ Source: US EPA, www.energystar.gov, Space Type Definitions, or definitions as developed through the Technical Advisory Committee.

types below, the user should apply custom assumptions where it is reasonable to estimate them, else the building of best fit should be utilized.

Building Type	Definition
Assisted Living MultiFamily	Applies to residential buildings of three or more units with staff to assist the occupants. Gross Floor Area should include all fully-enclosed space within the exterior walls of the building(s) including individual rooms or units, wellness centers, exam rooms, community rooms, small shops or service areas for residents and visitors (e.g. hair salons, convenience stores), staff offices, lobbies, atriums, cafeterias, kitchens, storage areas, hallways, basements, stairways, corridors between buildings, and elevator shafts.
Auditorium/Assembly	Applies to any performance space such as a theater, arena, or hall. Gross Floor Area should include all space within the building(s), including seating, stage and backstage areas, food service areas, retail areas, rehearsal studios, administrative/office space, mechanical rooms, storage areas, elevator shafts, and stairwells.
Childcare/Pre-school	Applies to any building providing childcare to pre-kindergarten age children.
College/University	Applies to facility space used for higher education. Relevant buildings include administrative headquarters, residence halls, athletic and recreation facilities, laboratories, etc. The total gross floor area should include all supporting functions such as kitchens used by staff, lobbies, atria, conference rooms and auditoria, fitness areas for staff, storage areas, stairways, elevator shafts, etc.
Convenience Store	Applies to facility space used for the retail sale of a limited selection of food and beverage products. The total gross floor area should include all supporting functions such as kitchens and break rooms used by staff, storage areas (refrigerated and non-refrigerated), and administrative areas.
Elementary School	Applies to a school serving children in any grades from Kindergarten through sixth grade. The total gross floor area should include all supporting functions such as administrative space, conference rooms, kitchens used by staff, lobbies, cafeterias, gymnasiums, auditoria, laboratory classrooms, portable classrooms, greenhouses, stairways, atria, elevator shafts, small landscaping sheds, storage areas, etc.
Exterior	Applies to unconditioned spaces that are outside of the building envelope.
Garage	Applies to unconditioned spaces either attached or detached from the primary building envelope that are not used for living space.
Grocery	Applies to facility space used for the retail sale of food and beverage products. It should not be used by restaurants. The total gross floor area should include all supporting functions such as kitchens and break rooms used by staff, storage areas (refrigerated and non-refrigerated), administrative areas, stairwells, atria, lobbies, etc.
Healthcare Clinic	Applies to a facility space used to provide diagnosis and treatment for medical, dental, or psychiatric outpatient care. Gross Floor Area should include all space within the building(s) including offices, exam rooms, laboratories, lobbies, atriums, conference rooms and auditoriums, employee break rooms and kitchens, rest rooms, elevator shafts, stairways, mechanical rooms, and storage areas.
High School/Middle School	Applies to facility space used as a school building for 7th through 12th grade students. This does not include college or university classroom facilities and laboratories, vocational, technical, or trade schools. The total gross floor area should include all supporting functions such as administrative space, conference rooms, kitchens used by staff, lobbies, cafeterias, gymnasiums, auditoria, laboratory classrooms, portable classrooms, greenhouses, stairways, atria, elevator shafts, small landscaping sheds, storage areas, etc.
Hospital	Applies to a general medical and surgical hospital (including critical access hospitals and children's hospitals) that is either a stand-alone building or a campus of buildings. Spaces more accurately characterized as a Healthcare Clinic should use that definition. The definition of Hospital accounts for all space types that are located within the Hospital

Building Type	Definition
	building/campus, such as medical offices, administrative offices, and skilled nursing. The total floor area should include the aggregate floor area of all buildings on the campus as well as all supporting functions such as: stairways, connecting corridors between buildings, medical offices, exam rooms, laboratories, lobbies, atria, cafeterias, storage areas, elevator shafts, and any space affiliated with emergency medical care, or diagnostic care.
Hotel/Motel Combined (All Spaces)	Applies to buildings that rent overnight accommodations on a room/suite basis, typically including a bath/shower and other facilities in guest rooms. The total gross floor area should include all interior space, including guestrooms, halls, lobbies, atria, food preparation and restaurant space, conference and banquet space, health clubs/spas, indoor pool areas, and laundry facilities, as well as all space used for supporting functions such as elevator shafts, stairways, mechanical rooms, storage areas, employee break rooms, back-of-house offices, etc. Hotel does not apply to fractional ownership properties such as condominiums or vacation timeshares. Hotel properties should be owned by a single entity and have rooms available on a nightly basis. Where distinction between Hotel and Motel is necessary: Hotel: Room entrances and Corridors are located in the <i>interior</i> of the building. Corridors are conditioned spaces. Building can be significantly larger in size/height. Motel: Room entrances and Corridors are located on the <i>exterior</i> of the building. Corridors are not conditioned spaces. Buildings tend to be two to three stories in height.
Hotel/Motel Common Areas	All the common areas open to guests of the hotel such as the lobby, corridors and stairways, and other spaces that may have continuous or large lighting and HVAC hours.
Hotel/Motel Guest Room	Applies to the guest rooms of the hotel or motel. These spaces are occupied intermittently.
Low-use Small Business	Any business type with low (<3000) operating hours (provided as option in lighting measures).
Manufacturing	Applies to buildings that are dedicated to manufacturing activities. Includes light industry buildings characterized by consumer product and component manufacturing and heavy industry buildings typically characterized by a plant that includes a main production area that has high-ceilings and contains heavy equipment used for assembly line production. These building types may be distinguished by categorizing NAICS (SIC) codes according to the needs of the Program Administrator.
Miscellaneous	Applies to spaces that do not fit clearly within any available categories should be designated as “miscellaneous”.
Multifamily-Mid Rise	Applies to residential buildings with up to four floors, including all public and multiuse spaces within the building envelope. Small Multifamily buildings best described as a house should use the residential measure characterizations.
Multifamily-High Rise Combined (All Spaces)	Applies to residential buildings with five or more floors, including all public and multiuse spaces within the building envelope. Gross Floor Area should include all fully-enclosed space within the exterior walls of the building(s) including living space in each unit (including occupied and unoccupied units), interior common areas (e.g. lobbies, offices, community rooms, common kitchens, fitness rooms, indoor pools), hallways, stairwells, elevator shafts, connecting corridors between buildings, storage areas, and mechanical space such as a boiler room. Open air stairwells, breezeways, and other similar areas that are not fully-enclosed should not be included in the Gross Floor Area.
Multifamily-High Rise Common Areas	All the common areas open to occupants of the building such as the lobby, corridors and stairways, and other spaces that may have continuous or high lighting and HVAC hours.
Multifamily-High Rise Residential Units	Applies to the residential units in the building only.
Movie Theater	Applies to buildings used for public or private film screenings. Gross Floor Area should

Building Type	Definition
	include all space within the building(s), including seating areas, lobbies, concession stands, bathrooms, administrative/office space, mechanical rooms, storage areas, elevator shafts, and stairwells.
Office-Low Rise	Applies to facility spaces in buildings with four floors or fewer used for general office, professional, and administrative purposes. The total gross floor area should include all supporting functions such as kitchens used by staff, lobbies, atria, conference rooms and auditoria, fitness areas for staff, storage areas, stairways, elevator shafts, etc.
Office-Mid Rise	Applies to facility spaces in buildings with five to nine floors used for general office, professional, and administrative purposes. The total gross floor area should include all supporting functions such as kitchens used by staff, lobbies, atria, conference rooms and auditoria, fitness areas for staff, storage areas, stairways, elevator shafts, etc.
Office-High Rise	Applies to facility spaces in buildings with ten floors or more used for general office, professional, and administrative purposes. The total gross floor area should include all supporting functions such as kitchens used by staff, lobbies, atria, conference rooms and auditoria, fitness areas for staff, storage areas, stairways, elevator shafts, etc.
Religious Worship/Church	Applies to buildings that are used as places of worship. This includes churches, temples, mosques, synagogues, meetinghouses, or any other buildings that primarily function as a place of religious worship. Gross Floor Area should include all areas inside the building that includes the primary worship area, including food preparation, community rooms, classrooms, and supporting areas such as restrooms, storage areas, hallways, and elevator shafts.
Restaurant	Applies to a subcategory of Retail/Service space that is used to provide commercial food services to individual customers, and includes kitchen, dining, and common areas.
Retail/Service- Department store	Applies to facility space used to conduct the retail sale of consumer product goods. Stores must be at least 30,000 square feet and have an exterior entrance to the public. The total gross floor area should include all supporting functions such as kitchens and break rooms used by staff, storage areas, administrative areas, elevators, stairwells, etc. Retail segments typically included under this definition are: Department Stores, Discount Stores, Supercenters, Warehouse Clubs, Drug Stores, Dollar Stores, Home Center/Hardware Stores, and Apparel/Hard Line Specialty Stores (e.g., books, clothing, office products, toys, home goods, electronics). Retail segments excluded under this definition are: Grocery, Convenience Stores, Automobile Dealerships, and Restaurants.
Retail/Service- Strip Mall	Applies to facility space used to conduct the retail sale of consumer product goods. Stores must less than 30,000 square feet and have an exterior entrance to the public. The total gross floor area should include all supporting functions such as kitchens and break rooms used by staff, storage areas, administrative areas, elevators, stairwells, etc. Retail segments excluded under this definition are: Grocery, Convenience Stores, Automobile Dealerships, and Restaurants.
Warehouse	Applies to unrefrigerated or refrigerated buildings that are used to store goods, manufactured products, merchandise or raw materials. The total gross floor area of Refrigerated Warehouses should include all temperature controlled area designed to store perishable goods or merchandise under refrigeration at temperatures below 50 degrees Fahrenheit. The total gross floor area of Unrefrigerated Warehouses should include space designed to store non-perishable goods and merchandise. Unrefrigerated warehouses also include distribution centers. The total gross floor area of refrigerated and unrefrigerated warehouses should include all supporting functions such as offices, lobbies, stairways, rest rooms, equipment storage areas, elevator shafts, etc. Existing atriums or areas with high ceilings should only include the base floor area that they occupy. The total gross floor area of refrigerated or unrefrigerated warehouse should not include outside loading bays or docks. Self-storage facilities, or facilities that rent individual storage units, are not eligible for a rating using the warehouse model.

Coincidence Factor (CF): Coincidence factors represent the fraction of connected load expected to be coincident with a particular system peak period, on a diversified basis. Coincidence factors are provided for summer peak periods.

Commercial & Industrial: The market sector that includes measures that apply to any of the building types defined in this TRM, which includes multifamily common areas and public housing²⁷.

Connected Load: The maximum wattage of the equipment, under normal operating conditions.

Deemed Value: A value that has been assumed to be representative of the average condition of an input parameter.

Default Value: When a measure indicates that an input to a prescriptive saving algorithm may take on a range of values, an average value is also provided in many cases. This value is considered the default input to the algorithm, and should be used when the other alternatives listed in the measure are not applicable.

End-use Category: A general term used to describe the categories of equipment that provide a service to an individual or building. See Table 2.1.1 for a list of the end-use categories that are incorporated in this TRM.

Energy Efficiency: "Energy efficiency" means measures that reduce the amount of electricity or natural gas consumed in order to achieve a given end use. "Energy efficiency" includes voltage optimization measures that optimize the voltage at points on the electric distribution voltage system and thereby reduce electricity consumption by electric customers' end use devices. "Energy efficiency" also includes measures that reduce the total Btus of electricity, natural gas and other fuels needed to meet the end use or uses (20 ILCS 3855/1-10). For purposes of this Section, "energy efficiency" means measures that reduce the amount of energy required to achieve a given end use. "Energy efficiency" also includes measures that reduce the total Btus of electricity and natural gas needed to meet the end use or uses (220 ILCS 5/8-104(b)).

Equivalent Full Load Hours (EFLH): The equivalent hours that equipment would need to operate at its peak capacity in order to consume its estimated annual kWh consumption (annual kWh/connected kW) or therms.

High Efficiency: General term for technologies and processes that require less energy, water, or other inputs to operate.

Lifetime: The number of years (or hours) that the new high efficiency equipment is expected to function. These are generally based on engineering lives, but sometimes adjusted based on expectations about frequency of removal, remodeling or demolition. Two important distinctions fall under this definition; Effective Useful Life (EUL) and Remaining Useful Life (RUL).

EUL – EUL is based on the manufacturers rating of the effective useful life; how long the equipment will last. For example, a CFL that operates x hours per year will typically have an EUL of y. A house boiler may have a lifetime of 20 years but the EUL is only 15 years since after that time it may be operating at a non-efficient point. An estimate of the median number of years that the measures installed under a program are still in place and operable.

RUL – Applies to retrofit or replacement measures. For example, if an existing working refrigerator is replaced with a high efficiency unit, the RUL is an assumption of how many more years the existing unit would have lasted. As a general rule the RUL is usually assumed to be 1/3 of the EUL.

Load Factor (LF): The fraction of full load (wattage) for which the equipment is typically run.

Measure Cost: The incremental (for time of sale measures) or full cost (both capital and labor for retrofit measures) of implementing the High Efficiency equipment. See Section 3.8 Measure Incremental Cost Definition for full definition.

Measure Description: A detailed description of the technology and the criteria it must meet to be eligible as an energy efficient measure.

²⁷ Measures that apply to the multifamily and public housing building types describe how to handle tenant versus master metered buildings.

Measure: An efficient technology or procedure that results in energy savings as compared to the baseline efficiency.

Residential: The market sector that includes measures that apply only to detached, residential buildings or duplexes.

Operation and Maintenance (O&M) Cost Adjustments: The dollar impact resulting from differences between baseline and efficient case Operation and Maintenance costs.

Operating Hours (HOURS): The annual hours that equipment is expected to operate.

Program: The mode of delivering a particular measure or set of measures to customers. See Table 2.4 for a list of program descriptions that are presently operating in Illinois.

Rating Period Factor (RPF): Percentages for defined times of the year that describe when energy savings will be realized for a specific measure.

Stakeholder Advisory Group (SAG): The Illinois Energy Efficiency Stakeholder Advisory Group (SAG) was first defined in the electric utilities' first energy efficiency Plan Orders to include "... the Utility, DCEO, Staff, the Attorney General, BOMA and CUB and representation from a variety of interests, including residential consumers, business consumers, environmental and energy advocacy organizations, trades and local government... [and] a representative from the ARES (alternative retail electric supplier) community should be included."²⁸ A group of stakeholders who have an interest in Illinois' energy efficiency programs and who meet regularly to share information and work toward consensus on various energy efficiency issues. The Utilities in Illinois have been directed by the ICC to work with the SAG on the development of a statewide TRM.

Table 3.2: Degree-Day Zones and Values by Market Sector

Zone	Residential		C&I		Weather Station / City
	HDD	CDD	HDD	CDD	
1	5,352	820	4,272	2,173	Rockford AP / Rockford
2	5,113	842	4,029	2,181	Chicago O'Hare AP / Chicago
3	4,379	1,108	3,406	2,666	Springfield #2 / Springfield
4	3,378	1,570	2,515	3,358	Belleville SIU RSCH / Belleville
5	3,438	1,370	2,546	3,090	Carbondale Southern IL AP / Marion
Average	4,860	947	3,812	2,362	Weighted by occupied housing units
Base Temp	60F	65F	55F	55F	Year climate normals, 1981-2010

3.5 Electrical Loadshapes (kWh)

Loadshapes are an integral part of the measure characterization and are used to divide energy savings into appropriate periods using Rating Period Factors (RPFs) such that each have variable avoided cost values allocated to them for the purpose of estimating cost effectiveness.

For the purposes of assigning energy savings (kWh) periods, the TRM TAC has agreed to use the industry standards for wholesale power market transactions as shown in the following table.

Table 3.3: On and Off Peak Energy Definitions

Period Category	Period Definition (Central Prevailing Time)
Winter On-Peak Energy	8AM - 11PM, weekdays, Oct – Apr, No NERC holidays

²⁸ ICC Docket No. 07-0540, Final Order at 32-33, February 6, 2008.

<http://www.icc.illinois.gov/downloads/public/edocket/215193.pdf>

Period Category	Period Definition (Central Prevailing Time)
Winter Off-Peak Energy	All other hours
Summer On-Peak Energy	8AM - 11PM, weekdays, May – Sept, No NERC holidays
Summer Off-Peak Energy	All other hours

Loadshapes have been developed for each end-use by assigning Rating Period Factor percentages to each of the four periods above. Two methodologies were used:

1. Itron eShapes data for Missouri, reconciled to Illinois loads and provided by Ameren, were used to calculate the percentage of load in to the four categories above.
2. Where the Itron eShapes data did not provide a particular end-use or specific measure load profile, loadshapes that have been developed over many years by Efficiency Vermont and that have been reviewed by the Vermont Department of Public Service, were adjusted to match Illinois period definitions. Note – no weather sensitive loadshapes were based on this method. Any of these load profiles that relate to High Impact Measures should be an area of future evaluation.

The following pages provide the loadshape values for most measures provided in the TRM²⁹. To distinguish the source of the loadshape, they are color coded. Rows that are shaded in green are Efficiency Vermont loadshapes adjusted for Illinois periods. Rows that are unshaded and are left in white are Itron eShapes data provided by Ameren.

ComEd uses the DSMore™ (Integral Analytics DSMore™ Demand Side Management Option/Risk Evaluator) software to screen the efficiency measures for cost effectiveness. Since this tool requires a loadshape value for weekdays and weekends in each month (i.e., 24 inputs), the percentages for the four period categories above were calculated by weighting the proportion of weekdays/weekends in each month to the total within each period. The results of these calculations are also provided below.

²⁹ All loadshape information has been posted to the VEIC Sharepoint site, and is publically accessible through the Stakeholder Advisory Group's web site. <http://www.ilsag.info/technical-reference-manual.html>
http://ilsagfiles.org/SAG_files/Technical_Reference_Manual/Residential_Loadshapes_References.zip
http://ilsagfiles.org/SAG_files/Technical_Reference_Manual/Commercial_Loadshapes_References.zip
http://ilsagfiles.org/SAG_files/Technical_Reference_Manual/Final_Draft/Sources%20and%20References%20-%20Loadshapes/TRM_Version_3_Loadshapes_2.24.zip

Table 3.4: Loadshapes by Season

		Winter Peak	Winter Off-peak	Summer Peak	Summer Off-peak
	Loadshape Reference Number	Oct-Apr, M-F, non-holiday, 8AM - 11PM	Oct-Apr, All other time	May-Sept, M-F, non-holiday, 8AM - 11PM	May- Sept, All other time
Residential Clothes Washer	R01	47.0%	11.1%	34.0%	8.0%
Residential Dish Washer	R02	49.3%	8.7%	35.7%	6.3%
Residential Electric DHW	R03	43.2%	20.6%	24.5%	11.7%
Residential Freezer	R04	38.9%	16.4%	31.5%	13.2%
Residential Refrigerator	R05	37.0%	18.1%	30.1%	14.7%
Residential Indoor Lighting	R06	48.1%	15.5%	26.0%	10.5%
Residential Outdoor Lighting	R07	18.0%	44.1%	9.4%	28.4%
Residential Cooling	R08	4.1%	0.7%	71.3%	23.9%
Residential Electric Space Heat	R09	57.8%	38.8%	1.7%	1.7%
Residential Electric Heating and Cooling	R10	35.2%	22.8%	31.0%	11.0%
Residential Ventilation	R11	25.8%	32.3%	18.9%	23.0%
Residential - Dehumidifier	R12	12.9%	16.2%	31.7%	39.2%
Residential Standby Losses - Entertainment Center	R13	26.0%	32.5%	18.9%	22.6%
Residential Standby Losses - Home Office	R14	23.9%	34.6%	17.0%	24.5%
Residential Pool Pumps	R15	0%	0%	58.9%	41.1%
Commercial Electric Cooking	C01	40.6%	18.2%	28.7%	12.6%
Commercial Electric DHW	C02	40.5%	18.2%	28.5%	12.8%
Commercial Cooling	C03	4.9%	0.8%	66.4%	27.9%
Commercial Electric Heating	C04	53.5%	43.2%	1.9%	1.4%
Commercial Electric Heating and Cooling	C05	19.4%	13.5%	47.1%	19.9%
Commercial Indoor Lighting	C06	40.1%	18.6%	28.4%	12.9%
Grocery/Conv. Store Indoor Lighting	C07	31.4%	26.4%	22.8%	19.3%
Hospital Indoor Lighting	C08	29.1%	29.0%	21.0%	20.9%
Office Indoor Lighting	C09	42.1%	16.0%	30.4%	11.5%

		Winter Peak	Winter Off-peak	Summer Peak	Summer Off-peak
	Loadshape Reference Number	Oct-Apr, M-F, non-holiday, 8AM - 11PM	Oct-Apr, All other time	May-Sept, M-F, non-holiday, 8AM - 11PM	May- Sept, All other time
Restaurant Indoor Lighting	C10	32.1%	25.7%	23.4%	18.8%
Retail Indoor Lighting	C11	35.5%	22.3%	25.8%	16.3%
Warehouse Indoor Lighting	C12	39.4%	18.5%	28.6%	13.5%
K-12 School Indoor Lighting	C13	45.8%	22.6%	20.2%	11.4%
Indust. 1-shift (8/5) (e.g., comp. air, lights)	C14	50.5%	7.2%	37.0%	5.3%
Indust. 2-shift (16/5) (e.g., comp. air, lights)	C15	47.5%	10.2%	34.8%	7.4%
Indust. 3-shift (24/5) (e.g., comp. air, lights)	C16	34.8%	23.2%	25.5%	16.6%
Indust. 4-shift (24/7) (e.g., comp. air, lights)	C17	25.8%	32.3%	18.9%	23.0%
Industrial Indoor Lighting	C18	44.3%	13.6%	32.4%	9.8%
Industrial Outdoor Lighting	C19	18.0%	44.1%	9.4%	28.4%
Commercial Outdoor Lighting	C20	23.4%	35.3%	13.0%	28.3%
Commercial Office Equipment	C21	37.7%	20.9%	26.7%	14.7%
Commercial Refrigeration	C22	38.5%	20.6%	26.7%	14.2%
Commercial Ventilation	C23	38.1%	20.6%	29.7%	11.6%
Traffic Signal - Red Balls, always changing or flashing	C24	25.8%	32.3%	18.9%	23.0%
Traffic Signal - Red Balls, changing day, off night	C25	37.0%	20.9%	27.1%	14.9%
Traffic Signal - Green Balls, always changing	C26	25.8%	32.3%	18.9%	23.0%
Traffic Signal - Green Balls, changing day, off night	C27	37.0%	20.9%	27.1%	14.9%
Traffic Signal - Red Arrows	C28	25.8%	32.3%	18.9%	23.0%
Traffic Signal - Green Arrows	C29	25.8%	32.3%	18.9%	23.0%
Traffic Signal - Flashing Yellows	C30	25.8%	32.3%	18.9%	23.0%
Traffic Signal - "Hand" Don't Walk Signal	C31	25.8%	32.3%	18.9%	23.0%
Traffic Signal - "Man" Walk Signal	C32	25.8%	32.3%	18.9%	23.0%
Traffic Signal - Bi-Modal Walk/Don't Walk	C33	25.8%	32.3%	18.9%	23.0%
Industrial Motor	C34	47.5%	10.2%	34.8%	7.4%
Industrial Process	C35	47.5%	10.2%	34.8%	7.4%
HVAC Pump Motor (heating)	C36	38.7%	48.6%	5.9%	6.8%

		Winter Peak	Winter Off-peak	Summer Peak	Summer Off-peak
	Loadshape Reference Number	Oct-Apr, M-F, non-holiday, 8AM - 11PM	Oct-Apr, All other time	May-Sept, M-F, non-holiday, 8AM - 11PM	May- Sept, All other time
HVAC Pump Motor (cooling)	C37	7.8%	9.8%	36.8%	45.6%
HVAC Pump Motor (unknown use)	C38	23.2%	29.2%	21.4%	26.2%
VFD - Supply fans <10 HP	C39	38.8%	16.1%	28.4%	16.7%
VFD - Return fans <10 HP	C40	38.8%	16.1%	28.4%	16.7%
VFD - Exhaust fans <10 HP	C41	34.8%	23.2%	20.3%	21.7%
VFD - Boiler feedwater pumps <10 HP	C42	42.9%	44.2%	6.6%	6.3%
VFD - Chilled water pumps <10 HP	C43	11.2%	5.5%	40.7%	42.6%
VFD Boiler circulation pumps <10 HP	C44	42.9%	44.2%	6.6%	6.3%
Refrigeration Economizer	C45	36.3%	50.8%	5.6%	7.3%
Evaporator Fan Control	C46	24.0%	35.9%	16.7%	23.4%
Standby Losses - Commercial Office	C47	8.2%	50.5%	5.6%	35.7%
VFD Boiler draft fans <10 HP	C48	37.3%	48.9%	6.4%	7.3%
VFD Cooling Tower Fans <10 HP	C49	7.9%	5.2%	54.0%	32.9%
Engine Block Heater Timer	C50	26.5%	61.0%	4.1%	8.5%
Door Heater Control	C51	30.4%	69.6%	0.0%	0.0%
Beverage and Snack Machine Controls	C52	10.0%	48.3%	7.4%	34.3%
Flat	C53	36.3%	21.8%	26.2%	15.7%
Religious Indoor Lighting	C54	26.8%	31.4%	18.9%	22.8%

Table 3.5: Loadshapes by Month and Day of Week

		Jan		Feb		Mar		Apr		May		Jun		Jul		Aug		Sep		Oct		Nov		Dec	
		M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S
Residential Clothes Washer	R01	7.0%	1.6%	6.3%	1.5%	6.6%	1.7%	6.7%	1.5%	6.9%	1.6%	6.5%	1.6%	7.1%	1.5%	6.8%	1.7%	6.6%	1.6%	7.0%	1.5%	6.5%	1.7%	6.9%	1.6%
Residential Dish Washer	R02	7.3%	1.2%	6.6%	1.2%	7.0%	1.4%	7.1%	1.2%	7.3%	1.2%	6.9%	1.3%	7.4%	1.2%	7.1%	1.3%	7.0%	1.2%	7.4%	1.2%	6.8%	1.3%	7.2%	1.3%
Residential Electric DHW	R03	6.4%	2.9%	5.8%	2.7%	6.1%	3.3%	6.2%	2.8%	5.0%	2.3%	4.7%	2.4%	5.1%	2.2%	4.9%	2.5%	4.8%	2.3%	6.5%	2.8%	6.0%	3.1%	6.3%	3.0%
Residential Freezer	R04	5.8%	2.3%	5.2%	2.2%	5.5%	2.6%	5.6%	2.2%	6.4%	2.6%	6.1%	2.7%	6.6%	2.5%	6.3%	2.8%	6.1%	2.6%	5.8%	2.2%	5.4%	2.4%	5.7%	2.4%
Residential Refrigerator	R05	5.5%	2.6%	4.9%	2.4%	5.2%	2.9%	5.3%	2.5%	6.2%	2.9%	5.8%	3.0%	6.3%	2.8%	6.0%	3.1%	5.9%	2.9%	5.5%	2.5%	5.1%	2.7%	5.4%	2.6%
Residential Indoor Lighting	R06	7.1%	2.2%	6.4%	2.1%	6.8%	2.4%	6.9%	2.1%	5.3%	2.1%	5.0%	2.2%	5.4%	2.0%	5.2%	2.2%	5.1%	2.1%	7.2%	2.1%	6.6%	2.3%	7.0%	2.2%
Residential Outdoor Lighting	R07	2.7%	6.2%	2.4%	5.9%	2.6%	7.0%	2.6%	6.0%	1.9%	5.7%	1.8%	5.8%	2.0%	5.3%	1.9%	6.0%	1.8%	5.7%	2.7%	6.0%	2.5%	6.6%	2.6%	6.4%
Residential Cooling	R08	0.6%	0.1%	0.5%	0.1%	0.6%	0.1%	0.6%	0.1%	14.6%	4.8%	13.7%	4.9%	14.9%	4.5%	14.2%	5.0%	13.9%	4.8%	0.6%	0.1%	0.6%	0.1%	0.6%	0.1%
Residential Electric Space Heat	R09	8.6%	5.5%	7.7%	5.1%	8.2%	6.1%	8.3%	5.3%	0.3%	0.3%	0.3%	0.3%	0.4%	0.3%	0.3%	0.4%	0.3%	0.3%	8.7%	5.3%	8.0%	5.8%	8.5%	5.6%
Residential Electric Heating and Cooling	R10	5.2%	3.2%	4.7%	3.0%	5.0%	3.6%	5.0%	3.1%	6.3%	2.2%	6.0%	2.3%	6.5%	2.1%	6.2%	2.3%	6.0%	2.2%	5.3%	3.1%	4.9%	3.4%	5.2%	3.3%
Residential Ventilation	R11	3.8%	4.6%	3.4%	4.3%	3.6%	5.1%	3.7%	4.4%	3.8%	4.6%	3.6%	4.7%	3.9%	4.3%	3.8%	4.8%	3.7%	4.6%	3.9%	4.4%	3.6%	4.8%	3.8%	4.7%
Residential - Dehumidifier	R12	1.9%	2.3%	1.7%	2.2%	1.8%	2.6%	1.8%	2.2%	6.5%	7.8%	6.1%	8.0%	6.6%	7.3%	6.3%	8.2%	6.2%	7.8%	1.9%	2.2%	1.8%	2.4%	1.9%	2.4%
Residential Standby Losses - Entertainment Center	R13	3.8%	4.6%	3.5%	4.3%	3.7%	5.1%	3.7%	4.4%	3.9%	4.5%	3.7%	4.6%	4.0%	4.2%	3.8%	4.8%	3.7%	4.5%	3.9%	4.4%	3.6%	4.8%	3.8%	4.7%
Residential Standby	R14	3.5%	4.9%	3.2%	4.6%	3.4%	5.5%	3.4%	4.7%	3.5%	4.9%	3.3%	5.0%	3.5%	4.6%	3.4%	5.2%	3.3%	4.9%	3.6%	4.7%	3.3%	5.2%	3.5%	5.0%

		Jan		Feb		Mar		Apr		May		Jun		Jul		Aug		Sep		Oct		Nov		Dec	
		M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S
Losses - Home Office																									
Commercial Electric Cooking	C01	6.0%	2.6%	5.4%	2.4%	5.7%	2.9%	5.8%	2.5%	5.9%	2.5%	5.5%	2.6%	6.0%	2.4%	5.7%	2.6%	5.6%	2.5%	6.1%	2.5%	5.6%	2.7%	5.9%	2.6%
Commercial Electric DHW	C02	6.0%	2.6%	5.4%	2.4%	5.7%	2.9%	5.8%	2.5%	5.8%	2.5%	5.5%	2.6%	6.0%	2.4%	5.7%	2.7%	5.6%	2.5%	6.1%	2.5%	5.6%	2.7%	5.9%	2.6%
Commercial Cooling	C03	0.7%	0.1%	0.6%	0.1%	0.7%	0.1%	0.7%	0.1%	13.6%	5.5%	12.8%	5.7%	13.9%	5.2%	13.3%	5.9%	13.0%	5.5%	0.7%	0.1%	0.7%	0.1%	0.7%	0.1%
Commercial Electric Heating	C04	7.9%	6.1%	7.1%	5.7%	7.6%	6.8%	7.7%	5.9%	0.4%	0.3%	0.4%	0.3%	0.4%	0.3%	0.4%	0.3%	0.4%	0.3%	8.0%	5.9%	7.4%	6.5%	7.8%	6.3%
Commercial Electric Heating and Cooling	C05	2.9%	1.9%	2.6%	1.8%	2.8%	2.1%	2.8%	1.9%	9.6%	4.0%	9.1%	4.1%	9.8%	3.7%	9.4%	4.2%	9.2%	4.0%	2.9%	1.9%	2.7%	2.0%	2.8%	2.0%
Commercial Indoor Lighting	C06	5.9%	2.6%	5.3%	2.5%	5.7%	2.9%	5.7%	2.6%	5.8%	2.6%	5.5%	2.6%	5.9%	2.4%	5.7%	2.7%	5.5%	2.6%	6.0%	2.6%	5.5%	2.8%	5.9%	2.7%
Grocery/Conv. Store Indoor Lighting	C07	4.7%	3.7%	4.2%	3.5%	4.4%	4.2%	4.5%	3.6%	4.7%	3.8%	4.4%	3.9%	4.8%	3.6%	4.6%	4.1%	4.5%	3.8%	4.7%	3.6%	4.3%	3.9%	4.6%	3.8%
Hospital Indoor Lighting	C08	4.3%	4.1%	3.9%	3.8%	4.1%	4.6%	4.2%	4.0%	4.3%	4.2%	4.0%	4.3%	4.4%	3.9%	4.2%	4.4%	4.1%	4.2%	4.4%	4.0%	4.0%	4.3%	4.3%	4.2%
Office Indoor Lighting	C09	6.2%	2.3%	5.6%	2.1%	6.0%	2.5%	6.0%	2.2%	6.2%	2.3%	5.9%	2.4%	6.4%	2.2%	6.1%	2.4%	5.9%	2.3%	6.3%	2.2%	5.8%	2.4%	6.2%	2.3%
Restaurant Indoor Lighting	C10	4.8%	3.6%	4.3%	3.4%	4.5%	4.1%	4.6%	3.5%	4.8%	3.7%	4.5%	3.8%	4.9%	3.5%	4.7%	4.0%	4.6%	3.7%	4.8%	3.5%	4.4%	3.8%	4.7%	3.7%
Retail Indoor Lighting	C11	5.3%	3.1%	4.7%	3.0%	5.0%	3.5%	5.1%	3.1%	5.3%	3.2%	5.0%	3.3%	5.4%	3.1%	5.2%	3.4%	5.0%	3.2%	5.3%	3.1%	4.9%	3.3%	5.2%	3.2%
Warehouse Indoor Lighting	C12	5.8%	2.6%	5.2%	2.5%	5.6%	2.9%	5.6%	2.5%	5.8%	2.7%	5.5%	2.8%	6.0%	2.5%	5.7%	2.8%	5.6%	2.7%	5.9%	2.5%	5.4%	2.8%	5.8%	2.7%
K-12 School Indoor	C13	6.8%	3.2%	6.1%	3.0%	6.5%	3.6%	6.6%	3.1%	4.1%	2.3%	3.9%	2.3%	4.2%	2.1%	4.0%	2.4%	3.9%	2.3%	6.9%	3.1%	6.3%	3.4%	6.7%	3.3%

		Jan		Feb		Mar		Apr		May		Jun		Jul		Aug		Sep		Oct		Nov		Dec	
		M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S
Lighting																									
Indust. 1-shift (8/5) (e.g., comp. air, lights)	C14	7.5%	1.0%	6.7%	1.0%	7.1%	1.1%	7.2%	1.0%	7.5%	1.1%	7.1%	1.1%	7.7%	1.0%	7.4%	1.1%	7.2%	1.1%	7.6%	1.0%	7.0%	1.1%	7.4%	1.0%
Indust. 2-shift (16/5) (e.g., comp. air, lights)	C15	7.0%	1.4%	6.3%	1.4%	6.7%	1.6%	6.8%	1.4%	7.1%	1.5%	6.7%	1.5%	7.3%	1.4%	6.9%	1.6%	6.8%	1.5%	7.1%	1.4%	6.6%	1.5%	7.0%	1.5%
Indust. 3-shift (24/5) (e.g., comp. air, lights)	C16	5.1%	3.3%	4.6%	3.1%	4.9%	3.7%	5.0%	3.2%	5.2%	3.3%	4.9%	3.4%	5.3%	3.1%	5.1%	3.5%	5.0%	3.3%	5.2%	3.2%	4.8%	3.5%	5.1%	3.4%
Indust. 4-shift (24/7) (e.g., comp. air, lights)	C17	3.8%	4.6%	3.4%	4.3%	3.6%	5.1%	3.7%	4.4%	3.8%	4.6%	3.6%	4.7%	3.9%	4.3%	3.8%	4.8%	3.7%	4.6%	3.9%	4.4%	3.6%	4.8%	3.8%	4.7%
Industrial Indoor Lighting	C18	6.6%	1.9%	5.9%	1.8%	6.3%	2.1%	6.3%	1.9%	6.6%	1.9%	6.2%	2.0%	6.8%	1.8%	6.5%	2.0%	6.3%	1.9%	6.6%	1.9%	6.1%	2.0%	6.5%	2.0%
Industrial Outdoor Lighting	C19	2.7%	6.2%	2.4%	5.9%	2.6%	7.0%	2.6%	6.0%	1.9%	5.7%	1.8%	5.8%	2.0%	5.3%	1.9%	6.0%	1.8%	5.7%	2.7%	6.0%	2.5%	6.6%	2.6%	6.4%
Commercial Outdoor Lighting	C20	3.5%	5.0%	3.1%	4.7%	3.3%	5.6%	3.3%	4.8%	2.7%	5.6%	2.5%	5.8%	2.7%	5.3%	2.6%	5.9%	2.5%	5.6%	3.5%	4.8%	3.2%	5.3%	3.4%	5.1%
Commercial Office Equipment	C21	5.6%	3.0%	5.0%	2.8%	5.3%	3.3%	5.4%	2.9%	5.4%	2.9%	5.1%	3.0%	5.6%	2.7%	5.3%	3.1%	5.2%	2.9%	5.6%	2.9%	5.2%	3.1%	5.5%	3.0%
Commercial Refrigeration	C22	5.7%	2.9%	5.1%	2.7%	5.4%	3.2%	5.5%	2.8%	5.5%	2.8%	5.1%	2.9%	5.6%	2.7%	5.3%	3.0%	5.2%	2.8%	5.8%	2.8%	5.3%	3.1%	5.6%	3.0%
Commercial Ventilation	C23	5.6%	2.9%	5.1%	2.7%	5.4%	3.3%	5.4%	2.8%	6.1%	2.3%	5.7%	2.4%	6.2%	2.2%	5.9%	2.4%	5.8%	2.3%	5.7%	2.8%	5.3%	3.1%	5.6%	3.0%
Traffic Signal - Red Balls, always changing or flashing	C24	3.8%	4.6%	3.4%	4.3%	3.6%	5.1%	3.7%	4.4%	3.8%	4.6%	3.6%	4.7%	3.9%	4.3%	3.8%	4.8%	3.7%	4.6%	3.9%	4.4%	3.6%	4.8%	3.8%	4.7%

		Jan		Feb		Mar		Apr		May		Jun		Jul		Aug		Sep		Oct		Nov		Dec	
		M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S
Traffic Signal - Red Balls, changing day, off night	C25	5.5%	2.9%	4.9%	2.8%	5.2%	3.3%	5.3%	2.9%	5.5%	3.0%	5.2%	3.1%	5.7%	2.8%	5.4%	3.1%	5.3%	3.0%	5.5%	2.9%	5.1%	3.1%	5.4%	3.0%
Traffic Signal - Green Balls, always changing	C26	3.8%	4.6%	3.4%	4.3%	3.6%	5.1%	3.7%	4.4%	3.8%	4.6%	3.6%	4.7%	3.9%	4.3%	3.8%	4.8%	3.7%	4.6%	3.9%	4.4%	3.6%	4.8%	3.8%	4.7%
Traffic Signal - Green Balls, changing day, off night	C27	5.5%	2.9%	4.9%	2.8%	5.2%	3.3%	5.3%	2.9%	5.5%	3.0%	5.2%	3.1%	5.7%	2.8%	5.4%	3.1%	5.3%	3.0%	5.5%	2.9%	5.1%	3.1%	5.4%	3.0%
Traffic Signal - Red Arrows	C28	3.8%	4.6%	3.4%	4.3%	3.6%	5.1%	3.7%	4.4%	3.8%	4.6%	3.6%	4.7%	3.9%	4.3%	3.8%	4.8%	3.7%	4.6%	3.9%	4.4%	3.6%	4.8%	3.8%	4.7%
Traffic Signal - Green Arrows	C29	3.8%	4.6%	3.4%	4.3%	3.6%	5.1%	3.7%	4.4%	3.8%	4.6%	3.6%	4.7%	3.9%	4.3%	3.8%	4.8%	3.7%	4.6%	3.9%	4.4%	3.6%	4.8%	3.8%	4.7%
Traffic Signal - Flashing Yellows	C30	3.8%	4.6%	3.4%	4.3%	3.6%	5.1%	3.7%	4.4%	3.8%	4.6%	3.6%	4.7%	3.9%	4.3%	3.8%	4.8%	3.7%	4.6%	3.9%	4.4%	3.6%	4.8%	3.8%	4.7%
Traffic Signal - "Hand" Don't Walk Signal	C31	3.8%	4.6%	3.4%	4.3%	3.6%	5.1%	3.7%	4.4%	3.8%	4.6%	3.6%	4.7%	3.9%	4.3%	3.8%	4.8%	3.7%	4.6%	3.9%	4.4%	3.6%	4.8%	3.8%	4.7%
Traffic Signal - "Man" Walk Signal	C32	3.8%	4.6%	3.4%	4.3%	3.6%	5.1%	3.7%	4.4%	3.8%	4.6%	3.6%	4.7%	3.9%	4.3%	3.8%	4.8%	3.7%	4.6%	3.9%	4.4%	3.6%	4.8%	3.8%	4.7%
Traffic Signal - Bi-Modal Walk/Don't Walk	C33	3.8%	4.6%	3.4%	4.3%	3.6%	5.1%	3.7%	4.4%	3.8%	4.6%	3.6%	4.7%	3.9%	4.3%	3.8%	4.8%	3.7%	4.6%	3.9%	4.4%	3.6%	4.8%	3.8%	4.7%
Industrial Motor	C34	7.0%	1.4%	6.3%	1.4%	6.7%	1.6%	6.8%	1.4%	7.1%	1.5%	6.7%	1.5%	7.3%	1.4%	6.9%	1.6%	6.8%	1.5%	7.1%	1.4%	6.6%	1.5%	7.0%	1.5%
Industrial Process	C35	7.0%	1.4%	6.3%	1.4%	6.7%	1.6%	6.8%	1.4%	7.1%	1.5%	6.7%	1.5%	7.3%	1.4%	6.9%	1.6%	6.8%	1.5%	7.1%	1.4%	6.6%	1.5%	7.0%	1.5%
HVAC Pump Motor (heating)	C36	5.7%	6.9%	5.2%	6.4%	5.5%	7.7%	5.5%	6.6%	1.2%	1.4%	1.1%	1.4%	1.2%	1.3%	1.2%	1.4%	1.2%	1.4%	5.8%	6.6%	5.3%	7.3%	5.7%	7.1%

		Jan		Feb		Mar		Apr		May		Jun		Jul		Aug		Sep		Oct		Nov		Dec	
		M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S
HVAC Pump Motor (cooling)	C37	1.2%	1.4%	1.0%	1.3%	1.1%	1.5%	1.1%	1.3%	7.5%	9.1%	7.1%	9.3%	7.7%	8.5%	7.3%	9.6%	7.2%	9.1%	1.2%	1.3%	1.1%	1.5%	1.1%	1.4%
HVAC Pump Motor (unknown use)	C38	3.4%	4.1%	3.1%	3.9%	3.3%	4.6%	3.3%	4.0%	4.4%	5.2%	4.1%	5.4%	4.5%	4.9%	4.3%	5.5%	4.2%	5.2%	3.5%	4.0%	3.2%	4.4%	3.4%	4.2%
VFD - Supply fans <10 HP	C39	5.7%	2.3%	5.2%	2.1%	5.5%	2.5%	5.6%	2.2%	5.8%	3.3%	5.5%	3.4%	5.9%	3.1%	5.7%	3.5%	5.5%	3.3%	5.8%	2.2%	5.4%	2.4%	5.7%	2.3%
VFD - Return fans <10 HP	C40	5.7%	2.3%	5.2%	2.1%	5.5%	2.5%	5.6%	2.2%	5.8%	3.3%	5.5%	3.4%	5.9%	3.1%	5.7%	3.5%	5.5%	3.3%	5.8%	2.2%	5.4%	2.4%	5.7%	2.3%
VFD - Exhaust fans <10 HP	C41	5.1%	3.3%	4.6%	3.1%	4.9%	3.7%	5.0%	3.2%	4.1%	4.3%	3.9%	4.4%	4.2%	4.1%	4.1%	4.6%	4.0%	4.3%	5.2%	3.2%	4.8%	3.5%	5.1%	3.4%
VFD - Boiler feedwater pumps <10 HP	C42	6.4%	6.2%	5.7%	5.9%	6.1%	7.0%	6.1%	6.0%	1.3%	1.3%	1.3%	1.3%	1.4%	1.2%	1.3%	1.3%	1.3%	1.3%	6.4%	6.0%	5.9%	6.6%	6.3%	6.4%
VFD - Chilled water pumps <10 HP	C43	1.7%	0.8%	1.5%	0.7%	1.6%	0.9%	1.6%	0.8%	8.3%	8.5%	7.8%	8.7%	8.5%	8.0%	8.1%	8.9%	7.9%	8.5%	1.7%	0.8%	1.6%	0.8%	1.6%	0.8%
VFD Boiler circulation pumps <10 HP	C44	6.4%	6.2%	5.7%	5.9%	6.1%	7.0%	6.1%	6.0%	1.3%	1.3%	1.3%	1.3%	1.4%	1.2%	1.3%	1.3%	1.3%	1.3%	6.4%	6.0%	5.9%	6.6%	6.3%	6.4%
Refrigeration Economizer	C45	5.4%	7.2%	4.8%	6.7%	5.1%	8.0%	5.2%	7.0%	1.1%	1.5%	1.1%	1.5%	1.2%	1.4%	1.1%	1.5%	1.1%	1.5%	5.4%	7.0%	5.0%	7.6%	5.3%	7.4%
Evaporator Fan Control	C46	3.6%	5.1%	3.2%	4.8%	3.4%	5.7%	3.4%	4.9%	3.4%	4.7%	3.2%	4.8%	3.5%	4.4%	3.3%	4.9%	3.3%	4.7%	3.6%	4.9%	3.3%	5.4%	3.5%	5.2%
Standby Losses - Commercial Office	C47	1.2%	7.1%	1.1%	6.7%	1.2%	8.0%	1.2%	6.9%	1.1%	7.1%	1.1%	7.3%	1.2%	6.7%	1.1%	7.5%	1.1%	7.1%	1.2%	6.9%	1.1%	7.5%	1.2%	7.3%
VFD Boiler draft fans <10 HP	C48	5.5%	6.9%	5.0%	6.5%	5.3%	7.7%	5.3%	6.7%	1.3%	1.5%	1.2%	1.5%	1.3%	1.4%	1.3%	1.5%	1.2%	1.5%	5.6%	6.7%	5.2%	7.3%	5.5%	7.1%
VFD Cooling Tower Fans <10 HP	C49	1.2%	0.7%	1.1%	0.7%	1.1%	0.8%	1.1%	0.7%	11.0%	6.5%	10.4%	6.7%	11.3%	6.2%	10.8%	6.9%	10.5%	6.5%	1.2%	0.7%	1.1%	0.8%	1.2%	0.8%

		Jan		Feb		Mar		Apr		May		Jun		Jul		Aug		Sep		Oct		Nov		Dec	
		M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S	M-F	S-S
Engine Block Heater Timer	C50	3.9%	8.6%	3.5%	8.1%	3.7%	9.6%	3.8%	8.3%	0.8%	1.7%	0.8%	1.7%	0.8%	1.6%	0.8%	1.8%	0.8%	1.7%	4.0%	8.3%	3.7%	9.1%	3.9%	8.9%
Door Heater Control	C51	4.5%	9.8%	4.0%	9.2%	4.3%	11.0%	4.3%	9.5%	0.0%	0.0%	0.0%	0.0%	0.0%	0.0%	0.0%	0.0%	0.0%	0.0%	4.5%	9.5%	4.2%	10.4%	4.4%	10.1%
Beverage and Snack Machine Controls	C52	1.5%	6.8%	1.3%	6.4%	1.4%	7.6%	1.4%	6.6%	1.5%	6.8%	1.4%	7.0%	1.5%	6.4%	1.5%	7.2%	1.4%	6.8%	1.5%	6.6%	1.4%	7.2%	1.5%	7.0%
Flat	C53	5.4%	3.1%	4.8%	2.9%	5.1%	3.4%	5.2%	3.0%	5.3%	3.1%	5.0%	3.2%	5.5%	2.9%	5.2%	3.3%	5.1%	3.1%	5.4%	3.0%	5.0%	3.3%	5.3%	3.2%
Religious Indoor Lighting	C54	4.0%	4.4%	3.6%	4.2%	3.8%	5.0%	3.8%	4.3%	3.9%	4.5%	3.6%	4.7%	3.9%	4.3%	3.8%	4.8%	3.7%	4.5%	4.0%	4.3%	3.7%	4.7%	3.9%	4.6%

3.6 Summer Peak Period Definition (kW)

To estimate the impact that an efficiency measure has on a utility's system peak, the peak itself needs to be defined. Illinois spans two different electrical control areas, the Pennsylvania – Jersey – Maryland (PJM) and the Midwest Independent System Operators (MISO). As a result, there is some disparity in the peak definition across the state. However, only PJM has a forward capacity market where an efficiency program can potentially participate. Because ComEd is part of the PJM control area, their definition of summer peak is being applied statewide in this TRM.

Because Illinois is a summer peaking state, only the summer peak period is defined for the purpose of this TRM. The coincident summer peak period is defined as 1:00-5:00 PM Central Prevailing Time on non-holiday weekdays, June through August.

Summer peak coincidence factors can be found within each measure characterization. The source is provided and is based upon evaluation results, analysis of load shape data (e.g., the Itron eShapes data provided by Ameren), or through a calculation using stated assumptions.

For measures that are not weather-sensitive, the summer peak coincidence factor is estimated whenever possible as the average of savings within the peak period defined above. For weather sensitive measures such as cooling, the summer peak coincidence factor is provided in two different ways. The first method is to estimate demand savings during the utility's peak hour (as provided by Ameren). This is likely to be the most indicative of actual peak benefits. The second way represents the average savings over the summer peak period, consistent with the non-weather sensitive end uses, and is presented so that savings can be bid into PJM's Forward Capacity Market.

3.7 Heating and Cooling Degree-Day Data

Many measures are weather sensitive. Because there is a range of climactic conditions across the state, VEIC engaged the Utilities to provide their preferences for what airports and cities are the best proxies for the weather in their service territories. The result of this engagement is in the table below. All of the data represents 30-year normals³⁰ from the National Climatic Data Center (NCDC). Note that the base temperature for the calculation of heating degree-days in this document does not follow the historical 65F degree base temperature convention. Instead VEIC used several different temperatures in this TRM to more accurately reflect the outdoor temperature when a heating or cooling system turns on.

Residential heating is based on 60F, in accordance with regression analysis of heating fuel use and weather by state by the Pacific Northwest National Laboratory³¹. Residential cooling is based on 65F in agreement with a field study in Wisconsin³². These are lower than typical thermostat set points because internal gains such as appliances, lighting, and people provide some heating. In C&I settings, internal gains are often much higher; the base temperatures for both heating and cooling is 55F³³. Custom degree-days with building specific base temperatures are recommended for large C&I projects.

³⁰ 30-year normals have been used instead of Typical Meteorological Year (TMY) data due to the fact that few of the measures in the TRM are significantly affected by solar insolation, which is one of the primary benefits of using the TMY approach.

³¹ Belzer and Cort, Pacific Northwest National Laboratory in "Statistical Analysis of Historical State-Level Residential Energy Consumption Trends," 2004.

³² Energy Center of Wisconsin, May 2008 metering study; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research", p. 32 (amended in 2010).

³³ This value is based upon experience, and it is preferable to use building-specific base temperatures when available.

Table 3.6: Degree-Day Zones and Values by Market Sector

Zone	Residential		C&I		Weather Station / City
	HDD	CDD	HDD	CDD	
1	5,352	820	4,272	2,173	Rockford AP / Rockford
2	5,113	842	4,029	3,357	Chicago O'Hare AP / Chicago
3	4,379	1,108	3,406	2,666	Springfield #2 / Springfield
4	3,378	1,570	2,515	3,090	Belleville SIU RSCH / Belleville
5	3,438	1,370	2,546	2,182	Carbondale Southern IL AP / Marion
Average	4,860	947	3,812	3,051	Weighted by occupied housing units
Base Temp	60F	65F	55F	55F	30 year climate normals, 1981-2010

This table assigns each of the proxy cities to one of five climate zones. The following graphics from the Illinois State Water Survey show isobars (lines of equal degree-days) and we have color-coded the counties in each of these graphics using those isobars as a dividing line. Using this approach, the state divides into five cooling degree-day zones and five heating degree-day zones. Note that although the heating and cooling degree-day maps are similar, they are not the same, and the result is that there are a total of 10 climate zones in the state. The counties are listed in the tables following the figures for ease of reference.

Figure 3.1: Cooling Degree-Day Zones by County

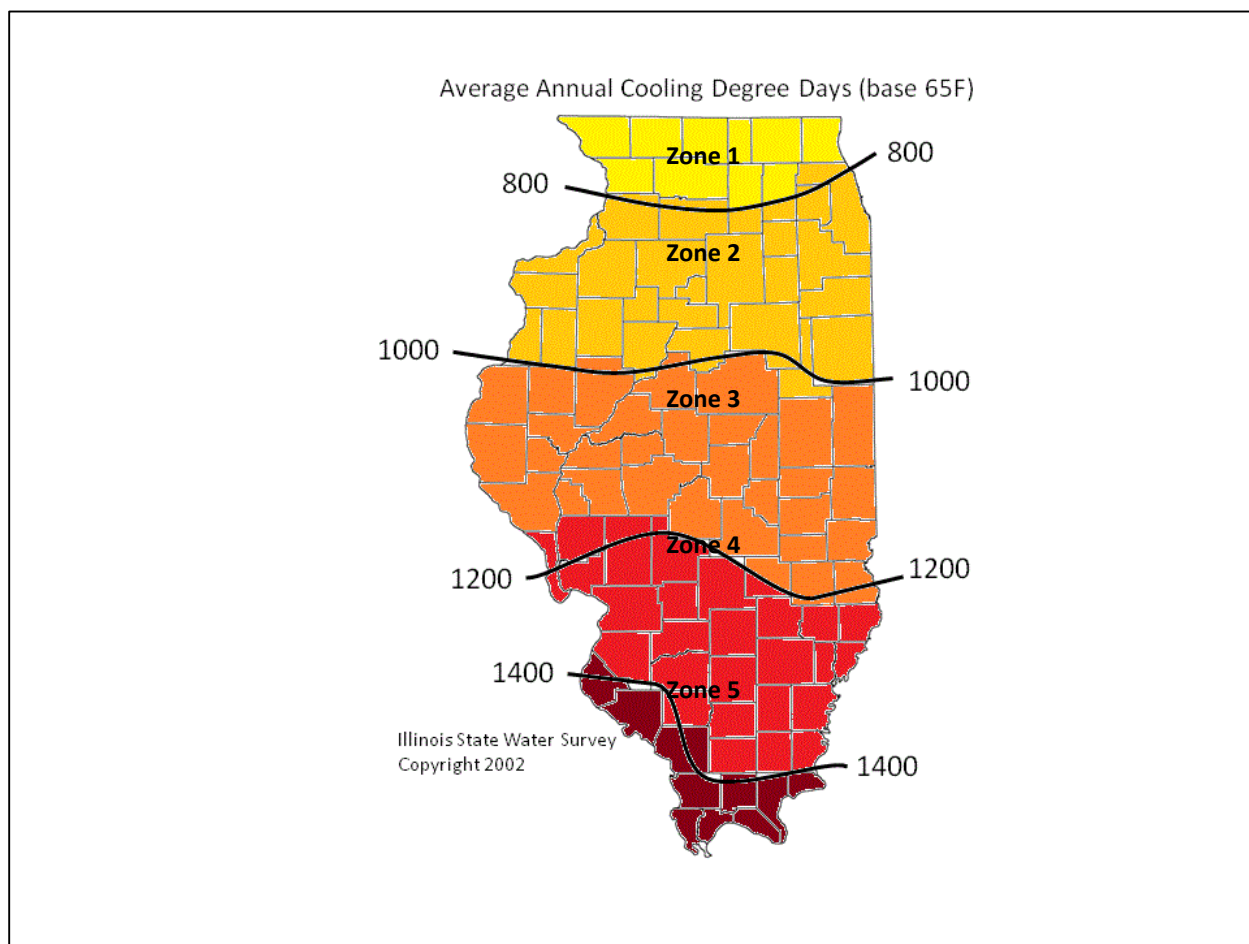


Figure 3.2: Heating Degree-Day Zones by County

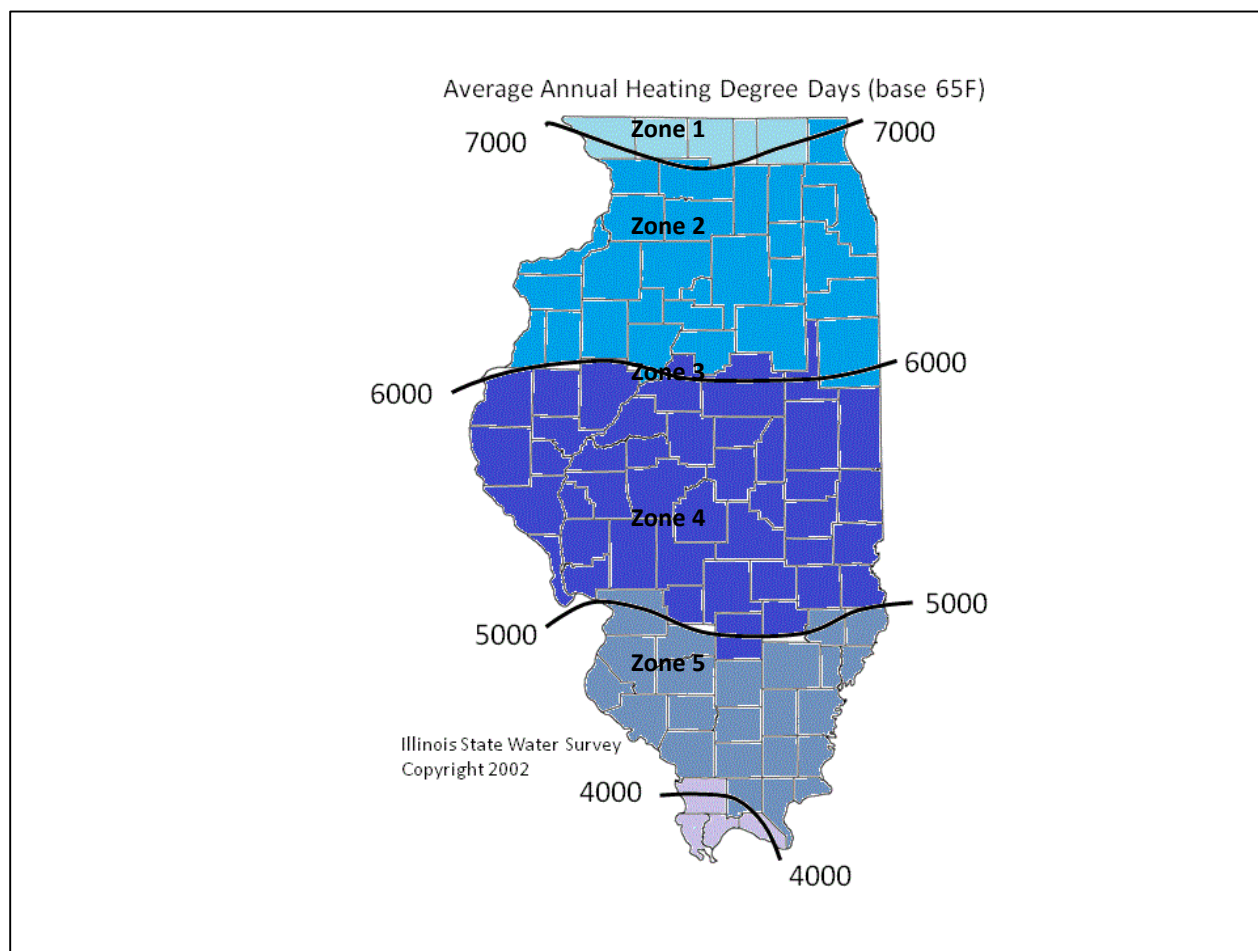


Table 3.7: Heating Degree-Day Zones by County

Zone 1	Zone 2	Zone 3	Zone 4	Zone 5
Boone County	Bureau County	Adams County	Clinton County	Alexander County
Jo Daviess County	Carroll County	Bond County	Edwards County	Massac County
Stephenson County	Cook County	Brown County	Franklin County	Pulaski County
Winnebago County	DeKalb County	Calhoun County	Gallatin County	Union County
	DuPage County	Cass County	Hamilton County	
	Grundy County	Champaign County	Hardin County	
	Henderson County	Christian County	Jackson County	
	Henry County	Clark County	Jefferson County	
	Iroquois County	Clay County	Johnson County	
	Kane County	Coles County	Lawrence County	
	Kankakee County	Crawford County	Madison County	
	Kendall County	Cumberland County	Marion County	
	Knox County	De Witt County	Monroe County	
	Lake County	Douglas County	Perry County	
	LaSalle County	Edgar County	Pope County	
	Lee County	Effingham County	Randolph County	
	Livingston County	Fayette County	Richland County	
	Marshall County	Ford County	Saline County	
	McHenry County	Fulton County	St. Clair County	
	Mercer County	Greene County	Wabash County	
	Ogle County	Hancock County	Washington County	
	Peoria County	Jasper County	Wayne County	
	Putnam County	Jersey County	White County	
	Rock Island County	Logan County	Williamson County	
	Stark County	Macon County		
	Warren County	Macoupin County		
	Whiteside County	Mason County		
	Will County	McDonough County		
	Woodford County	McLean County		
		Menard County		
		Montgomery		
		Morgan County		
		Moultrie County		
		Piatt County		
		Pike County		
		Sangamon County		
		Schuyler County		
		Scott County		
		Shelby County		
		Tazewell County		
		Vermilion County		

Table 3.8: Cooling Degree-day Zones by County

Zone 1	Zone 2	Zone 3	Zone 4	Zone 5
Boone County	Bureau County	Adams County	Bond County	Alexander County
Carroll County	Cook County	Brown County	Clay County	Hardin County

Zone 1	Zone 2	Zone 3	Zone 4	Zone 5
DeKalb County	DuPage County	Calhoun County	Clinton County	Johnson County
Jo Daviess County	Grundy County	Cass County	Edwards County	Massac County
Kane County	Henderson County	Champaign County	Fayette County	Pope County
Lake County	Henry County	Christian County	Franklin County	Pulaski County
McHenry County	Iroquois County	Clark County	Gallatin County	Randolph County
Ogle County	Kankakee County	Coles County	Hamilton County	Union County
Stephenson County	Kendall County	Crawford County	Jackson County	
Winnebago County	Knox County	Cumberland County	Jefferson County	
	LaSalle County	De Witt County	Jersey County	
	Lee County	Douglas County	Lawrence County	
	Livingston County	Edgar County	Macoupin County	
	Marshall County	Effingham County	Madison County	
	Mercer County	Ford County	Marion County	
	Peoria County	Fulton County	Monroe County	
	Putnam County	Greene County	Montgomery	
	Rock Island County	Hancock County	Perry County	
	Stark County	Jasper County	Richland County	
	Warren County	Logan County	Saline County	
	Whiteside County	Macon County	St. Clair County	
	Will County	Mason County	Wabash County	
	Woodford County	McDonough County	Washington County	
		McLean County	Wayne County	
		Menard County	White County	
		Morgan County	Williamson County	
		Moultrie County		
		Piatt County		
		Pike County		
		Sangamon County		
		Schuyler County		
		Scott County		
		Shelby County		
		Tazewell County		
		Vermilion County		

3.8 Measure Incremental Cost Definition

Incremental Costs means the difference between the cost of the efficient Measure and the cost of the most relevant baseline measure that would have been installed (if any) in the absence of the efficiency Program. Installation costs (material and labor) and Operations and Maintenance (O&M) costs shall be included if there is a difference between the efficient Measure and the baseline measure. In cases where the efficient Measure has a significantly shorter or longer life than the relevant baseline measure (e.g., LEDs versus halogens), the avoided baseline replacement measure costs should be accounted for in the TRC analysis. The Customer's value of service lost, the Customer's value of their lost amenity, and the Customer's transaction costs shall be included in the TRC analysis where a reasonable estimate or proxy of such costs can be easily obtained (e.g., Program Administrator payment to a Customer to reduce load during a demand response event, Program Administrator payment to a Customer as an inducement to give up duplicative functioning equipment). This Incremental Cost input in the TRC analysis is not reduced by the amount of any Incentives (any Financial Incentives Paid to Customers or Incentives Paid to Third Parties by a Program Administrator that is intended to reduce the price of the efficient Measure to the Customer). Incremental Cost calculations will vary depending on the type of efficient Measure being implemented, as outlined in the examples provided below and as set forth in the IL-TRM.

Examples of Incremental Cost calculations include:

- a. The Incremental Cost for an efficient Measure that is installed in new construction or is being purchased at the time of natural installation, investment, or replacement is the additional cost incurred to purchase an efficient Measure over and above the cost of the baseline/standard (i.e., less efficient) measure (including any incremental installation, replacement, or O&M costs if there is a difference between the efficient Measure and baseline measure).
- b. For a retrofit Measure where the efficiency Program caused the Customer to update their existing equipment, facility, or processes (e.g., air sealing, insulation, tank wrap, controls), where the Customer would not have otherwise made a purchase, the appropriate baseline is zero expenditure, and the Incremental Cost is the full cost of the new retrofit Measure (including installation costs).
- c. For the early replacement of a functioning measure with a new efficient Measure, where the Customer would not have otherwise made a purchase for a number of years, the appropriate baseline is a dual baseline that begins as the existing measure and shifts to the new standard measure after the expected remaining useful life of the existing measure ends. Thus, the Incremental Cost is the full cost of the new efficient Measure (including installation costs) being purchased to replace a still-functioning measure less the present value of the assumed deferred replacement cost of replacing the existing measure with a new baseline measure at the end of the existing measure's life (described in section 3.9). This deferred credit may not be necessary when the lifetime of the measure is short, the costs are very low, or for other reasons (e.g., certain Direct Install Measures, Measures provided in Kits to Customers).
- d. For study-based services (e.g., facility energy audits, energy surveys, energy assessments, retro-commissioning) that are truly necessary for a Customer to implement efficient Measures, as opposed to being principally intended to be a form of marketing, the Incremental Cost is the full cost of the study-based service. Even if the study-based service is performed entirely by a Program Administrator's implementation contractor, the full cost of the study-based service charged by the implementation contractor is the Incremental Cost, because this is assumed to be the cost of the study-based service that would have been incurred by the Customer if the Customer were to have the study-based service performed in the absence of the efficiency Program. If the Customer implements efficient Measures as a result of the study-based service provided by the efficiency Program, the Incremental Cost for those efficient Measures should also be classified as Incremental Costs in the TRC analysis.
- e. For the early retirement of duplicative functioning equipment before its expected life is over (e.g., appliance recycling Programs), the Incremental Costs are composed of the Customer's value placed on their lost amenity, any Customer transaction costs, and the pickup and recycling cost. The Incremental Costs include the actual cost of the pickup and recycling of the equipment (often paid for by a Program Administrator to

an implementation contractor) because this is assumed to be the cost of recycling the equipment that would have been incurred by the Customer if the Customer were to recycle the equipment on their own in the absence of the efficiency Program. The payment a Program Administrator makes to the Customer serves as a proxy for the value the Customer places on their lost amenity and any Customer transaction costs.

3.9 Discount Rates, Inflation Rates and O&M Costs

The Illinois Utilities utilize screening tools that apply an appropriate discount rate to any future costs or benefits. The societal discount rate, required for use by all electric utilities, is defined as a nominal discount rate of 2.38%, or a real (inflation-adjusted) discount rate of 0.46%³⁴.

Where a future cost is provided within the TRM (e.g. in early replacement measures where a deferred baseline replacement cost is provided) and the future cost has been adjusted using an inflation rate (based upon the 20-year Treasury yield of 1.91%³⁵), the nominal discount rate should be used to discount to the present value. Where future costs have not been adjusted for inflation, the real discount rate should be used to discount to present value.

Some measures specify an operations and maintenance (O&M) parameter that describes the incremental O&M cost savings that can be expected over the measure's lifetime. For most measures the TRM does not specify the NPV of the O&M costs. Instead, the necessary information required to calculate the NPV is included. An example is provided below:

Baseline Case:	O&M costs equal \$150 every two years.
Efficient Case:	O&M costs equal \$50 every five years.

Given this information, the incremental O&M costs can be determined by discounting the cash flows in the Baseline Case and the Efficient Case separately using the real discount rate.

For a select few measures that include baseline shifts that result in multiple component costs and lifetimes over the lifetime of the measure, this standard method cannot be used. In only these cases, the O&M costs are presented both as Annual Levelized equivalent cost (i.e., the annual payment that results in an equivalent NPV to the actual stream of O&M costs) and as NPVs using a real societal discount rate of 0.46%.

³⁴ Based on the current 10 year Treasury bond yield rates, as of January 2017. The 10 year rates are utilized to be consistent with the average measure life of the measures specified within this TRM.

³⁵ Established for use in the TRM in late 2015.

3.10 Interactive Effects

The TRM presents engineering equations for most measures. This approach is desirable because it conveys information clearly and transparently, and is widely accepted in the industry. Unlike simulation model results, engineering equations also provide flexibility and the opportunity for users to substitute local, specific information for specific input values. Furthermore, the parameters can be changed in TRM updates to be applied in future years as better information becomes available.

One limitation is that some interactive effects between measures are not automatically captured. Because we cannot know what measures will be implemented at the same time with the same customer, we cannot always capture the interactions between multiple measures within individual measure characterizations. However, interactive effects with different end-uses are included in individual measure characterizations whenever possible³⁶. For instance, waste heat factors are included in the lighting characterizations to capture the interaction between more-efficient lighting measures and the amount of heating and/or cooling that is subsequently needed in the building.

By contrast, no effort is made to account for interactive effects between an efficient air conditioning measure and an efficient lighting measure, because it is impossible to know the specifics of the other measure in advance of its installation. For custom measures and projects where a bundle of measures is being implemented at the same time, these kinds of interactive effects should be estimated.

³⁶ For more information, please refer to the document, “Dealing with interactive Effects During Measure Characterization” Memo to the Stakeholder Advisory Group dated 12/13/11.
http://portal.veic.org/projects/illinoistrm/Shared%20Documents/Memos/Interactive_Effects_Memo_121311.docx

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Version 6.0

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VOLUME 4: CROSS CUTTING MEASURES AND ATTACHMENTS

Volume 2: Commercial and Industrial Measures

4.1 Agricultural End Use

4.1.1 Engine Block Timer for Agricultural Equipment

DESCRIPTION

The measure is a plug-in timer that is activated below a specific outdoor temperature to control an engine block heater in agricultural equipment. Engine block heaters are typically used during cold weather to pre-warm an engine prior to start, for convenience heaters are typically plugged in considerably longer than necessary to improve startup performance. A timer allows a user to preset the heater to come on for only the amount of time necessary to pre-warm the engine block, reducing unnecessary run time even if the baseline equipment has an engine block temperature sensor.

This measure was developed to be applicable to the following program types: RF. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient measure is an engine block heater operated by an outdoor plug-in timer (15 amp or greater) that turns on the heater only when the outdoor temperature is below 25 °F.

DEFINITION OF BASELINE EQUIPMENT

The baseline scenario is an engine block heater that is manually plugged in by the farmer to facilitate equipment startup at a later time.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 3 years¹

DEEMED MEASURE COST

The incremental cost per installed plug-in timer is \$10.19².

COINCIDENCE FACTOR

Engine block timers only operate in the winter so the summer peak demand savings is zero.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\begin{aligned}\Delta\text{kWh} &= \text{ISR} * \text{Use Season} * \% \text{Days} * \text{HrSave/Day} * \text{kW}_{\text{heater}} - \text{ParaLd} \\ &= 78.39\% * 87 \text{ days} * 84.23\% * 7.765 \text{ Hr/Day} * 1.5 \text{ kW} - 5.46 \text{ kWh} \\ &= 664 \text{ kWh}\end{aligned}$$

¹Equipment life is expected to be longer, but measure life is more conservative to account for possible attrition in use over time.

²Based on bulk pricing reported by EnSave, which administers the rebate in Vermont

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-APL-ESDH-V01-120601

REVIEW DEADLINE: 1/1/2019

4.1.2 High Volume Low Speed Fans

DESCRIPTION

The measure applies to 20-24 foot diameter horizontally mounted ceiling high volume low speed (HVLS) fans that are replacing multiple non HVLS fans that have reached the end of useful life in agricultural applications.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be classified as HVLS and have a VFD³.

DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline condition is assumed to be multiple non HVLS existing fans that have reached the end of s useful life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years⁴.

DEEMED MEASURE COST

The incremental capital cost for the fans are as follows⁵:

Fan Diameter Size (feet)	Incremental Cost
20	\$4150
22	\$4180
24	\$4225

LOADSHAPE

Loadshape C34 - Industrial Motor

COINCIDENCE FACTOR

The measure has deemed kW savings therefor a coincidence factor is not applied.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS ⁶

The annual electric savings from this measure are deemed values depending on fan size and apply to all building types:

Fan Diameter Size (feet)	kWh Savings
20	6577

³ Act on Energy Commercial Technical Reference Manual No. 2010-4

⁴ Ibid.

⁵ Ibid.

⁶ Ibid.

Fan Diameter Size (feet)	kWh Savings
22	8543
24	10018

SUMMER COINCIDENT PEAK DEMAND SAVINGS⁷

The annual kW savings from this measure are deemed values depending on fan size and apply to all building types:

Fan Diameter Size (feet)	kW Savings
20	2.4
22	3.1
24	3.7

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-AGE-HVSF-V01-120601

REVIEW DEADLINE: 1/1/2019

⁷ Ibid.

4.1.3 High Speed Fans

DESCRIPTION

The measure applies to high speed exhaust, ventilation and circulation fans that are replacing an existing unit that reached the end of its useful life in agricultural applications.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be diffuser equipped and meet the following criteria⁸.

Diameter of Fan (inches)	Minimum Efficiency for Exhaust & Ventilation Fans	Minimum Efficiency for Circulation Fans
24 through 35	14.0 cfm/W at 0.10 static pressure	12.5 lbf/kW
36 through 47	17.1 cfm/W at 0.10 static pressure	18.2 lbf/kW
48 through 71	20.3 cfm/W at 0.10 static pressure	23.0 lbf/kW

DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline condition is assumed to be an existing fan that reached the end of its useful life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 7 years⁹.

DEEMED MEASURE COST

The incremental capital cost for all fan sizes is \$150¹⁰.

LOADSHAPE

Loadshape C34 - Industrial Motor

COINCIDENCE FACTOR

The measure has deemed kW savings therefor a coincidence factor is not applied.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS¹¹**

The annual electric savings from this measure are deemed values depending on fan size and apply to all building types:

⁸ Act on Energy Commercial Technical Reference Manual No. 2010-4

⁹ Ibid.

¹⁰ Ibid.

¹¹ Ibid.

Diameter of Fan (inches)	kWh
24 through 35	372
36 through 47	625
48 through 71	1122

SUMMER COINCIDENT PEAK DEMAND SAVINGS¹²

The annual kW savings from this measure are deemed values depending on fan size and apply to all building types:

Diameter of Fan (inches)	kW
24 through 35	0.118
36 through 47	0.198
48 through 71	0.356

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-AGE-HSF_-V01-120601

REVIEW DEADLINE: 1/1/2019

¹² Ibid.

4.1.4 Live Stock Waterer

DESCRIPTION

This measure applies to the replacement of electric open waterers with sinking or floating water heaters with equivalent herd size watering capacity of the old unit.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to an electrically heated thermally insulated waterer with minimum 2 inches of insulation. A thermostat is required on unit with heating element greater than or equal to 250 watts¹³.

DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline equipment is assumed to be an electric open waterer with sinking or floating water heaters that have reached the end of useful life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years¹⁴.

DEEMED MEASURE COST

The incremental capital cost for the waters are \$787.50:¹⁵

LOADSHAPE

Loadshape C04 - Non-Residential Electric Heating

COINCIDENCE FACTOR

The measure has deemed kW savings therefor a coincidence factor is not applied

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS ¹⁶

The annual electric savings from this measure is a deemed value and assumed to be 1592.85 kWh.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

The annual kW savings from this measure is a deemed value and assumed to be 0.525 kW. ¹⁷

¹³ Act on Energy Commercial Technical Reference Manual No. 2010-4

¹⁴ Ibid.

¹⁵ Ibid.

¹⁶ Ibid.

¹⁷ Ibid.

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-AGE-LSW1-V01-120601

REVIEW DEADLINE: 6/1/2018

4.2 Food Service Equipment End Use

4.2.1 Combination Oven

DESCRIPTION

This measure applies to both natural gas fired and electric high efficiency combination convection and steam ovens installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS, RF. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure, the installed equipment must be a new natural gas or electric combination oven meeting the ENERGY STAR idle rate and cooking efficiency requirements as specified below.¹⁸

ENERGY STAR Requirements (Version 2.1, Effective January 1, 2014)

Fuel Type	Operation	Idle Rate (Btu/h for Gas, kW for Electric)	Cooking-Energy Efficiency, (%)
Natural Gas	Steam Mode	$\leq 200P+6,511$	≥ 41
	Convection Mode	$\leq 150P+5,425$	≥ 56
Electric	Steam Mode	$\leq 0.133P+0.6400$	≥ 55
	Convection Mode	$\leq 0.080P+0.4989$	≥ 76

Note: P = Pan capacity as defined in Section 1.S, of the Commercial Ovens Program Requirements Version 2.1¹⁹

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is a natural gas or electric combination oven that is not ENERGY STAR certified.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years.²⁰

DEEMED MEASURE COST

The costs vary based on the efficiency and make of the equipment. Actual costs should be used.

LOADSHAPE

Loadshape C01 - Commercial Electric Cooking

COINCIDENCE FACTOR

Summer Peak Coincidence Factor for measure is provided below for different building type²¹:

Location	CF
Fast Food Limited Menu	0.32
Fast Food Expanded Menu	0.41

¹⁸ ENERGY STAR Commercial Ovens Key Product Criteria

http://www.energystar.gov/index.cfm?c=ovens.pr_crit_comm_ovens

¹⁹ Pan capacity is defined as the number of steam table pans the combination oven is able to accommodate as per the ASTM F-1495-05 standard specification.

<http://www.energystar.gov/products/specs/system/files/Commercial%20Ovens%20Program%20Requirements%20V2%201.pdf?965d-c5ec&3b06-d2f5>

²⁰ <http://www.fishnick.com/saveenergy/tools/calculators/gcombiCalc.php>

²¹ Values taken from Minnesota Technical Reference Manual, 'Electric Oven and Range' measure and is based upon "Project on Restaurant Energy Performance-End-Use Monitoring and Analysis", Appendixes I and II, Claar, et. al., May 1985

Location	CF
Pizza	0.46
Full Service Limited Menu	0.51
Full Service Expanded Menu	0.36
Cafeteria	0.36

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

The algorithm below applies to electric combination ovens only.²²

$$\Delta kWh = (\Delta CookingEnergy_{ConvElec} + \Delta CookingEnergy_{SteamElec} + \Delta IdleEnergy_{ConvElec} + \Delta IdleEnergy_{SteamElec}) * Days / 1,000$$

Where:

$\Delta CookingEnergy_{ConvElec}$ = Change in total daily cooking energy consumed by electric oven in convection mode

$$= LB_{Elec} * (EFOOD_{ConvElec} / ElecEFF_{ConvBase} - EFOOD_{ConvElec} / ElecEFF_{ConvEE}) * \%Conv$$

$\Delta CookingEnergy_{SteamElec}$ = Change in total daily cooking energy consumed by electric oven in steam mode

$$= LB_{Elec} * (EFOOD_{SteamElec} / ElecEFF_{SteamBase} - EFOOD_{SteamElec} / ElecEFF_{SteamEE}) * \%Steam$$

$\Delta IdleEnergy_{ConvElec}$ = Change in total daily idle energy consumed by electric oven in convection mode

$$= [(ElecIDLE_{ConvBase} * ((HOURS - LB_{Elec} / ElecPC_{ConvBase}) * \%Conv)) - (ElecIDLE_{ConvEE} * ((HOURS - LB_{Elec} / ElecPC_{ConvEE}) * \%Conv))]$$

$\Delta IdleEnergy_{SteamElec}$ = Change in total daily idle energy consumed by electric oven in convection mode

$$= [(ElecIDLE_{SteamBase} * ((HOURS - LB_{Elec} / ElecPC_{SteamBase}) * \%Steam)) - (ElecIDLE_{SteamEE} * ((HOURS - LB_{Elec} / ElecPC_{SteamEE}) * \%Steam))]$$

Where:

LB_{Elec} = Estimated mass of food cooked per day for electric oven (lbs/day)

= Custom, or if unknown, use 200 lbs (If P < 15) or 250 lbs (If P >= 15)

$EFOOD_{ConvElec}$ = Energy absorbed by food product for electric oven in convection mode

= Custom or if unknown, use 73.2 Wh/lb

$ElecEFF$ = Cooking energy efficiency of electric oven

= Custom or if unknown, use values from table below

²² Algorithms and assumptions derived from ENERGY STAR Commercial Kitchen Equipment Savings Calculator
https://www.energystar.gov/sites/default/files/asset/document/commercial_kitchen_equipment_calculator.xlsx

	Base	EE
ElecEFF _{Conv}	72%	76%
ElecEFF _{Steam}	49%	55%

%_{Conv} = Percentage of time in convection mode

= Custom or if unknown, use 50%

E_{FOOD}_{SteamElec} = Energy absorbed by food product for electric oven in steam mode

= Custom or if unknown, use 30.8 Wh/lb

%_{steam} = Percentage of time in steam mode

= 1 - %_{conv}

ElecIDLE_{Base} = Idle energy rate (W) of baseline electric oven

= Custom or if unknown, use values from table below

Pan Capacity	Convection Mode (ElecIDLE _{ConvBase})	Steam Mode (ElecIDLE _{SteamBase})
< 15	1,320	5,260
> = 15	2,280	8,710

HOURS = Average daily hours of operation

= Custom or if unknown, use 12 hours

ElecPC_{Base} = Production capacity (lbs/hr) of baseline electric oven

= Custom of if unknown, use values from table below

Pan Capacity	Convection Mode (ElecPC _{ConvBase})	Steam Mode (ElecPC _{SteamBase})
< 15	79	126
> = 15	166	295

ElecIDLE_{ConvEE} = Idle energy rate of ENERGY STAR electric oven in convection mode

= (0.08*P + 0.4989)*1000

ElecPC_{EE} = Production capacity (lbs/hr) of ENERGY STAR electric oven

= Custom of if unknown, use values from table below

Pan Capacity	Convection Mode (ElecPC _{ConvEE})	Steam Mode (ElecPC _{SteamEE})
< 15	119	177
> = 15	201	349

ElecIDLE_{SteamEE} = Idle energy rate of ENERGY STAR electric oven in steam mode

= (0.133* P + 0.64)*1000

Days = Days of operation per year

= Custom or if unknown, use 365 days per year

1,000 = Wh to kWh conversion factor

EXAMPLE

For example, a 10-pan capacity electric combination oven would save:

$$\Delta kWh = (\Delta CookingEnergy_{ConvElec} + \Delta CookingEnergy_{SteamElec} + \Delta IdleEnergy_{ConvElec} + \Delta IdleEnergy_{SteamElec}) * Days / 1,000$$

$$\begin{aligned}\Delta CookingEnergy_{ConvElec} &= 200 * (73.2 / 0.72 - 73.2 / 0.76) * 0.50 \\ &= 535 \text{ Wh}\end{aligned}$$

$$\begin{aligned}\Delta CookingEnergy_{SteamElec} &= 200 * (30.8 / 0.49 - 30.8 / 0.55) * (1 - 0.50) \\ &= 686 \text{ Wh}\end{aligned}$$

$$\begin{aligned}\Delta IdleEnergy_{ConvElec} &= [(1,320 * ((12 - 200/79) * 0.50)) - (1,299 * ((12 - 200/119) * 0.50))] \\ &= -453 \text{ Wh}\end{aligned}$$

$$\begin{aligned}\Delta IdleEnergy_{SteamElec} &= [(5,260 * ((12 - 200/126) * (1 - 0.50))) - (1,970 * ((12 - 200/177) * (1 - 0.50)))] \\ &= 16,678 \text{ Wh}\end{aligned}$$

$$\begin{aligned}\Delta kWh &= (535 + 686 + -453 + 16,678) * 365 / 1,000 \\ &= 6,368 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh / (HOURS * DAYS) * CF$$

Where:

CF = Summer peak coincidence factor is dependent on building type²³:

Location	CF
Fast Food Limited Menu	0.32
Fast Food Expanded Menu	0.41
Pizza	0.46
Full Service Limited Menu	0.51
Full Service Expanded Menu	0.36
Cafeteria	0.36

All other variables as defined above.

EXAMPLE

For example, a 10-pan capacity electric combination oven in a Full Service Limited Menu restaurant would save:

$$\begin{aligned}\Delta kW &= \Delta kWh / (HOURS * DAYS) * CF \\ &= 6,368 / (12 * 365) * 0.51 \\ &= 0.74 \text{ kW}\end{aligned}$$

²³Values taken from Minnesota Technical Reference Manual, 'Electric Oven and Range' measure and is based upon "Project on Restaurant Energy Performance-End-Use Monitoring and Analysis", Appendixes I and II, Claar, et. al., May 1985

NATURAL GAS ENERGY SAVINGS

The algorithm below applies to natural gas combination ovens only.²⁴

$$\Delta \text{Therms} = (\Delta \text{CookingEnergy}_{\text{ConvGas}} + \Delta \text{CookingEnergy}_{\text{SteamGas}} + \Delta \text{IdleEnergy}_{\text{ConvGas}} + \Delta \text{IdleEnergy}_{\text{SteamGas}}) * \text{Days} / 100,000$$

Where:

$\Delta \text{CookingEnergy}_{\text{ConvGas}}$ = Change in total daily cooking energy consumed by gas oven in convection mode

$$= \text{LB}_{\text{Gas}} * (\text{EFOOD}_{\text{ConvGas}} / \text{GasEFF}_{\text{ConvBase}} - \text{EFOOD}_{\text{ConvGas}} / \text{GasEFF}_{\text{ConvEE}}) * \%_{\text{Conv}}$$

$\Delta \text{CookingEnergy}_{\text{SteamGas}}$ = Change in total daily cooking energy consumed by gas oven in steam mode

$$= \text{LB}_{\text{Gas}} * (\text{EFOOD}_{\text{SteamGas}} / \text{GasEFF}_{\text{SteamBase}} - \text{EFOOD}_{\text{SteamGas}} / \text{GasEFF}_{\text{SteamEE}}) * \%_{\text{Steam}}$$

$\Delta \text{IdleEnergy}_{\text{ConvGas}}$ = Change in total daily idle energy consumed by gas oven in convection mode

$$= [(\text{GasIDLE}_{\text{ConvBase}} * ((\text{HOURS} - \text{LB}_{\text{Gas}} / \text{GasPC}_{\text{ConvBase}}) * \%_{\text{Conv}})) - (\text{GasIDLE}_{\text{ConvEE}} * ((\text{HOURS} - \text{LB}_{\text{Gas}} / \text{GasPC}_{\text{ConvEE}}) * \%_{\text{Conv}}))]$$

$\Delta \text{IdleEnergy}_{\text{SteamGas}}$ = Change in total daily idle energy consumed by gas oven in convection mode

$$= [(\text{GasIDLE}_{\text{SteamBase}} * ((\text{HOURS} - \text{LB}_{\text{Gas}} / \text{GasPC}_{\text{SteamBase}}) * \%_{\text{Steam}})) - (\text{GasIDLE}_{\text{SteamEE}} * ((\text{HOURS} - \text{LB}_{\text{Gas}} / \text{GasPC}_{\text{SteamEE}}) * \%_{\text{Steam}}))]$$

Where:

LB_{Gas} = Estimated mass of food cooked per day for gas oven (lbs/day)

= Custom, or if unknown, use 200 lbs (If $P < 15$), 250 lbs (If $15 \leq P < 30$), or 400 lbs (If $P \geq 30$)

$\text{EFOOD}_{\text{ConvGas}}$ = Energy absorbed by food product for gas oven in convection mode

= Custom or if unknown, use 250 Btu/lb

GasEFF = Cooking energy efficiency of gas oven

= Custom or if unknown, use values from table below

	Base	EE
$\text{GasEFF}_{\text{Conv}}$	52%	56%
$\text{GasEFF}_{\text{Steam}}$	39%	41%

$\text{EFOOD}_{\text{SteamGas}}$ = Energy absorbed by food product for gas oven in steam mode

= Custom or if unknown, use 105 Btu/lb

$\text{GasIDLE}_{\text{Base}}$ = Idle energy rate (Btu/hr) of baseline gas oven

= Custom or if unknown, use values from table below

²⁴ Algorithms and assumptions derived from ENERGY STAR Commercial Kitchen Equipment Savings Calculator
https://www.energystar.gov/sites/default/files/asset/document/commercial_kitchen_equipment_calculator.xlsx

Pan Capacity	Convection Mode (GasIDLE _{ConvBase})	Steam Mode (GasIDLE _{SteamBase})
< 15	8,747	18,656
15-30	10,788	24,562
>30	13,000	43,300

GasPC_{Base} = Production capacity (lbs/hr) of baseline gas oven
 = Custom of if unknown, use values from table below

Pan Capacity	Convection Mode (GasPC _{ConvBase})	Steam Mode (GasPC _{SteamBase})
< 15	125	195
15-30	176	211
>30	392	579

GasIDLE_{ConvEE} = Idle energy rate of ENERGY STAR gas oven in convection mode
 = $150 * P + 5,425$

GasPC_{EE} = Production capacity (lbs/hr) of ENERGY STAR gas oven
 = Custom of if unknown, use values from table below

Pan Capacity	Convection Mode (GasPC _{ConvEE})	Steam Mode (GasPC _{SteamEE})
< 15	124	172
15-30	210	277
>30	394	640

GasIDLE_{SteamEE} = Idle energy rate of ENERGY STAR gas oven in steam mode
 = $200 * P + 6511$

100,000 = Conversion factor from Btu to therms

All other variables as defined above.

EXAMPLE

For example, a 10-pan capacity gas combination oven would save:

$$\begin{aligned}\Delta\text{Therms} &= (\Delta\text{CookingEnergy}_{\text{ConvGas}} + \Delta\text{CookingEnergy}_{\text{SteamGas}} + \Delta\text{IdleEnergy}_{\text{ConvGas}} + \Delta\text{IdleEnergy}_{\text{SteamGas}}) * \text{Days} / 100,000 \\ \Delta\text{CookingEnergy}_{\text{ConvGas}} &= 200 * (250 / 0.52 - 250 / 0.56) * 0.50 \\ &= 3,434 \text{ therms} \\ \Delta\text{CookingEnergy}_{\text{SteamGas}} &= 200 * (105 / 0.39 - 105 / 0.41) * (1 - 0.50) \\ &= 1,313 \text{ therms} \\ \Delta\text{IdleEnergy}_{\text{ConvGas}} &= [(8,747 * ((12 - 200/125) * 0.50)) - (6,925 * ((12 - 200/124) * 0.50))] \\ &= 9,519 \text{ therms} \\ \Delta\text{IdleEnergy}_{\text{SteamGas}} &= [(18,658 * ((12 - 200/195) * (1 - 0.50))) - (8,511 * ((12 - 200/172) * (1 - 0.50)))] \\ &= 56,251 \text{ therms} \\ \Delta\text{Therms} &= (3,434 + 1,313 + 9,519 + 56,251) * 365 / 100,000 \\ &= 257 \text{ therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-FSE-CBOV-V02-160601

REVIEW DEADLINE: 1/1/2023

4.2.2 Commercial Solid and Glass Door Refrigerators & Freezers

DESCRIPTION

This measure relates to the installation of a new reach-in commercial refrigerator or freezer meeting ENERGY STAR efficiency standards. ENERGY STAR labeled commercial refrigerators and freezers are more energy efficient because they are designed with components such as ECM evaporator and condenser fan motors, hot gas anti-sweat heaters, or high-efficiency compressors, which will significantly reduce energy consumption.

This measure was developed to be applicable to the following program types: TOS and NC. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a new vertical solid or glass door refrigerator or freezer or vertical chest freezer meeting the minimum ENERGY STAR efficiency level standards.

DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline equipment is assumed to be an existing solid or glass door refrigerator or freezer meeting the minimum federal manufacturing standards as specified by the Energy Policy Act of 2005.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years²⁵.

DEEMED MEASURE COST

The incremental capital cost for this measure is provided below²⁶.

Type	Refrigerator incremental Cost, per unit	Freezer Incremental Cost, per unit
Solid or Glass Door		
0 < V < 15	\$143	\$142
15 ≤ V < 30	\$164	\$166
30 ≤ V < 50	\$164	\$166
V ≥ 50	\$249	\$407

LOADSHAPE

Loadshape C23 - Commercial Refrigeration

²⁵2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, "Effective/Remaining Useful Life Values", California Public Utilities Commission, December 16, 2008.

<http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf>

²⁶Estimates of the incremental cost of commercial refrigerators and freezers varies widely by source. Nadel, S., Packaged Commercial Refrigeration Equipment: A Briefing Report for Program Planners and Implementers, ACEEE, December 2002, indicates that incremental cost is approximately zero. Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, February, 19, 2010, assumed incremental cost ranging from \$75 to \$125 depending on equipment volume. ACEEE notes that incremental cost ranges from 0 to 10% of the baseline unit cost <http://www.aceee.org/ogeece/ch5_reach.htm>. For the purposes of this characterization, assume and incremental cost adder of 5% on the full unit costs presented in Goldberg et al, State of Wisconsin Public Service Commission of Wisconsin, Focus on Energy Evaluation, Business Programs: Incremental Cost Study, KEMA, October 28, 2009.

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 0.937.²⁷

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = (\text{kWh}_{\text{base}} - \text{kWh}_{\text{ee}}) * 365.25$$

Where:

kWh_{base} = baseline maximum daily energy consumption in kWh

= calculated using actual chilled or frozen compartment volume (V) of the efficient unit as shown in the table below.

Type	$\text{kWh}_{\text{base}}^{28}$
Solid Door Refrigerator	$0.10 * V + 2.04$
Glass Door Refrigerator	$0.12 * V + 3.34$
Solid Door Freezer	$0.40 * V + 1.38$
Glass Door Freezer	$0.75 * V + 4.10$

$\text{kWh}_{\text{ee}}^{29}$ = efficient maximum daily energy consumption in kWh

= calculated using actual chilled or frozen compartment volume (V) of the efficient unit as shown in the table below.

Type	Refrigerator kWh _{ee}	Freezer kWh _{ee}
$0 < V < 15$	$\leq 0.089V + 1.411$	$\leq 0.250V + 1.250$
$15 \leq V < 30$	$\leq 0.037V + 2.200$	$\leq 0.400V - 1.000$
$30 \leq V < 50$	$\leq 0.056V + 1.635$	$\leq 0.163V + 6.125$
$V \geq 50$	$\leq 0.060V + 1.416$	$\leq 0.158V + 6.333$
Glass Door		
$0 < V < 15$	$\leq 0.118V + 1.382$	$\leq 0.607V + 0.893$
$15 \leq V < 30$	$\leq 0.140V + 1.050$	$\leq 0.733V - 1.000$
$30 \leq V < 50$	$\leq 0.088V + 2.625$	$\leq 0.250V + 13.500$
$V \geq 50$	$\leq 0.110V + 1.500$	$\leq 0.450V + 3.500$

V = the chilled or frozen compartment volume (ft³) (as defined in the Association of Home Appliance Manufacturers Standard HRF1–1979)

= Actual installed

²⁷ The CF for Commercial Refrigeration was calculated based upon the Ameren provided eShapes

²⁸ Energy Policy Act of 2005. Accessed on 7/7/10. <http://www.epa.gov/oust/fedlaws/publ_109-058.pdf>

²⁹ ENERGY STAR Program Requirements for Commercial Refrigerators and Freezers Partner Commitments Version 2.0, U.S. Environmental Protection Agency, Accessed on 7/7/10. <

http://www.energystar.gov/ia/partners/product_specs/program_reqs/commer_refrig_glass_prog_req.pdf>

365.25 = days per year

For example a solid door refrigerator with a volume of 15 would save

$$\begin{aligned}\Delta \text{kWh} &= (3.54 - 2.76) * 365.25 \\ &= 285 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \Delta \text{kWh} / \text{HOURS} * \text{CF}$$

Where:

HOURS = equipment is assumed to operate continuously, 24 hours per day, 365.25 days per year.
= 8766

CF = Summer Peak Coincidence Factor for measure
= 0.937

For example a solid door refrigerator with a volume of 15 would save

$$\begin{aligned}\Delta \text{kW} &= 285 / 8766 * .937 \\ &= 0.030 \text{ kW}\end{aligned}$$

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-FSE-CSDO-V01-120601

REVIEW DEADLINE: 1/1/2019

4.2.3 Commercial Steam Cooker

DESCRIPTION

To qualify for this measure the installed equipment must be an ENERGY STAR® steamer in place of a standard steamer in a commercial kitchen. Savings are presented dependent on the pan capacity and corresponding idle rate at heavy load cooking capacity and if the steamer is gas or electric.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be as follows:

Gas	Electric
ENERGY STAR® qualified with 38% minimum cooking energy efficiency at heavy load (potato) cooking capacity for gas steam cookers.	ENERGY STAR® qualified with 50% minimum cooking energy efficiency at heavy load (potato) cooking capacity for electric steam cookers.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is assumed to be a non-ENERGY STAR® commercial steamer at end of life. It is assumed that the efficient equipment and baseline equipment have the same number of pans.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years³⁰

DEEMED MEASURE COST

The incremental capital cost for this measure is \$998³¹ for a natural gas steam cooker or \$2490³² for an electric steam cooker.

LOADSHAPE

Loadshape C01 - Commercial Electric Cooking

COINCIDENCE FACTOR

Summer Peak Coincidence Factor for measure is provided below for different building type³³:

Location	CF
Fast Food Limited Menu	0.32
Fast Food Expanded Menu	0.41
Pizza	0.46
Full Service Limited Menu	0.51
Full Service Expanded Menu	0.36

³⁰California DEER 2008 which is also used by both the Food Service Technology Center and ENERGY STAR®.

³¹Source for incremental cost for efficient natural gas steamer is RSG Commercial Gas Steamer Workpaper, January 2012.

³²Source for efficient electric steamer incremental cost is \$2,490 per 2009 PG&E Workpaper - PGECOFST104.1 - Commercial Steam Cooker - Electric and Gas as reference by KEMA in the ComEd C & I TRM.

³³ Values taken from Minnesota Technical Reference Manual, 'Electric Oven and Range' measure and is based upon "Project on Restaurant Energy Performance-End-Use Monitoring and Analysis", Appendixes I and II, Claar, et. al., May 1985. Unknown is an average of other location types

Location	CF
Cafeteria	0.36
Unknown	0.40

Algorithm

CALCULATION OF SAVINGS

Formulas below are applicable to both gas and electric steam cookers. Please use appropriate lookup values and identified flags.

ENERGY SAVINGS

$$\Delta \text{Savings} = (\Delta \text{Idle Energy} + \Delta \text{Preheat Energy} + \Delta \text{Cooking Energy}) * Z$$

$$\text{For a gas cooker:} \quad \Delta \text{Savings} = \Delta \text{Btu} * 1/100,000 * Z$$

$$\text{For an electric steam cooker:} \quad \Delta \text{Savings} = \Delta \text{kWh} * Z$$

Where:

Z = days/yr steamer operating (use 365.25 days/yr if heavy use restaurant and exact number unknown)

$$\Delta \text{Idle Energy} = (((1 - \text{CSM}_{\% \text{Baseline}}) * \text{IDLE}_{\text{BASE}} + \text{CSM}_{\% \text{Baseline}} * \text{PC}_{\text{BASE}} * E_{\text{FOOD}} / \text{EFF}_{\text{BASE}}) * (\text{HOURS}_{\text{day}} - (F / \text{PC}_{\text{Base}}) - (\text{PRE}_{\text{number}} * 0.25))) - (((1 - \text{CSM}_{\% \text{ENERGYSTAR}}) * \text{IDLE}_{\text{ENERGYSTAR}} + \text{CSM}_{\% \text{ENERGYSTAR}} * \text{PC}_{\text{ENERGY}} * E_{\text{FOOD}} / \text{EFF}_{\text{ENERGYSTAR}}) * (\text{HOURS}_{\text{Day}} - (F / \text{PC}_{\text{ENERGY}}) - (\text{PRE}_{\text{number}} * 0.25))))$$

Where:

CSM_{%Baseline} = Baseline Steamer Time in Manual Steam Mode (% of time)
= 90%³⁴

IDLE_{Base} = Idle Energy Rate of Base Steamer³⁵

Number of Pans	IDLE _{BASE} - Gas, Btu/hr	IDLE _{BASE} - Electric, kw
3	11,000	1.0
4	14,667	1.33
5	18,333	1.67
6	22,000	2.0

PC_{Base} = Production Capacity of Base Steamer³⁶

Number of Pans	PC _{BASE, gas} (lbs/hr)	PC _{BASE, electric} (lbs/hr)
3	65	70
4	87	93

³⁴Food Service Technology Center 2011 Savings Calculator

³⁵Food Service Technology Center 2011 Savings Calculator

³⁶Production capacity per Food Service Technology Center 2011 Savings Calculator of 23.3333 lb/hr per pan for electric baseline steam cookers and 21.6667 lb/hr per pan for natural gas baseline steam cookers. ENERGY STAR® savings calculator uses 23.3 lb/hr per pan for both electric and natural gas baseline steamers.

Number of Pans	PC _{BASE, gas} (lbs/hr)	PC _{BASE, electric} (lbs/hr)
5	108	117
6	130	140

E_{FOOD} = Amount of Energy Absorbed by the food during cooking known as ASTM Energy to Food (Btu/lb or kW/lb)

=105 Btu/lb³⁷ (gas steamers) or 0.0308⁸ (electric steamers)

EFF_{BASE} = Heavy Load Cooking Efficiency for Base Steamer

=15%³⁸ (gas steamers) or 26%⁹ (electric steamers)

$HOURS_{\text{day}}$ = Average Daily Operation (hours)

Type of Food Service	Hours _{day} ³⁹
Fast Food, limited menu	4
Fast Food, expanded menu	5
Pizza	8
Full Service, limited menu	8
Full Service, expanded menu	7
Cafeteria	6
Unknown	6 ⁴⁰
Custom	Varies

F = Food cooked per day (lbs/day)

= custom or if unknown, use 100 lbs/day⁴¹

$CSM_{\% \text{ENERGYSTAR}}$ = ENERGY STAR Steamer's Time in Manual Steam Mode (% of time)⁴²

= 0%

$IDLE_{\text{ENERGYSTAR}}$ = Idle Energy Rate of ENERGY STAR⁴³

Number of Pans	IDLE _{ENERGY STAR} – gas, (Btu/hr)	IDLE _{ENERGY STAR} – electric, (kW)
3	6250	0.40
4	8333	0.53
5	10417	0.67

³⁷Reference ENERGY STAR® savings calculator at

http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COC.

³⁸Reference Food Service Technology Center 2011 Savings Calculator values as used by Consortium for Energy Efficiency, Inc. for baseline electric and natural gas steamer heavy cooking load energy efficiencies.

³⁹ Values taken from Minnesota Technical Reference Manual, 'Electric Oven and Range' measure and is based upon "Project on Restaurant Energy Performance-End-Use Monitoring and Analysis", Appendixes I and II, Claar, et. al., May 1985.

⁴⁰Unknown is average of other locations

⁴¹Reference amount used by both Food Service Technology Center and ENERGY STAR® savings calculator

⁴²Reference information from the Food Service Technology Center citing that ENERGY STAR® steamers are not typically operated in constant steam mode, but rather are used in timed mode. Reference ENERGY STAR® savings calculator at http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COC for efficient steamer. Both baseline & efficient steamer mode values should be considered for users in Illinois market.

⁴³Food Service Technology Center 2011 Savings Calculator

Number of Pans	IDLE _{ENERGY STAR} – gas, (Btu/hr)	IDLE _{ENERGY STAR} – electric, (kW)
6	12500	0.80

PC_{ENERGY} = Production Capacity of ENERGY STAR® Steamer⁴⁴

Number of Pans	PC_{ENERGY} – gas (lbs/hr)	PC_{ENERGY} – electric (lbs/hr)
3	55	50
4	73	67
5	92	83
6	110	100

$EFF_{ENERGYSTAR}$ = Heavy Load Cooking Efficiency for ENERGY STAR® Steamer(%)

=38%⁴⁵ (gas steamer) or 50%¹⁵ (electric steamer)

PRE_{number} = Number of preheats per day

=1⁴⁶ (if unknown, use 1)

$\Delta Preheat Energy = (PRE_{number} * \Delta Preheat)$

Where:

PRE_{number} = Number of Preheats per Day

=1⁴⁷(if unknown, use 1)

PRE_{heat} = Preheat energy savings per preheat

= 11,000 Btu/preheat⁴⁸ (gas steamer) or 0.5 kWh/preheat⁴⁹ (electric steamer)

$\Delta Cooking Energy = ((1/ EFF_{BASE}) - (1/ EFF_{ENERGY STAR})) * F * E_{FOOD}$

Where:

EFF_{BASE} =Heavy Load Cooking Efficiency for Base Steamer

=15%⁵⁰ (gas steamer) or 26%²⁸ (electric steamer)

⁴⁴Production capacity per Food Service Technology Center 2011 Savings Calculator of 18.3333 lb/hr per pan for gas ENERGY STAR® steam cookers and 16.6667 lb/hr per pan for electric ENERGY STAR® steam cookers. ENERGY STAR® savings calculator uses 16.7 lb/hr per pan for electric and 20 lb/hr for natural gas ENERGY STAR® steamers.

⁴⁵Reference Food Service Technology Center 2011 Savings Calculator values as used by Consortium for Energy Efficiency, Inc. for Tier 1A and Tier 1B qualified electric and natural gas steamer heavy cooking load energy efficiencies and http://www.energystar.gov/ia/partners/product_specs/program_reqs/Commercial_Steam_Cookers_Program_Requirements.pdf?7010-36eb

⁴⁶Reference ENERGY STAR® savings calculator at

http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COC and Food

⁴⁷Reference ENERGY STAR® savings calculator at

http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COC and Food

⁴⁸Ohio TRM which references 2002 Food Service Technology Center "Commercial Cooking Appliance Technology Assessment" Chapter 8: Steamers. This is time also used by ENERGY STAR® savings calculator at

http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COC. 11,000 Btu/preheat is from 72,000 Btu/hr * 15 min/hr /60 min/hr for gas steamers and 0.5 kWh/preheat is from 6 kW/preheat * 15 min/hr / 60 min/hr

⁴⁹Reference Food Service Technology Center 2011 Savings Calculator values for Baseline Preheat Energy.

⁵⁰Reference Food Service Technology Center 2011 Savings Calculator values as used by Consortium for Energy Efficiency, Inc. for baseline electric and natural gas steamer heavy cooking load energy efficiencies.

EFF _{ENERGYSTAR}	=Heavy Load Cooking Efficiency for ENERGY STAR® Steamer =38% ⁵¹ (gas steamer) or 50% ²³ (electric steamer)
F	= Food cooked per day (lbs/day) = custom or if unknown, use 100 lbs/day ⁵²
E _{FOOD}	= Amount of Energy Absorbed by the food during cooking known as ASTM Energy to Food ⁵³

E _{FOOD} - gas(Btu/lb)	E _{FOOD} (kWh/lb)
105 ⁵⁴	0.0308 ⁵⁵

EXAMPLE

For a gas steam cooker: A 3 pan steamer in a full service restaurant

$$\begin{aligned}\Delta \text{Savings} &= (\Delta \text{Idle Energy} + \Delta \text{Preheat Energy} + \Delta \text{Cooking Energy}) * Z * 1/100,000 \\ \Delta \text{Idle Energy} &= (((1 - 0.9) * 11000 + 0.9 * 65 * 105 / 0.15) * (7 - (100 / 65) - (1 * 0.25))) - (((1 - 0) * 6250 + 0 * 55 * 105 / 0.38) * (7 - (100 / 55) - (1 * 0.25))) \\ &= 188,321 \\ \Delta \text{Preheat Energy} &= (1 * 11,000) \\ &= 11,000 \\ \Delta \text{Cooking Energy} &= (((1 / 0.15) - (1 / 0.38)) * (100 \text{ lb/day} * 105 \text{ btu/lb})) \\ &= 42368 \\ \Delta \text{Therms} &= (188321 + 11000 + 42368) * 365.25 * 1/100,000 \\ &= 883 \text{ therms}\end{aligned}$$

For an electric steam cooker: A 3 pan steamer in a cafeteria:

$$\begin{aligned}\Delta \text{Savings} &= (\Delta \text{Idle Energy} + \Delta \text{Preheat Energy} + \Delta \text{Cooking Energy}) * Z \\ \Delta \text{Idle Energy} &= (((1 - .9) * 1.0 + .9 * 70 * 0.0308 / 0.26) * (6 - (100 / 70) - (1 * .25))) - (((1 - 0) * 0.4 + 0 * 50 * 0.0308 / 0.50) * (6 - (100 / 50) - (1 * 0.25))) \\ &= 31.18 \\ \Delta \text{Preheat Energy} &= (1 * 0.5) \\ &= 0.5 \\ \Delta \text{Cooking Energy} &= (((1 / 0.26) - (1 / 0.5)) * (100 * 0.0308)) \\ &= 5.69 \\ \Delta \text{kWh} &= (31.18 + 0.5 + 5.69) * 365.25 \text{ days}\end{aligned}$$

⁵¹ Ibid.

⁵² Amount used by both Food Service Technology Center and ENERGY STAR® savings calculator

⁵³ Reference ENERGY STAR® savings calculator at

http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COC.

⁵⁴ Ibid.

⁵⁵ Ibid.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

This is only applicable to the electric steam cooker.

$$\Delta kW = (\Delta kWh / (HOURS_{Day} * Days_{Year})) * CF$$

Where:

CF = Summer Peak Coincidence Factor for measure is provided below for different locations⁵⁶:

Location	CF
Fast Food Limited Menu	0.32
Fast Food Expanded Menu	0.41
Pizza	0.46
Full Service Limited Menu	0.51
Full Service Expanded Menu	0.36
Cafeteria	0.36

Days_{Year} = Annual Days of Operation
 = custom or 365.25 days a year
 Other values as defined above

EXAMPLE

For 3 pan electric steam cooker located in a cafeteria:

$$\begin{aligned} \Delta kW &= (\Delta kWh / (HOURS_{Day} * Days_{Year})) * CF \\ &= (13,649 / (6 * 365.25)) * 0.36 \\ &= 2.24 kW \end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

This is applicable to both gas and electric steam cookers.

$$\Delta Water = (W_{BASE} - W_{ENERGYSTAR}) * HOURS_{Day} * Days_{Year}$$

Where

W_{BASE} = Water Consumption Rate of Base Steamer (gal/hr)
 = 40⁵⁷

W_{ENERGYSTAR} = Water Consumption Rate of ENERGY STAR® Steamer look up⁵⁸

CEE Tier	gal/hr
Tier 1A	15
Tier 1B	4

⁵⁶Values taken from Minnesota Technical Reference Manual, 'Electric Oven and Range' measure and is based upon "Project on Restaurant Energy Performance-End-Use Monitoring and Analysis", Appendixes I and II, Claar, et. al., May 1985.

⁵⁷FSTC (2002). Commercial Cooking Appliance Technology Assessment. Chapter 8: Steamers.

⁵⁸Source Consortium for Energy Efficiency, Inc. September 2010 "Program Design Guidance for Steamers" for Tier 1A and Tier 1B water requirements. Ohio Technical Reference Manual 2010 for 10 gal/hr water consumption which can be used when Tier level is not known.

CEE Tier	gal/hr
Avg Efficient	10
Avg Most Efficient	3

Days_{Year} =Annual Days of Operation
 =custom or 365.25 days a year⁵⁹

EXAMPLE

For example, an electric 3 pan steamer with average efficiency in a full service restaurant

$$\Delta \text{Water} = (40 - 10) * 7 * 365.25$$

$$= 76,703 \text{ gallons}$$

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-FSE-STMC-V04-160601

REVIEW DEADLINE: 1/1/2023

⁵⁹Source for 365.25 days/yr is ENERGY STAR® savings calculator which references Food Service Technology research on average use, 2009.

4.2.4 Conveyor Oven

DESCRIPTION

This measure applies to natural gas fired high efficiency conveyor ovens installed in commercial kitchens replacing existing natural gas units with conveyor width greater than 25 inches.

Conveyor ovens are available using four different heating processes: infrared, natural convection with a ceramic baking hearth, forced convection or air impingement, or a combination of infrared and forced convection. Conveyor ovens are typically used for producing a limited number of products with similar cooking requirements at high production rates. They are highly flexible and can be used to bake or roast a wide variety of products including pizza, casseroles, meats, breads, and pastries.

Some manufacturers offer an air-curtain feature at either end of the cooking chamber that helps to keep the heated air inside the conveyor oven. The air curtain operates as a virtual oven wall and helps reduce both the idle energy of the oven and the resultant heat gain to the kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a natural gas conveyor oven with a tested baking energy efficiency > 42% and an idle energy consumption rate < 57,000 Btu/hr utilizing ASTM standard F1817.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an existing pizza deck oven at end of life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 17 years.⁶⁰

DEEMED MEASURE COST

The incremental capital cost for this measure is \$1800⁶¹.

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

⁶⁰See 'Arkansas Deemed TRM Table for GasFoodService.xls' from v3.0 Arkansas Technical Reference Manual.

⁶¹Ibid.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

The annual natural gas energy savings from this measure is a deemed value equaling 884 Therms⁶².

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-FSE-CVOV-V02-180101

REVIEW DEADLINE: 1/1/2024

⁶² The Resource Solutions Group Commercial Conveyor Oven – Gas workpaper from January 2012; Commercial Gas Conveyor Oven – Large Gas Savings (therms/unit).

4.2.5 ENERGY STAR Convection Oven

DESCRIPTION

This measure applies to natural gas fired ENERGY STAR convection ovens installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a natural gas convection oven with a cooking efficiency $\geq 46\%$ utilizing ASTM standard 1496 and an idle energy consumption rate $< 12,000$ Btu/hr⁶³

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is a natural gas convection oven that is not ENERGY STAR certified and is at end of life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years⁶⁴

DEEMED MEASURE COST

The incremental capital cost for this measure is \$50⁶⁵

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

⁶³ Version 2.2. of the ENERGY STAR specification.

⁶⁴ Lifetime from ENERGY STAR commercial griddle which cites reference as "FSTC research on available models, 2009"
http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

⁶⁵ Measure cost from ENERGY STAR which cites reference as "EPA research on available models using AutoQuotes, 2010"
http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

⁶⁶ Algorithms and assumptions derived from ENERGY STAR Oven Commercial Kitchen Equipment Savings Calculator.
http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

$$\Delta \text{Therms} = (\Delta \text{DailyIdle Energy} + \Delta \text{DailyPreheat Energy} + \Delta \text{DailyCooking Energy}) * \text{Days} / 100000$$

Where:

$$\Delta \text{DailyIdleEnergy} = (\text{IdleBase} * \text{IdleBaseTime}) - (\text{IdleENERGYSTAR} * \text{IdleENERGYSTARTime})$$

$$\Delta \text{DailyPreheatEnergy} = (\text{PreHeatNumberBase} * \text{PreheatTimeBase} / 60 * \text{PreheatRateBase}) - (\text{PreheatNumberENERGYSTAR} * \text{PreheatTimeENERGYSTAR} / 60 * \text{PreheatRateENERGYSTAR})$$

$$\Delta \text{DailyCookingEnergy} = (\text{LB} * \text{EFOOD} / \text{EffBase}) - (\text{LB} * \text{EFOOD} / \text{EffENERGYSTAR})$$

Where:

HOURSday = Average Daily Operation
= custom or if unknown, use 12 hours

Days = Annual days of operation
= custom or if unknown, use 365.25 days a year

LB = Food cooked per day
= custom or if unknown, use 100 pounds

EffENERGYSTAR = Cooking Efficiency ENERGY STAR
= custom or if unknown, use 46%

EffBase = Cooking Efficiency Baseline
= custom or if unknown, use 30%

PCENERGYSTAR = Production Capacity ENERGY STAR
= custom or if unknown, use 80 pounds/hr

PCBase = Production Capacity base
= custom or if unknown, use 70 pounds/hr

PreheatNumberENERGYSTAR = Number of preheats per day
= custom or if unknown, use 1

PreheatNumberBase = Number of preheats per day
= custom or if unknown, use 1

PreheatTimeENERGYSTAR = preheat length
= custom or if unknown, use 15 minutes

PreheatTimeBase = preheat length
= custom or if unknown, use 15 minutes

PreheatRateENERGYSTAR = preheat energy rate high efficiency
= custom or if unknown, use 44000 btu/h

PreheatRateBase = preheat energy rate baseline
= custom or if unknown, use 76000 btu/h

IdleENERGYSTAR = Idle energy rate
= custom or if unknown, use 12000 btu/h

IdleBase	= Idle energy rate = custom or if unknown, use 18000 btu/h
IdleENERGYSTARTTime	= ENERGY STAR Idle Time = $\text{HOURSday-LB/PCENERGYSTAR -PreHeatTimeENERGYSTAR/60}$ = $12 - 100/80 - 15/60$ =10.5 hours
IdleBaseTime	= BASE Idle Time = $\text{HOURSday-LB/PCbase -PreHeatTimeBase/60}$ =Custom or if unknown, use = $12 - 100/70-15/60$ =10.3 hours
EFOOD	= ASTM energy to food = 250 btu/pound

EXAMPLE

For example, an ENERGY STAR Oven with a cooking energy efficiency of 46% and default values from above would save.

$$\Delta \text{Therms} = (\Delta \text{Idle Energy} + \Delta \text{Preheat Energy} + \Delta \text{Cooking Energy}) * \text{Days} / 100000$$

Where:

$\Delta \text{DailyIdleEnergy}$	$= (18000 * 10.3) - (12000 * 10.5)$ $= 59,400 \text{ btu}$
$\Delta \text{DailyPreheatEnergy}$	$= (1 * 15 / 60 * 76000) - (1 * 15 / 60 * 44000)$ $= 8,000 \text{ btu}$
$\Delta \text{DailyCookingEnergy}$	$= (100 * 250 / .30) - (100 * 250 / .46)$ $= 28,986 \text{ btu}$
ΔTherms	$= (59,400 + 8,000 + 28,986) * 365.25 / 100000$ $= 352 \text{ therms}$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-FSE-ESCV-V02-180101

REVIEW DEADLINE: 1/1/2024

4.2.6 ENERGY STAR Dishwasher

DESCRIPTION

This measure applies to ENERGY STAR high and low temp under counter, stationary single tank door type, single tank conveyor, and multiple tank conveyor dishwashers, as well as high temp pot, pan, and utensil dishwashers installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be an ENERGY STAR certified dishwasher meeting idle energy rate (kW) and water consumption (gallons/rack) limits, as determined by both machine type and sanitation approach (chemical/low temp versus high temp).

ENERGY STAR Requirements (Effective February 1, 2013)

Dishwasher Type	High Temp Efficiency Requirements		Low Temp Efficiency Requirements	
	Idle Energy Rate	Water Consumption	Idle Energy Rate	Water Consumption
Under Counter	≤ 0.50 kW	≤ 0.86 GPR	≤ 0.50 kW	≤ 1.19 GPR
Stationary Single Tank Door	≤ 0.70 kW	≤ 0.89 GPR	≤ 0.60 kW	≤ 1.18 GPR
Pot, Pan, and Utensil	≤ 1.20 kW	≤ 0.58 GPSF	≤ 1.00 kW	≤ 0.58 GPSF
Single Tank Conveyor	≤ 1.50 kW	≤ 0.70 GPR	≤ 1.50 kW	≤ 0.79 GPR
Multiple Tank Conveyor	≤ 2.25 kW	≤ 0.54 GPR	≤ 2.00 kW	≤ 0.54 GPR

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is a new dishwasher that is not ENERGY STAR certified.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be⁶⁷

Dishwasher Type		Equipment Life
Low Temp	Under Counter	10
	Stationary Single Tank Door	15
	Single Tank Conveyor	20
	Multi Tank Conveyor	20
High Temp	Under Counter	10
	Stationary Single Tank Door	15
	Single Tank Conveyor	20
	Multi Tank Conveyor	20
	Pot, Pan, and Utensil	10

DEEMED MEASURE COST

The incremental capital cost for this measure is provided below.⁶⁸

⁶⁷ Lifetime from ENERGY STAR Commercial Kitchen Equipment Savings Calculator which cites reference as “EPA/FSTC research on available models, 2013”

http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

⁶⁸ Measure cost from ENERGY STAR Commercial Kitchen Equipment Savings Calculator which cites reference as “EPA research on available models using AutoQuotes, 2012”

Dishwasher Type		Incremental Cost
Low Temp	Under Counter	\$50
	Stationary Single Tank Door	\$0
	Single Tank Conveyor	\$0
	Multi Tank Conveyor	\$970
High Temp	Under Counter	\$120
	Stationary Single Tank Door	\$770
	Single Tank Conveyor	\$2,050
	Multi Tank Conveyor	\$970
	Pot, Pan, and Utensil	\$1,710

LOADSHAPE

Loadshape C01 - Commercial Electric Cooking

COINCIDENCE FACTOR

Summer Peak Coincidence Factor for measure is provided below for different restaurant types⁶⁹:

Location	CF
Fast Food Limited Menu	0.32
Fast Food Expanded Menu	0.41
Pizza	0.46
Full Service Limited Menu	0.51
Full Service Expanded Menu	0.36
Cafeteria	0.36

Algorithm

CALCULATION OF SAVINGS

ENERGY STAR dishwashers save energy in three categories: building water heating, booster water heating and idle energy. Building water heating and booster water heating could be either electric or natural gas.

ELECTRIC ENERGY SAVINGS

Custom calculation below, otherwise use deemed values found within the tables that follow.

$$\Delta kWh^{70} = \Delta \text{BuildingEnergy} + \Delta \text{BoosterEnergy}^{71} + \Delta \text{IdleEnergy}$$

Where:

$$\begin{aligned} \Delta \text{BuildingEnergy} &= \text{Change in annual electric energy consumption of building water heater} \\ &= [(\text{WaterUse}_{\text{Base}} * \text{RacksWashed} * \text{Days}) * (\Delta T_{\text{in}} * 1.0 * 8.2 \div \text{Eff}_{\text{Heater}} \div 3,413)] - \\ &\quad [(\text{WaterUse}_{\text{ESTAR}} * \text{RacksWashed} * \text{Days}) * (\Delta T_{\text{in}} * 1.0 * 8.2 \div \text{Eff}_{\text{Heater}} \div 3,413)] \\ \Delta \text{BoosterEnergy} &= \text{Annual electric energy consumption of booster water heater} \end{aligned}$$

⁶⁹ Values taken from Minnesota Technical Reference Manual, 'Electric Oven and Range' measure and is based upon "Project on Restaurant Energy Performance-End-Use Monitoring and Analysis", Appendixes I and II, Claar, et. al., May 1985

⁷⁰ Algorithms and assumptions derived from ENERGY STAR Commercial Kitchen Equipment Savings Calculator. http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

⁷¹ Booster water heater energy only applies to high-temperature dishwashers.

$$\Delta \text{IdleEnergy} = \left[\left(\text{WaterUse}_{\text{Base}} * \text{RacksWashed} * \text{Days} \right) * \left(\Delta T_{\text{in}} * 1.0 * 8.2 \div \text{Eff}_{\text{Heater}} \div 3,413 \right) \right] - \left[\left(\text{WaterUse}_{\text{ESTAR}} * \text{RacksWashed} * \text{Days} \right) * \left(\Delta T_{\text{in}} * 1.0 * 8.2 \div \text{Eff}_{\text{Heater}} \div 3,413 \right) \right]$$

$\Delta \text{IdleEnergy}$ = Annual idle electric energy consumption of dishwasher

$$= \left[\text{IdleDraw}_{\text{Base}} * \left(\text{Hours} * \text{Days} - \text{Days} * \text{RacksWashed} * \text{WashTime} \div 60 \right) \right] - \left[\text{IdleDraw}_{\text{ESTAR}} * \left(\text{Hours} * \text{Days} - \text{Days} * \text{RacksWashed} * \text{WashTime} \div 60 \right) \right]$$

Where:

$\text{WaterUse}_{\text{Base}}$	<p>= Water use per rack (gal) of baseline dishwasher</p> <p>= Custom or if unknown, use value from table below as determined by machine type and sanitation method</p>
$\text{WaterUse}_{\text{ESTAR}}$	<p>= Water use per rack (gal) of ENERGY STAR dishwasher</p> <p>= Custom or if unknown, use value from table below as determined by machine type and sanitation method</p>
RacksWashed	<p>= Number of racks washed per day</p> <p>= Custom or if unknown, use value from table below as determined by machine type and sanitation method</p>
Days	<p>= Annual days of dishwasher operation</p> <p>= Custom or if unknown, use 365.25 days per year</p>
ΔT_{in}	<p>= Inlet water temperature increase (°F)</p> <p>= Custom or if unknown, use 70 °F for building water heaters and 40 °F for booster water heaters</p>
1.0	= Specific heat of water (Btu/lb/°F)
8.2	= Density of water (lb/gal)
$\text{Eff}_{\text{Heater}}$	<p>= Efficiency of water heater</p> <p>= Custom or if unknown, use 98% for electric building and booster water heaters</p>
3,413	= kWh to Btu conversion factor
$\text{IdleDraw}_{\text{Base}}$	<p>= Idle power draw (kW) of baseline dishwasher</p> <p>= Custom or if unknown, use value from table below as determined by machine type and sanitation method</p>
$\text{IdleDraw}_{\text{ESTAR}}$	<p>= Idle power draw (kW) of ENERGY STAR dishwasher</p> <p>= Custom or if unknown, use value from table below as determined by machine type and sanitation method</p>
Hours	<p>= Average daily hours of dishwasher operation</p> <p>= Custom or if unknown, use 18 hours per day</p>
WashTime	<p>= Typical wash time (min)</p> <p>= Custom or if unknown, use value from table below as determined by machine type and sanitation method</p>
60	= Minutes to hours conversion factor

EXAMPLE

For example, an ENERGY STAR high-temperature, under counter dishwasher with electric building and electric booster water heating with defaults from the calculation above and the table below would save:

$$\Delta kWh = \Delta BuildingEnergy + \Delta BoosterEnergy + \Delta IdleEnergy$$

Where:

$$\begin{aligned} \Delta BuildingEnergy &= [(1.09 * 75 * 365.25) * (70 * 1.0 * 8.2 \div 0.98 \div 3,413)] - [(0.86 * 75 * 365.25) * (70 * 1.0 * 8.2 \div 0.98 \div 3,413)] \\ &= 1,081 \text{ kWh} \\ \Delta BoosterEnergy &= [(1.09 * 75 * 365.25) * (40 * 1.0 * 8.2 \div 0.98 \div 3,413)] - [(0.86 * 75 * 365.25) * (40 * 1.0 * 8.2 \div 0.98 \div 3,413)] \\ &= 618 \text{ kWh} \\ \Delta IdleEnergy &= [0.76 * (18 * 365.25 - 365.25 * 75 * 2.0 \div 60)] - \\ &\quad [0.50 * (18 * 365.25 - 365.25 * 75 * 2.0 \div 60)] \\ &= 1,472 \text{ Wh} \\ \Delta kWh &= 1,081 + 618 + 1,472 \\ &= 3,171 \text{ kWh} \end{aligned}$$

Default values for WaterUse, RacksWashed, kW_{Idle}, and WashTime are presented in the table below.

	RacksWashed	WashTime	WaterUse		IdleDraw	
Low Temperature	All Dishwashers	All Dishwashers	Conventional	ENERGY STAR	Conventional	ENERGY STAR
Under Counter	75	2.0	1.73	1.19	0.50	0.50
Stationary Single Tank Door	280	1.5	2.10	1.18	0.60	0.60
Single Tank Conveyor	400	0.3	1.31	0.79	1.60	1.50
Multi Tank Conveyor	600	0.3	1.04	0.54	2.00	2.00
High Temperature	All Dishwashers	All Dishwashers	Conventional	ENERGY STAR	Conventional	ENERGY STAR
Under Counter	75	2.0	1.09	0.86	0.76	0.50
Stationary Single Tank Door	280	1.0	1.29	0.89	0.87	0.70
Single Tank Conveyor	400	0.3	0.87	0.70	1.93	1.50
Multi Tank Conveyor	600	0.2	0.97	0.54	2.59	2.25
Pot, Pan, and Utensil	280	3.0 3.0	0.70	0.58	1.20	1.20

Savings for all water heating combinations are presented in the tables below (calculated without rounding variables as provided above).

Electric building and electric booster water heating

Dishwasher type		kWh _{Base}	kWh _{ESTAR}	ΔkWh
Low Temp	Under Counter	10,972	8,431	2,541
	Stationary Single Tank Door	39,306	23,142	16,164
	Single Tank Conveyor	42,230	28,594	13,636
	Multi Tank Conveyor	50,112	31,288	18,824
	Under Counter	12,363	9,191	3,173

Dishwasher type		kWh _{Base}	kWh _{ESTAR}	ΔkWh
High Temp	Stationary Single Tank Door	39,852	27,981	11,871
	Single Tank Conveyor	45,593	36,375	9,218
	Multi Tank Conveyor	72,523	45,096	27,426
	Pot, Pan, and Utensil	21,079	17,766	3,313

Electric building and natural gas booster water heating

Dishwasher type		kWh _{Base}	kWh _{ESTAR}	ΔkWh
Low Temp	Under Counter	10,972	8,431	2,541
	Stationary Single Tank Door	39,306	23,142	16,164
	Single Tank Conveyor	42,230	28,594	13,636
	Multi Tank Conveyor	50,112	31,288	18,824
High Temp	Under Counter	9,432	6,878	2,554
	Stationary Single Tank Door	26,901	19,046	7,856
	Single Tank Conveyor	33,115	26,335	6,780
	Multi Tank Conveyor	51,655	33,479	18,176
	Pot, Pan, and Utensil	14,052	11,943	2,108

Natural gas building and electric booster water heating

Dishwasher type		kWh _{Base}	kWh _{ESTAR}	ΔkWh
Low Temp	Under Counter	2,831	2,831	0
	Stationary Single Tank Door	2,411	2,411	0
	Single Tank Conveyor	9,350	8,766	584
	Multi Tank Conveyor	10,958	10,958	0
High Temp	Under Counter	7,234	5,143	2,090
	Stationary Single Tank Door	17,188	12,344	4,844
	Single Tank Conveyor	23,757	18,806	4,951
	Multi Tank Conveyor	36,004	24,766	11,238
	Pot, Pan, and Utensil	8,781	7,576	1,205

Natural gas building and natural gas booster water heating

Dishwasher type		kWh _{Base}	kWh _{ESTAR}	ΔkWh
Low Temp	Under Counter	2,831	2,831	0
	Stationary Single Tank Door	2,411	2,411	0
	Single Tank Conveyor	9,350	8,766	584
	Multi Tank Conveyor	10,958	10,958	0
High Temp	Under Counter	4,303	2,831	1,472
	Stationary Single Tank Door	4,237	3,409	828
	Single Tank Conveyor	11,279	8,766	2,513
	Multi Tank Conveyor	15,136	13,149	1,987
	Pot, Pan, and Utensil	1,753	1,753	0

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh / \text{AnnualHours} * CF$$

Where:

$$\text{AnnualHours} = \text{Hours} * \text{Days}$$

= Custom or if unknown assume (18 * 365.25 =) 6575 annual hours

CF

= Summer Peak Coincidence Factor

= dependent on restaurant type⁷²:

Location	CF
Fast Food Limited Menu	0.32
Fast Food Expanded Menu	0.41
Pizza	0.46
Full Service Limited Menu	0.51
Full Service Expanded Menu	0.36
Cafeteria	0.36

Example:

A low temperature undercounter dishwasher in a Full Service Limited Menu restaurant with electric building and booster water heaters would save:

$$\begin{aligned}\Delta kW &= \Delta kWh / \text{AnnualHours} * CF \\ &= 2541 / 6575 * 0.51 \\ &= 0.197 \text{ kW}\end{aligned}$$

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms}^{73} = \Delta \text{BuildingEnergy} + \Delta \text{BoosterEnergy}$$

Where:

$$\begin{aligned}\Delta \text{BuildingEnergy} &= \text{Change in annual natural gas consumption of building water heater} \\ &= [(WaterUse_{Base} * RacksWashed * Days) * (\Delta T_{in} * 1.0 * 8.2 \div Eff_{Heater} \div 100,000)] - \\ &\quad [(WaterUse_{ESTAR} * RacksWashed * Days) * (\Delta T_{in} * 1.0 * 8.2 \div Eff_{Heater} \div 100,000)] \\ \Delta \text{BoosterEnergy} &= \text{Change in annual natural gas consumption of booster water heater} \\ &= [(WaterUse_{Base} * RacksWashed * Days) * (\Delta T_{in} * 1.0 * 8.2 \div Eff_{Heater} \div 100,000)] - \\ &\quad [(WaterUse_{ESTAR} * RacksWashed * Days) * (\Delta T_{in} * 1.0 * 8.2 \div Eff_{Heater} \div 100,000)]\end{aligned}$$

Where:

$$\begin{aligned}WaterUse_{Base} &= \text{Water use per rack (gal) of baseline dishwasher} \\ &= \text{Custom or if unknown, use value from table within the electric energy savings} \\ &\quad \text{characterization as determined by machine type and sanitation method} \\ WaterUse_{ESTAR} &= \text{Water use per rack (gal) of ENERGY STAR dishwasher} \\ &= \text{Custom or if unknown, use value from table within the electric energy savings} \\ &\quad \text{characterization as determined by machine type and sanitation method} \\ RacksWashed &= \text{Number of racks washed per day}\end{aligned}$$

⁷² Values taken from Minnesota Technical Reference Manual, 'Electric Oven and Range' measure and is based upon "Project on Restaurant Energy Performance-End-Use Monitoring and Analysis", Appendixes I and II, Claar, et. al., May 1985

⁷³ Algorithms and assumptions derived from ENERGY STAR Commercial Kitchen Equipment Savings Calculator

	= Custom or if unknown, use value from table within the electric energy savings characterization as determined by machine type and sanitation method
Days	= Annual days of dishwasher operation
	= Custom or if unknown, use 365 days per year
ΔT_{in}	= Inlet water temperature increase (°F)
	= Custom or if unknown, use 70 °F for building water heaters and 40 °F for booster water heaters
1.0	= Specific heat of water (Btu/lb/°F)
8.2	= Density of water (lb/gal)
Eff_{Heater}	= Efficiency of water heater
	= Custom or 80% for gas building and booster water heaters
100,000	= Therms to Btu conversion factor

EXAMPLE

For example, an ENERGY STAR high-temperature, under counter dishwasher with gas building and gas booster water heating with defaults from the calculation above and the table within the electric energy savings characterization would save:

$$\Delta \text{Therms} = \Delta \text{BuildingEnergy} + \Delta \text{BoosterEnergy}$$

Where:

$$\begin{aligned} \Delta \text{BuildingEnergy} &= [(1.09 * 75 * 365.25) * (70 * 1.0 * 8.2 \div 0.80 \div 100,000)] - [(0.86 * 75 * 365.25) * (70 * 1.0 * 8.2 \div 0.80 \div 100,000)] \\ &= 45 \text{ therms} \\ \Delta \text{BoosterEnergy} &= [(1.09 * 75 * 365.25) * (40 * 1.0 * 8.2 \div 0.80 \div 100,000)] - [(0.86 * 75 * 365.25) * (40 * 1.0 * 8.2 \div 0.80 \div 100,000)] \\ &= 26 \text{ therms} \\ \Delta \text{Therms} &= 45 + 26 \\ &= 71 \text{ therms} \end{aligned}$$

Savings for all water heating combinations are presented in the tables below.

Electric building and natural gas booster water heating

Dishwasher type		Therms _{Base}	Therms _{SESTAR}	Δ Therms
Low Temp	Under Counter	NA	NA	NA
	Stationary Single Tank Door	NA	NA	NA
	Single Tank Conveyor	NA	NA	NA
	Multi Tank Conveyor	NA	NA	NA
High Temp	Under Counter	123	97	26
	Stationary Single Tank Door	541	374	168
	Single Tank Conveyor	522	420	102
	Stationary Single Tank Door	872	486	387
	Pot, Pan, and Utensil	294	243	50

Natural gas building and natural gas booster water heating

	Dishwasher type	Therms _{Base}	Therms _{ESTAR}	ΔTherms
Low Temp	Under Counter	340	234	106
	Stationary Single Tank Door	1,543	867	676
	Single Tank Conveyor	1,375	829	546
	Multi Tank Conveyor	1,637	850	787
High Temp	Under Counter	337	266	71
	Stationary Single Tank Door	1,489	1,027	462
	Single Tank Conveyor	1,435	1,154	280
	Multi Tank Conveyor	2,399	1,336	1,064
	Pot, Pan, and Utensil	808	669	139

Natural gas building and electric booster water heating

	Dishwasher type	Therms _{Base}	Therms _{ESTAR}	ΔTherms
Low Temp	Under Counter	340	234	106
	Stationary Single Tank Door	1,543	867	676
	Single Tank Conveyor	1,375	829	546
	Multi Tank Conveyor	1,637	850	787
High Temp	Under Counter	214	169	45
	Stationary Single Tank Door	948	654	294
	Single Tank Conveyor	913	735	178
	Multi Tank Conveyor	1,527	850	677
	Pot, Pan, and Utensil	514	426	88

WATER IMPACT DESCRIPTIONS AND CALCULATION

$$\Delta \text{Water} = (\text{WaterUse}_{\text{Base}} * \text{RacksWashed} * \text{Days}) - (\text{WaterUse}_{\text{ESTAR}} * \text{RacksWashed} * \text{Days})$$

Where:

- WaterUse_{Base} = Water use per rack (gal) of baseline dishwasher
= Custom or if unknown, use value from table within the electric energy savings characterization as determined by machine type and sanitation method
- WaterUse_{ESTAR} = Water use per rack (gal) of ENERGY STAR dishwasher
= Custom or if unknown, use value from table within the electric energy savings characterization as determined by machine type and sanitation method
- RacksWashed = Number of racks washed per day
= Custom or if unknown, use value from table within the electric energy savings characterization as determined by machine type and sanitation method
- Days = Annual days of dishwasher operation
= Custom or if unknown, use 365 days per year

EXAMPLE

For example, an ENERGY STAR low-temperature, under counter dishwasher with defaults from the calculation above and the table within the electric energy savings characterization would save:

$$\Delta \text{Water} = (\text{WaterUse}_{\text{Base}} * \text{RacksWashed} * \text{Days}) - (\text{WaterUse}_{\text{ESTAR}} * \text{RacksWashed} * \text{Days})$$

$$\begin{aligned} \Delta \text{Water} &= (1.73 * 75 * 365.25) - (1.19 * 75 * 365.25) \\ &= 14,793 \text{ gallons} \end{aligned}$$

Savings for all dishwasher types are presented in the table below.

	Annual Water Consumption (gallons)		
	Baseline	ENERGY STAR	Savings
Low Temperature			
Under Counter	47,391	32,599	14,793
Stationary Single Tank Door	214,767	120,679	94,088
Single Tank Conveyor	191,391	115,419	75,972
Multi Tank Conveyor	227,916	118,341	109,575
High Temperature			
Under Counter	29,859	23,559	6,301
Stationary Single Tank Door	131,928	91,020	40,908
Single Tank Conveyor	127,107	102,270	24,837
Multi Tank Conveyor	212,576	118,341	94,235
Pot, Pan, and Utensil	71,589	59,317	12,272

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-FSE-ESDW-V03-180101

REVIEW DEADLINE: 1/1/2023

4.2.7 ENERGY STAR Fryer

DESCRIPTION

This measure applies to natural gas fired ENERGY STAR fryer installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a natural gas fryer with a heavy load cooking efficiency $\geq 50\%$ utilizing ASTM standard F1361 or F2144.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is a natural gas fryer that is not ENERGY STAR certified at end of life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years.⁷⁴

DEEMED MEASURE COST

The incremental capital cost for this measure is \$1200.⁷⁵

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS⁷⁶

Custom calculation below, otherwise use deemed value of 505 Therms.

$$\Delta \text{Therms} = (\Delta \text{DailyIdle Energy} + \Delta \text{DailyPreheat Energy} + \Delta \text{DailyCooking Energy}) * \text{Days} / 100000$$

⁷⁴Lifetime from ENERGY STAR commercial griddle which cites reference as “FSTC research on available models, 2009”
http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

⁷⁵Measure cost from ENERGY STAR which cites reference as “EPA research on available models using AutoQuotes, 2010”
http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

⁷⁶ Algorithms and assumptions derived from ENERGY STAR fryer Commercial Kitchen Equipment Savings Calculator.
http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

Where:

$$\Delta \text{DailyIdleEnergy} = (\text{IdleBase} * \text{IdleBaseTime}) - (\text{IdleENERGYSTAR} * \text{IdleENERGYSTARTime})$$

$$\Delta \text{DailyPreheatEnergy} = (\text{PreHeatNumberBase} * \text{PreheatTimeBase} / 60 * \text{PreheatRateBase}) - (\text{PreheatNumberENERGYSTAR} * \text{PreheatTimeENERGYSTAR} / 60 * \text{PreheatRateENERGYSTAR})$$

$$\Delta \text{DailyCookingEnergy} = (\text{LB} * \text{EFOOD} / \text{EffBase}) - (\text{LB} * \text{EFOOD} / \text{EffENERGYSTAR})$$

Where:

HOURSday	= Average Daily Operation = custom or if unknown, use 16 hours
Days	= Annual days of operation = custom or if unknown, use 365.25 days a year
LB	= Food cooked per day = custom or if unknown, use 150 pounds
EffENERGYSTAR	= Cooking Efficiency ENERGY STAR = custom or if unknown, use 50%
EffBase	= Cooking Efficiency Baseline = custom or if unknown, use 35%
PCENERGYSTAR	= Production Capacity ENERGY STAR = custom or if unknown, use 65 pounds/hr
PCBase	= Production Capacity base = custom or if unknown, use 60 pounds/hr
PreheatNumberENERGYSTAR	= Number of preheats per day = custom or if unknown, use 1
PreheatNumberBase	= Number of preheats per day = custom or if unknown, use 1
PreheatTimeENERGYSTAR	= preheat length = custom or if unknown, use 15 minutes
PreheatTimeBase	= preheat length = custom or if unknown, use 15 minutes
PreheatRateENERGYSTAR	= preheat energy rate high efficiency = custom or if unknown, use 62000 btu/h
PreheatRateBase	= preheat energy rate baseline = custom or if unknown, use 64000 btu/h
IdleENERGYSTAR	= Idle energy rate = custom or if unknown, use 9000 btu/h
IdleBase	= Idle energy rate = custom or if unknown, use 14000 btu/h

IdleENERGYSTARTTime	= ENERGY STAR Idle Time = $\text{HOURSday-LB/PCENERGYSTAR -PreHeatTimeENERGYSTAR/60}$ =Custom or if unknown, use = $16 - 150/65-15/60$ =13.44 hours
IdleBaseTime	= BASE Idle Time = $\text{HOURSday-LB/PCbase -PreHeatTimeBase/60}$ =Custom or if unknown, use = $16 - 150/60-15/60$ =13.25 hours
EFOOD	= ASTM energy to food = 570 btu/pound

EXAMPLE

For example, an ENERGY STAR fryer with a tested heavy load cooking energy efficiency of 50% and an idle energy rate of 120,981 btu and an Idle Energy Consumption Rate 9000 btu would save.

$$\Delta \text{Therms} = (\Delta \text{Idle Energy} + \Delta \text{Preheat Energy} + \Delta \text{Cooking Energy}) * \text{Days} / 100000$$

Where:

$\Delta \text{DailyIdleEnergy}$	$= (18550 * 13.25) - (120981 * 13.44)$ = 64519 btu
$\Delta \text{DailyPreheatEnergy}$	$= (1 * 15 / 60 * 64000) - (1 * 15 / 60 * 62000)$ = 500 btu
$\Delta \text{DailyCookingEnergy}$	$= (150 * 570 / .35) - (150 * 570 / .5)$ = 73286 btu
ΔTherms	$= (64519 + 500 + 73286) * 365.25 / 100000$ = 508 therms

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-FSE-ESFR-V01-120601

REVIEW DEADLINE: 1/1/2019

4.2.8 ENERGY STAR Griddle

DESCRIPTION

This measure applies to electric and natural gas fired high efficiency griddle installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be an ENERGY STAR natural gas or electric griddle with a tested heavy load cooking energy efficiency of 70 percent (electric) 38 percent (gas) or greater and an idle energy rate of 2,650 Btu/hr per square foot of cooking surface or less, utilizing ASTM F1275. The griddle must have an Idle Energy Consumption Rate < 2,600 Btu/hr per square foot of cooking surface.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an existing natural gas or electric griddle that's not ENERGY STAR certified and is at end of use.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years⁷⁷

DEEMED MEASURE COST

The incremental capital cost for this measure is \$0 for and electric griddle and \$60 for a gas griddle.⁷⁸

LOADSHAPE

Loadshape C01 - Commercial Electric Cooking

COINCIDENCE FACTOR

Summer Peak Coincidence Factor for measure is provided below for different building type⁷⁹:

Location	CF
Fast Food Limited Menu	0.32
Fast Food Expanded Menu	0.41
Pizza	0.46
Full Service Limited Menu	0.51
Full Service Expanded Menu	0.36
Cafeteria	0.36

⁷⁷ Lifetime from ENERGY STAR commercial griddle which cites reference as "FSTC research on available models, 2009"
http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

⁷⁸ Measure cost from ENERGY STAR which cites reference as "EPA research on available models using AutoQuotes, 2010"
http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

⁷⁹Values taken from Minnesota Technical Reference Manual, 'Electric Oven and Range' measure and is based upon "Project on Restaurant Energy Performance-End-Use Monitoring and Analysis", Appendixes I and II, Claar, et. al., May 1985

Algorithm

CALCULATION OF SAVINGS⁸⁰**ELECTRIC ENERGY SAVINGS**

$$\Delta kWh = (\Delta \text{Idle Energy} + \Delta \text{Preheat Energy} + \Delta \text{Cooking Energy}) * \text{Days} / 1000$$

Where:

$$\begin{aligned} \Delta \text{DailyIdleEnergy} &= [(\text{IdleBase} * \text{Width} * \text{Depth} * (\text{HOURSday} - (\text{LB}/(\text{PCBase} * \text{Width} * \text{Depth}))) - (\text{PreheatNumberBase} * \text{PreheatTimeBase}/60)] - [(\text{IdleENERGYSTAR} * \text{Width} * \text{Depth} * (\text{HOURSday} - (\text{LB}/(\text{PCENERGYSTAR} * \text{Width} * \text{Depth}))) - (\text{PreheatNumberENERGYSTAR} * \text{PreheatTimeENERGYSTAR}/60)] \\ \Delta \text{DailyPreheatEnergy} &= (\text{PreHeatNumberBase} * \text{PreheatTimeBase} / 60 * \text{PreheatRateBase} * \text{Width} * \text{Depth}) - (\text{PreheatNumberENERGYSTAR} * \text{PreheatTimeENERGYSTAR}/60 * \text{PreheatRateENERGYSTAR} * \text{Width} * \text{Depth}) \\ \Delta \text{DailyCookingEnergy} &= (\text{LB} * \text{EFOOD} / \text{EffBase}) - (\text{LB} * \text{EFOOD} / \text{EffENERGYSTAR}) \end{aligned}$$

Where:

HOURSday	= Average Daily Operation = custom or if unknown, use 12 hours
Days	= Annual days of operation = custom or if unknown, use 365.25 days a year
LB	= Food cooked per day = custom or if unknown, use 100 pounds
Width	= Griddle Width = custom or if unknown, use 3 feet
Depth	= Griddle Depth = custom or if unknown, use 2 feet
EffENERGYSTAR	= Cooking Efficiency ENERGY STAR = custom or if unknown, use 70%
EffBase	= Cooking Efficiency Baseline = custom or if unknown, use 65%
PCENERGYSTAR	= Production Capacity ENERGY STAR = custom or if unknown, use 40/6 = 6.67 pounds/hr/sq ft
PCBase	= Production Capacity base = custom or if unknown, use 35/6 = 5.83 pounds/hr/sq ft
PreheatNumberENERGYSTAR	= Number of preheats per day = custom or if unknown, use 1

⁸⁰ Algorithms and assumptions derived from ENERGY STAR Griddle Commercial Kitchen Equipment Savings Calculator. http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

PreheatNumberBase	= Number of preheats per day = custom or if unknown, use 1
PreheatTimeENERGYSTAR	= preheat length = custom or if unknown, use 15 minutes
PreheatTimeBase	= preheat length = custom or if unknown, use 15 minutes
PreheatRateENERGYSTAR	= preheat energy rate high efficiency = custom or if unknown, use $8000/6 = 1333$ W/sq ft
PreheatRateBase	= preheat energy rate baseline = custom or if unknown, use $16000/6 = 2667$ W/sq ft
IdleENERGYSTAR	= Idle energy rate = custom or if unknown, use 320 W/sq ft
IdleBase	= Idle energy rate = custom or if unknown, use 400 W/sq ft
EFOOD	= ASTM energy to food = 139 w/pound

For example, an ENERGY STAR griddle with a tested heavy load cooking energy efficiency of 70 percent or greater and an idle energy rate of 320 W per square foot of cooking surface or less would save.

$$\begin{aligned}
 \Delta \text{DailyIdleEnergy} &= [400 * 3 * 2 * (12 - (100/(35/6 * 3 * 2)) - (1 * 15/60))] - [320 * 3 * 2 * (12 - (100/(40/6 * 3 * 2)) - (1 * 15/60))] \\
 &= 3583 \text{ W} \\
 \Delta \text{DailyPreheatEnergy} &= (1 * 15 / 60 * 16000/6 * 3 * 2) - (1 * 15/60 * 8000/6 * 3 * 2) \\
 &= 2000 \text{ W} \\
 \Delta \text{DailyCookingEnergy} &= (100 * 139 / 0.65) - (100 * 139 / 0.70) \\
 &= 1527 \text{ W} \\
 \Delta \text{kWh} &= (2000 + 1527 + 3583) * 365.25 / 1000 \\
 &= 2597 \text{ kWh}
 \end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\text{kW} = \Delta \text{kWh/Hours} * \text{CF}$$

For example, an ENERGY STAR griddle in a cafeteria with a tested heavy load cooking energy efficiency of 70 percent or greater and an idle energy rate of 320 W per square foot of cooking surface or less would save

$$\begin{aligned}
 &= 2597 \text{ kWh} / 4308 * 0.36 \\
 &= 0.22 \text{ kW}
 \end{aligned}$$

NATURAL GAS ENERGY SAVINGS

Custom calculation below, otherwise use deemed value of 149 therms.

$$\Delta \text{Therms} = (\Delta \text{Idle Energy} + \Delta \text{Preheat Energy} + \Delta \text{Cooking Energy}) * \text{Days} / 100000$$

Where:

$$\Delta \text{DailyIdleEnergy} = [(\text{IdleBase} * \text{Width} * \text{Depth} * (\text{HOURSday} - \text{LB}/(\text{PCBase} * \text{Width} * \text{Depth})) - (\text{PreheatNumberBase} * \text{PreheatTimeBase}/60)] - [(\text{IdleENERGYSTAR} * \text{Width} * \text{Depth} * (\text{HOURSday} - (\text{LB}/(\text{PCENERGYSTAR} * \text{Width} * \text{Depth})) - (\text{PreheatNumberENERGYSTAR} * \text{PreheatTimeENERGYSTAR}/60)]$$

$$\Delta \text{DailyPreheatEnergy} = (\text{PreHeatNumberBase} * \text{PreheatTimeBase} / 60 * \text{PreheatRateBase} * \text{Width} * \text{Depth}) - (\text{PreheatNumberENERGYSTAR} * \text{PreheatTimeENERGYSTAR}/60 * \text{PreheatRateENERGYSTAR} * \text{Width} * \text{Depth})$$

$$\Delta \text{DailyCookingEnergy} = (\text{LB} * \text{EFOOD}/\text{EffBase}) - (\text{LB} * \text{EFOOD}/\text{EffENERGYSTAR})$$

Where (new variables only):

$$\text{EffENERGYSTAR} = \text{Cooking Efficiency ENERGY STAR}$$

= custom or if unknown, use 38%

$$\text{EffBase} = \text{Cooking Efficiency Baseline}$$

= custom or if unknown, use 32%

$$\text{PCENERGYSTAR} = \text{Production Capacity ENERGY STAR}$$

= custom or if unknown, use $45/6 = 7.5$ pounds/hr/sq ft

$$\text{PCBase} = \text{Production Capacity base}$$

= custom or if unknown, use $25/6 = 4.17$ pounds/hr/sq ft

$$\text{PreheatRateENERGYSTAR} = \text{preheat energy rate high efficiency}$$

= custom or if unknown, use $60000/6 = 10000$ btu/h/sq ft

$$\text{PreheatRateBase} = \text{preheat energy rate baseline}$$

= custom or if unknown, use $84000/6 = 14000$ btu/h/sq ft

$$\text{IdleENERGYSTAR} = \text{Idle energy rate}$$

= custom or if unknown, use $15900/6 = 2650$ btu/h/sq ft

$$\text{IdleBase} = \text{Idle energy rate}$$

= custom or if unknown, use $21000/6 = 3500$ btu/h/sq ft

$$\text{EFOOD} = \text{ASTM energy to food}$$

= 475 btu/pound

For example, an ENERGY STAR griddle with a tested heavy load cooking energy efficiency of 38 percent or greater and an idle energy rate of 2,650 Btu/h per square foot of cooking surface or less and an Idle Energy Consumption Rate < 2,600 Btu/h per square foot of cooking surface would save.

$$\begin{aligned}
 \Delta \text{DailyIdleEnergy} &= [3500 * 3 * 2 * (12 - 100 / (25 / 6 * 3 * 2)) - (1 * 15 / 60)] - [(2650 * 3 * 2 * (12 - 100 / (45 / 6 * 3 * 2)) - (1 * 15 / 60))] \\
 &= 11258 \text{ Btu} \\
 \Delta \text{DailyPreheatEnergy} &= (1 * 15 / 60 * 14,000 * 3 * 2) - (1 * 15 / 60 * 10000 * 3 * 2) \\
 &= 6000 \text{ btu} \\
 \Delta \text{DailyCookingEnergy} &= (100 * 475 / 0.32) - (100 * 475 / 0.38) \\
 &= 23438 \text{ btu} \\
 \Delta \text{Therms} &= (11258 + 6000 + 23438) * 365.25 / 100000 \\
 &= 149 \text{ therms}
 \end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-FSE-ESGR-V02-160601

REVIEW DEADLINE: 1/1/2023

4.2.9 ENERGY STAR Hot Food Holding Cabinets

DESCRIPTION

This measure applies to electric ENERGY STAR hot food holding cabinets (HFHC) installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be an ENERGY STAR certified HFHC.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an electric HFHC that's not ENERGY STAR certified and at end of life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years⁸¹

DEEMED MEASURE COST

The incremental capital cost for this measure is⁸²

HFHC Size	Incremental Cost
Full Size (20 cubic feet)	\$1200
¾ Size (12 cubic feet)	\$1800
½ Size (8 cubic feet)	\$1500

LOADSHAPE

Loadshape C01 - Commercial Electric Cooking

COINCIDENCE FACTOR

Summer Peak Coincidence Factor for measure is provided below for different building type⁸³:

Location	CF
Fast Food Limited Menu	0.32
Fast Food Expanded Menu	0.41
Pizza	0.46
Full Service Limited Menu	0.51
Full Service Expanded Menu	0.36
Cafeteria	0.36

⁸¹ Lifetime from ENERGY STAR HFHC which cites reference as "FSTC research on available models, 2009" http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

⁸² Measure cost from ENERGY STAR which cites reference as "EPA research on available models using AutoQuotes, 2010" http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

⁸³Values taken from Minnesota Technical Reference Manual, 'Electric Oven and Range' measure and is based upon "Project on Restaurant Energy Performance-End-Use Monitoring and Analysis", Appendixes I and II, Claar, et. al., May 1985

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

Custom calculation below, otherwise use deemed values depending on HFHC size⁸⁴

Cabinet Size	Savings (kWh)
Full Size HFHC	9308
¾ Size HFHC	3942
½ Size HFHC	2628

$$\Delta kWh = HFHC_{Baseline} kWh - HFHC_{ENERGYSTAR} kWh$$

Where:

$$HFHC_{Baseline} kWh = Power_{Baseline} * HOURS_{day} * Days / 1000$$

Power_{Baseline} = Custom, otherwise

Cabinet Size	Power (W)
Full Size HFHC	2500
¾ Size HFHC	1200
½ Size HFHC	800

HOURS_{day} = Average Daily Operation

= custom or if unknown, use 15 hours

Days = Annual days of operation

= custom or if unknown, use 365.25 days a year

$$HFHC_{ENERGYSTAR} kWh = Power_{ENERGYSTAR} * HOURS_{day} * Days / 1000$$

Power_{ENERGYSTAR} = Custom, otherwise

Cabinet Size	Power (W)
Full Size HFHC	800
¾ Size HFHC	480
½ Size HFHC	320

HOURS_{day} = Average Daily Operation

= custom or if unknown, use 15 hours

Days = Annual days of operation

= custom or if unknown, use 365.25 days a year

⁸⁴ Algorithms and assumptions derived from ENERGY STAR Commercial Kitchen Equipment Savings Calculator. http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=COG

For example, if a full size HFHC is installed the measure would save:

$$\begin{aligned}\Delta \text{kWh} &= (\text{PowerBaseline} * \text{HOURSday} * \text{Days}) / 1000 - (\text{PowerENERGYSTAR} * \text{HOURSday} * \text{Days}) / 1000 \\ &= (2500 * 15 * 365.25) / 1000 - (800 * 15 * 365.25) / 1000 \\ &= 9,314 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \Delta \text{kWh} / \text{Hours} * \text{CF}$$

Where: Hours = Hoursday * Days

For example, if a full size HFHC is installed in a cafeteria the measure would save:

$$\begin{aligned}&= 9,314 \text{ kWh} / (15 * 365.25) * .36 \\ &= 0.61 \text{ kW}\end{aligned}$$

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-FSE-ESHH-V02-160601

REVIEW DEADLINE: 1/1/2023

4.2.10 ENERGY STAR Ice Maker

DESCRIPTION

This measure relates to the installation of a new ENERGY STAR qualified commercial ice machine. The ENERGY STAR label applied to air-cooled, cube-type machines including ice-making head, self-contained, and remote-condensing units. This measure excludes flake and nugget type ice machines. This measure could relate to the replacing of an existing unit at the end of its useful life, or the installation of a new system in a new or existing building.

This measure was developed to be applicable to the following program types: TOS and NC. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a new commercial ice machine meeting the minimum ENERGY STAR efficiency level standards.

DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline equipment is assumed to be a commercial ice machine meeting federal equipment standards established January 1, 2010.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years⁸⁵.

DEEMED MEASURE COST

The incremental capital cost for this measure is provided below.⁸⁶

Harvest Rate (H)	Incremental Cost
100-200 lb ice machine	\$296
201-300 lb ice machine	\$312
301-400 lb ice machine	\$559
401-500 lb ice machine	\$981
501-1000 lb ice machine	\$1,485
1001-1500 lb ice machine	\$1,821
>1500 lb ice machine	\$2,194

LOADSHAPE

Loadshape C23 - Commercial Refrigeration

COINCIDENCE FACTOR

The Summer Peak Coincidence Factor is assumed to equal 0.937

⁸⁵DEER 2008

⁸⁶These values are from electronic work papers prepared in support of San Diego Gas & Electric's "Application for Approval of Electric and Gas Energy Efficiency Programs and Budgets for Years 2009-2011", SDGE, March 2, 2009. Accessed on 7/7/10 <<http://www.sdge.com/regulatory/documents/ee2009-2011Workpapers/SW-ComB/Food%20Service/Food%20Service%20Electric%20Measure%20Workpapers%2011-08-05.DOC>>.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = [(\text{kWh}_{\text{base}} - \text{kWh}_{\text{ee}}) / 100] * (\text{DC} * \text{H}) * 365.25$$

Where:

kWh_{base} = maximum kWh consumption per 100 pounds of ice for the baseline equipment

= calculated as shown in the table below using the actual Harvest Rate (H) of the efficient equipment.

kWh_{ee} = maximum kWh consumption per 100 pounds of ice for the efficient equipment

= calculated as shown in the table below using the actual Harvest Rate (H) of the efficient equipment.

Ice Machine Type	$\text{kWh}_{\text{base}}^{87}$	$\text{kWh}_{\text{ee}}^{88}$
Ice Making Head (H < 450)	10.26 - 0.0086*H	9.23 - 0.0077*H
Ice Making Head (H ≥ 450)	6.89 – 0.0011*H	6.20 - 0.0010*H
Remote Condensing Unit, without remote compressor (H < 1000)	8.85 – 0.0038*H	8.05 - 0.0035*H
Remote Condensing Unit, without remote compressor (H ≥ 1000)	5.1	4.64
Remote Condensing Unit, with remote compressor (H < 934)	8.85 – 0.0038*H	8.05 - 0.0035*H
Remote Condensing Unit, with remote compressor (H ≥ 934)	5.3	4.82
Self Contained Unit (H < 175)	18 - 0.0469*H	16.7 - 0.0436*H
Self Contained Unit (H ≥ 175)	9.8	9.11

100 = conversion factor to convert kWh_{base} and kWh_{ee} into maximum kWh consumption per pound of ice.

DC = Duty Cycle of the ice machine

= 0.57⁸⁹

H = Harvest Rate (pounds of ice made per day)

= Actual installed

⁸⁷Baseline reflects federal standards which apply to units manufactured on or after January 1, 2010
<<http://ecfr.gpoaccess.gov/cgi/t/text/text-idx?c=ecfr&rgn=div6&view=text&node=10:3.0.1.4.17.8&idno=10>>.

⁸⁸ENERGY STAR Program Requirements for Commercial Ice Machines, Partner Commitments, U.S. Environmental Protection Agency, Accessed on 7/7/10

<http://www.energystar.gov/ia/partners/product_specs/program_reqs/ice_machine_prog_req.pdf>

⁸⁹Duty cycle varies considerably from one installation to the next. TRM assumptions from Vermont, Wisconsin, and New York vary from 40 to 57%, whereas the ENERGY STAR Commercial Ice Machine Savings Calculator <http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_Ice_Machines.xls> assumes a value of 75%. A field study of eight ice machines in California indicated an average duty cycle of 57% (“A Field Study to Characterize Water and Energy Use of Commercial Ice-Cube Machines and Quantify Saving Potential”, Food Service Technology Center, December 2007). Furthermore, a report prepared by ACEEE assumed a value of 40% (Nadel, S., Packaged Commercial Refrigeration Equipment: A Briefing Report for Program Planners and Implementers, ACEEE, December 2002). The value of 57% was utilized since it appears to represent a high quality data source.

365.35 = days per year

For example an ice machine with an ice making head producing 450 pounds of ice would save

$$\begin{aligned}\Delta \text{kWh} &= [(6.4 - 5.8) / 100] * (0.57 * 450) * 365.25 \\ &= 562 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \Delta \text{kWh} / (\text{HOURS} * \text{DC}) * \text{CF}$$

Where:

HOURS = annual operating hours

$$= 8766^{90}$$

$$\text{CF} = 0.937$$

For example an ice machine with an ice making head producing 450 pounds of ice would save

$$\begin{aligned}\Delta \text{kW} &= 562 / (8766 * 0.57) * .937 \\ &= 0.105 \text{ kW}\end{aligned}$$

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

While the ENERGY STAR labeling criteria require that certified commercial ice machines meet certain “maximum potable water use per 100 pounds of ice made” requirements, such requirements are intended to prevent equipment manufacturers from gaining energy efficiency at the cost of water consumptions. A review of the AHRI Certification Directory⁹¹ indicates that approximately 81% of air-cooled, cube-type machines meet the ENERGY STAR potable water use requirement. Therefore, there are no assumed water impacts for this measure.

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-FSE-ESIM-V01-120601

REVIEW DEADLINE: 1/1/2019

⁹⁰Unit is assumed to be connected to power 24 hours per day, 365.25 days per year.

⁹¹AHRI Certification Directory, Accessed on 7/7/10. <<http://www.ahridirectory.org/ahridirectory/pages/home.aspx>>

4.2.11 High Efficiency Pre-Rinse Spray Valve

DESCRIPTION

Pre-rinse valves use a spray of water to remove food waste from dishes prior to cleaning in a dishwasher. More efficient spray valves use less water thereby reducing water consumption, water heating cost, and waste water (sewer) charges. Pre-rinse spray valves include a nozzle, squeeze lever, and dish guard bumper. The primary impacts of this measure are water savings. Reduced hot water consumption saves either natural gas or electricity, depending on the type of energy the hot water heater uses.

This measure was developed to be applicable to the following program types: TOS, RF, and DI. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure, the new or replacement pre-rinse spray nozzle must use less than 1.6 gallons per minute with a cleanability performance of 26 seconds per plate or less.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment will vary based on the delivery method and is defined below:

Time of Sale	Retrofit, Direct Install
The baseline equipment is assumed to be 1.6 gallons per minute. The Energy Policy Act (EPA) of 2005 sets the maximum flow rate for pre-rinse spray valves at 1.6 gallons per minute at 60 pounds per square inch of water pressure when tested in accordance with ASTM F2324-03. This performance standard went into effect January 1, 2006.	The baseline equipment is assumed to be an existing pre-rinse spray valve with a flow rate of 1.9 gallons per minute. ⁹² If existing pre-rinse spray valve flow rate is unknown, then existing pre-rinse spray valve must have been installed prior to 2006. The Energy Policy Act (EPA) of 2005 sets the maximum flow rate for pre-rinse spray valves at 1.6 gallons per minute at 60 pounds per square inch of water pressure when tested in accordance with ASTM F2324-03. This performance standard went into effect January 1, 2006. However, field data shows that not all nozzles in use have been replaced with the newer flow rate nozzle. Products predating this standard can use up to five gallons per minute

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 5 years⁹³

DEEMED MEASURE COST

When available, the actual cost of the measure (including labor where applicable) should be used. If unknown, a default value of \$92.90⁹⁴ may be assumed.

LOADSHAPE

Loadshape C01 - Commercial Electric Cooking

⁹² Verification measurements taken at 195 installations showed average pre and post flowrates of 2.23 and 1.12 gallon per minute, respectively." from IMPACT AND PROCESS EVALUATION FINAL REPORT for CALIFORNIA URBAN WATER CONSERVATION COUNCIL 2004-5 PRE-RINSE SPRAY VALVE INSTALLATION PROGRAM (PHASE 2) (PG&E Program # 1198-04; SoCalGas Program 1200-04) ("CUWCC Report", Feb 2007)

⁹³Reference 2010 Ohio Technical Reference Manual, Act on Energy Business Program Technical Reference Manual Rev05, and Federal Energy Management Program (2004), "How to Buy a Low-Flow Pre-Rinse Spray Valve."

⁹⁴Average of costs recognized by Ameren Missouri (\$85.8) and KCPL (\$100).

COINCIDENCE FACTOR

N/A

Algorithm**CALCULATION OF ENERGY SAVINGS****ELECTRIC ENERGY SAVINGS (NOTE WATER SAVINGS MUST FIRST BE CALCULATED)**

$$\Delta kWH = \Delta Gallons * 8.33 * 1 * (T_{out} - T_{in}) * (1/EFF_Elec) / 3,413 * FLAG$$

Where:

$\Delta Gallons$	= amount of water saved as calculated below
8.33	= specific mass in pounds of one gallon of water (lbm/gal)
1	= Specific heat of water: 1 Btu/lbm/°F
T_{out}	= Water Heater Outlet Water Temperature = custom, otherwise assume $T_{in} + 70^{\circ}F$ temperature rise from T_{in} ⁹⁵
T_{in}	= Inlet Water Temperature = custom, otherwise assume $54.1^{\circ}F$ ⁹⁶
EFF_Elec	= Efficiency of electric water heater supplying hot water to pre-rinse spray valve = custom, otherwise assume 97% ⁹⁷
Flag	= 1 if electric or 0 if gas

EXAMPLE

Time of Sale: For example, a new spray nozzle with 1.06 gal/min flow replacing a nozzle with 1.6 gal/min flow at a large institutional establishments with a cafeteria with 70 degree temperature rise of water used by the pre-rinse spray valve that is heated by electric hot water saves annually :

$$\begin{aligned}\Delta kWH &= 30,326 \times 8.33 \times 1 \times ((70+54.1) - 54.1) \times (1/.97) / 3,413 \times 1 \\ &= 5,341 \text{ kWh}\end{aligned}$$

Retrofit: For example, a new spray nozzle with 1.06 gal/min flow replacing a nozzle with 1.9 gal/min flow at a large institutional establishments with a cafeteria with 70 degree temperature rise of water used by the pre-rinse spray valve that is heated by electric hot water equals:

$$\begin{aligned}\Delta kWH &= 47,175 \times 8.33 \times 1 \times ((70+ 54.1) - 54.1) \times (1/.97) / 3,413 \times 1 \\ &= 8309 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

⁹⁵If unknown, assume a 70 degree temperature rise from T_{in} per Food Service Technology Center calculator assumptions to account for variations in mixing and water heater efficiencies

⁹⁶August 31, 2011 Memo of Savings for Hot Water Savings Measures to Nicor Gas from Navigant states that $54.1^{\circ}F$ was calculated from the weighted average of monthly water mains temperatures reported in the 2010 Building America Benchmark Study for Chicago-Waukegan, Illinois.

⁹⁷This efficiency value is based on IECC 2012/2015 performance requirement for electric resistant water heaters rounded without the slight adjustment allowing for reduction based on size of storage tank.

NATURAL GAS ENERGY SAVINGS

$$\Delta\text{Therms} = \Delta\text{Gallons} * 8.33 * 1 * (\text{Tout} - \text{Tin}) * (1/\text{EFF_Gas}) / 100,000 * (1 - \text{FLAG})$$

Where (new variables only):

$$\begin{aligned}\text{EFF_Gas} &= \text{Efficiency of gas water heater supplying hot water to pre-rinse spray valve} \\ &= \text{custom, otherwise assume } 80\%^{98}\end{aligned}$$

EXAMPLE

Time of Sale: For example, a new spray nozzle with 1.06 gal/min flow replacing a nozzle with 1.6 gal/min flow at a large institutional establishments with a cafeteria with 70 degree temperature of water used by the pre-rinse spray valve that is heated by fossil fuel hot water saves annually:

$$\begin{aligned}\Delta\text{Therms} &= 30,326 * 8.33 * 1 * ((70+54.1) - 54.1) * (1/.80) / 100,000 * (1-0) \\ &= 221 \text{ Therms}\end{aligned}$$

Retrofit: For example, a new spray nozzle with 1.06 gal/min flow replacing a nozzle with 1.9 gal/min flow at a busy large institutional establishments with a cafeteria with 70 degree temperature rise of water used by the pre-rinse spray valve that is heated by fossil fuel hot water saves annually:

$$\begin{aligned}\Delta\text{Therms} &= 47,175 * 8.33 * 1 * ((70+54.1) - 54.1) * (1/.80) / 100,000 * (1-0) \\ &= 344 \text{ Therms}\end{aligned}$$

WATER IMPACT CALCULATION⁹⁹

$$\Delta\text{Gallons} = (\text{FLObase} - \text{FLOeff}) * 60 * \text{HOURSday} * \text{DAYYear}$$

Where:

$$\text{FLObase} = \text{Base case flow in gallons per minute, or custom (Gal/min)}$$

Time of Sale	Retrofit, Direct Install
1.6 gal/min ¹⁰⁰	1.9 gal/min ¹⁰¹

$$\text{FLOeff} = \text{Efficient case flow in gallons per minute or custom (Gal/min)}$$

Time of Sale	Retrofit, Direct Install
1.06 gal/min ¹⁰²	1.06 gal/min ¹⁰³

⁹⁸ IECC 2012/2015, Table C404.2, Minimum Performance of Water-Heating Equipment

⁹⁹In order to calculate energy savings, water savings must first be calculated

¹⁰⁰The baseline equipment is assumed to be 1.6 gallons per minute. The Energy Policy Act (EPAct) of 2005 sets the maximum flow rate for pre-rinse spray valves at 1.6 gallons per minute at 60 pounds per square inch of water pressure when tested in accordance with ASTM F2324-03. This performance standard went into effect January 1, 2006.

www1.eere.energy.gov/femp/pdfs/spec_prerinsesprayvavles.pdf.

¹⁰¹ Verification measurements taken at 195 installations showed average pre and post flowrates of 2.23 and 1.12 gallon per minute, respectively." from IMPACT AND PROCESS EVALUATION FINAL REPORT for CALIFORNIA URBAN WATER CONSERVATION COUNCIL 2004-5 PRE-RINSE SPRAY VALVE INSTALLATION PROGRAM (PHASE 2) (PG&E Program # 1198-04; SoCalGas Program 1200-04) ("CUWCC Report", Feb 2007)

¹⁰²1.6 gallons per minute used to be the high efficiency flow, but more efficient spray valves are available ranging down to 0.64 gallons per minute per Federal Energy Management Program which references the Food Services Technology Center web site with the added note that even more efficient models may be available since publishing the data. The average of the nozzles listed on the FSTC website is 1.06.

¹⁰³1.6 gallons per minute used to be the high efficiency flow, but more efficient spray valves are available ranging down to 0.64 gallons per minute per Federal Energy Management Program which references the Food Services Technology Center web site

60 = Minutes per hour

HOURS_{day} = Hours per day that the pre-rinse spray valve is used at the site, custom, otherwise¹⁰⁴:

Application	Hours/day
Small, quick- service restaurants	1
Medium-sized casual dining restaurants	1.5
Large institutional establishments with cafeteria	3

DAYS_{year} = Days per year pre-rinse spray valve is used at the site, custom, otherwise 312 days/yr based on assumed 6 days/wk x 52 wk/yr = 312 day/yr.

EXAMPLE

Time of Sale: For example, a new spray nozzle with 1.06 gal/min flow replacing a nozzle with 1.6 gal/min flow at a large institutional establishment with a cafeteria equals

$$= (1.6 - 1.06) * 60 * 3 * 312$$

$$= 30,326 \text{ gal/yr}$$

Retrofit: For example, a new spray nozzle with 1.06 gal/min flow replacing a nozzle with 1.9 gal/min flow at a large institutional establishments with a cafeteria equals

$$= (1.9 - 1.06) * 60 * 3 * 312$$

$$= 47,175 \text{ gal/yr}$$

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-FSE-SPRY-V04-180101

REVIEW DEADLINE: 1/1/2023

with the added note that even more efficient models may be available since publishing the data. The average of the nozzles listed on the FSTC website is 1.06.

¹⁰⁴ Hours primarily based on PG& E savings estimates, algorithms, sources (2005), Food Service Pre-Rinse Spray Valves with review of 2010 Ohio Technical Reference Manual and Act on Energy Business Program Technical Resource Manual Rev05.

4.2.12 Infrared Charbroiler

DESCRIPTION

This measure applies to natural gas fired charbroilers that utilize infrared burners installed in a commercial kitchen

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a new natural gas charbroiler with infrared burners.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an existing natural gas charbroiler without infrared burners.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years¹⁰⁵

DEEMED MEASURE COST

The incremental capital cost for this measure is \$2173¹⁰⁶

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

Custom calculation below, otherwise use deemed value of 707 therms based on default values.¹⁰⁷

$$\Delta Therms = \frac{(\Delta PreheatEnergy + \Delta CookingEnergy) * Days}{100,000}$$

$$\Delta PreheatEnergy = (PreheatRate_{Base} - PreheatRate_{EE}) * Preheats * \frac{PreheatTime}{60}$$

¹⁰⁵ Lifecycle determined from Food Service Technology Center Gas Broiler Life-Cycle Cost Calculator and from FSTC Broiler Technology Assessment

¹⁰⁶ See 'Arkansas Deemed TRM Table for GasFoodService.xls' from v3.0 Arkansas Technical Reference Manual.

¹⁰⁷ Assumptions derived from Food Service Technology Center Gas Broiler Life-Cycle Cost Calculator and from FSTC Broiler Technology Assessment, http://www.fishnick.com/equipment/techassessment/4_broilers.pdf

$$\Delta \text{CookingEnergy} = (\text{InputRate}_{\text{Base}} - \text{InputRate}_{\text{EE}}) * (\text{Duty} * \text{Hours})$$

Where:

Days	= Annual days of operation
	= Custom or if unknown, use 312 days per year ¹⁰⁸
100,000	= Btu to therms conversion factor
PreheatRate _{Base}	= Preheat energy rate of baseline charbroiler
	= 64,000 Btu/hr
PreheatRate _{EE}	= Preheat energy rate of infrared charbroiler
	= Custom or if unknown, use 54,000 Btu/hr
Preheats	= Number of preheats per day
	= Custom or if unknown, use 1 preheat per day
PreheatTime	= Length of one preheat
	= Custom or if unknown, use 15 minutes per preheat ¹⁰⁹
60	= Minutes to hours conversion factor
InputRate _{Base}	= Input energy rate of baseline charbroiler
	= 140,000 Btu/hr
InputRate _{EE}	= Input energy rate of infrared charbroiler
	= Custom or if unknown, use 105,000 Btu/hr
Duty	= Duty cycle of charbroiler (%)
	= Custom or if unknown, use 80% ¹¹⁰
Hours	= Average daily hours of operation
	= Custom or if unknown, use 8 hours per day

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-FSE-IRCB-V02-180101

REVIEW DEADLINE: 1/1/2024

¹⁰⁸Typical annual operating time from FSTC Broiler Technology Assessment, Table 4.3

¹⁰⁹Typical preheat time from FSTC Broiler Technology Assessment.

¹¹⁰ Duty cycle from FSTC Broiler Technology Assessment, Table 4.3

4.2.13 Infrared Rotisserie Oven

DESCRIPTION

This measure applies to natural gas fired high efficiency rotisserie ovens utilizing infrared burners and installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a new natural gas rotisserie oven with infrared burners.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an existing natural gas rotisserie oven without infrared burners.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years¹¹¹

DEEMED MEASURE COST

The incremental capital cost for this measure is \$2665¹¹²

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

Custom calculation below based on Food Service Technology Center calculator, otherwise use deemed value of 599 therms, based on default values.

$$\Delta Therms = \frac{(InputRate_{Base} - InputRate_{EE}) * (Duty * Hours)}{100,000}$$

Where:

$InputRate_{Base}$ = Energy input rate of baseline rotisserie oven (Btu/hr)

¹¹¹Lifecycle determined from Food Service Technology Center Gas Oven Life-Cycle Cost Calculator.

¹¹²See 'Arkansas Deemed TRM Table for GasFoodService.xls' from v3.0 Arkansas Technical Reference Manual.

	= Custom of if unknown, use 90,000 Btu/hr ¹¹³
InputRate _{EE}	= Energy input rate of infrared rotisserie oven (Btu/hr)
	= Custom of if unknown, use 50,000 Btu/hr ¹¹⁴
Duty	= Duty cycle of rotisserie oven (%)
	= Custom or if unknown, use 60% ¹¹⁵
Hours	= Typical operating hours of rotisserie oven
	= Custom or if unknown, use 2,496 hours ¹¹⁶
100,000	= Btu to therms conversion factor

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-FSE-IROV-V02-180101

REVIEW DEADLINE: 1/1/2024

¹¹³ Median rated energy input for rotisserie ovens from FSTC Oven Technology Assessment, Table 7.2
http://www.fishnick.com/equipment/techassessment/7_ovens.pdf

¹¹⁴ Infrared energy input rate calculated based on efficient energy input rate of 50,000 Btu/hr, baseline cooking efficiency of 25%, and infrared cooking efficiency of 45%. Efficiencies and rates derived from FSTC Gas Rotisserie Oven Test Reports and FSTC Oven Technology Assessment.

¹¹⁵ Duty cycle from Food Service Technology Center Oven Technical Assessment, Table 7.2

¹¹⁶ Typical operating hours based on oven operating schedule of 8 hours per day, 6 days per week, 52 weeks per year, provided in Food Service Technology Center Oven Technical Assessment, Table 7.2

4.2.14 Infrared Salamander Broiler

DESCRIPTION

This measure applies to natural gas fired high efficiency salamander broilers utilizing infrared burners installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a new natural gas salamander broiler with infrared burners

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an existing natural gas salamander broiler without infrared burners

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years¹¹⁷

DEEMED MEASURE COST

The incremental capital cost for this measure is \$1000¹¹⁸

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

Custom calculation below based on Food Service Technology Center calculator, otherwise use deemed value of 240 therms, based on defaults.

$$\Delta Therms = \frac{(InputRate_{Base} - InputRate_{EE}) * (Duty * Hours)}{100,000}$$

¹¹⁷ Lifecycle determined from Food Service Technology Center Gas Broiler Life-Cycle Cost Calculator and from FSTC Broiler Technology Assessment.

¹¹⁸See 'Arkansas Deemed TRM Table for GasFoodService.xls' from v3.0 Arkansas Technical Reference Manual.

Where:

InputRate _{Base}	= Rated energy input rate of baseline salamander broiler (Btu/hr) = 38,500 Btu/hr ¹¹⁹
InputRate _{EE}	= Rated energy input rate of infrared salamander broiler (Btu/hr) = Custom or if unknown, use 24,750 Btu/hr ¹²⁰
Duty	= Duty cycle of salamander broiler (%) = Custom or if unknown, use 70% ¹²¹
Hours	= Typical operating hours of salamander broiler = Custom or if unknown, use 2,496 hours ¹²²
100,000	= Btu to therms conversion factor

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-FSE-IRBL-V02-180101

REVIEW DEADLINE: 1/1/2024

¹¹⁹ Median rated energy input for salamander broilers from FSTC Broiler Technology Assessment, Table 4.3
http://www.fishnick.com/equipment/techassessment/4_broilers.pdf

¹²⁰ Calculated energy input rate based on baseline energy input rate of 38,500 Btu/hr, baseline cooking efficiency of 22.5%, and infrared cooking efficiency of 35%

¹²¹ Duty cycle from Food Service Technology Center Broiler Technical Assessment, Table 4.3

¹²² Typical operating hours based on broiler operating schedule of 8 hours per day, 6 days per week, 52 weeks per year, provided in Food Service Technology Center Broiler Technical Assessment, Table 4.3

4.2.15 Infrared Upright Broiler

DESCRIPTION

This measure applies to natural gas fired high efficiency upright broilers utilizing infrared burners and installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a new natural gas upright broiler with infrared burners.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an existing natural gas upright broiler without infrared burners.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years¹²³

DEEMED MEASURE COST

The incremental capital cost for this measure is \$4400¹²⁴

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

Custom calculation below based on Food Service Technology Center calculator, otherwise use deemed value of 943 therms based on default values.

$$\Delta Therms = \frac{(InputRate_{Base} - InputRate_{EE}) * (Duty * Hours)}{100,000}$$

¹²³ Lifecycle determined from Food Service Technology Center Gas Broiler Life-Cycle Cost Calculator and from FSTC Broiler Technology Assessment.

¹²⁴See 'Arkansas Deemed TRM Table for GasFoodService.xls' from v3.0 Arkansas Technical Reference Manual.

Where:

InputRate _{Base}	= Rated energy input rate of baseline upright broiler (Btu/hr) = 144,000 Btu/hr ¹²⁵
InputRate _{EE}	= Rated energy input rate of infrared upright broiler (Btu/hr) = Custom or if unknown, use 90,000 Btu/hr ¹²⁶
Duty	= Duty cycle of upright broiler (%) = Custom or if unknown, use 70% ¹²⁷
Hours	= Typical operating hours of upright broiler = Custom or if unknown, use 2,496 hours ¹²⁸
100,000	= Btu to therms conversion factor

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-FSE-IRUB-V02-180101

REVIEW DEADLINE: 1/1/2024

¹²⁵ Baseline energy input rate calculated based on efficient energy input rate of 90,000 Btu/hr, baseline cooking efficiency of 25%, and infrared cooking efficiency of 40%

¹²⁶ Median rated energy input for upright broilers from FSTC Broiler Technology Assessment, Table 4.3
http://www.fishnick.com/equipment/techassessment/4_broilers.pdf

¹²⁷ Duty cycle from Food Service Technology Center Broiler Technical Assessment, Table 4.3

¹²⁸ Typical operating hours based on broiler operating schedule of 8 hours per day, 6 days per week, 52 weeks per year, provided in Food Service Technology Center Broiler Technical Assessment, Table 4.3

4.2.16 Kitchen Demand Ventilation Controls

DESCRIPTION

Installation of commercial kitchen demand ventilation controls that vary the ventilation based on cooking load and/or time of day.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a control system that varies the exhaust rate of kitchen ventilation (exhaust and/or makeup air fans) based on the energy and effluent output from the cooking appliances (i.e., the more heat and smoke/vapors generated, the more ventilation needed). This involves installing a new temperature sensor in the hood exhaust collar and/or an optic sensor on the end of the hood that sense cooking conditions which allows the system to automatically vary the rate of exhaust to what is needed by adjusting the fan speed accordingly.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is kitchen ventilation that has constant speed ventilation motor.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years.¹²⁹

DEEMED MEASURE COST

The incremental capital cost for this measure is¹³⁰

Measure Category	Incremental Cost \$/HP of fan
DVC Control Retrofit	\$1,988
DVC Control New	\$1,000

LOADSHAPE

Loadshape C23 - Commercial Ventilation

COINCIDENCE FACTOR

The measure has deemed peak kW savings therefore a coincidence factor does not apply

Algorithm

CALCULATION OF SAVINGS

Annual energy use was based on monitoring results from five different types of sites, as summarized in PG&E Food Service Equipment work paper.

¹²⁹ PG&E Workpaper: Commercial Kitchen Demand Ventilation Controls-Electric, 2004 - 2005

¹³⁰ Ibid.

ELECTRIC ENERGY SAVINGS

kWh savings are assumed to be 4966 kWh per horsepower of the fan¹³¹

SUMMER COINCIDENT PEAK DEMAND SAVINGS

kW savings are assumed to be 0.68 kW per horsepower of the fan¹³²

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms} = \text{CFM} * \text{HP} * \text{Annual Heating Load} / (\text{Eff}(\text{heat}) * 100,000)$$

Where:

CFM = the average airflow reduction with ventilation controls per hood

= 430 cfm/HP¹³³

HP = actual if known, otherwise assume 7.75 HP¹³⁴

Annual Heating Load = Annual heating energy required to heat fan exhaust make-up air, Btu/cfm dependent on location¹³⁵:

Zone	Annual Heating Load, Btu/cfm
1 (Rockford)	154,000
2-(Chicago)	144,000
3 (Springfield)	132,000
4-(Belleville)	102,000
5-(Marion)	104,000

Eff(heat) = Heating Efficiency

= actual if known, otherwise assume 80%¹³⁶

100,000 = conversion from Btu to Therm

¹³¹ Based on data provided in PGE Workpaper, Commercial Kitchen Demand Ventilation Controls, PGECOFST116, June 1, 2009. See 'Kitchen DCV.xls' for details.

¹³² Based on data provided in PGE Workpaper, Commercial Kitchen Demand Ventilation Controls, PGECOFST116, June 1, 2009. See 'Kitchen DCV.xls' for details.

¹³³ Based on data provided in PGE Workpaper, Commercial Kitchen Demand Ventilation Controls, PGECOFST116, June 1, 2009. See 'Kitchen DCV.xls' for details.

¹³⁴ Average of units in PGE Workpaper, Commercial Kitchen Demand Ventilation Controls, PGECOFST116, June 1, 2009.

¹³⁵ Food Service Technology Center Outside Air Load Calculator, <http://www.fishnick.com/ventilation/oalc/oac.php>, with inputs of one cfm, and hours from Commercial Kitchen Demand Ventilation Controls (Average 17.8 hours a day 4.45 am to 10.30 pm). Savings for Rockford, Chicago, and Springfield were obtained from the calculator; values for Belleville and Marion were obtained by using the average savings per HDD from the other values.

¹³⁶Work Paper WPRRSGNGRO301 CLEAResult"Boiler Tune-Up" which cites Focus on Energy Evaluation Business Programs: Deemed Savings Manual V1.0, PA Consulting, KEMA, March 22, 2010

EXAMPLE

For example, a kitchen hood in Rockford, IL with a 7.75 HP ventilation motor

$$\begin{aligned}\Delta\text{Therms} &= 430 * 7.75 * 154,000 / (0.80 * 100,000) \\ &= 6,415 \text{ Therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-FSE-VENT-V03-160601

REVIEW DEADLINE: 1/1/2021

4.2.17 Pasta Cooker

DESCRIPTION

This measure applies to natural gas fired dedicated pasta cookers as determined by the manufacturer and installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a new natural gas fired paste cooker.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an existing natural gas fired stove where pasta is cooked in a pan.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12¹³⁷.

DEEMED MEASURE COST

The incremental capital cost for this measure is \$2400¹³⁸.

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

The annual natural gas energy savings from this measure is a deemed value equaling 1380 Therms¹³⁹.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

¹³⁷See 'Arkansas Deemed TRM Table for GasFoodService.xls' from v3.0 Arkansas Technical Reference Manual.

¹³⁸Ibid.

¹³⁹ See 'Arkansas Deemed TRM Table for GasFoodService.xls' from v3.0 Arkansas Technical Reference Manual.

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-FSE-PCOK-V02-180101

REVIEW DEADLINE: 1/1/2024

4.2.18 Rack Oven - Double Oven

DESCRIPTION

This measure applies to natural gas fired high efficiency rack oven - double oven installed in a commercial kitchen.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a new natural gas rack oven –double oven with a baking efficiency $\geq 50\%$ utilizing ASTM standard 2093

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is an existing natural gas rack oven – double oven with a baking efficiency $< 50\%$.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years.¹⁴⁰

DEEMED MEASURE COST

The incremental capital cost for this measure is \$3000.¹⁴¹

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

Custom calculation below, otherwise use deemed value of 1930 therms based on default values.¹⁴²

¹⁴⁰ Lifecycle determined from Food Service Technology Center Gas Rack Oven Life-Cycle Cost Calculator and from FSTC Oven Technology Assessment

¹⁴¹See 'Arkansas Deemed TRM Table for GasFoodService.xls' from v3.0 Arkansas Technical Reference Manual.

¹⁴² Assumptions derived from Food Service Technology Center Gas Rack Oven Life-Cycle Cost Calculator, FSTC Oven Technology Assessment (http://www.fishnick.com/equipment/techassessment/7_ovens.pdf), and from FSTC Gas Double Rack Oven Test Reports.

$$\Delta Therms = InputRate * (BakingEfficiency_{EE} - BakingEfficiency_{Base}) * Duty * Hours * \frac{1}{100,000}$$

Where:

InputRate	= Input energy rate of rack oven – double oven = Custom or if unknown, 275,000 Btu/hr ¹⁴³
BakingEfficiency _{EE}	= Baking efficiency of energy efficiency rack oven – double oven = Custom or if unknown, use 55% ¹⁴⁴
BakingEfficiency _{Base}	= Baking efficiency of baseline rack oven – double oven = Custom or if unknown, 30%
Duty	= Duty cycle of double rack oven (%) = Custom or if unknown, use 75% ¹⁴⁵
Hours	= Average daily hours of operation = Custom or if unknown, use 3,744 hours ¹⁴⁶
100,000	= Btu to therms conversion factor

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE CI-FSE-RKOV-VO2-180101

REVIEW DEADLINE: 1/1/2024

¹⁴³ Median rated energy input for rack ovens from FSTC Oven Technology Assessment, http://www.fishnick.com/equipment/techassessment/7_ovens.pdf

¹⁴⁴ Average baking efficiency of double rack oven from FSTC Gas Double Rack Oven Test Reports.

¹⁴⁵ Duty cycle from FSTC Gas Double Rack Oven Test Reports on various double rack ovens.

¹⁴⁶ Typical operating hours based on oven operating schedule of 12 hours per day, 6 days per week, 52 weeks per year, provided in FSTC Gas Double Rack Oven Test Reports on various double rack ovens.

4.2.19 ENERGY STAR Electric Convection Oven

DESCRIPTION

Commercial convection ovens that are ENERGY STAR certified have higher heavy load cooking efficiencies, and lower idle energy rates, making them on average about 20 percent more efficient than standard models. Energy savings estimates are for ovens using full size (18" x 36") sheet pans.

This measure was developed to be applicable to the following program types; TOS.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is assumed to be an ENERGY STAR qualified electric convection oven.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be a standard convection oven with a heavy load efficiency of 65%.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years.¹⁴⁷

DEEMED MEASURE COST

The incremental cost for this measure is assumed to be \$800 for half size units and \$1000 for full size¹⁴⁸

LOADSHAPE

Loadshape C01 - Commercial Electric Cooking

COINCIDENCE FACTOR

Summer Peak Coincidence Factor for measure is provided below for different building type¹⁴⁹:

Location	CF
Fast Food Limited Menu	0.32
Fast Food Expanded Menu	0.41
Pizza	0.46
Full Service Limited Menu	0.51
Full Service Expanded Menu	0.36
Cafeteria	0.36
Unknown	0.40

¹⁴⁷ Food Service Technology Center (FSTC). Default value from life cycle cost calculator.

<http://www.fishnick.com/saveenergy/tools/calculators/eovencalc.php>

¹⁴⁸ Based on data from the Regional Technical Forum for the Northwest Council

(http://rtf.nwcouncil.org/measures/com/ComCookingConvectionOven_v2_0.xlsm) using actual list prices for 23 units from 2012, see "ComCookingConvectionOven_v2_0.xlsm".

¹⁴⁹ Minnesota 2012 Technical Reference Manual, [Electric Food Service_v03.2.xls](#),

<http://mn.gov/commerce/energy/topics/conservation/Design-Resources/Deemed-Savings.jspech>. Unknown is an average of other location types

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = \text{kWh}_{\text{base}} - \text{kWh}_{\text{eff}}$$

$$\text{kWh} = [(\text{LB} * E_{\text{FOOD}}/\text{EFF}) + (\text{IDLE} * (\text{HOURS}_{\text{DAY}} - \text{LB}/\text{PC} - \text{PRE}_{\text{TIME}}/60)) + \text{PRE}_{\text{ENERGY}}] * \text{DAYS}$$

Where:

kWh_{base} = the annual energy usage of the baseline equipment calculated using baseline values

kWh_{eff} = the annual energy usage of the efficient equipment calculated using efficient values

$\text{HOURS}_{\text{DAY}}$ = daily operating hours

= Actual, defaults:

Type of Food Service	$\text{HOURS}_{\text{DAY}}$ ¹⁵⁰
Fast Food, limited menu	4
Fast Food, expanded menu	5
Pizza	8
Full Service, limited menu	8
Full Service, expanded menu	7
Cafeteria	6
Unknown	6 ¹⁵¹
Custom	Varies

DAYS = Days per year of operation

= Actual, default = 365¹⁵²

PRE_{TIME} = Preheat time (min/day), the amount of time it takes a steamer to reach operating temperature when turned on
= 15 min/day¹⁵³

E_{FOOD} = ASTM Energy to Food (kWh/lb); the amount of energy absorbed by the food during cooking, per pound of food
= 0.0732¹⁵⁴

LB = pounds of food cooked per day (lb/day)

¹⁵⁰Minnesota 2012 Technical Reference Manual, Electric Food Service_v03.2.xls,
<http://mn.gov/commerce/energy/topics/conservation/Design-Resources/Deemed-Savings.jspech>

¹⁵¹Unknown is average of other locations

¹⁵² Food Service Technology Center (FSTC). Default value from life cycle cost calculator.
<http://www.fishnick.com/saveenergy/tools/calculators/eovencalc.php>

¹⁵³ Food Service Technology Center (2002). *Commercial Cooking Appliance Technology Assessment*. Prepared by Don Fisher.
Chapter 7: Ovens

¹⁵⁴ American Society for Testing and Materials. Industry standard for Commercial Ovens

= Actual, default = 100¹⁵⁵

EFF = Heavy load cooking energy efficiency (%). See table below.

IDLE = Idle energy rate. See table below.

PC = Production capacity (lbs/hr). See table below.

PRE_{ENERGY} = Preheat energy (kWh/day). See table below.

Performance Metrics: Baseline and Efficient Values

Metric	Baseline Model ¹⁵⁶	Energy Efficient Model ¹⁵⁷
PRE _{ENERGY} (kWh)	1.5	1
IDLE (kW)	2	Actual, default = 1.0
EFF	65%	Actual, default = 74%
PC (lb/hr)	70	Actual, default = 79

EXAMPLE

Using defaults provided above, the savings for a ENERGY STAR Electric Convection Oven in unknown location are:

$$\begin{aligned} \text{kWh}_{\text{base}} &= [(100 * 0.0732 / 0.65) + (2 * (6 - 100 / 70 - 15 / 60)) + 1.5] * 365 \\ &= 7,813 \text{ kWh} \end{aligned}$$

$$\begin{aligned} \text{kWh}_{\text{eff}} &= [(100 * 0.0732 / 0.74) + (1 * (6 - 100 / 79 - 15 / 60)) + 1.0] * 365 \\ &= 5,612 \text{ kWh} \end{aligned}$$

$$\begin{aligned} \Delta \text{kWh} &= \text{kWh}_{\text{base}} - \text{kWh}_{\text{eff}} \\ &= 7,813 - 5,612 \\ &= 2,200 \text{ kWh} \end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = (\Delta \text{kWh} / (\text{HOURS}_{\text{DAY}} * \text{DAYS})) * \text{CF}$$

Where:

$$\Delta \text{kWh} = \text{Annual energy savings (kWh)}$$

¹⁵⁵ Food Service Technology Center (FSTC). Default value from life cycle cost calculator.

<http://www.fishnick.com/saveenergy/tools/calculators/eovencalc.php>

¹⁵⁶ Food Service Technology Center (FSTC). Default values from life cycle cost calculator.

<http://www.fishnick.com/saveenergy/tools/calculators/eovencalc.php>

¹⁵⁷ Average ratings of units on ENERGY STAR qualified list as of 10/2014. Preheat energy is not provided so default is provided based on FSTC life cycle cost calculator.

CF = Summer Peak Coincidence Factor for measure is provided below for different building type¹⁵⁸:

Location	CF
Fast Food Limited Menu	0.32
Fast Food Expanded Menu	0.41
Pizza	0.46
Full Service Limited Menu	0.51
Full Service Expanded Menu	0.36
Cafeteria	0.36
Unknown	0.40

EXAMPLE

Using defaults provided above, the savings for a ENERGY STAR Electric Convection Oven in unknown location are:

$$\begin{aligned}\Delta kW &= (2200 / (6 * 365)) * 0.40 \\ &= 0.40\end{aligned}$$

FOSSIL FUEL IMPACT DESCRIPTIONS AND CALCULATION

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE CI-FSE-ECON-VO1-150601

REVIEW DEADLINE: 1/1/2022

¹⁵⁸Minnesota 2012 Technical Reference Manual, [Electric Food Service v03.2.xls](http://mn.gov/commerce/energy/topics/conservation/Design-Resources/Deemed-Savings.jspech), <http://mn.gov/commerce/energy/topics/conservation/Design-Resources/Deemed-Savings.jspech>. Unknown is an average of other location types

4.3 Hot Water

4.3.1 Storage Water Heater

DESCRIPTION

This measure is for upgrading from minimum code to a high efficiency storage-type water heater. Storage water heaters are used to supply hot water for a variety of commercial building types. Storage capacities vary greatly depending on the application. Large consumers of hot water include (but not limited to) industries, hotels/motels and restaurants.

This measure was developed to be applicable to the following program types: TOS.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The minimum specifications of the high efficiency equipment should be defined by the programs.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is assumed to be a new standard water heater of same type as existing, meeting the Federal Standard for <75,000 Btu/h units and IECC 2015 for all others. If existing type is unknown, assume Gas Storage Water Heater.

Equipment Type	Sub Category	Federal Standard Minimum Efficiency ¹⁵⁹
Gas Storage Water Heaters ≤ 75,000 Btu/h	≤55 gallon tanks	$0.675 - (0.0015 * \text{Rated Storage Volume in Gallons}) \text{ EF}$
	>55 gallon tanks	$0.8012 - (0.00078 * \text{Rated Storage Volume in Gallons}) \text{ EF}$
Gas Storage Water Heaters > 75,000 Btu/h	< 4000 Btu/h/gal	$80\% E_t$ Standby Loss: $(Q/800 + 110vV)$
Electric Water Heaters ≤ 75,000 Btu/h	≤55 gallon tanks	$0.96 - (0.0003 * \text{rated volume in gallons}) \text{ EF}$
	>55 gallon tanks ¹⁶⁰	$2.057 - (0.00113 * \text{rated volume in gallons}) \text{ EF}$
Electric Water Heaters > 75,000 Btu/h	≤12 kW	$0.97 - (0.00132 * \text{rated volume in gallons}) \text{ EF}$
	> 12kW	Standby Loss: $0.30 + 27/V_m (\%/\text{hr})$

V = Rated volume in gallons, V_m = measured volume in gallons.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 Years¹⁶¹

DEEMED MEASURE COST

The full install cost and incremental cost assumptions are provided below. Actual costs should be used where available¹⁶²:

Equipment Type	Category	Install Cost	Incremental Cost
Gas Storage Water Heaters ≤ 75,000 Btu/h, ≤55 Gallons	Baseline	\$616	N/A
	Efficient	\$1,055	\$440

¹⁵⁹ ≤75,000 Btu/h Storage Water Heater and <200,000 Btu/h Tankless Water Heater Federal Standard is DOE Standard 10 CFR 430.32(d). All other standards are from 10 CFR 431.110.

¹⁶⁰ It is assumed that tanks <75,000Btu/h and >55 gallons will not be eligible measures due to the high baseline.

¹⁶¹ DEER 08, EUL_Summary_10-1-08.xls.

¹⁶² Cost information is based upon data from “2010-2012 WA017 Ex Ante Measure Cost Study Draft Report”, Itron, February 28, 2014. See “NR HW Heater_WA017_MCS Results Matrix - Volume I.xls” for more information.

Equipment Type	Category	Install Cost	Incremental Cost
Gas Storage Water Heaters > 75,000 Btu/h	0.80 Et	\$4,886	N/A
	0.83 Et	\$5,106	\$220
	0.84 Et	\$5,299	\$413
	0.85 Et	\$5,415	\$529
	0.86 Et	\$5,532	\$646
	0.87 Et	\$5,648	\$762
	0.88 Et	\$5,765	\$879
	0.89 Et	\$5,882	\$996
	0.90 Et	\$6,021	\$1,135

Tank Size	Incremental Cost
50 gallons	\$1050
80 gallons	\$1050
100 gallons	\$1950

LOADSHAPE

For electric hot water heaters, use Loadshape C02 - Non-Residential Electric DHW.

COINCIDENCE FACTOR

The coincidence factor is assumed to be 0.925 ¹⁶⁴.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Electric energy savings are calculated for electric storage water heaters per the equations given below.

Electric units ≤12 kW:

$$\Delta kWh = \frac{(T_{out} - T_{in}) * HotWaterUse_{Gallon} * \gamma Water * 1 * \left(\frac{1}{EF_{elecbase}} - \frac{1}{EF_{Eff}} \right)}{3412}$$

Where:

T_{OUT} = Tank temperature
= 125°F

T_{IN} = Incoming water temperature from well or municipal system
= 54°F¹⁶⁵

¹⁶³ Act on Energy Commercial Technical Reference Manual, Table 9.6.1-4

¹⁶⁴ Coincidence factor based on Average W in peak period/Max W from Itron eShape data for Missouri, calibrated to Illinois loads,

¹⁶⁵ US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL
http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html

HotWaterUse_{Gallon}

= Estimated annual hot water consumption (gallons)

= Actual if possible to provide reasonable custom estimate. If not, two methodologies are provided to develop an estimate:

1. Consumption per usable storage tank capacity
= Capacity * Consumption/cap

Where:

Capacity = Usable capacity of hot water storage tank in gallons
= Actual

Consumption/cap = Estimate of consumption per gallon of usable tank capacity, based on building type:¹⁶⁶

Building Type ¹⁶⁷	Consumption/Cap
Convenience	528
Education	568
Grocery	528
Health	788
Large Office	511
Large Retail	528
Lodging	715
Other Commercial	341
Restaurant	622
Small Office	511
Small Retail	528
Warehouse	341
Nursing	672
Multi-Family	894

2. Consumption per unit area by building type
= (Area/1000) * Consumption/1,000 sq.ft.

Where:

Area = Area in sq.ft that is served by DHW boiler
= Actual

Consumption/1,000 sq.ft. = Estimate of DHW consumption per 1,000 sq.ft. based on building type:¹⁶⁸

¹⁶⁶ Methodology based on Cadmus analysis. Annual hot water usage in gallons based on CBECS (2012) and RECS (2009) consumption data of East North Central (removed outliers of 1,000 kBtuh or less) to calculate hot water usage. Annual hot water gallons per tank size gallons based on the tank sizing methodology found in ASHRAE 2011 HVAC Applications. Chapter 50 Service Water Heating. Demand assumptions (gallons per day) for each building type based on ASHRAE Chapter 50 and to LBNL White Paper. LBL-37398 Technology Data Characterizing Water Heating in Commercial Buildings: Application to End Use Forecasting. Assumes hot water heater efficiency of 80%.

¹⁶⁷ According to CBECS 2012 “Lodging” buildings include Dormitories, Hotels, Motel or Inns and other Lodging and “Nursing” buildings include Assisted Living and Nursing Homes.

¹⁶⁸ Methodology based on Cadmus analysis. Annual hot water usage in gallons based on CBECS (2012) and RECS (2009) consumption data of East North Central (removed outliers of 1,000 kBtuh or less) to calculate hot water usage. Annual hot water gallons per tank size gallons based on the tank sizing methodology found in ASHRAE 2011 HVAC Applications. Chapter 50 Service Water Heating. Demand assumptions (gallons per day) for each building type based on ASHRAE Chapter 50 and to LBNL

Building Type ¹⁶⁹	Consumption/1,000 sq.ft.
Convenience	4,594
Education	7,285
Grocery	697
Health	24,540
Large Office	1,818
Large Retail	1,354
Lodging	29,548
Other Commercial	3,941
Restaurant	44,439
Small Office	1,540
Small Retail	6,111
Warehouse	1,239
Nursing	30,503
Multi-Family	15,434

γ_{Water} = Specific weight capacity of water (lb/gal)

= 8.33 lbs/gal

1 = Specific heat of water (Btu/lb.°F)

EF_{elecbase} = Rated efficiency of baseline water heater expressed as Energy Factor (EF);

Equipment Type	Sub Category	Federal Standard Minimum Efficiency ¹⁷⁰
Electric Water Heaters ≤ 75,000 Btu/h	≤55 gallon tanks	0.96 – (0.0003 * rated volume in gallons) EF
	>55 gallon tanks ¹⁷¹	2.057 – (0.00113 * rated volume in gallons) EF
Electric Water Heaters > 75,000 Btu/h	≤12 kW	0.97 – (0.00132 * rated volume in gallons) EF
	> 12kW	N/A (For >12 kW Units see below)

EF_{eff} = Rated efficiency of efficient water heater expressed as Energy Factor (EF) or Thermal Efficiency (E_t)

= Actual

3412 = Converts Btu to kWh

For example, for a 200,000 Btu/h, 150 gallon, 90% Thermal Efficiency storage unit with rated standby loss of 1029 BTU/h installed in a 1500 ft² restaurant:

$$\Delta \text{kWh} = ((125 - 54) * ((1,500/1,000) * 44,439) * 8.33 * 1 * (1/0.8 - 1/0.9))/3412$$

$$= 1,605 \text{ kWh}$$

Electric units > 12kW:

White Paper. LBL-37398 Technology Data Characterizing Water Heating in Commercial Buildings: Application to End Use Forecasting. Assumes hot water heater efficiency of 80%.

¹⁶⁹ According to CBECS 2012 “Lodging” buildings include Dormitories, Hotels, Motel or Inns and other Lodging and “Nursing” buildings include Assisted Living and Nursing Homes.

¹⁷⁰ ≤75,000 Btu/h Storage Water Heater and <200,000 Btu/h Tankless Water Heater Federal Standard is DOE Standard 10 CFR 430.32(d). All other standards are from 10 CFR 431.110.

¹⁷¹ It is assumed that tanks <75,000Btu/h and >55 gallons will not be eligible measures due to the high baseline.

$$\Delta kWh = \frac{((T_{out} - T_{air}) * V * \gamma_{Water} * 1 * (SL_{elecbase} - SL_{eff})) * 8766}{3412}$$

T_{air} = Ambient Air Temperature

= 70°F

V = Rated tank volume in gallons

= Actual

$SL_{elecbase}$ = Standby loss of electric baseline unit (%/hr)

= 0.30 + 27/V

SL_{eff} = Nameplate standby loss of new water heater, in BTU/h

8766 = Hours per year

For example, >12kW, 100 gallon storage unit with rated standby loss of 0.5 %/hr:

$$SL_{base} = 0.3 + (27 / 100)$$

$$= 0.57\%/hr$$

$$\Delta kWh = (((125 - 70) * 100 * 8.33 * 1 * (0.57 - 0.5)) * 8766) / 3412$$

$$= 8,239 \text{ kWh}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \frac{\Delta kWh}{Hours} * CF$$

Where:

Hours = Full load hours of water heater

= 6461¹⁷²

CF = Summer Peak Coincidence Factor for measure

= 0.925¹⁷³

For example, >12kW, 100 gallon storage unit with rated standby loss of 0.5 %/hr:

$$\Delta kW = 8,239 / 6,461 * 0.925$$

$$= 1.18 \text{ kW}$$

NATURAL GAS ENERGY SAVINGS

Natural gas energy savings are calculated for natural gas storage water heaters per the equations given below.

$$\Delta Therms = \frac{(T_{out} - T_{in}) * HotWaterUse_{Gallon} * \gamma_{Water} * 1 * \left(\frac{1}{EF_{gasbase}} - \frac{1}{EF_{Eff}} \right)}{100,000}$$

Where:

¹⁷² Full load hours assumption based on Wh/Max W Ratio from Itron eShape data for Missouri, calibrated to Illinois loads,

¹⁷³ Coincidence factor based on Average W in peak period/Max W from Itron eShape data for Missouri, calibrated to Illinois loads,

100,000 = Converts Btu to Therms

$EF_{gasbase}$ = Rated efficiency of baseline water heater expressed as Energy Factor (EF) or Thermal Efficiency (E_t);

Equipment Type	Sub Category	Federal Standard Minimum Efficiency ¹⁷⁴
Gas Storage Water Heaters ≤ 75,000 Btu/h	≤55 gallon tanks	$0.675 - (0.0015 * \text{Rated Storage Volume in Gallons}) EF$
	>55 gallon tanks	$0.8012 - (0.00078 * \text{Rated Storage Volume in Gallons}) EF$
Gas Storage Water Heaters > 75,000 Btu/h	< 4000 Btu/h/gal	80% E_t

Additional Standby Loss Savings

Gas Storage Water Heaters >75,000 Btu/h can claim additional savings due to lower standby losses.

$$\Delta Therms_{Standby} = \frac{(SL_{gasbase} - SL_{eff}) * 8766}{100,000}$$

Where:

$SL_{gasbase}$ = Standby loss of gas baseline unit (Btu/h)

$$= Q/800 + 110\sqrt{V}$$

Q = Nameplate input rating in Btu/h

V = Rated volume in gallons

SL_{eff} = Nameplate standby loss of new water heater, in Btu/h

8766 = Hours per year

For example, for a 200,000 Btu/h, 150 gallon, 90% Thermal Efficiency storage unit with rated standby loss of 1029 BTU/h installed in a 1500 ft² restaurant:

$$\begin{aligned} \Delta Therms &= ((125 - 54) * ((1,500/1,000) * 44,439) * 8.33 * 1 * (1/0.8 - 1/0.9))/100,000 \\ &= 54.8 \text{ Therms} \end{aligned}$$

$$\begin{aligned} \Delta Therms_{Standby} &= (((200000/800 + 110 * \sqrt{150}) - 1079) * 8766)/100,000 \\ &= 49.8 \text{ Therms} \end{aligned}$$

$$\begin{aligned} \Delta Therms_{Total} &= 54.8 + 49.8 \\ &= 104.6 \text{ Therms} \end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

¹⁷⁴ ≤75,000 Btu/h Storage Water Heater and <200,000 Btu/h Tankless Water Heater Federal Standard is DOE Standard 10 CFR 430.32(d). All other standards are from 10 CFR 431.110.

MEASURE CODE: CI-HWE-STWH-V03-180101

REVIEW DEADLINE: 1/1/2019

4.3.2 Low Flow Faucet Aerators

DESCRIPTION

This measure relates to the direct installation of a low flow faucet aerator in a commercial building. Expected applications include small business, office, restaurant, or motel. Health care-specific inputs are defined for Laminar Flow Restrictor (LFR) devices. For multifamily or senior housing, the residential low flow faucet aerator should be used.

This measure was developed to be applicable to the following program types, DI.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be an energy efficient faucet aerator, for bathrooms rated at 1.5 gallons per minute (GPM) or less, or for kitchens rated at 2.2 GPM or less. For LFR devices, the installed equipment must be a device rated at 2.2 GPM or less. Savings are calculated on an average savings per faucet fixture basis.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is assumed to be a standard bathroom faucet aerator rated at 2.25 GPM or more, or a standard kitchen faucet aerator rated at 2.75 GPM or more. For LFR devices, the baseline condition is assumed to be no aerator at all, due to the contamination risk caused by faucet aerators in health care facilities and the baseline flow rate is assumed to be 3.74 GPM¹⁷⁵. Note if flow rates are measured, for example through a Direct Install program, then actual baseline flow rates should be used as opposed to the deemed values.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 9 years.¹⁷⁶

DEEMED MEASURE COST

The full install cost (including labor) for this measure is \$8¹⁷⁷ or program actual. For LFRs, The incremental cost is \$14.27¹⁷⁸ or program actual.

LOADSHAPE

Loadshape C02 - Commercial Electric DHW

COINCIDENCE FACTOR

The coincidence factor for this measure is dependent on building type as presented below.

¹⁷⁵ Workpaper WPSCGNRWH150827A, Laminar Flow Restrictors For Hospitals and Health Care Facilities.

¹⁷⁶ Table C-6, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. "http://neep.org/Assets/uploads/files/emv/emv-library/measure_life_GDS%5B1%5D.pdf"

¹⁷⁷ Direct-install price per faucet assumes cost of aerator and install time. (2011, Market research average of \$3 and assess and install time of \$5 (20min @ \$15/hr)

¹⁷⁸ Direct install price per faucet assumes cost of LFR (\$7.27) and install time (\$7) (Southern California Gas Company, Workpaper WPSCGNRWH150827A Revision #0, September, 2015).

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

NOTE THESE SAVINGS ARE *PER* FAUCET RETROFITTED¹⁷⁹.

$$\Delta \text{kWh} = \% \text{ElectricDHW} * ((\text{GPM}_{\text{base}} - \text{GPM}_{\text{low}}) / \text{GPM}_{\text{base}}) * \text{Usage} * \text{EPG}_{\text{electric}} * \text{ISR}$$

Where:

$\% \text{ElectricDHW}$ = proportion of water heating supplied by electric resistance heating

DHW fuel	$\% \text{Electric_DHW}$
Electric	100%
Fossil Fuel	0%

GPM_{base} = Average flow rate, in gallons per minute, of the baseline faucet “as-used”

= 1.39¹⁸⁰ or custom based on metering studies¹⁸¹ or if measured during DI:

= Measured full throttle flow * 0.83 throttling factor¹⁸²

Baseline for LFRs¹⁸³: = 3.74 * 0.83 = 3.10

GPM_{low} = Average flow rate, in gallons per minute, of the low-flow faucet aerator “as-used”

= 0.94¹⁸⁴ or custom based on metering studies¹⁸⁵ or if measured during DI:

= Rated full throttle flow * 0.95 throttling factor¹⁸⁶

¹⁷⁹ This algorithm calculates the amount of energy saved per aerator by determining the fraction of water consumption savings for the upgraded fixture. Due to the distribution of water consumption by fixture type, as well as the different number of fixtures in a building, several variables must be incorporated.

¹⁸⁰ DeOreo, B., and P. Mayer. Residential End Uses of Water Study Update. Forthcoming. ©2015 Water Research Foundation. Reprinted With Permission.

¹⁸¹ Measurement should be based on actual average flow consumed over a period of time rather than a onetime spot measurement for maximum flow. Studies have shown maximum flow rates do not correspond well to average flow rate due to occupant behavior which does not always use maximum flow.

¹⁸² 2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings. Page 1-265. www.seattle.gov/light/Conserve/Reports/paper_10.pdf

¹⁸³ Using measured flow rate assumption from Workpaper WPSCGNRWH150827A, Laminar Flow Restrictors For Hospitals and Health Care Facilities.

¹⁸⁴ Average retrofit flow rate for kitchen and bathroom faucet aerators from sources 2, 4, 5, and 7. This accounts for all throttling and differences from rated flow rates. Assumes all kitchen aerators at 2.2 gpm or less and all bathroom aerators at 1.5 gpm or less. The most comprehensive available studies did not disaggregate kitchen use from bathroom use, but instead looked at total flow and length of use for all faucets. This makes it difficult to reliably separate kitchen water use from bathroom water use. It is possible that programs installing low flow aerators lower than the 2.2 gpm for kitchens and 1.5 gpm for bathrooms will see a lower overall average retrofit flow rate.

¹⁸⁵ Measurement should be based on actual average flow consumed over a period of time rather than a onetime spot measurement for maximum flow. Studies have shown maximum flow rates do not correspond well to average flow rate due to occupant behavior which does not always use maximum flow.

¹⁸⁶ 2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings. Page 1-265. www.seattle.gov/light/Conserve/Reports/paper_10.pdf

$$\text{For LFRs}^{187}: \quad = 2.2 * 0.95 = 2.09$$

Usage = Estimated usage of mixed water (mixture of hot water from water heater line and cold water line) per faucet (gallons per year)

= If data is available to provide a reasonable custom estimate it should be used, if not use the following defaults (or substitute custom information in to the calculation):

Building Type	Gallons hot water per unit per day ¹⁸⁸ (A)	Unit	Estimated % hot water from Faucets ¹⁸⁹ (B)	Multiplier ¹⁹⁰ (C)	Unit	Days per year (D)	Annual gallons mixed water per faucet (A*B*C*D)
Small Office	1	person	100%	10	employees per faucet	250	2,500
Large Office	1	person	100%	45	employees per faucet	250	11,250
Fast Food Rest	0.7	meal/day	50%	75	meals per faucet	365	9,581
Sit-Down Rest	2.4	meal/day	50%	36	meals per faucet	365	15,768
Retail	2	employee	100%	5	employees per faucet	365	3,650
Grocery	2	employee	100%	5	employees per faucet	365	3,650
Warehouse	2	employee	100%	5	employees per faucet	250	2,500
Elementary School	0.6	person	50%	50	students per faucet	200	3,000
Jr High/High School	1.8	person	50%	50	students per faucet	200	9,000
Health	90	patient	25%	2	Patients per faucet	365	16,425
Motel	20	room	25%	1	faucet per room	365	1,825
Hotel	14	room	25%	1	faucet per room	365	1,278
Other	1	employee	100%	20	employees per faucet	250	5,000

EPG_{electric} = Energy per gallon of mixed water used by faucet (electric water heater)

$$= (8.33 * 1.0 * (\text{WaterTemp} - \text{SupplyTemp})) / (\text{RE}_{\text{electric}} * 3412)$$

= 0.0795 kWh/gal for Bath, 0.0969 kWh/gal for Kitchen, 0.139 kWh/gal for LFRs, 0.0919 kWh/gal for unknown

8.33 = Specific weight of water (lbs/gallon)

1.0 = Heat Capacity of water (btu/lb-°F)

WaterTemp = Assumed temperature of mixed water

¹⁸⁷ Using measured flow rate assumption from Workpaper WPCGNRWH150827A, Laminar Flow Restrictors For Hospitals and Health Care Facilities.

¹⁸⁸ Table 2-45 Chapter 49, Service Water Heating, 2007 ASHRAE Handbook, HVAC Applications.

¹⁸⁹ Estimated based on data provided in Appendix E; "Waste Not, Want Not: The Potential for Urban Water Conservation in California"; http://www.pacinst.org/reports/urban_usage/appendix_e.pdf

¹⁹⁰ Based on review of the Illinois plumbing code (Employees and students per faucet). Retail, grocery, warehouse and health are estimates. Meals per faucet estimated as 4 bathroom and 3 kitchen faucets and average meals per day of 250 (based on California study above) – 250/7 = 36. Fast food assumption estimated.

	= 86F for Bath, 93F for Kitchen 91F for Unknown ¹⁹¹ , 110F for health care facilities ¹⁹²
SupplyTemp	= Assumed temperature of water entering building = 54.1°F ¹⁹³
RE_electric	= Recovery efficiency of electric water heater = 98% ¹⁹⁴
3412	= Converts Btu to kWh (Btu/kWh)
ISR	= In service rate of faucet aerators dependant on install method as listed in table below ¹⁹⁵

Selection	ISR
Direct Install - Deemed	0.95

EXAMPLE

For example, a direct installed kitchen faucet in a large office with electric DHW:

$$\Delta \text{kWh} = 1 * ((1.39 - 0.94)/1.39) * 11,250 * 0.0969 * 0.95$$

$$= 335.3 \text{ kWh}$$

For example, a direct installed bathroom faucet in an Elementary School with electric DHW:

$$\Delta \text{kWh} = 1 * ((1.39 - 0.94)/1.39) * 3,000 * 0.0795 * 0.95$$

$$= 73.4 \text{ kWh}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = (\Delta \text{kWh} / \text{Hours}) * \text{CF}$$

Where:

ΔkWh = calculated value above on a per faucet basis

Hours = Annual electric DHW recovery hours for faucet use

$$= (\text{Usage} * 0.545^{196}) / \text{GPH}$$

= Calculate if usage is custom, if using default usage use:

¹⁹¹ Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group. If the aerator location is unknown an average of 91% should be used which is based on the assumption that 70% of household water runs through the kitchen faucet and 30% through the bathroom $(0.7*93)+(0.3*86)=0.91$.

¹⁹² Southern California Gas Company, Workpaper WPSCGNRWH150827A Revision #0, September, 2015

¹⁹³ US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL

http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html.

¹⁹⁴ Electric water heaters have recovery efficiency of 98%: <http://www.ahridirectory.org/ahridirectory/pages/home.aspx>

¹⁹⁵ ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program Table 3-8

http://ilsagfiles.org/SAG_files/Evaluation_Documents/ComEd/ComEd%20EPY2%20Evaluation%20Reports/ComEd_All_Electric_Single_Family_HEP_PY2_Evaluation_Report_Final.pdf

¹⁹⁶ 54.5% is the proportion of hot 120F water mixed with 54.1F supply water to give 90°F mixed faucet water.

Building Type	Annual Recovery Hours
Small Office	24
Large Office	109
Fast Food Rest	93
Sit-Down Rest	153
Retail	36
Grocery	36
Warehouse	24
Elementary School	29
Jr High/High School	88
Health	160
Motel	18
Hotel	12
Other	49

Where:

GPH = Gallons per hour recovery of electric water heater calculated for 85.9F temp rise (140-54.1), 98% recovery efficiency, and typical 12kW electric resistance storage tank.

= 56

CF = Coincidence Factor for electric load reduction

= Dependent on building type¹⁹⁷

Building Type	Coincidence Factor
Small Office	0.0064
Large Office	0.0288
Fast Food Rest	0.0084
Sit-Down Rest	0.0184
Retail	0.0043
Grocery	0.0043
Warehouse	0.0064
Elementary School	0.0096
Jr High/High School	0.0288
Health	0.0144
Motel	0.0006
Hotel	0.0004
Other	0.0128

¹⁹⁷ Calculated as follows: Assumptions for percentage of usage during peak period (1-5pm) were made and then multiplied by 65/365 (65 being the number of days in peak period) and by the number of total annual recovery hours to give an estimate of the number of hours of recovery during peak periods. There are 260 hours in the peak period so the probability you will see savings during the peak period is calculated as the number of hours of recovery during peak divided by 260. See 'C&I Faucet Aerator.xls' for details.

EXAMPLE

For example, a direct installed kitchen faucet in a large office with electric DHW:

$$\begin{aligned}\Delta kW &= 335.3/109 * 0.0288 \\ &= 0.0886 \text{ kW}\end{aligned}$$

For example, a direct installed bathroom faucet in an Elementary School with electric DHW:

$$\begin{aligned}\Delta kW &= 73.4/29 * 0.0096 \\ &= 0.0243 \text{ kW}\end{aligned}$$

FOSSIL FUEL IMPACT DESCRIPTIONS AND CALCULATION

$$\Delta \text{Therms} = \% \text{FossilDHW} * ((\text{GPM_base} - \text{GPM_low})/\text{GPM_base}) * \text{Usage} * \text{EPG_gas} * \text{ISR}$$

Where:

$\% \text{FossilDHW}$ = proportion of water heating supplied by fossil fuel heating

DHW fuel	$\% \text{Fossil_DHW}$
Electric	0%
Fossil Fuel	100%

EPG_gas = Energy per gallon of mixed water used by faucet (gas water heater)
 $= (8.33 * 1.0 * (\text{WaterTemp} - \text{SupplyTemp})) / (\text{RE_gas} * 100,000)$
 = 0.00397 Therm/gal for Bath, 0.00484 Therm/gal for Kitchen, 0.00695 Therm/gal for LFRs, 0.00459 Therm/gal for unknown

Where:

RE_gas = Recovery efficiency of gas water heater
 = 67%¹⁹⁸

100,000 = Converts Btus to Therms (Btu/Therm)

Other variables as defined above.

¹⁹⁸ Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 75%. Commercial properties are more similar to MF homes than SF homes. MF hot water is often provided by a larger commercial boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of .59 and the .75 for single family home. An average is used for this analysis by default.

EXAMPLE

For example, a direct installed kitchen faucet in a large office with gas DHW:

$$\begin{aligned}\Delta\text{Therms} &= 1 * ((1.39 - 0.94)/1.39) * 11,250 * 0.00484 * 0.95 \\ &= 16.7 \text{ Therms}\end{aligned}$$

For example, a direct installed bathroom faucet in an Elementary School with gas DHW:

$$\begin{aligned}\Delta\text{Therms} &= 1 * ((1.39 - 0.94)/1.39) * 3,000 * 0.00397 * 0.95 \\ &= 3.66 \text{ Therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

$$\Delta\text{gallons} = ((\text{GPM}_{\text{base}} - \text{GPM}_{\text{low}})/\text{GPM}_{\text{base}}) * \text{Usage} * \text{ISR}$$

Variables as defined above

EXAMPLE

For example, a direct installed faucet in a large office:

$$\begin{aligned}\Delta\text{gallons} &= ((1.39 - 0.94)/1.39) * 11,250 * 0.95 \\ &= 3,640 \text{ gallons}\end{aligned}$$

For example, a direct installed faucet in a Elementary School:

$$\begin{aligned}\Delta\text{gallons} &= ((1.39 - 0.94)/1.39) * 3,000 * 0.95 \\ &= 971 \text{ gallons}\end{aligned}$$

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

SOURCES USED FOR GPM ASSUMPTIONS

Source ID	Reference
1	2011, DeOreo, William. California Single Family Water Use Efficiency Study. April 20, 2011.
2	2000, Mayer, Peter, William DeOreo, and David Lewis. Seattle Home Water Conservation Study. December 2000.
3	1999, Mayer, Peter, William DeOreo. Residential End Uses of Water. Published by AWWA Research Foundation and American Water Works Association. 1999.
4	2003, Mayer, Peter, William DeOreo. Residential Indoor Water Conservation Study. Aquacraft, Inc. Water Engineering and Management. Prepared for East Bay Municipal Utility District and the US EPA. July 2003.
5	2011, DeOreo, William. Analysis of Water Use in New Single Family Homes. By Aquacraft. For Salt Lake City Corporation and US EPA. July 20, 2011.
6	2011, Aquacraft. Albuquerque Single Family Water Use Efficiency and Retrofit Study. For Albuquerque Bernalillo County Water Utility Authority. December 1, 2011.

Source ID	Reference
7	2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings.

MEASURE CODE: CI-HWE-LFFA-V07-180101

REVIEW DEADLINE: 1/1/2023

4.3.3 Low Flow Showerheads

DESCRIPTION

This measure relates to the direct installation of a low flow showerhead in a commercial building. Expected applications include small business, office, restaurant, or small motel. For multifamily or senior housing, the residential low flow showerhead should be used.

This measure was developed to be applicable to the following program types: DI.

If applied to other program types, the measure savings should be verified

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be an energy efficient showerhead rated at 2.0 gallons per minute (GPM) or less. Savings are calculated on a per showerhead fixture basis.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is assumed to be a standard showerhead rated at 2.5 GPM.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years.¹⁹⁹

DEEMED MEASURE COST

The full install cost (including labor) for this measure is \$12²⁰⁰ or program actual.

LOADSHAPE

Loadshape C02 - Commercial Electric DHW

COINCIDENCE FACTOR

The coincidence factor for this measure is assumed to be 2.78%²⁰¹.

Algorithm

CALCULATION OF SAVINGS²⁰²**ELECTRIC ENERGY SAVINGS**

Note these savings are per showerhead fixture

¹⁹⁹ Table C-6, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. Evaluations indicate that consumer dissatisfaction may lead to reductions in persistence, particularly in Multi-Family , ["http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf"](http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf)

²⁰⁰ Direct-install price per showerhead assumes cost of showerhead (Market research average of \$7 and assess and install time of \$5 (20min @ \$15/hr)

²⁰¹ Calculated as follows: Assume 11% showers take place during peak hours (based on: <http://www.aquacraft.com/sites/default/files/pub/DeOreo-%282001%29-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf>). There are 65 days in the summer peak period, so the percentage of total annual aerator use in peak period is $0.11 * 65 / 365 = 1.96\%$. The number of hours of recovery during peak periods is therefore assumed to be $1.96\% * 369 = 7.23$ hours of recovery during peak period. There are 260 hours in the peak period so the probability you will see savings during the peak period is $7.23 / 260 = 0.0278$

²⁰²Based on excel spreadsheet 120911.xls ...on SharePoint

$\Delta kWh =$

$$\%ElectricDHW * ((GPM_base * L_base - GPM_low * L_low) * NSPD * 365.25) * EPG_electric * ISR$$

Where:

$\%ElectricDHW$ = proportion of water heating supplied by electric resistance heating
= 1 if electric DHW, 0 if fuel DHW, if unknown assume 16%²⁰³

GPM_base = Flow rate of the baseline showerhead
= 2.67 for Direct-install programs²⁰⁴

GPM_low = As-used flow rate of the low-flow showerhead, which may, as a result of measurements of program evaluations deviate from rated flows, see table below:

Rated Flow
2.0 GPM
1.75 GPM
1.5 GPM
Custom or Actual ²⁰⁵

L_base = Shower length in minutes with baseline showerhead
= 8.20 min²⁰⁶

L_low = Shower length in minutes with low-flow showerhead
= 8.20 min²⁰⁷

365.25 = Days per year, on average.

$NSPD$ = Estimated number of showers taken per day for one showerhead

$EPG_electric$ = Energy per gallon of hot water supplied by electric
= $(8.33 * 1.0 * (ShowerTemp - SupplyTemp)) / (RE_electric * 3412)$
= $(8.33 * 1.0 * (105 - 54.1)) / (0.98 * 3412)$
= 0.127 kWh/gal

8.33 = Specific weight of water (lbs/gallon)

1.0 = Heat Capacity of water (btu/lb-°F)

$ShowerTemp$ = Assumed temperature of water
= 105°F²⁰⁸

²⁰³ Table HC8.9. Water Heating in U.S. Homes in Midwest Region, Divisions, and States, 2009 (RECS)

²⁰⁴ Based on measured data from Ameren IL EM&V of Direct-Install program. Program targets showers that are rated 2.5 GPM or above.

²⁰⁵ Note that actual values may be either a) program-specific minimum flow rate, or b) program-specific evaluation-based value of actual effective flow-rate due to increased duration or temperatures. The latter increases in likelihood as the rated flow drops and may become significant at or below rated flows of 1.5 GPM. The impact can be viewed as the inverse of the throttling described in the footnote for baseline flowrate.

²⁰⁶ Representative value from sources 1, 2, 3, 4, 5, and 6 (See Source Table at end of measure section)

²⁰⁷ Set equal to L_base .

²⁰⁸ Shower temperature cited from SBW Consulting, Evaluation for the Bonneville Power Authority, 1994, http://www.bpa.gov/energy/n/reports/evaluation/residential/faucet_aerator.cfm

SupplyTemp	= Assumed temperature of water entering house = 54.1°F ²⁰⁹
RE_electric	= Recovery efficiency of electric water heater = 98% ²¹⁰
3412	= Converts Btu to kWh (btu/kWh)
ISR	= In service rate of showerhead = Dependant on program delivery method as listed in table below

Selection	ISR ²¹¹
Direct Install - Deemed	0.98

EXAMPLE

For example, a direct-installed 1.5 GPM showerhead in an office with electric DHW where the number of showers is estimated at 3 per day:

$$\begin{aligned}\Delta\text{kWh} &= 1 * ((2.67*8.20) - (1.5*8.20)) * 3*365.25 * 0.127 * 0.98 \\ &= 1308.4 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta\text{kW} = \Delta\text{kWh}/\text{Hours} * \text{CF}$$

Where:

$$\begin{aligned}\Delta\text{kWh} &= \text{calculated value above} \\ \text{Hours} &= \text{Annual electric DHW recovery hours for showerhead use} \\ &= ((\text{GPM_base} * \text{L_base}) * \text{NSPD} * 365.25) * 0.773^{212} / \text{GPH}\end{aligned}$$

Where:

$$\begin{aligned}\text{GPH} &= \text{Gallons per hour recovery of electric water heater calculated for 65.9°F temp rise (120-54.1), 98\% recovery efficiency, and typical 4.5kW electric resistance storage tank.} \\ &= 27.51 \\ \text{CF} &= \text{Coincidence Factor for electric load reduction} \\ &= 0.0278^{213}\end{aligned}$$

²⁰⁹ US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL

http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html.

²¹⁰ Electric water heaters have recovery efficiency of 98%: <http://www.ahridirectory.org/ahridirectory/pages/home.aspx>

²¹¹ Deemed values are from ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program Table 3-8. Alternative ISRs may be developed for program delivery methods based on evaluation results.

²¹² 77.3% is the proportion of hot 120°F water mixed with 54.1°F supply water to give 105°F shower water

²¹³ Calculated as follows: Assume 11% showers take place during peak hours (based on:

<http://www.aquacraft.com/sites/default/files/pub/DeOreo-%282001%29-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf>). There are 65 days in the summer peak period, so the percentage of total annual aerator use in peak period is $0.11 * 65 / 365.25 = 1.96\%$. The number of hours of recovery during peak periods is therefore assumed to be $1.96\% * 369 = 7.23$ hours of recovery during peak period where 369 equals the average annual electric DHW recovery hours for

EXAMPLE

For example, a direct-installed 1.5 GPM showerhead in an office with electric DHW where the number of showers is estimated at 3 per day:

$$\begin{aligned}\Delta kW &= (1308.4 / 674.1) * 0.0278 \\ &= 0.054 \text{ kW}\end{aligned}$$

FOSSIL FUEL IMPACT DESCRIPTIONS AND CALCULATION

$$\Delta \text{Therms} = \% \text{FossilDHW} * ((\text{GPM_base} * \text{L_base} - \text{GPM_low} * \text{L_low}) * \text{NSPD} * 365.25) * \text{EPG_gas} * \text{ISR}$$

Where:

$\% \text{FossilDHW}$ = proportion of water heating supplied by fossil fuel heating

DHW fuel	$\% \text{Fossil_DHW}$
Electric	0%
Fossil Fuel	100%
Unknown	84% ²¹⁴

$$\begin{aligned}\text{EPG_gas} &= \text{Energy per gallon of Hot water supplied by gas} \\ &= (8.33 * 1.0 * (\text{ShowerTemp} - \text{SupplyTemp})) / (\text{RE_gas} * 100,000) \\ &= 0.0063 \text{ Therm/gal}\end{aligned}$$

Where:

$$\begin{aligned}\text{RE_gas} &= \text{Recovery efficiency of gas water heater} \\ &= 67\%²¹⁵\end{aligned}$$

$$100,000 = \text{Converts Btus to Therms (btu/Therm)}$$

Other variables as defined above.

showerhead use including SF and MF homes with Direct Install and Retrofit/TOS measures. There are 260 hours in the peak period so the probability you will see savings during the peak period is $7.23/260 = 0.0278$

²¹⁴ Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

²¹⁵ Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 75%. Commercial properties are more similar to MF homes than SF homes. MF hot water is often provided by a larger commercial boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of .59 and the .75 for single family home. An average is used for this analysis by default.

EXAMPLE

For example, a direct-installed 1.5 GPM showerhead in an office with gas DHW where the number of showers is estimated at 3 per day:

$$\begin{aligned}\Delta\text{Therms} &= 1.0 * ((2.67 * 8.2) - (1.5 * 8.2)) * 3 * 365.25 * 0.0063 * 0.98 \\ &= 64.9 \text{ therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

$$\Delta\text{gallons} = ((\text{GPM}_{\text{base}} * L_{\text{base}} - \text{GPM}_{\text{low}} * L_{\text{low}}) * \text{NSPD} * 365.25 * \text{ISR})$$

Variables as defined above

EXAMPLE

For example, a direct-installed 1.5 GPM showerhead in an office with where the number of showers is estimated at 3 per day:

$$\begin{aligned}\Delta\text{gallons} &= ((2.67 * 8.2) - (1.5 * 8.2)) * 3 * 365.25 * 0.98 \\ &= 10,302 \text{ gallons}\end{aligned}$$

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

SOURCES

Source ID	Reference
1	2011, DeOreo, William. California Single Family Water Use Efficiency Study. April 20, 2011.
2	2000, Mayer, Peter, William DeOreo, and David Lewis. Seattle Home Water Conservation Study. December 2000.
3	1999, Mayer, Peter, William DeOreo. Residential End Uses of Water. Published by AWWA Research Foundation and American Water Works Association. 1999.
4	2003, Mayer, Peter, William DeOreo. Residential Indoor Water Conservation Study. Aquacraft, Inc. Water Engineering and Management. Prepared for East Bay Municipal Utility District and the US EPA. July 2003.
5	2011, DeOreo, William. Analysis of Water Use in New Single Family Homes. By Aquacraft. For Salt Lake City Corporation and US EPA. July 20, 2011.
6	2011, Aquacraft. Albuquerque Single Family Water Use Efficiency and Retrofit Study. For Albuquerque Bernalillo County Water Utility Authority. December 1, 2011.
7	2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings.

MEASURE CODE: CI-HWE-LFSH-V04-180101

REVIEW DEADLINE: 1/1/2020

4.3.4 Commercial Pool Covers

DESCRIPTION

This measure refers to the installation of covers on commercial use pools that are heated with gas-fired equipment located either indoors or outdoors. By installing pool covers, the heating load on the pool boiler will be reduced by reducing the heat loss from the water to the environment and the amount of actual water lost due to evaporation (which then requires additional heated water to make up for it).

The main source of energy loss in pools is through evaporation. This is particularly true of outdoor pools where wind plays a larger role. The point of installing pool covers is threefold. First, it will reduce convective losses due to the wind by shielding the water surface. Second, it will insulate the water from the colder surrounding air. And third, it will reduce radiative losses to the night sky. In doing so, evaporative losses will also be minimized, and the boiler will not need to work as hard in replenishing the pool with hot water to keep the desired temperature.

This measure can be used for pools that (1) currently do not have pool covers, (2) have pool covers that are past the useful life of the existing cover, or (3) have pool covers that are past their warranty period and have failed.

DEFINITION OF EFFICIENT EQUIPMENT

For indoor pools, the efficient case is the installation of an indoor pool cover with a 5 year warranty on an indoor pool that operates all year.

For outdoor pools, the efficient case is the installation of an outdoor pool cover with a 5 year warranty on an outdoor pool that is open through the summer season.

DEFINITION OF BASELINE EQUIPMENT

For indoor pools, the base case is an uncovered indoor pool that operates all year.

For outdoor pools, the base case is an outdoor pool that is uncovered and is open through the summer season.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The useful life of this measure is assumed to be 6 years ²¹⁶

DEEMED MEASURE COST

The table below shows the costs for the various options and cover sizes. Since this measure covers a mix of various sizes, the average cost of these options is taken to be the incremental measure cost. ²¹⁷

Cover Size	Edge Style	
	Hemmed (indoor)	Weighted (outdoor)
1000-1,999 sq. ft.	\$2.19	\$2.24
2,000-2,999 sq. ft.	\$2.01	\$2.06
3,000+ sq. ft.	\$1.80	\$1.83
Average	\$2.00	\$2.04

²¹⁶ The effective useful life of a pool cover is typically one year longer than its warranty period. SolaPool Covers. Pool Covers Website, FAQ- "How long will my SolaPool cover blanket last?". Pool covers are typically offered with 3 and 5 year warranties with at least one company offering a 6 year warranty. Conversation with Trade Ally. Knorr Systems

²¹⁷ Pool Cover Costs: Lincoln Commercial Pool Equipment website. Accessed 8/26/11.

<http://www.lincolnaquatics.com/shop/catalog/Pool+and+Spa+Covers+and+Accessories/product.html?ProductID=84-010>

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

NET TO GROSS RATIO

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS SAVINGS

The calculations are based on modeling runs using RSPEC! Energy Smart Pools Software that was created by the U.S. Department of Energy.²¹⁸

$$\Delta \text{Therms} = \text{SavingFactor} \times \text{Size of Pool}$$

Where

Savings factor = dependant on pool location and listed in table below²¹⁹

Location	Therm / sq-ft
Indoor	2.61
Outdoor	1.01

Size of Pool = custom input

WATER IMPACT DESCRIPTIONS AND CALCULATION

$$\Delta \text{Therms} = \text{WaterSavingFactor} \times \text{Size of Pool}$$

Where

WaterSavingFactor = Water savings for this measure dependant on pool location and listed in table below.²²⁰

Location	Annual Savings Gal / sq-ft
Indoor	15.28
Outdoor	8.94

²¹⁸ Full method and supporting information found in reference document: IL TRM - Business Pool Covers WorkPaper.docx. Note that the savings estimates are based upon Chicago weather data.

²¹⁹ Business Pool Covers.xlsx

²²⁰ Ibid.

Size of Pool = Custom input

DEEMED O&M COST ADJUSTMENT CALCULATION

There are no O&M cost adjustments for this measure.

MEASURE CODE: CI-HWE-PLCV-V01-130601

REVIEW DEADLINE: 1/1/2020

4.3.5 Tankless Water Heater

DESCRIPTION

This measure covers the installation of on-demand or instantaneous tankless water heaters. Tankless water heaters function similar to standard hot water heaters except they do not have a storage tank. When there is a call for hot water, the water is heated instantaneously as it passes through the heating element and then proceeds to the user or appliance calling for hot water. Tankless water heaters achieve savings by eliminating the standby losses that occur in stand-alone or tank-type water heaters and by being more efficient than the baseline storage hot water heater.

This measure was developed to be applicable to the following program types: TOS, RF, ER.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

Electric	Gas
To qualify for this measure, the tankless water heater shall be a new electric powered tankless hot water heater with an energy factor greater than or equal to 0.98 with an output greater than or equal to 5 GPM output at 70° F temperature rise.	To qualify for this measure, the tankless water heater shall meet or exceed the efficiency requirements for tankless hot water heaters mandated by the International Energy Conservation Code (IECC) 2012/2015, Table C404.2.

DEFINITION OF BASELINE EQUIPMENT

Electric	Gas
The baseline condition is assumed to be an electric commercial-grade tanked water heater 50 or more gallon storage capacity with an energy factor less than or equal to 0.9 or the water heater is five or more years old.	The baseline condition is assumed to be a gas-fired tank-type water heater meeting the efficiency requirements mandated by the International Energy conservation Code (IECC) 2012/2015, Table C404.2

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

Electric	Gas
The expected measure life is assumed to be 5 years ²²¹ .	The expected measure life is assumed to be 20 years ²²²

DEEMED MEASURE COST

The incremental capital cost for an electric tankless heater this measure is assumed to be²²³

Output (gpm) at delta T 70	Incremental Cost
5	\$1050
10	\$1050
15	\$1950

The incremental capital cost for a gas fired tankless heater is as follows:

²²¹ Ohio Technical Reference Manual 8/2/2010 referencing CenterPoint Energy-Triennial CIP/DSM Plan 2010-2012 Report; Additional reference stating >20 years is at Energy Savers.Gov online at

http://www.energysavers.gov/your_home/water_heating/index.cfm/mytopic=12820

²²² Ibid.

²²³ Act on Energy Technical Reference Manual, Table 9.6.2-3

Program	Capital Cost, \$ per unit
Retrofit	\$3,255 ²²⁴
Time of Sale or New Construction	\$2,526 ²²⁵

DEEMED O&M COST ADJUSTMENTS\$100²²⁶**LOADSHAPE**

Loadshape C02 - Commercial Electric DHW

COINCIDENCE FACTOR

The measure has deemed kW savings therefor a coincidence factor is not applied

Algorithm

CALCULATION OF ENERGY SAVINGS**ELECTRIC ENERGY SAVINGS²²⁷**

The annual electric savings from an electric tankless heater is a deemed value and assumed to be:

Output (gpm) at delta T 70	Savings (kWh)
5.0	2,992
10.0	7,905
15.0	12,879

SUMMER COINCIDENT PEAK DEMAND SAVINGS²²⁸

The annual kW savings from an electric tankless heater is a deemed value and assumed to be:

Output (gpm) at delta T 70	Savings (kW)
5.0	0.34
10.0	0.90
15.0	1.47

²²⁴ Based on AOE historical average installation data of 42 tankless gas hot water heaters²²⁵ <http://www.mncee.org/getattachment/7b8982e9-4d95-4bc9-8e64-f89033617f37/>, Low contractor estimate used to reflect less labor required in new construction of venting.²²⁶ Water heaters (WH) require annual maintenance. There are different levels of effort for annual maintenance depending if the unit is gas or electric, tanked or tankless. Electric and gas tank water heater manufacturers recommend an annual tank drain to clear sediments. Also recommended are “periodic” inspections by qualified service professionals of operating controls, heating element and wiring for electric WHs and thermostat, burner, relief valve internal flue-way and venting systems for gas WHs. Tankless WH require annual maintenance by licensed professionals to clean control compartments, burners, venting system and heat exchangers. This information is from WH manufacturer product brochures including GE, Rinnai, Rheem, Takagi and Kenmore. References for incremental O&M costs were not found. Therefore the incremental cost of the additional annual maintenance for tankless WH is estimated at \$100.²²⁷ Act on Energy Technical Reference Manual, Table 9.6.2-3²²⁸ Ibid.

NATURAL GAS SAVINGS

$$\Delta \text{Therms} = \frac{[W_{\text{gal}} \times 8.33 \times 1 \times (T_{\text{out}} - T_{\text{in}}) \times [(1/\text{Eff}_{\text{base}}) - (1/\text{Eff}_{\text{ee}})]]}{100,000} + \frac{[(SL \times 8,766)/\text{Eff}_{\text{base}}]}{100,000} \text{ Btu/Therms}$$

Where:

Wgal	= Annual water use for equipment in gallons = custom, otherwise assume 21,915 gallons ²²⁹
8.33 lbm/gal	= weight in pounds of one gallon of water
1 Btu/lbm°F	= Specific heat of water: 1 Btu/lbm/°F
8,766 hr/yr	= hours a year
Tout	= Unmixed Outlet Water Temperature = custom, otherwise assume 130 °F ²³⁰
Tin	= Inlet Water Temperature = custom, otherwise assume 54.1 °F ²³¹
Eff base	= Rated efficiency of baseline water heater expressed as Energy Factor (EF) or Thermal Efficiency (Et); see table below ²³²

Input Btu/hr of existing, tanked water heater	Eff base	Units
Size: ≤ 75,000 Btu/hr	0.67 - 0.0019 * Tank Volume	Energy Factor
Size: >75,000 Btu/hr and ≤ 155,000 Btu/hr	80%	Thermal Efficiency
Size: >155,000 Btu/hr	80%	Thermal Efficiency

Where:

Tank Volume = custom input, if unknown assume 60 gallons for Size: ≤ 75,000 Btu/hr

Please note: Units in base case must match units in efficient case. If Energy Factor used in base case, Energy Factor to be used in efficient case. If Thermal Efficiency is used in base case, Thermal Efficiency must be used in efficient case.

Eff ee	= Rated efficiency of efficient water heater expressed as Energy Factor (EF) or Thermal Efficiency (Eff t) = custom input, if unknown assume 0.84 ²³³
--------	---

²²⁹ 21,915 gallons is an estimate of 60 gal/day for 365.25 days/yr. If building type is known, reference 2007 ASHRAE Handbook HVAC Applications p. 49.14 Table 7 Hot Water Demands and Use for Various Types of Buildings to help estimate hot water consumption.

²³⁰ Based on 2010 Ohio Technical Reference Manual and NAEHB Research Center, (2002) Performance Comparison of Residential hot Water Systems. Prepared for National Renewable Energy Laboratory, Golden, Colorado.

²³¹ August 31, 2011 Memo of Savings for Hot Water Savings Measures to Nicor Gas from Navigant states that 54.1°F was calculated from the weighted average of monthly water mains temperatures reported in the 2010 Building America Benchmark Study for Chicago-Waukegan, Illinois.

²³² International Energy Conservation Code (IECC) 2012/2015, Table C404.2, Minimum Performance of Water-Heating Equipment

²³³ Specifications of energy efficient tankless water heater. Reference Consortium for Energy Efficiency (CEE) which maintains a list of high efficiency tankless water heaters which currently have Energy Factors up to .96. Ameren currently requires minimum .82 energy factor.

SL = Stand-by Loss in Base Case Btu/hr
 = custom input based on formula in table below, if unknown assume unit size in table below²³⁴

Input Btu/h of new, tankless water heater	Standby Loss (SL)
Size: ≤ 75,000 Btu/hr	0
Size: >75,000 Btu/hr	(Input rating/800)+(110*√Tank Volume))

Where:

Tank Volume = custom input, if unknown assume, 60 gallons for <75,000 Btu/hr, 75 gallons for >75,000 Btu/hr and ≤ 155,000 Btu/hr and 150 for Size >155,000 Btu/hr

Input Rating = nameplate Btu/hr rating of water heater

EXAMPLE

For example, a 75,000 Btu/hr tankless unit using 21,915 gal/yr with outlet temperature at 130.0 and inlet temperature at 54.1, replacing a baseline unit with 0.8 thermal efficiency and standby losses of 1008.3 btu/hr:

$$\Delta \text{Therms} = \left[\left[\left(\frac{21,915 \times 8.33 \times 1}{100,000} \times (130 - 54.1) \times \left[\left(\frac{1}{0.8} \right) - \left(\frac{1}{0.84} \right) \right] \right) + (1008.3 \times 8,766) \right] \right] / 100,000$$

$$= 115 \text{ Therms}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

The deemed O&M cost adjustment for a gas fired tankless heater is \$100

REFERENCE TABLES

Minimum Performance Water Heating Equipment²³⁵

²³⁴ Stand-by loss is provided in 2012/2015 IECC, Table C404.2, Minimum Performance of Water-Heating Equipment

²³⁵ International Energy Conservation Code (IECC)2012/2015

TABLE C404.2
MINIMUM PERFORMANCE OF WATER-HEATING EQUIPMENT

EQUIPMENT TYPE	SIZE CATEGORY (input)	SUBCATEGORY OR RATING CONDITION	PERFORMANCE REQUIRED ^{a, b}	TEST PROCEDURE
Water heaters, electric	≤ 12 kW	Resistance	0.97 - 0.00132 V, EF	DOE 10 CFR Part 430
	> 12 kW	Resistance	1.73 V - 155 SL, Btu/h	ANSI Z21.10.3
	≤ 24 amps and ≤ 260 volts	Heat pump	0.93 - 0.00132 V, EF	DOE 10 CFR Part 430
Storage water heaters, gas	≤ 75,000 Btu/h	≥ 20 gal	0.67 - 0.0019 V, EF	DOE 10 CFR Part 430
	> 75,000 Btu/h and ≤ 155,000 Btu/h	< 4,000 Btu/h/gal	$80\% E_r$ $(Q/800 + 110/\sqrt{V})$ SL, Btu/h	ANSI Z21.10.3
	> 155,000 Btu/h	< 4,000 Btu/h/gal	$80\% E_r$ $(Q/800 + 110/\sqrt{V})$ SL, Btu/h	
Instantaneous water heaters, gas	> 60,000 Btu/h and < 200,000 Btu/h ^c	≥ 4,000 (Btu/h)/gal and < 2 gal	0.62 - 0.0019 V, EF	DOE 10 CFR Part 430
	≥ 200,000 Btu/h	≥ 4,000 Btu/h/gal and < 10 gal	80% E_r	ANSI Z21.10.3
	≥ 200,000 Btu/h	≥ 4,000 Btu/h/gal and ≥ 10 gal	$80\% E_r$ $(Q/800 + 110/\sqrt{V})$ SL, Btu/h	
Storage water heaters, oil	≤ 105,000 Btu/h	≥ 20 gal	0.59 - 0.0019 V, EF	DOE 10 CFR Part 430
	≥ 105,000 Btu/h	< 4,000 Btu/h/gal	$78\% E_r$ $(Q/800 + 110/\sqrt{V})$ SL, Btu/h	ANSI Z21.10.3
Instantaneous water heaters, oil	≤ 210,000 Btu/h	≥ 4,000 Btu/h/gal and < 2 gal	0.59 - 0.0019 V, EF	DOE 10 CFR Part 430
	> 210,000 Btu/h	≥ 4,000 Btu/h/gal and < 10 gal	80% E_r	ANSI Z21.10.3
	> 210,000 Btu/h	≥ 4,000 Btu/h/gal and ≥ 10 gal	$78\% E_r$ $(Q/800 + 110/\sqrt{V})$ SL, Btu/h	
Hot water supply boilers, gas and oil	≥ 300,000 Btu/h and < 12,500,000 Btu/h	≥ 4,000 Btu/h/gal and < 10 gal	80% E_r	ANSI Z21.10.3
Hot water supply boilers, gas	≥ 300,000 Btu/h and < 12,500,000 Btu/h	≥ 4,000 Btu/h/gal and ≥ 10 gal	$80\% E_r$ $(Q/800 + 110/\sqrt{V})$ SL, Btu/h	
Hot water supply boilers, oil	> 300,000 Btu/h and < 12,500,000 Btu/h	> 4,000 Btu/h/gal and > 10 gal	$78\% E_r$ $(Q/800 + 110/\sqrt{V})$ SL, Btu/h	
Pool heaters, gas and oil	All	—	78% E_r	ASHRAE 146
Heat pump pool heaters	All	—	4.0 COP	AHRI 1160
Unfired storage tanks	All	—	Minimum insulation requirement R-12.5 (h · ft ² · °F)/Btu	(none)

For SI: °C = [(°F) - 32]/1.8, 1 British thermal unit per hour = 0.2931 W, 1 gallon = 3.785 L, 1 British thermal unit per hour per gallon = 0.078 W/L.

a. Energy factor (EF) and thermal efficiency (E_r) are minimum requirements. In the EF equation, V is the rated volume in gallons.

b. Standby loss (SL) is the maximum Btu/h based on a nominal 70°F temperature difference between stored water and ambient requirements. In the SL equation, Q is the nameplate input rate in Btu/h. In the SL equation for electric water heaters, V is the rated volume in gallons. In the SL equation for oil and gas water heaters and boilers, V is the rated volume in gallons.

c. Instantaneous water heaters with input rates below 200,000 Btu/h must comply with these requirements if the water heater is designed to heat water to temperatures 180°F or higher.

MEASURE CODE: CI-HWE-TKWH-V03-160601

REVIEW DEADLINE: 1/1/2019

4.3.6 Ozone Laundry

DESCRIPTION

A new ozone laundry system(s) is added-on to new or existing commercial washing machine(s) using hot water heated with natural gas. The system generates ozone (O₃), a naturally occurring molecule, which helps clean fabrics by chemically reacting with soils in cold water. Adding an ozone laundry system(s) will reduce the amount of chemicals, detergents, and hot water needed to wash linens. Using ozone also reduces the total amount of water consumed, saving even more in energy.

Natural gas energy savings will be achieved at the hot water heater/boiler as they will be required to produce less hot water to wash each load of laundry. The decrease in hot water usage will increase cold water usage, but overall water usage at the facility will decrease.

Electric savings can be realized through reduced washer cycle length and reduced pumping load at the boiler feeding the commercial washers. The increased usage associated with operating the ozone system should also be accounted for when determining total kWh impact. Data reviewed for this measure characterization indicated that pumping savings should be accounted for, but washer savings and ozone generator consumption are comparatively so small that they can be ignored.

The reduced washer cycle length may decrease the dampness of the clothes when they move to the dryer. This can result in shorter runtimes which result in gas and electrical savings. However, at this time, there is inconclusive evidence that energy savings are achieved from reduced dryer runtimes so the resulting dryer effects are not included in this analysis. Additionally, there would be challenges verifying that dryer savings will be achieved throughout the life of the equipment.

This incentive only applies to the following facilities with on-premise laundry operations:

- Hotels/motels
- Fitness and recreational sports centers.
- Healthcare (excluding hospitals)
- Assisted living facilities

Ozone laundry system(s) could create significant energy savings opportunities at other larger facility types with on-premise laundry operations (such as correctional facilities, universities, and staff laundries), however, the results included in this analysis are based heavily on past project data for the applicable facility types listed above and may not apply to facilities outside of this list due to variances in number of loads and average pound (lbs.)-capacity per project site. Projects at these facilities should continue to be evaluated through custom programs and the applicable facility types and the resulting analysis should be updated based on new information.

This measure was developed to be applicable to the following program types: TOS, RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

A new ozone laundry system(s) is added-on to new or existing commercial washing machine(s) using hot water heated with natural gas. The ozone laundry system(s) must transfer ozone into the water through:

- Venturi Injection
- Bubble Diffusion
- Additional applications may be considered upon program review and approval on a case by case basis

DEFINITION OF BASELINE EQUIPMENT

The base case equipment is a conventional washing machine system with no ozone generator installed. The washing machines are provided hot water from a gas-fired boiler.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure equipment effective useful life (EUL) is estimated at 10 years based on typical lifetime of the ozone generator's corona discharge unit.²³⁶

DEEMED MEASURE COST

The actual measure costs should be used if available. If not a deemed value of \$79.84 / lbs capacity should be used²³⁷.

LOADSHAPE

Loadshape C53 – Flat

COINCIDENCE FACTOR

Past project documentation and data collection is not sufficient to determine a coincidence factor for this measure. Value should continue to be studied and monitored through additional studies due to limited data points used for this determination

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

Electric savings can be realized through reduced washer cycle length and reduced pumping load at the boiler feeding the commercial washers. There is also an increased usage associated with operating the ozone system. Data reviewed for this measure characterization indicated that while pumping savings is significant and should be accounted for, washer savings and ozone generator consumption are negligible, counter each other out and are well within the margin of error so these are not included to simplify the characterization²³⁸.

$$\Delta \text{kWh}_{\text{PUMP}} = \text{HP} * \text{HP}_{\text{CONVERSION}} * \text{Hours} * \% \text{water_savings}$$

Where:

$\Delta \text{kWh}_{\text{PUMP}}$ = Electric savings from reduced pumping load
 HP = Brake horsepower of boiler feed water pump;
 = Actual or use 5 HP if unknown²³⁹

²³⁶ Aligned with other national energy efficiency programs and confirmed with national vendors

²³⁷ Average costs per unit of capacity were generated using data collected from existing ozone laundry projects that received incentives under the Non-Residential Retrofit Demand Reduction program (NRR-DR), as well as from the Nicor Custom Incentive Program, and the Nicor Emerging Technology Program (ETP). See referenced document Table 2 and RSMMeans Mechanical Cost Data, 31st Annual Edition (2008)

²³⁸ Washer savings were reviewed but were considered negligible and not included in the algorithm (0.00082 kWh / lbs-capacity, determined through site analysis through Nicor Emerging Technology Program (ETP) and confirmed with national vendors). Note that washer savings from Nicor's site analysis are smaller than those reported in a WI Focus on Energy case study (0.23kWh/100lbs, Hampton Inn Brookfield, November 2010). Electric impact of operating ozone generator (0.0021 kWh / lbs-capacity same source as washer savings) was also considered negligible and not included in calculations. Values should continue to be studied and monitored through additional studies due to limited data points used for this determination.

²³⁹ Assumed average horsepower for boilers connected to applicable washer

HP_{CONVERSION} = Conversion from Horsepower to Kilowatt
= 0.746

Hours = Actual associated boiler feed water pump hours
= 800 hours if unknown²⁴⁰

%water_savings = water reduction factor: how much more efficient an ozone injection washing machine is compared to a typical conventional washing machine as a rate of hot and cold water reduction.
= 25%²⁴¹

Using defaults above:

$$\Delta \text{kWh}_{\text{PUMP}} = 5 * 0.746 * 800 * 0.25$$

$$= 746 \text{ kWh}$$

Default per lb capacity: = $\Delta \text{kWh}_{\text{PUMP}} / \text{lb capacity}$

Where:

$$\text{Lbs-Capacity} = \text{Average Capacity in lbs of washer}$$

$$= 254.38^{242}$$

$$\Delta \text{kWh}_{\text{PUMP}} / \text{lb capacity} = 746 / 254.38$$

$$= 2.93 \text{ kWh/lb-capacity}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

Past project documentation and data collection is not sufficient to determine summer coincident peak demand savings for this measure. Value should continue to be studied and monitored through additional studies due to limited data points used for this determination. In absence of site-specific data, the summer coincident peak demand savings should be assumed to be zero.

$$\Delta \text{kW} = 0$$

NATURAL GAS SAVINGS

$$\Delta \text{Therm} = \text{Therm}_{\text{Baseline}} * \% \text{hot_water_savings}$$

Where:

ΔTherm = Gas savings resulting from a reduction in hot water use, in therm.
 $\text{Therm}_{\text{Baseline}}$ = Annual Baseline Gas Consumption

²⁴⁰ Engineered estimate provided by CLEAResult review of Nicor custom projects. Machines spent approximately 7 minutes per hour filling with water and were in operation approximately 20 hours per day. Total pump time therefore estimated as $7/60 * 20 * 365 = 852$ hours, and rounded down conservatively to 800 hours.

²⁴¹ Average water reduction factors were generated using data collected from existing ozone laundry projects that received incentives under the Non-Residential Retrofit Demand Reduction program (NRR-DR). Table 6 summarizes data gathered from several NRR-DR projects, Nicor Custom projects, and Nicor ETP projects. Nicor Savings Numbers are associated with ACEE_AWE_Ozone Laundry / From Gas Savings Calculations

²⁴² Average lbs-capacity per project site was generated using data collected from existing ozone laundry projects that received incentives under the Non-Residential Retrofit Demand Reduction program (NRR-DR), as well as from the Nicor Custom Incentive Program, and the Nicor Emerging Technology Program (ETP). See referenced document Table 2

$$= \text{WHE} * \text{WUtiliz} * \text{WUsage_hot}$$

Where:

WHE = water heating energy: energy required to heat the hot water used
 = 0.00885 therm/gallon²⁴³

WUtiliz = washer utilization factor: the annual pounds of clothes washed per year
 = actual, if unknown use 916,150 lbs laundry²⁴⁴, approximately equivalent to 13 cycles/day

WUsage_hot = hot water usage factor: how much hot water a typical conventional washing machine utilizes, normalized per pounds of clothes washed
 = 1.19 gallons/lbs laundry²⁴⁵

Using defaults above:

$$\begin{aligned} \text{Therm}_{\text{Baseline}} &= 0.00885 * 916,150 * 1.19 \\ &= 9,648 \text{ therms} \end{aligned}$$

Default per lb capacity:

$$\begin{aligned} \text{Therm}_{\text{Baseline}} / \text{lb capacity} &= 9,648 / 254.38 \\ &= 37.9 \text{ therms / lb-capacity} \end{aligned}$$

%hot_water_savings = hot water reduction factor: how much more efficient an ozone injection washing machine is, compared to a typical conventional washing machine, as a rate of hot water reduction
 = 81%²⁴⁶

Savings using defaults above:

$$\begin{aligned} \Delta \text{Therm} &= \text{Therm}_{\text{Baseline}} * \% \text{hot_water_savings} \\ &= 9648 * 0.81 \\ &= 7,815 \text{ therms} \end{aligned}$$

Default per lb capacity:

$$\begin{aligned} \Delta \text{Therm} / \text{lb-capacity} &= 7815 / 254.38 \\ &= 30.7 \text{ therms / lb-capacity} \end{aligned}$$

²⁴³ Assuming boiler efficiency is the regulated minimum efficiency (80%), per Title 20 Appliance Standard of the California Energy Regulations (October 2007). The incoming municipal water temperature is assumed to be 55 °F with an average hot water supply temperature of 140°F, based on default test procedures on clothes washers set by the Department of Energy's Office of Energy Efficiency and Renewable Energy (Federal Register, Vol. 52, No. 166). Enthalpies for these temperatures (107 btu/lbs at 140F, 23.07 btu/lbs at 55F) were obtained from ASHRAE Fundamentals

²⁴⁴ Average utilization factors were generated using data collected from existing ozone laundry projects that received incentives under the NRR-DR program. Table 3 summarizes data gathered from several NRR-DR projects, Nicor Custom projects, and Nicor ETP projects

²⁴⁵ Average hot water usage factors were generated using data collected from existing ozone laundry projects that received incentives under the NRR-DR program. summarizes data gathered from several NRR-DR projects:

²⁴⁶ Average hot water reduction factors were generated using data collected from existing ozone laundry projects that received incentives under the Non-Residential Retrofit Demand Reduction program (NRR-DR). Table 5 summarizes data gathered from several NRR-DR projects, Nicor Custom projects, and Nicor ETP projects. Nicor Savings Numbers are associated with ACEE_AWE_Ozone Laundry / From Gas Savings Calculations

WATER IMPACT DESCRIPTIONS AND CALCULATION

The water savings calculations listed here account for the combination of hot and cold water used. Savings calculations for this measure were based on the reduction in total water use from implementing an ozone washing system to the base case. There are three main components in obtaining this value:

$$\Delta \text{gallons} = W\text{Usage} * W\text{Utiliz} * \% \text{water_savings}$$

Where:

- $\Delta \text{gallons}$ = reduction in total water use from implementing an ozone washing system to the base case
- $W\text{Usage}$ = water usage factor: how efficiently a typical conventional washing machine utilized hot and cold water normalized per unit of clothes washed
= 2.03 gallons/lbs laundry²⁴⁷
- $W\text{Utiliz}$ = washer utilization factor: the annual pounds of clothes washed per year
= actual, if unknown use 916,150 lbs laundry²⁴⁸, approximately equivalent to 13 cycles/day
- $\% \text{water_savings}$ = water reduction factor: how much more efficient an ozone injection washing machine is compared to a typical conventional washing machine as a rate of hot and cold water reduction.
= 25%²⁴⁹

Savings using defaults above:

$$\begin{aligned}\Delta \text{Gallons} &= W\text{Usage} * W\text{Utiliz} * \% \text{water_savings} \\ &= 2.03 * 916,150 * 0.25 \\ &= 464,946 \text{ gallons}\end{aligned}$$

Default per lb capacity:

$$\begin{aligned}\Delta \text{ Gallons / lb-capacity} &= 464,946 / 254.38 \\ &= 1,828 \text{ gallons / lb-capacity}\end{aligned}$$

DEEMED O&M COST ADJUSTMENT CALCULATION

Maintenance is required for the following components annually:²⁵⁰

- Ozone Generator: filter replacement, check valve replacement, fuse replacement, reaction chamber inspection/cleaning, reaction chamber o-ring replacement
- Air Preparation – Heat Regenerative: replacement of two medias
- Air Preparation – Oxygen Concentrators: filter replacement, pressure relief valve replacement, compressor rebuild
- Venturi Injector: check valve replacement

²⁴⁷ Average water usage factors were generated using data collected from existing ozone laundry projects that received incentives under the NRR-DR program. summarizes data gathered from several NRR-DR projects

²⁴⁸ Average utilization factors were generated using data collected from existing ozone laundry projects that received incentives under the NRR-DR program. Table 3 summarizes data gathered from several NRR-DR projects, Nicor Custom projects, and Nicor ETP projects

²⁴⁹ Average water reduction factors were generated using data collected from existing ozone laundry projects that received incentives under the Non-Residential Retrofit Demand Reduction program (NRR-DR). Table 6 summarizes data gathered from several NRR-DR projects, Nicor Custom projects, and Nicor ETP projects. Nicor Savings Numbers are associated with ACEE_AWE_Ozone Laundry / From Gas Savings Calculations

²⁵⁰ Confirmed through communications with national vendors and available references E.g.
<http://ozonelaundry.wordpress.com/2010/11/17/the-importance-of-maintenance/>

Maintenance is expected to cost \$0.79 / lbs capacity.

REFERENCES

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- 6 The Benefits of Ozone in Hospitality On-Premise Laundry Operations, PG&E Emerging Technologies Program,
Application Assessment Report #0802, April 2009.
- 7 Federal Register, Vol. 52, No. 166
- 8 2009 ASHRAE Handbook – Fundamentals, Thermodynamic Properties of Water at Saturation, Section 1.1 (Table
3), 2009
- 9 Table 2 through 6: Excel file summarizing data collected from existing ozone laundry projects that received
incentives under the NRR-DR program

MEASURE CODE CI-HWE-OZLD-VO1-140601

REVIEW DEADLINE: 1/1/2020

4.3.7 Multifamily Central Domestic Hot Water Plants

DESCRIPTION

This measure covers multifamily central domestic hot water (DHW) plants with thermal efficiencies greater than or equal to 88%. This measure is applicable to any combination of boilers and storage tanks provided the thermal efficiency of the boilers is greater than 88%. Plants providing other than solely DHW are not applicable to this measure.

This measure was developed to be applicable to the following program types: TOS, NC, ER.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify the boiler(s) must have a Thermal Efficiency of 88% or greater and supply domestic hot water to multi-family buildings.

DEFINITION OF BASELINE EQUIPMENT

For TOS the baseline boiler is assumed to have a Thermal Efficiency of 80%.²⁵¹

For Early Replacement the savings are calculated between existing unit and efficient unit consumption during the remaining life of the existing unit, and between new baseline unit as above and efficient unit consumption for the remainder of the measure life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for the domestic hot water boilers is 15 years.²⁵²

DEEMED MEASURE COST

TOS: The actual install cost should be used for the efficient case, minus the baseline cost assumption provided below:

Capacity Range	Baseline Installed Cost per kBtuh ²⁵³
<300kBtuh	\$65 per kBTU/h
300 – 2500 kBtuh	\$38 per kBTU/h
>2500 kBtuh	\$32 per kBTU/h

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

²⁵¹ International Energy Conservation Code (IECC) 2012/2015, Table C404.2, Minimum Performance of Water-Heating Equipment

²⁵² Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011.

²⁵³ Baseline install costs are based on data from the W017 Itron California Measure Cost Study, accessed via <http://www.energydataweb.com/cpuc/search.aspx>. The data is provided in a file named "MCS Results Matrix – Volume I".

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

There are no anticipated electrical savings from this measure.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS SAVINGS

Time of Sale:

$$\begin{aligned} \Delta \text{Therms} &= \text{Hot Water Savings} + \text{Standby Loss Savings} \\ &= [(MFHH * \#Units * GPD * Days/yr * \nu \text{Water} * (T_{out} - T_{in}) * (1/Eff_{base} - 1/Eff_{ee})) / 100,000] + [((SL * Hours/yr * (1/Eff_{base} - 1/Eff_{ee})) / 100,000] \end{aligned}$$

Early Replacment²⁵⁴:

$$\begin{aligned} \Delta \text{Therms for remaining life of existing unit (1st 5 years):} \\ &= [(MFHH * \#Units * GPD * Days/yr * \nu \text{Water} * (T_{out} - T_{in}) * (1/Eff_{exist} - 1/Eff_{ee})) / 100,000] + [((SL * Hours/yr * (1/Eff_{exist} - 1/Eff_{ee})) / 100,000] \end{aligned}$$

$$\begin{aligned} \Delta \text{Therms for remaining measure life (next 10 years):} \\ &= [(MFHH * \#Units * GPD * Days/yr * \nu \text{Water} * (T_{out} - T_{in}) * (1/Eff_{base} - 1/Eff_{ee})) / 100,000] + [((SL * Hours/yr * (1/Eff_{base} - 1/Eff_{ee})) / 100,000] \end{aligned}$$

Where:

MFHH	= number of people in Multi-Family House Hold = Actual. If unknown assume 2.1 persons/unit ²⁵⁵
#Units	= Number of units served by hot water boiler = Actual
GPD	= Gallons of hot water used per person per day = Actual. If unknown assume 17.6 gallons per person per day ²⁵⁶
Days/yr	= 365.25
νWater	= Specific Weight of Water = 8.33 gal/lb
T _{out}	= tank temperature of hot water = 125°F or custom
T _{in}	= Incoming water temperature from well or municiple system

²⁵⁴ The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a “number of years to adjustment” and “savings adjustment” input which would be the (new base to efficient savings)/(existing to efficient savings).

²⁵⁵ Navigant, ComEd PY3 Multi-Family Home Energy Savings Program Evaluation Report Final, May 16, 2012.

²⁵⁶ Deoreo, B., and P. Mayer. Residential End Uses of Water Study Update. Forthcoming. ©2015 Water Research Foundation. Reprinted With Permission.

	= 54°F ²⁵⁷	
Eff_base	= thermal efficiency of base unit	
	= 80% ²⁵⁸	
Eff_ee	= thermal efficiency of efficient unit complying with this measure	
	= Actual. If unknown assume 88%	
Eff_exist	= thermal efficiency of existing unit	
	= Actual. If unknown assume 73% ²⁵⁹	
SL	= Standby Loss ²⁶⁰	
	= (Input rating / 800) + (110 * vTank Volume)	
	Input rating	= Name plate input capacity in Btuh
	Tank Volume	= Rated volume of the tank in gallons
Hours / yr	= 8766 hours	
100,000	= btu/therm	

²⁵⁷ US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL
http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html

²⁵⁸ IECC 2012/2015, Table C404.2, Minimum Performance of Water-Heating Equipment

²⁵⁹ Based upon DCEO data provided 10/2014; average age adjusted efficiency of existing units replaced through the program. Efficiency age adjustment of 0.5% per year based upon NREL “Building America Performance Analysis Procedures for Existing Homes”.

²⁶⁰ Stand-by loss is provided in IECC 2012/2015, Table C404.2, Minimum Performance of Water-Heating Equipment

EXAMPLES

Time of Sale:

For example, an 88% 1000 gallon boiler with 150,000 Btuh input rating installed serving 50 units.

$$\begin{aligned}
 \Delta \text{Therms} &= \text{Hot Water Savings} + \text{Standby Loss Savings} \\
 &= [(MFHH * \#Units * GPD * Days/yr * \nu \text{Water} * (T_{out} - T_{in}) * (1/Eff_{base} - 1/Eff_{ee})) / 100,000] + [(SL * Hours/yr * (1/Eff_{base} - 1/Eff_{ee})) / 100,000] \\
 &= [(2.1 * 50 * 17.6 * 8.33 * 365.25 * 1.0 * (125-54) * (1/0.8 - 1/0.88)) / 100000] + \\
 &\quad [((150000/800 + (110 * \nu 1000)) * 8766 * (1/0.8 - 1/0.88)) / 100000] \\
 &= 454 + 37 \\
 &= 490 \text{ therms}
 \end{aligned}$$

Early Replacement:

For example, an 88% 1000 gallon boiler with 150,000 Btuh input rating installed serving 50 units replaces a working unit with unknown efficiency.

$$\begin{aligned}
 \Delta \text{Therms for remaining life of existing unit (1st 5 years):} \\
 &= [(2.1 * 50 * 17.6 * 8.33 * 365.25 * 1.0 * (125-54) * (1/0.73 - 1/0.88)) / 100000] + \\
 &\quad [((150000/800 + (110 * \nu 1000)) * 8766 * (1/0.73 - 1/0.88)) / 100000] \\
 &= 932 + 75 \\
 &= 1007 \text{ therms} \\
 \Delta \text{Therms for remaining measure life (next 10 years):} \\
 &= 454 + 37 \text{ (as above)} \\
 &= 490 \text{ therms}
 \end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HWE-MDHW-V02-160601**REVIEW DEADLINE: 1/1/2023**

4.3.8 Controls for Central Domestic Hot Water

DESCRIPTION

Demand control recirculation pumps seek to reduce inefficiency by combining control via temperature and demand inputs, whereby the controller will not activate the recirculation pump unless both (a) the recirculation loop return water has dropped below a prescribed temperature (e.g. 100°F) and (b) a CDHW demand is sensed as water flow through the CDHW system.

This measure was developed to be applicable to the following program types: TOS, RF, NC. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

Re-circulating pump shall cycle on based on (a) the recirculation loop return water dropping below a prescribed temperature (e.g. 100°F) and (b) a CDHW demand is sensed as water flow through the CDHW system.

DEFINITION OF BASELINE EQUIPMENT

The base case for this measure category are existing, un-controlled Recirculation Pumps on gas-fired Central Domestic Hot Water Systems.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The effective useful life is 15 years²⁶¹.

DEEMED MEASURE COST

The average cost of the demand controller circulation kit is \$1,608 with an installation cost of \$400 for a total measure cost of \$2,008.²⁶²

LOADSHAPE

Loadshape C02 - Non-Residential Electric DHW

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

Deemed at 656 kWh²⁶³.

²⁶¹ Benningfield Group. (2009). *PY 2009 Monitoring Report: Demand Control for Multifamily Central Domestic Hot Water*. Folsom, CA: Prepared for Southern California Gas Company, October 30, 2009.

²⁶² The incremental costs were averaged based on the following multi-family and dormitory building studies-

- Gas Technology Institute. (2014). *1003: Demand-based domestic hot water recirculation Public project report*. Des Plaines, IL: Prepared for Nicor Gas, January 7, 2014.
- Studies performed in multiple dormitory buildings in the California region for Southern California Gas' PREPS Program, 2012.

²⁶³ This value is the average kWh saved per pump based on results from Multi-Family buildings studied in Nicor Gas Emerging Technology Program study and Southern California Gas' study in multiple dormitory buildings. Note this value does not reflect

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS SAVINGS

Gas savings for this measure can be calculated by using site specific boiler size and boiler usage information or deemed values are provided based on number of rooms for Dormitories and number of apartments for Multi-Family buildings²⁶⁴.

$$\Delta \text{Therms} = \text{Boiler Input Capacity} * (t_{\text{normal occ}} * R_{\text{normal occ}} + t_{\text{low occ}} * R_{\text{low occ}}) / 100,000$$

Where:

Boiler Input Capacity	<p>= Input capacity of the Domestic Hot Water boiler in BTU/hr.</p> <p>= If the facility uses the same boiler for space heat and domestic hot water, estimate the boiler input capacity for only domestic hot water loads. If this cannot be estimated, use 22.75%²⁶⁵ of total boiler input capacity for Multi-Family Buildings and 16.48%²⁶⁶ of total boiler input capacity for Dormitories, as domestic hot water load.</p> <p>= If unknown capacity use 4,938 BTU/hr per room for Dormitories²⁶⁷ and 12,493 BTU/hr per apartment for Multi-Family Buildings²⁶⁸</p>
$t_{\text{normal occ}}$	= Total operating hours of domestic hot water burner, when the facility has normal occupancy. If unknown, assume 1,688 hours for Dormitories ²⁶⁹ and 2,089 hours for Multi-Family buildings ²⁷⁰ .
$t_{\text{low occ}}$	= Total operating hours of domestic hot water burner, when the facility has low occupancy ²⁷¹ . If unknown, assume 520 hours for Dormitories and 0 hours for Multi-Family buildings.

savings from electric units but electrical savings from gas-fired units. See 'CDHW Controls Summary Calculations.xlsx' for more information.

²⁶⁴ See 'CDHW Controls Summary Calculations.xlsx' for more information.

²⁶⁵ This is an average number based on Residential Energy Consumption Survey (2009) data and Commercial Building Energy Consumption Survey (2012) data compiled by U.S. Energy Information Administration, for buildings with more than 5 apartments in Illinois and Nursing Home and Assisted Living facilities in Midwest.

²⁶⁶ This is based on Commercial Building Energy Consumption Survey (2012) data compiled by U.S. Energy Information Administration, for Education facilities in East North Central.

²⁶⁷ This is based on studies done in multiple university dormitory buildings in the California region, for Southern California Gas' PREPS Program, 2012. It closely matches the design guidelines outlined in 2007 ASHRAE Handbook, Chapter 49: Service Water Heating, Table 7, and assumes 1 to 2 students per dorm room based on typical dorm room layouts. This source provides the source for dormitory assumptions of Boiler Input Capacity, $t_{\text{low occ}}$, $R_{\text{normal occ}}$ and $R_{\text{low occ}}$.

²⁶⁸ This is based on studies done at Multi-Family Buildings for the Nicor Gas Emerging Technology Program by Gas Technology Institute. It closely matches the design guidelines outlined in 2007 ASHRAE Handbook, Chapter 49: Service Water Heating, Table 9, and assumes 2.1 persons per apartment as per ComEd PY3 Multi-Family Home Energy Savings Program Evaluation Report Final, May 16, 2012 by Navigant. This source provides the source for dormitory assumptions of Boiler Input Capacity, $t_{\text{low occ}}$, $R_{\text{normal occ}}$ and $R_{\text{low occ}}$.

²⁶⁹ Based on results of studies performed in multiple university dormitory buildings in the California region, for Southern California Gas' PREPS Program, 2012.

²⁷⁰ Based on results of the studies done at Multi-Family Buildings for the Nicor Gas Emerging Technology Program:

- Gas Technology Institute. (2014). *1003: Demand-based domestic hot water recirculation Public project report*. Des Plaines, IL: Prepared for Nicor Gas, January 7, 2014.

²⁷¹ Low occupancy periods for dormitory buildings can be assumed as vacation day or holiday occupancy.

$R_{\text{normal occ}}$	<p>= Reduction(%) in total operating hours of domestic hot water burner, due to installed central domestic hot water controls, during normal occupancy period.</p> <p>= 22.44% for Dormitories</p> <p>= 24.02% for Multi-Family Buildings</p>
$R_{\text{low occ}}$	<p>= Reduction(%) in total operating hours of domestic hot water burner, due to installed central domestic hot water controls, during low occupancy period.</p> <p>= 44.57% for Dormitories</p> <p>= 0% for Multi-Family Buildings</p>

Based on defaults above:

$$\Delta \text{Therms} = 30.1 * \text{number of rooms (for Dormitories)}$$

$$= 62.7 * \text{number of apartments (for Multi-Family buildings)}$$

EXAMPLE

For example, a dormitory building has a 400,000 BTU/hr boiler whose burner operates for an estimated 580 hours during vacation months and 1,300 hours during regular occupancy months. Savings from installing central domestic hot water controls in this building are -

$$\Delta \text{Therms} = 400,000 \text{ BTU/hr} * (1,300 * 0.2244 + 580 * 0.4457) / 100,000$$

$$= 2,200.9 \text{ therms}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HWE-CDHW-V02-180101

REVIEW DEADLINE: 1/1/2022

4.3.9 Heat Recovery Grease Trap Filter

DESCRIPTION

A heat recovery grease trap filter combines grease filters and a heat exchanger to recover heat leaving kitchen hoods. As a direct replacement for conventional hood mounted filters in commercial kitchens, they are plumbed to the domestic hot water system to provide preheating energy to incoming water.

This measure was developed to be applicable to the following program types: TOS and RF. If applied to other program types, the measure savings should be verified. For NC projects, this measure may be applicable if code requirements are otherwise satisfied.

DEFINITION OF EFFICIENT EQUIPMENT

Grease filters with heat exchangers carrying domestic hot water in kitchen exhaust air ducts.

DEFINITION OF BASELINE EQUIPMENT

Kitchen exhaust air duct with constant air flow²⁷² and no heat recovery.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

THE EXPECTED MEASURE LIFE IS ASSUMED TO BE 15 YEARS.²⁷³

DEEMED MEASURE COST

FULL INSTALLATION COSTS, INCLUDING PLUMBING MATERIALS, LABOR AND ANY ASSOCIATED CONTROLS, SHOULD BE USED FOR SCREENING PURPOSES.

LOADSHAPE

Loadshape C01 - Commercial Electric Cooking

COINCIDENCE FACTOR

Summer Peak Coincidence Factor for measure is provided below for different building type²⁷⁴:

Location	CF
Fast Food Limited Menu	0.32
Fast Food Expanded Menu	0.41
Pizza	0.46
Full Service Limited Menu	0.51
Full Service Expanded Menu	0.36
Cafeteria	0.36
Unknown	0.40

Algorithm

²⁷² Savings methodology factors are for a constant speed fan.

²⁷³ Professional judgement, consistent with expected lifetime of kitchen demand ventilation controls and other kitchen equipment.

²⁷⁴ Minnesota 2012 Technical Reference Manual, Electric Food Service_v03.2.xls,
<http://mn.gov/commerce/energy/topics/conservation/Design-Resources/Deemed-Savings.jspech>

CALCULATION OF ENERGY SAVINGS**ELECTRIC ENERGY SAVINGS**

For electric hot water heaters:

$$\Delta kWh = \frac{[(Meal/Day * HW/Meal * Days/Year) * lbs/gal * BTU/lb.^{\circ}F * (\Delta T/filter * Qty_Filter) * 0.00293]}{(\eta_{HeaterElec})}$$

Where:

Meal/Day	= Average number of meals served per day. If not directly available, see Table 1.
HW/Meal	= Hot water required per meal = 3 gal/meal ²⁷⁵
Days/Year	= Number of days kitchen operates per year. If not directly available, see Table 1.
Lbs/gal	= weight of water = 8.3 lbs/gal
BTU/lb.°F	= Specific heat of water = 1.0
ΔT/filter	= Temperature difference of domestic water across each filter = 5.8°F/filter ²⁷⁶
Qty_Filter	= Number of heat recovery grease trap filters installed. If not directly available, see Table 1.

Commercial Kitchen Load based on Building Type

Building Type	Meals/Day ²⁷⁷	Assumed days/Year	Number of Filters ²⁷⁸
Primary School	400	312	2
Secondary School	600	312	3
Quick Service Restaurant	800	312	5
Full Service Restaurant	780	312	4
Large Hotel	780	356	4
Hospital	800	356	4

$\eta_{HeaterElec}$	= Efficiency of the Electric water heater. = Actual. If unknown, use the table C404.2 in IECC 2012 (or IECC 2015 if through new construction) to assume values based on code estimates
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²⁷⁵ Average dishwashing and faucet water usage taken from Chapter 8, Table 8.3.3 Normalized Annual End Uses of Water in Select Restaurants in Western United States.

²⁷⁶ Average value based on case studies. Northwinds Sailing, Inc. and North Shore Sustainable Energy, LLC. *Angry Trout Café Kitchen Exhaust Heat Recovery*. Minnesota Department of Commerce, Division of Energy Resources, 2012.

²⁷⁷ Commercial Kitchen Loads for listed buildings in U.S. Department of Energy Commercial Reference Building Models of the National Building Stock, NREL

²⁷⁸ Each filter is 20 X 20 inches.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh / \text{Hours} * CF$$

Where:

Hours = Hours of operation of kitchen exhaust air fan. If not directly available use:

Building Type	Kitchen Exhaust Fan Annual Operating Hours ²⁷⁹
Primary School	4,056
Secondary School	4,056
Quick Service Restaurant	5,616
Full Service Restaurant	5,616
Large Hotel	5,340
Hospital	3,916

CF = Summer Peak Coincidence Factor for measure²⁸⁰:

Location	CF
Fast Food Limited Menu	0.32
Fast Food Expanded Menu	0.41
Pizza	0.46
Full Service Limited Menu	0.51
Full Service Expanded Menu	0.36
Cafeteria	0.36
Unknown	0.40

NATURAL GAS SAVINGS

For natural gas hot water heaters:

$$\Delta \text{Therm} = \frac{[(\text{Meal/Day} * \text{HW/Meal} * \text{Days/Year}) * \text{lbs/gal} * \text{BTU/lb} * ^\circ\text{F} * (\Delta T/\text{filter} * \text{Qty_Filter})]}{(\eta_{\text{HeaterGas}} * 100,000)}$$

Where:

$\eta_{\text{HeaterGas}}$

= Efficiency of the Gas water heater. If not directly available, use:

= Actual. If unknown, use the table C404.2 in IECC 2012 (or IECC 2015 if through new construction) to assume values based on code estimates

Other variables as above

WATER AND OTHER NON-ENERGY IMPACT DESCRIPTIONS AND CALCULATION

N/A

²⁷⁹ Exhaust Fan Schedules for listed buildings in U.S. Department of Energy Commercial Reference Building Models of the National Building Stock, NREL

²⁸⁰ Minnesota 2012 Technical Reference Manual, [Electric Food Service v03.2.xls](http://mn.gov/commerce/energy/topics/conservation/Design-Resources/Deemed-Savings.jspech), <http://mn.gov/commerce/energy/topics/conservation/Design-Resources/Deemed-Savings.jspech>

DEEMED O&M COST ADJUSTMENT CALCULATION

O&M savings may result from reduced filter and hood cleaning frequencies. More research should be done to understand any potential savings and the associated value.

MEASURE CODE: CI-HWE-GRTF-V01-160601

REVIEW DEADLINE: 1/1/2024

4.3.10 DHW Boiler Tune-up

DESCRIPTION

Domestic hot water (DHW) boilers provide hot water for bathrooms, kitchens, tubs and other appliances. Several commercial and industrial facilities such as multi-family buildings, lodging and restaurants have a separate hot water boiler serving DHW loads. Unlike space heating boilers, DHW boilers operate year round, which means they have a greater need to be properly maintained and tuned up.

This measure calculates savings for tuning up a DHW boiler to improve its efficiency and reduce its consumption. A boiler tune-up involves cleaning/inspecting burners, burner nozzles and combustion chambers, adjusting air flow and burner gas input to reduce stack temperatures, and checking venting and safety controls. A pre- and post- tune up combustion efficiency ticket (from combustion analyzer) can be used to confirm the improvement in boiler efficiency.

Boilers that serve only a DHW load are eligible for this measure.

This measure was developed to be applicable to the following program types: RF. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the facility must, as applicable, complete the tune-up requirements²⁸¹ listed below, by approved technician:

- Measure combustion efficiency using an electronic flue gas analyzer
- Adjust airflow and reduce excessive stack temperatures
- Adjust burner and gas input, manual or motorized draft control
- Check for proper venting
- Complete visual inspection of system piping and insulation
- Check safety controls
- Check adequacy of combustion air intake
- Clean fireside surfaces.
- Inspect all refractory. Patch and wash coat as required.
- Inspect gaskets on front and rear doors and replace as necessary.
- Seal and close front and rear doors properly.
- Clean low and auxiliary low water cut-off controls, then re-install using new gaskets.
- Clean plugs in control piping.
- Remove all hand hole and man hole plates. Flush boiler with water to remove loose scale and sediment.
- Replace all hand hole and man hole plates with new gaskets.
- Open feedwater tank manway, inspect and clean as required. Replace manway plate with new gasket.
- Clean burner and burner pilot.
- Check pilot electrode and adjust or replace.
- Clean air damper and blower assembly.
- Clean motor starter contacts and check operation.
- Make necessary adjustments to burner for proper combustion.
- Perform all flame safeguard and safety trip checks.
- Check all hand hole plates and man hole plates for leaks at normal operating temperatures and pressures.
- Troubleshoot any boiler system problems as requested by on-site personnel

DEFINITION OF BASELINE EQUIPMENT

The baseline condition of this measure is a boiler that has not had a tune-up within the past 36 months.

²⁸¹ Act on Energy Commercial Technical Reference Manual No. 2010-4, 9.2.2 Gas Boiler Tune-up

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The life of this measure is 3 years.²⁸²

DEEMED MEASURE COST

The cost of this measure is \$0.83/MBtu/hr per tune-up.²⁸³

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS SAVINGS

$$\Delta \text{Therms} = ((T_{\text{out}} - T_{\text{in}}) * \text{HotWaterUse}_{\text{Gallon}} * \gamma_{\text{water}} * 1 * (1/\text{Eff}_{\text{before}} - 1/\text{Eff}_{\text{after}}))/100,000$$

Where:

- | | |
|--------------------------------------|---|
| T_{OUT} | = Hot water storage tank temperature
= 125°F |
| T_{IN} | = Incoming water temperature from well or municipal system
= 54°F ²⁸⁴ |
| $\text{HotWaterUse}_{\text{Gallon}}$ | = Estimated annual hot water consumption (gallons)
= Actual if possible to provide reasonable custom estimate. If not, the following methods are provided to develop an estimate ²⁸⁵ :
1. Consumption per usable storage tank capacity |

²⁸² Act on Energy Commercial Technical Reference Manual No. 2010-4, 9.2.2 Gas Boiler Tune-up

²⁸³ Work Paper – Tune up for Boilers serving Space Heating and Process Load by Resource Solutions Group, January 2012

²⁸⁴ US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL
http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html

²⁸⁵ Methodology based on Cadmus analysis. Annual hot water usage in gallons based on CBECS (2012) and RECS (2009) consumption data of East North Central (removed outliers of 1,000 kBtu/h or less) to calculate hot water usage. Annual hot water gallons per tank size gallons based on the tank sizing methodology found in ASHRAE 2011 HVAC Applications. Chapter 50 Service Water Heating. Demand assumptions (gallons per day) for each building type based on ASHRAE Chapter 50 and to LBNL White Paper. LBL-37398 Technology Data Characterizing Water Heating in Commercial Buildings: Application to End Use Forecasting. Assumes hot water heater efficiency of 80%.

$$= \text{Capacity} * \text{Consumption/cap}$$

Where:

Capacity = Usable capacity of hot water storage tank in gallons
= Actual

Consumption/cap = Estimate of consumption per gallon of usable tank capacity, based on building type:

Building Type ²⁸⁶	Consumption/Cap
Convenience	528
Education	568
Grocery	528
Health	788
Large Office	511
Large Retail	528
Lodging	715
Other Commercial	341
Restaurant	622
Small Office	511
Small Retail	528
Warehouse	341
Nursing	672
Multi-Family	894

2. Consumption per unit area by building type

$$= (\text{Area}/1000) * \text{Consumption}/1,000 \text{ sq.ft.}$$

Where:

Area = Area in sq.ft that is served by DHW boiler
= Actual

Consumption/1,000 sq.ft. = Estimate of DHW consumption per 1,000 sq.ft. based on building type:

Building Type	Consumption/1,000 sq.ft.
Convenience	4,594
Education	7,285
Grocery	697
Health	24,540
Large Office	1,818
Large Retail	1,354
Lodging	29,548
Other Commercial	3,941
Restaurant	44,439
Small Office	1,540
Small Retail	6,111
Warehouse	1,239

²⁸⁶ According to CBECS 2012 “Lodging” buildings include Dormitories, Hotels, Motel or Inns and other Lodging and “Nursing” buildings include Assisted Living and Nursing Homes.

Building Type	Consumption/1,000 sq.ft.
Nursing	30,503
Multi-Family	15,434

γ_{water} = Specific weight capacity of water (lb/gal)

= 8.33 lbs/gal

1 = Specific heat of water (Btu/lb.°F)

$\text{Eff}_{\text{before}}$ = Efficiency of the boiler before tune-up

$\text{Eff}_{\text{after}}$ = Efficiency of the boiler after tune-up

100,000 = Converts Btu to therms

Note: Contractors should select a mid-level firing rate that appropriately represents the average building operating condition over the course of the year and take readings at a consistent firing rate for pre and post tune-up.

EXAMPLE

Tune up of a DHW Boiler heating a 100 gallon storage tank in a nursing home, measuring 80% AFUE prior to tune up and 82.2% AFUE after.

$$\begin{aligned}\Delta\text{Therms} &= ((T_{\text{out}} - T_{\text{in}}) * \text{HotWaterUse}_{\text{Gallon}} * \gamma_{\text{water}} * 1 * (1/\text{Eff}_{\text{before}} - 1/\text{Eff}_{\text{after}}))/100,000 \\ &= ((125 - 54) * (100 * 672) * 8.33 * 1 * (1/0.8 - 1/0.822))/100,000 \\ &= 13.3 \text{ therms}\end{aligned}$$

WATER AND OTHER NON-ENERGY IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HWE-DBTU-V01-180101

REVIEW DEADLINE: 1/1/2024

4.4 HVAC End Use

Many of the commercial HVAC measures use equivalent full load hours (EFLH) to calculate heating and cooling savings. The tables with these values are included in this section and referenced in each measure.

To calculate the updated EFLHs by building type and climate zone provided below, a TAC Subcommittee utilized building energy models originally developed for ComEd²⁸⁷, applying some adjustments and additions for new building type models and mechanical systems. Based on comparisons with available field data from Navigant²⁸⁸, the EFLH calculation was finalized by the Subcommittee to be the annual total (heating or cooling) output (in Btu) divided by the 95th percentile hourly peak output (heating or cooling) demand (in Btu/hr). This calculation keeps EFLH independent of modeled systems efficiency (which is utilized in the TRM savings calculation) and buffers EFLH value from hourly variances in the modeling that are not representative of actual buildings. See “EFLH Description 2015-02-11.doc” for further explanation.

The building characteristics can be found in the reference table named “EFLH Building Descriptions Updated 2014-11-21.xlsx”.

Note where a measure installation is within a building or application that does not fit with any of the defined building types below, the user should apply custom assumptions where it is reasonable to estimate them, else the building of best fit should be utilized.

Building Type	Heating EFLH				
	Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
Assembly	1,787	1,831	1,635	1,089	1,669
Assisted Living	1,683	1,646	1,446	1,063	1,277
College	1,530	1,430	1,276	709	849
Convenience Store	1,481	1,368	1,214	871	973
Elementary School	1,781	1,736	1,531	1,057	1,283
Garage	985	969	852	680	752
Grocery	1,608	1,602	1,404	876	1,047
Healthcare Clinic	1,579	1,620	1,414	963	1,019
High School	1,845	1,857	1,666	1,187	1,388
Hospital - CAV no econ ²⁸⁹	1,764	1,818	1,549	1,332	1,512
Hospital - CAV econ ²⁹⁰	1,788	1,853	1,580	1,369	1,555
Hospital - VAV econ ²⁹¹	731	695	522	314	340
Hospital - FCU	1,325	1,512	1,232	1,448	1,946
Hotel/Motel	1,761	1,712	1,544	1,056	1,290
Hotel/Motel - Common	1,601	1,626	1,548	1,260	1,323
Hotel/Motel - Guest	1,758	1,702	1,521	1,018	1,252
Manufacturing Facility	1,048	1,013	939	567	634
MF - High Rise	1,526	1,506	1,373	1,169	1,172
MF - High Rise - Common	1,815	1,762	1,580	1,089	1,406
MF - High Rise - Residential	1,475	1,464	1,330	1,152	1,123
MF - Mid Rise	1,666	1,685	1,450	1,067	1,216
Movie Theater	1,916	1,905	1,718	1,288	1,538

²⁸⁷ A full description of the ComEd model development is found in “ComEd Portfolio Modeling Report. Energy Center of Wisconsin July 30, 2010”

²⁸⁸ <http://www.icc.illinois.gov/downloads/public/edocket/397867.pdf>

²⁸⁹ Based on model with single duct reheat system with a fixed outdoor air volume.

²⁹⁰ Based on model with single duct reheat system with airside economizer controls, with constant volume zone reheat boxes and single speed fan motors.

²⁹¹ Based on model with single duct reheat system with airside economizer controls, zone VAV reheat boxes and VFD fan motors.

Building Type	Heating EFLH				
	Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
Office - High Rise - CAV no econ	2,020	2,050	1,869	1,252	1,363
Office - High Rise - CAV econ	2,089	2,132	1,960	1,351	1,487
Office - High Rise - VAV econ	1,528	1,558	1,284	759	846
Office - High Rise - FCU	1,118	1,102	952	505	530
Office - Low Rise	1,428	1,425	1,132	692	793
Office - Mid Rise	1,585	1,587	1,342	855	950
Religious Building	1,603	1,504	1,440	1,054	1,205
Restaurant	1,350	1,354	1,216	920	1,091
Retail - Department Store	1,392	1,278	1,200	781	891
Retail - Strip Mall	1,332	1,233	1,090	751	810
Warehouse	1,456	1,357	1,400	875	1,078
Unknown	1,553	1,539	1,369	982	1,139

Equivalent Full Load Hours for Cooling (EFLH_{cooling}) :

Building Type	Cooling EFLH				
	Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
Assembly	725	796	937	1,183	932
Assisted Living	1,475	1,457	1,773	2,110	1,811
College	475	481	662	746	806
Convenience Store	1,088	1,067	1,368	1,541	1,371
Elementary School	725	764	905	1,142	956
Garage	934	974	1,226	1,582	1,383
Grocery	1,033	1,000	1,236	1,499	1,286
Healthcare Clinic	1,282	1,305	1,519	1,767	1,571
High School	675	721	840	1,060	920
Hospital - CAV no econ	4,166	4,275	4,319	4,692	4,445
Hospital - CAV econ	1,751	1,814	2,120	2,411	2,112
Hospital - VAV econ	1,531	1,592	1,853	2,163	1,876
Hospital - FCU	3,245	3,291	3,451	4,128	3,806
Hotel/Motel	1,233	1,186	1,436	1,274	1,616
Hotel/Motel - Common	2,186	2,103	2,344	1,391	2,651
Hotel/Motel - Guest	1,042	1,019	1,269	1,216	1,418
Manufacturing Facility	1,010	1,055	1,209	1,453	1,273
MF - High Rise	921	845	1,048	1,779	1,099
MF - High Rise - Common	914	839	1,055	2,893	1,132
MF - High Rise - Residential	899	831	1,011	1,569	1,055
MF - Mid Rise	809	767	992	1,119	993
Movie Theater	876	745	1,036	1,178	1,010
Office - High Rise - CAV no econ	1,688	1,708	1,811	1,865	1,725
Office - High Rise - CAV econ	1,454	1,452	1,551	1,568	1,416
Office - High Rise - VAV econ	875	919	1,057	1,275	1,077
Office - High Rise - FCU	1,117	1,170	1,277	1,642	1,412
Office - Low Rise	949	1,010	1,182	1,452	1,281
Office - Mid Rise	883	938	1,072	1,286	1,083
Religious Building	861	817	967	1,159	1,067

Building Type	Cooling EFLH				
	Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
Restaurant	1,074	1,134	1,279	1,627	1,325
Retail - Department Store	949	889	1,124	1,367	1,157
Retail - Strip Mall	950	919	1,149	1,351	1,215
Warehouse	357	338	422	647	533
Unknown	1,215	1,221	1,408	1,670	1,480

4.4.1 Air Conditioner Tune-up

DESCRIPTION

An air conditioning system that is operating as designed saves energy and provides adequate cooling and comfort to the conditioned space

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a unitary or split system air conditioner least 3 tons and preapproved by program. The measure requires that a certified technician performs the following items:

- Check refrigerant charge
- Identify and repair leaks if refrigerant charge is low
- Measure and record refrigerant pressures
- Measure and record temperature drop at indoor coil
- Clean condensate drain line
- Clean outdoor coil and straighten fins
- Clean indoor and outdoor fan blades
- Clean indoor coil with spray-on cleaner and straighten fins
- Repair damaged insulation – suction line
- Change air filter
- Measure and record blower amp draw

A copy of contractor invoices that detail the work performed to identify tune-up items, as well as additional labor and parts to improve/repair air conditioner performance must be submitted to the program

DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline condition is assumed to be an AC system that that does not have a standing maintenance contract or a tune up within in the past 36 months.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 3 years.²⁹²

DEEMED MEASURE COST

The incremental capital cost for this measure is \$35²⁹³ per ton.

LOADSHAPE

Loadshape C03 - Commercial Cooling

COINCIDENCE FACTOR

$$\begin{aligned} CF_{SSP} &= \text{Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)} \\ &= 91.3\% \text{ }^{294} \end{aligned}$$

²⁹²3 years is given for “Clean Condenser Coils – Commercial” and “Clean Evaporator Coils”. DEER2014 EUL Table.

²⁹³http://www.deeresources.com/files/DEER2013codeUpdate/download/DEER2014-EUL-table-update_2014-02-05.xlsx

²⁹⁴Act on Energy Commercial Technical Reference Manual No. 2010-4

²⁹⁴ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility’s peak hour is divided by the maximum AC load during the year.

$$\begin{aligned} CF_{PJM} &= \text{PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)} \\ &= 47.8\%^{295} \end{aligned}$$

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = (\text{kBtu/hr}) * [(1/\text{EER}_{\text{before}}) - (1/\text{EER}_{\text{after}})] * \text{EFLH}$$

Where:

kBtu/hr	= capacity of the cooling equipment actually installed in kBtu per hour (1 ton of cooling capacity equals 12 kBtu/hr). =Actual
EER _{before}	= Energy Efficiency Ratio ²⁹⁶ of the baseline equipment prior to tune-up =Actual
EER _{after}	= Energy Efficiency Ratio of the baseline equipment after to tune-up =Actual
EFLH	= Equivalent Full Load Hours for cooling are provided in section 4.4 HVAC End Use

Where it is not possible or appropriate to perform Test in and Test out of the equipment, the following deemed methodology can be used:

$$\Delta \text{kWh} = (\text{kBtu/hr}) / \text{EER}_{\text{before}} * \text{EFLH} * (1 - \% \text{Savings})$$

Where:

%Savings	= Deemed percent savings per Tune-Up component. These are additive if condenser cleaning, evaporator cleaning and refrigerant charge correction are performed (totals provided below) ²⁹⁷
----------	--

Tune-Up Component	% savings
Condenser Cleaning	6.10%
Evaporator Cleaning	0.22%
Refrig. Charge Off. <=20%	0.68%

²⁹⁵Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

²⁹⁶ In the context of this measure Energy Efficiency Ratio (EER) refers to field-measured steady-state rate of heat energy removal (e.g., cooling capacity) by the equipment in Btuh divided by the steady-state rate of energy input to the equipment in watts. This ratio is expressed in Btuh per watt (Btuh/watt). The cooling capacity may be derived using either refrigerant or air-side measurements. The measurement is performed at the outdoor and indoor environmental conditions that are present at the time the tune-up is being performed, and should be normalized using a correction function to the AHRI 210/240 Standard test conditions. The correction function should be developed based on manufacturer's performance data. Care must be taken to ensure the unit is fully loaded and operating at or near steady-state. Generally, this requires that the outside air temperature is at least 60°F, and that the unit runs with all stages of cooling enabled for 10 to 15 minutes prior to making measurements. For more information, please see "IL TRM_Normalizing to AHRI Conditions Method".

²⁹⁷ Savings estimates are determined by applying the findings from DNV-GL "[Impact Evaluation of 2013-2014 HVAC3 Commercial Quality Maintenance Programs](#)", April 2016, to simulate the inefficient condition within select eQuest models and across climate zones. The percent savings were consistent enough across building types and climate zones that it was determined appropriate to apply a single set of assumptions for all. See 'eQuest C&I Tune up Analysis.xlsx' for more information.

Tune-Up Component	% savings
Refrig. Charge Off. >20%	8.44%
Combined (Refrig. Charge Off. ≤20%)	7.00%
Combined (Refrig. Charge Off. >20%)	14.76%

For example, a 12 EER 5-ton rooftop air conditioner on a department store in Rockford receives a tune-up that includes both condenser and evaporator cleaning:

$$\begin{aligned}\Delta \text{kWh} &= (5 \times 12) / 12 \times 1,392 \times 6.32\% \\ &= 440 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW}_{\text{SSP}} = (\text{kBtu/hr} \times (1/\text{EER}_{\text{before}} - 1/\text{EER}_{\text{after}})) \times \text{CF}_{\text{SSP}}$$

$$\Delta \text{kW}_{\text{PJM}} = (\text{kBtu/hr} \times (1/\text{EER}_{\text{before}} - 1/\text{EER}_{\text{after}})) \times \text{CF}_{\text{PJM}}$$

Where:

$$\begin{aligned}\text{CF}_{\text{SSP}} &= \text{Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)} \\ &= 91.3\%^{298}\end{aligned}$$

$$\begin{aligned}\text{CF}_{\text{PJM}} &= \text{PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)} \\ &= 47.8\%^{299}\end{aligned}$$

Where it is not possible or appropriate to perform Test in and Test out of the equipment, the following deemed methodology can be used:

$$\Delta \text{kW} = (\text{kBtu/hr}) / \text{EER}_{\text{before}} \times \% \text{Savings} \times \text{CF}$$

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-ACTU-V04-180101

REVIEW DEADLINE: 1/1/2021

²⁹⁸ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

²⁹⁹ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

4.4.2 Space Heating Boiler Tune-up

DESCRIPTION

This measure is for a non-residential boiler that provides space heating. The tune-up will improve boiler efficiency by cleaning and/or inspecting burners, combustion chamber, and burner nozzles. Adjust air flow and reduce excessive stack temperatures, adjust burner and gas input. Check venting, safety controls, and adequacy of combustion air intake. Combustion efficiency should be measured before and after tune-up using an electronic flue gas analyzer.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the facility must, as applicable, complete the tune-up requirements³⁰⁰ listed below, by approved technician:

- Measure combustion efficiency using an electronic flue gas analyzer
- Adjust airflow and reduce excessive stack temperatures
- Adjust burner and gas input, manual or motorized draft control
- Check for proper venting
- Complete visual inspection of system piping and insulation
- Check safety controls
- Check adequacy of combustion air intake
- Clean fireside surfaces.
- Inspect all refractory. Patch and wash coat as required.
- Inspect gaskets on front and rear doors and replace as necessary.
- Seal and close front and rear doors properly.
- Clean low and auxiliary low water cut-off controls, then re-install using new gaskets.
- Clean plugs in control piping.
- Remove all hand hole and man hole plates. Flush boiler with water to remove loose scale and sediment.
- Replace all hand hole and man hole plates with new gaskets.
- Open feedwater tank manway, inspect and clean as required. Replace manway plate with new gasket.
- Clean burner and burner pilot.
- Check pilot electrode and adjust or replace.
- Clean air damper and blower assembly.
- Clean motor starter contacts and check operation.
- Make necessary adjustments to burner for proper combustion.
- Perform all flame safeguard and safety trip checks.
- Check all hand hole plates and man hole plates for leaks at normal operating temperatures and pressures.
- Troubleshoot any boiler system problems as requested by on-site personnel

DEFINITION OF BASELINE EQUIPMENT

The baseline condition of this measure is a boiler that has not had a tune-up within the past 36 months

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The life of this measure is 3 years³⁰¹

DEEMED MEASURE COST

The cost of this measure is \$0.83/MBtu/hr³⁰² per tune-up

³⁰⁰ Act on Energy Commercial Technical Reference Manual No. 2010-4, 9.2.2 Gas Boiler Tune-up

³⁰¹ Act on Energy Commercial Technical Reference Manual No. 2010-4, 9.2.2 Gas Boiler Tune-up

³⁰² Work Paper – Tune up for Boilers serving Space Heating and Process Load by Resource Solutions Group, January 2012

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms} = (\text{Capacity} * \text{EFLH} * (((\text{Effbefore} + \text{Ei}) / \text{Effbefore}) - 1)) / 100,000$$

Where:

Capacity = Boiler gas input size (Btu/hr)
= custom

EFLH = Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use

Effbefore = Efficiency of the boiler before the tune-up

Note: Contractors should select a mid-level firing rate that appropriately represents the average building operating condition over the course of the heating season and take readings at a consistent firing rate for pre and post tune-up.

Ei = Efficiency Improvement of the boiler tune-up measure

100,000 = Converts Btu to therms

EXAMPLE

For example, a 1050 kBtu boiler in a Chicago high rise office records an efficiency prior to tune up of 82% AFUE and a 1.8% improvement in efficiency are tune up:

$$\begin{aligned} \Delta \text{therms} &= (1,050,000 * 2050 * ((0.82 + 0.018) / 0.82 - 1)) / 100,000 \\ &= 473 \text{ Therms} \end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-BLRT-V06-160601

REVIEW DEADLINE: 1/1/2022

4.4.3 Process Boiler Tune-up

DESCRIPTION

This measure is for a non-residential boiler for process loads. For space heating, see measure 4.4.2. The tune-up will improve boiler efficiency by cleaning and/or inspecting burners, combustion chamber, and burner nozzles. Adjust air flow and reduce excessive stack temperatures, adjust burner and gas input. Check venting, safety controls, and adequacy of combustion air intake. Combustion efficiency should be measured before and after tune-up using an electronic flue gas analyzer.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the facility must, as applicable, complete the tune-up requirements³⁰³ by approved technician, as specified below:

- Measure combustion efficiency using an electronic flue gas analyzer
- Adjust airflow and reduce excessive stack temperatures
- Adjust burner and gas input, manual or motorized draft control
- Check for proper venting
- Complete visual inspection of system piping and insulation
- Check safety controls
- Check adequacy of combustion air intake
- Clean fireside surfaces
- Inspect all refractory. Patch and wash coat as required.
- Inspect gaskets on front and rear doors and replace as necessary.
- Seal and close front and rear doors properly.
- Clean low and auxiliary low water cut-off controls, then re-install using new gaskets.
- Clean plugs in control piping.
- Remove all hand hole and man hole plates. Flush boiler with water to remove loose scale and sediment.
- Replace all hand hole and man hole plates with new gaskets.
- Open feedwater tank manway, inspect and clean as required. Replace manway plate with new gasket.
- Clean burner and burner pilot.
- Check pilot electrode and adjust or replace.
- Clean air damper and blower assembly.
- Clean motor starter contacts and check operation.
- Make necessary adjustments to burner for proper combustion.
- Perform all flame safeguard and safety trip checks.
- Check all hand hole plates and man hole plates for leaks at normal operating temperatures and pressures.
- Troubleshoot any boiler system problems as requested by on-site personnel

DEFINITION OF BASELINE EQUIPMENT

The baseline condition of this measure is a boiler that has not had a tune-up within the past 36 months

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The life of this measure is 3 years³⁰⁴

DEEMED MEASURE COST

The cost of this measure is \$0.83/MBtu/hr³⁰⁵ per tune-up

³⁰³ Act on Energy Commercial Technical Reference Manual No. 2010-4, 9.2.2 Gas Boiler Tune-up

³⁰⁴ Act on Energy Commercial Technical Reference Manual No. 2010-4, 9.2.2 Gas Boiler Tune-up

³⁰⁵ Work Paper – Tune up for Boilers serving Space Heating and Process Load by Resource Solutions Group, January 2012

DEEMED O&M COST ADJUSTMENTS

N/A

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms} = ((\text{Ngi} * 8766 * \text{UF}) / 100) * (1 - (\text{Eff}_{\text{pre}} / \text{Eff}_{\text{measured}}))$$

Where:

Ngi = Boiler gas input size (kBtu/hr)
= custom

UF = Utilization Factor
= 41.9%³⁰⁶ or custom

Eff_{pre} = Boiler Combustion Efficiency Before Tune-Up
= Actual

Note: Contractors should select a mid-level firing rate that appropriately represents the average building operating condition over the course of the heating season and take readings at a consistent firing rate for pre and post tune-up.

Eff_{measured} = Boiler Combustion Efficiency After Tune-Up
= Actual

100 = conversion from kBtu to therms

8766 = hours a year

³⁰⁶ Work Paper – Tune up for Boilers serving Space Heating and Process Load by Resource Solutions Group, January 2012

EXAMPLE

For example, a 80% 1050 kBtu boiler is tuned-up resulting in final efficiency of 81.3%:

$$\begin{aligned}\Delta\text{therms} &= ((1050 * 8766 * 0.419) / 100) * (1 - (0.80 / 0.813)) \\ &= 617 \text{ therms}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-PBTU-V05-160601

REVIEW DEADLINE: 1/1/2022

4.4.4 Boiler Lockout/Reset Controls

DESCRIPTION

This measure relates to improving combustion efficiency by adding controls to non-residential building heating boilers to vary the boiler entering water temperature relative to heating load as a function of the outdoor air temperature to save energy. Energy is saved by increasing the temperature difference between the water temperature entering the boiler in the boiler's heat exchanger and the boiler's burner flame temperature. The flame temperature remains the same while the water temperature leaving the boiler decreases with the decrease in heating load due to an increase in outside air temperature. A lockout temperature is also set to prevent the boiler from turning on when it is above a certain temperature outdoors.

This measure was developed to be applicable to the following program types: RF. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

Natural gas customer adding boiler reset controls capable of resetting the boiler supply water temperature in an inverse linear fashion with outdoor air temperature. Boiler lockout temperatures should be set to 55 °F at this time as well, to turn the boiler off when the temperature goes above a certain setpoint.

DEFINITION OF BASELINE EQUIPMENT

Existing boiler without boiler reset controls, any size with constant hot water flow.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The life of this measure is 20 years³⁰⁷

DEEMED MEASURE COST

The cost of this measure is \$612³⁰⁸

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

³⁰⁷CLEAResultreferences the Brooklyn Union Gas Company, High Efficiency Heating and Water and Controls, Gas Energy Efficiency Program Implementation Plan.

³⁰⁸ Nexant. Questar DSM Market Characterization Report. August 9, 2006.

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms} = \text{Binput} * \text{SF} * \text{EFLH} / (100)$$

Where:

Binput = Boiler Input Capacity (kBtu/hr)

= custom

SF = Savings factor

= 8%³⁰⁹ or custom

EFLH = Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use

100 = conversion from kBtu to therms

EXAMPLE

For example, a 800 kBtu/hr boiler at a restaurant in Rockford, IL

$$\Delta \text{Therms} = 800 * 0.08 * 1,350 / (100)$$

$$= 864 \text{ Therms}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-BLRC-V03-150601

REVIEW DEADLINE: 1/1/2021

³⁰⁹ Savings factor is the estimate of annual gas consumption that is saved due to adding boiler reset controls. The CLEAResult uses a boiler tuneup savings value derived from Xcel Energy "DSM Biennial Plan-Technical Assumptions," Colorado. Focus on Energy uses 8%, citing multiple sources. Vermont Energy Investment Corporation's boiler reset savings estimates for custom projects further indicate 8% savings estimate is better reflection of actual expected savings.

4.4.5 Condensing Unit Heaters

DESCRIPTION

This measure applies to a gas fired condensing unit heater installed in a commercial application.

This measure was developed to be applicable to the following program types: TOS, NC. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a condensing unit heater up to 300 MBH with a Thermal Efficiency > 90% and the heater must be vented, and condensate drained per manufacturer specifications. The unit must be replacing existing natural gas equipment.

DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline condition is assumed to be a non-condensing natural gas unit heater at end of life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years³¹⁰

DEEMED MEASURE COST

The incremental capital cost for a unit heater is \$676³¹¹

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

The annual natural gas energy savings from this measure is a deemed value equaling 266 Therms.

³¹⁰DEER 2008

³¹¹ENERGY STAR and CEE do not currently provide calculators for this type of equipment therefore deemed values from Nicor Gas were used. Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-CUHT-V01-120601

REVIEW DEADLINE: 1/1/2019

4.4.6 Electric Chiller

DESCRIPTION

This measure relates to the installation of a new electric chiller meeting the efficiency standards presented below. This measure could relate to the replacement of an existing unit at the end of its useful life, or the installation of a new system in an existing building (i.e. time of sale). Only single-chiller applications should be assessed with this methodology. The characterization is not suited for multiple chillers projects or chillers equipped with variable speed drives (VSDs).

This measure was developed to be applicable to the following program types: TOS, NC. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to exceed the efficiency requirements defined by the program.

DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline equipment is assumed to meet the efficiency requirements within Table 403.2.3(7) of either the 2012 or the 2015 IECC (applicable from 01/01/2016), depending on the IECC in effect on the date of the building permit (if unknown assume IECC 2015).

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 20 years³¹².

DEEMED MEASURE COST

The incremental capital cost for this measure is provided below.

Equipment Type	Size Category	Incremental Cost (\$/ton)
Air cooled, electrically operated	All capacities	\$127/ton ³¹³
Water cooled, electrically operated, positive displacement (reciprocating)	All capacities	\$22/ton ³¹⁴
Water cooled, electrically operated, positive displacement (rotary screw and scroll)	< 150 tons	\$351/ton ³¹⁵
	>= 150 tons and < 300 tons	\$127/ton
	>= 300 tons	\$87/ton

LOADSHAPE

Loadshape C03 - Commercial Cooling

³¹² 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, "Effective/Remaining Useful Life Values", California Public Utilities Commission, December 16, 2008 (http://deerresources.com/deer0911planning/downloads/EUL_Summary_10-1-08.xls)

³¹³ 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, "Cost Values and Summary Documentation", California Public Utilities Commission, December 16, 2008. Calculated as the simple average of screw and reciprocating air-cooled chiller incremental costs from DEER2008. This assumes that baseline shift from IECC 2012 to IECC 2015 carries the same incremental costs. Values should be verified during evaluation

³¹⁴ 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, "Cost Values and Summary Documentation"

³¹⁵ Incremental costs for water-cooled, electrically operated, positive displacement (rotary screw and scroll) from the W017 Itron California Measure Cost Study, accessed via <http://www.energydataweb.com/cpuc/search.aspx>. The data is provided in a file named "MCS Results Matrix – Volume I".

COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market. Both values provided are based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren.

CF_{SSP} = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)
= 91.3%³¹⁶

CF_{PJM} = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)
= 47.8%³¹⁷

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$\Delta kWH = TONS * ((IPLV_{base}) - (IPLV_{ee})) * EFLH$

Where:

$TONS$ = chiller nominal cooling capacity in tons (note: 1 ton = 12,000 Btu/hr)
= Actual installed

$IPLV_{base}$ = efficiency of baseline equipment expressed as Integrated Part Load Value(kW/ton). Chiller units are dependent on chiller type. See Chiller Units, Conversion Values and Baseline Efficiency Values by Chiller Type and Capacity in the Reference Tables section.

$IPLV_{ee}$ ³¹⁸ = efficiency of high efficiency equipment expressed as Integrated Part Load Value (kW/ton)³¹⁹
= Actual installed

$EFLH$ = Equivalent Full Load Hours for cooling are provided in section 4.4 HVAC End Use.

For example, a 100 ton air-cooled electrically operated chiller with IPLV of 14 EER (0.86 kW/ton) and baseline EER of 12.5 (0.96 kW/ton) ,in a low-rise office building in Rockford with a building permit dated on 1/1/2015 would save:

$$\begin{aligned}\Delta kWH &= 100 * ((0.96) - (0.86)) * 949 \\ &= 9,490 \text{ kWh}\end{aligned}$$

³¹⁶ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

³¹⁷ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

³¹⁸ Integrated Part Load Value is a seasonal average efficiency rating calculated in accordance with ARI Standard 550/590. It may be calculated using any measure of efficiency (EER, kW/ton, COP), but for consistency with IECC 2012, it is expressed in terms of IPLV here.

³¹⁹ Can determine IPLV from standard testing or looking at engineering specs for design conditions. Standard data is available from AHRnetl.org. <http://www.ahrinet.org/>

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW_{SSP} = \text{TONS} * ((PE_{base}) - (PE_{ee})) * CF_{SSP}$$

$$\Delta kW_{PJM} = \text{TONS} * ((PE_{base}) - (PE_{ee})) * CF_{PJM}$$

Where:

PE_{base} = Peak efficiency of baseline equipment expressed as Full Load (kW/ton)

PE_{ee} = Peak efficiency of high efficiency equipment expressed as Full Load (kW/ton)

= Actual installed

CF_{SSP} = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)

= 91.3%

CF_{PJM} = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)

= 47.8%

For example, a 100 ton air-cooled electrically operated chiller with a peak efficiency of 1.05 kW/ton and a baseline peak efficiency of 1.2 kW/ton would save:

$$\Delta kW_{SSP} = 100 * (1.2 - 1.05) * 0.913$$

$$= 13.7 \text{ kW}$$

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

REFERENCE TABLES

Chillers Ratings- Chillers are rated with different units depending on equipment type as shown below

Equipment Type	Unit
Air cooled, electrically operated	EER
Water cooled, electrically operated, positive displacement (reciprocating)	kW/ton
Water cooled, electrically operated, positive displacement (rotary screw and scroll)	kW/ton

In order to convert chiller equipment ratings to IPLV the following relationships are provided

$$\text{kW/ton} = 12 / \text{EER}$$

$$\text{kW/ton} = 12 / (\text{COP} \times 3.412)$$

$$\text{COP} = \text{EER} / 3.412$$

$$\text{COP} = 12 / (\text{kW/ton}) / 3.412$$

$$\text{EER} = 12 / \text{kW/ton}$$

$$\text{EER} = \text{COP} \times 3.412$$

2012 IECC Baseline Efficiency Values by Chiller Type and Capacity

TABLE C403.2.3(7)
MINIMUM EFFICIENCY REQUIREMENTS:
WATER CHILLING PACKAGES*

EQUIPMENT TYPE	SIZE CATEGORY	UNITS	BEFORE 1/1/2010		AS OF 1/1/2010 ^b				TEST PROCEDURE
			FULL LOAD	IPLV	PATH A		PATH B		
					FULL LOAD	IPLV	FULL LOAD	IPLV	
Air-cooled chillers	< 150 tons	EER	≥ 9.562	≥ 10.4 16	≥ 9.562	≥ 12.500	NA	NA	AHRI 550/590
	≥ 150 tons	EER			≥ 9.562	≥ 12.750	NA	NA	
Air cooled without condenser, electrical operated	All capacities	EER	≥ 10.586	≥ 11.782	Air-cooled chillers without condensers shall be rated with matching condensers and comply with the air-cooled chiller efficiency requirements				
Water cooled, electrically operated, reciprocating	All capacities	kW/ton	≤ 0.837	≤ 0.696	Reciprocating units shall comply with water cooled positive displacement efficiency requirements				
Water cooled, electrically operated, positive displacement	< 75 tons	kW/ton	≤ 0.790	≤ 0.676	≤ 0.780	≤ 0.630	≤ 0.800	≤ 0.600	
	≥ 75 tons and < 150 tons	kW/ton			≤ 0.775	≤ 0.615	≤ 0.790	≤ 0.586	
	≥ 150 tons and < 300 tons	kW/ton	≤ 0.717	≤ 0.627	≤ 0.680	≤ 0.580	≤ 0.718	≤ 0.540	
	≥ 300 tons	kW/ton	≤ 0.639	≤ 0.571	≤ 0.620	≤ 0.540	≤ 0.639	≤ 0.490	
Water cooled, electrically operated, centrifugal	< 150 tons	kW/ton	≤ 0.703	≤ 0.669	≤ 0.634	≤ 0.596	≤ 0.639	≤ 0.450	
	≥ 150 tons and < 300 tons	kW/ton	≤ 0.634	≤ 0.596					
	≥ 300 tons and < 600 tons	kW/ton	≤ 0.576	≤ 0.549	≤ 0.576	≤ 0.549	≤ 0.600	≤ 0.400	
	≥ 600 tons	kW/ton	≤ 0.576	≤ 0.549	≤ 0.570	≤ 0.539	≤ 0.590	≤ 0.400	
Air cooled, absorption single effect	All capacities	COP	≥ 0.600	NR	≥ 0.600	NR	NA	NA	AHRI 560
Water cooled, absorption single effect	All capacities	COP	≥ 0.700	NR	≥ 0.700	NR	NA	NA	
Absorption double effect, indirect fired	All capacities	COP	≥ 1.000	≥ 1.050	≥ 1.000	≥ 1.050	NA	NA	
Absorption double effect, direct fired	All capacities	COP	≥ 1.000	≥ 1.000	≥ 1.000	≥ 1.000	NA	NA	

For SI: 1 ton = 3517 W, 1 British thermal unit per hour = 0.2931 W, °C = [(°F) - 32]/1.8.

NA = Not applicable, not to be used for compliance; NR = No requirement.

- The centrifugal chiller equipment requirements, after adjustment in accordance with Section C403.2.3.1 or Section C403.2.3.2, do not apply to chillers used in low-temperature applications where the design leaving fluid temperature is less than 36°F. The requirements do not apply to positive displacement chillers with leaving fluid temperatures less than or equal to 32°F. The requirements do not apply to absorption chillers with design leaving fluid temperatures less than 40°F.
- Compliance with this standard can be obtained by meeting the minimum requirements of Path A or B. However, both the full load and IPLV shall be met to fulfill the requirements of Path A or B.
- Chapter 6 of the referenced standard contains a complete specification of the referenced test procedure, including the referenced year version of the test procedure.

2015 IECC Baseline Efficiency Values by Chiller Type and Capacity

TABLE C403.2.3(7)
WATER CHILLING PACKAGES – EFFICIENCY REQUIREMENTS^{a, b, d}

WATER CHILLING PACKAGES – EFFICIENCY REQUIREMENTS							
EQUIPMENT TYPE	SIZE CATEGORY	UNITS	BEFORE 1/1/2015		AS OF 1/1/2015		TEST PROCEDURE
			Path A	Path B	Path A	Path B	
Air-cooled chillers	< 150 Tons	EER (Btu/W)	≥ 9.562 FL	NA ^c	≥ 10.100 FL	≥ 9.700 FL	AHRI 550/ 590
			≥ 12.500 IPLV		≥ 13.700 IPLV	≥ 15.800 IPLV	
	≥ 150 Tons		≥ 9.562 FL	NA ^c	≥ 10.100 FL	≥ 9.700 FL	
			≥ 12.500 IPLV		≥ 14.000 IPLV	≥ 16.100 IPLV	
Air cooled without condenser, electrically operated	All capacities	EER (Btu/W)	Air-cooled chillers without condenser shall be rated with matching condensers and complying with air-cooled chiller efficiency requirements.				
Water cooled, electrically operated positive displacement	< 75 Tons	kW/ton	≤ 0.780 FL	≤ 0.800 FL	≤ 0.750 FL	≤ 0.780 FL	
	≥ 75 tons and < 150 tons		≤ 0.630 IPLV	≤ 0.600 IPLV	≤ 0.600 IPLV	≤ 0.500 IPLV	
			≤ 0.775 FL	≤ 0.790 FL	≤ 0.720 FL	≤ 0.750 FL	
	≥ 150 tons and < 300 tons		≤ 0.615 IPLV	≤ 0.586 IPLV	≤ 0.560 IPLV	≤ 0.490 IPLV	
			≤ 0.680 FL	≤ 0.718 FL	≤ 0.660 FL	≤ 0.680 FL	
	≥ 300 tons and < 600 tons		≤ 0.580 IPLV	≤ 0.540 IPLV	≤ 0.540 IPLV	≤ 0.440 IPLV	
			≤ 0.620 FL	≤ 0.639 FL	≤ 0.610 FL	≤ 0.625 FL	
	≥ 600 tons		≤ 0.540 IPLV	≤ 0.490 IPLV	≤ 0.520 IPLV	≤ 0.410 IPLV	
			≤ 0.620 FL	≤ 0.639 FL	≤ 0.560 FL	≤ 0.585 FL	
	≤ 0.540 IPLV		≤ 0.490 IPLV	≤ 0.500 IPLV	≤ 0.380 IPLV		
Water cooled, electrically operated centrifugal	< 150 Tons	kW/ton	≤ 0.634 FL	≤ 0.639 FL	≤ 0.610 FL	≤ 0.695 FL	
	≥ 150 tons and < 300 tons		≤ 0.596 IPLV	≤ 0.450 IPLV	≤ 0.550 IPLV	≤ 0.440 IPLV	
			≤ 0.634 FL	≤ 0.639 FL	≤ 0.610 FL	≤ 0.635 FL	
	≥ 300 tons and < 400 tons		≤ 0.596 IPLV	≤ 0.450 IPLV	≤ 0.550 IPLV	≤ 0.400 IPLV	
			≤ 0.576 FL	≤ 0.600 FL	≤ 0.560 FL	≤ 0.595 FL	
	≥ 400 tons and < 600 tons		≤ 0.549 IPLV	≤ 0.400 IPLV	≤ 0.520 IPLV	≤ 0.390 IPLV	
			≤ 0.576 FL	≤ 0.600 FL	≤ 0.560 FL	≤ 0.585 FL	
	≥ 600 Tons		≤ 0.549 IPLV	≤ 0.400 IPLV	≤ 0.500 IPLV	≤ 0.380 IPLV	
			≤ 0.570 FL	≤ 0.590 FL	≤ 0.560 FL	≤ 0.585 FL	
	≤ 0.539 IPLV		≤ 0.400 IPLV	≤ 0.500 IPLV	≤ 0.380 IPLV		
Air cooled, absorption, single effect	All capacities	COP	≥ 0.600 FL	NA ^c	≥ 0.600 FL	NA ^c	AHRI 560
Water cooled absorption, single effect	All capacities	COP	≥ 0.700 FL	NA ^c	≥ 0.700 FL	NA ^c	
Absorption, double effect, indirect fired	All capacities	COP	≥ 1.000 FL	NA ^c	≥ 1.000 FL	NA ^c	
			≥ 1.050 IPLV		≥ 1.050 IPLV		
Absorption double effect direct fired	All capacities	COP	≥ 1.000 FL	NA ^c	≥ 1.000 FL	NA ^c	
			≥ 1.000 IPLV		≥ 1.050 IPLV		

a. The requirements for centrifugal chiller shall be adjusted for nonstandard rating conditions in accordance with Section C403.2.3.1 and are only applicable for the range of conditions listed in Section C403.2.3.1. The requirements for air-cooled, water-cooled positive displacement and absorption chillers are at standard rating conditions defined in the reference test procedure.

b. Both the full-load and IPLV requirements shall be met or exceeded to comply with this standard. Where there is a Path B, compliance can be with either Path A or Path B for any application.

c. NA means the requirements are not applicable for Path B and only Path A can be used for compliance.

d. FL represents the full-load performance requirements and IPLV the part-load performance requirements.

MEASURE CODE: CI-HVC-CHIL-V05-180101

REVIEW DEADLINE: 1/1/2022

4.4.7 ENERGY STAR and CEE Tier 1 Room Air Conditioner

DESCRIPTION

This measure relates to the purchase and installation of a room air conditioning unit that meets either the ENERGY STAR or CEE TIER 1 minimum qualifying efficiency specifications, in place of a baseline unit meeting minimum Federal Standard efficiency ratings presented below:³²⁰

Product Class (Btu/H)	Federal Standard EER, with louvered sides	Federal Standard EER, without louvered sides	ENERGY STAR EER, with louvered sides	ENERGY STAR EER, without louvered sides	CEE TIER 1 EER
< 8,000	9.7	9	10.7	9.9	11.2
8,000 to 13,999	9.8	8.5	10.8	9.4	11.3
14,000 to 19,999	9.7	8.5	10.7	9.4	11.2
>= 20,000	8.5	8.5	9.4	9.4	9.8

Casement	Federal Standard (EER)	ENERGY STAR (EER)
Casement-only	8.7	9.6
Casement-slider	9.5	10.5

Reverse Cycle - Product Class (Btu/H)	Federal Standard EER, with louvered sides	Federal Standard EER, without louvered sides	ENERGY STAR EER, with louvered sides	ENERGY STAR EER, without louvered sides
< 14,000	N/A	8.5	N/A	9.4
>= 14,000	N/A	8	N/A	8.8
< 20,000	9	N/A	9.9	N/A
>= 20,000	8.5	N/A	9.4	N/A

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the new room air conditioning unit must meet the ENERGY STAR efficiency standards presented above.

³²⁰ http://www.energystar.gov/index.cfm?c=roomac.pr_crit_room_ac and http://www.cee1.org/resid/seha/rm-ac/rm-ac_specs.pdf

Side louvers that extend from a room air conditioner model in order to position the unit in a window. A model without louvered sides is placed in a built-in wall sleeve and are commonly referred to as "through-the-wall" or "built-in" models.

Casement-only refers to a room air conditioner designed for mounting in a casement window of a specific size.

Casement-slider refers to a room air conditioner with an encased assembly designed for mounting in a sliding or casement window of a specific size.

Reverse cycle refers to the heating function found in certain room air conditioner models.

http://www.energystar.gov/ia/partners/product_specs/program_reqs/room_air_conditioners_prog_req.pdf

DEFINITION OF BASELINE EQUIPMENT

The baseline assumption is a new room air conditioning unit that meets the current minimum federal efficiency standards presented above.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 9 years.³²¹

DEEMED MEASURE COST

The incremental cost for this measure is assumed to be \$40 for an ENERGY STAR unit and \$80 for a CEE TIER 1 unit.³²²

LOADSHAPE

Loadshape C03 - Commercial Cooling

COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market. Both values provided are based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren.

CF_{SSP} = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)
= 91.3%³²³

CF_{PJM} = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)
= 47.8%³²⁴

Algorithm

CALCULATION OF SAVINGS**ENERGY SAVINGS**

$$\Delta kWh = (FLH_{RoomAC} * Btu/H * (1/EER_{base} - 1/EER_{ee}))/1000$$

Where:

FLH_{RoomAC} = Full Load Hours of room air conditioning unit
= dependent on location:³²⁵

³²¹ Energy Star Room Air Conditioner Savings Calculator,

http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=AC

http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf

³²² Based on field study conducted by Efficiency Vermont

³²³ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

³²⁴ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

³²⁵ Full load hours for room AC is significantly lower than for central AC. The average ratio of FLH for Room AC (provided in RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008:

http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RAC.pdf) to FLH for Central Cooling for the same location (provided by AHRI:

Zone	FLH _{RoomAC}
1 (Rockford)	253
2-(Chicago)	254
3 (Springfield)	310
4-(Belleville)	391
5-(Marion)	254

Btu/H = Size of unit

= Actual. If unknown assume 8500 Btu/hr ³²⁶

EER_{base} = Efficiency of baseline unit

= As provided in tables above

EER_{ee} = Efficiency of ENERGY STAR or CEE Tier 1 unit

= Actual. If unknown assume minimum qualifying standard as provided in tables above

For example for an 8,500 Btu/H capacity ENERGY STAR unit, with louvered sides, in Rockford:

$$\begin{aligned}\Delta \text{kWh}_{\text{ENERGY STAR}} &= (253 * 8500 * (1/9.8 - 1/10.8)) / 1000 \\ &= 20.3 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \text{Btu/H} * ((1/\text{EER}_{\text{base}} - 1/\text{EER}_{\text{ee}}))/1000 * \text{CF}$$

Where:

CF_{SSP} = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)
= 91.3% ³²⁷

CF_{PJM} = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)
= 47.8% ³²⁸

Other variable as defined above

For example for an 8,500 Btu/H capacity ENERGY STAR unit, with louvered sides, in Rockford during system peak

$$\begin{aligned}\Delta \text{kW}_{\text{ENERGY STAR}} &= (8500 * (1/9.8 - 1/10.8)) / 1000 * 0.913 \\ &= 0.073 \text{ kW}\end{aligned}$$

FOSSIL FUEL SAVINGS

N/A

http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls is 31%. This ratio has been applied to the FLH from the unitary and split system air conditioning measure.

³²⁶ Based on maximum capacity average from the RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008

³²⁷ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

³²⁸ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-ESRA-V01-120601

REVIEW DEADLINE: 1/1/2019

4.4.8 Guest Room Energy Management (PTAC & PTHP)

DESCRIPTION

This measure applied to the installation of a temperature setback and lighting control system for individual guest rooms. The savings are achieved based on Guest Room Energy Management's (GREM's) ability to automatically adjust the guest room's set temperatures and control the HVAC unit for various occupancy modes.

This measure was developed to be applicable to the following program types: TOS, NC. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

Guest room temperature set point must be controlled by automatic occupancy detectors or keycard that indicates the occupancy status of the room. During unoccupied periods the default setting for controlled units differs by at least 5 degrees from the operating set point. Theoretically, the control system may also be tied into other electric loads, such as lighting and plug loads to shut them off when occupancy is not sensed. This measure bases savings on improved HVAC controls. If system is connected to lighting and plug loads, additional savings would be realized. The incentive is per guestroom controlled, rather than per sensor, for multi-room suites. Replacement or upgrades of existing occupancy-based controls are not eligible for an incentive.

DEFINITION OF BASELINE EQUIPMENT

Guest room energy management thermostats replace manual heating/cooling temperature set-point and fan On/Off/Auto thermostat controls. Two possible baselines exist based on whether housekeeping staff are directed to set-back (or turn off) thermostats when rooms are not rented.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for GREM is 15 years³²⁹.

DEEMED MEASURE COST

\$260/unit

The IMC documented for this measure is \$260 per room HVAC controller, which is the cost difference between a non-programmable thermostat and a GREM³³⁰.

DEEMED O&M COST ADJUSTMENTS

N/A

LOADSHAPE

Loadshape C03 - Commercial Cooling

COINCIDENCE FACTOR

A coincidence factor is not used in the determination of coincident peak kW savings.

³²⁹ DEER 2008 value for energy management systems

³³⁰ This value was extracted from Smart Ideas projects in PY1 and PY2.

Algorithm

CALCULATION OF SAVINGS

Below are the annual kWh savings per installed EMS for different sizes and types of HVAC units. The savings are achieved based on GREM's ability to automatically adjust the guest room's set temperatures and control the HVAC unit to maintain set temperatures for various occupancy modes. Note that care should be taken in selecting a value consistent with actual baseline conditions (e.g. whether housekeeping staff are directed to set-back/turn-off the thermostats when rooms are unrented). Different values are provided for Motels and Hotels since significant differences in shell performance, number of external walls per room and typical heating and cooling efficiencies result in significantly different savings estimates. Energy savings estimates are derived using a prototypical EnergyPlus simulation of a motel and a hotel³³¹. Model outputs are normalized to the installed capacity and reported here as kWh/Ton, coincident peak kW/Ton and Therms/Ton.

ELECTRIC ENERGY SAVINGS

Motel Electric Energy Savings			
Climate Zone (City based upon)	Heating Source	Baseline	Electric Savings (kWh/Ton)
1 (Rockford)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	744
		No Housekeeping Setback	1,786
	PTAC w/ Gas Heating	Housekeeping Setback	63
		No Housekeeping Setback	155
	PTHP	Housekeeping Setback	385
		No Housekeeping Setback	986
2 (Chicago)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	506
		No Housekeeping Setback	1,582
	PTAC w/ Gas Heating	Housekeeping Setback	51
		No Housekeeping Setback	163
	PTHP	Housekeeping Setback	211
		No Housekeeping Setback	798
3 (Springfield)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	462
		No Housekeeping Setback	1,382
	PTAC w/ Gas Heating	Housekeeping Setback	65
		No Housekeeping Setback	198
	PTHP	Housekeeping Setback	202
		No Housekeeping Setback	736
4 (Belleville)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	559
		No Housekeeping Setback	1,877
	PTAC w/ Gas Heating	Housekeeping Setback	85
		No Housekeeping Setback	287
	PTHP	Housekeeping Setback	260
		No Housekeeping Setback	1,023
5 (Marion-Williamson)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	388
		No Housekeeping Setback	1,339
	PTAC w/ Gas Heating	Housekeeping Setback	81

³³¹ For motels, see S. Keates, ADM Associates Workpaper: "Suggested Revisions to Guest Room Energy Management (PTAC & PTHP)", 11/14/2013 and spreadsheet summarizing the results: 'GREM Savings Summary_IL TRM_1_22_14.xlsx'. In 2014 the hotel models were also run to compile results, rather than by applying adjustment factors to the motel results as had been done in V3.0 of the TRM. The updated values can be found in 'GREM Savings Summary (Hotel)_IL TRM_10_16_14.xls'.

Motel Electric Energy Savings			
Climate Zone (City based upon)	Heating Source	Baseline	Electric Savings (kWh/Ton)
	PTHP	No Housekeeping Setback	274
		Housekeeping Setback	174
		No Housekeeping Setback	682

Hotel Electric Energy Savings			
Climate Zone (City based upon)	Heating Source	Baseline	Electric Savings (kWh/Ton)
1 (Rockford)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	204
		No Housekeeping Setback	345
	PTAC w/ Gas Heating	Housekeeping Setback	121
		No Housekeeping Setback	197
	PTHP	Housekeeping Setback	152
		No Housekeeping Setback	253
	Central Hot Water Fan Coil w/ Electric Resistance Heating	Housekeeping Setback	177
		No Housekeeping Setback	296
	Central Hot Water Fan Coil w/ Gas Heating	Housekeeping Setback	94
		No Housekeeping Setback	148
2 (Chicago)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	188
		No Housekeeping Setback	342
	PTAC w/ Gas Heating	Housekeeping Setback	119
		No Housekeeping Setback	195
	PTHP	Housekeeping Setback	145
		No Housekeeping Setback	250
	Central Hot Water Fan Coil w/ Electric Resistance Heating	Housekeeping Setback	161
		No Housekeeping Setback	294
	Central Hot Water Fan Coil w/ Gas Heating	Housekeeping Setback	92
		No Housekeeping Setback	147
3 (Springfield)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	182
		No Housekeeping Setback	291
	PTAC w/ Gas Heating	Housekeeping Setback	123
		No Housekeeping Setback	197
	PTHP	Housekeeping Setback	145
		No Housekeeping Setback	233
	Central Hot Water Fan Coil w/ Electric Resistance Heating	Housekeeping Setback	153
		No Housekeeping Setback	240
	Central Hot Water Fan Coil w/ Gas Heating	Housekeeping Setback	94
		No Housekeeping Setback	146
4 (Belleville)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	182
		No Housekeeping Setback	308
	PTAC w/ Gas Heating	Housekeeping Setback	125
		No Housekeeping Setback	199
	PTHP	Housekeeping Setback	146
		No Housekeeping Setback	240
	Central Hot Water Fan Coil w/ Electric Resistance Heating	Housekeeping Setback	152
		No Housekeeping Setback	255

Hotel Electric Energy Savings			
Climate Zone (City based upon)	Heating Source	Baseline	Electric Savings (kWh/Ton)
5 (Marion-Williamson)	Central Hot Water Fan Coil w/ Gas Heating	Housekeeping Setback	95
		No Housekeeping Setback	147
	PTAC w/ Electric Resistance Heating	Housekeeping Setback	171
		No Housekeeping Setback	295
	PTAC w/ Gas Heating	Housekeeping Setback	122
		No Housekeeping Setback	199
	PTHP	Housekeeping Setback	140
		No Housekeeping Setback	235
	Central Hot Water Fan Coil w/ Electric Resistance Heating	Housekeeping Setback	141
		No Housekeeping Setback	243
	Central Hot Water Fan Coil w/ Gas Heating	Housekeeping Setback	92
		No Housekeeping Setback	146

SUMMER COINCIDENT PEAK DEMAND SAVINGS

Motel Coincident Peak Demand Savings			
Climate Zone (City based upon)	Heating Source	Baseline	Coincident Peak Demand Savings (kW/Ton)
1 (Rockford)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	0.08
		No Housekeeping Setback	0.17
	PTAC w/ Gas Heating	Housekeeping Setback	0.08
		No Housekeeping Setback	0.17
	PTHP	Housekeeping Setback	0.08
		No Housekeeping Setback	0.17
2 (Chicago)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	0.06
		No Housekeeping Setback	0.17
	PTAC w/ Gas Heating	Housekeeping Setback	0.06
		No Housekeeping Setback	0.17
	PTHP	Housekeeping Setback	0.06
		No Housekeeping Setback	0.17
3 (Springfield)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	0.07
		No Housekeeping Setback	0.17
	PTAC w/ Gas Heating	Housekeeping Setback	0.07
		No Housekeeping Setback	0.17
	PTHP	Housekeeping Setback	0.07
		No Housekeeping Setback	0.17
4 (Belleville)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	0.10
		No Housekeeping Setback	0.28
	PTAC w/ Gas Heating	Housekeeping Setback	0.10
		No Housekeeping Setback	0.28
	PTHP	Housekeeping Setback	0.10
		No Housekeeping Setback	0.28
5 (Marion-Williamson)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	0.08
		No Housekeeping Setback	0.21
	PTAC w/ Gas Heating	Housekeeping Setback	0.08
		No Housekeeping Setback	0.21

Motel Coincident Peak Demand Savings			
Climate Zone (City based upon)	Heating Source	Baseline	Coincident Peak Demand Savings (kW/Ton)
	PTHP	Housekeeping Setback	0.08
		No Housekeeping Setback	0.21

Hotel Coincident Peak Demand Savings			
Climate Zone (City based upon)	Heating Source	Baseline	Coincident Peak Demand Savings (kW/Ton)
1 (Rockford)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	0.08
		No Housekeeping Setback	0.11
	PTAC w/ Gas Heating	Housekeeping Setback	0.08
		No Housekeeping Setback	0.11
	PTHP	Housekeeping Setback	0.08
		No Housekeeping Setback	0.11
	Central Hot Water Fan Coil w/ Electric Resistance Heating	Housekeeping Setback	0.05
		No Housekeeping Setback	0.08
2 (Chicago)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	0.07
		No Housekeeping Setback	0.11
	PTAC w/ Gas Heating	Housekeeping Setback	0.07
		No Housekeeping Setback	0.11
	PTHP	Housekeeping Setback	0.07
		No Housekeeping Setback	0.11
	Central Hot Water Fan Coil w/ Electric Resistance Heating	Housekeeping Setback	0.05
		No Housekeeping Setback	0.07
3 (Springfield)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	0.08
		No Housekeeping Setback	0.11
	PTAC w/ Gas Heating	Housekeeping Setback	0.08
		No Housekeeping Setback	0.11
	PTHP	Housekeeping Setback	0.08
		No Housekeeping Setback	0.11
	Central Hot Water Fan Coil w/ Electric Resistance Heating	Housekeeping Setback	0.05
		No Housekeeping Setback	0.07
4 (Belleville)	PTAC w/ Electric Resistance Heating	Housekeeping Setback	0.08
		No Housekeeping Setback	0.11
	PTAC w/ Gas Heating	Housekeeping Setback	0.08
		No Housekeeping Setback	0.11
	PTHP	Housekeeping Setback	0.08
		No Housekeeping Setback	0.11
	Central Hot Water Fan Coil w/ Electric Resistance Heating	Housekeeping Setback	0.05
		No Housekeeping Setback	0.08

Hotel Coincident Peak Demand Savings			
Climate Zone (City based upon)	Heating Source	Baseline	Coincident Peak Demand Savings (kW/Ton)
5 (Marion-Williamson)	Central Hot Water Fan Coil w/ Gas Heating	Housekeeping Setback	0.05
		No Housekeeping Setback	0.08
	PTAC w/ Electric Resistance Heating	Housekeeping Setback	0.08
		No Housekeeping Setback	0.11
	PTAC w/ Gas Heating	Housekeeping Setback	0.08
		No Housekeeping Setback	0.11
	PTHP	Housekeeping Setback	0.08
		No Housekeeping Setback	0.11
	Central Hot Water Fan Coil w/ Electric Resistance Heating	Housekeeping Setback	0.05
		No Housekeeping Setback	0.08
	Central Hot Water Fan Coil w/ Gas Heating	Housekeeping Setback	0.05
		No Housekeeping Setback	0.08

NATURAL GAS ENERGY SAVINGS

For PTACs with gas heating:

Motel Natural Gas Energy Savings		
Climate Zone (City based upon)	Baseline	Gas Savings (Therms/Ton)
1 (Rockford)	Housekeeping Setback	30
	No Housekeeping Setback	71
2 (Chicago)	Housekeeping Setback	20
	No Housekeeping Setback	62
3 (Springfield)	Housekeeping Setback	17
	No Housekeeping Setback	52
4 (Belleville)	Housekeeping Setback	21
	No Housekeeping Setback	70
5 (Marion-Williamson)	Housekeeping Setback	13
	No Housekeeping Setback	47

Hotel Natural Gas Energy Savings			
Climate Zone (City based upon)	Heating Source	Baseline	Gas Savings (Therms/Ton)
1 (Rockford)	PTAC w/ Gas Heating	Housekeeping Setback	3.6
		No Housekeeping Setback	6.4
	Central Hot Water Fan Coil w/ Gas Heating	Housekeeping Setback	3.6
		No Housekeeping Setback	6.4
2 (Chicago)	PTAC w/ Gas Heating	Housekeeping Setback	3.0
		No Housekeeping Setback	6.5
	Central Hot Water Fan Coil w/ Gas Heating	Housekeeping Setback	3.0
		No Housekeeping Setback	6.5
3 (Springfield)	PTAC w/ Gas Heating	Housekeeping Setback	2.6
		No Housekeeping Setback	4.1
		Housekeeping Setback	2.6

Hotel Natural Gas Energy Savings			
Climate Zone (City based upon)	Heating Source	Baseline	Gas Savings (Therms/Ton)
	Central Hot Water Fan Coil w/ Gas Heating	No Housekeeping Setback	4.1
4 (Belleville)	PTAC w/ Gas Heating	Housekeeping Setback	2.5
		No Housekeeping Setback	4.8
	Central Hot Water Fan Coil w/ Gas Heating	Housekeeping Setback	2.5
		No Housekeeping Setback	4.8
5 (Marion-Williamson)	PTAC w/ Gas Heating	Housekeeping Setback	2.1
		No Housekeeping Setback	4.2
	Central Hot Water Fan Coil w/ Gas Heating	Housekeeping Setback	2.1
		No Housekeeping Setback	4.2

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-GREM-V05-150601

REVIEW DEADLINE: 1/1/2022

4.4.9 Heat Pump Systems

DESCRIPTION

This measure applies to the installation of high-efficiency air cooled, water source, ground water source, and ground source heat pump systems. This measure could apply to replacing an existing unit at the end of its useful life, or installation of a new unit in a new or existing building.

This measure was developed to be applicable to the following program types: TOS NC. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a high-efficiency air cooled, water source, ground water source, or ground source heat pump system that exceeds the energy efficiency requirements of the 2012 or 2015 (applicable from 01/01/2016) International Energy Conservation Code (IECC), depending on the IECC in effect on the date of the building permit (if unknown assume IECC 2015).

DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline equipment is assumed to be a standard-efficiency air cooled, water source, ground water source, or ground source heat pump system that meets the energy efficiency requirements of the 2012 or 2015 IECC, depending on the IECC in effect on the date of the building permit (if unknown assume IECC 2015). Note the Time of Sale baseline is assumed to be IECC 2015. The rating conditions for the baseline and efficient equipment efficiencies must be equivalent.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years³³² except for geothermal heat pump systems which have an expected measures life of 25 years³³³.

DEEMED MEASURE COST

For analysis purposes, the incremental capital cost for this measure is assumed as \$100 per ton for air-cooled units.³³⁴ The incremental cost for all other equipment types should be determined on a site-specific basis.

LOADSHAPE

Loadshape C05 - Commercial Electric Heating and Cooling

COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market. Both values provided are based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren.

$$\begin{aligned} CF_{SSP} &= \text{Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)} \\ &= 91.3\% \text{ }^{335} \end{aligned}$$

³³²Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007.

³³³ System life of indoor components as per DOE estimate <http://energy.gov/energysaver/articles/geothermal-heat-pumps>. The ground loop has a much longer life, but the compressor and other mechanical components are the same as an ASHP.

³³⁴ Based on a review of TRM incremental cost assumptions from Vermont, Wisconsin, and California.

³³⁵ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

$$\begin{aligned} CF_{PJM} &= \text{PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)} \\ &= 47.8\%^{336} \end{aligned}$$

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

For units with cooling capacities less than 65 kBtu/hr:

$$\begin{aligned} \Delta kWh &= \text{Annual kWh Savings}_{\text{cool}} + \text{Annual kWh Savings}_{\text{heat}} \\ \text{Annual kWh Savings}_{\text{cool}} &= (\text{kBtu/hr}_{\text{cool}}) * [(1/\text{SEER}_{\text{base}}) - (1/\text{SEER}_{\text{ee}})] * \text{EFLH}_{\text{cool}} \\ \text{Annual kWh Savings}_{\text{heat}} &= (\text{kBtu/hr}_{\text{heat}}) * [(1/\text{HSPF}_{\text{base}}) - (1/\text{HSPF}_{\text{ee}})] * \text{EFLH}_{\text{heat}} \end{aligned}$$

For units with cooling capacities equal to or greater than 65 kBtu/hr:

$$\begin{aligned} \Delta kWh &= \text{Annual kWh Savings}_{\text{cool}} + \text{Annual kWh Savings}_{\text{heat}} \\ \text{Annual kWh Savings}_{\text{cool}} &= (\text{kBtu/hr}_{\text{cool}}) * [(1/\text{EER}_{\text{base}}) - (1/\text{EER}_{\text{ee}})] * \text{EFLH}_{\text{cool}} \\ \text{Annual kWh Savings}_{\text{heat}} &= (\text{kBtu/hr}_{\text{heat}})/3.412 * [(1/\text{COP}_{\text{base}}) - (1/\text{COP}_{\text{ee}})] * \text{EFLH}_{\text{heat}} \end{aligned}$$

Where:

kBtu/hr _{cool}	= capacity of the cooling equipment in kBtu per hour (1 ton of cooling capacity equals 12 kBtu/hr).
	= Actual installed
SEER _{base}	= Seasonal Energy Efficiency Ratio of the baseline equipment
	= SEER from tables below, based on the applicable IECC on the date of the building permit (if unknown assume IECC 2015).
SEER _{ee}	= Seasonal Energy Efficiency Ratio of the energy efficient equipment.
	= Actual installed
EFLH _{cool}	= Equivalent Full Load Hours for cooling are provided in section 4.4 HVAC End Use.
HSPF _{base}	= Heating Seasonal Performance Factor of the baseline equipment
	= HSPF from tables below, based on the applicable IECC on the date of the building permit (if unknown assume IECC 2015).
HSPF _{ee}	= Heating Seasonal Performance Factor of the energy efficient equipment.
	= Actual installed. If rating is COP, HSPF = COP * 3.413
EFLH _{heat}	= heating mode equivalent full load hours are provided in section 4.4 HVAC End Use.
EER _{base}	= Energy Efficiency Ratio of the baseline equipment

³³⁶ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

= EER from tables below, based on the applicable IECC on the date of the building permit (if unknown assume IECC 2015).. For air-cooled units < 65 kBtu/hr, assume the following conversion from SEER to EER for calculation of peak savings:³³⁷

$$EER = (-0.02 * SEER^2) + (1.12 * SEER)$$

EER _{ee}	= Energy Efficiency Ratio of the energy efficient equipment. For air-cooled units < 65 kBtu/hr, if the actual EER _{ee} is unknown, assume the conversion from SEER to EER as provided above.
	= Actual installed
kBtu/hr _{heat}	= capacity of the heating equipment in kBtu per hour.
	= Actual installed
3.412	= Btu per Wh.
COP _{base}	= coefficient of performance of the baseline equipment
	= COP from tables below, based on the applicable IECC on the date of the building permit (if unknown assume IECC 2015). If rating is HSPF, COP = HSPF / 3.413
COP _{ee}	= coefficient of performance of the energy efficient equipment.
	= Actual installed

³³⁷ Based on Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder. Note this is appropriate for single speed units only.

Minimum Efficiency Requirements: 2012 IECC

TABLE C403.2.3(2)
MINIMUM EFFICIENCY REQUIREMENTS:
ELECTRICALLY OPERATED UNITARY AND APPLIED HEAT PUMPS

EQUIPMENT TYPE	SIZE CATEGORY	HEATING SECTION TYPE	SUBCATEGORY OR RATING CONDITION	MINIMUM EFFICIENCY	TEST PROCEDURE ^a
Air cooled (cooling mode)	< 65,000 Btu/h ^b	All	Split System	13.0 SEER	AHRI 210/240
			Single Packaged	13.0 SEER	
Through-the-wall, air cooled	≤ 30,000 Btu/h ^b	All	Split System	13.0 SEER	
			Single Packaged	13.0 SEER	
Single-duct high-velocity air cooled	< 65,000 Btu/h ^b	All	Split System	10.0 SEER	
Air cooled (cooling mode)	≥ 65,000 Btu/h and < 135,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	11.0 EER 11.2 IEER	AHRI 340/360
		All other	Split System and Single Package	10.8 EER 11.0 IEER	
	≥ 135,000 Btu/h and < 240,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	10.6 EER 10.7 IEER	
		All other	Split System and Single Package	10.4 EER 10.5 IEER	
	≥ 240,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	9.5 EER 9.6 IEER	
		All other	Split System and Single Package	9.3 EER 9.4 IEER	
Water source (cooling mode)	< 17,000 Btu/h	All	86°F entering water	11.2 EER	ISO 13256-1
	≥ 17,000 Btu/h and < 65,000 Btu/h	All	86°F entering water	12.0 EER	
	≥ 65,000 Btu/h and < 135,000 Btu/h	All	86°F entering water	12.0 EER	
Ground water source (cooling mode)	< 135,000 Btu/h	All	59°F entering water	16.2 EER	
		All	77°F entering water	13.4 EER	
Water-source water to water (cooling mode)	< 135,000 Btu/h	All	86°F entering water	10.6 EER	ISO 13256-2
			59°F entering water	16.3 EER	
Ground water source Brine to water (cooling mode)	< 135,000 Btu/h	All	77°F entering fluid	12.1 EER	
Air cooled (heating mode)	< 65,000 Btu/h ^b	—	Split System	7.7 HSPF	AHRI 210/240
		—	Single Package	7.7 HSPF	
Through-the-wall, (air cooled, heating mode)	≤ 30,000 Btu/h ^b (cooling capacity)	—	Split System	7.4 HSPF	
		—	Single Package	7.4 HSPF	
Small-duct high velocity (air cooled, heating mode)	< 65,000 Btu/h ^b	—	Split System	6.8 HSPF	

(continued)

TABLE C403.2.3(2)—continued
 MINIMUM EFFICIENCY REQUIREMENTS:
 ELECTRICALLY OPERATED UNITARY AND APPLIED HEAT PUMPS

EQUIPMENT TYPE	SIZE CATEGORY	HEATING SECTION TYPE	SUB-CATEGORY OR RATING CONDITION	MINIMUM EFFICIENCY	TEST PROCEDURE ^a
Air cooled (heating mode)	≥ 65,000 Btu/h and < 135,000 Btu/h (cooling capacity)	—	47°F db/43°F wb Outdoor Air	3.3 COP	AHRI 340/360
			17°F db/15°F wb Outdoor Air	2.25 COP	
	≥ 135,000 Btu/h (cooling capacity)	—	47°F db/43°F wb Outdoor Air	3.2 COP	
			17°F db/15°F wb Outdoor Air	2.05 COP	
Water source (heating mode)	< 135,000 Btu/h (cooling capacity)	—	68°F entering water	4.2 COP	ISO 13256-1
Ground water source (heating mode)	< 135,000 Btu/h (cooling capacity)	—	50°F entering water	3.6 COP	
Ground source (heating mode)	< 135,000 Btu/h (cooling capacity)	—	32°F entering fluid	3.1 COP	
Water-source water to water (heating mode)	< 135,000 Btu/h (cooling capacity)	—	68°F entering water	3.7 COP	ISO 13256-2
		—	50°F entering water	3.1 COP	
Ground source brine to water (heating mode)	< 135,000 Btu/h (cooling capacity)	—	32°F entering fluid	2.5 COP	

For SI: 1 British thermal unit per hour = 0.2931 W, °C = [(°F) - 32]/1.8.

a. Chapter 6 of the referenced standard contains a complete specification of the referenced test procedure, including the reference year version of the test procedure.

b. Single-phase, air-cooled air conditioners less than 65,000 Btu/h are regulated by NAECA. SEER values are those set by NAECA.

Minimum Efficiency Requirements: 2015 IECC

TABLE C403.2.3(2)
 MINIMUM EFFICIENCY REQUIREMENTS:
 ELECTRICALLY OPERATED UNITARY AND APPLIED HEAT PUMPS

EQUIPMENT TYPE	SIZE CATEGORY	HEATING SECTION TYPE	SUBCATEGORY OR RATING CONDITION	MINIMUM EFFICIENCY		TEST PROCEDURE ^a
				Before 1/1/2016	As of 1/1/2016	
Air cooled (cooling mode)	< 65,000 Btu/h ^b	All	Split System	13.0 SEER ^c	14.0 SEER ^c	AHRI 210/240
			Single Package	13.0 SEER ^c	14.0 SEER ^c	
Through-the-wall, air cooled	≤ 30,000 Btu/h ^b	All	Split System	12.0 SEER	12.0 SEER	
			Single Package	12.0 SEER	12.0 SEER	
Single-duct high-velocity air cooled	< 65,000 Btu/h ^b	All	Split System	11.0 SEER	11.0 SEER	
Air cooled (cooling mode)	≥ 65,000 Btu/h and < 135,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	11.0 EER 11.2 IEER	11.0 EER 12.0 IEER	AHRI 340/360
		All other	Split System and Single Package	10.8 EER 11.0 IEER	10.8 EER 11.8 IEER	
	≥ 135,000 Btu/h and < 240,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	10.6 EER 10.7 IEER	10.6 EER 11.6 IEER	
		All other	Split System and Single Package	10.4 EER 10.5 IEER	10.4 EER 11.4 IEER	
	≥ 240,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	9.5 EER 9.6 IEER	9.5 EER 10.6 IEER	
		All other	Split System and Single Package	9.3 EER 9.4 IEER	9.3 EER 9.4 IEER	
Water to Air: Water Loop (cooling mode)	< 17,000 Btu/h	All	86°F entering water	12.2 EER	12.2 EER	ISO 13256-1
	≥ 17,000 Btu/h and < 65,000 Btu/h	All	86°F entering water	13.0 EER	13.0 EER	
	≥ 65,000 Btu/h and < 135,000 Btu/h	All	86°F entering water	13.0 EER	13.0 EER	
Water to Air: Ground Water (cooling mode)	< 135,000 Btu/h	All	59°F entering water	18.0 EER	18.0 EER	ISO 13256-1
Brine to Air: Ground Loop (cooling mode)	< 135,000 Btu/h	All	77°F entering water	14.1 EER	14.1 EER	ISO 13256-1
Water to Water: WaterLoop (cooling mode)	< 135,000 Btu/h	All	86°F entering water	10.6 EER	10.6 EER	ISO 13256-2
Water to Water: Ground Water (cooling mode)	< 135,000 Btu/h	All	59°F entering water	16.3 EER	16.3 EER	
Brine to Water: Ground Loop (cooling mode)	< 135,000 Btu/h	All	77°F entering fluid	12.1 EER	12.1 EER	

(continued)

TABLE C403.2.3(2)—continued
 MINIMUM EFFICIENCY REQUIREMENTS:
 ELECTRICALLY OPERATED UNITARY AND APPLIED HEAT PUMPS

EQUIPMENT TYPE	SIZE CATEGORY	HEATING SECTION TYPE	SUBCATEGORY OR RATING CONDITION	MINIMUM EFFICIENCY		TEST PROCEDURE ^a
				Before 1/1/2016	As of 1/1/2016	
Air cooled (heating mode)	< 65,000 Btu/h ^b	—	Split System	7.7 HSPF ^c	8.2 HSPF ^c	AHRI 210/240
		—	Single Package	7.7 HSPF ^c	8.0 HSPF ^c	
Through-the-wall, (air cooled, heating mode)	≤ 30,000 Btu/h ^b (cooling capacity)	—	Split System	7.4 HSPF	7.4 HSPF	
		—	Single Package	7.4 HSPF	7.4 HSPF	
Small-duct high velocity (air cooled, heating mode)	< 65,000 Btu/h ^b	—	Split System	6.8 HSPF	6.8 HSPF	
Air cooled (heating mode)	≥ 65,000 Btu/h and < 135,000 Btu/h (cooling capacity)	—	47°F db/43°F wb outdoor air	3.3 COP	3.3 COP	AHRI 340/360
			17°F db/15°F wb outdoor air	2.25 COP	2.25 COP	
	≥ 135,000 Btu/h (cooling capacity)	—	47°F db/43°F wb outdoor air	3.2 COP	3.2 COP	
			17°F db/15°F wb outdoor air	2.05 COP	2.05 COP	
Water to Air: Water Loop (heating mode)	< 135,000 Btu/h (cooling capacity)	—	68°F entering water	4.3 COP	4.3 COP	ISO 13256-1
Water to Air: Ground Water (heating mode)	< 135,000 Btu/h (cooling capacity)	—	50°F entering water	3.7 COP	3.7 COP	
Brine to Air: Ground Loop (heating mode)	< 135,000 Btu/h (cooling capacity)	—	32°F entering fluid	3.2 COP	3.2 COP	
Water to Water: Water Loop (heating mode)	< 135,000 Btu/h (cooling capacity)	—	68°F entering water	3.7 COP	3.7 COP	ISO 13256-2
Water to Water: Ground Water (heating mode)	< 135,000 Btu/h (cooling capacity)	—	50°F entering water	3.1 COP	3.1 COP	
Brine to Water: Ground Loop (heating mode)	< 135,000 Btu/h (cooling capacity)	—	32°F entering fluid	2.5 COP	2.5 COP	

For SI: 1 British thermal unit per hour = 0.2931 W, °C = [(°F) - 32]/1.8.

a. Chapter 6 contains a complete specification of the referenced test procedure, including the reference year version of the test procedure.

b. Single-phase, air-cooled air conditioners less than 65,000 Btu/h are regulated by NAECA. SEER values are those set by NAECA.

c. Minimum efficiency as of January 1, 2015.

For example a 5 ton cooling unit with 60 kbtu heating, an efficient SEER of 16, and an efficient HSPF of 9.5, at a restaurant in Chicago with a building permit dated after 1/1/2016 saves:

$$\begin{aligned}\Delta kWh &= [(60) * [(1/14) - (1/16)] * 1134] + [(60) * [(1/8.2) - (1/9.5)] * 1354] \\ &= 1963.2 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = ((kBtu/hr_{cool}) * (1/EER_{base} - 1/EER_{ee})) * CF$$

Where CF value is chosen between:

$$\begin{aligned}CF_{SSP} &= \text{Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)} \\ &= 91.3\%^{338}\end{aligned}$$

$$\begin{aligned}CF_{PJM} &= \text{PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)} \\ &= 47.8\%^{339}\end{aligned}$$

For example a 5 ton cooling unit with 60 kbtu heating, an efficient EER of 12.5 with a building permit dated after 1/1/2016 saves:

$$\begin{aligned}\Delta kW &= (60 * (1/11 - 1/12.5)) * 0.913 \\ &= 0.598 \text{ kW}\end{aligned}$$

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-HPSY-V05-180101

REVIEW DEADLINE: 1/1/2019

³³⁸ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

³³⁹ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

4.4.10 High Efficiency Boiler

DESCRIPTION

To qualify for this measure the installed equipment must be replacement of an existing boiler at the end of its service life, in a commercial or multifamily space with a high efficiency, gas-fired steam or hot water boiler. High efficiency boilers achieve gas savings through the utilization of a sealed combustion chamber and multiple heat exchangers that remove a significant portion of the waste heat from flue gasses. Because multiple heat exchangers are used to remove waste heat from the escaping flue gasses, some of the flue gasses condense and must be drained.

This measure was developed to be applicable to the following program types: TOS, RF. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a boiler used 80% or more for space heating, not process, and boiler AFUE, TE (thermal efficiency), or Ec (combustion efficiency) rating must be rated greater than or equal to 85% for hot water boilers and 81% for steam boilers.

DEFINITION OF BASELINE EQUIPMENT

Dependent on when the unit is installed and whether the unit is hot water or steam. The baseline efficiency source is the Energy Independence and Security Act of 2007 with technical amendments from Federal Register, volume 73, Number 145, Monday, July 28, 2008 for boilers <300,000 Btu/hr and is Final Rule, Federal Register, volume 74, Number 139, Wednesday, July 22, 2009 for boiler ≥300,000 Btu/hr.

Hot water boiler baseline:

Year	Efficiency
Hot Water <300,000 Btu/hr < June 1, 2013 ³⁴⁰	80% AFUE
Hot Water <300,000 Btu/hr ≥ June 1, 2013	82% AFUE
Hot Water ≥300,000 & ≤2,500,000 Btu/hr	80% TE
Hot Water >2,500,000 Btu/hr	82% Ec

Steam boiler baseline:

Year	Efficiency
Steam <300,000 Btu/hr < June 1, 2013 ³⁴¹	75% AFUE
Steam <300,000 Btu/hr ≥ June 1, 2013	80% AFUE
Steam - all except natural draft ≥300,000 & ≤2,500,000 Btu/hr	79% TE
Steam - natural draft ≥300,000 & ≤2,500,000 Btu/hr	77% TE
Steam - all except natural draft >2,500,000 Btu/hr	79% TE
Steam - natural draft >2,500,000 Btu/hr	77% TE

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 20 years³⁴²

³⁴⁰ The Federal baseline for boilers <300,000 btu/hr changes from 80% to 82% in September 2012. To prevent a change in baseline mid-program, the increase in efficiency is delayed until June 2013 when a new program year starts.

³⁴¹ Ibid.

³⁴² The Technical support documents for federal residential appliance standards: http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/appendix_e.pdf Note that this value is below the 20 years used by CA's DEER and the range of 20-40 year estimate made by the Consortium for Energy Efficiency in 2010

DEEMED MEASURE COST

The incremental capital cost for this measure depends on efficiency as listed below³⁴³

Measure Tier	Incr. Cost, per unit
ENERGY STAR® Minimum	\$1,470
AFUE 90%	\$2,400
AFUE 95%	\$3,370
AFUE ≥ 96%	\$4,340
Boilers > 300,000 Btu/hr with TE (thermal efficiency) rating	Custom

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF ENERGY SAVINGS**ELECTRIC ENERGY SAVINGS**

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms} = \text{EFLH} * \text{Capacity} * ((\text{EfficiencyRating}(\text{actual}) - \text{EfficiencyRating}(\text{base})) / \text{EfficiencyRating}(\text{base})) / 100,000$$

Where:

EFLH = Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use

Capacity = Nominal Heating Input Capacity Boiler Size (Btu/hr) for efficient unit not existing unit
= custom Boiler input capacity in Btu/hr

EfficiencyRating(base) = Baseline Boiler Efficiency Rating, dependant on year and boiler type. Baseline efficiency values by boiler type and capacity are found in the Definition of Baseline Equipment Section

EfficiencyRating(actual) = Efficient Boiler Efficiency Rating use actual value

Measure Type	Actual AFUE
ENERGY STAR® Minimum	85%

³⁴³ Average of low and high incremental cost based on Nicor Gas program data for non-condensing and condensing boilers. Nicor Gas Energy Efficiency Plan 2011 - 2014, May 27, 2011 \$1,470 for ≤ 300,000 Btu/hr for non-condensing hydronic boilers >85% AFUE & \$3,365 for condensing boilers > 90% AFUE. The exception is \$4,340 for AFUE ≥ 96% AFUE which was obtained from extrapolation above the size range that Nicor Gas Energy Efficiency Plan provided for incremental cost.

Measure Type	Actual AFUE
AFUE 90%	90%
AFUE 95%	95%
AFUE ≥ 96%	≥ 96%
Custom	Value to one significant digit i.e. 95.7%

EXAMPLE

For example, a 150,000 btu/hr water boiler meeting AFUE 90% in Rockford at a high rise office building , in the year 2012

$$\begin{aligned}\Delta\text{Therms} &= 2,089 * 150,000 * (0.90-0.80)/0.80 / 100,000 \text{ Btu/Therm} \\ &= 392 \text{ Therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-BOIL-V05-150601

REVIEW DEADLINE: 1/1/2019

4.4.11 High Efficiency Furnace

DESCRIPTION

This measure covers the installation of a high efficiency gas furnace in lieu of a standard efficiency gas furnace in a commercial or industrial space. High efficiency gas furnaces achieve savings through the utilization of a sealed, super insulated combustion chamber, more efficient burners, and multiple heat exchangers that remove a significant portion of the waste heat from the flue gasses. Because multiple heat exchangers are used to remove waste heat from the escaping flue gasses, most of the flue gasses condense and must be drained. Furnaces equipped with ECM fan motors can save additional electric energy

This measure was developed to be applicable to the following program types: TOS RF and EREP. If applied to other program types, the measure savings should be verified.

Time of sale:

- a. The installation of a new high efficiency, gas-fired condensing furnace in a commercial location. This could relate to the replacement of an existing unit at the end of its useful life, or the installation of a new system.

Early replacement:

Early Replacement determination will be based on meeting the following conditions:

- The existing unit is operational when replaced, or
- The existing unit requires minor repairs (<\$528)³⁴⁴.
- All other conditions will be considered Time of Sale.

The Baseline AFUE of the existing unit replaced:

- If the AFUE of the existing unit is known and $\leq 75\%$, the Baseline AFUE is the actual AFUE value of the unit replaced. If the AFUE is $> 75\%$, the Baseline AFUE = 80%.
- If the AFUE of the existing unit is unknown, use assumptions in variable list below (AFUE(exist)).
- If the operational status or repair cost of the existing unit is unknown, use time of sale assumptions.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a furnace with input energy less than 225,000 Btu/hr rated natural gas fired furnace with an Annual Fuel Utilization Efficiency (AFUE) rating and fan electrical efficiency exceeding the program requirements:

DEFINITION OF BASELINE EQUIPMENT

Time of Sale: Although the current Federal Standard for gas furnaces is an AFUE rating of 78%, based upon review of available product in the AHRI database, the baseline efficiency for this characterization is assumed to be 80%. The baseline will be adjusted when the Federal Standard is updated.

Early replacement: The baseline for this measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and a new baseline unit for the remainder of the measure life. As discussed above we estimate that the new baseline unit that could be purchased in the year the existing unit would have needed replacing is 90%

³⁴⁴ The Technical Advisory Committee agreed that if the cost of repair is less than 20% of the new baseline replacement cost it can be considered early replacement. Note the non-inflated cost is used as this would be a cost consideration in the program year.

DEFINITION OF MEASURE LIFE

The expected measure life is assumed to be 16.5 years³⁴⁵

Remaining life of existing equipment is assumed to be 5.5 years³⁴⁶.

DEEMED MEASURE COST

Time of Sale: The incremental capital cost for this measure depends on efficiency as listed below³⁴⁷:

AFUE	Installation Cost	Incremental Install Cost
80%	\$2011	n/a
90%	\$2641	\$630
91%	\$2727	\$716
92%	\$2813	\$802
93%	\$3049	\$1,038
94%	\$3286	\$1,275
95%	\$3522	\$1,511
96%	\$3758	\$1,747

Early Replacement: The full installation cost is provided in the table above. The assumed deferred cost (after 5.5 years) of replacing existing equipment with a new baseline unit is assumed to be \$2876³⁴⁸. This cost should be discounted to present value using the nominal discount rate.

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = \text{Heating Savings} + \text{Cooling Savings} + \text{Shoulder Season Savings}$$

Where:

$$\begin{aligned} \text{Heating Savings} &= \text{Brushless DC motor or Electronically commutated motor (ECM)} \\ &= 418 \text{ kWh}^{349} \end{aligned}$$

³⁴⁵ Average of 15-18 year lifetime estimate made by the Consortium for Energy Efficiency in 2010.

³⁴⁶ Assumed to be one third of effective useful life

³⁴⁷ Based on data from Appendix E of the Appliance Standards Technical Support Documents including equipment cost and installation labor (http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/appendix_e.pdf). Where efficiency ratings are not provided, the values are interpolated from those that are.

³⁴⁸ \$2641 inflated using 1.91% rate.

³⁴⁹ To estimate heating, cooling and shoulder season savings for Illinois, VEIC adapted results from a 2009 Focus on Energy study of BPM blower motor savings in Wisconsin. This study included effects of behavior change based on the efficiency of new motor greatly increasing the amount of people that run the fan continuously. The savings from the Wisconsin study were adjusted to account for different run hour assumptions (average values used) for Illinois. See: FOE to IL Blower Savings.xlsx.

Cooling Savings = Brushless DC motor or electronically commutated motor (ECM) savings during cooling season

If air conditioning = 263 kWh

If no air conditioning = 175 kWh

If unknown (weighted average)= 241 kWh³⁵⁰

Shoulder Season Savings = Brushless DC motor or electronically commutated motor (ECM) savings during shoulder seasons

= 51 kWh

EXAMPLE

For example, a blower motor in a low rise office building where air conditioning presence is unknown:

$\Delta \text{kWh} = \text{Heating Savings} + \text{Cooling Savings} + \text{Shoulder Season Savings}$

= 418 + 241 + 51

= 710 kWh

SUMMER COINCIDENT PEAK DEMAND SAVINGS

For units that have evaporator coils and condensing units and are cooling in the summer in addition to heating in the winter the summer coincident peak demand savings should be calculated. If the unit is not equipment with coils or condensing units, the summer peak demand savings will not apply.

$\Delta \text{kW} = (\text{Cooling Savings} / \text{HOURSyear}) * \text{CF}$

Where:

HOURSyear = Actual hours per year if known, otherwise use hours from Table below for building type³⁵¹.

Building Type	HOURSyear
Assembly	2150
Assisted Living	4373
College	1605
Convenience Store	2084
Elementary School	3276
Garage	2102
Grocery	2096
Healthcare Clinic	1987
High School	3141
Hospital - VAV econ	2788
Hospital - CAV econ	2881
Hospital - CAV no econ	8760
Hospital - FCU	8729
Manufacturing Facility	2805

³⁵⁰ The weighted average value is based on assumption that 75% of buildings installing BPM furnace blower motors have Central AC.

³⁵¹ Hours per year are estimated using the eQuest models as the total number of hours the cooling system is operating for each building type.

Building Type	HOURSyear
MF - High Rise	4237
MF - Mid Rise	2899
Hotel/Motel – Guest	4479
Hotel/Motel - Common	8712
Movie Theater	2120
Office - High Rise - VAV econ	2038
Office - High Rise - CAV econ	4849
Office - High Rise - CAV no econ	5682
Office - High Rise - FCU	3069
Office - Low Rise	2481
Office - Mid Rise	1881
Religious Building	2830
Restaurant	3350
Retail - Department Store	2528
Retail - Strip Mall	2266
Warehouse	770
Unknown	2718

CF =Summer Peak Coincidence Factor for measure is provided below for different building types³⁵²:

HVAC Pumps	CF
Assembly	48.3%
Assisted Living	52.9%
College	14.2%
Convenience Store	57.1%
Elementary School	33.3%
Garage	61.9%
Grocery	47.5%
Healthcare Clinic	61.9%
High School	28.8%
Hospital - VAV econ	57.6%
Hospital - CAV econ	61.5%
Hospital - CAV no econ	64.8%
Hospital - FCU	60.9%
Manufacturing Facility	43.3%
MF - High Rise - Common	43.7%
MF - Mid Rise	24.3%
Hotel/Motel - Guest	62.9%
Hotel/Motel - Common	64.6%
Movie Theater	41.9%
Office - High Rise - VAV econ	43.2%
Office - High Rise - CAV econ	48.3%
Office - High Rise - CAV no econ	50.3%
Office - High Rise - FCU	46.2%

³⁵² Coincidence Factors are estimated using the eQuest models..

HVAC Pumps	CF
Office - Low Rise	47.4%
Office - Mid Rise	42.8%
Religious Building	43.3%
Restaurant	48.8%
Retail - Department Store	50.5%
Retail - Strip Mall	52.8%
Warehouse	22.5%
Unknown	42.4%

EXAMPLE

For example, a blower motor in an low rise office building where air conditioning presence is unknown:

$$\begin{aligned}\Delta kW &= (241 / 2481) * 0.474 \\ &= 0.05 \text{ kW}\end{aligned}$$

NATURAL GAS ENERGY SAVINGS

Time of Sale:

$$\Delta \text{Therms} = \text{EFLH} * \text{Capacity} * ((\text{AFUE}(\text{eff}) - \text{AFUE}(\text{base})) / \text{AFUE}(\text{base})) / 100,000 \text{ Btu/Therm}$$

Early replacement³⁵³:

Δ Therms for remaining life of existing unit (1st 5.5 years):

$$\Delta \text{Therms} = \text{EFLH} * \text{Capacity} * ((\text{AFUE}(\text{eff}) - \text{AFUE}(\text{exist})) / \text{AFUE}(\text{exist})) / 100,000 \text{ Btu/Therm}$$

Δ Therms for remaining measure life (next 11 years):

$$\Delta \text{Therms} = \text{EFLH} * \text{Capacity} * ((\text{AFUE}(\text{eff}) - \text{AFUE}(\text{base})) / \text{AFUE}(\text{base})) / 100,000 \text{ Btu/Therm}$$

Where:

EFLH	= Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use
Capacity	= Nominal Heating Input Capacity Furnace Size (Btu/hr) for efficient unit not existing unit = custom Furnace input capacity in Btu/hr
AFUE(exist)	= Existing Furnace Annual Fuel Utilization Efficiency Rating = Use actual AFUE rating where it is possible to measure or reasonably estimate. If unknown, assume 64.4 AFUE% ³⁵⁴ .
AFUE(base)	= Baseline Furnace Annual Fuel Utilization Efficiency Rating, dependant on year as listed below:

³⁵³ The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a “number of years to adjustment” and “savings adjustment” input which would be the (new base to efficient savings)/(existing to efficient savings).

³⁵⁴ Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

Dependent on program type as listed below³⁵⁵:

Program Year	AFUE(base)
Time of Sale	80%
Early Replacement	90%

AFUE(eff) = Efficient Furnace Annual Fuel Utilization Efficiency Rating.
 = Actual. If Unknown, assume 95%³⁵⁶

EXAMPLE

$$\begin{aligned}\Delta\text{Therms} &= 1428 * 150,000 * ((0.92-0.80)/0.80)/ 100,000 \\ &= 321 \text{ Therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-FRNC-V07-180101

REVIEW DEADLINE: 1/1/2019

³⁵⁵ Though the Federal Minimum AFUE is 78%, there were only 50 models listed in the AHRI database at that level. At AFUE 79% the total rises to 308. There are 3,548 active furnace models listed with AFUE ratings between 78 and 80.

³⁵⁶ Minimum ENERGY STAR efficiency after 2.1.2012.

4.4.12 Infrared Heaters (all sizes), Low Intensity

DESCRIPTION

This measure applies to natural gas fired low-intensity infrared heaters with an electric ignition that use non-conditioned air for combustion

This measure was developed to be applicable to the following program types: TOS, NC. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a natural gas heater with an electric ignition that uses non-conditioned air for combustion

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is a standard natural gas fired heater warm air heater.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years³⁵⁷

DEEMED MEASURE COST

The incremental capital cost for this measure is \$1716³⁵⁸

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

The annual natural gas energy savings from this measure is a deemed value equaling 451 Therms³⁵⁹

³⁵⁷ENERGY STAR and CEE do not currently provide calculators for this type of equipment therefore deemed values from Nicor Gas were used. Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011

³⁵⁸ibid.

³⁵⁹Nicor Gas Energy Efficiency Plan 2011-2014. Revised Plan Filed Pursuant to Order Docket 10-0562, May 27, 2011. These deemed values should be compared to PY evaluation and revised as necessary.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-IRHT-V01-120601

REVIEW DEADLINE: 1/1/2019

4.4.13 Package Terminal Air Conditioner (PTAC) and Package Terminal Heat Pump (PTHP)

DESCRIPTION

A PTAC is a packaged terminal air conditioner that cools and sometimes provides heat through an electric resistance heater (heat strip). A PTHP is a packaged terminal heat pump. A PTHP uses its compressor year round to heat or cool. In warm weather, it efficiently captures heat from inside your building and pumps it outside for cooling. In cool weather, it captures heat from outdoor air and pumps it into your home, adding heat from electric heat strips as necessary to provide heat.

This measure characterizes:

- a) Time of Sale: the purchase and installation of a new efficient PTAC or PTHP.
- b) Early Replacement: the early removal of an existing PTAC or PTHP from service, prior to its natural end of life, and replacement with a new efficient PTAC or PTHP unit. Savings are calculated between existing unit and efficient unit consumption during the remaining life of the existing unit, and between new baseline unit and efficient unit consumption for the remainder of the measure life. The measure is only valid for non-fuel switching installations – for example replacing a cooling only PTAC with a PTHP can currently not use the TRM.

This measure was developed to be applicable to the following program types: TOS NC, EREP. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be PTACs or PTHPs that exceed baseline efficiencies.

DEFINITION OF BASELINE EQUIPMENT

Time of Sale: the baseline conditions is provided in the Federal Baseline reference table provided below.

Early Replacement: the baseline is the existing PTAC or PTHP for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years.³⁶⁰

Remaining life of existing equipment is assumed to be 5 years³⁶¹

DEEMED MEASURE COST

Time of Sale: The incremental capital cost for this equipment is estimated to be \$84/ton.³⁶²

Early Replacement: The measure cost is the full cost of removing the existing unit and installing a new one. The actual program cost should be used. If unknown assume \$1,047 per ton³⁶³.

The assumed deferred cost (after 5 years) of replacing existing equipment with new baseline unit is assumed to be \$1,039 per ton³⁶⁴. This cost should be discounted to present value using the nominal discount rate.

³⁶⁰ Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007

³⁶¹ Standard assumption of one third of effective useful life.

³⁶² DEER 2008. This assumes that baseline shift from IECC 2012 to IECC 2015 carries the same incremental costs. Values should be verified during evaluation

³⁶³ Based on DCEO – IL PHA Efficient Living Program data.

³⁶⁴ Based on subtracting TOS incremental cost from the DCEO data and incorporating inflation rate of 1.91%.

LOADSHAPE

Loadshape C03 - Commercial Cooling

COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market. Both values provided are based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren.

$$\begin{aligned} CF_{SSP} &= \text{Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)} \\ &= 91.3\%^{365} \end{aligned}$$

$$\begin{aligned} CF_{PJM} &= \text{PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)} \\ &= 47.8\%^{366} \end{aligned}$$

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Electric savings for PTACs and PTHPs should be calculated using the following algorithms

ENERGY SAVINGS

Time of Sale:

$$PTAC \Delta kWh^{367} = \text{Annual kWh Savings}_{cool}$$

$$PTHP \Delta kWh = \text{Annual kWh Savings}_{cool} + \text{Annual kWh Savings}_{heat}$$

$$\text{Annual kWh Savings}_{cool} = (kBtu/hr_{cool}) * [(1/EER_{base}) - (1/EER_{ee})] * EFLH_{cool}$$

$$\text{Annual kWh Savings}_{heat} = (kBtu/hr_{heat})/3.412 * [(1/COP_{base}) - (1/COP_{ee})] * EFLH_{heat}$$

Early Replacement:

$$\Delta kWh \text{ for remaining life of existing unit (1}^{st} \text{ 5 years)} = \text{Annual kWh Savings}_{cool} + \text{Annual kWh Savings}_{heat}$$

$$\text{Annual kWh Savings}_{cool} = (kBtu/hr_{cool}) * [(1/EER_{exist}) - (1/EER_{ee})] * EFLH_{cool}$$

$$\text{Annual kWh Savings}_{heat} = (kBtu/hr_{heat})/3.412 * [(1/COP_{exist}) - (1/COP_{ee})] * EFLH_{heat}$$

$$\Delta kWh \text{ for remaining measure life (next 10 years)} = \text{Annual kWh Savings}_{cool} + \text{Annual kWh Savings}_{heat}$$

$$\text{Annual kWh Savings}_{cool} = (kBtu/hr_{cool}) * [(1/EER_{base}) - (1/EER_{ee})] * EFLH_{cool}$$

$$\text{Annual kWh Savings}_{heat} = (kBtu/hr_{heat})/3.412 * [(1/COP_{base}) - (1/COP_{ee})] * EFLH_{heat}$$

Where:

³⁶⁵ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

³⁶⁶ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

³⁶⁷ There are no heating efficiency improvements for PTACs since although some do provide heating, it is always through electric resistance and therefore the COPbase and COPee would be 1.0.

kBtu/hr _{cool}	= capacity of the cooling equipment in kBtu per hour (1 ton of cooling capacity equals 12 kBtu/hr).
	= Actual installed
EFLH _{cool}	= Equivalent Full Load Hours for cooling are provided in section 4.4 HVAC End Use:
EFLH _{heat}	= Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use
EER _{exist}	= Energy Efficiency Ratio of the existing equipment
	= Actual. If unknown assume 8.1 EER ³⁶⁸
EER _{base}	= Energy Efficiency Ratio of the baseline equipment; see the table below for values.
	= Based on applicable IECC code on date of building permit (if unknown assume IECC 2015).

Copy of Table C403.2.3(3): Minimum Efficiency Requirements: Electrically operated packaged terminal air conditioners, packaged terminal heat pumps

Equipment Type	IECC 2012 Minimum Efficiency	IECC 2015 Minimum Efficiency
PTAC (Cooling mode) New Construction	13.8 – (0.300 x Cap/1000) EER	14.0 – (0.300 x Cap/1000) EER
PTAC (Cooling mode) Replacements	10.9 – (0.213 x Cap/1000) EER	10.9 – (0.213 x Cap/1000) EER
PTHP (Cooling mode) New Construction	14.0 – (0.300 x Cap/1000) EER	14.0 – (0.300 x Cap/1000) EER
PTHP (Cooling mode) Replacements	10.8 – (0.213 x Cap/1000) EER	10.8 – (0.213 x Cap/1000) EER
PTHP (Heating mode) New Construction	3.2 – (0.026 x Cap/1000) COP	3.2 – (0.026 x Cap/1000) COP
PTHP (Heating mode) Replacements	2.9 – (0.026 x Cap/1000) COP	2.9 – (0.026 x Cap/1000) COP

“Cap” = The rated cooling capacity of the project in Btu/hr. If the units capacity is less than 7000 Btu/hr, use 7,000 Btu/hr in the calculation. If the unit’s capacity is greater than 15,000 Btu/hr, use 15,000 Btu/hr in the calculations.

Replacement unit shall be factory labeled as follows “MANUFACTURED FOR REPLACEMENT APPLICATIONS ONLY; NOT TO BE INSTALLED IN NEW CONSTRUCTION PROJECTS”, Replacement efficiencies apply only to units with existing sleeves less than 16 inches (406mm) in height and less than 42 inches (1067 mm) in width.

EER _{ee}	= Energy Efficiency Ratio of the energy efficient equipment. For air-cooled units < 65 kBtu/hr, if the actual EER _{ee} is unknown, assume the following conversion from SEER to EER for calculation of peak savings ³⁶⁹ : $EER = (-0.02 * SEER^2) + (1.12 * SEER)$
	= Actual installed
kBtu/hr _{heat}	= capacity of the heating equipment in kBtu per hour.
	= Actual installed

³⁶⁸ Estimated using the IECC building energy code up until year 2003 (p107; <https://law.resource.org/pub/us/code/ibr/icc.iecc.2000.pdf>) and assuming a 1 ton unit; $EER = 10 - (0.16 * 12,000/1,000) = 8.1$.

³⁶⁹ Based on Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder. Note this is appropriate for single speed units only.

3.412	= Btu per Wh.
COP _{exist}	= coefficient of performance of the existing equipment = Actual. If unknown assume 1.0 COP for PTAC units and 2.6 COP ³⁷⁰ for PTHPs.
COP _{base}	= coefficient of performance of the baseline equipment; see table above for values.
COP _{ee}	= coefficient of performance of the energy efficient equipment. = Actual installed

EXAMPLE:

Time of Sale (assuming new construction baseline):

For example a 1 ton PTAC with an efficient EER of 12 at a guest hotel in Rockford with a building permit dated before 1/1/2016 saves:

$$= [(12) * [(1/10.2) - (1/12)] * 1,042]$$

$$= 184 \text{ kWh}$$

Early Replacement (assuming replacement baseline for deferred replacement in 5 years):

For example a 1 ton PTHP with an efficient EER of 12, COP of 3.0 at a guest hotel in Rockford replaces a PTAC unit (with electric resistance heat) with unknown efficiency.

Δ kWh for remaining life of existing unit (1st 5years)

$$= (12 * (1/8.1 - 1/12) * 1,042) + (12/3.412 * (1/1.0 - 1/3.0) * 1,758)$$

$$= 502 + 4,122$$

$$= 4,624 \text{ kWh}$$

Δ kWh for remaining measure life (next 10 years)

$$= (12 * (1/8.3 - 1/12) * 1,042) + (12/3.412 * (1/1.0 - 1/3.0) * 1,758)$$

$$= 465 + 4,122$$

$$= 34,587 \text{ kWh}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

Time of Sale:

$$\Delta \text{kW} = (\text{kBtu/hr}_{\text{cool}}) * [(1/\text{EER}_{\text{base}}) - (1/\text{EER}_{\text{ee}})] * \text{CF}$$

Early Replacement:

$$\Delta \text{kW for remaining life of existing unit (1}^{\text{st}} \text{ 5years)} = (\text{kBtu/hr}_{\text{cool}}) * [(1/\text{EER}_{\text{exist}}) - (1/\text{EER}_{\text{ee}})] * \text{CF}$$

$$\Delta \text{kW for remaining measure life (next 10 years)} = (\text{kBtu/hr}_{\text{cool}}) * [(1/\text{EER}_{\text{base}}) - (1/\text{EER}_{\text{ee}})] * \text{CF}$$

Where:

$$\text{CF}_{\text{SSP}} = \text{Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)}$$

³⁷⁰Estimated using the IECC building energy code up until year 2003 (p107; <https://law.resource.org/pub/us/code/ibr/icc.iecc.2000.pdf>) and assuming a 1 ton unit; $\text{COP} = 2.9 - (0.026 * 12,000/1,000) = 2.6$

$$= 91.3\%^{371}$$

$$\begin{aligned} CF_{PJM} &= \text{PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)} \\ &= 47.8\%^{372} \end{aligned}$$

EXAMPLE

Time of Sale:

For example a 1 ton replacement cooling unit with no heating with an efficient EER of 12 saves:

$$\begin{aligned} \Delta kW_{SSP} &= (12 * (1/10.2 - 1/12) * 0.913 \\ &= 0.16 \text{ kW} \end{aligned}$$

For example a 1 ton PTHP with an efficient EER of 12, COP of 3.0 replacing a PTAC unit with unknown efficiency saves:

ΔkW for remaining life of existing unit (1st 5years):

$$\begin{aligned} \Delta kW_{SSP} &= 12 * (1/8.1 - 1/12) * 0.913 \\ &= 0.44 \text{ kW} \end{aligned}$$

ΔkW for remaining measure life (next 10 years):

$$\begin{aligned} \Delta kW_{SSP} &= 12 * (1/8.3 - 1/12) * 0.913 \\ &= 0.41 \text{ kW} \end{aligned}$$

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-PTAC-V08-180101

REVIEW DEADLINE: 1/1/2020

³⁷¹ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

³⁷² Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

4.4.14 Pipe Insulation

DESCRIPTION

This measure provides rebates for installation of $\geq 1"$ or $\geq 2"$ fiberglass, foam, calcium silicate or other types of insulation with similar insulating properties to existing bare pipe on straight piping as well as other pipe components such as elbows, tees, valves, and flanges for all non-residential installations.

Default per linear foot savings estimates are provided for the both exposed indoor or above ground outdoor piping distributing fluid in the following system types (natural gas fired systems only):

- Hydronic heating systems (with or without outdoor reset controls), including:
 - boiler systems that do not circulate water around a central loop and operate upon demand from a thermostat ("non-recirculation")
 - systems that recirculate during heating season only ("Recirculation – heating season only")
 - systems recirculating year round ("Recirculation – year round")
- Domestic hot water
- Low and high-pressure steam systems
 - non-recirculation
 - recirculation - heating season only
 - recirculation - year round

Process piping can also use the algorithms provided but requires custom entry of hours.

Minimum qualifying nominal pipe diameter is 1." Indoor piping must have at least 1" of insulation and outdoor piping must have at least 2" of insulation and include an all-weather protective jacket. New advanced insulating materials may be thinner and savings can be calculated with 3E Plus.

This measure was developed to be applicable to the following program types: RF, DI

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient case is installing pipe wrap insulation to a length of pipe. Indoor piping must have at least 1" of insulation (or equivalent R-value) and outdoor piping must have at least 2" of insulation (or equivalent R-value) and include an all-weather protective jacket. Minimum qualifying pipe diameter is 1." Insulation must be continuous and contiguous over fittings that directly connect to straight pipe, including elbows and tees.³⁷³

DEFINITION OF BASELINE EQUIPMENT

The base case for savings estimates is a bare pipe. Pipes are required by new construction code to be insulated but are still commonly found uninsulated in older commercial buildings.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 15 years.³⁷⁴

³⁷³ ASHRAE Handbook—Fundamentals, 23.14; Hart, G., "Saving energy by insulating pipe components on steam and hot water distribution systems", *ASHRAE Journal*, October 2011

³⁷⁴ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf

DEEMED MEASURE COST

Actual costs should be used if known. Otherwise the deemed measure costs below based on RS Means³⁷⁵ pricing reference materials may be used.³⁷⁶ The following table summarizes the estimated costs for this measure per foot of insulation added and include installation costs:

Insulation Thickness		
	1 Inch (Indoor)	2 Inches (Outdoor)
Pipe- RS Means #	220719.10.5170	220719.10.5530
Jacket- RS Means #	220719.10.0156	220719.10.0320
Jacket Type	PVC	Aluminum
Insulation Cost per foot	\$9.40	\$13.90
Jacket Cost per foot	\$4.57	\$7.30
Total Cost per foot	\$13.97	\$21.20

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS SAVINGS

$$\Delta \text{therms per foot}^{377} = [((Q_{\text{base}} - Q_{\text{eff}}) * \text{EFLH}) / (100,000 * \eta_{\text{Boiler}})] * \text{TRF}$$

$$= [\text{Modeled or provided by tables below}] * \text{TRF}$$

$$\Delta \text{therms} = (L_{\text{sp}} + L_{\text{oc,i}}) * \Delta \text{therms per foot}$$

Where:

EFLH = Equivalent Full Load Hours for Heating
 = Actual or defaults by building type provided in Section 4.4, HVAC end use

For year round recirculation or domestic hot water:

$$= 8,766$$

For heating season recirculation, hours with the outside air temperature below 55°F:

³⁷⁵ RS Means 2008. Mechanical Cost Data, pages 106 to 119

³⁷⁶ RS Means 2010: "for fittings, add 3 linear feet for each fitting plus 4 linear feet for each flange of the fitting"

³⁷⁷ This value comes from the reference table "Savings Summary by Building Type and System Type." The formula and the input tables in this section document assumptions used in calculation spreadsheet "Pipe Insulation Savings 2013-11-12.xlsx"

Zone	Hours
Zone 1 (Rockford)	5,039
Zone 2 (Chicago)	4,963
Zone 3 (Springfield)	4,495
Zone 4 (Belleville/	4,021
Zone 5 (Marion)	4,150
Zone 1 (Rockford)	5,039

Q_{base}	<p>= Heat Loss from Bare Pipe (Btu/hr/ft)</p> <p>= Calculated where possible using 3E Plusv4.0 software. For defaults see table below</p>
Q_{eff}	<p>= Heat Loss from Insulated Pipe (Btu/hr/ft)</p> <p>= Calculated where possible using 3E Plusv4.0 software. For defaults see table below</p>
100,000	= conversion factor (1 therm = 100,000 Btu)
η_{Boiler}	<p>= Efficiency of the boiler being used to generate the hot water or steam in the pipe</p> <p>= Actual or if unknown use default values given below:</p> <p>= 81.9% for water boilers ³⁷⁸</p> <p>= 80.7% for steam boilers, except multifamily low-pressure ³⁷⁹</p> <p>= 64.8% for multifamily low-pressure steam boilers ³⁸⁰</p>
TRF	<p>= Thermal Regain Factor for space type, applied only to space heating energy and is applied to values resulting from Δtherms/ft tables below ³⁸¹</p> <p>= See table below for base TRF values by pipe location</p> <p>May vary seasonally such as: TRF[summer] * summer hours + TRF[winter] * winter hours where TRF values reflecting summer and winter conditions are apportioned by the hours for those conditions. TRF may also be adjusted by building specific balance temperature and operating hours above and below that balance temperature. ³⁸²</p>

Pipe Location	Assumed Regain	TRF, Thermal Regain Factor
Outdoor	0%	1.0
Indoor, heated space	85%	0.15
Indoor, semi- heated, (unconditioned space, with heat transfer to conditioned space. E.g.: boiler room, ceiling plenum, basement, crawlspace, wall)	30%	0.70

³⁷⁸ Average efficiencies of units from the California Energy Commission (CEC).

³⁷⁹ Ibid.

³⁸⁰ Katrakis, J. and T.S. Zawacki. "Field-Measured Seasonal Efficiency of Intermediate-sized Low-Pressure Steam Boilers". ASHRAE V99, pt. 2, 1993.

³⁸¹ Thermal regain for *residential* pipe insulation measures is discussed in Home Energy Services Impact Evaluation, prepared for the Massachusetts Residential Retrofit and Low Income Program Area Evaluation, Cadmus Group, Inc., August 2012 and Andrews, John, Better Duct Systems for Home Heating and Cooling, U.S. Department of Energy, 2001. Recognizing the differences between residential and commercial heating systems, the factors have been adjusted based on professional judgment. This factor would benefit from additional study and evaluation.

³⁸² Thermal Regain Factor_4-30-14.docx

Pipe Location	Assumed Regain	TRF, Thermal Regain Factor
Indoor, unheated, (no heat transfer to conditioned space)	0%	1.0
Location not specified	85%	0.15
Custom	Custom	1 – assumed regain

L_{sp} = Length of straight pipe to be insulated (linear foot)

= actual installed ((linear foot)

$L_{oc,l}$ = Total equivalent length of the other components (valves and tees) of pipe to be insulated

= Actual installed (linear foot). See table “Equivalent Length of Other Components – Elbows and Tees” for equivalent lengths.

The heat loss estimates (Q_{base} and Q_{eff}) were developed using the 3E Plus v4.0 software program.³⁸³ The energy savings analysis is based on adding 1-inch (indoor) or 2-inch (outdoor) thick insulation around bare pipe. The thermal conductivity of pipe insulation varies by material and temperature rating; to obtain a typical value, a range of materials allowed for this measure were averaged. For insulation materials not in the table below, use 3E Plusv4.0 software to calculate Q_{base} and Q_{eff} .

Insulation Type	Conductivity (Btu.in / hr.ft ² .°F @ 75F)	Max temp (°F)
Polyethylene foam	0.25	200
Flexible polyurethane-based foam	0.27	200
Fiberglass	0.31	250
Melamine foam	0.26	350
Flexible silicon foam	0.40	392
Calcium silicate	0.40	1200
Cellular glass	0.31	400
Average conductivity of all these materials (Btu.in / hr.ft ² .°F @ 75°F)	0.31	

The pipe fluid temperature assumption used depends upon both the system type and whether there is outdoor reset controls:

System Type	Fluid temperature assumption (°F)
Hot Water space heating with outdoor reset - Non recirculation	145
Hot Water space heating without outdoor reset - Non recirculation	170
Hot Water space heating with outdoor reset – Recirculation heating season only	145
Hot Water space heating without outdoor reset – Recirculation heating season only	170
Hot Water space heating with outdoor reset – Recirculation year round	130
Hot Water space heating without outdoor reset – Recirculation year round	170
Domestic Hot Water	125
Low Pressure Steam	225
High Pressure Steam	312

³⁸³ 3E Plus is a heat loss calculation software provided by the NAIMA (North American Insulation Manufacturer Association).

	Indoor Insulation, Hot Water	Indoor Insulation, Low Pressure Steam	Indoor Insulation, High Pressure Steam	Domestic Hot Water	Outdoor Insulation, Hot Water	Outdoor Insulation, Low Pressure Steam	Outdoor Insulation, High Pressure Steam
Insulation thickness (inch)	1	1	1	1	2	2	2
Temperature, Fluid in Pipe (°F)	170 (w/o reset) 145 (w/ reset heat) 130 (w/reset year)	225	312	125	170 (w/o reset) 145 (w/ reset heat) 130 (w/reset year)	225	312
Av. steam pressure (psig)	n/a	10.9	82.8	n/a	n/a	10.9	82.8
Operating Time (hrs/yr)	2,746 (non-recirc) 5,039 (recirc heating season) 8,760 (recirc year round)						
Ambient Temperature (°F) ³⁸⁴	75	75	75	75	48.6	48.6	48.6
Wind speed (mph) ³⁸⁵	0	0	0	0	9.4	9.4	9.4
Pipe parameters							
Pipe material	Copper	Steel	Steel	Copper	Copper	Steel	Steel
Pipe size for Heat Loss Calc	2"	2"	2"	2"	2"	2"	2"
Outer Diameter, Pipe, actual	2.38"	2.38"	2.38"	2.38"	2.38"	2.38"	2.38"
Heat Loss, Bare Pipe (from 3EPlus) (Btu/hr.ft)	114 (w/o reset) 78 (w/ reset heat) 58 (w/reset year)	232	432	52	460 (w/o reset) 363 (w/ reset heat) 306 (w/reset year)	710	1101
Insulation parameters							
Outer diameter, insulation	4.38"	4.38"	4.38"	4.38"	4.38"	4.38"	4.38"
Average Heat Loss, Insulation (from 3EPlus) (Btu/hr.ft)	24 (w/o reset) 17 (w/ reset heat) 13 (w/reset year)	40	70	13.25	21 (w/o reset) 16 (w/ reset heat) 13 (w/reset year)	32	52
Annual Energy Savings							
Boiler / Water Heater efficiency	81.9%	80.7% (64.8% for MF)	80.7%	67%	81.9%	80.7% (64.8% for MF)	80.7%
Annual Gas Use, Base Case (therms/yr/ft)	3.8 (w/o reset) 4.8 (w/ reset heat) 6.2 (w/reset year)	7.9 (non recirc) 14.5 (recirc heat) 25.2 (recirc year)	14.7 (non recirc) 27.0 (recirc heat) 46.9 (recirc year)	6.76	15.4 (w/o reset) 22.5 (w/ reset heat) 32.7 (w/reset year)	24.1 (non recirc) 44.3 (recirc heat) 77.0 (recirc year)	37.5 (non recirc) 68.7 (recirc heat) 119.5 (recirc year)
Annual Gas Use, Measure case (therms/yr/ft)	0.8 (w/o reset) 1.1 (w/ reset heat) 1.4 (w/reset year)	1.4 (non recirc) 2.5 (recirc heat) 4.4 (recirc year)	2.4 (non recirc) 4.4 (recirc heat) 7.6 (recirc year)	1.73	0.7 (w/o reset) 1.0 (w/ reset heat) 1.4 (w/reset year)	1.1 (non recirc) 2.0 (recirc heat) 3.4 (recirc year)	1.8 (non recirc) 3.2 (recirc heat) 5.6 (recirc year)
Annual Gas Savings (therms/yr/ft)	3.0 (w/o reset) 3.7 (w/ reset heat) 4.8 (w/reset year)	6.5 (non recirc) 12.0 (recirc heat) 20.8 (recirc year)	12.3 (non recirc) 22.6 (recirc heat) 39.3 (recirc year)	5.0	14.7 (w/o reset) 21.4 (w/ reset heat) 31.3 (w/reset year)	23.1 (non recirc) 42.3 (recirc heat) 73.6 (recirc year)	35.7 (non recirc) 65.5 (recirc heat) 113.9 (recirc year)

Heat = heating season only, year = year round

³⁸⁴ DOE Weather Data.

http://apps1.eere.energy.gov/buildings/energyplus/weatherdata/4_north_and_central_america_wmo_region_4/1_usa/USA_IL_Aurora.Muni.AP.744655_TMY3.stat Ibid.

³⁸⁵ Ibid.

Values below must be multiplied by the appropriate Thermal Regain Factor (TRF). All variables were the same except for hours of operation in the calculation of the default savings per foot for the various building types and applications as presented in the table below:

Savings Summary for Indoor pipe insulation by System Type and Building Type (Δtherms per foot)
(continues for 3.5 pages)

Location	System Type	Building Type	Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam)				
			Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
Indoor	Hot Water Space Heating with outdoor reset – non-recirculation	Assembly	1.32	1.36	1.21	0.81	1.24
		Assisted Living	1.25	1.22	1.07	0.79	0.95
		College	1.13	1.06	0.95	0.53	0.63
		Convenience Store	1.10	1.01	0.90	0.65	0.72
		Elementary School	1.32	1.29	1.13	0.78	0.95
		Garage	0.73	0.72	0.63	0.50	0.56
		Grocery	1.19	1.19	1.04	0.65	0.78
		Healthcare Clinic	1.17	1.20	1.05	0.71	0.75
		High School	1.37	1.38	1.23	0.88	1.03
		Hospital - CAV no econ	1.31	1.35	1.15	0.99	1.12
		Hospital - CAV econ	1.33	1.37	1.17	1.01	1.15
		Hospital - VAV econ	0.54	0.51	0.39	0.23	0.25
		Hospital - FCU	0.98	1.12	0.91	1.07	1.44
		Hotel/Motel	1.31	1.27	1.14	0.78	0.96
		Hotel/Motel - Common	1.19	1.21	1.15	0.93	0.98
		Hotel/Motel - Guest	1.30	1.26	1.13	0.75	0.93
		Manufacturing Facility	0.78	0.75	0.70	0.42	0.47
		MF - High Rise	1.13	1.12	1.02	0.87	0.87
		MF - High Rise - Common	1.35	1.31	1.17	0.81	1.04
		MF - High Rise - Residential	1.09	1.08	0.99	0.85	0.83
		MF - Mid Rise	1.23	1.25	1.07	0.79	0.90
		Movie Theater	1.35	1.33	1.24	0.94	1.12
		Office - High Rise - CAV no econ	1.50	1.52	1.38	0.93	1.01
		Office - High Rise - CAV econ	1.55	1.58	1.45	1.00	1.10
		Office - High Rise - VAV econ	1.13	1.15	0.95	0.56	0.63
		Office - High Rise - FCU	0.83	0.82	0.71	0.37	0.39
		Office - Low Rise	1.06	1.06	0.84	0.51	0.59
		Office - Mid Rise	1.17	1.18	0.99	0.63	0.70
		Religious Building	1.19	1.11	1.07	0.78	0.89
		Restaurant	1.00	1.00	0.90	0.68	0.81
		Retail - Department Store	1.03	0.95	0.89	0.58	0.66
		Retail - Strip Mall	0.99	0.91	0.81	0.56	0.60
		Warehouse	1.08	1.01	1.04	0.65	0.80
		Unknown	1.15	1.14	1.01	0.73	0.84
	Hot Water Space Heating without outdoor reset – non-recirculation	Assembly	1.96	2.00	1.79	1.19	1.83
		Assisted Living	1.84	1.80	1.58	1.16	1.40
		College	1.67	1.56	1.40	0.78	0.93
		Convenience Store	1.62	1.50	1.33	0.95	1.06
		Elementary School	1.95	1.90	1.68	1.16	1.40
		Garage	1.08	1.06	0.93	0.74	0.82

Location	System Type	Building Type	Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam)				
			Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
		Grocery	1.76	1.75	1.54	0.96	1.15
		Healthcare Clinic	1.73	1.77	1.55	1.05	1.11
		High School	2.02	2.03	1.82	1.30	1.52
		Hospital - CAV no econ	1.93	1.99	1.69	1.46	1.65
		Hospital - CAV econ	1.96	2.03	1.73	1.50	1.70
		Hospital - VAV econ	0.80	0.76	0.57	0.34	0.37
		Hospital - FCU	1.45	1.65	1.35	1.58	2.13
		Hotel/Motel	1.93	1.87	1.69	1.16	1.41
		Hotel/Motel - Common	1.75	1.78	1.69	1.38	1.45
		Hotel/Motel - Guest	1.92	1.86	1.66	1.11	1.37
		Manufacturing Facility	1.15	1.11	1.03	0.62	0.69
		MF - High Rise	1.67	1.65	1.50	1.28	1.28
		MF - High Rise - Common	1.99	1.93	1.73	1.19	1.54
		MF - High Rise - Residential	1.61	1.60	1.46	1.26	1.23
		MF - Mid Rise	1.82	1.84	1.59	1.17	1.33
		Movie Theater	1.99	1.96	1.83	1.39	1.66
		Office - High Rise - CAV no econ	2.21	2.24	2.04	1.37	1.49
		Office - High Rise - CAV econ	2.29	2.33	2.14	1.48	1.63
		Office - High Rise - VAV econ	1.67	1.70	1.40	0.83	0.93
		Office - High Rise - FCU	1.22	1.21	1.04	0.55	0.58
		Office - Low Rise	1.56	1.56	1.24	0.76	0.87
		Office - Mid Rise	1.73	1.74	1.47	0.94	1.04
		Religious Building	1.75	1.65	1.58	1.15	1.32
		Restaurant	1.48	1.48	1.33	1.01	1.19
		Retail - Department Store	1.52	1.40	1.31	0.85	0.97
		Retail - Strip Mall	1.46	1.35	1.19	0.82	0.89
		Warehouse	1.59	1.49	1.53	0.96	1.18
		Unknown	1.70	1.68	1.50	1.07	1.25
	Hot Water with outdoor reset	All buildings, Recirculation heating season only (Hours below 55F)	3.73	3.68	3.33	2.98	3.08
	Hot Water w/o outdoor reset	All buildings, Recirculation heating season only (Hours below 55F)	5.51	5.43	4.92	4.40	4.54
	Hot Water with outdoor reset	All buildings, Recirculation year round (All hours)	4.79	4.79	4.79	4.79	4.79
	Hot Water w/o outdoor reset	All buildings, Recirculation year round (All hours)	9.58	9.58	9.58	9.58	9.58
	Domestic Hot Water	DHW circulation loop	5.02	5.02	5.02	5.02	5.02
	LP Steam – non-recirculation	Assembly	4.25	4.36	3.89	2.59	3.97
		Assisted Living	4.01	3.92	3.44	2.53	3.04
		College	3.64	3.40	3.04	1.69	2.02
		Convenience Store	3.52	3.26	2.89	2.07	2.32
		Elementary School	4.24	4.13	3.64	2.52	3.05
		Garage	2.34	2.31	2.03	1.62	1.79
		Grocery	3.83	3.81	3.34	2.08	2.49
		Healthcare Clinic	3.76	3.85	3.36	2.29	2.42

Location	System Type	Building Type	Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam)				
			Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
		High School	4.39	4.42	3.96	2.82	3.30
		Hospital - CAV no econ	4.20	4.33	3.69	3.17	3.60
		Hospital - CAV econ	4.25	4.41	3.76	3.26	3.70
		Hospital - VAV econ	1.74	1.65	1.24	0.75	0.81
		Hospital - FCU	3.15	3.60	2.93	3.44	4.63
		Hotel/Motel	4.19	4.07	3.67	2.51	3.07
		Hotel/Motel - Common	3.81	3.87	3.68	3.00	3.15
		Hotel/Motel - Guest	4.18	4.05	3.62	2.42	2.98
		Manufacturing Facility	2.49	2.41	2.23	1.35	1.51
		MF - High Rise	4.52	4.46	4.07	3.46	3.47
		MF - High Rise - Common	5.38	5.22	4.68	3.23	4.17
		MF - High Rise - Residential	4.37	4.34	3.94	3.41	3.33
		MF - Mid Rise	4.94	4.99	4.30	3.16	3.60
		Movie Theater	4.33	4.26	3.98	3.03	3.61
		Office - High Rise - CAV no econ	4.81	4.88	4.45	2.98	3.24
		Office - High Rise - CAV econ	4.97	5.07	4.66	3.21	3.54
		Office - High Rise - VAV econ	3.64	3.71	3.06	1.81	2.01
		Office - High Rise - FCU	2.66	2.62	2.27	1.20	1.26
		Office - Low Rise	3.40	3.39	2.69	1.65	1.89
		Office - Mid Rise	3.77	3.78	3.19	2.03	2.26
		Religious Building	3.82	3.58	3.43	2.51	2.87
		Restaurant	3.21	3.22	2.89	2.19	2.60
		Retail - Department Store	3.31	3.04	2.86	1.86	2.12
		Retail - Strip Mall	3.17	2.94	2.59	1.79	1.93
		Warehouse	3.46	3.23	3.33	2.08	2.56
		Unknown	3.70	3.66	3.26	2.34	2.71
	LP Steam	All buildings, Recirculation heating season only (Hours below 55F)	11.99	11.81	10.70	9.57	9.88
	LP Steam	All buildings, Recirculation year round (All hours)	20.84	20.84	20.84	20.84	20.84
	HP Steam – non-recirculation	Assembly	8.02	8.22	7.34	4.89	7.49
		Assisted Living	7.56	7.39	6.49	4.77	5.73
		College	6.87	6.42	5.73	3.18	3.81
		Convenience Store	6.65	6.14	5.45	3.91	4.37
		Elementary School	8.00	7.79	6.87	4.75	5.76
		Garage	4.42	4.35	3.82	3.05	3.38
		Grocery	7.22	7.19	6.30	3.93	4.70
		Healthcare Clinic	7.09	7.27	6.35	4.32	4.57
		High School	8.28	8.34	7.48	5.33	6.23
		Hospital - CAV no econ	7.92	8.16	6.95	5.98	6.79
		Hospital - CAV econ	8.03	8.32	7.09	6.14	6.98
		Hospital - VAV econ	3.28	3.12	2.35	1.41	1.53
		Hospital - FCU	5.95	6.79	5.53	6.50	8.73
		Hotel/Motel	7.91	7.69	6.93	4.74	5.79
		Hotel/Motel - Common	7.18	7.30	6.95	5.65	5.94
		Hotel/Motel - Guest	7.89	7.64	6.83	4.57	5.62

Location	System Type	Building Type	Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam)				
			Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
		Manufacturing Facility	4.70	4.55	4.22	2.55	2.84
		MF - High Rise	6.85	6.76	6.16	5.25	5.26
		MF - High Rise - Common	8.15	7.91	7.09	4.89	6.31
		MF - High Rise - Residential	6.62	6.57	5.97	5.17	5.04
		MF - Mid Rise	7.48	7.57	6.51	4.79	5.46
		Movie Theater	8.16	8.04	7.52	5.71	6.80
		Office - High Rise - CAV no econ	9.07	9.20	8.39	5.62	6.12
		Office - High Rise - CAV econ	9.38	9.57	8.80	6.06	6.67
		Office - High Rise - VAV econ	6.86	6.99	5.76	3.41	3.80
		Office - High Rise - FCU	5.02	4.95	4.27	2.27	2.38
		Office - Low Rise	6.41	6.40	5.08	3.11	3.56
		Office - Mid Rise	7.12	7.12	6.03	3.84	4.27
		Religious Building	7.20	6.75	6.46	4.73	5.41
		Restaurant	6.06	6.08	5.46	4.13	4.90
		Retail - Department Store	6.25	5.74	5.39	3.51	4.00
		Retail - Strip Mall	5.98	5.54	4.89	3.37	3.63
		Warehouse	6.53	6.09	6.29	3.93	4.84
		Unknown	6.97	6.91	6.14	4.41	5.11
	HP Steam	All buildings, Recirculation heating season only (Hours below 55F)	22.62	22.28	20.18	18.05	18.63
	HP Steam	All buildings, Recirculation year round (All hours)	39.32	39.32	39.32	39.32	39.32

**Savings Summary for Outdoor pipe insulation by System Type and Building Type (Δ therms per foot)
(continues for 3.5 pages)**

Location	System Type	Building Type	Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam)				
			Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
Outdoor	Hot Water Space Heating with outdoor reset – non-recirculation	Assembly	7.58	7.77	6.94	4.62	7.08
		Assisted Living	7.14	6.98	6.13	4.51	5.42
		College	6.49	6.07	5.41	3.01	3.60
		Convenience Store	6.28	5.80	5.15	3.70	4.13
		Elementary School	7.56	7.36	6.50	4.49	5.44
		Garage	4.18	4.11	3.61	2.88	3.19
		Grocery	6.82	6.80	5.96	3.72	4.44
		Healthcare Clinic	6.70	6.87	6.00	4.09	4.32
		High School	7.83	7.88	7.07	5.03	5.89
		Hospital - CAV no econ	7.49	7.71	6.57	5.65	6.41
		Hospital - CAV econ	7.59	7.86	6.70	5.81	6.60
		Hospital - VAV econ	3.10	2.95	2.22	1.33	1.44
		Hospital - FCU	5.62	6.42	5.23	6.14	8.26
		Hotel/Motel	7.47	7.26	6.55	4.48	5.47
		Hotel/Motel - Common	6.79	6.90	6.57	5.34	5.61

Location	System Type	Building Type	Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam)				
			Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
		Hotel/Motel - Guest	7.46	7.22	6.45	4.32	5.31
		Manufacturing Facility	4.45	4.30	3.98	2.41	2.69
		MF - High Rise	6.48	6.39	5.83	4.96	4.97
		MF - High Rise - Common	7.70	7.48	6.70	4.62	5.96
		MF - High Rise - Residential	6.26	6.21	5.64	4.89	4.77
		MF - Mid Rise	7.07	7.15	6.15	4.53	5.16
		Movie Theater	7.71	7.60	7.10	5.40	6.43
		Office - High Rise - CAV no econ	8.57	8.70	7.93	5.31	5.78
		Office - High Rise - CAV econ	8.86	9.04	8.32	5.73	6.31
		Office - High Rise - VAV econ	6.48	6.61	5.45	3.22	3.59
		Office - High Rise - FCU	4.75	4.67	4.04	2.14	2.25
		Office - Low Rise	6.06	6.05	4.80	2.94	3.36
		Office - Mid Rise	6.73	6.73	5.70	3.63	4.03
		Religious Building	6.80	6.38	6.11	4.47	5.11
		Restaurant	5.73	5.75	5.16	3.90	4.63
		Retail - Department Store	5.91	5.42	5.09	3.31	3.78
		Retail - Strip Mall	5.65	5.23	4.62	3.19	3.44
		Warehouse	6.18	5.76	5.94	3.71	4.57
		Unknown	6.59	6.53	5.81	4.17	4.83
	Hot Water Space Heating without outdoor reset – non-recirculation	Assembly	9.59	9.83	8.77	5.85	8.96
		Assisted Living	9.04	8.83	7.76	5.70	6.86
		College	8.21	7.68	6.85	3.80	4.56
		Convenience Store	7.95	7.34	6.52	4.68	5.22
		Elementary School	9.56	9.32	8.22	5.68	6.89
		Garage	5.28	5.20	4.57	3.65	4.04
		Grocery	8.63	8.60	7.54	4.70	5.62
		Healthcare Clinic	8.47	8.70	7.59	5.17	5.47
		High School	9.90	9.97	8.94	6.37	7.45
		Hospital - CAV no econ	9.47	9.76	8.31	7.15	8.11
		Hospital - CAV econ	9.60	9.95	8.48	7.35	8.34
		Hospital - VAV econ	3.93	3.73	2.80	1.68	1.82
		Hospital - FCU	7.11	8.12	6.61	7.77	10.45
		Hotel/Motel	9.45	9.19	8.29	5.67	6.92
		Hotel/Motel - Common	8.59	8.73	8.31	6.76	7.10
		Hotel/Motel - Guest	9.44	9.13	8.16	5.47	6.72
		Manufacturing Facility	5.63	5.44	5.04	3.05	3.40
		MF - High Rise	8.19	8.08	7.37	6.27	6.29
		MF - High Rise - Common	9.74	9.46	8.48	5.85	7.54
		MF - High Rise - Residential	7.92	7.86	7.14	6.18	6.03
		MF - Mid Rise	8.94	9.05	7.78	5.73	6.53
		Movie Theater	9.76	9.61	8.99	6.83	8.14
		Office - High Rise - CAV no econ	10.84	11.01	10.03	6.72	7.32
		Office - High Rise - CAV econ	11.21	11.44	10.52	7.25	7.98
		Office - High Rise - VAV econ	8.20	8.36	6.89	4.07	4.54
		Office - High Rise - FCU	6.00	5.91	5.11	2.71	2.84
		Office - Low Rise	7.67	7.65	6.08	3.72	4.25

Location	System Type	Building Type	Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam)				
			Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
		Office - Mid Rise	8.51	8.52	7.21	4.59	5.10
		Religious Building	8.61	8.07	7.73	5.66	6.47
		Restaurant	7.25	7.27	6.53	4.94	5.85
		Retail - Department Store	7.47	6.86	6.44	4.19	4.78
		Retail - Strip Mall	7.15	6.62	5.85	4.03	4.35
		Warehouse	7.81	7.29	7.52	4.69	5.78
		Unknown	8.34	8.26	7.35	5.27	6.11
	Hot Water with outdoor reset	All buildings, Recirculation heating season only (Hours below 55F)	21.38	21.06	19.07	17.06	17.61
	Hot Water without outdoor reset	All buildings, Recirculation heating season only (Hours below 55F)	27.05	26.64	24.13	21.58	22.28
	Hot Water with outdoor reset	All buildings, Recirculation year round (All hours)	31.30	31.30	31.30	31.30	31.30
	Hot Water without outdoor reset	All buildings, Recirculation year round (All hours)	47.02	47.02	47.02	47.02	47.02
	LP Steam – non-recirculation	Assembly	15.01	15.38	13.73	9.15	14.02
		Assisted Living	14.14	13.82	12.15	8.93	10.73
		College	12.85	12.01	10.72	5.95	7.13
		Convenience Store	12.44	11.49	10.20	7.32	8.17
		Elementary School	14.96	14.58	12.86	8.88	10.78
		Garage	8.27	8.14	7.15	5.71	6.32
		Grocery	13.51	13.46	11.80	7.36	8.79
		Healthcare Clinic	13.26	13.61	11.88	8.09	8.56
		High School	15.50	15.60	13.99	9.97	11.66
		Hospital - CAV no econ	14.82	15.27	13.01	11.19	12.70
		Hospital - CAV econ	15.02	15.57	13.27	11.50	13.06
		Hospital - VAV econ	6.14	5.84	4.39	2.64	2.85
		Hospital - FCU	11.13	12.71	10.35	12.16	16.35
		Hotel/Motel	14.80	14.38	12.97	8.87	10.84
		Hotel/Motel - Common	13.45	13.66	13.00	10.58	11.12
		Hotel/Motel - Guest	14.77	14.29	12.78	8.56	10.52
		Manufacturing Facility	8.80	8.51	7.89	4.77	5.32
		MF - High Rise	15.97	15.76	14.37	12.23	12.26
		MF - High Rise - Common	18.99	18.44	16.53	11.39	14.71
		MF - High Rise - Residential	15.43	15.31	13.92	12.05	11.75
		MF - Mid Rise	17.43	17.63	15.17	11.16	12.72
		Movie Theater	15.27	15.05	14.07	10.69	12.73
		Office - High Rise - CAV no econ	16.97	17.22	15.70	10.51	11.45
		Office - High Rise - CAV econ	17.55	17.91	16.47	11.35	12.49
		Office - High Rise - VAV econ	12.83	13.09	10.79	6.37	7.11
		Office - High Rise - FCU	9.40	9.26	8.00	4.25	4.45
		Office - Low Rise	12.00	11.97	9.51	5.82	6.66
		Office - Mid Rise	13.32	13.33	11.28	7.18	7.98
		Religious Building	13.47	12.64	12.10	8.86	10.13
		Restaurant	11.34	11.38	10.21	7.73	9.16
		Retail - Department Store	11.69	10.74	10.08	6.56	7.48
		Retail - Strip Mall	11.19	10.36	9.15	6.31	6.80

Location	System Type	Building Type	Annual therm Savings per linear foot (therm /ft) (2" pipe / 1" insulation for hot water, 2" insulation for steam)				
			Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
		Warehouse	12.23	11.40	11.77	7.35	9.05
		Unknown	13.05	12.93	11.50	8.25	9.57
	LP Steam	All buildings, Recirculation heating season only (Hours below 55F)	42.33	41.69	37.76	33.78	34.86
	LP Steam	All buildings, Recirculation year round (All hours)	73.59	73.59	73.59	73.59	73.59
	HP Steam – non-recirculation	Assembly	23.24	23.81	21.26	14.16	21.70
		Assisted Living	21.89	21.40	18.80	13.82	16.61
		College	19.90	18.60	16.60	9.22	11.04
		Convenience Store	19.26	17.79	15.79	11.33	12.65
		Elementary School	23.16	22.57	19.91	13.75	16.69
		Garage	12.80	12.60	11.08	8.84	9.78
		Grocery	20.91	20.83	18.26	11.39	13.61
		Healthcare Clinic	20.53	21.07	18.39	12.53	13.25
		High School	23.99	24.15	21.66	15.43	18.05
		Hospital - CAV no econ	22.94	23.64	20.14	17.32	19.66
		Hospital - CAV econ	23.25	24.10	20.54	17.80	20.22
		Hospital - VAV econ	9.51	9.03	6.79	4.08	4.42
		Hospital - FCU	17.24	19.67	16.02	18.82	25.31
		Hotel/Motel	22.90	22.27	20.08	13.74	16.77
		Hotel/Motel - Common	20.81	21.15	20.13	16.38	17.21
		Hotel/Motel - Guest	22.87	22.13	19.78	13.24	16.28
		Manufacturing Facility	13.63	13.18	12.21	7.38	8.24
		MF - High Rise	19.85	19.59	17.86	15.20	15.24
		MF - High Rise - Common	23.60	22.92	20.55	14.16	18.28
		MF - High Rise - Residential	19.18	19.03	17.30	14.98	14.61
		MF - Mid Rise	21.67	21.92	18.86	13.87	15.81
		Movie Theater	23.64	23.29	21.78	16.55	19.71
		Office - High Rise - CAV no econ	26.27	26.66	24.30	16.28	17.73
		Office - High Rise - CAV econ	27.16	27.72	25.49	17.57	19.33
		Office - High Rise - VAV econ	19.87	20.26	16.70	9.87	11.00
		Office - High Rise - FCU	14.54	14.33	12.38	6.57	6.89
		Office - Low Rise	18.58	18.53	14.72	9.00	10.31
		Office - Mid Rise	20.61	20.64	17.46	11.12	12.36
		Religious Building	20.85	19.56	18.72	13.71	15.67
		Restaurant	17.55	17.61	15.81	11.96	14.18
		Retail - Department Store	18.10	16.63	15.61	10.16	11.58
		Retail - Strip Mall	17.32	16.04	14.17	9.77	10.53
		Warehouse	18.93	17.65	18.21	11.37	14.02
		Unknown	20.20	20.01	17.80	12.77	14.81
	HP Steam	All buildings, Recirculation heating season only (Hours below 55F)	65.53	64.54	58.45	52.29	53.97
	HP Steam	All buildings, Recirculation year round (All hours)	113.92	113.92	113.92	113.92	113.92

For insulation covering elbows and tees that connect straight pipe, a calculated surface area will be assumed based on the dimensions for fittings given by ANSI/ASME B36.19. The surface area is then converted to an

equivalent length of pipe that must be added to the total length of straight pipe in order to calculate total savings. Equivalent pipe lengths are given in 1" increments in pipe diameter for simplicity. In the case of pipe diameters in between full inch diameters, the closest equivalent length should be used. The larger pipe sizes mostly apply to steam header piping, which has the most heat loss per foot.

Calculated Surface Areas of Elbows and Tees

Nominal Pipe Diameter	Calculated Surface Area (ft)	
	90 Degree Elbow ³⁸⁶	Straight Tee ³⁸⁷
1"	0.10	0.13
2"	0.41	0.39
3"	0.93	0.77
4"	1.64	1.21
5"	2.57	1.77
6"	3.70	2.44
8"	6.58	3.95
10"	10.28	5.98
12"	14.80	8.34

Equivalent Length of Other Components – Elbows and Tees (L_{oc})

Nominal Pipe Diameter	Equivalent Length of Other Components (ft)	
	90 Degree Elbow	Straight Tee
1"	0.30	0.38
2"	0.66	0.63
3"	1.01	0.84
4"	1.40	1.03
5"	1.76	1.22
6"	2.13	1.41
8"	2.91	1.75
10"	3.65	2.13
12"	4.44	2.50

For insulation around valves or flanges, a surface area from ASTM standard C1129-12 will be assumed for 2" pipes. For 1" pipes, which weren't included in the standard, a linear-trended value will be used. The surface area is then converted to an equivalent length of either 1" or 2" straight pipe that must be added to the total length of straight pipe in order to calculate total savings.

Calculated Surface Areas of Flanges and Valves

Valves					Flanges				
Class (psi)	150	300	600	900	Class (psi)	150	300	600	900
NPS (in)	ft ²	ft ²	ft ²	ft ²	NPS (in)	ft ²	ft ²	ft ²	ft ²
1	0.69	1.8	1.8	2.4	1	0.36	0.36	0.4	1.23
2	2.21	2.94	2.94	5.2	2	0.71	0.84	0.88	1.54
2.5	2.97	3.51	3.91	6.6					
3	3.37	4.39	4.69	6.5	3	1.06	1.32	1.36	1.85
4	4.68	6.06	7.64	9.37	4	1.44	1.83	2.23	2.64

³⁸⁶ Based on the dimensions for diameter, long radius, and short radius given by ANSI/ASME 36.19

³⁸⁷ Based on the center to face and diameter dimensions given by ANSI/ASME B36.19

Valves				
Class (psi)	150	300	600	900
NPS (in)	ft ²	ft ²	ft ²	ft ²
6	7.03	9.71	13.03	15.8
8	10.3	13.5	18.4	23.8
10	13.8	18	26.5	32.1
12	16.1	24.1	31.9	41.9

Flanges				
Class (psi)	150	300	600	900
NPS (in)	ft ²	ft ²	ft ²	ft ²
6	2.04	2.72	3.6	4.37
8	2.92	3.74	4.89	6.4
10	3.68	4.8	6.93	8.47
12	5.01	6.34	7.97	10.43

Equivalent Length of Other Components - Flanges and Valves (L_{oc})

ANSI Class (psi)	Equivalent Length of Other Components (ft)			
	1" Valve	1" Flange	2" Valve	2" Flange
150	3.56	1.05	3.56	1.14
300	4.73	1.05	4.73	1.35
600	4.73	1.16	4.73	1.42
900	8.37	3.57	8.37	2.48

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-PINS-V04-160601

REVIEW DEADLINE: 1/1/2023

4.4.15 Single-Package and Split System Unitary Air Conditioners

DESCRIPTION

This measure promotes the installation of high-efficiency unitary air-, water-, and evaporatively-cooled air conditioning equipment, both single-package and split systems. Air conditioning (AC) systems are a major consumer of electricity and systems that exceed baseline efficiency requirements can significantly reduce energy consumption. This measure could apply to the replacing of an existing unit at the end of its useful life or the installation of a new unit in a new or existing building.

This measure was developed to be applicable to the following program types: TOS, NC, EREP. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is assumed to be a high-efficiency air-, water-, or evaporatively-cooled air conditioner that exceeds the energy efficiency requirements of the current International Energy Conservation Code (IECC).

DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline equipment is assumed to be a standard-efficiency air-, water, or evaporatively-cooled air conditioner that meets the energy efficiency requirements of the IECC in effect on the date of the building permit (if date is unknown, assume current IECC code minimum).

For Early Replacement programs, use the actual efficiency of the existing unit or assume IECC code base in place at the original time of existing unit installation. To qualify under the early replacement characterization, baseline equipment must meet these additional qualifications:

- The existing unit is operational when replaced or the existing unit would be operational with minor repairs³⁸⁸.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years.³⁸⁹

For early replacement, the remaining life of existing equipment is assumed to be 5 years³⁹⁰.

DEEMED MEASURE COST

The incremental capital cost for this measure is based upon capacity and efficiency level (defined by CEE specifications³⁹¹), as outlined in the following table:³⁹²

Capacity	Incremental cost (\$/ton)	
	Up to and including CEE Tier 1 units	CEE Tier 2 and above
< 135,000 Btu/hr	\$63	\$127
135,000 Btu/hr to > 250,000 Btu/hr	\$63	\$127
250,000 Btu/hr and greater	\$19	\$38

³⁸⁸ Based on ComEd Small Business Trade Ally feedback. For units rated at less than 20 ton units, the cost of common repairs is under \$2,000, significantly less than the cost of purchasing new equipment. Therefore, if the cost of repair is less than \$2,000, it can be considered early replacement because customers would repair instead of replace a failed unit. Repair cost data was not available for units larger than 20 tons.

³⁸⁹ Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007.

³⁹⁰ Assumed to be one third of effective useful life

³⁹¹ For specification details see; <https://library.cee1.org/content/cee-commercial-unitary-ac-and-hp-specification-0>

³⁹² NEEP Incremental Cost Study (ICS) Final Report – Phase 3, May 2014.

For early replacement the full cost of the installed unit should be used. If unknown use defaults below. The assumed deferred cost (after 5 years) of replacing existing equipment with a new baseline unit is also provided. This future cost should be discounted to present value using the real discount rate:

Capacity	Full Install Cost (\$/ton)		
	Base Units	Up to and including CEE Tier 1 units	CEE Tier 2 and above
< 135,000 Btu/hr	\$895	\$958	\$1,021
135,000 Btu/hr to > 250,000 Btu/hr	\$762	\$825	\$889
250,000 Btu/hr and greater	\$673	\$691	\$710

LOADSHAPE

Loadshape C03 - Commercial Cooling

COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market. Both values provided are based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren.

CF_{SSP} = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)
= 91.3%³⁹³

CF_{PJM} = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)
= 47.8%³⁹⁴

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Time of Sale:

For units with cooling capacities less than 65 kBtu/hr:

$$\Delta \text{kWH} = (\text{kBtu/hr}) * [(1/\text{SEER}_{\text{base}}) - (1/\text{SEER}_{\text{ee}})] * \text{EFLH}$$

For units with cooling capacities equal to or greater than 65 kBtu/hr:

$$\Delta \text{kWH} = (\text{kBtu/hr}) * [(1/\text{IEER}_{\text{base}}) - (1/\text{IEER}_{\text{ee}})] * \text{EFLH}$$

Early replacement³⁹⁵:

For units with cooling capacities less than 65 kBtu/hr:

³⁹³ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

³⁹⁴ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

³⁹⁵ The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input which would be the (new base to efficient savings)/(existing to efficient savings).

For remaining life of existing unit (1st 5 years):

$$\Delta \text{kWH} = (\text{kBtu/hr}) * [(1/\text{SEER}_{\text{exist}}) - (1/\text{SEER}_{\text{ee}})] * \text{EFLH}$$

For remaining measure life (next 10 years):

$$\Delta \text{kWH} = (\text{kBtu/hr}) * [(1/\text{SEER}_{\text{base}}) - (1/\text{SEER}_{\text{ee}})] * \text{EFLH}$$

For units with cooling capacities equal to or greater than 65 kBtu/hr:

For remaining life of existing unit (1st 5 years):

$$\Delta \text{kWH} = (\text{kBtu/hr}) * [(1/\text{IEER}_{\text{exist}}) - (1/\text{IEER}_{\text{ee}})] * \text{EFLH}$$

For remaining measure life (next 10 years):

$$\Delta \text{kWH} = (\text{kBtu/hr}) * [(1/\text{IEER}_{\text{base}}) - (1/\text{IEER}_{\text{ee}})] * \text{EFLH}$$

Where:

kBtu/hr = capacity of the cooling equipment actually installed in kBtu per hour (1 ton of cooling capacity equals 12 kBtu/hr)

SEER_{base} = Seasonal Energy Efficiency Ratio of the baseline equipment
= SEER values from tables below, based on applicable IECC on date of building permit (if unknown assume IECC 2015).

SEER_{ee} = Seasonal Energy Efficiency Ratio of the energy efficient equipment (actually installed)

SEER_{exist} = Seasonal Energy Efficiency Ratio of the existing equipment
= Actual, or assume IECC code base in place at the original time of existing unit installation

IEER_{base} = Integrated Energy Efficiency Ratio of the baseline equipment. See table below based on applicable IECC on date of building permit (if unknown assume IECC 2015).

IEER_{ee} = Integrated Energy Efficiency Ratio of the energy efficient equipment (actually installed)

IEER_{exist} = Integrated Energy Efficiency Ratio of the existing equipment
= Actual, or assume IECC code base in place at the original time of existing unit installation

EFLH = Equivalent Full Load Hours for cooling are provided in section 4.4 HVAC End Use

The rating conditions for the baseline and efficient equipment efficiencies must be equivalent.

2012 IECC Minimum Efficiency Requirements

TABLE C403.2.3(1)
MINIMUM EFFICIENCY REQUIREMENTS:
ELECTRICALLY OPERATED UNITARY AIR CONDITIONERS AND CONDENSING UNITS

EQUIPMENT TYPE	SIZE CATEGORY	HEATING SECTION TYPE	SUBCATEGORY OR RATING CONDITION	MINIMUM EFFICIENCY		TEST PROCEDURE ^a
				Before 6/1/2011	As of 6/1/2011	
Air conditioners, air cooled	< 65,000 Btu/h ^b	All	Split System	13.0 SEER	13.0 SEER	AHRI 210/240
			Single Package	13.0 SEER	13.0 SEER	
Through-the-wall (air cooled)	≤ 30,000 Btu/h ^b	All	Split system	12.0 SEER	12.0 SEER	
			Single Package	12.0 SEER	12.0 SEER	
Small-duct high-velocity (air cooled)	< 65,000 Btu/h ^b	All	Split System	10.0 SEER	10.0 SEER	
Air conditioners, air cooled	≥ 65,000 Btu/h and < 135,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	11.2 EER 11.4 IEER	11.2 EER 11.4 IEER	AHRI 340/360
		All other	Split System and Single Package	11.0 EER 11.2 IEER	11.0 EER 11.2 IEER	
	≥ 135,000 Btu/h and < 240,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	11.0 EER 11.2 IEER	11.0 EER 11.2 IEER	
		All other	Split System and Single Package	10.8 EER 11.0 IEER	10.8 EER 11.0 IEER	
	≥ 240,000 Btu/h and < 760,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	10.0 EER 10.1 IEER	10.0 EER 10.1 IEER	
		All other	Split System and Single Package	9.8 EER 9.9 IEER	9.8 EER 9.9 IEER	
	≥ 760,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	9.7 EER 9.8 IEER	9.7 EER 9.8 IEER	
		All other	Split System and Single Package	9.5 EER 9.6 IEER	9.5 EER 9.6 IEER	
Air conditioners, water cooled	< 65,000 Btu/h ^b	All	Split System and Single Package	12.1 EER 12.3 IEER	12.1 EER 12.3 IEER	AHRI 210/240
	≥ 65,000 Btu/h and < 135,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	11.5 EER 11.7 IEER	12.1 EER 12.3 IEER	AHRI 340/360
		All other	Split System and Single Package	11.3 EER 11.5 IEER	11.9 EER 12.1 IEER	
	≥ 135,000 Btu/h and < 240,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	11.0 EER 11.2 IEER	12.5 EER 12.7 IEER	
		All other	Split System and Single Package	10.8 EER 11.0 IEER	12.3 EER 12.5 IEER	
	≥ 240,000 Btu/h and < 760,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	11.0 EER 11.1 IEER	12.4 EER 12.6 IEER	
		All other	Split System and Single Package	10.8 EER 10.9 IEER	12.2 EER 12.4 IEER	
	≥ 760,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	11.0 EER 11.1 IEER	12.0 EER 12.4 IEER	
		All other	Split System and Single Package	10.8 EER 10.9 IEER	12.0 EER 12.2 IEER	

(continued)

**TABLE C403.2.3(1)—continued
MINIMUM EFFICIENCY REQUIREMENTS:
ELECTRICALLY OPERATED UNITARY AIR CONDITIONERS AND CONDENSING UNITS**

EQUIPMENT TYPE	SIZE CATEGORY	HEATING SECTION TYPE	SUB-CATEGORY OR RATING CONDITION	MINIMUM EFFICIENCY		TEST PROCEDURE ^a
				Before 6/1/2011	As of 6/1/2011	
Air conditioners, evaporatively cooled	< 65,000 Btu/h ^b	All	Split System and Single Package	12.1 EER 12.3 IEER	12.1 EER 12.3 IEER	AHRI 210/240
	≥ 65,000 Btu/h and < 135,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	11.5 EER 11.7 IEER	12.1 EER 12.3 IEER	AHRI 340/360
		All other	Split System and Single Package	11.3 EER 11.5 IEER	11.9 EER 12.1 IEER	
	≥ 135,000 Btu/h and < 240,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	11.0 EER 11.2 IEER	12.0 EER 12.2 IEER	
		All other	Split System and Single Package	10.8 EER 11.0 IEER	11.8 EER 12.0 IEER	
	≥ 240,000 Btu/h and < 760,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	11.0 EER 11.1 IEER	11.9 EER 12.1 IEER	
		All other	Split System and Single Package	10.8 EER 10.9 IEER	12.2 EER 11.9 IEER	
	≥ 760,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	10.0 EER 11.1 IEER	11.7 EER 11.9 IEER	
		All other	Split System and Single Package	10.8 EER 10.9 IEER	11.5 EER 11.7 IEER	
Condensing units, air cooled	≥ 135,000 Btu/h			10.1 EER 11.4 IEER	10.5 EER 14.0 IEER	AHRI 365
Condensing units, water cooled	≥ 135,000 Btu/h			13.1 EER 13.6 IEER	13.5 EER 14.0 IEER	
Condensing units, evaporatively cooled	≥ 135,000 Btu/h			13.1 EER 13.6 IEER	13.5 EER 14.0 IEER	

For SI: 1 British thermal unit per hour = 0.2931 W.

- a. Chapter 6 of the referenced standard contains a complete specification of the referenced test procedure, including the reference year version of the test procedure.
- b. Single-phase, air-cooled air conditioners less than 65,000 Btu/h are regulated by NAECA. SEER values are those set by NAECA.

2015 IECC Minimum Efficiency Requirements

TABLE C403.2.3(1)
MINIMUM EFFICIENCY REQUIREMENTS:
ELECTRICALLY OPERATED UNITARY AIR CONDITIONERS AND CONDENSING UNITS

EQUIPMENT TYPE	SIZE CATEGORY	HEATING SECTION TYPE	SUBCATEGORY OR RATING CONDITION	MINIMUM EFFICIENCY		TEST PROCEDURE ^a
				Before 1/1/2016	As of 1/1/2016	
Air conditioners, air cooled	< 65,000 Btu/h ^b	All	Split System	13.0 SEER	13.0 SEER	AHRI 210/240
			Single Package	13.0 SEER	14.0 SEER ^c	
Through-the-wall (air cooled)	≤ 30,000 Btu/h ^b	All	Split system	12.0 SEER	12.0 SEER	
			Single Package	12.0 SEER	12.0 SEER	
Small-duct high-velocity (air cooled)	< 65,000 Btu/h ^b	All	Split System	11.0 SEER	11.0 SEER	
Air conditioners, air cooled	≥ 65,000 Btu/h and < 135,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	11.2 EER 11.4 IEER	11.2 EER 12.8 IEER	AHRI 340/360
		All other	Split System and Single Package	11.0 EER 11.2 IEER	11.0 EER 12.6 IEER	
	≥ 135,000 Btu/h and < 240,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	11.0 EER 11.2 IEER	11.0 EER 12.4 IEER	
		All other	Split System and Single Package	10.8 EER 11.0 IEER	10.8 EER 12.2 IEER	
	≥ 240,000 Btu/h and < 760,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	10.0 EER 10.1 IEER	10.0 EER 11.6 IEER	
		All other	Split System and Single Package	9.8 EER 9.9 IEER	9.8 EER 11.4 IEER	
	≥ 760,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	9.7 EER 9.8 IEER	9.7 EER 11.2 IEER	
		All other	Split System and Single Package	9.5 EER 9.6 IEER	9.5 EER 11.0 IEER	
	< 65,000 Btu/h ^b	All	Split System and Single Package	12.1 EER 12.3 IEER	12.1 EER 12.3 IEER	
Air conditioners, water cooled	≥ 65,000 Btu/h and < 135,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	12.1 EER 12.3 IEER	12.1 EER 13.9 IEER	AHRI 340/360
		All other	Split System and Single Package	11.9 EER 12.1 IEER	11.9 EER 13.7 IEER	
	≥ 135,000 Btu/h and < 240,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	12.5 EER 12.5 IEER	12.5 EER 13.9 IEER	
		All other	Split System and Single Package	12.3 EER 12.5 IEER	12.3 EER 13.7 IEER	
	≥ 240,000 Btu/h and < 760,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	12.4 EER 12.6 IEER	12.4 EER 13.6 IEER	
		All other	Split System and Single Package	12.2 EER 12.4 IEER	12.2 EER 13.4 IEER	
	≥ 760,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	12.2 EER 12.4 IEER	12.2 EER 13.5 IEER	
		All other	Split System and Single Package	12.0 EER 12.2 IEER	12.0 EER 13.3 IEER	
	< 65,000 Btu/h ^b	All	Split System and Single Package	12.1 EER 12.3 IEER	12.1 EER 12.3 IEER	

(continued)

TABLE C403.2.3(1)—continued
MINIMUM EFFICIENCY REQUIREMENTS:
ELECTRICALLY OPERATED UNITARY AIR CONDITIONERS AND CONDENSING UNITS

EQUIPMENT TYPE	SIZE CATEGORY	HEATING SECTION TYPE	SUB-CATEGORY OR RATING CONDITION	MINIMUM EFFICIENCY		TEST PROCEDURE ^c
				Before 1/1/2016	As of 1/1/2016	
Air conditioners, evaporatively cooled	< 65,000 Btu/h ^b	All	Split System and Single Package	12.1 EER 12.3 IEER	12.1 EER 12.3 IEER	AHRI 210/240
	≥ 65,000 Btu/h and < 135,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	12.1 EER 12.3 IEER	12.1 EER 12.3 IEER	AHRI 340/360
		All other	Split System and Single Package	11.9 EER 12.1 IEER	11.9 EER 12.1 IEER	
	≥ 135,000 Btu/h and < 240,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	12.0 EER 12.2 IEER	12.0 EER 12.2 IEER	
		All other	Split System and Single Package	11.8 EER 12.0 IEER	11.8 EER 12.0 IEER	
	≥ 240,000 Btu/h and < 760,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	11.9 EER 12.1 IEER	11.9 EER 12.1 IEER	
		All other	Split System and Single Package	11.7 EER 11.9 IEER	11.7 EER 11.9 IEER	
	≥ 760,000 Btu/h	Electric Resistance (or None)	Split System and Single Package	11.7 EER 11.9 IEER	11.7 EER 11.9 IEER	
		All other	Split System and Single Package	11.5 EER 11.7 IEER	11.5 EER 11.7 IEER	
Condensing units, air cooled	≥ 135,000 Btu/h			10.5 EER 11.8 IEER	10.5 EER 11.8 IEER	AHRI 365
Condensing units, water cooled	≥ 135,000 Btu/h			13.5 EER 14.0 IEER	13.5 EER 14.0 IEER	
Condensing units, evaporatively cooled	≥ 135,000 Btu/h			13.5 EER 14.0 IEER	13.5 EER 14.0 IEER	

For SI: 1 British thermal unit per hour = 0.2931 W.

a. Chapter 6 contains a complete specification of the referenced test procedure, including the reference year version of the test procedure.

b. Single-phase, air-cooled air conditioners less than 65,000 Btu/h are regulated by NAECA. SEER values are those set by NAECA.

c. Minimum efficiency as of January 1, 2015.

For example a 5 ton air cooled split system with a SEER of 15 at a retail strip mall in Rockford would save:

$$\begin{aligned}\Delta \text{kWh} &= (60) * [(1/13) - (1/15)] * 950 \\ &= 585 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

Time of Sale:

$$\Delta \text{kW} = (\text{kBtu/hr} * (1/\text{EER}_{\text{base}} - 1/\text{EER}_{\text{ee}})) * \text{CF}$$

Early Replacement:

For remaining life of existing unit (1st 5 years):

$$\Delta \text{kW} = (\text{kBtu/hr} * [(1/\text{EER}_{\text{exist}}) - (1/\text{EER}_{\text{ee}})]) * \text{CF}$$

For remaining measure life (next 10 years):

$$\Delta \text{kWh} = (\text{kBtu/hr} * [(1/\text{EER}_{\text{base}}) - (1/\text{EER}_{\text{ee}})]) * \text{CF}$$

Where:

EER_{base} = Energy Efficiency Ratio of the baseline equipment

= EER values from tables below, based on applicable IECC on date of the building permit (if unknown assume IECC 2015). (For air-cooled units < 65 kBtu/hr, assume the following

conversion from SEER to EER for calculation of peak savings:³⁹⁶ $EER = (-0.02 * SEER^2) + (1.12 * SEER)$

EERee = Energy Efficiency Ratio of the energy efficient equipment. If the actual EERee is unknown, assume the conversion from SEER to EER for calculation of peak savings as above).

= Actual installed

EERexist = Energy Efficiency Ratio of the existing equipment

= Actual, or assume IECC code base in place at the original time of existing unit installation

CF_{SSP} = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)
= 91.3%³⁹⁷

CF_{PJM} = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)
= 47.8%³⁹⁸

For example, a 5 ton air cooled split system with a SEER of 15 in Rockford would save:

$$\Delta kW_{SSP} = (60) * [(1/13) - (1/15)] * .913$$

$$= 0.562 \text{ kW}$$

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

MEASURE CODE: CI-HVC-SPUA-V05-180101

REVIEW DEADLINE: 1/1/2019

³⁹⁶ Based on Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder. Note this is appropriate for single speed units only.

³⁹⁷ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

³⁹⁸ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

4.4.16 Steam Trap Replacement or Repair

DESCRIPTION

The measure is for the repair or replacement of faulty steam traps that are allowing excess steam to escape and thereby increasing steam generation. The measure is applicable to commercial applications, commercial HVAC (low pressure steam) including multifamily buildings, low pressure industrial applications, medium pressure industrial applications, applications and high pressure industrial applications.

This measure was developed to be applicable to the following program types: TOS, RF. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

Customers must have leaking traps to qualify for rebates. However, if a commercial customer opts to replace all traps without inspection, rebates and the savings are discounted to take into consideration the fact that some traps are being replaced that have not yet failed.

DEFINITION OF BASELINE EQUIPMENT

The baseline criterion is a faulty steam trap in need of replacing. No minimum leak rate is required. Any leaking or blow through trap can be repaired or replaced. If a commercial customer chooses to repair or replace all the steam traps at the facility without verification, the savings are adjusted. Savings for commercial full replacement projects are reduced by the percentage of traps found to be leaking on average from the studies listed. If an audit is performed on a commercial site, then the leaking and blowdown can be adjusted.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The life of this measure is 6 years³⁹⁹

DEEMED MEASURE COST

Steam System	Cost per trap ⁴⁰⁰ (\$)
Commercial Dry Cleaners	77
Commercial Heating (including Multifamily), low pressure steam	77
Industrial Medium Pressure >15 psig < 30 psig	180
Steam Trap, Industrial Medium Pressure ≥30 <75 psig	223
Steam Trap, Industrial High Pressure ≥75 <125 psig	276
Steam Trap, Industrial High Pressure ≥125 <175 psig	322
Steam Trap, Industrial High Pressure ≥175 <250 psig	370
Steam Trap, Industrial High Pressure ≥250 psig	418

LOADSHAPE

N/A

³⁹⁹Source paper is the CLEAResult "Steam Traps Revision #1" dated August 2011. Primary studies used to prepare the source paper include Enbridge Steam Trap Survey, KW Engineering Steam Trap Survey, Enbridge Steam Saver Program 2005, Armstrong Steam Trap Survey, DOE Federal Energy Management Program Steam Trap Performance Assessment, Oak Ridge National Laboratory Steam System Survey Guide, KEMA Evaluation of PG&E's Steam Trap Program, Sept. 2007. Communication with vendors suggested a inverted bucket steam trap life typically in the range of 5 - 7 years, float and thermostatic traps 4- 6 years, float and thermodynamic disc traps of 1 - 3 years. Cost does not include installation.

⁴⁰⁰ Ibid.

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS SAVINGS

$$\Delta \text{Therm} = S_a * (H_v/B) * \text{Hours} * L / 100,000$$

Where:

S_a = Average actual steam loss per leaking trap
 = $24.24 \times P_{ia} \times D^2 \times A \times FF$

Where:

24.24 = Constant lb/(hr-psia-in²)

P_{ia} = $P_{ig} + P_{atm}$

= Average steam trap inlet pressure, absolute, psia

P_{ig} = Average steam trap inlet pressure, gauge, psig

P_{atm} = Atmospheric pressure, 14.7 psia

D = Diameter of Orifice, in.

A = Adjustment factor

= 50%,⁴⁰¹ all steam systems. This factor is to account for reducing the maximum theoretical steam flow to the average steam flow (the Enbridge factor).

FF = Flow Factor. In addition to the Adjustment factor (A), an additional 50 percent flow factor adjustment is recommended for medium and high pressure steam systems to address industrial float and thermostatic style traps where additional blockage is possible.

⁴⁰¹ Enbridge adjustment factor used as referenced in CLEAResult "Work Paper Steam Traps Revision #2" Revision 3 dated March 2, 2012 and DOE Federal Energy Management Program Steam Trap Performance Assessment.

Steam System	Average Steam Trap Inlet Pressure psig ⁴⁰²	Diameter of Orifice in	Adjustment Factor	Flow Factor	Average Actual Steam Loss per Leaking Trap (lb/hr/trap)
Commercial Dry Cleaners	-	-	50%	100%	19.1
Commercial Heating (including Multifamily) LPS	-	-	50%	100%	6.9
Industrial or Process Low Pressure, <15 psig	-	-	50%	100%	6.9
Medium Pressure >15 psig < 30 psig	16	0.1875	50%	50%	6.5
Medium Pressure ≥30 <75 psig	47	0.2500	50%	50%	23.4
High Pressure ≥75 <125 psig	101	0.2500	50%	50%	43.8
High Pressure ≥125 <175 psig	146	0.2500	50%	50%	60.9
High Pressure ≥175 <250 psig	202	0.2500	50%	50%	82.1
High Pressure ≥250 ≤300 psig	263	0.2500	50%	50%	105.2
High Pressure > 300 psig	Custom	Custom	50%	50%	Calculated

Hv = Heat of vaporization of steam

Steam System	Average Inlet Pressure psig	Heat of Vaporization ⁴⁰³ (Btu/lb)
Commercial Dry Cleaners	--	890
Commercial Heating (including Multifamily) LPS	--	951
Industrial and Process Low Pressure ≤15 psig	--	951
Medium Pressure >15 psig < 30 psig	16	944
Medium Pressure ≥30 <75 psig	47	915
High Pressure ≥75 <125 psig	101	880
High Pressure ≥125 <175 psig	146	859
High Pressure ≥175 <250 psig	202	837
High Pressure ≥250 ≤300 psig	263	816
High Pressure > 300 psig	--	Custom

B = Boiler efficiency

= custom, if unknown:

= 80.7% for steam boilers, except multifamily low-pressure ⁴⁰⁴

= 64.8% for multifamily low-pressure steam boilers ⁴⁰⁵

Hours = Annual operating hours of steam plant

⁴⁰² Medium and high pressure steam trap inlet pressure based on Navigant analysis of source collected during program implementation by Nicor Gas for GPY1 through GPY4. For each steam trap project, the data provided measure savings description, operating pressure, installation Zip code, business building type, program year, and annual operating hours.

⁴⁰³ Heat of vaporization of steam at the inlet pressure to the steam trap. Implicit assumption that the average boiler nominal pressure where the vaporization occurs, is essentially that same pressure. Referenced in CLEAResult "Work Paper Steam Traps Revision #2" Revision 3 dated March 2, 2012.

⁴⁰⁴ Ibid.

⁴⁰⁵ Katrakis, J. and T.S. Zawacki. "Field-Measured Seasonal Efficiency of Intermediate-sized Low-Pressure Steam Boilers". ASHRAE V99, pt. 2, 1993.

= custom, if unknown:

Steam System	Zone (where applicable)	Hours/Yr ⁴⁰⁶
Commercial Dry Cleaners	All Climate Zones	2,425
Industrial and Process Low Pressure ≤15 psig		8,282
Medium Pressure >15 psig < 30 psig		8,282
Medium Pressure ≥30 <75 psig		8,282
High Pressure ≥75 <125 psig		8,282
High Pressure ≥125 <175 psig		8,282
High Pressure ≥175 <250 psig		8,282
High Pressure ≥250 psig		8,282
Commercial Heating (including Multifamily)LPS ⁴⁰⁷	1 (Rockford)	4,272
	2 (Chicago O'Hare)	4,029
	3 (Springfield)	3,406
	4 (Belleville)	2,515
	5 (Marion)	2,546

L = Leaking & blow-thru

L is 1.0 when applied to the replacement of an individual leaking trap. If a number of steam traps are replaced and the system has not been audited, the leaking and blow-thru is applied to reflect the assumed percentage of steam traps that were actually leaking and need to be replaced. A custom value can be utilized if supported by an evaluation.

Steam System	L (%) ⁴⁰⁸
Custom	Custom
Commercial Dry Cleaners	27%
Commercial Heating (including Multifamily) LPS	27%
Industrial and Process Low Pressure ≤15 psig	16%
Medium Pressure >15 psig < 30 psig	16%
Medium Pressure ≥30 <75 psig	16%
High Pressure ≥75 <125 psig	16%
High Pressure ≥125 <175 psig	16%
High Pressure ≥175 <250 psig	16%
High Pressure > 300 psig	16%

⁴⁰⁶ Medium and high pressure steam trap annual operating hours based on Navigant analysis of source collected during program implementation by Nicor Gas for GPY1 through GPY4. For each steam trap project, the data provided measure savings description, operating pressure, installation Zip code, business building type, program year, and annual operating hours.

⁴⁰⁷ Since commercial LPS reflect heating systems, Hours/yr are equivalent to HDD55 zone table

⁴⁰⁸ Dry cleaners survey data as referenced in CLEAResult "Work Paper Steam Traps Revision #2" Revision 3 dated March 2, 2012.

EXAMPLE

For example, a commercial dry cleaning facility with the default hours of operation and boiler efficiency;

$$\begin{aligned}\Delta\text{Therms} &= \text{Sa} * (\text{Hv/B}) * \text{Hours} * \text{L} \\ &= 19.1 \text{ lbs/hr/trap} * (890 \text{ Btu/lb} / 80\%) / 100,000 * 2,425 * 27\% \\ &= 138.8 \text{ therms per trap}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-STRE-V05-180101

REVIEW DEADLINE: 1/1/2020

4.4.17 Variable Speed Drives for HVAC Pumps and Cooling Tower Fans

DESCRIPTION

This measure is applied to variable speed drives (VSD) which are installed on the following HVAC system applications: chilled water pump, hot water pumps and cooling tower fans. There is a separate measure for HVAC supply and return fans. All other VSD applications require custom analysis by the program administrator. The VSD will modulate the speed of the motor when it does not need to run at full load. Since the power of the motor is proportional to the cube of the speed for these types of applications, significant energy savings will result.

This measure is not applicable for:

- Cooling towers, chilled or hot water pumps with any process load.
- VSD installation in existing cooling towers with 2-speed motors. (IECC 2007 requires 2-speed motors for cooling towers with motors greater than 7.5 HP)
- VSD installation in new cooling towers with motors greater than 7.5 HP

This measure was developed to be applicable to the following program types: TOS, RF. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The VSD is applied to a motor which does not have a VSD. This measure is not applicable for replacing failed VSDs. The application must have a variable load and installation is to include the necessary controls. Savings are based on application of VSDs to a range of baseline load conditions including no control, inlet guide vanes, outlet guide vanes and throttling valves.

DEFINITION OF BASELINE EQUIPMENT

The time of sale baseline is a new motor installed without a VSD or other methods of control. Retrofit baseline is an existing motor operating as is. Retrofit baselines may or may not include guide vanes, throttling valves or other methods of control. This information shall be collected from the customer.

Installations of new equipment with VSDs which are required by IECC 2012 or 2015 as adopted by the State of Illinois are not eligible for incentives.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life for HVAC application is 15 years;⁴⁰⁹ measure life for process is 10 years.⁴¹⁰

DEEMED MEASURE COST

Customer provided costs will be used when available. Default measure costs⁴¹¹ are noted below for up to 20 hp motors. Custom costs must be gathered from the customer for motor sizes not listed below.

HP	Cost
1 -5 HP	\$ 1,330
7.5 HP	\$ 1,622
10 HP	\$ 1,898
15 HP	\$ 2,518
20 HP	\$ 3,059

⁴⁰⁹ Efficiency Vermont TRM 10/26/11 for HVAC VSD motors

⁴¹⁰ DEER 2008

⁴¹¹ Ohio TRM 8/6/2010 varies by motor/fan size based on equipment costs from Granger 2008 Catalog pp 286-289, average across available voltages and models. Labor costs from RS Means Data 2008 Ohio average cost adjustment applied.

LOADSHAPE

Loadshape C42 - VFD - Boiler feedwater pumps <10 HP

Loadshape C43 - VFD - Chilled water pumps <10 HP

Loadshape C44 - VFD Boiler circulation pumps <10 HP

Loadshape C48 - VFD Boiler draft fans <10 HP

Loadshape C49 - VFD Cooling Tower Fans <10 HP

COINCIDENCE FACTOR

The demand savings factor (DSF) is already based upon coincident savings, and thus there is no additional coincidence factor for this characterization.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = \text{BHP} / \text{EFFi} * \text{Hours} * \text{ESF}$$

Where:

BHP = System Brake Horsepower

(Nominal motor HP * Motor load factor)

Motors are assumed to have a load factor of 65% for calculating kW if actual values cannot be determined⁴¹². Custom load factor may be applied if known.

EFFi = Motor efficiency, installed. Actual motor efficiency shall be used to calculate kW. If not known a default value of 93% shall be used.⁴¹³

Hours = Default hours are provided for HVAC applications which vary by HVAC application and building type⁴¹⁴. When available, actual hours should be used.

Building Type	Heating Run Hours	Cooling Run Hours
Assembly	4888	2150
Assisted Living	4711	4373
College	3990	1605
Convenience Store	4136	2084
Elementary School	5105	3276
Garage	4849	2102
Grocery	4200	2096
Healthcare Clinic	5481	1987
High School	5480	3141

⁴¹² Del Balso, Ryan J. "Investigation into the Reliability of Energy Efficiency/Demand Side Management Savings Estimates for Variable Frequency Drives in Commercial Applications", University of Colorado, Department of Civil, Environmental and Architectural Engineering, 2013.

⁴¹³ Ohio TRM 8/6/2010 pp207-209, Com Ed TRM June 1, 2010.

⁴¹⁴ Hours per year are estimated using the eQuest models as the total number of hours the heating or cooling system is operating for each building type. "Heating and Cooling Run Hours" are estimated as the total number of hours fans are operating for heating, cooling and ventilation for each building type. This may overclaim certain applications (e.g. pumps) and so where possible actual hours should be used for these applications.

Building Type	Heating Run Hours	Cooling Run Hours
Hospital - VAV econ	3718	2788
Hospital - CAV econ	7170	2881
Hospital - CAV no econ	7139	8760
Hospital - FCU	5844	8729
Manufacturing Facility	3821	2805
MF - High Rise	4522	4237
MF - Mid Rise	5749	2899
Hotel/Motel - Guest	4480	4479
Hotel/Motel - Common	3292	8712
Movie Theater	5063	2120
Office - High Rise - VAV econ	4094	2038
Office - High Rise - CAV econ	5361	4849
Office - High Rise - CAV no econ	5331	5682
Office - High Rise - FCU	3758	3069
Office - Low Rise	3834	2481
Office - Mid Rise	3977	1881
Religious Building	5199	2830
Restaurant	4579	3350
Retail - Department Store	4249	2528
Retail - Strip Mall	4475	2266
Warehouse	4606	770
Unknown	4649	2718

The type of hours to apply depends on the VFD application, according to the table below.

Application	Hours Type
Hot Water Pump	Heating
Chilled Water Pump	Cooling
Cooling Tower Fan	Cooling

ESF = Energy savings factor varies by VFD application. Units are kW/HP.

Application	ESF
Hot Water Pump	0.424 ⁴¹⁵
Chilled Water Pump	0.411 ⁴¹⁶
Cooling Tower Fan	0.126 ⁴¹⁷

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = BHP/EFFi * DSF$$

Where:

DSF = Demand Savings Factor varies by VFD application.⁴¹⁸ Units are kW/HP. Values listed below are based on typical peak load for the listed application.

⁴¹⁵ Based on the methodology described in the Connecticut TRM, 8th Edition (2013); derived using a temperature BIN analysis of typical heating, cooling and fan load profiles.

⁴¹⁶ Ibid

⁴¹⁷ Based on eQuest model for VSD v one-speed fan, see "CT Savings Factors.xlsx".

⁴¹⁸ DSF assumptions are based upon the same source as the ESFs.

Application	DSF
Hot Water Pump	0
Chilled Water Pump	0.299
Cooling Tower Fan	0.378

FOSSIL FUEL IMPACT DESCRIPTIONS AND CALCULATION

There are no expected fossil fuel impacts for this measure.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-VSDHP-V04-180101

REVIEW DEADLINE: 1/1/2021

4.4.18 Small Commercial Programmable Thermostats

DESCRIPTION

This measure characterizes the energy savings from the installation of a new Programmable Thermostat for reduced heating energy consumption through temperature set-back during unoccupied or reduced demand times. This measure is limited to small businesses, as they have smaller HVAC systems that are similar to residential HVAC systems and may be controlled by a simple manual adjustment thermostat. Mid to large sized businesses will typically have a building automation system or some other form of automated HVAC controls. Therefore, it is limited to select building types, including small office, retail – strip mall, restaurants (characterized as 1, 2 or 3 meal), small manufacturing, religious facilities, and convenience stores. This measure is only appropriate for single zone heating systems. Custom calculations are required for savings for programmable thermostats installed in multi-zone systems.

This measure was developed to be applicable to the following program types: RF, DI.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The criteria for this measure are established by replacement of a manual-only temperature control, with one that has the capability to adjust temperature setpoints according to a schedule without manual intervention.

DEFINITION OF BASELINE EQUIPMENT

For new thermostats the baseline is a non-programmable thermostat requiring manual intervention to change temperature setpoint.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life of a programmable thermostat is assumed to be 8 years⁴¹⁹ based upon equipment life only⁴²⁰. For the purposes of claiming savings for a new programmable thermostat, this is reduced by a 50% persistence factor to give a final measure life of 4 years.

DEEMED MEASURE COST

Actual material and labor costs should be used if the implementation method allows. If unknown the capital and labor cost for this measure is assumed to be \$181 per thermostat⁴²¹. For the purposes of screening and planning it should be assumed that one thermostat will serve 5 tons of Cooling Capacity at a cost of \$36.20 / ton or 115kBtuh of Heating Capacity at a cost of \$1.57 / kBtu.

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

⁴¹⁹ Table 1, HVAC Controls, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007

⁴²⁰ Future evaluation is strongly encouraged to inform the persistence of savings to further refine measure life assumption. As this characterization depends heavily upon a large scale but only 2-year study of the energy impacts of programmable thermostats, the longer term impacts should be assessed.

⁴²¹ Nicor Rider 30 Business EER Program Database, Paid Rebates with Programmable Thermostat Installation Costs, Program to Date as of January 11, 2013.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS⁴²²**

$$\Delta \text{kWh} = [\text{Baseline Energy Use (kWh/Ton)} - \text{Proposed Energy Use (kWh/Ton)}] * \text{Cooling Capacity (Tons)}$$

The following equations are used to calculate baseline and proposed electric energy use. The savings is the difference between the proposed and baseline calculated usage. This approach allows the savings estimate to account for the operational attributes of the baseline as well as the proposed case, yielding a better estimate than an approach that assumes a particular baseline or proposed energy use to determine savings.

Electric Energy Use Equations (kWh / ton)

Building Type	Fan Mode During Occupied Period (Fo)	Equation
Assembly	Continuous	$CZ + Fu * (0.83 * Tc + 0.83 * Th + 1.67 * Ws - 293.018) - 0.0922 * Tc * Th + 1.291 * Ws$
	Intermittent	$CZ + Fu * (1.911 - 0.12 * Tc) + Tc * (0.00311 * Ws - 0.229) + 0.11 * Ws$
Convenience Store	Continuous	$CZ + Fu * (-28.629 * Tc - 11.69 * Th + 19.118 * Ws - 2935.12) + 0.909 * Ws$
	Intermittent	$CZ + Tc * (0.0863 * Ws - 12.688) + Th * (0.043 * Ws - 6.38) + 1.669 * Ws$
Office – Low Rise	Continuous	$CZ + Fu * (7.082 * Tc - 41.199 * Th + 18.734 * Ws - 3288.55) + Tc * (0.205 * Ws - 34.929)$
	Intermittent	$CZ + Tc * (0.0806 * Ws - 8.984) + Th * (0.0864 * Ws - 9.558) + 1.178 * Ws$
Religious	Continuous	$CZ + Fu * (-1.579 * Tc - 18.14 * Th + 15.01 * Ws - 2417.74) + Tc * (0.177 * Ws - 26.412)$
	Intermittent	$CZ + Fu * (0.266 * Tc - 2.067) + Tc * (0.0295 * Ws - 4.502) + Th * (0.0517 * Ws - 8.251) + 0.735 * Ws$
Restaurant – Fast Food	Continuous	$CZ + Fu * (0.678 * Tc + 0.257 * Th + 2.88 * Ws - 494.006) + Tc * (0.0231 * Ws - 4.074) + Th * (0.00936 * Ws - 1.655) + 0.918 * Ws$
	Intermittent	$CZ + Fu * (0.377 * Tc + 0.124 * Th + 0.13 * Ws - 24.893) + Tc * (-0.0143 * Th + 0.0166 * Ws - 2.691) + 0.898 * Ws$
Restaurant – Full Service	Continuous	$CZ + Fu * (-8.41 * Th + 11.766 * Ws - 1910.81) + Tc * (0.282 * Ws - 43.851)$
	Intermittent	$CZ + 0.123 * Fu * Tc + Tc * (0.0561 * Ws - 8.237) + Th * (0.0219 * Ws - 3.284) + 1.038 * Ws$
Retail – Department Store	Continuous	$CZ + Fu * (-1.475 * Th + 0.755 * Ws - 114.373) + Th * (0.151 * Ws - 24.016) + 1.612 * Ws$
	Intermittent	$CZ + Tc * (0.0173 * Ws - 1.912) + Th * (0.0249 * Ws - 3.29) + 0.511 * Ws$
Retail – Strip Mall	Continuous	$CZ + Fu * (1.077 * Tc - 10.697 * Th + 6.91 * Ws - 1117.18) + Tc * (0.0583 * Ws - 7.54) + 1.231 * Ws$
	Intermittent	$CZ + 0.0894 * Fu * Tc + Th * (-0.0142 * Tc + 0.04 * Ws - 5.278) + 0.884 * Ws$

Where:

- CZ = Climate Zone Coefficient
= Depends on Building Type and Fan Mode During Occupied Period (see table below)
- Tc = Degrees of Cooling Setback °F
= Must be between 0-15°F
- Th = Degrees of Heating Setback °F
= Must be between 0-15°F

⁴²² Savings equations and factors determined by regression of results of a series of eQuest simulations. See Programmable T-Stat Work Paper_PECI_FinalDraft_140730_Redline.docx for details.

- Fo** = Fan Mode During Occupied Period (Note: Commercial mechanical code requires continuous fan operation during occupied periods to meet ventilation requirements.)
 = Continuous for occupied fan that runs continuously (e.g. Fan Mode Set to 'On')
 = Intermittent for occupied fan that runs intermittently (e.g. Fan Mode Set to 'Auto')
- Fu** = Fan Mode During Unoccupied Period
 = 0 for unoccupied fan that runs continuously (e.g. Fan Mode Set to 'On')
 = 1 for unoccupied fan that runs intermittently (e.g. Fan Mode Set to 'Auto')
- Ws** = Weekly Hours thermostat is in Occupied mode
 = Minimum values depends on Building Type (see table below), maximum value of 168 (24/7)
 (e.g.: Weekly occupancy schedule of Mon-Sat 8AM-5PM, Sun 9AM-2PM, Ws = 59)

Electric Energy Use Climate Zone Coefficients and Minimum Weekly Hours Occupied

Building Type	Fan Mode During Occupied Period (<i>Fo</i>)	Climate Zone Coefficient (CZ) ⁴²³					Minimum Ws
		1	2	3	4	5	
Assembly	Continuous	911.366	928.924	1152.83	1208.999	1210.173	98
	Intermittent	735.752	762.831	966.562	998.927	1028.906	
Convenience Store	Continuous	4817.094	4832.784	5139.133	5182.161	5208.608	108
	Intermittent	1478.133	1514.568	1784.384	1843.463	1930.47	
Office - Low Rise	Continuous	5047.662	5039.592	5187.924	5217.672	5177.449	55
	Intermittent	825.072	808.965	946.571	979.421	945.418	
Religious Facility	Continuous	4197.117	4172.858	4380.025	4370.008	4356.054	133
	Intermittent	632.404	603.395	678.294	664.717	616.853	
Restaurant – Fast Food	Continuous	1342.988	1378.661	1664.018	1714.201	1727.841	108
	Intermittent	993.764	1039.643	1307.8	1340.544	1389.791	
Restaurant – Full Service	Continuous	4070.35	4094.742	4428.966	4501.829	4522.522	117
	Intermittent	1472.014	1516.05	1856.108	1938.441	2056.45	
Retail – Department Store	Continuous	1510.201	1496.47	1706.105	1716.128	1688.464	93
	Intermittent	701.27	702.129	847.735	875.12	881.677	
Retail – Strip Mall	Continuous	1926.294	1930.137	2156.856	2174.435	2165.03	93
	Intermittent	656.479	673.257	835.906	850.322	869.921	

⁴²³ Climate Zones Referenced in Section 3.7, Table 3.6

EXAMPLE

A low rise office in Rockford (Climate Zone 1) is occupied Mon-Fri 7AM-6PM and has a 10 ton DX RTU controlled by a manual thermostat. The fan runs continuously during the occupied hours and building staff do not manually change the fan mode, cooling or heating setpoints during unoccupied periods.

A programmable thermostat is installed by a contractor who sets the occupied schedule to Mon-Fri 7AM-6PM with a 10°F cooling and heating unoccupied temperature setback. The contractor also programs the fan to operate continuously during the occupied periods and to intermittent “auto” during the unoccupied periods.

$$\begin{aligned} \Delta \text{kWh} &= [\text{Baseline Energy Use (kWh/Ton)} - \text{Proposed Energy Use (kWh/Ton)}] * \text{Cooling Capacity (Tons)} \\ \text{Baseline Energy Use (kWh/Ton)} &= \text{Equation for Office Low Rise, } F_o = \text{Continuous} \\ &= CZ + Fu * (7.082 * T_c - 41.199 * Th + 18.734 * Ws - 3288.55) + T_c * (0.205 * Ws - 34.929) \\ &= 5047.662 + 0 * (7.082 * 0 - 41.199 * 0 + 18.734 * 168 - 3288.55) + 0 * (0.205 * 168 - 34.929) \\ &= 5,047.662 \text{ kWh/Ton} \\ \text{Proposed Energy Use (kWh/Ton)} &= \text{Equation for Office Low Rise, } F_o = \text{Continuous} \\ &= CZ + Fu * (7.082 * T_c - 41.199 * Th + 18.734 * Ws - 3288.55) + T_c * (0.205 * Ws - 34.929) \\ &= 5047.662 + 1 * (7.082 * 10 - 41.199 * 10 + 18.734 * 55 - 3288.55) + 10 * (0.205 * 55 - 34.929) \\ &= 2,211.722 \text{ kWh/Ton} \\ \Delta \text{kWh} &= [5,047.622 \text{ (kWh/Ton)} - 2,211.722 \text{ (kWh/Ton)}] * 10 \text{ Tons} \\ &= 2,835.89 \text{ kWh/Ton} * 10 \text{ Tons} \\ &= 28,358.9 \text{ kWh} \end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms} = [\text{Baseline Energy Use (Therms/kBtu)} - \text{Proposed Energy Use (Therms/kBtu)}] * \text{Output Heating Capacity (kBtu)}$$

The following equations are used to calculate baseline and proposed natural gas energy use. The savings is the difference between the proposed and baseline calculated usage. This approach allows the savings estimate to account for the operational attributes of the baseline as well as the proposed case, yielding a better estimate than an approach that assumes a particular baseline or proposed energy use to determine savings.

Natural Gas Energy Use Equations (therms / kbtu output)

Building Type	Fan Mode During Occupied Period (Fo)	Equation
Assembly	Continuous	$CZ + Fu * (0.232 * Th + 0.0984 * Ws - 18.79) + Th * (0.00271 * Ws - 0.535) + 0.0142 * Ws$
	Intermittent	$CZ + Fu * (0.00405 * Th + 0.000519 * Ws - 0.11) + Th * (0.0000689 * Ws - 0.0118) + 0.0022 * Ws$
Convenience Store	Continuous	$CZ + Fu * (0.00545 * Th - 0.00251 * Ws + 0.416) + Th * (0.000123 * Ws - 0.0204) + 0.00183 * Ws$
	Intermittent	$CZ + Fu * (0.00231 * Th - 0.0349) + Th * (0.000309 * Ws - 0.0494) + 0.00266 * Ws$
Office – Low Rise	Continuous	$CZ + Fu * (0.0205 * Th + 0.364) + Th * (0.00046 * Ws - 0.0554) + 0.00169 * Ws$
	Intermittent	$CZ + Fu * (0.00745 * Th - 0.142) + Th * (0.00077 * Ws - 0.111) + 0.00199 * Ws$

Building Type	Fan Mode During Occupied Period (F_o)	Equation
Religious	Continuous	$CZ + 0.00791 * Fu * Th + Th * (0.00096 * Ws - 0.167) + 0.00184 * Ws$
	Intermittent	$CZ + Fu * (0.00143 * Th - 0.0309) + Th * (0.0008 * Ws - 0.134) + 0.00219 * Ws$
Restaurant – Fast Food	Continuous	$CZ + Fu * (0.0431 * Th + 0.0424 * Ws - 7.517) + Th * (0.00113 * Ws - 0.213) + 0.0119 * Ws$
	Intermittent	$CZ + Fu * (0.0125 * Th + 0.0036 * Ws - 0.71) + Th * (0.000329 * Ws - 0.0615) + 0.00738 * Ws$
Restaurant – Full Service	Continuous	$CZ + Fu * (0.00445 * Ws - 0.535) + Th * (0.000679 * Ws - 0.1) + 0.00218 * Ws$
	Intermittent	$CZ + Fu * (0.00144 * Th + 0.000262 * Ws - 0.0553) + Th * (0.00018 * Ws - 0.0299) + 0.00166 * Ws$
Retail – Department Store	Continuous	$CZ + 0.00203 * Fu * Th + Th * (0.000591 * Ws - 0.0812) + 0.00194 * Ws$
	Intermittent	$CZ + Th * (0.000406 * Ws - 0.0611) + 0.00228 * Ws$
Retail – Strip Mall	Continuous	$CZ + Fu * (0.00998 * Th + 0.00207 * Ws - 0.206) + Th * (0.000665 * Ws - 0.101) + 0.00292 * Ws$
	Intermittent	$CZ + Fu * (0.00383 * Th - 0.0656) + Th * (0.000575 * Ws - 0.0912) + 0.00249 * Ws$

Where:

CZ = Climate Zone Coefficient

= Depends on Building Type and Fan Mode During Occupied Period (see table below)

Th = Degrees of Heating Setback °F

= Must be between 0-15°F

Fo = Fan Mode During Occupied Period (Note: Commercial mechanical code requires continuous fan operation during occupied periods to meet ventilation requirements.)

= Continuous for occupied fan that runs continuously (e.g. Fan Mode Set to 'On')

= Intermittent for occupied fan that runs intermittently (e.g. Fan Mode Set to 'Auto')

Fu = Fan Mode During Unoccupied Period

= 0 for unoccupied fan that runs continuously (e.g. Fan Mode Set to 'On')

= 1 for unoccupied fan that runs intermittently (e.g. Fan Mode Set to 'Auto')

Ws = Weekly Hours thermostat is in Occupied mode

= Minimum values depends on Building Type (see table below), maximum value of 168 (24/7)

(e.g.: Weekly occupancy schedule of Mon-Sat 8AM-5PM, Sun 9AM-2PM, Ws = 59)

Natural Gas Energy Use Climate Zone Coefficients and Minimum Weekly Hours Occupied

Building Type	Fan Mode During Occupied Period (F_o)	Climate Zone Coefficient (CZ)					Minimum Ws
		1	2	3	4	5	
Assembly	Continuous	19.872	17.83	15.828	15.282	13.482	98
	Intermittent	0.237	0.0989	0.0267	-0.0131	-0.0871	
Convenience Store	Continuous	1.493	1.081	0.782	0.544	0.114	108
	Intermittent	1.128	0.854	0.619	0.437	0.0854	
Office - Low Rise	Continuous	1.718	1.317	0.971	0.739	0.319	55
	Intermittent	3.447	3.022	2.503	2.251	1.646	
Religious Facility	Continuous	6.294	5.55	4.678	4.202	3.122	133
	Intermittent	5.914	5.368	4.557	4.137	3.246	
Restaurant – Fast Food	Continuous	8.383	7.211	6.034	5.767	4.71	108

Building Type	Fan Mode During Occupied Period (<i>Fo</i>)	Climate Zone Coefficient (CZ)					Minimum <i>Ws</i>
		1	2	3	4	5	
	Intermittent	1.227	0.636	0.302	0.102	-0.262	
Restaurant – Full Service	Continuous	5.247	4.484	3.753	3.465	2.627	117
	Intermittent	0.951	0.704	0.51	0.381	0.0746	
Retail – Department Store	Continuous	4.385	3.854	3.192	2.784	1.858	93
	Intermittent	3.061	2.672	2.182	1.829	1.008	
Retail – Strip Mall	Continuous	3.917	3.394	2.728	2.394	1.617	93
	Intermittent	2.659	2.292	1.811	1.543	0.909	

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-PROG-V02-150601**REVIEW DEADLINE: 1/1/2022**

4.4.19 Demand Controlled Ventilation

DESCRIPTION

Demand control ventilation (DCV) adjusts outside ventilation air based on the number of occupants and the ventilation demands that those occupants create. DCV is part of a building's ventilation system control strategy. It may include hardware, software, and controls as an integral part of a building's ventilation design. Active control of the ventilation system provides the opportunity to reduce heating and cooling energy use.

The primary component is a control sensor to communicate either directly with the economizer or with a central computer. The component is most typically a carbon dioxide (CO₂) sensor, occupancy sensor, or turnstile counter. This measure is applicable to multiple building types, and savings are classified by the specific building types defined in the Illinois TRM. This measure is modeled to assume night time set backs are in operation and minimum outside air is being used when the building is unoccupied. Systems that have static louvers or that are open at night will likely have greater savings by using the custom program.

This measure was developed to be applicable to the following program types: RF. If applied to other program types, the measure savings should be verified

DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment condition is defined by new CO₂ sensors installed on return air systems where no other sensors were previously installed. For heating savings, this measure does not apply to any system with terminal reheat (constant volume or variable air volume). For terminal reheat system a custom savings calculation should be used.

DEFINITION OF BASELINE EQUIPMENT

The base case for this measure is a space with no demand control capability. The current code minimum for outside air (OA) is 17 CFM per occupant (ASHRAE 62.1) which is the value assumed in this measure.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The deemed measure life is 10 years and based on CO₂ sensor estimated life.⁴²⁴

DEEMED MEASURE COST

The deemed measure cost is assumed to be the full cost of installation of a DCV retrofit including sensor cost (\$500) and installation (\$1000 labor) for a total of \$1500⁴²⁵.

LOADSHAPE

Commercial ventilation C23

COINCIDENCE FACTOR

N/A

⁴²⁴ During the course of conversations with vendors and Building Automation System (BAS) contractors, it was determined that sensors have to be functional for up to 10 years. It is recommended that they are part of a normal preventive maintenance program in which calibration is an important part of extending useful life. Although they are not subject to mechanical failure, they do fall out of tolerance over time.

⁴²⁵ Discussion with vendors

Algorithm

CALCULATION OF ENERGY SAVINGS**ELECTRIC ENERGY SAVINGS**

For facilities heated by natural gas,

$$\Delta \text{kWh} = \text{Condition Space}/1000 * \text{SF}_{\text{cooling}}$$

For facilities heated by heat pumps,

$$\Delta \text{kWh} = \text{Condition Space}/1000 * \text{SF}_{\text{cooling}} + \text{Condition Space}/1000 * \text{SF}_{\text{Heat HP}}$$

For facilities heated by electric resistance,

$$\Delta \text{kWh} = \text{Condition Space}/1000 * \text{SF}_{\text{cooling}} + \text{Condition Space}/1000 * \text{SF}_{\text{Heat ER}}$$

Where:

Conditioned Space = actual square footage of conditioned space controlled by sensor

$\text{SF}_{\text{cooling}}$ = Cooling Savings Factor

= value in table below based on building type and weather zone

$\text{SF}_{\text{Heat HP}}$ = Heating Savings factor for facilities heated by Heat Pump (HP)

= value in table below based on building type and weather zone

$\text{SF}_{\text{Heat ER}}$ = Heating Savings factor for facilities heated by Electric Resistance (ER)

= value in table below based on building type and weather zone

Saving Factor Tables⁴²⁶

Building Type	SF _{cooling} (kWh/1000 SqFt)				
	Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
Office - Low-rise	454	456	460	456	462
Office - Mid-rise	430	431	432	428	433
Office - High-rise	448	450	452	449	454
Religious Building	493	509	573	584	605
Restaurant	505	515	553	569	581
Retail - Department Store	620	625	630	638	642
Retail - Strip Mall	380	376	356	406	407
Convenience Store	602	603	610	612	614
Elementary School	317	327	352	352	363
High School	305	316	340	340	352
College/University	392	410	434	449	462
Healthcare Clinic	353	358	379	383	389
Lodging	576	578	586	588	591

⁴²⁶ The electric energy savings was calculated using TMY3 weather data and methodology consistent with ASHRAE standards. Savings are calculated on an annual basis for each given temperature zone in Illinois. Energy savings for DCV were developed utilizing standards, inputs and approaches as set forth by ASHRAE 62.1 and 90.1, respectively. Building input parameters like square footage, equipment efficiencies and occupancy match those used in the EFLH calculations. Reference calculation found in Demand Control Ventilation 12-30-13.xls.

Building Type	SF _{cooling} (kWh/1000 SqFt)				
	Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
Manufacturing	481	482	482	477	482
Special Assembly Auditorium	410	427	479	494	514
Default	451	458	475	482	490

Building Type	SF _{Heat HP} (kWh/1000 SqFt)				
	Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
Office - Low-rise	234	203	180	172	148
Office - Mid-rise	156	133	117	117	102
Office - High-rise	209	183	164	153	133
Religious Building	1,495	1,322	1,172	1,116	1,000
Restaurant	1,058	954	828	810	711
Retail - Department Store	365	326	289	283	250
Retail - Strip Mall	244	214	195	185	164
Convenience Store	179	161	141	137	117
Elementary School	652	567	500	470	414
High School	636	553	492	457	406
College/University	1,257	1,105	969	937	789
Healthcare Clinic	443	393	344	331	297
Lodging	204	182	156	153	156
Manufacturing	166	145	125	120	109
Special Assembly Auditorium	1,759	1,551	1,399	1,366	1,202
Default	604	533	472	454	400

Building Type	SF _{Heat ER} (kWh/1000 SqFt)				
	Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
Office - Low-rise	703	610	539	516	445
Office - Mid-rise	467	399	352	352	305
Office - High-rise	627	549	492	458	399
Religious Building	4,486	3,966	3,517	3,348	3,001
Restaurant	3,175	2,862	2,485	2,429	2,134
Retail - Department Store	1,094	979	868	848	750
Retail - Strip Mall	732	641	586	554	492
Convenience Store	537	484	422	410	352
Elementary School	1,956	1,701	1,501	1,409	1,243
High School	1,908	1,659	1,477	1,372	1,219
College/University	3,770	3,314	2,907	2,810	2,368
Healthcare Clinic	1,330	1,179	1,032	992	891
Lodging	611	546	469	458	469
Manufacturing	499	436	375	359	328
Special Assembly Auditorium	5,276	4,652	4,197	4,099	3,606
Default	1,811	1,598	1,415	1,361	1,200

For example: 7,500 SqFt of low-rise office space in Chicago with gas heat.

$$\begin{aligned}\Delta \text{kWh} &= 7,500 / 1000 * 456 \\ &= 3,420 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

NA

NATURAL GAS SAVINGS

$$\Delta \text{therms} = \text{Condition Space} / 1000 * \text{SF}_{\text{Heat Gas}}$$

Where:

$\text{SF}_{\text{Heat Gas}}$ = value in table below based on building type and weather zone⁴²⁷

Building Type	SF _{Heat Gas} (Therm/1000 sq ft)				
	Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
Office - Low-rise	30	26	23	22	19
Office - Mid-rise	20	17	15	15	13
Office- High-rise	27	23	21	20	17
Religious Building	191	169	150	143	128
Restaurant	135	122	106	104	91
Retail - Department Store	47	42	37	36	32
Retail - Strip Mall	31	27	25	24	21
Convenience Store	23	21	18	17	15
Elementary School	83	73	64	60	53
High School	81	71	63	59	52
College/ University	161	141	124	120	101
Healthcare Clinic	57	50	44	42	38
Lodging	26	23	20	20	20
Manufacturing	21	19	16	15	14
Special Assembly Auditorium	225	198	179	175	154
De-fault	77	68	60	58	51

For example: 7500 SqFt of low-rise office space in Chicago.

$$\begin{aligned}\Delta \text{Therms} &= 7,500 * 26 \\ &= 195 \text{ Therms}\end{aligned}$$

⁴²⁷ The natural gas energy savings was calculated using TMY3 weather data and methodology consistent with ASHRAE standards. Savings are calculated on an annual basis for each given temperature zone in Illinois. Energy savings for DCV were developed utilizing standards, inputs and approaches as set forth by ASHRAE 62.1 and 90.1, respectively. Building input parameters like square footage, equipment efficiencies and occupancy match those used in the EFLH calculations. Reference calculation found in Demand Control Ventilation 12-30-13.xls.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-DCV-V04-180101

REVIEW DEADLINE: 1/1/2022

4.4.20 High Turndown Burner for Space Heating Boilers

DESCRIPTION

This measure is for a non-residential boilers equipped with linkageless controls providing space heating with burners having a turndown less than 6:1.⁴²⁸ Turndown is the ratio of the high firing rate to the low firing rate. When boilers are subjected to loads below the low firing rate, the boiler must cycle on/off to meet the load requirements. A higher turndown ratio reduces burner startups, provides better load control, saves wear-and-tear on the burner, and reduces purge-air requirements, all of these benefits result in better overall efficiency.

This measure was developed to be applicable to the following program types: NC, TOS, RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify the boiler linkageless burner must operate with a turndown greater than or equal to 10:1 and be subjected to loads less than or equal to 30%⁴²⁹ of the full fire input MBH for greater than 60%⁴³⁰ of the operating hours.

DEFINITION OF BASELINE EQUIPMENT

The baseline boiler utilizes a linkageless burner with a turndown ration of 6:1 or less and is used primarily for space heating. Redundant boilers do not qualify.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 21 years.⁴³¹

DEEMED MEASURE COST

The deemed installed measure cost including labor is approximately \$2.53/MBtu/hr.⁴³²

DEEMED O&M COST ADJUSTMENTS

N/A

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

⁴²⁸ The standard turndown ratio for boilers is 6:1. Understanding Fuel Savings in the Boiler Room, ASHRAE Journal, David Eoff, December, 2008 p 38

⁴²⁹ Focus on Energy Evaluation, Business Programs: Deemed Savings Manual V1.0, March 22, 2010. This factor implies that boilers are 30% oversized on average.

⁴³⁰ FES Analysis of bin hours based upon a 30% oversizing factor.

⁴³¹ "Burner," Obtained from a nation-wide survey conducted by ASHRAE TC 1.8 (Akalin 1978). Data changed by TC 1.8 in 1986.

⁴³² FES review of PY2/PY3 costs for custom People's and North Shore high turndown burner projects. See High Turndown Costs.xlsx for details.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS SAVINGS

$$\Delta \text{therms} = \text{Ngi} * \text{SF} * \text{EFLH} / 100$$

Where:

Ngi = Boiler gas input size (kBtu/hr) = custom

SF = Savings Factor = Percentage of energy loss per hour

$$= (\sum ((\text{EL}_{\text{base}} - \text{EL}_{\text{eff}}) * \text{H}_{\text{cycling}})) / \text{H}) * 100$$

Where:

EL_{base} = Base Boiler Percentage of energy loss due to cycling at % of Base Boiler Load where
BL_{base} ≤ TDR_{base}

$$= 0.003 * (\text{Cycles}_{\text{base}})^2 - 0.001 * \text{Cycles}_{\text{base}}^{433}$$

Where:

Cycles_{base} = Number of Cycles/hour of base boiler

$$= \text{TDR}_{\text{base}} / \text{BL}$$

Where:

BL = % of full boiler load at bin hours being evaluated. This is assumed to be a straight line based on 0% load at the building balance point (assumed to be 55F), and full load corrected for the oversizing (OSF) at the lowest temperature bin of -10 to -5F.

OSF = Oversizing Factor = 1.3⁴³⁴ or customTDR_{base} = Turndown ratio = 0.33⁴³⁵ or custom

EL_{eff} = Efficient Boiler Percentage of energy loss due to cycling at % of Efficient Boiler Load

$$= 0.003 * (\text{Cycles}_{\text{eff}})^2 - 0.001 * \text{Cycles}_{\text{eff}}$$

Where:

Cycles_{eff} = Number of Cycles/hour

⁴³³ Release 3.0 Operations & Maintenance Best Practices A Guide to Achieving Operational Efficiency, August 2010, Federal Energy Management Program, US Department of Energy. The equation was determined by plotting the values in Table 9.2.1 – Boiler Cycling Energy Loss.

⁴³⁴ PA Consulting, KEMA, Focus on Energy Evaluation, Business Programs: Deemed Savings Manual V1.0, March 22, 2010, Page 4-12.

⁴³⁵ Ibid.

$$= \text{TDR}_{\text{eff}} / \text{BL}$$

Where:

$$\text{TDR}_{\text{eff}} = \text{Turndown ratio} = 0.10^{436} \text{ or custom}$$

$\text{H}_{\text{cycling}}$ = Hours base boiler is cycling at % of base boiler load

= see table below or custom

H = Total Number of Hours in Heating Season

= 4,946 or custom

100 = convert to a percentage

$$\text{SF} = 69.1 / 4946 * 100 = 1.4\% \text{ or custom (see table below for summary of values)}$$

Temperature	H_cycling	BL	EL_base	EL_eff	(EL_base-EL_eff)* Hours
50 to 55	601	6.0%	8.5%	0.7%	47.2
45 to 50	603	12.0%	2.0%	0.0%	12.0
40 to 45	455	18.0%	0.8%	0.0%	3.8
35 to 40	925	24.0%	0.4%	0.0%	4.0
30 to 35	814	30.0%	0.3%	0.0%	2.1
Total					69.1

EFLH = Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use.

100 = convert kBtu to therms

Water IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVAC-HTBC-V04-140601

REVIEW DEADLINE: 1/1/2020

⁴³⁶ 10:1 ratio used to qualify for efficient equipment.

4.4.21 Linkageless Boiler Controls for Space Heating

DESCRIPTION

This measure is for a non-residential boiler providing space heating and currently having single point positioning combustion control. In single-point positioning control, the fuel valve is linked to the combustion air damper via a jackshaft mechanism to maintain correspondence between fuel and combustion air input. Most boilers with single point positioning control do not maintain low excess air levels over their entire firing range. Generally these boilers are calibrated at high fire, but due to the non-linearity required for efficient combustion, excess air levels tend to dramatically increase as the firing rate decreases. Boiler efficiency drops as the excess air levels are increased.

This measure was developed to be applicable to the following program types: TOS, RF.
If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify the boiler burner must have a linkageless control system allowing the combustion air damper position to be adjusted and set for optimal efficiency at several firing rates throughout the burner's firing range. This requires the fuel valve and combustion air damper to each be powered by a separate actuator. An alternative to the combustion air damper is a Variable Speed Drive on the combustion air fan.

DEFINITION OF BASELINE EQUIPMENT

The baseline boiler utilizes single point positioning for the burner combustion control.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 16 years.⁴³⁷

DEEMED MEASURE COST

The deemed measure cost is estimated at \$2.50/MBtu/hr burner input.⁴³⁸

DEEMED O&M COST ADJUSTMENTS

N/A

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

When a Variable Speed Drive is incorporated, electrical savings are calculated according to the "4.4.17 Variable Speed Drive for HVAC Pumps and Cooling Tower Fans" measure.

⁴³⁷ Total number of hours for heating with a base temperature of 55°F for Chicago, IL as noted by National Climate Data Center

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS SAVINGS

$$\Delta \text{Therms} = \text{Ngi} * \text{SF} * \text{EFLH} / 100$$

Where:

Ngi = Boiler gas input size (kBtu/hr) = custom

SF = Savings factor

Note: Savings factor is the percentage increase in efficiency as a result of the addition of linkageless burner controls. At an average boiler load of 35%, single point controls are assumed to have excess air of 91%, while linkageless controls are assumed to have 34% excess air.⁴³⁹ The difference between controls types is 57% at this average operating condition. A 15% reduction in excess air is approximately a 1% increase in efficiency.⁴⁴⁰ Therefore the nominal combustion efficiency increase is $57 / 15 * 1\% = 3.8\%$.

= 3.8%

EFLH = Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use

100 = convert kBtu to therms

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-LBC-V05-160601

REVIEW DEADLINE: 1/1/2022

⁴³⁹ Available and Emerging Technologies for Reducing Greenhouse Gas Emissions from Industrial, Commercial, and Institutional Boilers, Prepared by the Sector Policies and Programs Division Office of Air Quality Planning and Standards U.S. Environmental Protection Agency Research Triangle Park, North Carolina 27711, October 2010, Table 1. ICI Boilers – Summary of Greenhouse Gas Emission Reduction Measures, pg. 8

⁴⁴⁰ Department of Energy (DOE). January 2012, Steam Tip Sheet #4, Improve Your Boiler's Combustion Efficiency. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. This value was determined as an appropriate average over the stack temperatures and excess air levels indicated.

4.4.22 Oxygen Trim Controls for Space Heating Boilers

DESCRIPTION

This measure is for a non-residential boiler providing space heating without oxygen trim combustion controls. Oxygen trim controls limit the amount of excess oxygen provided to the burner for combustion. This oxygen level is dependent upon the amount of air provided. Oxygen trim control converts parallel positioning, linkageless controls, into a closed-loop control configuration with the addition of an exhaust gas analyzer and PID controller. Boilers with oxygen trim controls can maintain a predetermined excess air rate (generally 15% to 30% excess air) over the entire burner firing rate. Boilers without these controls typically have excess air rates around 30% over the entire firing rate. Boiler efficiency drops as the excess air levels are increased.

This measure was developed to be applicable to the following program types: NC, TOS, RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify the boiler burner must have an oxygen control system allowing the combustion air to be adjusted to maintain a predetermined excess oxygen level in the flue exhaust at all firing rates throughout the burner's firing range. This requires an oxygen sensor in the flue exhaust and linkageless fuel valve and combustion air controls.

DEFINITION OF BASELINE EQUIPMENT

The baseline boiler utilizes single point positioning for the burner combustion control.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for the O2 Trim controls is 18 years.⁴⁴¹

DEEMED MEASURE COST

The deemed measure cost is approximately \$23,250.⁴⁴²

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

⁴⁴¹ State of Wisconsin Public Service Commission of Wisconsin Focus on Energy Evaluation Business Programs: Measure Life Study Final Report: August 25, 2009, Table 1-2. Recommended Measure Life by WISEerts Group Description, pg. 1-4.

⁴⁴² CODES AND STANDARDS ENHANCEMENT INITIATIVE (CASE) PROCESS BOILERS, 2013 California Building Energy Efficiency Standards, California Utilities Statewide Codes and Standards Team, October 2011, pg. 22

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms} = \text{Ngi} * \text{SF} * \text{EFLH} / 100$$

Where:

Ngi = Boiler gas input size (kBtu/hr)

= Custom

SF = Savings factor

Note: Savings factor is the percentage reduction in gas consumption as a result of the addition of O2 trim controls. Linkageless controls have an excess air rate of 28% over the entire firing range.⁴⁴³ O2 trim controls have an excess air rate of 15%.⁴⁴⁴ The average difference is 13%. A 15% reduction in excess air is approximately a 1% increase in efficiency.⁴⁴⁵ Therefore the nominal combustion efficiency increase is $13 / 15 * 1\% = 0.87\%$.

= 0.87%

EFLH = Default Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use. When available, actual hours should be used.

100 = convert kBtu to therms

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

The deemed annual Operations and Maintenance cost is \$800.⁴⁴⁶

MEASURE CODE: CI-HVC-O2TC-V01-140601

REVIEW DEADLINE: 1/1/2022

⁴⁴³ Department of Energy (DOE). 2009. Energy Matters newsletter. Fall 2009- Vol. 1, Iss. 1. Washington, DC: U.S. Department of Energy, Office of Energy Efficiency and Renewable Energy, Industrial Technologies Program.

⁴⁴⁴ Ibid

⁴⁴⁶ Department of Energy (DOE). January 2012, Steam Tip Sheet #4, Improving Your Boiler's Combustion Efficiency. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. This value was determined as an appropriate average over the stack temperatures and excess air levels indicated.

4.4.23 Shut Off Damper for Space Heating Boilers or Furnaces

DESCRIPTION

This measure is for non-residential atmospheric boilers or furnaces providing space heating without a shut off damper. When appliances are on standby mode warm room air is drawn through the stack via the draft hood or dilution air inlet at a rate proportional to the stack height, diameter and outdoor temperature. More air is drawn through the vent immediately after the appliance shuts off and the flue is still hot. Installation of a new shut off damper can prevent heat from being drawn up the warm vent and reducing the amount of air that passes through the furnace or boiler heat exchanger. This reduction in air can slightly increase overall operating efficiency by reducing the time needed to achieve steady-state operating conditions.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify the space heating boiler or furnace must have a new electrically or thermally activated shut off damper installed on either the exhaust flue or combustion air intake. Barometric dampers do not qualify. The damper actuation shall be interlocked with the firing controls.

DEFINITION OF BASELINE EQUIPMENT

The baseline boiler or furnace incorporates no shut off damper on the combustion air intake or flue exhaust.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for the shut off damper is 15 years.⁴⁴⁷

DEEMED MEASURE COST

The deemed measure cost for this approximately \$1,500.⁴⁴⁸

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

⁴⁴⁷ State of Wisconsin Public Service Commission of Wisconsin Focus on Energy Evaluation Business Programs: Measure Life Study Final Report: August 25, 2009, Table 1-2. Recommended Measure Life by WISEerts Group Description, pg. 1-4.

⁴⁴⁸ CODES AND STANDARDS ENHANCEMENT INITIATIVE (CASE) PROCESS BOILERS, 2013 California Building Energy Efficiency Standards, California Utilities Statewide Codes and Standards Team, October 2011, pg. 22

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms} = \text{Ngi} * \text{SF} * \text{EFLH} / 100$$

Where:

Ngi = Boiler gas input size (kBtu/hr)

= Custom

SF = Savings factor

= 1%⁴⁴⁹

Note: The savings factor assumes the boiler or furnace is located in an unconditioned space. The savings factor can be higher for those units located within conditioned space.

EFLH = Default Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use. When available, actual hours should be used.

100 = convert kBtu to therms

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

The deemed annual Operations and Maintenance cost is \$112.⁴⁵⁰

MEASURE CODE: CI-HVC-SODP-V01-140601

REVIEW DEADLINE: 1/1/2020

⁴⁴⁹ Based on internet review of savings potential;

“Up to 4%”: Use of Automatic Vent Dampers for New and Existing Boilers and Furnaces, Energy Innovators Initiative Technical Fact Sheet, Office of Energy Efficiency, Canada, 2002

“Up to 1%”: Page 9, The Carbon Trust, “Steam and high temperature hot water boilers”

http://www.carbontrust.com/media/13332/ctv052_steam_and_high_temperature_hot_water_boilers.pdf,

“1 - 2%”: Page 2, Sustainable Energy Authority of Ireland “Steam Systems Technical Guide”,

http://www.seai.ie/Your_Business/Technology/Buildings/Steam_Systems_Technical_Guide.pdf.

⁴⁵⁰ CODES AND STANDARDS ENHANCEMENT INITIATIVE (CASE) PROCESS BOILERS, 2013 California Building Energy Efficiency Standards, California Utilities Statewide Codes and Standards Team, October 2011, pg. 22

4.4.24 Small Pipe Insulation

DESCRIPTION

This measure provides rebates for adding insulation to bare pipes with inner diameters of ½" and ¾". Insulation must be at least one inch thick. Since new construction projects are required by code to have pipe insulation, this measure is only for retrofits of existing facilities. This covers bare straight pipe as well as all fittings.

Default savings are provided on a per linear foot basis. It is assumed that the majority of pipes less than one inch in commercial facilities are used for domestic hot water. However, this measure can cover hydronic heating systems as well as low and high pressure steam systems.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient case is a ½" or ¾" diameter pipe with at least one inch of insulation. Insulation must be protected from damage which includes moisture, sunlight, equipment maintenance and wind. Outdoor pipes should have a weather protective jacket. Insulation must be continuous over straight pipe, elbows and tees.

DEFINITION OF BASELINE EQUIPMENT

The base case for savings estimates is a bare hot water or steam pipe with a fluid temperature of 105 degrees Fahrenheit or greater. Current new construction code requires insulation amounts similar to this measure though this base case is commonly found in older existing buildings.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 15 years.⁴⁵¹

DEEMED MEASURE COST

The incremental measure cost for insulation is the full cost of adding insulation to the pipe. Actual installation costs should be used for the measure cost. For planning purposes, the following costs can be used to estimate the full cost of materials and labor.⁴⁵²

Insulation Thickness	¾" pipe	½" pipe
1"	\$4.45	\$4.15

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

⁴⁵¹ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf

⁴⁵² A market survey was performed to determine these costs.

Algorithm

CALCULATION OF ENERGY SAVINGS**ELECTRIC ENERGY SAVINGS**

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS SAVINGS

$$\begin{aligned}\Delta \text{Therms per foot}^{453} &= [((Q_{\text{base}} - Q_{\text{eff}}) * \text{EFLH}) / (100,000 * \eta_{\text{Boiler}})] * \text{TRF} \\ &= [\text{Modeled or provided by tables below}] * \text{TRF} \\ \Delta \text{Therms} &= (L_{\text{sp}} + L_{\text{oc,i}}) * \Delta \text{therms per foot}\end{aligned}$$

Where:

EFLH = Equivalent Full Load Hours for Heating
 = Actual or defaults by building type provided in Section 4.4, HVAC end use
 For year round recirculation or domestic hot water:
 = 8,766

For heating season recirculation, hours with the outside air temperature below 55°F:

Zone	Hours
Zone 1 (Rockford)	5,039
Zone 2 (Chicago)	4,963
Zone 3 (Springfield)	4,495
Zone 4 (Belleville/	4,021
Zone 5 (Marion)	4,150

Q_{base} = Heat Loss from Bare Pipe (Btu/hr/ft)
 = Calculated where possible using 3E Plusv4.0 software. For defaults see table below

Q_{eff} = Heat Loss from Insulated Pipe (Btu/hr/ft)
 = Calculated where possible using 3E Plusv4.0 software. For defaults see table below

100,000 = conversion factor (1 therm = 100,000 Btu)

η_{Boiler} = Efficiency of the boiler being used to generate the hot water or steam in the pipe
 = 81.9% for water boilers ⁴⁵⁴
 = 80.7% for steam boilers, except multifamily low-pressure ⁴⁵⁵

⁴⁵³This value comes from the reference table "Savings Summary by Building Type and System Type." The formula and the input tables in this section document assumptions used in calculation spreadsheet "Pipe Insulation Savings 2013-11-12.xlsx"

⁴⁵⁴ Average efficiencies of units from the California Energy Commission (CEC).

⁴⁵⁵ Ibid.

= 64.8% for multifamily low-pressure steam boilers ⁴⁵⁶

TRF = Thermal Regain Factor for space type, applied only to space heating energy and is applied to values resulting from Δ therms/ft tables below ⁴⁵⁷

= See table below for base TRF values by pipe location

May vary seasonally such as: TRF[summer] * summer hours + TRF[winter] * winter hours where TRF values reflecting summer and winter conditions are apportioned by the hours for those conditions. TRF may also be adjusted by building specific balance temperature and operating hours above and below that balance temperature.⁴⁵⁸

Pipe Location	Assumed Regain	TRF, Thermal Regain Factor
Outdoor	0%	1.0
Indoor, heated space	85%	0.15
Indoor, semi- heated, (unconditioned space, with heat transfer to conditioned space. E.g.: boiler room, ceiling plenum, basement, crawlspace, wall)	30%	0.70
Indoor, unheated, (no heat transfer to conditioned space)	0%	1.0
Location not specified	85%	0.15
Custom	Custom	1 – assumed regain

L_{sp} = Length of straight pipe to be insulated (linear foot)

L_{oc,i} = Total equivalent length of (elbows and tees) of pipe to be insulated. Use table below to determine equivalent lengths.

Nominal Pipe Diameter	Equivalent Length (ft)	
	90 Degree Elbow	Straight Tee
1/2"	0.04	0.03
3/4"	0.06	0.05

The table below shows the deemed therm savings by building type and region on a per linear foot basis for both ½" and ¾" copper pipe.

The following table provides deemed values for 1/2" copper pipe, temperatures are assumed by category below, and insulation is assumed to be one inch fiberglass.

⁴⁵⁶ Katrakis, J. and T.S. Zawacki. "Field-Measured Seasonal Efficiency of Intermediate-sized Low-Pressure Steam Boilers". ASHRAE V99, pt. 2, 1993.

⁴⁵⁷ Thermal regain for *residential* pipe insulation measures is discussed in Home Energy Services Impact Evaluation, prepared for the Massachusetts Residential Retrofit and Low Income Program Area Evaluation, Cadmus Group, Inc., August 2012 and Andrews, John, Better Duct Systems for Home Heating and Cooling, U.S. Department of Energy, 2001. Recognizing the differences between residential and commercial heating systems, the factors have been adjusted based on professional judgment. This factor would benefit from additional study and evaluation.

⁴⁵⁸ Thermal Regain Factor_4-30-14.docx

Piping Use	Building Type	Annual Therms Saved / Linear Foot				
		Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
Space Heating Non-recirculating	Assembly	0.117	0.120	0.107	0.071	0.109
	Assisted Living	0.110	0.107	0.094	0.069	0.083
	College	0.100	0.093	0.083	0.046	0.055
	Convenience Store	0.097	0.089	0.079	0.057	0.064
	Elementary School	0.116	0.113	0.100	0.069	0.084
	Garage	0.064	0.063	0.056	0.044	0.049
	Grocery	0.105	0.105	0.092	0.057	0.068
	Healthcare Clinic	0.103	0.106	0.092	0.063	0.066
	High School	0.120	0.121	0.109	0.077	0.091
	Hospital - CAV no econ	0.115	0.119	0.101	0.087	0.099
	Hospital - CAV econ	0.117	0.121	0.103	0.089	0.101
	Hospital - VAV econ	0.048	0.045	0.034	0.020	0.022
	Hospital - FCU	0.087	0.099	0.080	0.094	0.127
	Hotel/Motel	0.115	0.112	0.101	0.069	0.084
	Hotel/Motel - Common	0.104	0.106	0.101	0.082	0.086
	Hotel/Motel - Guest	0.115	0.111	0.099	0.066	0.082
	Manufacturing Facility	0.068	0.066	0.061	0.037	0.041
	MF - High Rise	0.100	0.098	0.090	0.076	0.076
	MF - High Rise - Common	0.118	0.115	0.103	0.071	0.092
	MF - High Rise - Residential	0.096	0.096	0.087	0.075	0.073
	MF - Mid Rise	0.109	0.110	0.095	0.070	0.079
	Movie Theater	0.119	0.117	0.109	0.083	0.099
	Office - High Rise - CAV no econ	0.132	0.134	0.122	0.082	0.089
	Office - High Rise - CAV econ	0.136	0.139	0.128	0.088	0.097
	Office - High Rise - VAV econ	0.100	0.102	0.084	0.050	0.055
	Office - High Rise - FCU	0.073	0.072	0.062	0.033	0.035
	Office - Low Rise	0.093	0.093	0.074	0.045	0.052
	Office - Mid Rise	0.103	0.104	0.088	0.056	0.062
	Religious Building	0.105	0.098	0.094	0.069	0.079
	Restaurant	0.088	0.088	0.079	0.060	0.071
	Retail - Department Store	0.091	0.083	0.078	0.051	0.058
	Retail - Strip Mall	0.087	0.081	0.071	0.049	0.053
	Warehouse	0.095	0.089	0.091	0.057	0.070
	Unknown	0.101	0.100	0.089	0.064	0.074
Space Heating - recirculation heating season only	All buildings (Hours below 55°F)	0.329	0.324	0.293	0.262	0.271
Space Heating - recirculation year round	All buildings (All hours)	0.572	0.572	0.572	0.572	0.572
DHW	Recirculation loop	0.572	0.572	0.572	0.572	0.572
Process	Custom	Custom				

The following table provides deemed savings values for 3/4" copper pipe with temperatures assumed by category below, insulation is assumed to be one inch fiberglass.

Piping Use	Building Type	Annual Therms Saved / Linear Foot				
		Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
Space Heating Non-recirculating	Assembly	0.142	0.145	0.129	0.086	0.132
	Assisted Living	0.133	0.130	0.115	0.084	0.101
	College	0.121	0.113	0.101	0.056	0.067
	Convenience Store	0.117	0.108	0.096	0.069	0.077
	Elementary School	0.141	0.137	0.121	0.084	0.102
	Garage	0.078	0.077	0.067	0.054	0.060
	Grocery	0.127	0.127	0.111	0.069	0.083
	Healthcare Clinic	0.125	0.128	0.112	0.076	0.081
	High School	0.146	0.147	0.132	0.094	0.110
	Hospital - CAV no econ	0.140	0.144	0.123	0.105	0.120
	Hospital - CAV econ	0.142	0.147	0.125	0.108	0.123
	Hospital - VAV econ	0.058	0.055	0.041	0.025	0.027
	Hospital - FCU	0.105	0.120	0.098	0.115	0.154
	Hotel/Motel	0.140	0.136	0.122	0.084	0.102
	Hotel/Motel - Common	0.127	0.129	0.123	0.100	0.105
	Hotel/Motel - Guest	0.139	0.135	0.120	0.081	0.099
	Manufacturing Facility	0.083	0.080	0.074	0.045	0.050
	MF - High Rise	0.121	0.119	0.109	0.093	0.093
	MF - High Rise - Common	0.144	0.140	0.125	0.086	0.111
	MF - High Rise - Residential	0.117	0.116	0.105	0.091	0.089
	MF - Mid Rise	0.132	0.134	0.115	0.085	0.096
	Movie Theater	0.144	0.142	0.133	0.101	0.120
	Office - High Rise - CAV no econ	0.160	0.162	0.148	0.099	0.108
	Office - High Rise - CAV econ	0.165	0.169	0.155	0.107	0.118
	Office - High Rise - VAV econ	0.121	0.123	0.102	0.060	0.067
	Office - High Rise - FCU	0.089	0.087	0.075	0.040	0.042
	Office - Low Rise	0.113	0.113	0.090	0.055	0.063
	Office - Mid Rise	0.126	0.126	0.106	0.068	0.075
	Religious Building	0.127	0.119	0.114	0.084	0.095
	Restaurant	0.107	0.107	0.096	0.073	0.086
	Retail - Department Store	0.110	0.101	0.095	0.062	0.071
	Retail - Strip Mall	0.106	0.098	0.086	0.059	0.064
	Warehouse	0.115	0.108	0.111	0.069	0.085
	Unknown	0.123	0.122	0.108	0.078	0.090

Piping Use	Building Type	Annual Therms Saved / Linear Foot				
		Zone 1 (Rockford)	Zone 2 (Chicago)	Zone 3 (Springfield)	Zone 4 (Belleville)	Zone 5 (Marion)
Space Heating - recirculation heating season only	All buildings (Hours below 55°F)	0.399	0.393	0.356	0.319	0.329
Space Heating - recirculation year round	All buildings (All hours)	0.694	0.694	0.694	0.694	0.694
DHW	Recirculation loop	0.694	0.694	0.694	0.694	0.694
Process	Custom	Custom				

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-SPIN-V02-160601

REVIEW DEADLINE: 1/1/2023

4.4.25 Small Commercial Programmable Thermostat Adjustments

DESCRIPTION

This measure involves reprogramming existing commercial programmable thermostats or building automation systems for reduced energy consumption through adjustments of unoccupied heating/cooling setpoints and/or fan control. This measure is limited to packaged HVAC units that are controlled by a commercial thermostat or building automation system. The measure is limited to select building types presented below.

Eligible Small Commercial Building Types

Building Type
Assembly
Convenience Store
Office - Low Rise
Restaurant - Fast Food
Religious Facility
Restaurant - Full Service
Retail - Strip Mall
Retail - Department Store

This measure was developed to be applicable to the following program types: RF, DI.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The criteria for this measure is established by optimizing heating/cooling temperature setbacks and fan operation with a commercial programmable thermostat or building automation system, which reprogrammed to match actual facility occupancy.

DEFINITION OF BASELINE EQUIPMENT

The baseline for this measure is a commercial programmable thermostat or building automation system that is currently operating packaged HVAC units with heating/cooling temperature setbacks and fan operation that do not align with a facilities actual occupancy.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life of a programmable thermostat is assumed to be 8 years⁴⁵⁹ based upon equipment life only⁴⁶⁰. For the purposes of claiming savings for a adjustment of an existing programmable thermostat, this is reduced to a 25% persistence factor to give a final measure life of 2 years. It is recommended that this assumption be evaluated by future energy measurement and verification activities.

DEEMED MEASURE COST

Actual labor costs should be used if the implementation method allows. If unknown the labor cost for this measure is assumed to be \$70.34⁴⁶¹ per thermostat, as summarized in the table below.

⁴⁵⁹ Table 1, HVAC Controls, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007

⁴⁶⁰ Future evaluation is strongly encouraged to inform the persistence of savings to further refine measure life assumption.

⁴⁶¹ RSMears, "Instrumentation and Control for HVAC", Mechanical Cost Data, Kingston, MA: Reed Construction Data, 2010, pg. 255 & 632

Measure	Units	Materials	Labor	Total Cost (including O&P)	City Cost Index (Install Only)*	Total	Source
Adjust Temperature Set Points	4	\$0.00	\$5.95	\$6.55	134.5%	\$35.24	RS Means 2010 (pg 255, Section 23-09-8100)
Adjust Fan Schedule	2	\$0.00	\$11.86	\$13.05	134.5%	\$35.10	RS Means 2010 (pg 255, Section 23-09-8120)
Totals						\$70.34	

* Chicago, IL - Division 23

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS⁴⁶²

$$\Delta \text{kWh} = [\text{Baseline Energy Use (kWh/Ton)} - \text{Proposed Energy Use (kWh/Ton)}] * \text{Cooling Capacity (Tons)}$$

The following equations are used to calculate baseline and proposed electric energy use. The savings is the difference between the proposed and baseline calculated usage. This approach allows the savings estimate to account for the operational attributes of the baseline as well as the proposed case, yielding a better estimate than an approach that assumes a particular baseline or proposed energy use to determine savings.

Electric Energy Use Equations (kWh / ton)

Building Type	Fan Mode During Occupied Period (Fo)	Equation
Assembly	Continuous	$CZ + Fu * (0.83 * Tc + 0.83 * Th + 1.67 * Ws - 293.018) - 0.0922 * Tc * Th + 1.291 * Ws$
	Intermittent	$CZ + Fu * (1.911 - 0.12 * Tc) + Tc * (0.00311 * Ws - 0.229) + 0.11 * Ws$
Convenience Store	Continuous	$CZ + Fu * (-28.629 * Tc - 11.69 * Th + 19.118 * Ws - 2935.12) + 0.909 * Ws$
	Intermittent	$CZ + Tc * (0.0863 * Ws - 12.688) + Th * (0.043 * Ws - 6.38) + 1.669 * Ws$
Office – Low Rise	Continuous	$CZ + Fu * (7.082 * Tc - 41.199 * Th + 18.734 * Ws - 3288.55) + Tc * (0.205 * Ws - 34.929)$
	Intermittent	$CZ + Tc * (0.0806 * Ws - 8.984) + Th * (0.0864 * Ws - 9.558) + 1.178 * Ws$
Religious	Continuous	$CZ + Fu * (-1.579 * Tc - 18.14 * Th + 15.01 * Ws - 2417.74) + Tc * (0.177 * Ws - 26.412)$
	Intermittent	$CZ + Fu * (0.266 * Tc - 2.067) + Tc * (0.0295 * Ws - 4.502) + Th * (0.0517 * Ws - 8.251) + 0.735 * Ws$
Restaurant – Fast Food	Continuous	$CZ + Fu * (0.678 * Tc + 0.257 * Th + 2.88 * Ws - 494.006) + Tc * (0.0231 * Ws - 4.074) + Th * (0.00936 * Ws - 1.655) + 0.918 * Ws$
	Intermittent	$CZ + Fu * (0.377 * Tc + 0.124 * Th + 0.13 * Ws - 24.893) + Tc * (-0.0143 * Th + 0.0166 * Ws - 2.691) + 0.898 * Ws$
Restaurant – Sit Down	Continuous	$CZ + Fu * (-8.41 * Th + 11.766 * Ws - 1910.81) + Tc * (0.282 * Ws - 43.851)$
	Intermittent	$CZ + 0.123 * Fu * Tc + Tc * (0.0561 * Ws - 8.237) + Th * (0.0219 * Ws - 3.284) + 1.038 * Ws$
Retail – Large	Continuous	$CZ + Fu * (-1.475 * Th + 0.755 * Ws - 114.373) + Th * (0.151 * Ws - 24.016) + 1.612 * Ws$

⁴⁶² Savings equations and factors determined by regression of results of a series of eQuest simulations. See Programmable T-Stat Work Paper_PECI_FinalDraft_140730_Redline.docx for details.

Building Type	Fan Mode During Occupied Period (Fo)	Equation
	Intermittent	$CZ + Tc * (0.0173 * Ws - 1.912) + Th * (0.0249 * Ws - 3.29) + 0.511 * Ws$
Retail – Strip Mall	Continuous	$CZ + Fu * (1.077 * Tc - 10.697 * Th + 6.91 * Ws - 1117.18) + Tc * (0.0583 * Ws - 7.54) + 1.231 * Ws$
	Intermittent	$CZ + 0.0894 * Fu * Tc + Th * (-0.0142 * Tc + 0.04 * Ws - 5.278) + 0.884 * Ws$

Where:

- CZ** = Climate Zone Coefficient
 = Depends on Building Type and Fan Mode During Occupied Period (see table below)
- Tc** = Degrees of Cooling Setback °F
 = Must be between 0-15°F
- Th** = Degrees of Heating Setback °F
 = Must be between 0-15°F
- Fo** = Fan Mode During Occupied Period (Note: Commercial mechanical code requires continuous fan operation during occupied periods to meet ventilation requirements.)
 = Continuous for occupied fan that runs continuously (e.g. Fan Mode Set to 'On')
 = Intermittent for occupied fan that runs intermittently (e.g. Fan Mode Set to 'Auto')
- Fu** = Fan Mode during Unoccupied Period
 = 0 for unoccupied fan that runs continuously (e.g. Fan Mode Set to 'On')
 = 1 for unoccupied fan that runs intermittently (e.g. Fan Mode Set to 'Auto')
- Ws** = Weekly Hours thermostat is in Occupied mode,
 = Minimum values depend on Building Type (see table below), maximum value of 168 (24/7)
 ex: Weekly occupancy schedule of Mon-Sat 8AM-5PM, Sun 9AM-2PM, Ws = 59

Electric Energy Use Climate Zone Coefficients and Minimum Weekly Hours Occupied

Building Type	Fan Mode During Occupied Period (Fo)	Climate Zone Coefficient (CZ)					Minimum Ws
		1	2	3	4	5	
Assembly	Continuous	911.366	928.924	1152.83	1208.999	1210.173	98
	Intermittent	735.752	762.831	966.562	998.927	1028.906	
Convenience Store	Continuous	4817.094	4832.784	5139.133	5182.161	5208.608	108
	Intermittent	1478.133	1514.568	1784.384	1843.463	1930.47	
Office - Low Rise	Continuous	5047.662	5039.592	5187.924	5217.672	5177.449	55
	Intermittent	825.072	808.965	946.571	979.421	945.418	
Religious Facility	Continuous	4197.117	4172.858	4380.025	4370.008	4356.054	133
	Intermittent	632.404	603.395	678.294	664.717	616.853	
Restaurant – Fast Food	Continuous	1342.988	1378.661	1664.018	1714.201	1727.841	108
	Intermittent	993.764	1039.643	1307.8	1340.544	1389.791	
Restaurant – Full Service	Continuous	4070.35	4094.742	4428.966	4501.829	4522.522	117
	Intermittent	1472.014	1516.05	1856.108	1938.441	2056.45	
	Continuous	1510.201	1496.47	1706.105	1716.128	1688.464	93

Building Type	Fan Mode During Occupied Period (Fo)	Climate Zone Coefficient (CZ)					Minimum Ws
		1	2	3	4	5	
Retail – Department Store	Intermittent	701.27	702.129	847.735	875.12	881.677	
Retail – Strip Mall	Continuous	1926.294	1930.137	2156.856	2174.435	2165.03	93
	Intermittent	656.479	673.257	835.906	850.322	869.921	

EXAMPLE

A low rise office building in Rockford (Climate Zone 1) is occupied Mon-Fri 7AM-6PM and is heated and cooled with a packaged Gas (150 kBtu output) / DX (10 Ton) RTU which is controlled by a programmable thermostat. When the technician reviews the thermostat schedule they find the unoccupied schedule is programmed incorrectly. During the unoccupied periods the fan is programmed correctly, and runs in intermittent “auto” mode, although the heating and cooling temperature setpoints are not setback.

The technician adjusts the unoccupied schedule to include a 10°F cooling and heating temperature setback during the unoccupied periods.

$$\Delta \text{kWh} = [\text{Baseline Energy Use (kWh/Ton)} - \text{Proposed Energy Use (kWh/Ton)}] * \text{Cooling Capacity (Tons)}$$

$$\text{Baseline Energy Use (kWh/Ton)} = \text{Equation for Office Low Rise, } Fo=\text{Continuous}$$

$$= CZ + Fu * (7.082 * Tc - 41.199 * Th + 18.734 * Ws - 3288.55) + Tc * (0.205 * Ws - 34.929)$$

$$= 5047.662 + 1 * (7.082 * 0 - 41.199 * 0 + 18.734 * 55 - 3288.55) + 0 * (0.205 * 55 - 34.929)$$

$$= 2,789.482 \text{ kWh/Ton}$$

$$\text{Proposed Energy Use (kWh/Ton)} = \text{Equation for Office Low Rise, } Fo=\text{Continuous}$$

$$= CZ + Fu * (7.082 * Tc - 41.199 * Th + 18.734 * Ws - 3288.55) + Tc * (0.205 * Ws - 34.929)$$

$$= 5047.662 + 1 * (7.082 * 10 - 41.199 * 10 + 18.734 * 55 - 3288.55) + 10 * (0.205 * 55 - 34.929)$$

$$= 2,211.722 \text{ kWh/Ton}$$

$$\Delta \text{kWh} = [2,789.482 \text{ (kWh/Ton)} - 2,211.722 \text{ (kWh/Ton)}] * 10 \text{ Tons}$$

$$= 577.71 \text{ kWh/Ton} * 10 \text{ Tons}$$

$$= 5777.1 \text{ kWh}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms} = [\text{Baseline Energy Use (Therms/kBtu/h)} - \text{Proposed Energy Use (Therms/kBtu/h)}] * \text{Output Heating Capacity (kBtu/h)}$$

The following equations are used to calculate baseline and proposed natural gas energy use. The savings is the difference between the proposed and baseline calculated usage. This approach allows the savings estimate to account for the operational attributes of the baseline as well as the proposed case, yielding a better estimate than an approach that assumes a particular baseline or proposed energy use to determine savings.

Natural Gas Energy Use Equations (therms / kbtu)

Building Type	Fan Mode During Occupied Period (Fo)	Equation
Assembly	Continuous	$CZ + Fu * (0.232 * Th + 0.0984 * Ws - 18.79) + Th * (0.00271 * Ws - 0.535) + 0.0142 * Ws$
	Intermittent	$CZ + Fu * (0.00405 * Th + 0.000519 * Ws - 0.11) + Th * (0.0000689 * Ws - 0.0118) + 0.0022 * Ws$
Convenience Store	Continuous	$CZ + Fu * (0.00545 * Th - 0.00251 * Ws + 0.416) + Th * (0.000123 * Ws - 0.0204) + 0.00183 * Ws$
	Intermittent	$CZ + Fu * (0.00231 * Th - 0.0349) + Th * (0.000309 * Ws - 0.0494) + 0.00266 * Ws$
Office – Low Rise	Continuous	$CZ + Fu * (0.0205 * Th + 0.364) + Th * (0.00046 * Ws - 0.0554) + 0.00169 * Ws$
	Intermittent	$CZ + Fu * (0.00745 * Th - 0.142) + Th * (0.00077 * Ws - 0.111) + 0.00199 * Ws$
Religious	Continuous	$CZ + 0.00791 * Fu * Th + Th * (0.00096 * Ws - 0.167) + 0.00184 * Ws$
	Intermittent	$CZ + Fu * (0.00143 * Th - 0.0309) + Th * (0.0008 * Ws - 0.134) + 0.00219 * Ws$
Restaurant – Fast Food	Continuous	$CZ + Fu * (0.0431 * Th + 0.0424 * Ws - 7.517) + Th * (0.00113 * Ws - 0.213) + 0.0119 * Ws$
	Intermittent	$CZ + Fu * (0.0125 * Th + 0.0036 * Ws - 0.71) + Th * (0.000329 * Ws - 0.0615) + 0.00738 * Ws$
Restaurant – Sit Down	Continuous	$CZ + Fu * (0.00445 * Ws - 0.535) + Th * (0.000679 * Ws - 0.1) + 0.00218 * Ws$
	Intermittent	$CZ + Fu * (0.00144 * Th + 0.000262 * Ws - 0.0553) + Th * (0.00018 * Ws - 0.0299) + 0.00166 * Ws$
Retail – Large	Continuous	$CZ + 0.00203 * Fu * Th + Th * (0.000591 * Ws - 0.0812) + 0.00194 * Ws$
	Intermittent	$CZ + Th * (0.000406 * Ws - 0.0611) + 0.00228 * Ws$
Retail – Strip Mall	Continuous	$CZ + Fu * (0.00998 * Th + 0.00207 * Ws - 0.206) + Th * (0.000665 * Ws - 0.101) + 0.00292 * Ws$
	Intermittent	$CZ + Fu * (0.00383 * Th - 0.0656) + Th * (0.000575 * Ws - 0.0912) + 0.00249 * Ws$

Where:

- CZ** = Climate Zone Coefficient
= Depends on Building Type and Fan Mode During Occupied Period (see table below)
- Th** = Degrees of Heating Setback °F
= Must be between 0-15°F
- Fo** = Fan Mode During Occupied Period (Note: Commercial mechanical code requires continuous fan operation during occupied periods to meet ventilation requirements.)
= Continuous for occupied fan that runs continuously (e.g. Fan Mode Set to 'On')
= Intermittent for occupied fan that runs intermittently (e.g. Fan Mode Set to 'Auto')
- Fu** = Fan Mode during Unoccupied Period

= 0 for unoccupied fan that runs continuously (e.g. Fan Mode Set to 'On')

= 1 for unoccupied fan that runs intermittently (e.g. Fan Mode Set to 'Auto')

Ws = Weekly Hours thermostat is in Occupied mode,

= Minimum values depends on Building Type (see table below), maximum value of 168 (24/7)

ex: Weekly occupancy schedule of Mon-Sat 8AM-5PM, Sun 9AM-2PM, Ws = 59.

Natural Gas Energy Use Climate Zone Coefficients and Minimum Weekly Hours Occupied

Building Type	Fan Mode During Occupied Period (Fo)	Climate Zone Coefficient (CZ)					Minimum Ws
		1	2	3	4	5	
Assembly	Continuous	19.872	17.83	15.828	15.282	13.48 2	98
	Intermittent	0.237	0.0989	0.0267	0.0131	0.087 1	
Convenience Store	Continuous	1.493	1.081	0.782	0.544	0.114	108
	Intermittent	1.128	0.854	0.619	0.437	0.085 4	
Office - Low Rise	Continuous	1.718	1.317	0.971	0.739	0.319	55
	Intermittent	3.447	3.022	2.503	2.251	1.646	
Religious Facility	Continuous	6.294	5.55	4.678	4.202	3.122	133
	Intermittent	5.914	5.368	4.557	4.137	3.246	
Restaurant – Fast Food	Continuous	8.383	7.211	6.034	5.767	4.71	108
	Intermittent	1.227	0.636	0.302	0.102	-0.262	
Restaurant – Full Service	Continuous	5.247	4.484	3.753	3.465	2.627	117
	Intermittent	0.951	0.704	0.51	0.381	0.074 6	
Retail – Department Store	Continuous	4.385	3.854	3.192	2.784	1.858	93
	Intermittent	3.061	2.672	2.182	1.829	1.008	
Retail – Strip Mall	Continuous	3.917	3.394	2.728	2.394	1.617	93
	Intermittent	2.659	2.292	1.811	1.543	0.909	

EXAMPLE

A low rise office building in Rockford (Climate Zone 1) is occupied Mon-Fri 7AM-6PM and is heated and cooled with a packaged Gas (150 kBtu output) / DX (10 Ton) RTU which is controlled by a programmable thermostat. When the technician reviews the thermostat schedule they find the unoccupied schedule is programmed incorrectly. During the unoccupied periods the fan is programmed correctly, and runs in intermittent "auto" mode, although the heating and cooling temperature setpoints are not setback.

The technician adjusts the unoccupied schedule to include a 10°F cooling and heating temperature setback during the unoccupied periods.

$$\Delta \text{Therms} = [\text{Baseline Energy Use (Therms/kBtuh)} - \text{Proposed Energy Use (Therms/kBtuh)}] * \text{Output Heating Capacity (kBtuh)}$$

Baseline Energy Use (Therms/kBtuh) = Equation for Office Low Rise, F_o =Continuous

$$= CZ + Fu * (0.0205 * Th + 0.364) + Th * (0.00046 * Ws - 0.0554) + 0.00169 * Ws$$

$$= 1.718 + 1 * (0.0205 * 0 + 0.364) + 0 * (0.00046 * 55 - 0.0554) + 0.00169 * 55$$

$$= 2.17495 \text{ Therms/kBtuh output}$$

Proposed Energy Use (Therms/kBtuh) = Equation for Office Low Rise, F_o =Continuous

$$= CZ + Fu * (0.0205 * Th + 0.364) + Th * (0.00046 * Ws - 0.0554) + 0.00169 * Ws$$

$$= 1.718 + 1 * (0.0205 * 10 + 0.364) + 10 * (0.00046 * 55 - 0.0554) + 0.00169 * 55$$

$$= 2.07895 \text{ Therms/kBtuh output}$$

$$\Delta \text{Therms} = [2.17495 \text{ (Therms/kBtuh output)} - 2.07895 \text{ (Therms/kBtuh output)}] * 150 \text{ kBtuh output}$$

$$= 0.096 \text{ (Therms/kBtuh output)} * 150 \text{ kBtuh output}$$

$$= 14.4 \text{ Therms}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-PRGA-V01-150601

REVIEW DEADLINE: 1/1/2022

4.4.26 Variable Speed Drives for HVAC Supply and Return Fans

DESCRIPTION

This measure is applied to variable speed drives (VSD) which are installed on HVAC supply fans and return fans. There is a separate measure for HVAC pumps and cooling tower fans. All other VSD applications require custom analysis by the program administrator. The VSD will modulate the speed of the motor when it does not need to run at full load. Since the power of the motor is proportional to the cube of the speed for these types of applications, significant energy savings will result.

This measure was developed to be applicable to the following program types: TOS, RF. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The VSD is applied to a motor which does not have a VSD. The application must have a variable load and installation is to include the necessary controls. Savings are based on application of VSDs to a range of baseline load conditions including no control, inlet guide vanes, outlet guide vanes and throttling valves.

DEFINITION OF BASELINE EQUIPMENT

The time of sale baseline is a new motor installed without a VSD or other methods of control. Retrofit baseline is an existing motor operating as is. Retrofit baselines may or may not include guide vanes, throttling valves or other methods of control. This information shall be collected from the customer.

Installations of new equipment with VSDs which are required by IECC 2012 or 2015 as adopted by the State of Illinois are not eligible for incentives.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life for HVAC application is 15 years;⁴⁶³ measure life for process is 10 years.⁴⁶⁴

DEEMED MEASURE COST

Customer provided costs will be used when available. Default measure costs⁴⁶⁵ are noted below for up to 20 hp motors. Custom costs must be gathered from the customer for motor sizes not listed below.

HP	Cost
1 -5 HP	\$ 1,330
7.5 HP	\$ 1,622
10 HP	\$ 1,898
15 HP	\$ 2,518
20 HP	\$ 3,059

LOADSHAPE

Loadshape C39 - VFD - Supply fans <10 HP

Loadshape C40 - VFD - Return fans <10 HP

Loadshape C41 - VFD - Exhaust fans <10 HP

⁴⁶³ Efficiency Vermont TRM 10/26/11 for HVAC VSD motors

⁴⁶⁴ DEER 2008

⁴⁶⁵ Ohio TRM 8/6/2010 varies by motor/fan size based on equipment costs from Granger 2008 Catalog pp 286-289, average across available voltages and models. Labor costs from RS Means Data 2008 Ohio average cost adjustment applied.

COINCIDENCE FACTOR

The demand savings factor (DSF) is already based upon coincident savings, and thus there is no additional coincidence factor for this characterization.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS⁴⁶⁶

$$\begin{aligned}
 kWh_{Base} &= \left(0.746 \times HP \times \frac{LF}{\eta_{motor}} \right) \times RHRS_{Base} \times \sum_{0\%}^{100\%} (\%FF \times PLR_{Base}) \\
 kWh_{Retrofit} &= \left(0.746 \times HP \times \frac{LF}{\eta_{motor}} \right) \times RHRS_{base} \times \sum_{0\%}^{100\%} (\%FF \times PLR_{Retrofit}) \\
 \Delta kWh_{fan} &= kWh_{Base} - kWh_{Retrofit} \\
 \Delta kWh_{total} &= \Delta kWh_{fan} \times (1 + IE_{energy})
 \end{aligned}$$

Where:

kWh_{Base}	= Baseline annual energy consumption (kWh/yr)
$kWh_{Retrofit}$	= Retrofit annual energy consumption (kWh/yr)
ΔkWh_{fan}	= Fan-only annual energy savings
ΔkWh_{total}	= Total project annual energy savings
0.746	= Conversion factor for HP to kWh
HP	= Nominal horsepower of controlled motor
LF	= Load Factor; Motor Load at Fan Design CFM (Default = 65%) ⁴⁶⁷
η_{motor}	= Installed nominal/nameplate motor efficiency
Default motor is a NEMA Premium Efficiency, ODP, 4-pole/1800 RPM fan motor	

⁴⁶⁶ Methodology developed and tested in Del Balso, Ryan Joseph. "Investigation into the Reliability of Energy Efficiency/Demand Side Management Savings Estimates for Variable Frequency Drives in Commercial Applications". A project report submitted to the Faculty of the Graduate School of the University of Colorado, 2013.

⁴⁶⁷ Lawrence Berkeley National Laboratory, and Resource Dynamics Corporation. (2008). "Improving Motor and Drive System Performance; A Sourcebook for Industry". U.S. Department of Energy, Office of Energy Efficiency and Renewable Energy. Golden, CO: National Renewable Energy Laboratory.

NEMA Premium Efficiency Motors Default Efficiencies⁴⁶⁸

Size HP	Open Drip Proof (ODP)			Totally Enclosed Fan-Cooled (TEFC)		
	# of Poles			# of Poles		
	6	4	2	6	4	2
	Speed (RPM)			Speed (RPM)		
	1200	1800 Default	3600	1200	1800	3600
1	0.825	0.855	0.770	0.825	0.855	0.770
1.5	0.865	0.865	0.840	0.875	0.865	0.840
2	0.875	0.865	0.855	0.885	0.865	0.855
3	0.885	0.895	0.855	0.895	0.895	0.865
5	0.895	0.895	0.865	0.895	0.895	0.885
7.5	0.902	0.910	0.885	0.910	0.917	0.895
10	0.917	0.917	0.895	0.910	0.917	0.902
15	0.917	0.930	0.902	0.917	0.924	0.910
20	0.924	0.930	0.910	0.917	0.930	0.910
25	0.930	0.936	0.917	0.930	0.936	0.917
30	0.936	0.941	0.917	0.930	0.936	0.917
40	0.941	0.941	0.924	0.941	0.941	0.924
50	0.941	0.945	0.930	0.941	0.945	0.930
60	0.945	0.950	0.936	0.945	0.950	0.936
75	0.945	0.950	0.936	0.945	0.954	0.936
100	0.950	0.954	0.936	0.950	0.954	0.941
125	0.950	0.954	0.941	0.950	0.954	0.950
150	0.954	0.958	0.941	0.958	0.958	0.950
200	0.954	0.958	0.950	0.958	0.962	0.954
250	0.954	0.958	0.950	0.958	0.962	0.958
300	0.954	0.958	0.954	0.958	0.962	0.958
350	0.954	0.958	0.954	0.958	0.962	0.958
400	0.958	0.958	0.958	0.958	0.962	0.958
450	0.962	0.962	0.958	0.958	0.962	0.958
500	0.962	0.962	0.958	0.958	0.962	0.958

$RHRS_{Base}$ = Annual operating hours for fan motor based on building type

Default hours are provided for HVAC applications which vary by HVAC application and building type⁴⁶⁹. When available, actual hours should be used.

Building Type	Total Fan Run Hours
Assembly	7235
Assisted Living	8760
College	6103
Convenience Store	7004
Elementary School	7522

⁴⁶⁸ Douglass, J. (2005). Induction Motor Efficiency Standards. Washington State University and the Northwest Energy Efficiency Alliance, Extension Energy Program, Olympia, WA. Retrieved October 17, 2013, from http://www1.eere.energy.gov/manufacturing/tech_assistance/pdfs/motor_efficiency_standards.pdf

⁴⁶⁹ Hours per year are estimated using the eQuest models as the total number of hours the fans are operating for heating, cooling and ventilation for each building type.

Building Type	Total Fan Run Hours
Garage	7357
Grocery	7403
Healthcare Clinic	6345
High School	7879
Hospital - VAV econ	8760
Hospital - CAV econ	8760
Hospital - CAV no econ	8760
Hospital - FCU	8760
Manufacturing Facility	8706
MF - High Rise	8760
MF - Mid Rise	8760
Hotel/Motel - Guest	8760
Hotel/Motel - Common	8760
Movie Theater	7505
Office - High Rise - VAV econ	6064
Office - High Rise - CAV econ	5697
Office - High Rise - CAV no econ	5682
Office - High Rise - FCU	6163
Office - Low Rise	6288
Office - Mid Rise	6125
Religious Building	7380
Restaurant	7809
Retail - Department Store	6890
Retail - Strip Mall	6846
Warehouse	6786
Unknown	7100

$\%FF$ = Percentage of run-time spent within a given flow fraction range

Default Fan Duty Cycle Based on 2012 ASHRAE Handbook; HVAC Systems and Equipment, page 45.11, Figure 12.

Flow Fraction (% of design cfm)	Percent of Time at Flow Fraction
0% to 10%	0.0%
10% to 20%	1.0%
20% to 30%	5.5%
30% to 40%	15.5%
40% to 50%	22.0%
50% to 60%	25.0%
60% to 70%	19.0%
70% to 80%	8.5%
80% to 90%	3.0%
90% to 100%	0.5%

PLR_{Base} = Part load ratio for a given flow fraction range based on the baseline flow control type

$PLR_{Retrofit}$ = Part load ratio for a given flow fraction range based on the retrofit flow control type

Control Type	Flow Fraction									
	10%	20%	30%	40%	50%	60%	70%	80%	90%	100%
No Control or Bypass Damper	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00	1.00
Discharge Dampers	0.46	0.55	0.63	0.70	0.77	0.83	0.88	0.93	0.97	1.00
Outlet Damper, BI & Airfoil Fans	0.53	0.53	0.57	0.64	0.72	0.80	0.89	0.96	1.02	1.05
Inlet Damper Box	0.56	0.60	0.62	0.64	0.66	0.69	0.74	0.81	0.92	1.07
Inlet Guide Vane, BI & Airfoil Fans	0.53	0.56	0.57	0.59	0.60	0.62	0.67	0.74	0.85	1.00
Inlet Vane Dampers	0.38	0.40	0.42	0.44	0.48	0.53	0.60	0.70	0.83	0.99
Outlet Damper, FC Fans	0.22	0.26	0.30	0.37	0.45	0.54	0.65	0.77	0.91	1.06
Eddy Current Drives	0.17	0.20	0.25	0.32	0.41	0.51	0.63	0.76	0.90	1.04
Inlet Guide Vane, FC Fans	0.21	0.22	0.23	0.26	0.31	0.39	0.49	0.63	0.81	1.04
VFD with duct static pressure controls	0.09	0.10	0.11	0.15	0.20	0.29	0.41	0.57	0.76	1.01
VFD with low/no duct static pressure	0.05	0.06	0.09	0.12	0.18	0.27	0.39	0.55	0.75	1.00

Provided below is the resultant values based upon the defaults provided above:

Control Type	$\sum_{0\%}^{100\%} (\%FF \times PLR_{Base})$
No Control or Bypass Damper	1.00
Discharge Dampers	0.80
Outlet Damper, BI & Airfoil Fans	0.78
Inlet Damper Box	0.69
Inlet Guide Vane, BI & Airfoil Fans	0.63
Inlet Vane Dampers	0.53
Outlet Damper, FC Fans	0.53
Eddy Current Drives	0.49
Inlet Guide Vane, FC Fans	0.39
VFD with duct static pressure controls	0.30
VFD with low/no duct static pressure	0.27

IE_{energy} = HVAC interactive effects factor for energy (default = 15.7%)

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\begin{aligned}
 kW_{Base} &= \left(0.746 \times HP \times \frac{LF}{\eta_{motor}} \right) \times PLR_{Base,FFpeak} \\
 kW_{Retrofit} &= \left(0.746 \times HP \times \frac{LF}{\eta_{motor}} \right) \times PLR_{Retrofit,FFpeak} \\
 \Delta kW_{fan} &= kW_{Base} - kW_{Retrofit} \\
 \Delta kW_{total} &= \Delta kW_{fan} \times (1 + IE_{demand})
 \end{aligned}$$

Where:

kW_{Base} = Baseline summer coincident peak demand (kW)

$kW_{Retrofit}$	= Retrofit summer coincident peak demand (kW)
ΔkW_{fan}	= Fan-only summer coincident peak demand impact
ΔkW_{total}	= Total project summer coincident peak demand impact
$PLR_{Base,FFpeak}$	= The part load ratio for the average flow fraction between the peak daytime hours during the weekday peak time period based on the baseline flow control type (default average flow fraction during peak period = 90%)
$PLR_{Retrofit,FFpeak}$	= The part load ratio for the average flow fraction between the peak daytime hours during the weekday peak time period based on the retrofit flow control type (default average flow fraction during peak period = 90%)
IE_{demand}	= HVAC interactive effects factor for summer coincident peak demand (default = 15.7%)

FOSSIL FUEL IMPACT DESCRIPTIONS AND CALCULATION

There are no expected fossil fuel impacts for this measure.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-VSDF-V02-160601

REVIEW DEADLINE: 1/1/2021

4.4.27 Energy Recovery Ventilator

DESCRIPTION

This measure includes the addition of energy recovery equipment on existing or new unitary equipment, where energy recovery is not required by the IECC 2012/2015. This measure analyzes the heating savings potential from recovering energy from exhaust or relief building air. This measure assumes during unoccupied hours of the building no exhaust or relief air is available for energy recovery.

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

Efficient equipment is unitary equipment that incorporates energy recovery not required by the IECC 2012/2015.

DEFINITION OF BASELINE EQUIPMENT

The baseline is unitary equipment not required by IECC 2012/2015 to incorporate energy recovery.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for the domestic energy recovery equipment is 15 years.⁴⁷⁰

DEEMED MEASURE COST

The incremental cost for this measure assumes cost of cabinet and controls incorporated into packaged and built up air handler units. Additionally it assumes 1 to 1 ratio of fresh and exhausted air.

Energy Recovery Equipment Type	Incremental Cost \$/CFM ⁴⁷¹
Fixed Plate	\$6
Rotary Wheel	\$6
Heat Pipe	\$6

DEEMED O&M COST ADJUSTMENTS

There are no expected O&M savings associated with this measure.

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

⁴⁷⁰ Assumed service life limited by controls -" Demand Control Ventilation Using CO2 Sensors", pg. 19, by US Department of Energy Efficiency and Renewable Energy

⁴⁷¹"Map to HVAC Solutions", by Michigan Air, Issue 3, 2006

Algorithm

CALCULATION OF ENERGY SAVINGS**ELECTRIC ENERGY SAVINGS****SUMMER COINCIDENT PEAK DEMAND SAVINGS**

There are no anticipated electrical savings from this measure as it is assumed that the additional fan energy due to the increased static pressure drop offsets cooling energy savings. Where this is not expected to be the case, a custom calculation should be used to determine the savings.

NATURAL GAS SAVINGS

Gas savings algorithm is derived from the following:

$$\Delta \text{Therms} = (\text{Design Heating Load} * \text{TE_ERV} * \text{EFLH} * \text{OccHours}/24) / (100,000 * \mu \text{Heat})$$

Where:

$$\text{Design Heating Load} = (1.08 * \text{CFM} * \Delta T)$$

$$1.08 = \text{A constant for sensible heat equations (BTU/h/CFM.}^{\circ}\text{F)}$$

$$\text{CFM} = \text{Cubic Feet per Minute of Energy Recovery Ventilator}$$

$$\Delta T = T_{\text{RA}} - T_{\text{DD}}$$

$$T_{\text{RA}} = \text{Temperature of the Return Air} = 70^{\circ}\text{F or custom}$$

$$T_{\text{DD}} = \text{Temperature on design day of outside air}^{472}$$

$$= (\text{see Table below}) \text{ or custom}$$

Zone	Weather Station	T_DD, Temperature, °F
1	Greater Rockford	-5.8
2	Chicago/O'Hare ARPT.	-1.5
3	Springfield/Capital	0.4
4	Scott AFB MidAmerica	9.0
5	Cape Girardeau Regional	9.7
Average	-	2.4

$$\text{TE_ERV} = \text{Thermal Effectiveness of Energy Recovery Equipment}^{473}$$

$$= (\text{see Table below}) \text{ or custom}$$

Heat Recovery Equipment Type	TE_ERV (%)
Fixed Plate	0.65
Rotary Equipment	0.68
Heat Pipe	0.55

⁴⁷²Weather Station Data, 99.6% Heating DB - 2013 Fundamentals, ASHRAE Handbook

⁴⁷³Energy Recovery Fact Sheet - Center Point Energy, MN

EFLH = Equivalent Full Load Hours for heating are provided in section 4.4 HVAC End Use

OccHours = Average Hours per day facility is occupied
= custom or use Modeling Inputs in eQuest models:

	Weekday	Saturday	Sunday	Holiday	Annual Operating Hours	OccHours
Assembly/Convention Center	10am-9pm	10am-9pm	10am-9pm	closed	3905	10.7
Assisted Living	24/7	24/7	24/7	24/7	8760	24.0
College	8am-9pm	closed	closed	closed	3263	8.9
Convenience Store	7am-10pm	9am-9pm	10am-5pm	10am-5pm	4823	13.2
Elementary School	8am-4pm (20% in summer)	closed	closed	closed	1606	4.4
Garage	7am-5pm	8am-12pm	closed	closed	3342	9.1
Grocery	7am-9pm	7am-9pm	9am-8pm	closed	4814	13.2
Healthcare Clinic	7am-7pm	9am-5pm	closed	closed	3428	9.4
High School	8am-4pm (20% in summer)	closed	closed	closed	1606	4.4
Hospital	24/7	24/7	24/7	24/7	8760	24.0
Motel	24/7	24/7	24/7	24/7	8760	24.0
Manufacturing Facility (Light Industry)	Mfg: 6am-10pm, Office: 8am-5pm	Mfg: 6am-10pm, Office: closed	closed	closed	4848	13.3
Multi-Family Mid-Rise	24/7; Reduced occupancy 7am - 5pm	24/7; Reduced occupancy 9am - 3pm	24/7; Reduced occupancy 9am - 3pm	24/7; Reduced occupancy 9am - 3pm	7038	19.3
Multi-Family High-Rise	24/7; Reduced occupancy 7am - 5pm	24/7; Reduced occupancy 9am - 3pm	24/7; Reduced occupancy 9am - 3pm	24/7; Reduced occupancy 9am - 3pm	7038	19.3
Movie Theater	10am-Midnight	10am-Midnight	10am-Midnight	10am-Midnight	5110	14.0
Office - Low-rise	8am-5pm	closed	closed	closed	2259	6.2
Office - Mid-rise	8am-5pm	20% 8am-noon	closed	closed	2301	6.3
Office - High-rise	8am-5pm	20% 8am-noon	closed	closed	2301	6.3
Religious Building	Office: 8am-5pm, other: closed	closed	8am-1pm	closed	260	0.7
Restaurant	7am-8pm	7am-8pm	7am-8pm	closed	4615	12.6
Retail - Department Store	9am-9pm	9am-9pm	10am-5pm	10am-5pm	4070	11.1
Retail - Strip Mall	9am-9pm	9am-9pm	10am-5pm	10am-5pm	4070	11.1
Warehouse (Conditioned Storage)	7am-7pm	7am-7pm (reduced occupancy)	closed	closed	3324	9.1

μ Heat = Efficiency of heating system

= Actual

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-ERVE-V02-160601

REVIEW DEADLINE: 1/1/2022

4.4.28 Stack Economizer for Boilers Serving HVAC Loads

MEASURE DESCRIPTION

Stack economizers are designed to recover heat from hot boiler flue gasses. Recovered heat is used to preheat boiler feed water. This measure describes the retrofit of HVAC boilers with stack economizers. HVAC boilers are defined as those used for space heating applications. There is another, similar measure for boilers that serve process loads.

This measure was developed to be applicable to the following program types: NC, TOS, RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify the economizer must be installed on a boiler exhaust stack. Heat captured by the economizer is to be used to pre-heat boiler feed water.

DEFINITION OF BASELINE EQUIPMENT

The baseline boiler does not have an economizer installed.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for the boiler stack economizer is 15 years.⁴⁷⁴

DEEMED MEASURE COST

The incremental and full measure cost for this measure is custom.

DEEMED O&M COST ADJUSTMENTS

The O&M cost for this measure is custom.

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS SAVINGS

$$\Delta \text{therms} = \text{SF} * \text{MBH}_{\text{In}} * \text{EFLH} / 100$$

⁴⁷⁴ PA Consulting, Focus on Energy Evaluation, Business Programs: Measure Life Study, August 25, 2009.

Where:

$$SF = (T_{\text{existing}} - T_{\text{eff}}) / 40^{\circ}\text{F} * TRE$$

= see default Savings Factor table below

Where:

$$T_{\text{existing}} = \text{Existing Full Fire Boiler Flue Gas Temperature as it exits the Stack}$$

$$= 425^{\circ}\text{F}^{475} \text{ (water, 81.9\% eff) or custom}$$

$$= 480^{\circ}\text{F}^3 \text{ (steam, 80.7\% eff) or custom}$$

$$T_{\text{eff}} = \text{Efficient Full Fire Boiler Flue Gas Temperature as it exits the Stack}$$

$$= 338^{\circ}\text{F} \text{ (conventional economizer – Water Boiler)}^{476} \text{ or custom}$$

$$= 365^{\circ}\text{F} \text{ (conventional economizer – Steam Boiler)}^{477} \text{ or custom}$$

$$= 280^{\circ}\text{F} \text{ (condensing economizer – Water Boiler)}^{478} \text{ or custom}$$

$$= 308^{\circ}\text{F} \text{ (condensing economizer – Steam Boiler)}^{479} \text{ or custom}$$

$$TRE = \% \text{ efficiency increase for } 40^{\circ}\text{F} \text{ of stack temperature reduction}$$

$$= 1\%^{480} \text{ or custom}$$

Based on defaults provided above:

Boiler Type	SF ⁴⁸¹	
	Conventional Economizer	Condensing Economizer
Hot Water Boiler	2.19% average SF or custom	3.63% average SF or custom
Steam Boiler	2.88% average SF or custom	4.31% average SF or custom

$$MBH_{\text{In}} = \text{Rated boiler input capacity, in MBH}$$

$$= \text{Actual}$$

⁴⁷⁵ Cleaver Brooks. March 2012, Boiler Efficiency Guide, Pg. 7, Figure 1.

⁴⁷⁶ The minimum stack temperature for a non-condensing economizer is 250°F from Department of Energy (DOE). January 2012, Steam Tip Sheet #26A, Consider Installing a Condensing Economizer. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. The average temperature drop is assumed to be ½ way between the existing and efficient temperature minimum, $(425^{\circ}\text{F} + 250^{\circ}\text{F}) / 2 = 338^{\circ}\text{F}$.

⁴⁷⁷ The minimum stack temperature for a non-condensing economizer is 250°F from Department of Energy (DOE). January 2012, Steam Tip Sheet #26A, Consider Installing a Condensing Economizer. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. The average temperature drop is assumed to be ½ way between the existing and efficient temperature minimum, $(480^{\circ}\text{F} + 250^{\circ}\text{F}) / 2 = 365^{\circ}\text{F}$.

⁴⁷⁸ The minimum stack temperature for a condensing economizer is 250°F from Department of Energy (DOE). January 2012, Steam Tip Sheet #26A, Consider Installing a Condensing Economizer. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. The average temperature drop is assumed to be ½ way between the existing and efficient temperature minimum, $(425^{\circ}\text{F} + 135^{\circ}\text{F}) / 2 = 280^{\circ}\text{F}$.

⁴⁷⁹ The minimum stack temperature for a condensing economizer is 250°F from Department of Energy (DOE). January 2012, Steam Tip Sheet #26A, Consider Installing a Condensing Economizer. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. The average temperature drop is assumed to be ½ way between the existing and efficient temperature minimum, $(480^{\circ}\text{F} + 135^{\circ}\text{F}) / 2 = 308^{\circ}\text{F}$.

⁴⁸⁰ United States EPA, Climate Wise: Wise Rules for Industrial Efficiency, July 1998. The Wise Rules indicate savings range of 1-2% per 40°F reduction, so utilizing 1% is a conservative approach.

⁴⁸¹ These average values should be utilized in absence of actual temperature data. An economizer with a zero temperature change between the existing and the efficient temperatures would not be installed, so these average values are conservative.

EFLH = Equivalent Full Load Hours for heating are provided in Section 4.4 HVAC End Use

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-BECO-V01-150601

REVIEW DEADLINE: 1/1/2021

4.4.29 Stack Economizer for Boilers Serving Process Loads

MEASURE DESCRIPTION

Stack economizers are designed to recover heat from hot boiler flue gasses. Recovered heat is used to preheat boiler feed water. This measure describes the retrofit of process boilers with stack economizers. Process boilers are defined as those used for industrial, manufacturing, or other non-HVAC applications. There is another, similar measure for boilers that serve HVAC loads.

This measure was developed to be applicable to the following program types: NC, TOS, RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify the economizer must be installed on a boiler exhaust stack. Heat captured by the economizer is to be used to pre-heat boiler feed water.

DEFINITION OF BASELINE EQUIPMENT

The baseline boiler does not have an economizer installed.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for the boiler stack economizer is 15 years.⁴⁸²

DEEMED MEASURE COST

The incremental and full measure cost for this measure is custom.

DEEMED O&M COST ADJUSTMENTS

The O&M cost for this measure is custom.

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

⁴⁸² PA Consulting, Focus on Energy Evaluation, Business Programs: Measure Life Study, August 25, 2009.

NATURAL GAS SAVINGS

$$\Delta \text{therms} = \text{SF} * \text{MBH_In} * 8766 * \text{UF} / 100$$

Where:

$$\text{SF} = (\text{T_existing} - \text{T_eff}) / 40^\circ\text{F} * \text{TRE}$$

= see default Savings Factor table below

$$\begin{aligned} \text{T_existing} &= \text{Existing Full Fire Boiler Flue Gas Temperature as it exits the Stack} \\ &= 425^\circ\text{F}^{483} \text{ (water, 81.9\% eff per IL TRM) or custom} \\ &= 480^\circ\text{F}^3 \text{ (steam, 80.7\% eff per IL TRM) or custom} \end{aligned}$$

$$\begin{aligned} \text{T_eff} &= \text{Efficient Full Fire Boiler Flue Gas Temperature as it exits the Stack} \\ &= 338^\circ\text{F} \text{ (conventional economizer – Water Boiler)}^{484} \text{ or custom} \\ &= 365^\circ\text{F} \text{ (conventional economizer – Steam Boiler)}^{485} \text{ or custom} \\ &= 280^\circ\text{F} \text{ (condensing economizer – Water Boiler)}^{486} \text{ or custom} \\ &= 308^\circ\text{F} \text{ (condensing economizer – Water Boiler)}^{487} \text{ or custom} \end{aligned}$$

$$\begin{aligned} \text{TRE} &= \% \text{ efficiency increase for } 40^\circ\text{F} \text{ of stack temperature reduction} \\ &= 1\%^{488} \text{ or custom} \end{aligned}$$

Based on defaults provided above:

Boiler Type	SF ⁴⁸⁹	
	Conventional Economizer	Condensing Economizer
Hot Water Boiler	2.19% average SF or custom	3.63% average SF or custom
Steam Boiler	2.88% average SF or custom	4.31% average SF or custom

$$\text{MBH_In} = \text{Rated boiler input capacity, in MBH}$$

⁴⁸³ Cleaver Brooks. March 2012, Boiler Efficiency Guide, Pg. 7, Figure 1.

⁴⁸⁴ The minimum stack temperature for a non-condensing economizer is 250°F from Department of Energy (DOE). January 2012, Steam Tip Sheet #26A, Consider Installing a Condensing Economizer. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. The average temperature drop is assumed to be ½ way between the existing and efficient temperature minimum, (425°F + 250°F) / 2 = 338°F.

⁴⁸⁵ The minimum stack temperature for a non-condensing economizer is 250°F from Department of Energy (DOE). January 2012, Steam Tip Sheet #26A, Consider Installing a Condensing Economizer. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. The average temperature drop is assumed to be ½ way between the existing and efficient temperature minimum, (480°F + 250°F) / 2 = 365°F.

⁴⁸⁶ The minimum stack temperature for a condensing economizer is 250°F from Department of Energy (DOE). January 2012, Steam Tip Sheet #26A, Consider Installing a Condensing Economizer. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. The average temperature drop is assumed to be ½ way between the existing and efficient temperature minimum, (425°F + 135°F) / 2 = 280°F.

⁴⁸⁷ The minimum stack temperature for a condensing economizer is 250°F from Department of Energy (DOE). January 2012, Steam Tip Sheet #26A, Consider Installing a Condensing Economizer. Advanced Manufacturing Office. Washington, DC: U.S. Department of Energy. The average temperature drop is assumed to be ½ way between the existing and efficient temperature minimum, (480°F + 135°F) / 2 = 308°F.

⁴⁸⁸ United States EPA, Climate Wise: Wise Rules for Industrial Efficiency, July 1998. The Wise Rules indicate savings range of 1-2% per 40°F reduction, so utilizing 1% is a conservative approach.

⁴⁸⁹ These average values should be utilized in absence of actual temperature data. An economizer with a zero temperature change between the existing and the efficient temperatures would not be installed, so these average values are conservative.

	= Actual
8766	= Hours a year
UF	= Utilization Factor
	= 41.9% ⁴⁹⁰ or custom

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-PECO-V01-150601

REVIEW DEADLINE: 1/1/2022

⁴⁹⁰ Work Paper – Tune up for Boilers serving Space Heating and Process Load by Resource Solutions Group, January 2012

4.4.30 Notched V Belts for HVAC Systems

MEASURE DESCRIPTION

This measure is for replacement of smooth v-belts in non-residential package and split HVAC systems with notched v-belts or for installing new equipment with synchronous belts instead of smooth v-belts. Typically there is a v-belt between the motor and the supply air fan and/or return air fan in larger package and split HVAC systems (RTU).

In general there are two styles of grooved v-belts, notched and synchronous. The DOE defines each as follows;

Notched V-Belts - A notched belt has grooves or notches that run perpendicular to the belt's length, which reduces the bending resistance of the belt. Notched belts can use the same pulleys as cross-section standard V-belts. They run cooler, last longer, and are about 2% more efficient than standard V-belts.

Synchronous Belts - Synchronous belts (also called cogged, timing, positive-drive, or high-torque drive belts) are toothed and require the installation of mating grooved sprockets. These belts operate with a consistent efficiency of 98% and maintain their efficiency over a wide load range.

Smooth v-belts are usually referred to in five basic groups:

- "L" belts are low end belts that are for small, fractional horsepower motors and these are not used in RTUs.
- "A" and "B" belts are the two types typically used in RTUs. The "A" belt is a ½ inch width by 5/16 inch thickness and the "B" belt is larger, 21/32 inch wide and 12/32 inch thick so it can carry more power. V-belts come in a wide variety of lengths where 20 to 100 inches is typical.
- "C" and "D" belts are primarily for industrial applications with high power transmission requirements.
- V-belts are provided by various vendors. The notched version of these belts typically have an "X" added to the designation. For this HVAC fans notched v-belt Replacement measure, only the "A" and "B" v-belts are considered. A typical "A" v-belt is replaced by a notched "AX" v-belt and a "B" is replaced by a "BX." In general, smooth v-belts have an efficiency of 90% to 98% while notched v-belts have an efficiency of 95% to 98%. Because notched v-belts are more flexible they work with smaller diameter pulleys and they have less resistance to bending. Lower bending resistance increases the power transmission efficiency, lowers the waste heat, and allows the belt to last longer than a smooth belt.

Three research papers⁴⁹¹ ⁴⁹² ⁴⁹³ show that the notched v-belt efficiency is 2% to 5% better than a typical smooth v-belt. A fourth paper by USDOE's Energy Efficiency and Renewable Energy⁴⁹⁴ group reviewed most of the earlier literature and recommended using a conservative 2% efficiency improvement for energy savings for calculations.

For this measure it is assumed that upgrading a standard smooth v-belt with a new notched v-belt will result in a fan energy reduction of 2%.

DEFINITION OF EFFICIENT EQUIPMENT

For the Notched V-Belt characterization to apply, the Efficient Equipment is HVAC RTUs that have notched v-belts installed on the supply and/or return air fans. This can be done as a retrofit, TOS, or NC project.

⁴⁹¹"Gates Corporation Announces New EPDM Molded Notch V-Belts," The Gates Rubber Co., June 2010 (Assumed 3% efficiency improvement) https://ww2.gates.com/news/index.cfm?id=11296&show=newsitem&location_id=753&view=Gates

⁴⁹² "Synchronous Belt Drives Offer Low Cost Energy Savings," Baldor., February 2009. (attached in Reference Documents)

⁴⁹³ "Energy Savings from Synchronous Belts," The Gates Rubber Co., February 2014. (Assumed 5% efficiency improvement) <http://www.gates.com/~media/Files/Gates/Industrial/Power%20Transmission/White%20Papers/Energy%20Savings%20from%20Synchronous%20Belt%20Drives.pdf>

⁴⁹⁴ "Motor System Tip Sheet #5, Replace V-Belts with Cogged or Synchronous Belt Drives," USDOE-EERE, September 2005. (Assumed 2% efficiency improvement) http://www1.eere.energy.gov/industry/bestpractices/pdfs/replace_vbelts_motor_systemts5.pdf

For the Synchronous Belt characterization to apply, the Efficient Equipment is HVAC RTUs that have synchronous belts installed on the supply and/or return air fans. This can be done as a TOS or NC project. Retrofit projects can also claim savings, but costs should be verified independently (typically the cost of installing synchronous belts as a retrofit is not economically viable).

DEFINITION OF BASELINE EQUIPMENT

The Baseline Equipment is HVAC RTUs that have smooth v-belts installed on the supply and/or return air fans (i.e. RTU does not already have a notched v-belt installed).

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

A v-belt has a life based on fan run hours which varies by building type based primarily on occupancy schedule because the fans are required by code to operate continuously during occupied hours. The supply and return fans will also run a few hours during unoccupied hours for heating and cooling as needed. For the notched v-belt EUL calculation, the default hours⁴⁹⁵ in the following table are used for a variety of building types and HVAC applications.

$$\text{EUL} = \text{Belt Life} / \text{Occupancy Hours per year}$$

Where:

$$\text{Belt Life} = 24,000 \text{ hours}^{496}$$

$$\text{Occupancy Hours per year} = \text{values from Table below}$$

The notched v-belt measure EUL is summarized by building type in the following table.

Notched v-belt Effective Useful Life (EUL)

Building Type	Total Fan Run Hours	EUL (Years)
Assembly	7235	3.3
Assisted Living	8760	2.7
College	6103	3.9
Convenience Store	7004	3.4
Elementary School	7522	3.2
Garage	7357	3.3
Grocery	7403	3.2
Healthcare Clinic	6345	3.8
High School	7879	3.0
Hospital - VAV econ	8760	2.7
Hospital - CAV econ	8760	2.7
Hospital - CAV no econ	8760	2.7
Hospital - FCU	8760	2.7
Manufacturing Facility	8706	2.8
MF - High Rise	8760	2.7
MF - Mid Rise	8760	2.7
Hotel/Motel - Guest	8760	2.7
Hotel/Motel - Common	8760	2.7
Movie Theater	7505	3.2
Office - High Rise - VAV econ	6064	4.0

⁴⁹⁵ ComEd Trm June 1, 2010 page 139. The Office hours is based upon occupancy from the eQuest model developed for EFLH, since it was agreed the ComEd value was too low.

⁴⁹⁶ "DEER2014-EUL-table-update_2014-02-05.xlsx," Database for Energy Efficiency Resources (DEER), Deer 2014. www.deerresources.com (attached in Reference Documents)

Building Type	Total Fan Run Hours	EUL (Years)
Office - High Rise - CAV econ	5697	4.2
Office - High Rise - CAV no econ	5682	4.2
Office - High Rise - FCU	6163	3.9
Office - Low Rise	6288	3.8
Office - Mid Rise	6125	3.9
Religious Building	7380	3.3
Restaurant	7809	3.1
Retail - Department Store	6890	3.5
Retail - Strip Mall	6846	3.5
Warehouse	6786	3.5
Unknown	7100	3.4

The lifetime of a synchronous belt system is the same as the lifetime of the equipment it is installed on because it is a permanent upgrade, involving the installation of toothed pulleys. Typical HVAC RTU lifetime is 15 years, which applies to synchronous belts as well. This is not to suggest that the actual belt component has an equivalent lifetime because they do require replacement. However, their O&M cost savings (derived from not having to tension, etc.) are assumed to offset the replacement cost of the belt, resulting in a net cost of zero. As a result, neither a separate lifetime nor O&M savings are quantified for synchronous belts and lifetime can therefore be considered as the lifetime of the equipment they're installed on because it would not be possible to install a traditional or notched belt on the synchronous pulleys.

DEEMED MEASURE COST

A review of the Grainger online⁴⁹⁷ pricing for "A," "B," "AX," and "BX" v-belts showed the incremental cost to upgrade to notched v-belts would result in a 28% price increase. The notched v-belt incremental cost is summarized in the table below:

Notched V-belt Incremental Cost Summary

Smooth V-Belt Industry Number	Outside Length (Inches)	Dayton Smooth V-Belt*	Notched V-belt Industry Number	Dayton Notched v-belt*	Price Increase	% Increase
A30 (Item # 1A095)	32	\$12.70	AX29 (Item # 3GWU4)	\$17.65	\$4.95	28%
B29 (Item # 6L208)	32	\$16.75	BX29 (Item # 5TXL4)	\$23.23	\$6.48	28%
* Pricing based on Dayton Belts as found on Grainger Website 10/30/14						

Note that the incremental cost for notched V-Belts assumes that the notched belt is purchased and installed instead of a smooth v-belt. There is no difference in the cost of installation, only the material.

Synchronous Belt Incremental Cost Summary

Smooth V-Belt Industry Number	Smooth belt system Price*	Synchronous Belt Industry Number	Synchronous System Price*	Price Difference
Belt A30 (Item # 1A095)	\$12.70	Belt 1DHL5 (Item # 322L050)	\$20.51	\$7.81
Gearbelt pulley BK47 (Item #5UHD5)	\$45.90	Gearbelt sprocket	\$113.00	\$67.10

⁴⁹⁷ Grainger catalog on-line web-site for Dayton v-belt pricing
<http://www.grainger.com/Grainger/ecatalog/N-1z0r596/Ntt-v-belts>

Smooth V-Belt Industry Number	Smooth belt system Price*	Synchronous Belt Industry Number	Synchronous System Price*	Price Difference
		GTR-36G-8M-12 (Item # 2UWH6)		
* Costs based on Grainger pricing.				

Incremental cost for a NC or TOS project is \$142. This is the price of synchronous equipment (belt, two sprockets) subtract v-belt equipment (belt, two pulleys). Labor cost is assumed to be equal in the baseline and efficient cases.

Incremental cost for a RF project is \$383.81. This is the price of synchronous equipment and labor⁴⁹⁸ to install it (not including a trip charge) subtract the cost of the v-belt (but not the pulleys).

DEEMED O&M COST ADJUSTMENTS

N/A

LOADSHAPE

Loadshape C05 - Commercial Electric Heating and Cooling

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = \text{kW}_{\text{connected}} * \text{Hours} * \text{ESF}$$

Where:

$\text{kW}_{\text{Connected}}$ = kW of equipment is calculated using motor efficiency⁴⁹⁹.

= (HP * 0.746 kW/HP * Load Factor) / Motor Efficiency

Load Factor = Motors are assumed to have a load factor of 80% for calculating KW if actual values cannot be determined⁵⁰⁰. Custom load factor may be applied if known.

Motor Efficiency = Actual motor efficiency shall be used to calculate KW. If not known a value from the motor efficiency reference tables below should be used⁵⁰¹. Default motor is a NEMA Premium Efficiency, ODP, 4-pole/1800 RPM fan motor

⁴⁹⁸ Assumed to be \$150 based on mechanical contractor estimate.

⁴⁹⁹ Note that kW_{Connected} may be determined using various methodologies. The examples provided use rated HP and assumed load factor. Other methodologies include rated voltage and full load current with assumed load factor, or actual measured voltage and current.

⁵⁰⁰ Com Ed TRM June 1, 2010

⁵⁰¹ Efficiency values for motors less than one HP taken from Baldor Electric Catalog 501:

http://www.baldor.com/pdf/501_Catalog/CA501.pdf

Baseline Motor Efficiencies (EPACT)						
Size HP	Open Drip Proof (ODP)			Totally Enclosed Fan-Cooled (TEFC)		
	# of Poles					
	6	4	2	6	4	2
	Speed (RPM)					
	1200	1800	3600	1200	1800	3600
1/8	-	44.00%	-	-	-	-
1/6	57.50%	62.00%	-	-	-	-
1/4	68.00%	68.00%	-	68.00%	64.00%	-
1/3	70.00%	70.00%	72.00%	70.00%	68.00%	72.00%
1/2	78.50%	80.00%	68.00%	72.00%	74.00%	68.00%
3/4	77.00%	78.50%	74.00%	77.00%	75.50%	74.00%
1	80.00%	82.50%	75.50%	80.00%	82.50%	75.50%
1.5	84.00%	84.00%	82.50%	85.50%	84.00%	82.50%
2	85.50%	84.00%	84.00%	86.50%	84.00%	84.00%
3	86.50%	86.50%	84.00%	87.50%	87.50%	85.50%
5	87.50%	87.50%	85.50%	87.50%	87.50%	87.50%
7.5	88.50%	88.50%	87.50%	89.50%	89.50%	88.50%
10	90.20%	89.50%	88.50%	89.50%	89.50%	89.50%
15	90.20%	91.00%	89.50%	90.20%	91.00%	90.20%
20	91.00%	91.00%	90.20%	90.20%	91.00%	90.20%
25	91.70%	91.70%	91.00%	91.70%	92.40%	91.00%

Efficient Motor Efficiencies (NEMA Premium)						
Size HP	Open Drip Proof (ODP)			Totally Enclosed Fan-Cooled (TEFC)		
	# of Poles			# of Poles		
	2	4	6	2	4	6
	Speed (RPM)			Speed (RPM)		
	1200	1800 (Default)	3600	1200	1800	3600
0.125 *	-	44.00%	-	-	-	-
1/6	57.50%	62.00%	-	-	-	-
1/4	68.00%	68.00%	-	68.00%	64.00%	-
1/3	70.00%	70.00%	72.00%	70.00%	68.00%	72.00%
1/2	78.50%	80.00%	68.00%	72.00%	74.00%	68.00%
3/4	77.00%	78.50%	74.00%	77.00%	75.50%	74.00%
1	82.50%	85.50%	77.00%	82.50%	85.50%	77.00%
1.5	86.50%	86.50%	84.00%	87.50%	86.50%	84.00%
2	87.50%	86.50%	85.50%	88.50%	86.50%	85.50%
3	88.50%	89.50%	85.50%	89.50%	89.50%	86.50%
5	89.50%	89.50%	86.50%	89.50%	89.50%	88.50%
7.5	90.20%	91.00%	88.50%	91.00%	91.70%	89.50%
10	91.70%	91.70%	89.50%	91.00%	91.70%	90.20%
15	91.70%	93.00%	90.20%	91.70%	92.40%	91.00%
20	92.40%	93.00%	91.00%	91.70%	93.00%	91.00%
25	93.00%	93.60%	91.70%	93.00%	93.60%	91.70%

Hours = When available, actual hours should be used. If actual hours are not available default hours⁵⁰² are provided in table below for HVAC fan operation which varies by building type:

Building Type	Total Fan Run Hours
Assembly	7235
Assisted Living	8760
College	6103
Convenience Store	7004
Elementary School	7522
Garage	7357
Grocery	7403
Healthcare Clinic	6345
High School	7879
Hospital - VAV econ	8760
Hospital - CAV econ	8760
Hospital - CAV no econ	8760
Hospital - FCU	8760
Manufacturing Facility	8706
MF - High Rise	8760
MF - Mid Rise	8760
Hotel/Motel - Guest	8760
Hotel/Motel - Common	8760
Movie Theater	7505
Office - High Rise - VAV econ	6064
Office - High Rise - CAV econ	5697
Office - High Rise - CAV no econ	5682
Office - High Rise - FCU	6163
Office - Low Rise	6288
Office - Mid Rise	6125
Religious Building	7380
Restaurant	7809
Retail - Department Store	6890
Retail - Strip Mall	6846
Warehouse	6786
Unknown	7100

ESF = Energy Savings Factor, the ESF for notched v-belt Installation is assumed to be 2%
= the ESF for notched Synchronous Belt Installation is assumed to be 3.1%⁵⁰³

⁵⁰² Hours per year are estimated using the eQuest models as the total number of hours the fans are operating for heating, cooling and ventilation for each building type.

⁵⁰³ Based on information found in Advanced Manufacturing Office, US DOE, "Replace V-Belts with Notched or Synchronous Drives", (US Department of Energy Motor Systems Tip Sheet #5, DOE/GO-102012-3740, November 2012). V-belt drives can have a peak efficiency of 95% and synchronous belts operate at 98%, therefore ESF is $(1-95\%/98\%) = 3.1\%$.

EXAMPLE

For example, a notched v-belt installation in an low rise office building RTU with a 5 HP NEMA premium efficiency motor using the default hours of operation, motor load and 89.5% motor efficiency;

$$\begin{aligned}
 \Delta \text{kWh} &= \text{kW}_{\text{connected}} * \text{Hours} * \text{ESF} \\
 &= ((\text{HP} * 0.746 \text{ kW/HP} * \text{Load Factor}) / \text{Motor Efficiency}) * \text{Hours} * \text{ESF} \\
 &= ((5 \text{ HP} * 0.746 \text{ kW/HP} * 80\%) / 89.5\%) * 6288 * 2\% \\
 &= 419 \text{ kWh Savings}
 \end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \text{kW}_{\text{connected}} * \text{ESF}$$

Where:

$$\begin{aligned}
 \text{kW}_{\text{Connected}} &= \text{kW of equipment is calculated using motor efficiency.} \\
 &= (\text{HP} * 0.746 \text{ kW/HP} * \text{Load Factor}) / \text{Motor Efficiency} \\
 &\text{Variables as provided above}
 \end{aligned}$$

EXAMPLE

For example, an office building RTU with a 5 HP NEMA premium efficiency motor using the default motor load and 89.5% motor efficiency;

$$\begin{aligned}
 \Delta \text{kW} &= \text{kW}_{\text{connected}} * \text{ESF} \\
 &= ((\text{HP} * 0.746 \text{ kW/HP} * \text{Load Factor}) / \text{Motor Efficiency}) * \text{ESF} \\
 &= ((5 \text{ HP} * 0.746 \text{ kW/HP} * 80\%) / 89.5\%) * 2\% \\
 &= 0.0667 \text{ kW Savings}
 \end{aligned}$$

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-NVBE-V03-180101

REVIEW DEADLINE: 1/1/2022

4.4.31 Small Business Furnace Tune-Up

DESCRIPTION

This measure is for a natural gas Small Business furnace that provides space heating. The tune-up will improve furnace performance by inspecting, cleaning and adjusting the furnace and appurtenances for correct and efficient operation. Additional savings maybe realized through a complete system tune-up.

This measure was developed to be applicable to the following program types: Small business.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure an approved technician must complete the tune-up requirements⁵⁰⁴ listed below:

- Measure combustion efficiency using an electronic flue gas analyzer
- Check and clean blower assembly and components per manufacturer's recommendations
- Where applicable Lubricate motor and inspect and replace fan belt if required
- Inspect for gas leaks
- Clean burner per manufacturer's recommendations and adjust as needed
- Check ignition system and safety systems and clean and adjust as needed
- Check and clean heat exchanger per manufacturer's recommendations
- Inspect exhaust/flue for proper attachment and operation
- Inspect control box, wiring and controls for proper connections and performance
- Check air filter and clean or replace per manufacturer's
- Inspect duct work connected to furnace for leaks or blockages
- Measure temperature rise and adjust flow as needed
- Check for correct line and load volts/amps
- Check thermostat operation is per manufacturer's recommendations (if adjustments made, refer to 'Small Commercial Programmable Thermostat Adjustment' measure for savings estimate)
- Perform Carbon Monoxide test and adjust heating system until results are within standard industry acceptable limits

DEFINITION OF BASELINE EQUIPMENT

The baseline is furnace assumed not to have had a tune-up in the past 2 years.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for the tune up is 2 years.⁵⁰⁵

DEEMED MEASURE COST

The incremental cost for this measure should be the actual cost of tune up.

DEEMED O&M COST ADJUSTMENTS

There are no expected O&M savings associated with this measure.

LOADSHAPE

Loadshape C04 - Commercial Electric Heating

⁵⁰⁴ American Standard Maintenance for Indoor Units: <http://www.americanstandardair.com/owner-support/maintenance.html>

⁵⁰⁵ Act on Energy Commercial Technical Reference Manual No. 2010-4, 9.2.3 Gas Forced-Air Furnace Tune-up.

COINCIDENCE FACTOR

N/A

Algorithms

CALCULATION OF ENERGY SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = \Delta \text{Therms} * F_e * 29.3$$

Where:

$$\Delta \text{Therms} = \text{as calculated below}$$

$$F_e = \text{Furnace Fan energy consumption as a percentage of annual fuel consumption}$$

$$= 3.14\%^{506}$$

$$29.3 = \text{kWh per therm}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS SAVINGS

$$\Delta \text{Therms} = (\text{Capacity} * \text{EFLH} * (((\text{Effbefore} + \text{Ei}) / \text{Effbefore}) - 1)) / 100,000$$

Where:

$$\text{Capacity} = \text{Furnace gas input size (Btu/hr)}$$

$$= \text{Actual}$$

$$\text{EFLH} = \text{Equivalent Full Load Hours for heating are provided}$$

$$\text{in section 4.4 HVAC End Use}$$

$$\text{Effbefore} = \text{Efficiency of the furnace before the tune-up}$$

$$= \text{Actual}$$

Note: Contractors should select a mid-level firing rate that appropriately represents the average building operating condition over the course of the heating season and take readings at a consistent firing rate for pre and post tune-up.

$$\text{Ei} = \text{Efficiency Improvement of the furnace tune-up measure}$$

$$= \text{Actual}$$

$$100,000 = \text{Converts Btu to therms}$$

⁵⁰⁶ F_e is not one of the AHRI certified ratings provided for residential furnaces, but can be reasonably estimated from a calculation based on the certified values for fuel energy (E_f in MMBtu/yr) and E_{ae} (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the Energy Star version 3 criteria for 2% F_e . See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.

EXAMPLE

A 200 kBtu furnace in a Rockford low rise office records an efficiency prior to tune up of 82% AFUE and a 1.8% improvement in efficiency are tune up:

$$\begin{aligned}\Delta \text{therms} &= (200,000 * 1428 * (((0.82 + 0.018) / 0.82) - 1)) / 100,000 \\ &= 62.3 \text{ therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-FTUN-V02-160601

REVIEW DEADLINE: 1/1/2022

4.4.32 Combined Heat and Power

DESCRIPTION

The Combined Heat and Power (CHP) measure can provide energy savings within the State of Illinois through the development and operation of CHP projects. This measure is applicable for Conventional or Topping Cycle CHP systems, as well as Waste Heat-to-Power (WHP) or Bottoming Cycle CHP systems. The measure will reduce the total Btu's of energy required to meet the end use needs of the facility.

It is recognized that CHP system design and configuration may be complex, and as such the calculation of energy savings may not be reducible to the equations within this measure. In such cases a more comprehensive engineering and financial analysis may be developed that more accurately incorporates the attributes of complex CHP configurations such as variable-capacity systems, and partial combined-cycle CHP systems. Where noted, the use of values that are determined through an external engineering analysis may be substituted by agreement between the participant, the program administrator and independent evaluator. This substitution of values does not eliminate ex post evaluation risk (retroactive adjustments to savings claims) that exists when using custom inputs.

This measure was developed to be applicable to the following program types: Retrofit (RF), New Construction (NC). If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

Conventional or Topping Cycle CHP is defined as an integrated system that is located at or near the building or facility (on-site, on the customer side of the meter) that utilizes a prime mover (reciprocating engine, gas turbine, micro-turbine, fuel cell, boiler/steam turbine combination) for the purpose of generating electricity and useful thermal energy (such as steam, hot water, or chilled water) where the primary function of the facility where the CHP is located is not to generate electricity for use on the grid. An eligible system must demonstrate a minimum total system efficiency of 60% (HHV)⁵⁰⁷ with at least 20% of the system's total useful energy output in the form of useful thermal energy on an annual basis.

Measuring and Calculating Conventional CHP Total System Efficiency:

CHP efficiency is calculated using the following equation:

$$CHP_{Efficiency(HHV)} = \frac{\left[CHP_{thermal} \left(\frac{kBtu}{yr} \right) + E_{CHP} \left(\frac{kWh}{yr} \right) * 3.412 \left(\frac{kBtu}{kWh} \right) \right]}{F_{totalCHP} \left(\frac{kBtu}{yr} \right)}$$

Where:

- CHP_{thermal} = Useful annual thermal energy output from the CHP system, defined as the annual thermal energy output of the CHP system that is actually recovered and utilized in the facility/process.
- E_{CHP} = Useful annual electricity output produced by the CHP system, defined as the annual electric energy output of the CHP system that is actually utilized to replace purchased electricity required to meet the requirements of the facility/process.
- F_{totalCHP} = Total annual fuel consumed by the CHP system

For further definition of the terms, please see "Calculation of Energy Savings" Section below.

⁵⁰⁷ Higher Heating Value (HHV): refers to the heating value of the fuel and is defined as the total thermal energy available, including the heat of condensation of water vapors, resulting from complete combustion of the fuel versus the Lower Heating Value (LHV) which assumes the heat of condensation is not available

Waste Heat-to-Power or Bottoming Cycle CHP is defined as an integrated system that is located at or near the building or facility (on-site, on the customer side of the meter) that does one of the following:

- Utilizes exhaust heat from an industrial/commercial process to generate electricity (except for exhaust heat from a facility whose primary purpose is the generation of electricity for use on the grid); or
- Utilizes the pressure drop in an industrial/commercial facility to generate electricity through a backpressure steam turbine where the facility normally uses a pressure reducing valve (PRV) to reduce the pressure in their facility; or
- Utilizes the pressure reduction in natural gas pipelines (located at natural gas compressor stations) before the gas is distributed through the pipeline to generate electricity, provided that the conversion of energy to electricity is achieved without using additional fossil fuels.

Since these types of systems utilize waste heat as their fuel, they do not have to meet any specific total system efficiency level (assuming they use no additional fossil fuel in their operation) If additional fuel is used onsite, it should be accounted for using the following methodology:

- Treat the portion of Waste-Heat-to-Power that does not require any additional fuel using the Waste-Heat-to-Power methodology outlined in this document.
- Treat the portion of Waste-Heat-to-Power that requires additional fuel (if natural gas) using the Conventional CHP methodology outlined in this document. If the additional fuel is not natural gas, custom carbon equivalency calculations would be needed – refer to section “Calculation of Energy Savings” for more details.
- Add the energy savings together.

These systems may export power to the grid.

DEFINITION OF BASELINE EQUIPMENT

Electric Baseline: The baseline facility would be a facility that purchases its electric power from the grid.

Heating Baseline (for CHP applications that displace onsite heat): The baseline equipment would be the boiler/furnace operating onsite, or a boiler/furnace meeting the baseline equipment defined in the High Efficiency Boiler (Section 4.4.10)/Furnace (Section 4.4.11) measures of this TRM.

Cooling Baseline (for CHP applications that displace onsite cooling demands): The baseline equipment would be the chiller (or chillers) operating onsite, or a chiller (or chillers) meeting the definition of baseline equipment defined in the Electric Chiller (Section 4.4.6) measure of this TRM.

Facilities that use biogas or waste gas: Facilities that use (but are not purchasing) biogas or waste gas that is not otherwise used, whether they are using biogas or waste gas only or a combination of biogas or waste gas and natural gas to meet their energy demands are also eligible for this measure. If additional fuel is purchased to power the CHP system, then the additional natural gas should be taken into account using the following methodology:

- Treat the portion of CHP system that does not require any additional fuel, or that requires additional fuel that would otherwise be wasted (e.g. flared), using the Waste-Heat-to-Power methodology outlined in this document.
- Treat the portion of CHP that requires additional fuel (if natural gas) using the Conventional CHP methodology outlined in this document. If the additional fuel is not natural gas, custom carbon equivalency calculations would be needed – refer to section “Calculation of Energy Savings” for more details.
- Add the energy savings together.

Consumption of any biogas or waste gas that would not otherwise being wasted (e.g., flared) will be accounted for in the overall net BTU savings calculations the same as for purchased natural gas.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

Measure life is a custom assumption, dependent on the technology selected and the system installation.

DEEMED MEASURE COST

Custom installation and equipment cost will be used. These costs should include the cost of the equipment and the cost of installing the equipment. Equipment costs include, but are not limited to: prime mover, heat recovery system(s), exhaust gas treatment system(s), controls, and any interconnection/electrical connection costs.

The installations costs include labor and material costs such as, but not limited to: labor costs, materials such as ductwork, piping, and wiring, project and construction management, engineering costs, commissioning costs, and other fees.

Measure costs will also include the present value of expected maintenance costs over the life of the CHP system.

LOADSHAPE

Use Custom Loadshape. The loadshape should be obtained from the actual CHP operation strategy, based on the On-Peak and Off-Peak Energy definitions specified in Table 3.3 of “Section 3.5 Electrical Loadshapes” of the TRM.

COINCIDENCE FACTOR

Custom coincidence factor will be used. Actual value based on the CHP operation strategy will be used.

Algorithm

CALCULATION OF ENERGY SAVINGS**i) Conventional or Topping Cycle CHP Systems:****Step 1: (Calculating Total Annual Source Fuel Savings in Btus)**

The first step is to calculate the total annual source fuel savings associated with the CHP installation, in order to ensure the CHP project produces positive total annual source fuel savings (i.e. reduction in source Btus):

$$\begin{aligned}
 S_{\text{FuelCHP}} &= \text{Annual fuel savings (Btu) associated with the use of a Conventional CHP system to generate the useful electricity output (kWh, converted to Btu) and useful thermal energy output (Btu) versus the use of the equivalent electricity generated and delivered by the local grid and the equivalent thermal energy provided by the onsite boiler/furnace.} \\
 &= (F_{\text{grid}} + F_{\text{thermalCHP}}) - F_{\text{total CHP}}
 \end{aligned}$$

Where:

$$\begin{aligned}
 F_{\text{grid}} &= \text{Annual fuel in Btu that would have been used to generate the useful electricity output of the CHP system if that useful electricity output was provided by the local utility grid.} \\
 &= E_{\text{CHP}} * H_{\text{grid}}
 \end{aligned}$$

Where:

$$E_{\text{CHP}} = \text{Useful annual electricity output produced by the CHP system, defined as the annual electric energy output of the CHP system that is actually utilized to replace purchased electricity required to meet the requirements of the facility/process.}^{508}$$

⁵⁰⁸ For complex systems this value may be obtained from a CHP System design/financial analysis study.

$$= (\text{CHP}_{\text{capacity}} * \text{Hours}) - E_{\text{Parasitic}}$$

$\text{CHP}_{\text{capacity}}$ = CHP nameplate capacity

= Custom input

Hours = Annual operating hours of the system

= Custom input

$E_{\text{parasitic}}$ = The electricity required to operate the CHP system that would otherwise not be required by the facility/process

= Custom input

H_{grid} = Heat rate of the grid in Btu/kWh, based on the average fossil heat rate for the EPA eGRID subregion, adjusted to take into account T&D losses.

For systems operating less than 6,500 hrs per year:

Use the Non-baseload heat rate provided by EPA eGRID for RFC West region for ComEd territory (including independent providers connected to RFC West), and SERC Midwest region for Ameren territory (including independent providers connected to SERC Midwest)⁵⁰⁹. Also include any line losses.

For systems operating more than 6,500 hrs per year:

Use the All Fossil Average heat rate provided by EPA eGRID for RFC West region for ComEd territory (including independent providers connected to RFC West), and SERC Midwest region for Ameren territory (including independent providers connected to SERC Midwest). Also include any line losses.

$F_{\text{thermalCHP}}$ = Annual fuel in Btu that would have been used on-site by a boiler/furnace to provide the useful thermal energy output of the CHP system.⁵¹⁰

= $\text{CHP}_{\text{thermal}} / \text{Boiler}_{\text{eff}}$ (or $\text{CHP}_{\text{thermal}} / \text{Furnace}_{\text{eff}}$)

$\text{CHP}_{\text{thermal}}$ = Useful annual thermal energy output from the CHP system, defined as the annual thermal energy output of the CHP system that is actually recovered and utilized in the facility/process.

= Custom input

$\text{Boiler}_{\text{eff}} / \text{Furnace}_{\text{eff}}$ = Efficiency of the on-site Boiler/Furnace that is displaced by the CHP system or if unknown, the baseline equipment value stated in the High Efficiency Boiler (Section 4.4.10) measure or High Efficiency Furnace (Section 4.4.11) measure in this TRM. .

= Custom input

$F_{\text{total CHP}}$ = Total fuel in Btus consumed by the CHP system

⁵⁰⁹ These values are subject to regular updates so should be reviewed regularly to ensure the current assumptions are correct. Refer to the latest EPA eGRID data. Current values, based on eGrid 2014 are:

- Non-Baseload RFC West: 9,346 Btu/kWh * (1 + Line Losses)
- Non-Baseload SERC Midwest: 9,157 Btu/kWh * (1 + Line Losses)
- All Fossil Average RFC West: 9,931 Btu/kWh * (1 + Line Losses)
- All Fossil Average SERC Midwest: 10,209 Btu/kWh * (1 + Line Losses)

⁵¹⁰ For complex systems this value may be obtained from a CHP System design/financial analysis study.

= Custom input

Step 2: (Savings Allocation to Program Administrators for Purposes of Assessing Compliance with Energy Savings Goals (Not for Use in Load Reduction Forecasting))

Savings claims are a function of the electric output of the CHP system (E_{CHP}), the used thermal output of the CHP system ($F_{thermalCHP}$), and the CHP system efficiency ($CHP_{Eff(HHV)}$). The percentages of electric output and used thermal output that can be claimed also differ slightly depending on whether the project was included in both electric⁵¹¹ and gas⁵¹² Energy Efficiency Portfolio Standard (EEPS)⁵¹³ efficiency programs, only an electric EEPS program or only a gas EEPS program. The tables below provide the specific percentages of electric and/or thermal output that can be claimed under each of those three scenarios. These percentages apply only to cases in which natural gas is the fuel used by the CHP system. Saving estimates for systems using other fuels should be calculated on a custom basis. If the waste heat recovered from the CHP system is offsetting electric equipment, such as an absorption chiller offsetting an electric chiller, then the net change in electricity consumption associated with the electric equipment should be added to the allocated electric savings.

1) For systems participating in both electric EEPS and gas EEPS programs:

CHP Annual System Efficiency (HHV)	Allocated Electric Savings	Allocated Gas Savings
60%	65% of E_{CHP} (kWh)	No gas savings
>60% to 65%	65% of E_{CHP} (kWh) + one percentage point increase for every one percentage point increase in CHP system efficiency (max 70% of E_{CHP} in kWh)	No gas Savings
>65%	70% of E_{chp} (kWh)	2.5% of $F_{thermal}$ (useful thermal output of the CHP system) for every one percentage point increase in CHP system efficiency above 65%.

Example: System with measured annual system efficiency (HHV) of 70%: Electric savings (kWh) = 70% of E_{CHP} measured over 12 months, and Gas savings (therms) = 12.5% of $F_{thermal}$ measured over 12 months (70% - 65% = 5 X 2.5% = 12.5%)

2) For systems participating in only an electric EEPS program:

CHP Annual System Efficiency (HHV)	Allocated Electric Savings	Allocated Gas Savings
60%	65% of E_{CHP} (useful electric output of CHP system in kWh)	No gas Savings
Greater than 60%	65% + one percentage point increase for every one percentage point increase in CHP system efficiency (no max)	No gas Savings

Example: System with measured annual fuel use efficiency of 75%: Electric savings (kWh) = 65% + 15% = 80% of E_{CHP} measured over 12 months (15% = 1% for every 1% increase in system efficiency). No gas savings (therms).

⁵¹¹ 220 ILCS 5/8-103; 220 ILCS 5/16-111.5B

⁵¹² 220 ILCS 5/8-104

⁵¹³ As used in this measure characterization, EEPS programs are defined as those energy efficiency programs implemented pursuant to Sections 8-103, 8-104, and 16-111.5B of the Illinois Public Utilities Act. Technically, EEPS programs pertain to energy efficiency programs implemented pursuant to 220 ILCS 5/8-103 and 220 ILCS 5/8-104. However, for simplicity in presentation, this measure defines EEPS programs as also including those programs implemented pursuant to 220 ILCS 5/16-111.5B (these programs are funded through the same energy efficiency riders established pursuant to Section 8-103).

3) For systems participating in only a gas EEPS program:

CHP Annual System Efficiency (HHV)	Allocated Electric Savings	Allocated Gas Savings
60% or greater	No electric savings	2.5% of F_{thermal} (useful thermal output of the CHP system) for every one percentage point increase in CHP system efficiency above 60%.

Example: System with measured annual system efficiency (HHV) of 70%: No Electric savings (kWh). Gas savings (therms) = 25% of F_{thermal} measured over 12 months (70% - 60% = 10 X 2.5% = 25%)

Conventional or topping cycle CHP systems virtually always require an increase in the use of fuel on-site in order to produce electricity. Different jurisdictions and experts across the country have employed and/or put forward a variety of approaches⁵¹⁴ to address how increased on-site fuel consumption should be reflected in the attribution of electric savings to CHP systems. The approach reflected in the tables above is generally consistent – for CHP systems consuming natural gas – with approaches recently put forward by the Southwest Energy Efficiency Project (SWEET) and Institute for Industrial Productivity (IIP) that determine reduced electric savings based on the equivalent amount of carbon dioxide generated from the increased fuel used⁵¹⁵.

There are a variety of ways one could treat the potential for gas utilities to claim savings from CHP projects in their EEPS portfolios. For projects in which a natural gas EEPS program is involved, the tables above treat savings from CHP installations in two steps: (1) a fuel-switch from electricity to natural gas (i.e. using more natural gas to eliminate the need to generate as much electricity on the grid); and (2) possible increases in CHP efficiency above a “benchmark” level. When both electric EEPS and natural gas EEPS programs are involved in a project, the program administrator claims all the electricity savings associated with a fuel-switch up to a “benchmark” 65% efficient CHP system. All the savings associated with increasing CHP efficiencies above that benchmark level are allocated to natural gas (e.g. if the CHP efficiency is 75%, the natural gas savings associated with an increase in CHP efficiency from 65% to 75% are allocated to natural gas). That is consistent with the notion that CHP efficiency typically increases primarily by increasing the use of the thermal output of the system (increasing the displacement of baseline gas use). For projects that involve only a natural gas EEPS program, the “benchmark” above which the gas utility can claim savings is lowered to 60%.

ii) **Waste-Heat-to-Power CHP Systems :****ELECTRIC ENERGY SAVINGS:**

$$\Delta \text{kWh} = E_{\text{CHP}}$$

⁵¹⁴ Approaches range from ignoring the increased gas use entirely (i.e., no “penalty”) to applying approximately 40-60% “penalties”, depending on the CHP efficiency and based on the equivalent grid kWh that the increased gas use represents.

⁵¹⁵ Consider, for example, a hypothetical CHP system that produces 5 million kWh annually, consumes 50 million kBtu of gas annual to generate that electricity (i.e. electric efficiency of approximately 34.8% HHV), reduces on-site gas use for space heating by 26 million kBtu of gas (i.e. equivalent to approximately 81.5% CHP thermal output utilization displacing gas used in a 70% efficient space heating boiler) and has a total annual CHP efficiency of 70.6% HHV. In this example, the net increase in on-site gas use is 24 million kBtu. At a carbon dioxide emission rate of 53.06 kg/MMBtu for burning natural gas, that translates to an increase in on-site carbon dioxide emissions of 1404 tons per year. At an estimated marginal emission rate of 1.098 tons of carbon dioxide per MWh in Illinois, that is equivalent to electric grid production of approximately 1.28 million kWh, or penalty of about 25.6% of the CHP system’s electrical output if a precise calculation of carbon equivalency was utilized to assign savings. In comparison, the simplified table above would entitle an electric utility to claim savings equal to 75.6% of the electric output (i.e. a penalty of 24.4% of electrical output) if it was the only utility promoting the system. In a gas and electric example, the electric savings claimed would be 70% of the production (a penalty of 30% of the CHP system’s electrical output) and 12.5% of the recovered thermal output, equivalent to 2.23 million kBtu. The difference between the electric only scenario and the electric and gas, on the electric side, is 5% of the electric output or 250,000 kWh, which would require 2.45 million kBtu input at an efficiency of 34.8% HHV.

Where:

E_{CHP} = Useful annual electricity output produced by the CHP system, defined as the annual electric energy output of the CHP system that is actually utilized to replace purchased electricity required to meet the requirements of the facility/process.

= Custom input

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = CF * CHP_{capacity}$$

Where:

CF = Summer Coincidence factor. This factor should also consider any displaced chiller capacity⁵¹⁶

= Custom input

$CHP_{Capacity}$ = CHP nameplate capacity

= Custom input

NATURAL GAS ENERGY SAVINGS:

$$\Delta \text{Therms} = F_{\text{thermalCHP}} \div 100,000$$

Where:

$F_{\text{thermalCHP}}$ = Net savings in annual purchased fuel in Btu, if any, that would have been used on-site by a boiler/furnace to provide some or all of the useful thermal energy output of the CHP system⁵¹⁷.

100,000 = Conversion factor for Btu to therms

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

Custom estimates of maintenance costs that will be incurred for the life of the measure will be used. Maintenance costs vary with type and size of the prime mover. These costs include, but are not limited to:

- Maintenance labor
- Engine parts and materials such as oil filters, air filters, spark plugs, gaskets, valves, piston rings, electronic components, etc. and consumables such as oil
- Minor and major overhauls

For screening purposes, the US EPA has published resource guides that provide average maintenance costs based on CHP technology and system size⁵¹⁸.

⁵¹⁶ If some or all of the existing electric chiller peak demand is no longer needed due to new waste heat powered chillers (e.g., absorption), the coincidence factor should be adjusted appropriately.

⁵¹⁷ In most cases, it is expected that waste-heat-to-power systems will not provide any new net useful thermal energy output, since the CHP system will be driven by thermal energy that was otherwise being wasted. If additional natural gas or other purchased energy is used onsite, it should be properly accounted for.

⁵¹⁸ "EPA Combined Heat and Power Partnership Resources" Oct 07, 2014, <http://www.epa.gov/chp/resources.html> in the document "Catalog of CHP technologies" http://www.epa.gov/chp/documents/catalog_chptech_full.pdf pages 2-16, 3-14, 4-14, 5-14, and 6-16.

COST-EFFECTIVENESS SCREENING AND LOAD REDUCTION FORECASTING

For the purposes of forecasting load reductions due to CHP projects per Section 16-111.5B, changes in site energy use at the customer's meter – reduced consumption of utility provided electricity – adjusted for utility line losses (at-the-busbar savings), customer switching estimates, NTG, and any other adjustment factors deemed appropriate, should be used.

For the purposes of screening a CHP measure application for cost-effectiveness, changes in site energy use – reduced consumption of utility provided electricity and the net change in consumption of fuel – should be used. In general, the benefit and cost components used in evaluating the cost-effectiveness of a CHP project would include at least the following terms:

Benefits: $E_{\text{CHP}} + \Delta kW + F_{\text{thermal_CHP}}$

Costs: $F_{\text{total_CHP}} + \text{CHP}_{\text{COSTS}} + \text{O\&M}_{\text{COSTS}}$

Where:

$\text{CHP}_{\text{COSTS}}$ = CHP equipment and installation costs as defined in the "Deemed Measure Costs" section

$\text{O\&M}_{\text{COSTS}}$ = CHP operations and maintenance costs as defined in the "Deemed O&M Cost Adjustment Calculation" section

MEASURE CODE: CI-HVC-CHAP-V02-180101

REVIEW DEADLINE: 1/1/2020

4.4.33 Industrial Air Curtain

DESCRIPTION

This measure applies to buildings with exterior entryways that utilize overhead doors. All other air curtain applications, such as through sliding door entryways or conventional foot-traffic entryways, require custom analysis as air curtain designs must often accommodate other factors that may change their effectiveness.

The use of overhead doors within exterior entryways during the heating season leads to the exfiltration of warm air from the upper portion of the door opening and the infiltration of colder air from the lower portion of the door opening. This results in increase heating energy use to compensate for heat losses every time a door is opened. By reducing heat losses, air curtains can also enhance the physical comfort of employees or customers near the entryway as there will be reduced temperature fluctuations when the door is opened and closed. In addition, in some cases excess heating capacity may be installed in buildings to meet this larger heating load. The addition of air curtains to exterior entryways that currently utilize overhead doors will result in energy savings and enhanced personal comfort, and also possibly in reduced equipment sizing and corresponding costs.

The primary markets for this measure are commercial and industrial facilities with overhead doors in exterior entryways, including but not limited to the following building types: retail, manufacturing, and warehouse (non-refrigerated).

Limitations

- For use in conditioned spaces with an overhead door in an exterior entryway. This measure does include other door types such doorways to commercial spaces such as retail.
- This measure should only be applied to spaces in which the overhead door separates a conditioned space and an unconditioned space.
- Installation must follow manufacturer recommendations to attain proper air velocity, discharge angle down to the floor level, and unit position.
- Certain heating systems may not be a good fit for air curtains, such as locations with undersized heating capacity. In these cases, the installation of an air curtain may not effectively reduce heating system cycling given the inappropriately sized heating capacity.
- Buildings with slightly positive to slightly negative (~ 5 Pa to -10 Pa). For all other scenarios, custom analysis is recommended.
- Measure assumes that wind speeds at near ground level are less than or equal to 12 mph for 90% of the heating or cooling season. For areas with more extreme weather, custom analysis is necessary.
- Note: for cost effectiveness, it is recommended that minimum door open times should be approximately 15 hours per week.⁵¹⁹

This measure was developed to be applicable to the following program types: NC, RF. If applied to other program types, the measure savings should be verified.

The following methodology is highly complex and requires significant data collection. It is hoped that simplifying steps can be made in future iterations based on continued metering and evaluation of installations. Also the data collected through implementing the measure in the way currently drafted will aid in simplifying efforts at a future date.

DEFINITION OF EFFICIENT EQUIPMENT

Overhead air curtains designed for commercial and industrial applications that have been tested and certified in accordance with ANSI/AMCA 220 and installed following manufacturer guidelines. Measure is for standard models without added heating.

⁵¹⁹ Spentzas, Steve, et. al, "1009: Commercial and Industrial Air Curtains – Public Project Report," Nicor Gas Emerging Technology Program (Oct 2014): 9

DEFINITION OF BASELINE EQUIPMENT

No air curtain or other currently installed means to effectively reduce heat loss and air mixing during door openings, such as a vestibule or strip curtain.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years.⁵²⁰

DEEMED MEASURE COST

The incremental capital cost for overhead air curtains for exterior entryways are as follows, with an added average installation cost approximately equal to the capital cost.⁵²¹

Door Size	Capital Cost
8'w x 8'h	\$3,600
10'w x 10'h	\$4,500
10'w x 12'h	\$5,400
12'w x 14'h	\$8,000
16'w x 16'h	\$13,300

LOADSHAPE

Heating Season: If electric heating, use Commercial Electric Heating Loadshape: C04. Otherwise, N/A

Cooling Season: Commercial Cooling Loadshape C03. Or, if applicable, use Commercial Electric Heating and Cooling Loadshape C05.

COINCIDENCE FACTOR

CF_{SSP} = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)
= 91.3%⁵²²

CF_{PJM} = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)
= 47.8%⁵²³

Algorithm

CALCULATION OF ENERGY SAVINGS

The following formulas provide a methodology for estimating cooling load (kWh) and heating load (therm) savings associated with the installation of air curtains on exterior entryways such as a single door or loading bay. This algorithm is based on the assumption that therm savings are directly related to the difference in cooling or heating losses due to infiltration or exfiltration through an entryway before and after the installation of an AMCA certified air curtain. Energy savings are assumed to be the result of a reduction of natural infiltration effects due to wind and thermal forces and follow the calculation methodology outlined by the ASHRAE Handbook.⁵²⁴ The calculation

⁵²⁰ Navigant Consulting Inc, Measures and Assumptions for Demand Side Management (DSM) Planning: Appendix C: Substantiation Sheets, "Air Curtains – Single Door," Ontario Energy Board, (April 2009): C-137.

2014 Database for Energy-Efficient Resources, EUL/RUL (Effective/Remaining Useful Life) Values, February 4, 2014.

⁵²¹ Based on manufacturer interviews and air curtain specification sheets.

⁵²² Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

⁵²³ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

⁵²⁴ ASHRAE, "Ventilation and Infiltration," in 2013 ASHRAE Handbook – Fundamentals (2013): Ch 16.1 - 16.37

assumes that the air curtain is appropriately sized and commissioned to be effective in mitigating infiltration of winds of up to 12 mph for at least 90% of the year (based on manufacturer literature and TMY3 wind speed ranges at near ground level for Illinois).⁵²⁵ Additionally, this measure assumes the HVAC systems are appropriately balanced such that the maximum pressure differential between indoor air and outdoor air is within the range of $5 \text{ Pa} < \Delta P < -10 \text{ Pa}$.⁵²⁶ Custom analysis is necessary if building pressurization exceeds this range. However, while effectiveness decreases, some studies suggest that air curtains outperform vestibules and single door construction for negatively pressurized buildings with a ΔP of above -30 Pa .⁵²⁷

This algorithm allows either actual inputs or provides estimates if actual data is not available. All weather dependent values are derived from TMY3 data for the closest weather station to those locations defined elsewhere in the Illinois TRM (which are based on 30 year climate normals). If TMY3 weather station data was not available for the data used in the Illinois TRM, the next closest weather station was used. It is assumed that weather variations are negligible between the weather stations located within the same region. This approach was followed as the air curtain algorithm has a number of weather dependent variables which are all calculated in relation to the heating season or cooling season as defined by the balance point temperature deemed appropriate for the facility. All weather dependent data is based on TMY3 data and is listed in tables by both climate zone and balance point temperature, which is then normalized to the Illinois TRM climate zoned HDD/CDD definitions unless otherwise noted.

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh}_{\text{cooling}} = [(Q_{\text{tbc}} - Q_{\text{tac}}) / \text{EER} - (\text{HP} * 0.7457)] * t_{\text{open}} * \text{CD}$$

$$\Delta \text{kWh}_{\text{HPheating}} = [(Q_{\text{tbc}} - Q_{\text{tac}}) / \text{HSPF} - (\text{HP} * 0.7457)] * t_{\text{open}} * \text{HD}$$

$$\Delta \text{kWh}_{\text{Gasheating}} = -(\text{HP} * 0.7457) * t_{\text{open}} * \text{HD}$$

Where:

Q_{tbc} = rate of total heat transfer through the open entryway, before air curtain (kBtu/hr)

Q_{tac} = rate of total heat transfer through the open entryway, after air curtain (kBtu/hr)

(see calculation in 'Heat Transfer Through Open Entryway with/without Air Curtain' sections below)

EER = energy efficiency ratio of the cooling equipment (kBtu/kWh)

= Actual. If unknown, use the table C403.2.3(2) in IECC 2012 (or IECC 2015 if through new construction) to assume values based on code estimates

HP = Input power for air curtain (hp)

= Actual value. If actual value not available, use the following estimates based on manufacturer specs

Door Size	Fan HP
8'w x 8'h	1
10'w x 10'h	1.5
10'w x 12'h	4
12'w x 14'h	6

⁵²⁵ National Solar Radiation Data Base – 1991 – 2005 Update: Typical Meteorological year 3. http://rredc.nrel.gov/solar/old_data/nsrdb/1991-2005/tmy3/by_state_and_city.html

⁵²⁶ Spentzas, Steve, et. al, "1009: Commercial and Industrial Air Curtains – Public Project Report," Nicor Gas Emerging Technology Program (Oct 2014): 10

Wang, Liangzhu, "Investigation of the Impact of Building Entrance Air Curtain on Whole Building Energy Use," Air Movement and Control International, Inc. (2013). 4

⁵²⁷ Wang, Liangzhu, "Investigation of the Impact of Building Entrance Air Curtain on Whole Building Energy Use," Air Movement and Control International, Inc. (2013). 4

Door Size	Fan HP
16'w x 16'h	12

0.7457 = unit conversion factor, brake horsepower to electric power (kW/HP)

t_{open} = average hours per day the door is open (hr/day)

= Actual or user defined estimated value.

CD = cooling days per year, total days in year above balance point temperature (day)

= use table below to select the best value for location⁵²⁸

Climate Zone - Weather Station/City	CD (Balance Point Temperature)				
	45 °F	50 °F	55 °F	60 °F	65 °F
1 - Rockford AP / Rockford	194	168	148	124	97
2 - Chicago O'Hare AP / Chicago	194	173	153	127	95
3 - Springfield #2 / Springfield	214	194	174	148	114
4 - Belleville SIU RSCH / Belleville	258	229	208	174	138
5 - Carbondale Southern IL AP / Marion	222	201	181	158	130

HSPF = Heating System Performance Factor of heat pump equipment

= Actual. If unknown, use the table C403.2.3(2) in IECC 2012 (or IECC 2015 if through new construction) to assume values based on code estimates

HD = heating days per year, total days in year above balance point temperature (day)

= use table below to select an appropriate value⁵²⁹:

Climate Zone - Weather Station/City	HD				
	45 °F	50 °F	55 °F	60 °F	65 °F
1 -Rockford AP / Rockford	142	160	183	204	228
2 - Chicago O'Hare AP / Chicago	150	166	192	219	253
3 - Springfield #2 / Springfield	125	142	167	194	230
4 - Belleville SIU RSCH / Belleville	101	115	134	156	180
5 - Carbondale Southern IL AP / Marion	103	123	148	174	205

Heat Transfer Through Open Entryway without Air Curtain (Cooling Season)

⁵²⁸ National Solar Radiation Data Base – 1991 – 2005 Update: Typical Meteorological year 3. http://rredc.nrel.gov/solar/old_data/nsrdb/1991-2005/tmy3/by_state_and_city.html

Note that cooling days (CD) are calculated by first determining its value from the TMY3 data associated with the appropriate weather station as defined by and used elsewhere in the Illinois TRM. Using the TMY3 outdoor air dry bulb hourly data, ^{Error!} ~~reference source not found~~ the annual hours are totaled for every hour that the outdoor air dry bulb temperature is above a designated zero heat loss balance point temperature or base temperature for cooling. For commercial and industrial (C&I) buildings, a base temperature for heating of 55 °F is designated in the Illinois TRM, but building specific base temperatures are recommended for large C&I projects. Additionally, the TRM uses a 30-year normal data for degree-days while the CD calculation was based on TMY3 data; in order to account for this, calculations of CD were also normalized by the ratio of CDD to align the calculated values more closely with the TRM.

⁵²⁹ Note that Heating Days (HD) are calculated following the same approach outlined in the Cooling Days section.

$$Q_{tbc} = 4.5 * CFM_{tot} * (h_{oc} - h_{ic}) / (1,000 \text{ Btu/kBtu})$$

Where:

4.5 = unit conversion factor with density of air: 60 min/hr * 0.075 lbm/ft³ (lb*min/(ft*hr))

CFM_{tot} = Total air flow through entryway (cfm), see calculation below

h_{oc} = average enthalpy of outside air during the cooling season (Btu/lb)

= use the below table to determine the approximate outdoor air enthalpy associated with an indoor temperature setpoint and climate zone.⁵³⁰

Climate Zone - Weather Station/City	h _{oc}		
	67 °F	72 °F	77 °F
1 -Rockford AP / Rockford	31.6	33.0	35.3
2 - Chicago O'Hare AP / Chicago	32.0	33.6	35.4
3 - Springfield #2 / Springfield	32.9	34.6	36.6
4 - Belleville SIU RSCH / Belleville	33.5	35.0	36.4
5 - Carbondale Southern IL AP / Marion	34.6	36.2	37.7

h_{ic} = average enthalpy of indoor air, cooling season (Btu/lb)

= use the below table to determine the approximate indoor air enthalpy associated with an indoor temperature setpoint in indoor relative humidity.

Relative Humidity (%)	h _{ic}		
	67 °F	72 °F	77 °F
60	25.5	28.5	31.8
50	23.9	26.6	29.5
40	22.3	24.7	27.3

= an estimate 26.6 Btu/lb associated with the 72 °F and 50% indoor relative humidity case can be used as an approximation if no other data is available. For other indoor temperature setpoints and RH, enthalpies may be interpolated.

The total airflow through the entryway, CFM_{tot}, includes both infiltration due to wind as well as thermal forces, as follows:

$$CFM_{tot} = \text{sqrt}[(CFM_w)^2 + (CFM_t)^2]$$

Where:

CFM_w = Infiltration due to the wind (cfm)

CFM_t = Infiltration due to thermal forces (cfm)

The infiltration due to the wind is calculated as follows:

$$CFM_w = (v_{wc} * C_{wc}) * C_v * A_d * (88 \text{ fpm/mpH})$$

Where:

⁵³⁰ Average enthalpies were estimated following ASHRAE guidelines for perfect gas relationships for dry air associated with hourly TMY3 data. Error! Reference source not found. Enthalpies were then averaged for all values associated with a dry-bulb outdoor air temperature that exceeded the indoor air temperature setpoint. Other enthalpy values may be interpolated for indoor air temperature setpoints not represented in the table. Note that while outdoor air enthalpies increase with higher temperature setpoints, the change in enthalpy from indoor to outdoor will decrease.

V_{wc} = average wind speed during the cooling season based on entryway orientation (mph)
 = use the below table to for the wind speed effects based on climate zone and entryway orientation⁵³¹:

Climate Zone -Weather Station /City	Entryway Orientation			
	N	E	S	W
1 -Rockford AP / Rockford	4.2	4.1	4.7	4.8
2 - Chicago O'Hare AP / Chicago	4.7	4.5	5.4	4.6
3 - Springfield #2 / Springfield	4.1	3.7	6.0	5.0
4 - Belleville SIU RSCH / Belleville	3.3	2.7	3.8	4.2
5 - Carbondale Southern IL AP / Marion	3.1	2.9	4.4	3.8

C_{wc} = wind speed correction factor due to wind direction in cooling season, (%)
 = because wind direction is not constant, a wind speed correction factor is used to adjust for the amount of time during the cooling season prevailing winds can be expected to impact the entryway. Use the following table to determine the correct wind speed correction factor for cooling applications.

Climate Zone -Weather Station/City	Entryway Orientation			
	N	E	S	W
1 -Rockford AP / Rockford	0.18	0.13	0.30	0.31
2 - Chicago O'Hare AP / Chicago	0.18	0.17	0.36	0.26
3 - Springfield #2 / Springfield	0.17	0.12	0.46	0.21
4 - Belleville SIU RSCH / Belleville	0.21	0.15	0.35	0.16
5 - Carbondale Southern IL AP / Marion	0.18	0.15	0.37	0.11

Note that correction factors do not add up to 1 (100%). This is attributed to periods of calm winds.

C_v = effectiveness of openings,
 = 0.3, assumes diagonal wind²⁰

A_d = area of the doorway (ft²)
 = user defined

The infiltration due to thermal forces is calculated as follows:

$$CFM_t = A_d * C_{dc} * (60 \text{ sec/min}) * \sqrt{2 * g * H/2 * (T_{oc} - T_{ic}) / (459.7 + T_{oc})}$$

Where:

C_{dc} = the discharge coefficient during the cooling season⁵³²
 = $0.4 + 0.0025 * |T_{ic} - T_{oc}|$
 = 0.42, Illinois average at indoor air temp of 72°F

Note, values for C_{dc} show little variation due to balance point temperature, indoor air temperature, and climate zone. As such, if estimating results, the Illinois average value may be used as a simplification.

⁵³¹ Average wind speeds are calculated based on the TMY3 wind speed data. Because this data is collected at an altitude of 33 ft, wind speed is approximated for a 5 ft level based on ASHRAE Handbook guidelines using the urban/suburban parameters for adjusting wind speed based on altitude ($\alpha = 1200$, $\beta = 0.22$).

ASHRAE, "Airflow Around Buildings," in 2013 ASHRAE Handbook – Fundamentals (2013): p 24.3

⁵³² ASHRAE, "Ventilation and Infiltration," in 2013 ASHRAE Handbook – Fundamentals (2013): p 16.13

g = acceleration due to gravity
= 32.2 ft/sec²

H = the height of the entryway (ft)
= user input

T_{ic} = Average indoor air temperature during cooling season
= User input, can assume indoor cooling temperature set-point

T_{oc} = Average outdoor temp during cooling season (°F)
= the average outdoor temperature is dependent on the CD period and zone. As such, the following table may be used for average outdoor temperature during the cooling period⁵³³:

Climate Zone - Weather Station/City	T _{oc}				
	62 °F	67 °F	72 °F	77 °F	82 °F
1 -Rockford AP / Rockford	72.9	76.0	79.2	82.5	85.5
2 - Chicago O'Hare AP / Chicago	72.9	76.0	79.4	82.8	85.5
3 - Springfield #2 / Springfield	73.7	76.7	79.9	83.4	86.4
4 - Belleville SIU RSCH / Belleville	74.9	77.7	81.0	84.3	86.9
5 - Carbondale Southern IL AP / Marion	75.1	77.7	80.9	84.7	87.4

459.7 = conversion factor from °F to °R

= calculation requires absolute temperature for values not calculated as a difference of temperatures.

Heat Transfer Through Open Entryway with Air Curtain (Cooling Season)

$$Q_{tac} = Q_{tbc} * (1 - E)$$

Where:

E = the effectiveness of the air curtain (%)
= 0.60⁵³⁴

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = (\Delta kW_{cooling} / (CD * 24)) * CF$$

Where:

CF_{SSP} = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)
= 91.3%⁵³⁵

CF_{PJM} = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)

⁵³³ Based on binned data from TMY3 & adjusted bracketed thermostat setpoint temperatures. Interpolate other values as needed.

⁵³⁴ Assumed conservative estimate based on referenced study results and ASHRAE 2004 effectiveness range of 60-80% for air curtains. Jaramillo, Julian, et. Al. "Application of Air Curtains in Refrigerated Chambers," International Refrigeration and Air-Conditioning Conference, Purdue University e-Pubs (July 14-17, 2008): <http://docs.lib.purdue.edu/cgi/viewcontent.cgi?article=1972&context=iracc>

ASHRAE, "Room Air Distribution Equipment," in 2004 ASHRAE Handbook – HVAC Systems and Equipment (2004): p 17.8

⁵³⁵ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

$$= 47.8\%^{536}$$

NATURAL GAS SAVINGS

Natural gas savings, Δ therms, associated with reduced infiltration through an entryway during the heating season are calculated by determining the difference between heat loss through the entryway before and after the installation of the air curtain.

$$\Delta\text{therms} = (Q_{bc} - Q_{ac}) * t_{open} * HD / \eta$$

Where:

Q_{bc} = rate of sensible heat transfer through the open entryway, before air curtain (therm/hr)

Q_{ac} = rate of sensible heat transfer through the open entryway, after air curtain (therm/hr)

t_{open} = average hours per day the door is open (hr/day)

= Actual or estimated user input value

HD = heating days per year, total days in year above balance point temperature (day)

= use table below to select an appropriate value⁵³⁷:

Climate Zone - Weather Station/City	HD				
	45 °F	50 °F	55 °F	60 °F	65 °F
1 -Rockford AP / Rockford	142	160	183	204	228
2 - Chicago O'Hare AP / Chicago	150	166	192	219	253
3 - Springfield #2 / Springfield	125	142	167	194	230
4 - Belleville SIU RSCH / Belleville	101	115	134	156	180
5 - Carbondale Southern IL AP / Marion	103	123	148	174	205

η = efficiency of heating equipment

= Actual. If unknown, assume 0.8

Heat Transfer Through Open Entryway without Air Curtain (Heating Season)

$$Q_{bc} = (1.08 \text{ Btu}/(\text{hr} * ^\circ\text{F} * \text{cfm})) * \text{CFM}_{tot} * (T_{ih} - T_{oh}) / (100,000 \text{ Btu/therm})$$

Where:

1.08 = sensible heat transfer coefficient (specific heat of air and unit conversions)

CFM_{tot} = Total air flow through entryway (cfm)

T_{ih} = Average indoor air temperature during heating season

= User input, can assume indoor heating temperature set-point

T_{oh} = Average outdoor temp during heating season (°F)

= use table below, based on binned data from TMY3 & balance point temperature

⁵³⁶Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

⁵³⁷ Note that Heating Days (HD) are calculated following the same approach outlined in the Cooling Days section.

Climate Zone - Weather Station/City	Avg Outdoor Air Temp - Heating Season				
	45 °F	50 °F	55 °F	60 °F	65 °F
1 -Rockford AP / Rockford	26.3	28.8	31.6	34.2	37.3
2 - Chicago O'Hare AP / Chicago	29.4	31.2	34.0	36.8	40.3
3 - Springfield #2 / Springfield	29.4	31.5	34.6	37.7	41.6
4 - Belleville SIU RSCH / Belleville	31.7	33.6	36.2	39.2	42.3
5 - Carbondale Southern IL AP / Marion	32.5	34.9	37.8	40.7	44.0

The total airflow through the entryway, CFM_{tot} , includes both infiltration due to wind as well as thermal forces, as follows:

$$CFM_{tot} = \sqrt{(CFM_w)^2 + (CFM_t)^2}$$

Where:

CFM_w = Infiltration due to the wind (cfm)

CFM_t = Infiltration due to thermal forces (cfm)

The infiltration due to the wind is calculated as follows:

$$CFM_w = (v_{wh} * C_{wh}) * C_v * A_d * (88 \text{ fpm/mpH})$$

Where:

v_{wh} = average wind speed during the heating season (mph)

= similar to cooling season wind speed assumptions, use the following table to determine average wind speed based on entryway orientation:

Climate Zone -Weather Station/ City	Entryway Orientation			
	N	E	S	W
1 -Rockford AP / Rockford	5.0	4.6	4.9	5.6
2 - Chicago O'Hare AP / Chicago	5.5	5.2	4.9	5.1
3 - Springfield #2 / Springfield	5.0	4.9	5.3	5.1
4 - Belleville SIU RSCH / Belleville	4.3	3.4	3.5	5.3
5 - Carbondale Southern IL AP / Marion	4.6	3.2	4.2	4.4

C_{wh} = wind speed correction factor due to wind direction in heating season, (%)

= because wind direction is not constant, a wind speed correction factor is used to adjust for the amount of time during the heating season prevailing winds can be expected to impact the entryway. Use the following table to determine the correct wind speed correction factor for the heating applications.

Climate Zone -Weather Station/ City	Entryway Orientation			
	N	E	S	W
1 -Rockford AP / Rockford	0.18	0.13	0.30	0.31
2 - Chicago O'Hare AP / Chicago	0.21	0.10	0.26	0.39
3 - Springfield #2 / Springfield	0.21	0.14	0.27	0.34
4 - Belleville SIU RSCH / Belleville	0.31	0.15	0.22	0.29
5 - Carbondale Southern IL AP / Marion	0.31	0.11	0.27	0.18

Note that correction factors do not add up to 1 (100%). This is attributed to periods of calm winds.

C_v = effectiveness of openings,
= 0.3, assumes diagonal wind²⁴

A_d = area of the doorway (ft²)
= user input

The infiltration due to thermal forces is calculated as follows:

$$CFM_t = A_d * C_{dh} * (60 \text{ sec/min}) * \sqrt{2 * g * H/2 * (T_{ih} - T_{oh}) / (459.7 + T_{ih})}$$

Where:

C_{dh} = the discharge coefficient during the heating season
= $0.4 + 0.0025 * |T_{ih} - T_{oh}|$
= 0.49, Illinois average at indoor air temp of 72°F

Note, values for C_{dh} show little variation due to balance point temperature, indoor air temperature, and climate zone. As such, if estimating results, the Illinois average value may be used as a simplification.

g = acceleration due to gravity
= 32.2 ft/sec²

H = the height of the entryway (ft)
= user defined

Heat Transfer Through Open Entryway without Air Curtain (Heating Season)

$$Q_{ac} = Q_{bc} * (1 - E)$$

Where:

E = the effectiveness of the air curtain (%)
= 0.60⁵³⁸

WATER AND OTHER NON-ENERGY IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

The air curtain would need to be regularly serviced and commissioned to ensure that it is appropriately operating. This is estimated at a cost of \$150⁵³⁹.

MEASURE CODE: CI-MSC-AIRC-V01-160601

REVIEW DEADLINE: 1/1/2022

⁵³⁸ Assumed conservative estimate based on referenced study results and ASHRAE 2004 effectiveness range of 60-80% for air curtains. Jaramillo, Julian, et. Al. "Application of Air Curtains in Refrigerated Chambers," International Refrigeration and Air-Conditioning Conference, Purdue University e-Pubs (July 14-17, 2008): <http://docs.lib.purdue.edu/cgi/viewcontent.cgi?article=1972&context=iracc>

ASHRAE, "Room Air Distribution Equipment," in 2004 ASHRAE Handbook – HVAC Systems and Equipment (2004): p 17.8

⁵³⁹ Assumes approximately 1 hour of maintenance (include cleaning out filters, greasing, and checking that the designed angle of attack on the blower nozzle is at the designed position) based on manufacturer input and product spec sheets.

4.4.34 Destratification Fan

DESCRIPTION

This measure applies to buildings with high bay ceiling construction without fans currently installed for the purpose of destratifying air. There is also a separate measure for destratification fans as applied to agricultural settings (“High Volume Low Speed Fans”). All other destratification fan applications require custom analysis.

Air stratification leads to higher temperatures at the ceiling and lower temperatures at the ground. During the heating season, destratification fans improve air temperature distribution in a space by circulating warmer air from the ceiling back down to the floor level, thereby enhancing comfort and saving energy. Energy savings are realized by a reduction of heat loss through the roof-deck and walls as a result of a smaller temperature differential between indoor temperature and outdoor air.

Note that further, but limited, empirical evidence suggests that improved air mixing due to destratification would also result in shorter heating system runtimes due to warmer air reaching the thermostat level sooner, and possibly even allow a facility to lower the thermostat set point while maintaining a similar level of occupant comfort. This is supported by measured data in which an increase in temperatures was observed at the thermostat (5 foot level) level when air is destratified, resulting in an approximate temperature increase at the 5 foot level in the range of 1 - 3°F⁵⁴⁰. This measure does not currently attempt to quantify the potential impacts of air mixing from destratification; however, it should be noted that additional therms savings may be possible.

This measure was developed to be applicable to the following program types: NC, RF. If applied to other program types, the measure savings should be verified.

Limitations

- For use in conditioned, high bay structures. Recommended minimum ceiling height of 20 ft.
- This measure should only be applied to spaces in which the ceiling is subject to heat loss to outdoor air (i.e., single story or top floor spaces) and where there is sufficient space to allow for appropriate spacing of the fans. Other applications require custom analysis.
- Installation must follow manufacturer recommendations sufficient to effectively destratify the entire space. Please see calculation of effective area, A_{eff} , in the therms savings algorithm as a check if this criteria is met. Otherwise, custom calculation is necessary.
- Measure does not currently support facilities with night setbacks on heating equipment. Custom analysis is needed in this case.
- Certain heating systems may not be a good fit for destratification fans, such as locations with: high velocity vertical throw unit heaters, radiant heaters, and centralized forced air systems. In these cases, measured evidence of stratification should be confirmed and custom analysis may be necessary.

DEFINITION OF EFFICIENT EQUIPMENT

High Volume, Low Speed (HVLS) fans with a minimum diameter of 14 ft with Variable Speed Drive (VSD) installed⁵⁴¹.

Note that bell-shaped fans are currently excluded from this measure due to limited validation of the technology available. Further verification of effectiveness compared to HVLS is needed. A manufacturer of bell shaped fans indicates that four bell-shaped fans provide an equivalent effective area as a typical HVLS fan. However, there is a need for further review of bell shaped fan field test data supporting manufacturer claims regarding comparable effectiveness to HVLS technologies.

⁵⁴⁰ Kosar, Doug, “1026: Destratification Fans – Public Project Report,” Nicor Gas, Emerging Technology Program (Oct 2014): 16

⁵⁴¹ Ibid.

DEFINITION OF BASELINE EQUIPMENT

No destratification fans or other means to effectively mix indoor air.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years⁵⁴²

DEEMED MEASURE COST

Measure cost = [incremental cost of HVLS fans] + [installation costs (including materials and labor)]

The incremental capital cost for HVLS fans are as follows⁵⁴³:

Fan Diameter (ft)	Incremental Cost
14	\$6,600
16	\$6,650
18	\$6,700
20	\$6,750
22	\$6,800
24	\$6,850

Since installation cost is depended on a variety of factors, this is a custom entry. Actual costs should be used.

LOADSHAPE

Loadshape C04: Commercial Electric Heating.

COINCIDENCE FACTOR

N/A due to no savings attributable to cooling during the summer peak period.

Algorithm

CALCULATION OF SAVINGS

The following formulas provide a methodology for estimating heating load savings associated with destratification fan use. This algorithm is based on the assumption that savings are directly related to the difference in heat loss through the envelope before and after destratification.

ELECTRIC ENERGY SAVINGS

The algorithm for this measure was developed for natural gas heating applications, however, for electric heating applications, the same methodology presented in the Natural Gas Savings Section may be used with the standard conversion factor from therms to kWh of 29.31 kWh/therm and an equipment efficiency as follows:

⁵⁴² Consistent with both 2008 Database for Energy-Efficient Resources, EUL/RUL (Effective/Remaining Useful Life) Values, October 10, 2008 and GDS Associates, Inc, "Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures," New England Stat Program Working Group (June 2007), p30.

⁵⁴³ Costs were obtained from manufacturer interviews and are based off of average or typical prices for base model HVLS fans. Costs include materials and labor to install the fans and tie fans into an existing electrical supply located near the fan.

System Type	Age of Equipment	HSPF Estimate	η (Effective COP Estimate) (HSPF/3.413)
Heat Pump	Before 2006	6.8	2.0
	2006 - 2014	7.7	2.3
	2015 on	8.2	2.40
Resistance	N/A	N/A	1

Regardless of how the building is heated, the energy consumption of the fans must be accounted for. If the building is electrically heated, fan energy shall be subtracted from the savings as calculated above. If the building is heated with natural gas, this shall represent an electric penalty, i.e., an increase in consumption. This is calculated as follows:

$$\Delta \text{kWh} = - (W_{\text{fan}} * N_{\text{fan}}) * t_{\text{eff}}$$

W_{fan} = fan input power (kW)

N_{fan} = number of fans

t_{eff} = effective annual operation time, based on balance point temperature (hr)

= see table below in Natural Gas Savings section for further detail

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms} = [(\Delta Q_r + \Delta Q_w) * t_{\text{eff}}] / (100,000 * \eta)$$

Where:

ΔQ_r = the heat loss reduction through the roof due to the destratification fan (Btu/hr)

= See calculation section below

ΔQ_w = the heat loss reduction through the exterior walls due to destratification fan (Btu/hr)

= See calculation section below

t_{eff} = effective annual operation time, based on balance point temperature (hr)

= use table below to select an appropriate value⁵⁴⁴:

Climate Zone - Weather Station/City	t_{eff}				
	45 °F	50 °F	55 °F	60 °F	65 °F
1 - Rockford AP / Rockford	3810	4226	4880	5571	6436
2 - Chicago O'Hare AP / Chicago	3593	3986	4603	5254	6070
3 - Springfield #2 / Springfield	3038	3370	3891	4442	5131
4 - Belleville SIU RSCH / Belleville	2243	2488	2873	3280	3789
5 - Carbondale Southern IL AP / Marion	2271	2519	2909	3320	3836

100,000 = conversion factor (1 therm = 100,000 Btu)

⁵⁴⁴ These were calculated at various base temperatures using TMY3 data and adjusted to make consistent with the 30 year normal data used elsewhere. For more information see 'Destratification Fan Workpaper'; Robert Irmiger, Gas Technology Institute, 9/6/2015.

η = thermal efficiency of heating equipment
 = Actual. If unknown assume 0.8.

EXAMPLE:

For a warehouse facility located in Rockford, IL, installing destratification fans could reduce heat loss through the roof of 95,000 Btu/hr and a reduced heat loss through the wall of 51,228 Btu/hr. Assuming a balance point of 55°F the therms savings for the facility would be estimated as:

$$\begin{aligned}\Delta\text{Therms} &= [(\Delta Q_r + \Delta Q_w) * t_{\text{eff}}] / (100,000 * \eta) \\ &= [(95,000 \text{ Btu/hr} + 51,228 \text{ Btu/hr}) * 4880 \text{ hr}] / [(100,000 \text{ Btu/therm}) * 0.8] \\ &= 8,923 \text{ therms}\end{aligned}$$

Heat loss reduction through the roof

$$\begin{aligned}\Delta Q_r &= Q_{r,s} - Q_{r,d} \\ &= (1/R_r) * A_r * [(T_{r,s} - T_{oa}) - (T_{r,d} - T_{oa})] \\ &= (1/R_r) * A_r * (T_{r,s} - T_{r,d})\end{aligned}$$

Where:

$Q_{r,s}$ = roof heat loss for stratified space
 $Q_{r,d}$ = roof heat loss for destratified space
 R_r = overall thermal resistance through the roof (hr * ft² * °F / Btu)
 = Actual or estimated based on construction type. If unknown, assume the following:

Thermal Resistance Factor (R-Factor) for Roof	Retrofit ⁵⁴⁵	New Construction ⁵⁴⁶ (2010 or newer)
R_r	10.0 (hr * ft ² * °F / Btu)	20.0 (hr * ft ² * °F / Btu)

A_r = roof area (ft²)
 = user input
 = can be approximated with floor area
 T_{oa} = outside air temperature, note: therm savings calculations are actually independent of outside air because this term drops out of the heat loss reduction equation
 $T_{r,s}$ = indoor temperature at roof deck, stratified case (°F)
 = Actual. If unknown, use the following equation

⁵⁴⁵ ANSI/ASHRAE/IESNA 100-1995, "Energy Conservation in Existing Buildings," ASHRAE Standard (1995). Additionally, professional judgement was used to address older vintage structure prior to adoption of the 1995 standard and an estimate of 50% of current code standard was used.

⁵⁴⁶ ANSI/ASHRAE/IESNA Standard 90.1-2007, "Energy Standard for Buildings Except Low-Rise Residential Buildings," ASHRAE Standard (2007): Table 5.5-4 and Table 5.5-5

$$= m_s * h_r + T_{f,s}$$

h_r = ceiling height/roof deck (ft)

m_s = estimated heat gain per foot elevation, stratified case (°F/ft)

$$= 0.8 \text{ °F/ft}$$

= Professional judgement used to define value based on result from a Nicor Gas ETP Pilot field testing results and the Ansley article^{547,548}. Estimates from these sources fall on the conservative side of the industry rule of thumb range of 1-2 °F/ft heat gain.

$T_{f,s}$ = estimated floor temperature, stratified case (°F)

$$= T_{tstat} - m_s * h_{tstat}$$

$$= T_{tstat} - 4 \text{ °F}$$

T_{tstat} = temperature set point at the thermostat

h_{tstat} = vertical distance between the floor and the thermostat, assumed 5ft

$T_{r,d}$ = indoor temp at roof, destratified case

= actual value, or may be estimated using the following:^{549,550}

$$= T_{tstat} + 1 \text{ °F}$$

EXAMPLE:

For a 50,000 ft² warehouse built in 1997 with 30 ft ceilings and a thermostat set point of 65 °F. No further measured values available.

$$\begin{aligned} \Delta Q_r &= (1/R_r) * A_r * (T_{r,s} - T_{r,d}) = (1/R_r) * A_r * [(m_s * h_r + T_{tstat} - 4 \text{ °F}) - (T_{tstat} + 1 \text{ °F})] \\ &= (1/R_r) * A_r * [(0.8 \text{ °F/ft} * h_r) - 5 \text{ °F}] \\ &= 1/(10 \text{ hr} * \text{ft}^2 * \text{°F} / \text{Btu}) * (50,000 \text{ ft}^2) * [(0.8 \text{ °F/ft} * 30 \text{ ft}) - 5 \text{ °F}] \\ &= 95,000 \text{ Btu/hr} \end{aligned}$$

Heat loss reduction through exterior walls

Note: a conservative estimate for therms savings would neglect the impact of heat loss through the walls. However, Ansley suggests that estimates based on the roof deck losses alone underestimate actual savings by up to 46%.⁵⁵¹

$$\begin{aligned} \Delta Q_w &= Q_{w,s} - Q_{w,d} \\ &= (1/R_w) * A_w * (T_{w,s} - T_{w,d}) \end{aligned}$$

Where:

⁵⁴⁷ Kosar, Doug, "1026: Destratification Fans – Public Project Report," Nicor Gas, Emerging Technology Program (Oct 2014): 10-11. Field testing results indicated approximately 0.6 oF/ft for a garden center.

⁵⁴⁸ Aynsley, Richard, "Saving Heating Costs in Warehouses," ASHRAE Journal (Dec 2005): 48. Identifies a 0.8 oF/ft gain.

⁵⁴⁹ 12. Kosar, Doug, "1026: Destratification Fans – Public Project Report," Nicor Gas, Emerging Technology Program (Oct 2014): 10-11. Field testing results indicated approximately 0.6 oF/ft for a garden center.

⁵⁵⁰ 13. Aynsley, Richard, "Saving Heating Costs in Warehouses," ASHRAE Journal (Dec 2005): 48.

⁵⁵¹ Aynsley, Richard, "Saving Heating Costs in Warehouses," ASHRAE Journal (Dec 2005): 51

R_w = overall thermal resistance through the exterior walls ($\text{hr} \cdot \text{ft}^2 \cdot ^\circ\text{F} / \text{Btu}$)
 = Actual or estimated based on construction type⁵⁵². If unknown, assume the following

Thermal Resistance Factor (R-Factor) for Wall	Retrofit ⁵⁵³	New Construction ⁵⁵⁴ (2010 or newer)
R_w	6.5 ($\text{hr} \cdot \text{ft}^2 \cdot ^\circ\text{F} / \text{Btu}$)	13.0 ($\text{hr} \cdot \text{ft}^2 \cdot ^\circ\text{F} / \text{Btu}$)

A_w = area of exterior walls (ft^2)

= user input

$T_{w,s}$ = average indoor air temperature for wall heat loss, stratified case

= If actual $T_{r,s}$ measurement is available⁵⁵⁵

$$= [(T_{r,s} \cdot h_a) + (T_{tstat} \cdot h_b)] / h_r$$

h_a = vertical distance between the heat source and the ceiling

h_b = vertical distance between the floor and the heat source

= Otherwise, use the linear stratification equation at average space height, see definition above.

$$= m_s \cdot (h_r / 2) + T_{f,s}$$

$$= m_s \cdot (h_r / 2) + (T_{tstat} - 4)$$

$T_{w,d}$ = average indoor air temperature for wall heat loss, destratified case

$$= T_{tstat} + 0.5$$

⁵⁵² Because heat loss through the walls is estimated using the average space temperature pre- and post- destratification. There are a number of factors that can impact the average space temperature causing deviations from estimates of many degrees in some cases. As such, it is recommended that a conservative value for the thermal resistance through the walls, R_w , be used. A recommended method for determining R_w would be to use the highest R-value for the wall space, neglecting lower R-values associated with windows, thermal bridges, etc.

⁵⁵³ ANSI/ASHRAE/IESNA 100-1995, "Energy Conservation in Existing Buildings," ASHRAE Standard (1995). Additionally, professional judgement was used to address older vintage structure prior to adoption of the 1995 standard and an estimate of 50% of current code standard was used.

⁵⁵⁴ ANSI/ASHRAE/IESNA Standard 90.1-2007, "Energy Standard for Buildings Except Low-Rise Residential Buildings," ASHRAE Standard (2007): Table 5.5-4 and Table 5.5-5

⁵⁵⁵ Aynsley, Richard, "Saving Heating Costs in Warehouses," ASHRAE Journal (Dec 2005): 48

= conservative estimate using engineering judgment based on the same assumption used for $T_{r,f}$ estimate.

EXAMPLE:

For a 50,000 ft² warehouse built in 1997 with 1200 ft length of perimeter wall and 30 ft ceilings and a thermostat set point of 65 °F and a measured temperature at the ceiling of 85 °F and unit heaters located 10 feet from the roof:

$$\begin{aligned}
 \Delta Q_w &= (1/R_w) * A_w * (T_{w,s} - T_{w,d}) \\
 &= (1/R_w) * A_w * [((T_{r,s} * h_a) + (T_{tstat} * h_b)) / h_r] - (T_{tstat} + 0.5 \text{ °F})] \\
 &= 1/(6.5 \text{ hr} * \text{ft}^2 * \text{°F}/\text{Btu}) * (1200 * 30) * [((85 \text{ °F} * 10\text{ft}) + (65 \text{ °F} * 20\text{ft})) / 30\text{ft}] - (65 + 0.5 \text{ °F})] \\
 &= 1/(6.5 \text{ hr} * \text{ft}^2 * \text{°F}/\text{Btu}) * (36,000\text{ft}^2) * (71.7 \text{ °F} - 65.5 \text{ °F}) \\
 &= 34,338 \text{ Btu/hr}
 \end{aligned}$$

Measure eligibility check

Use the following algorithm to verify a fan system is sufficiently sized to destratify air across the entire area.

Effective area, A_{eff} , is the area over which a fan or a group of fans can be expected to effectively destratify a space. If A_{eff} is less than the roof area, A_r , a custom analysis approach should be followed to account for the change in the effectiveness of the system. In lieu of more detailed studies, effective area is defined based on the measured results from an Enbridge Gas field study in which the area a fan was expected to effectively destratify was equal to 5 times the fan diameter⁵⁵⁶. Effective area, is calculated as follows:

$$\begin{aligned}
 A_{eff} &= [\pi * (5 * D_{fan})^2] / 4 * N_{fan} \\
 &= 6.25 * \pi * D_{fan}^2 * N_{fan}
 \end{aligned}$$

Where:

A_{eff} = the effective area fan area on the floor (ft²)

D_{fan} = fan diameter

N_{fan} = the number of fans

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-DSFN-V02-180101

REVIEW DEADLINE: 1/1/2021

⁵⁵⁶ Enbridge Gas Distribution, Inc., "Big Fans Deliver Big Bonus," (Aug 2007) https://www.enbridgegas.com/businesses/assets/docs/hunter_douglas_case_study.pdf. Additionally, multiple utilities have adopted this definition in their programs including Enbridge Gas and Consumers Energy.

4.4.35 Economizer Repair and Optimization

DESCRIPTION

Economizers are designed to use unconditioned outside air (OSA) instead of mechanical cooling to provide cooling when exterior conditions permit. When the OSA temperature is less than the changeover temperature (determined by a static setpoint or a reference return air sensor) up to 100% OSA is supplied to help meet the facility's cooling needs, thus reducing mechanical cooling energy and saving energy. An economizer that is not working or is not properly adjusted can waste energy and cause comfort issues. This HVAC Economizer Optimization measure involves the repair and optimization of common economizer problems such as adjusting changeover setpoint, repairing damper motors & linkages and replacing non-working sensors and/or controllers. These repairs and adjustments result in proper operation which maximizes both occupant comfort and energy savings.

This measure is only appropriate for single zone packaged rooftop units. Custom calculations are required for savings for multi-zone systems.

In general the HVAC Economizer Optimization measure may involve both repair and/or optimization;

Economizer Repair – The Economizer repair work is performed to ensure that the existing economizer is working properly. This allows the system to take advantage of free cooling and ensure that the system is not supplying an excess amount of outside air (OSA) during non-economizing periods.

- **Replace Damper Motor** – If the existing damper motor is not operational, the unit will be replaced with a functioning motor to allow proper damper modulation.
- **Repair Damper linkage** – If the existing linkage is broken or not adjusted properly, the unit will be replaced or adjusted to allow proper damper modulation.
- **Repair Economizer Wiring** – If the existing economizer is not operational due to a wiring issue, the issue will be repaired to allow proper economizer operation.
- **Reduce Over Ventilation** – If the unit is supplying excess OSA, the OSA damper position will be adjusted to meet minimum ventilation requirements.
- **Economizer Sensor Replacement** – If the unit is equipped with a nonadjustable dry bulb (i.e. snapdisk) or malfunctioning analog sensor, the sensor is replaced with a new selectable sensor.
- **Economizer Control Replacement** – If the existing economizer controller is not operational, the unit will be replaced or upgraded to allow for proper economizer operation.

Economizer Optimization– The economizer optimization work is performed to ensure that the existing economizer system is set up properly to maximize use of free cooling for units located in a particular climate zone.

- **Economizer Changeover Setpoint Adjustment** – If the unit is equipped with a fully operational economizer, the controller is adjusted to the appropriate changeover setpoint based on ASHRAE 90.1 (Figure 1 - *Table 6.5.1.1.3 High-Limit Shutoff Control Settings for Air Economizers*) for the corresponding climate zone.
- **Enable Integrated Operation** – If the unit is equipped with a fully operational economizer and is not set up to allow a minimum of two stages of cooling (1st stage – Economizer Only & 2nd Stage – Economizer & Mechanical cooling), the unit will be wired to allow two stage cooling

This measure was developed to be applicable to the following program types: RF, DI.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment condition is defined by fully functional economizer that is programmed to meet ASHRAE 90.1 economizer changeover setpoint requirements for the facility's climate zone and changeover control type (Figure 1 - Table 6.5.1.1.3 High-Limit Shutoff Control Settings for Air Economizers)⁵⁵⁷.

⁵⁵⁷ ASHRAE, Standard 90.1-2013 - <https://www.ashrae.org/resources--publications/bookstore/standard-90-1>

Figure 1 – Baseline ASHRAE High-Limit Shutoff Control Settings**TABLE 6.5.1.1.3 High-Limit Shutoff Control Settings for Air Economizers^b**

Control Type	Allowed Only in Climate Zone at Listed Setpoint	Required High-Limit Setpoints (Economizer Off When):	
		Equation	Description
Fixed dry-bulb temperature	1b, 2b, 3b, 3c, 4b, 4c, 5b, 5c, 6b, 7, 8	$T_{OA} > 75^{\circ}\text{F}$	Outdoor air temperature exceeds 75°F
	5a, 6a	$T_{OA} > 70^{\circ}\text{F}$	Outdoor air temperature exceeds 70°F
	1a, 2a, 3a, 4a,	$T_{OA} > 65^{\circ}\text{F}$	Outdoor air temperature exceeds 65°F
Differential dry-bulb temperature	1b, 2b, 3b, 3c, 4b, 4c, 5a, 5b, 5c, 6a, 6b, 7, 8	$T_{OA} > T_{RA}$	Outdoor air temperature exceeds return air temperature
Fixed enthalpy with fixed dry-bulb temperature	All	$h_{OA} > 28 \text{ Btu/lb}^a$ or $T_{OA} > 75^{\circ}\text{F}$	Outdoor air enthalpy exceeds 28 Btu/lb ^a of dry air ^a or outdoor air temperature exceeds 75°F
Differential enthalpy with fixed dry-bulb temperature	All	$h_{OA} > h_{RA}$ or $T_{OA} > 75^{\circ}\text{F}$	Outdoor air enthalpy exceeds return air enthalpy or outdoor air temperature exceeds 75°F

a. At altitudes substantially different than sea level, the fixed enthalpy limit shall be set to the enthalpy value at 75°F and 50% RH. As an example, at approximately 6000 ft elevation, the fixed enthalpy limit is approximately 30.7 Btu/lb.

b. Devices with selectable rather than adjustable setpoints shall be capable of being set to within 2°F and 2 Btu/lb of the setpoint listed.

Figure 2 – ASHRAE Climate Zone Map

NORMATIVE APPENDIX B CLIMATE ZONES FOR U.S. STATES AND COUNTIES

This normative appendix provides the climate zones for U.S. states and counties. Figure B-1 contains the county-level climate zone map for the United States. Table B-1 lists each state and major counties within the state and shows the climate number and letter for each county listed.

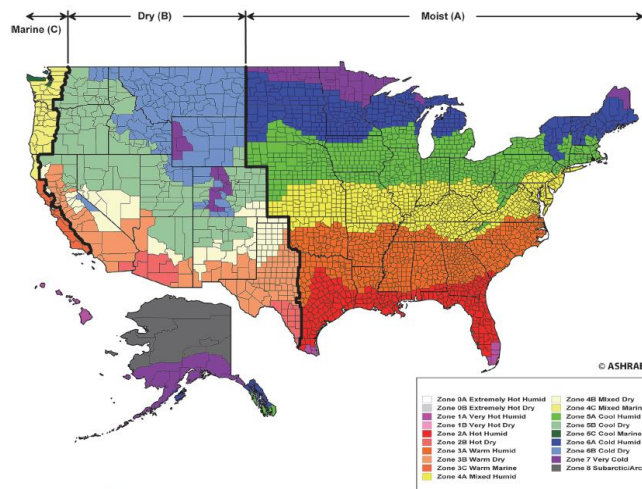


FIGURE B-1 Climate zones for United States counties.

DEFINITION OF BASELINE EQUIPMENT

The baseline for this measure is an existing economizer installed on a packaged single zone rooftop HVAC unit. The existing economizer system is currently not operating as designed due to mechanical and/or control problems, and/or is not optimally adjusted.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 5 years⁵⁵⁸.

DEEMED MEASURE COST

The cost for this measure can vary considerably depending upon the existing condition of the economizer and the work required to achieve the required efficiency levels. Measure cost should be determined on a site-specific basis.

⁵⁵⁸ [California Public Utilities Commission, DEER 2014 EUL Table D08 v2.05](#)

LOADSHAPE

Loadshape C03 - Commercial Cooling

COINCIDENCE FACTOR

N/A

Algorithm**CALCULATION OF ENERGY SAVINGS**

The savings calculation methodology uses a regression equation to calculate the energy savings for a variety of common situations⁵⁵⁹. The equation variables are limited to the ranges listed; if the actual conditions fall outside of these ranges custom calculations are required.

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = [\text{Baseline Energy Use (kWh/Ton)} - \text{Proposed Energy Use (kWh/Ton)}] * \text{Cooling Capacity (Tons)}$$

The following equations are used to calculate baseline and proposed electric energy use⁵⁶⁰.

Electric Energy Use Equations (kWh / ton)

Building Type	Changeover Type	Equation
Assembly	Fixed Dry-Bulb (DB)	$\text{cz} + \text{CSP} * -2.021 + \text{EL} * -16.362 + \text{OAn} * 1.665 + \text{OAx} * -3.13$
	Dual Temperature Dry-Bulb (DTDB)	$\text{cz} + \text{EL} * -11.5 + \text{OAn} * 1.635 + \text{OAx} * -2.817$
	Dual Temperature Enthalpy (DTEnth)	$\text{cz} + \text{EL} * -17.772 + \text{OAn} * 1.853 + \text{OAx} * -3.044$
	Fixed Enthalpy (Enth)	$\text{cz} + \text{CSP} * -5.228 + \text{EL} * -17.475 + \text{OAn} * 1.765 + \text{OAx} * -3.003$
	Analog ABCD Economizers (ABCD)	$\text{cz} + \text{CSP} * -2.234 + \text{EL} * -16.394 + \text{OAn} * 1.744 + \text{OAx} * -3.01$
Convenience Store	DB	$\text{cz} + \text{CSP} * -3.982 + \text{EL} * -27.508 + \text{OAn} * 2.486 + \text{OAx} * -4.684$
	DTDB	$\text{cz} + \text{EL} * -20.798 + \text{OAn} * 2.365 + \text{OAx} * -3.773$
	DTEnth	$\text{cz} + \text{EL} * -30.655 + \text{OAn} * 2.938 + \text{OAx} * -4.461$
	Enth	$\text{cz} + \text{CSP} * -8.648 + \text{EL} * -25.678 + \text{OAn} * 2.092 + \text{OAx} * -3.754$
	ABCD	$\text{cz} + \text{CSP} * -3.64 + \text{EL} * -24.927 + \text{OAn} * 2.09 + \text{OAx} * -3.788$
Office - Low Rise	DB	$\text{cz} + \text{CSP} * -0.967 + \text{EL} * -6.327 + \text{OAn} * 2.87 + \text{OAx} * -1.047$
	DTDB	$\text{cz} + \text{OAn} * 2.968 + \text{OAx} * -0.943$
	DTEnth	$\text{cz} + \text{EL} * -9.799 + \text{OAn} * 3.106 + \text{OAx} * -1.085$
	Enth	$\text{cz} + \text{CSP} * -2.773 + \text{EL} * -7.392 + \text{OAn} * 2.941 + \text{OAx} * -0.974$
	ABCD	$\text{cz} + \text{CSP} * -1.234 + \text{EL} * -7.229 + \text{OAn} * 2.936 + \text{OAx} * -0.995$
Religious Facility	DB	$\text{cz} + \text{CSP} * -1.131 + \text{OAn} * 3.542 + \text{OAx} * -1.01$
	DTDB	$\text{cz} + \text{EL} * -10.198 + \text{OAn} * 4.056 + \text{OAx} * -1.279$
	DTEnth	$\text{cz} + \text{OAn} * 3.775 + \text{OAx} * -1.031$
	Enth	$\text{cz} + \text{CSP} * -2.13 + \text{OAn} * 3.317 + \text{OAx} * -0.629$
	ABCD	$\text{cz} + \text{CSP} * -0.95 + \text{OAn} * 3.313 + \text{OAx} * -0.647$

⁵⁵⁹ For more information on methodology, please refer to workpaper submitted by CLEAResult titled "CLEAResult_Economizer Repair_151020_Finalv2.doc". Note that the original ComEd eQuest models were used in the analysis, rather than the VEIC developed models used elsewhere. VEIC do not consider this a significant issue as adjustments from the ComEd models were focused on calibrating EFLH values, not to overall energy use metrics. We also believe using the ComEd models is likely more conservative. It may be appropriate to update the analysis with the updated models at a later time.

⁵⁶⁰ This approach allows the savings estimate to account for the operational attributes of the baseline as well as the proposed case, yielding a better estimate than an approach that assumes a particular baseline or proposed energy use to determine savings.

Building Type	Changeover Type	Equation
Restaurant	DB	$cz + CSP^* - 2.243 + EL^* - 21.523 + OAx^* - 1.909$
	DTDB	$cz + EL^* - 14.427 + OAn^* 0.295 + OAx^* - 1.451$
	DTEnth	$cz + EL^* - 25.99 + OAn^* 0.852 + OAx^* - 1.951$
	Enth	$cz + CSP^* - 4.962 + EL^* - 16.868 + OAn^* - 0.12 + OAx^* - 1.418$
	ABCD	$cz + CSP^* - 2.115 + EL^* - 16.15 + OAn^* - 0.125 + OAx^* - 1.432$
Retail - Department Store	DB	$cz + CSP^* - 1.003 + OAn^* 3.765 + OAx^* - 0.938$
	DTDB	$cz + OAn^* 3.688 + OAx^* - 0.676$
	DTEnth	$cz + OAn^* 4.081 + OAx^* - 1.072$
	Enth	$cz + CSP^* - 2.545 + OAn^* 3.725 + OAx^* - 0.788$
	ABCD	$cz + CSP^* - 1.175 + OAn^* 3.708 + OAx^* - 0.809$
Retail - Strip Mall	DB	$cz + CSP^* - 1.192 + EL^* - 5.62 + OAn^* 3.353 + OAx^* - 1.142$
	DTDB	$cz + OAn^* 3.355 + OAx^* - 0.915$
	DTEnth	$cz + EL^* - 9.202 + OAn^* 3.642 + OAx^* - 1.215$
	Enth	$cz + CSP^* - 2.997 + EL^* - 5.938 + OAn^* 3.312 + OAx^* - 0.964$
	ABCD	$cz + CSP^* - 1.36 + EL^* - 5.884 + OAn^* 3.3 + OAx^* - 0.987$

Where:

CZ = Climate Zone Coefficient

= Depends on Building Type and Changover Type (see table below)

Building Type	Changeover Type	Electric Climate Zone Coefficients				
		CZ1 (Rockford)	CZ2 (Chicago)	CZ3 (Springfield)	CZ4 (Belleville)	CZ5 (Marion)
Assembly	DB	874.07	886.73	1043.38	1071.48	1072.20
	DTDB	698.45	711.89	870.13	899.51	903.10
	DTEnth	702.06	715.42	873.43	902.76	906.50
	Enth	851.95	865.43	1020.65	1047.10	1053.32
	ABCD	884.19	897.63	1053.12	1080.58	1086.35
Convenience Store	DB	1739.12	1787.09	2128.78	2206.65	2245.93
	DTDB	1389.28	1436.30	1780.99	1863.45	1904.89
	DTEnth	1398.42	1446.82	1789.71	1869.89	1912.59
	Enth	1643.51	1691.34	2032.83	2112.21	2157.63
	ABCD	1692.80	1740.62	2082.35	2162.73	2207.68
Office - Low Rise	DB	674.06	687.17	899.17	993.84	989.16
	DTDB	583.62	597.02	811.39	907.61	903.58
	DTEnth	588.94	602.11	816.02	912.49	908.26
	Enth	668.83	682.23	893.61	987.52	986.59
	ABCD	690.27	703.52	915.27	1009.94	1008.59
Religious Facility	DB	613.26	630.50	853.53	923.99	931.74
	DTDB	518.40	535.45	760.76	832.57	840.72
	DTEnth	513.59	531.20	756.26	829.13	837.26
	Enth	576.94	594.17	817.64	888.37	897.18
	ABCD	593.78	611.04	834.69	905.83	914.27
Restaurant	DB	1397.27	1430.45	1763.21	1837.63	1872.18
	DTDB	1191.82	1225.12	1558.32	1633.95	1669.13
	DTEnth	1192.84	1226.77	1559.41	1635.13	1671.11

Building Type	Changeover Type	Electric Climate Zone Coefficients				
		CZ1 (Rockford)	CZ2 (Chicago)	CZ3 (Springfield)	CZ4 (Belleville)	CZ5 (Marion)
Retail - Department Store	Enth	1343.56	1377.52	1710.11	1783.66	1821.67
	ABCD	1373.72	1407.70	1740.43	1814.74	1852.55
	DB	717.89	730.07	968.85	1034.78	1035.06
	DTDB	628.83	641.70	883.37	951.09	951.33
	DTEnth	629.35	641.90	882.84	951.33	951.44
	Enth	705.06	717.99	956.42	1020.57	1024.45
Retail - Strip Mall	ABCD	728.60	741.47	980.19	1045.30	1048.57
	DB	800.69	818.68	1070.39	1129.87	1133.84
	DTDB	692.97	711.31	965.63	1026.68	1030.41
	DTEnth	698.12	716.34	970.06	1031.78	1035.72
	Enth	784.54	803.35	1054.37	1112.72	1120.74
	ABCD	810.10	828.86	1080.11	1139.39	1146.95

CSP = Economizer Changeover Setpoint (°F or Btu/lb) (actual in ranges below)

Economizer Control Type		Economizer Changeover Setpoint
Dry-Bulb		60°F - 80°F
Dual Temperature Dry-Bulb		0°F -5°F delta
Dual Temperature Enthalpy		0 Btu/lb -5 Btu/lb delta
Enthalpy		18 Btu/lb – 28 Btu/lb
Analog ABCD Economizers	A	73°F
	B	70°F
	C	67°F
	D	63°F
	E	55°F

EL = Integrated Economizer Operation (Economizer Lockout)

= 0 for Economizer w/ Integrated Operation (Two Stage Cooling)

= 1 for Economizer w/ out Integrated Operation (One Stage Cooling)

Oan = Minimum Outside Air (% OSA)⁵⁶¹

= Actual. Must be between 15% -70%. If unknown assume

Functional Economizer – 30%

Non functional Economizer (Damper failed closed) – 15%

Non functional Economizer (Damper failed open) - 30% (Assume Minimum Ventilation (Three Fingers)⁵⁶²)

Oax = Maximum Outside Air (%)ⁱ

= Actual. Must be between 15% -70%. If unknown assume

Functional Economizer – 70%

Non functional Economizer (Damper failed closed) – 15%

⁵⁶¹ DNV GL, "HVAC Impact Evaluation Final Report WO32 HVAC – Volume 1: Report," California Public Utilities Commission, Energy Division, HVAC Commercial Quality Maintenance (CQM) (1/28/14)

⁵⁶² Technician rule of thumb taken from CPUC 'HVAC Impact Evaluation Final Report', WO32, 28Jan 2015, p18.

Non functional Economizer (Damper failed open) — 30% (Assume Minimum Ventilation (Three Fingers))

EXAMPLE

A low rise office building in Rockford (Climate Zone 1) is heated and cooled with a packaged Gas (92 kBtu output) / DX (5 Ton) RTU. The RTU is equipped with a fixed dry-bulb outside air economizer and is programmed for integrated operation. When the technician inspects the RTU they find that the changeover setpoint is programmed to 62°F, which does not meet ASHRAE economizer high limit shut off air economizer recommendations. After further investigation it is found that the OSA damper motor is not operational and is providing 30% outside air.

The technician replaces the damper motor and allow for proper OSA damper modulation (30% Min OSA & 70% Max OSA). They also adjust the fixed dry-bulb changeover setpoint to meet the ASHRAE economizer high limit shut off air economizer recommendation of 70°F.

$$\Delta \text{kWh} = [\text{Baseline Energy Use (kWh/Ton)} - \text{Proposed Energy Use (kWh/Ton)}] * \text{Cooling Capacity (Tons)}$$

Baseline Energy Use (kWh/Ton) = Equation for Office Low Rise

$$= cz + CSP * -0.967 + EL * -6.327 + OAn * 2.87 + OAx * -1.047$$

$$= 674.06 + 62 * -0.967 + 0 * -6.327 + 30 * 2.87 + 30 * -1.047$$

$$= 668.8 \text{ kWh/Ton}$$

Proposed Energy Use (kWh/Ton) = Equation for Office Low Rise

$$= cz + CSP * -0.967 + EL * -6.327 + OAn * 2.87 + OAx * -1.047$$

$$= 674.06 + 70 * -0.967 + 0 * -6.327 + 30 * 2.87 + 70 * -1.047$$

$$= 619.2 \text{ kWh/Ton}$$

$$\Delta \text{kWh} = [668.8 \text{ (kWh/Ton)} - 619.2 \text{ (kWh/Ton)}] * 5 \text{ Tons}$$

$$= 49.6 \text{ kWh/Ton} * 5 \text{ Tons}$$

$$= 248.08 \text{ kWh}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A - It is assumed that repair or optimization of the economizer will not typically have a significant impact summer peak demand.

NATURAL GAS SAVINGS

$$\Delta \text{Therms} = [\text{Baseline Energy Use (Therms/kBtu)} - \text{Proposed Energy Use (Therms/kBtu)}] * \text{Output Heating Capacity (kBtu)}$$

The following equations are used to calculate baseline and proposed electric energy use.

Natural Gas Energy Use Equations (therms / kbtu output)

Building Type	Changeover Type	Equation
Assembly	Fixed Dry-Bulb (DB)	$cz + OAn * 0.0853$
	Dual Temperature Dry-Bulb (DTDB)	$cz + OAn * 0.0866$
	Dual Temperature Enthalpy (DTEth)	$cz + OAn * 0.0866$
	Fixed Enthalpy (Enth)	$cz + OAn * 0.0855$
	Analog ABCD Economizers (ABCD)	$cz + OAn * 0.0855$

Building Type	Changeover Type	Equation
Convenience Store	DB	$cz+OAn*0.26$
	DTDB	$cz+OAn*0.263$
	DTEnth	$cz+OAn*0.263$
	Enth	$cz+OAn*0.261$
	ABCD	$cz+OAn*0.261$
Office - Low Rise	DB	$cz+OAn*0.3$
	DTDB	$cz+OAn*0.301$
	DTEnth	$cz+OAn*0.301$
	Enth	$cz+OAn*0.3$
	ABCD	$cz+OAn*0.3$
Religious Facility	DB	$cz+OAn*0.35$
	DTDB	$cz+OAn*0.348$
	DTEnth	$cz+OAn*0.348$
	Enth	$cz+OAn*0.349$
	ABCD	$cz+OAn*0.349$
Restaurant	DB	$cz+OAn*0.0867$
	DTDB	$cz+OAx*0.038+OAn*OAx*0.00149$
	DTEnth	$cz+OAx*0.038+OAn*OAx*0.00149$
	Enth	$cz+OAn*0.0878$
	ABCD	$cz+OAn*0.0878$
Retail - Department Store	DB	$cz+OAn*0.319$
	DTDB	$cz+OAn*0.318$
	DTEnth	$cz+OAn*0.318$
	Enth	$cz+OAn*0.318$
	ABCD	$cz+OAn*0.318$
Retail - Strip Mall	DB	$cz+OAn*0.215$
	DTDB	$cz+OAn*0.216$
	DTEnth	$cz+OAn*0.216$
	Enth	$cz+OAn*0.215$
	ABCD	$cz+OAn*0.215$

Where:

CZ = Climate Zone Coefficient

= Depends on Building Type and Changeover Type (see table below)

Building Type	Changeover Type	Natural Gas Climate Zone Coefficients				
		CZ1 (Rockford)	CZ2 (Chicago)	CZ3 (Springfield)	CZ4 (Belleville)	CZ5 (Marion)
Assembly	DB	-0.03	-0.55	-1.06	-1.28	-1.71
	DTDB	-0.02	-0.57	-1.11	-1.34	-1.79
	DTEnth	-0.02	-0.57	-1.11	-1.34	-1.79
	Enth	-0.03	-0.55	-1.06	-1.29	-1.72
	ABCD	-0.03	-0.55	-1.06	-1.29	-1.72
Convenience Store	DB	2.95	0.50	-1.48	-2.96	-5.56
	DTDB	3.06	0.52	-1.56	-3.11	-5.81
	DTEnth	3.06	0.52	-1.56	-3.11	-5.81

Building Type	Changeover Type	Natural Gas Climate Zone Coefficients				
		CZ1 (Rockford)	CZ2 (Chicago)	CZ3 (Springfield)	CZ4 (Belleville)	CZ5 (Marion)
	Enth	2.96	0.50	-1.49	-2.98	-5.59
	ABCD	2.96	0.50	-1.49	-2.98	-5.59
Office - Low Rise	DB	5.83	3.02	0.46	-0.92	-4.13
	DTDB	5.98	3.08	0.41	-1.03	-4.36
	DTEnth	5.98	3.08	0.41	-1.03	-4.36
	Enth	5.85	3.03	0.46	-0.93	-4.16
	ABCD	5.85	3.03	0.46	-0.93	-4.16
Religious Facility	DB	9.23	6.71	3.75	2.40	-0.80
	DTDB	9.41	6.83	3.77	2.39	-0.86
	DTEnth	9.41	6.83	3.77	2.39	-0.86
	Enth	9.25	6.73	3.75	2.40	-0.80
	ABCD	9.25	6.73	3.75	2.40	-0.80
Restaurant	DB	8.30	6.54	4.94	4.00	1.95
	DTDB	10.51	8.71	7.07	6.10	4.00
	DTEnth	10.51	8.71	7.07	6.10	4.00
	Enth	8.28	6.51	4.91	3.96	1.90
	ABCD	8.28	6.51	4.91	3.96	1.90
Retail - Department Store	DB	8.20	5.86	3.19	1.25	-2.59
	DTDB	8.35	5.94	3.18	1.18	-2.75
	DTEnth	8.35	5.94	3.18	1.18	-2.75
	Enth	8.21	5.87	3.18	1.24	-2.61
	ABCD	8.21	5.87	3.18	1.24	-2.61
Retail - Strip Mall	DB	6.40	4.35	2.07	0.49	-2.18
	DTDB	6.51	4.38	2.03	0.39	-2.34
	DTEnth	6.51	4.38	2.03	0.39	-2.34
	Enth	6.41	4.35	2.06	0.48	-2.20
	ABCD	6.41	4.35	2.06	0.48	-2.20

EXAMPLE

A low rise office building in Rockford (Climate Zone 1) is heated and cooled with a packaged Gas (92 kBtu output) / DX (5 Ton) RTU. The RTU is equipped with a fixed dry-bulb outside air economizer and is programmed for integrated operation. When the technician inspects the RTU they find that the changeover setpoint is programmed to 62°F, which does not meet ASHRAE economizer high limit shut off air economizer recommendations. After further investigation it is found the OSA damper motor is not operational and is providing 30% outside air.

The technician replaces the damper motor and allow for proper OSA damper modulation (30% Min OSA & 70% Max OSA). They also adjust the fixed dry-bulb changeover setpoint to meet the ASHRAE economizer high limit shut off air economizer recommendation of 70°F.

$$\Delta \text{Therms} = [\text{Baseline Energy Use (Therms/kBtuh)} - \text{Proposed Energy Use (Therms/kBtuh)}] * \text{Output Heating Capacity (kBtuh)}$$

$$\text{Baseline Energy Use (Therms/kBtuh)} = \text{Equation for Office Low Rise}$$

$$= cz + OAn * 0.3$$

$$= 5.83 + 30 * .3$$

$$= 14.8 \text{ Therms/kBtuh output}$$

$$\text{Proposed Energy Use (Therms/kBtuh)} = \text{Equation for Office Low Rise}$$

$$= cz + OAn * 0.3$$

$$= 5.83 + 30 * .3$$

$$= 14.8 \text{ Therms/kBtuh output}$$

$$\Delta \text{Therms} = [14.8 (\text{Therms/kBtuh output}) - 14.8 (\text{Therms/kBtuh output})] * 92 \text{ kBtuh output}$$

$$= 0.0 (\text{Therms/kBtuh output}) * 92 \text{ kBtuh output}$$

$$= 0 \text{ Therms}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-ECRP-V02-160601**REVIEW DEADLINE: 1/1/2023**

4.4.36 Multi-Family Space Heating Steam Boiler Averaging Controls

DESCRIPTION

This measure covers multi-family space heating boiler averaging controls. Temperature sensors are placed in interior spaces to monitor the average temperature of the building. At minimum a sensor must be placed at each corner and at one central location. Additionally, a temperature sensor must monitor the outside air temperature. These sensors shall provide data to the averaging controls. The averaging controls will adjust the boiler operation based upon an average of the indoor sensors and the outside air temperature. These controls shall also incorporate a night-time setback capability. Buildings utilizing thermostatic radiator valves, or other modulating control valves or sequences to control the temperature in individual spaces are not eligible.

This measure was developed to be applicable to the following program types: RF.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify the boiler(s) must incorporate an averaging control system utilizing at least 5 indoor sensors and 1 outdoor sensor. The controls shall have the capability to incorporate a nighttime setback throughout the building.

DEFINITION OF BASELINE EQUIPMENT

The baseline is a boiler system without averaging controls or other steam supply modulating controls. Current boiler control system can utilize a single thermostat or aquastat and timer.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for the domestic hot water boilers is 20 years.⁵⁶³

DEEMED MEASURE COST

AS A RETROFIT MEASURE, THE ACTUAL INSTALLED COST SHOULD BE USED FOR SCREENING PURPOSES.

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

⁵⁶³ The Brooklyn Union Gas Company, High Efficiency Heating and Water and Controls, Gas Energy Efficiency Program Implementation Plan.

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms} = \text{Capacity} \times \text{EFLH} \times \text{SF} / 100,000$$

Where:

Capacity	= Boiler gas input size (Btu/h)
	= Actual
EFLH	= Effective Full Load Hours for heating are provided in section 4.4. HVAC End Use
SF	= Savings Factor
	= 5% ⁵⁶⁴ or custom if savings can be substantiated
100,000	= converts Btu/h to therm

For Example:

A 1,000,000 btu/h steam boiler in a Mid-Rise Multi-Family building in Chicago has averaging controls installed.

$$\begin{aligned} \Delta \text{Therms} &= 1,000,000 \times 1,685 \times 0.05 / 100,000 \\ &= 843 \text{ therms} \end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-SBAC-V01-160601

REVIEW DEADLINE: 1/1/2022

⁵⁶⁴ A conservative estimate considering only setback savings, with DOE estimates as much as 1% per degree per 8 hours of setback.
<http://energy.gov/energysaver/thermostats>

4.4.37 Unitary HVAC Condensing Furnace

DESCRIPTION

Condensing furnaces recover energy in combustion exhaust flue gasses that would otherwise simply be vented to the atmosphere, making them more efficient than non-condensing furnaces. This measure applies to a constant volume (CV), dedicated outside air system (DOAS), make-up air system (MUAS), or any unitary HVAC system that is utilizing an indirect gas fired process to heat 100% OA to provide ventilation or make-up air to commercial and industrial (C&I) building spaces. The unitary package must contain an indirect gas-fired, warm air furnace section, but the unitary package can be with or without an electric air conditioning section. The unitary package can be either a single package or split system that is applied indoors (non-weatherized) or outdoors (weatherized).

This measure excludes demand control ventilation, condensing unit heaters, and high efficiency (condensing) furnaces with annual fuel utilization efficiency (AFUE) ratings (for furnaces with less than 225,000 Btu/hr input capacity), which are covered by other measures for the C&I sector in the Technical Reference Manual (TRM)⁵⁶⁵.

This measure was developed to be applicable to the following program types: TOS, NC. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure, the efficient unitary equipment must contain a condensing, warm air furnace with a natural gas thermal efficiency (TE) rating of 90% or higher, or alternatively, the unitary package must have equipment nameplate information for natural gas that identifies a heating output and heating input rating that has an output over input ratio of 0.90 or higher. These ratings must be certified by a recognized testing laboratory in accordance with American National Standards Institute (ANSI) Standard Z21.47 for Gas-Fired Central Furnaces⁵⁶⁶. The furnace must be vented and condensate disposed of in accordance with the equipment manufacturer installation instructions and applicable codes.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is expected to be unitary equipment that contains a non-condensing, warm air furnace with a natural gas thermal efficiency (TE) rating of 80%, or alternatively, the unitary package will have equipment nameplate information for natural gas that identifies a heating output and heating input rating that has an output over input ratio of 0.80. These ratings must be certified by a recognized testing laboratory in accordance with American National Standards Institute (ANSI) Standard Z21.47 for Gas-Fired Central Furnaces.

Note the current Department of Energy (DOE) federal minimum efficiency standard is 80% for 225,000 Btu/hr and higher input capacity furnaces per the Energy Conservation Standard for Commercial Warm Air Furnaces⁵⁶⁷. In the American Society of Heating, Refrigeration and Air-Conditioning Engineers (ASHRAE) Standard 90.1 Energy Standard for Buildings Except Low-Rise Residential Buildings⁵⁶⁸ that minimum TE requirement is extended below 225,000 Btu/hr input capacity to require all commercial warm air furnaces and combination warm air furnace/air conditioning units to meet the minimum 80% TE.

⁵⁶⁵ Illinois Statewide Technical Reference Manual (TRM), Version 4.0 (effective June 1, 2015), 2015. <http://www.ilsag.info/technical-reference-manual.html> (Accessed September 25, 2015).

⁵⁶⁶ American National Standards Institute (ANSI), ANSI Z21.47 Standard for Central Gas-Fired Central Furnaces, 2012. <http://www.techstreet.com/products/1837013#product> (Accessed September 25, 2015).

⁵⁶⁷ Department of Energy (DOE), Commercial Warm Air Furnace Standard DOE 10 CFR, Part 431, Subpart D – Commercial Warm Air Furnaces, 2004. <https://www.law.cornell.edu/cfr/text/10/part-431/subpart-D> (Accessed September 25, 2015).

⁵⁶⁸ American Society of Heating, Refrigeration and Air-Conditioning Engineers (ASHRAE), ASHRAE Standard 90.1 Energy Standard for Buildings Except Low-Rise Residential Buildings, 2013. <https://www.ashrae.org/resources--publications/bookstore/standard-90-1> (Accessed September 25, 2015).

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years, which is consistent with the established TRM measure life for single-package and split system unitary air conditioners, since in colder climates these unitary packages typically contain a gas-fired, warm air furnace section, with an electric air conditioning section.

DEEMED MEASURE COST

The actual incremental equipment and installation costs should be used, if available. If not, the incremental cost of \$5.42 per 1000 Btu/hr of output capacity should be used for the condensing furnace equipment (as part of a unitary package) and its installation (including the combustion condensate drainage and disposal system). This incremental cost is from the DOE Technical Support Document for the Notice of Proposed Rulemaking (NPR) for the Commercial Warm Air Furnace Standard⁵⁶⁹. Per the DOE documentation, it is based on their representative 250,000 Btu/hr input capacity furnace at a 92% TE.

LOADSHAPE

Loadshape C23 - Commercial Ventilation

COINCIDENCE FACTOR

The coincidence factor is assumed to be 1.0 – that is, building ventilation will always be provided during peak periods.

Algorithm

CALCULATION OF SAVINGS

The following methodology provides formulas for estimating gas heating savings associated with condensing furnaces in unitary HVAC packages when applied as a CV, DOAS, MUAS, or any RTU that is indirectly heating 100% outside air (OA). These types of HVAC systems typically run continuously during the HVAC operating schedule to provide building ventilation and maintain indoor air quality or to compensate for exhaust and maintain neutral or slightly positive building pressurization. The algorithm estimates the gas use reduction resulting from utilizing condensing heating of 90% or higher thermal efficiency (TE) in place of the federal minimum TE of 80% (or other user defined baseline TE) for commercial warm air furnaces.

The methodology provides a representative group of operating schedules for the market sector applications highlighted earlier based on DOE commercial reference building models⁵⁷⁰. Heating loads during the operating schedule are determined based on hourly differences between a range of supply air (SA) heated to temperatures and the OA temperature using Typical Meteorological Year (TMY3)⁵⁷¹ weather data. These hourly heating loads are generated for all hours when the OA temperature is below the base temperature of 55 °F for heating in C&I settings per the TRM. To accommodate the variability in heating base temperatures in C&I settings, these hourly heating loads are also generated for base temperatures of 45 °F and 65 °F for heating. The hourly heating loads are then summed for the entire year. The annual heating loads are calculated in this manner for the climate zone 2 weather station (Chicago O'Hare Airport), which is then normalized to its National Climatic Data Center (NCDC)⁵⁷² 30 year (1981-2010) weather average by multiplying by the heating degree day (HDD) ratio of the NCDC/TRM HDD55 over the TMY3 HDD55 (HDD at base temperature of 55 °F), and likewise for the annual heating loads for HDD45 (HDD at

⁵⁶⁹ Department of Energy (DOE), Rulemaking for Commercial Warm Air Furnace Standard, Technical Support Document 2015. https://www1.eere.energy.gov/buildings/appliance_standards/rulemaking.aspx/ruleid/70 (Accessed September 25, 2015).

⁵⁷⁰ Department of Energy (DOE) National Renewable Energy Laboratory, Commercial Reference Building Models of the National Building Stock, 2011. <http://www.nrel.gov/docs/fy11osti/46861.pdf> (Accessed September 25, 2015).

⁵⁷¹ Department of Energy (DOE) National Renewable Energy Laboratory, Users Manual for TMY3 Data Sets, 2008. <http://www.nrel.gov/docs/fy08osti/43156.pdf> (Accessed September 25, 2015).

⁵⁷² National Climatic Data Center, 1981-2010 Climate Normals, 2015. <https://www.ncdc.noaa.gov/data-access/land-based-station-data/land-based-datasets/climate-normals/1981-2010-normals-data> (Accessed November 4, 2015).

base temperature of 45 °F) and HDD65 (HDD at base temperature of 65 °F), using the values in Table 1 and Table 2. Since detailed hourly weather data is not available for all 5 of the TRM climate zone weather stations, the annual heating loads for the other climate zones are determined by multiplying the climate zone 2 annual heating loads by the ratio of the other climate zone NCDC HDD over the climate zone 2 NCDC HDD, using the values in Table 1.

These annual heating loads on a per unit airflow basis are then used in conjunction with the actual airflow of the 100% OA system and its condensing efficiency to calculate the gas heating savings versus the baseline (non-condensing) heating efficiency. This measure results in additional electric use by the unitary HVAC package due to the additional pressure drop of the condensing heat exchanger of the warm air furnace section.

Table 1. NCDC/TRM HDD Values for All Climate Zones

Climate Zone - Weather Station/City	NCDC 30 Year Average HDD45 ⁸	NCDC 30 Year Average HDD55 ^{1,8}	NCDC 30 Year Average HDD65 ⁸
1 - Rockford AP / Rockford	2495	4272	6569
2 - Chicago O'Hare AP / Chicago	2263	4029	6340
3 - Springfield #2 / Springfield	1812	3406	5495
4 - Belleville SIU RSCH / Belleville	1197	2515	4379
5 - Carbondale Southern IL AP / Marion	1183	2546	4477

Table 2. TMY3 HDD Values for Climate Zone 2

Climate Zone - Weather Station/City	TMY3 HDD45 ⁷	TMY3 HDD55 ⁷	TMY3 HDD65 ⁷
2 - Chicago O'Hare AP / Chicago	2422	4188	6497

ELECTRIC ENERGY SAVINGS

As noted previously, this measure results in additional SA fan electric use by the unitary HVAC system due to the additional pressure drop of the condensing heat exchanger of the warm air furnace section.

$$\Delta kWh = - (t_{FAN} * cfm * \Delta P) / (\eta_{FAN/MOTOR} * 8520)$$

Where:

t_{FAN} = annual fan runtime (hr), refer to Tables 1 through 4

cfm = airflow (cfm), use actual or rated system airflow

ΔP = incremental pressure drop (inch W.G.), assume 0.15 if actual value not known

$\eta_{FAN/MOTOR}$ = combined fan and motor efficiency, assume 0.60 if actual value not known

8520 = conversion factor (fan horsepower – HP – calculation constant of 6356 for standard air conditions adjusted by 1 HP = 0.746 kW, or 6356/ 0.746 = 8520 for this kW calculation)

EXAMPLE:

For a “big box” retail store operating 24 hours a day and 7 days a week (8760 hours per year) with a 5000 cfm DOAS that has an incremental pressure drop of 0.15 inch W.G. and a combined fan and motor efficiency of 0.6 has annual kWh savings of:

$$\begin{aligned}\Delta \text{kWh} &= - (t_{\text{FAN}} * \text{cfm} * \Delta P) / (\eta_{\text{FAN/MOTOR}} * 8520) \\ &= - (8760 * 5000 * 0.15) / (0.6 * 8520) \\ &= - 1285 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

The additional SA fan electric use by the unitary HVAC system will typically result in a modest electric demand increase.

$$\Delta \text{kW} = (\Delta \text{kWh} / t_{\text{FAN}}) * \text{CF}$$

Where:

$$\text{CF} = 1.0$$

EXAMPLE:

Continuing the previous example:

$$\begin{aligned}\Delta \text{kW} &= (\Delta \text{kWh} / t_{\text{FAN}}) * \text{CF} \\ &= (- 1285 / 8760) * 1.0 \\ &= - 0.15 \text{ kW}\end{aligned}$$

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms} = [Q_{\text{OA}} * \text{cfm} * (1/TE_{\text{NC}} - 1/TE_{\text{C}})] / 100,000$$

Where:

$$Q_{\text{OA}} = \text{annual outside air (OA) heating load per cfm of OA (Btu/cfm)}$$

First, select the most representative operating schedule for the application from among the four (4) scenarios listed below and its set of three (3) applicable tables. Second, select the table in that set with the most representative HDD base temperature – the base temperature for OA below which heating is required. If that base temperature is not readily determined, select the TRM default base temperature of 55 °F (HDD55) for heating in C&I settings. Third, select the climate zone within that table. Fourth, select an appropriate heated to supply air (SA) temperature within that table. Use the resulting Q_{OA} value, with linear interpolation allowed between SA temperatures.

The four (4) scenarios available are indicative of the following building applications and operating schedules:

1. 24 hour a day and 7 day a week (24/7) operation, with HVAC operating schedule of 8760 hours per year, typical of large retail stores with DOAS, hotel/multifamily buildings with corridor MUAS, and healthcare facilities with DOAS. Use Table 3 through Table 5.
2. 6:00 AM to 1:00 AM every day operation, with HVAC operating schedule of 7300 hours per year, typical of full service and quick service restaurants with kitchen MUAS. Use Table 6 through Table 8.
3. 7:00 AM to 9:00 PM Monday-Friday, 7:00 AM to 10:00 PM Saturday, and 9:00 AM to 7:00 PM Sunday operations, with HVAC operating schedule of 5266 hours per year, typical of non-24/7 retail stores with DOAS. Use Table 9 through Table 11.
4. 7:00 AM to 9:00 PM Monday-Friday operation, with HVAC operating schedule of 3911 hours per year, typical of school buildings with DOAS. Use Table 12 through Table 14.

TE_{NC} = non-condensing thermal efficiency (TE), use federal minimum TE of 80% (0.80) or actual TE if known

TE_C = condensing thermal efficiency (TE), use actual TE or if unknown assume 90% (0.90)

100,000 = conversion factor (1 therm = 100,000 Btu)

EXAMPLE:

Continuing the previous example, for a climate zone 2 (Chicago O'Hare AP / Chicago) application using a 90% TE condensing DOAS with a supply air temperature from the DOAS of 95 °F:

$$\begin{aligned}\Delta\text{Therms} &= [Q_{OA} * \text{cfm} * (1/TE_{NC} - 1/TE_C)] / 100,000 \\ &= 303,268 * 5,000 * (1/0.80 - 1/0.90) / 100,000 \\ &= 2,106 \text{ therms}\end{aligned}$$

8760 Hour Annual Operation Scenario

Table 3. 8760 Hour Annual Operation Scenario for HDD45

Supply Air Fan Runtime = 8760 Hours	Q _{oa} (Annual Btu/cfm) At Supply Air Temperature Of			
Climate Zone - Weather Station/City	75°F	85°F	95°F	105°F
1 - Rockford AP / Rockford	189,343	230,897	272,451	314,004
2 - Chicago O'Hare AP / Chicago	171,737	209,427	247,116	284,806
3 - Springfield #2 / Springfield	137,511	167,689	197,868	228,046
4 - Belleville SIU RSCH / Belleville	90,839	110,775	130,711	150,647
5 - Carbondale Southern IL AP / Marion	89,777	109,479	129,182	148,885

Table 4. 8760 Hour Annual Operation Scenario for HDD55

Supply Air Fan Runtime = 8760 Hours	Q _{oa} (Annual Btu/cfm) At Supply Air Temperature Of			
Climate Zone - Weather Station/City	75°F	85°F	95°F	105°F
1 - Rockford AP / Rockford	216,145	268,852	321,559	374,266
2 - Chicago O'Hare AP / Chicago	203,850	253,559	303,268	352,977
3 - Springfield #2 / Springfield	172,329	214,351	256,374	298,397
4 - Belleville SIU RSCH / Belleville	127,248	158,278	189,307	220,337
5 - Carbondale Southern IL AP / Marion	128,817	160,229	191,641	223,053

Table 5. 8760 Hour Annual Operation Scenario for HDD65

Supply Air Fan Runtime = 8760 Hours	Q _{oa} (Annual Btu/cfm) At Supply Air Temperature Of			
Climate Zone - Weather Station/City	75°F	85°F	95°F	105°F
1 - Rockford AP / Rockford	239,158	308,050	376,942	445,834
2 - Chicago O'Hare AP / Chicago	230,820	297,311	363,802	430,292
3 - Springfield #2 / Springfield	200,056	257,685	315,314	372,943
4 - Belleville SIU RSCH / Belleville	159,426	205,351	251,276	297,200
5 - Carbondale Southern IL AP / Marion	162,994	209,947	256,899	303,852

7300 Hour Annual Operation Scenario**Table 6. 7300 Hour Annual Operation Scenario for HDD45**

Supply Air Fan Runtime = 7300 Hours	Q _{oa} (Annual Btu/cfm) At Supply Air Temperature Of			
Climate Zone - Weather Station/City	75°F	85°F	95°F	105°F
1 - Rockford AP / Rockford	151,914	185,369	218,823	252,278
2 - Chicago O'Hare AP / Chicago	137,788	168,132	198,476	228,819
3 - Springfield #2 / Springfield	110,328	134,624	158,921	183,217
4 - Belleville SIU RSCH / Belleville	72,882	88,932	104,982	121,033
5 - Carbondale Southern IL AP / Marion	72,030	87,892	103,755	119,617

Table 7. 7300 Hour Annual Operation Scenario for HDD55

Supply Air Fan Runtime = 7300 Hours	Q _{oa} (Annual Btu/cfm) At Supply Air Temperature Of			
Climate Zone - Weather Station/City	75°F	85°F	95°F	105°F
1 - Rockford AP / Rockford	173,511	215,950	258,389	300,828
2 - Chicago O'Hare AP / Chicago	163,641	203,666	243,691	283,716
3 - Springfield #2 / Springfield	138,338	172,174	206,010	239,846
4 - Belleville SIU RSCH / Belleville	102,149	127,133	152,118	177,103
5 - Carbondale Southern IL AP / Marion	103,408	128,701	153,993	179,286

Table 8. 7300 Hour Annual Operation Scenario for HDD65

Supply Air Fan Runtime = 7300 Hours	Q _{oa} (Annual Btu/cfm) At Supply Air Temperature Of			
Climate Zone - Weather Station/City	75°F	85°F	95°F	105°F
1 - Rockford AP / Rockford	191,803	247,046	302,288	357,531
2 - Chicago O'Hare AP / Chicago	185,117	238,434	291,750	345,067
3 - Springfield #2 / Springfield	160,444	206,655	252,866	299,076
4 - Belleville SIU RSCH / Belleville	127,859	164,685	201,510	238,336
5 - Carbondale Southern IL AP / Marion	130,720	168,370	206,020	243,670

5266 Hour Annual Operation Scenario**Table 9. 5266 Hour Annual Operation Scenario for HDD45**

Supply Air Fan Runtime = 5266 Hours	Q _{oa} (Annual Btu/cfm) At Supply Air Temperature Of			
Climate Zone - Weather Station/City	75°F	85°F	95°F	105°F
1 - Rockford AP / Rockford	104,175	127,350	150,524	173,699
2 - Chicago O'Hare AP / Chicago	94,488	115,508	136,527	157,547
3 - Springfield #2 / Springfield	75,657	92,488	109,319	126,149
4 - Belleville SIU RSCH / Belleville	49,979	61,097	72,215	83,334
5 - Carbondale Southern IL AP / Marion	49,394	60,383	71,371	82,359

Table 10. 5266 Hour Annual Operation Scenario for HDD55

Supply Air Fan Runtime = 5266 Hours	Q _{oa} (Annual Btu/cfm) At Supply Air Temperature Of			
Climate Zone - Weather Station/City	75°F	85°F	95°F	105°F
1 - Rockford AP / Rockford	118,320	147,406	176,492	205,578
2 - Chicago O'Hare AP / Chicago	111,590	139,021	166,452	193,884
3 - Springfield #2 / Springfield	94,335	117,524	140,714	163,904
4 - Belleville SIU RSCH / Belleville	69,657	86,780	103,904	121,027
5 - Carbondale Southern IL AP / Marion	70,516	87,850	105,184	122,519

Table 11. 5266 Hour Annual Operation Scenario for HDD65

Supply Air Fan Runtime = 5266 Hours	Q _{oa} (Annual Btu/cfm) At Supply Air Temperature Of			
Climate Zone - Weather Station/City	75°F	85°F	95°F	105°F
1 - Rockford AP / Rockford	130,903	168,718	206,532	244,347
2 - Chicago O'Hare AP / Chicago	126,339	162,836	199,333	235,829
3 - Springfield #2 / Springfield	109,501	141,133	172,765	204,398
4 - Belleville SIU RSCH / Belleville	87,262	112,470	137,678	162,886
5 - Carbondale Southern IL AP / Marion	89,215	114,987	140,759	166,531

3911 Hour Annual Operation Scenario

Table 12. 3911 Hour Annual Operation Scenario for HDD45

Supply Air Fan Runtime = 3911 Hours	Q _{oa} (Annual Btu/cfm) At Supply Air Temperature Of			
Climate Zone - Weather Station/City	75°F	85°F	95°F	105°F
1 - Rockford AP / Rockford	75,029	91,729	108,428	125,128
2 - Chicago O'Hare AP / Chicago	68,053	83,199	98,346	113,492
3 - Springfield #2 / Springfield	54,490	66,618	78,746	90,874
4 - Belleville SIU RSCH / Belleville	35,996	44,008	52,019	60,031
5 - Carbondale Southern IL AP / Marion	35,575	43,493	51,411	59,329

Table 13. 3911 Hour Annual Operation Scenario for HDD55

Supply Air Fan Runtime = 3911 Hours	Q _{oa} (Annual Btu/cfm) At Supply Air Temperature Of			
Climate Zone - Weather Station/City	75°F	85°F	95°F	105°F
1 - Rockford AP / Rockford	85,672	106,825	127,979	149,132
2 - Chicago O'Hare AP / Chicago	80,799	100,749	120,699	140,649
3 - Springfield #2 / Springfield	68,305	85,170	102,035	118,901
4 - Belleville SIU RSCH / Belleville	50,436	62,890	75,343	87,797
5 - Carbondale Southern IL AP / Marion	51,058	63,665	76,272	88,879

Table 14. 3911 Hour Annual Operation Scenario for HDD65

Supply Air Fan Runtime = 3911 Hours	Q _{oa} (Annual Btu/cfm) At Supply Air Temperature Of			
Climate Zone - Weather Station/City	75°F	85°F	95°F	105°F
1 - Rockford AP / Rockford	95,460	123,294	151,128	178,963
2 - Chicago O'Hare AP / Chicago	92,132	118,996	145,860	172,724
3 - Springfield #2 / Springfield	79,853	103,136	126,420	149,703
4 - Belleville SIU RSCH / Belleville	63,635	82,190	100,745	119,299
5 - Carbondale Southern IL AP / Marion	65,059	84,029	102,999	121,969

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

The actual incremental annual maintenance costs should be used, if available. If not, the incremental cost of \$0.05 per 1000 Btu/hr of output capacity should be used for maintaining the combustion condensate disposal system yearly. This incremental cost is from the DOE Technical Support Document for the Notice of Proposed Rulemaking (NOPR) for the Commercial Warm Air Furnace Standard⁶. Per the DOE documentation, it is based on their representative 250,000 Btu/hr input capacity furnace at a 92% TE.

MEASURE CODE: CI-HVC-DSFN-V01-160601

REVIEW DEADLINE: 1/1/2019

4.4.38 Covers and Gap Sealers for Room Air Conditioners

DESCRIPTION

Room air conditioners (window ACs, through-the-wall or sleeve ACs, PTACs or PTHPs) constitute a permanent or semi-permanent penetration through the building's envelope. These units are often poorly installed, resulting in gaps that act like air leakage pathways through the building's envelope. The uncontrolled movement of air across the gaps in the envelope (infiltration) increases the building's winter heating requirements and reduces its overall energy performance.

The heat loss and infiltration can be reduced by installing a rigid or flexible insulated cover on the inside of a room AC. These covers should be maintained by building staff and should remain installed through the heating season. Simple uninsulated cloth covers with no sealing at edges do not qualify for this measure.

There are several types of AC covers available that may be eligible for this measure:

1. If the room AC is left in the window or sleeve, a rigid cover that covers the indoor side of the AC unit with foam gaskets to seal the edges may be installed.
2. If the room AC is absent or is removed during the heating months, a rigid cover that fits inside the sleeve with foam gaskets along the edges for proper air sealing may be installed.
3. Flexible covers that are well insulated and perfectly cover the indoor side of the AC unit may also be eligible for this measure.

This measure was developed to be applicable to the following program types: RF, DI. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The installed equipment is a rigid cover that fits inside the empty sleeve or completely covers the indoor side of a window AC unit, with foam gaskets sealing the edges. A flexible insulated cover that perfectly covers the indoor side of the unit and seals gaps may also be installed. Covers should remain installed throughout the winter heating season.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is a room AC (window AC, through-the-wall or sleeve AC, PTAC or PTHP) that is poorly installed with gaps around the edges and does not use AC covers or gap sealers during the winter heating months.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The estimated useful life of typical AC covers is 5 years⁵⁷³.

DEEMED MEASURE COST

The measure cost is the full cost of installing AC covers. Actual installation costs (material and labor) should be used if available. In actual costs are unknown, assume material cost⁵⁷⁴ of \$24 (flexible covers) up to \$119, depending on size of the AC unit. The install time per unit is 15 to 30 minutes at assumed labor rate of \$20/hour.

LOADSHAPE

Loadshape C04 – Commercial Electric Heating

COINCIDENCE FACTOR

N/A

⁵⁷³ New York Standard Approach for Estimating Energy Savings from Energy Efficiency Programs V4, April 2016 (New York TRM).

⁵⁷⁴ Cost estimates from customer invoices and vendors. Material costs can be lower for bulk orders.

Algorithm

CALCULATION OF ENERGY SAVINGS**ELECTRIC ENERGY SAVINGS**

If the building is electrically heated, electric energy savings are calculated as follows:

$$\Delta kWh = (Q_{infiltration} * 1.08 * (T_{OA} - T_{SA}) * EFLH_{heat}) / (3,412 * COP)$$

Where:

$$Q_{infiltration} = \text{Air infiltration (CFM) due to poor installation of window or through-the-wall AC}^{575}$$

$$= ELA * 0.000645 * (f_s^2 * (T_{OA} - T_{SA}) + f_w^2 * U^2)^{1/2} * 2118.88$$

Where:

ELA = Effective Leakage Area (sq. in.)

= Can be collected on site; if unknown, assume 6 sq. in.⁵⁷⁶

0.000645= Converts square inches to square meters

f_s = Stack Coefficient

$$= 1/3 * (9.81 * \text{Height} * 0.3048) / (T_{OA})^{0.5}$$

f_w = Wind Coefficient

$$= A * B * (\text{Height} * 0.3048) / (10)^C$$

Where:

9.81 = Acceleration due to gravity (m/s²)

Height = Height of the location of the leakage area in feet

= Assume 8 ft per floor

T_{OA} = Average Outside Air Temperature during heating period⁵⁷⁷. Use values from table below, based on facility location⁵⁷⁸. This figure must be in Kelvin to determine Stack Coefficient (f_s) and infiltration ($Q_{infiltration}$), but in Fahrenheit to determine energy savings (ΔkWh , $\Delta Therms$).

Zone	T_{OA} (°F)	T_{OA} (K)
Zone 1 (Rockford)	31.63	272.94
Zone 2 (Chicago)	33.99	274.26
Zone 3 (Springfield)	34.58	274.58
Zone 4 (Belleville)	36.24	275.51
Zone 5 (Marion)	39.07	277.08

⁵⁷⁵ Infiltration equation and values for stack and wind coefficient equations from “The Use of Blower Door Data.” Max Sherman, 1998. The equation is adjusted for wall leakage area (i.e. no ceiling or floor leakage).

⁵⁷⁶ Average effective leakage area for multi-family building AC units from “There are Holes in Our Walls.” Prepared for Urban Green Council by Steven Winter Associates, April 2011.

⁵⁷⁷ “Heating Period” is defined as hours when the TMY3 dry bulb temperature is less than 55°F (balance point)

⁵⁷⁸ Based on NREL’s Typical Meteorological Year 3 (TMY3) data for different weather stations.

A, B and C = Constants based on the facility site's shielding and terrain parameters. Use values from the tables below⁵⁷⁹.

Shielding Class	Shielding Type	Shielding Description	A
1	None	No obstructions or local shielding whatsoever (i.e. isolated building)	0.324
2	Light	Light local shielding with few obstructions (e.g. A few trees or a shed in the vicinity)	0.285
3	Moderate	Moderate local shielding; some obstructions within two house heights (e.g. Thick hedge fence on fence and nearby building)	0.24
4	Heavy	Heavy shielding; obstructions around most of perimeter buildings or trees within five building heights in most directions (e.g. Well developed/dense tract house)	0.185
5	Very Heavy	Very heavy shielding, large obstruction surrounding perimeter within two house heights (e.g. Typical downtown area)	0.102

Terrain Class	Terrain Type	Terrain Description	B	C
1	None	Ocean or other body of water with at least 5 km of unrestricted space	1.3	0.1
2	Light	Flat terrain with some isolated obstacles (e.g. Buildings or trees well separated from each other)	1	0.15
3	Moderate	Rural areas with low buildings, trees etc.	0.85	0.2
4	Heavy	Urban, industrial or forest areas	0.67	0.25
5	Very Heavy	Center of large city (e.g. Manhattan)	0.47	0.35

0.3048 = Converts feet to meters

T_{SA} = Average Indoor Air Temperature during heating period. This figure will need to be in Kelvin to calculate infiltration ($Q_{infiltration}$) and Fahrenheit to calculate energy savings (ΔkWh , $\Delta Therms$).

= Collected on site. If unknown, assume 72°F (295 K). If known, convert °F to K by using the following equation: $K = (°F + 459.67) * (5/9)$.

U = Average Wind Velocity (m/s) during heating period. Use table below, based on facility location⁵⁸⁰.

Zone	U (m/s)
Zone 1 (Rockford)	4.50
Zone 2 (Chicago)	4.67
Zone 3 (Springfield)	4.60
Zone 4 (Belleville)	3.92
Zone 5 (Marion)	3.07

2118.88 = Converts m^3/s to CFM

⁵⁷⁹ Shielding and terrain class descriptions and constants from "The Use of Blower Door Data." Max Sherman, 1998" and "Wind and Infiltration Interaction for Small Buildings." MH Sherman and DT Grimsrud, Lawrence Berkley Laboratory, 1982.

⁵⁸⁰ Based on TMY3 data, see "Covers for Room AC_11092016.xls" for more information.

- 1.08 = Sensible heat transfer constant (Btu/hr.CFM.°F)
- EFLH_{heat} = Equivalent Full Load Hours for heating from section 4.4 HVAC End Use⁵⁸¹
- 3,412 = Converts Btus to kWh
- COP = Coefficient of Performance of the heating unit
= Collected on site. If unknown assume 2.6 for PTHP⁵⁸²

Deemed per-unit savings for the Multi-Family Building type for Shielding Class 3 and Terrain Class 3 are as follows:

Multi-Family - Electric Savings per Unit (kWh/unit)						
Floor	Height	Rockford	Chicago	Springfield	Belleville	Marion
1	8	55.18	53.16	45.70	31.09	25.67
2	16	68.19	65.31	56.17	38.72	32.66
3	24	77.92	74.34	63.96	44.45	37.97
4	32	86.04	81.85	70.44	49.25	42.44
5	40	93.15	88.42	76.11	53.46	46.37
6	48	99.56	94.34	81.22	57.26	49.93
7	56	105.44	99.76	85.90	60.75	53.20
8	64	110.91	104.80	90.25	63.99	56.24
9	72	116.04	109.53	94.33	67.04	59.11
10	80	120.89	114.00	98.19	69.92	61.81
12	96	129.92	122.31	105.36	75.29	66.85
14	112	138.21	129.94	111.95	80.22	71.49
16	128	145.93	137.04	118.08	84.81	75.82
18	144	153.19	143.72	123.84	89.13	79.88
20	160	160.05	150.03	129.29	93.21	83.72
22	176	166.59	156.03	134.47	97.10	87.38
24	192	172.83	161.77	139.42	100.82	90.88
26	208	178.82	167.28	144.18	104.38	94.23
28	224	184.58	172.57	148.75	107.81	97.46
30	240	190.15	177.69	153.17	111.12	100.58

⁵⁸¹ Although in theory the hours should be all hours that infiltration is expected (i.e. all hours <55F), the IL TAC has agreed to use the Equivalent Full Load Hours to keep the savings at a more conservative level.

⁵⁸² From IECC 2012 Minimum Efficiency Requirements. For a 1 ton PTHP, COP = 2.9 – (0.026 * 12,000/1,000).

EXAMPLE

A mid-rise multi-family building located in the moderate terrain class and shielding class of Chicago, has 16 rooms on the 10th floor (80 feet high) with PTHPs that get covered with a cover and foam gasket during the heating months. The indoor temperature during the heating months is maintained at 74°F. The air infiltration and the related energy savings from the AC covers and seals are calculated as follows -

For Shielding Class 3 and Terrain Class 3,

A = 0.24, B = 0.85 and C = 0.2

Therefore,

$$f_s = 1/3 * (9.81 \text{ m/s}^2 * 80 \text{ ft} * 0.3048 \text{ m/ft} / 274.26 \text{ K})^{0.5} = 0.3 \text{ m/K}^{1/2} \cdot \text{s}$$

$$f_w = 0.24 * 0.85 * (80 \text{ ft} * 0.3048 \text{ m/ft} / 10 \text{ m})^{0.2} = 0.24$$

Total effective leakage area (ELA) = 16 units * 6 sq. in. = 96 sq. in.

$$Q_{\text{infiltration}} = \text{ELA} * 0.000645 * (f_s^2 * (T_{\text{OA}} - T_{\text{SA}}) + f_w^2 * U^2)^{1/2} * 2118.88$$

$$= 96 * 0.000645 * (0.3^2 * (296.48 \text{ K} - 274.26 \text{ K}) + 0.24^2 * 4.67^2)^{1/2} * 2118.88$$

$$= 237 \text{ CFM}$$

$$\Delta \text{kWh} = (237 * 1.08 \text{ Btu/hr.CFM.}^\circ\text{F} * (74^\circ\text{F} - 33.99^\circ\text{F}) * 1,685) / (3,412 \text{ Btu/kWh} * 2.6)$$

$$= 1,945 \text{ kWh}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

As the savings occur during the winter season (non-peak), there are no demand savings associated with this measure.

NATURAL GAS SAVINGS

If the building is heated with gas, the natural gas savings are calculated as follows:

$$\Delta \text{Therms} = (Q_{\text{infiltration}} * 1.08 \text{ Btu/hr.CFM.}^\circ\text{F} * (T_{\text{OA}} - T_{\text{SA}}) * \text{EFLH}_{\text{heat}}) / (100,000 \text{ Btu/therm} * \eta)$$

Where,

η = Efficiency of heating equipment.

= Collected on site. If unknown, assume 80%⁵⁸³.

100,000 = Converts Btus to therms

Other factors as defined above

Deemed per-unit savings per unit for the Multi-Family Building type for Shielding Class 3 and Terrain Class 3 are as follows:

Multi-Family - Gas Savings per Unit (Therms/Unit)						
Floor	Height	Rockford	Chicago	Springfield	Belleville	Marion
1	8	6.12	5.90	5.07	3.45	2.85
2	16	7.56	7.24	6.23	4.29	3.62
3	24	8.64	8.24	7.09	4.93	4.21
4	32	9.54	9.08	7.81	5.46	4.71
5	40	10.33	9.81	8.44	5.93	5.14

⁵⁸³ Energy Independence and Security Act of 2007 – averaged for hot water and steam boilers.

Multi-Family - Gas Savings per Unit (Therms/Unit)						
Floor	Height	Rockford	Chicago	Springfield	Belleville	Marion
6	48	11.04	10.46	9.01	6.35	5.54
7	56	11.69	11.06	9.53	6.74	5.90
8	64	12.30	11.62	10.01	7.10	6.24
9	72	12.87	12.15	10.46	7.43	6.55
10	80	13.41	12.64	10.89	7.75	6.85
12	96	14.41	13.56	11.68	8.35	7.41
14	112	15.33	14.41	12.41	8.90	7.93
16	128	16.18	15.20	13.09	9.40	8.41
18	144	16.99	15.94	13.73	9.88	8.86
20	160	17.75	16.64	14.34	10.34	9.28
22	176	18.47	17.30	14.91	10.77	9.69
24	192	19.16	17.94	15.46	11.18	10.08
26	208	19.83	18.55	15.99	11.57	10.45
28	224	20.47	19.14	16.50	11.96	10.81
30	240	21.09	19.70	16.98	12.32	11.15

EXAMPLE

A gas-heated mid-rise multi-family building located in the moderate terrain class and shielding class of Chicago, has 16 rooms on the 10th floor (80 feet high) with room air conditioners that get covered with an AC cover and foam gasket during the heating months. The indoor temperature during the heating months is maintained at 74°F. The air infiltration and the related therm savings from the AC covers and seals are calculated as follows

For Shielding Class 3 and Terrain Class 3,

A = 0.24, B = 0.85 and C = 0.2

Therefore,

$$f_s = 1/3 * (9.81 \text{ m/s}^2 * 80 \text{ ft} * 0.3048 \text{ m/ft} / 274.26 \text{ K})^{0.5} = 0.3 \text{ m/K}^{1/2} \cdot \text{s}$$

$$f_w = 0.24 * 0.85 * (80 \text{ ft} * 0.3048 \text{ m/ft} / 10 \text{ m})^{0.2} = 0.24$$

Total effective leakage area (ELA) = 16 units * 6 sq.in = 96 sq. in

$$\begin{aligned} Q_{\text{infiltration}} &= \text{ELA} * 0.000645 * (f_s^2 * (T_{\text{OA}} - T_{\text{SA}}) + f_w^2 * U^2)^{1/2} * 2118.88 \\ &= 96 * 0.000645 * (0.3^2 * (296.48 \text{ K} - 274.26 \text{ K}) + 0.24^2 * 4.67^2)^{1/2} * 2118.88 \\ &= 237 \text{ CFM} \end{aligned}$$

$$\begin{aligned} \Delta \text{Therms} &= (237 * 1.08 \text{ Btu/hr.CFM.}^\circ\text{F} * (74^\circ\text{F} - 33.99^\circ\text{F}) * 1,685) / (100,000 \text{ Btu/therm} * 80\%) \\ &= 216 \text{ therms} \end{aligned}$$

WATER AND OTHER NON-ENERGY IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-CRAC-V01-180101

REVIEW DEADLINE: 1/1/2023

4.4.39 High Temperature Heating and Ventilation (HTHV) Direct Fired Heater

DESCRIPTION

This measure applies to 100% outside air, high temperature heating and ventilation (HTHV) direct fired gas heaters. These units replace unit heaters (indirect gas fired or steam coil) or rooftop units in warehouses which suffer from extreme temperature stratification, minimal controls and reduced heating efficiencies.

Warehouses have high ceilings (~30 ft high), and suffer from stratification of air. The warm air rises and remains near the roof, which keeps the thermostat from reaching its desired setpoint. This increases the run hours of the heating unit and causes discomfort among the occupants. The HTHV units have high pressure fans that direct high temperature and high velocity air towards the floor and thus help minimize temperature stratification. On average, a 30 ft high warehouse could reduce its linear stratification from 0.53°F/ft to 0.13°F/ft, thus maintaining a more uniform temperature in the room and reducing the operating hours of the heating unit.

Since the HTHV units are direct fired, they also have improved efficiencies of 92% compared to 80% for a typical indirect fired unit heater or rooftop unit. They transfer the latent heat of the flue gases into the space instead of venting it out.

This measure only applies to high ceiling warehouses that do not have any other destratification technologies installed (i.e. destratification fans, air rotation units etc.). New HTHV units must be the warehouse's primary heat source.

This measure was developed to be applicable to the following program types: RF, TOS, NC. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment must be a 100% outside air, HTHV direct fired gas heater, with a discharge temperature greater than or equal to 150°F, a temperature rise greater than or equal to 140°F, and an efficiency exceeding 92%.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment must be an indirect fired gas or steam unit heater or a rooftop unit used as the primary space heating source. Warehouses with existing destratification technologies (high volume, low speed fans or air turnover units) do not qualify for this measure.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 15 years⁵⁸⁴.

DEEMED MEASURE COST

The measure cost should be based on a contractor's evaluation of the project scope and may vary significantly on a project to project basis. If unknown, for early replacement or retrofit projects, assume \$14.50/MBtu/hr (material cost for an HTHV unit) or \$26/MBTUh (sum of material and installation cost)⁵⁸⁵.

The incremental measure cost, assuming a baseline of standard efficiency unit heaters, is \$7.43/MBtu/hr (material cost)⁵⁸⁶.

⁵⁸⁴ Based on "Field Demonstration of High Efficiency Gas Heaters", prepared for Better Buildings Alliance, US. DOE, Jim Young, Navigant Consulting, 2014.

⁵⁸⁵ Average costs from CLEAResult's evaluation of 9 different projects in the Chicagoland area.

⁵⁸⁶ Based on data collected in "Field Demonstration of High Efficiency Gas Heaters", prepared for Better Buildings Alliance, US. DOE, Jim Young, Navigant Consulting, 2014.

LOADSHAPE

Loadshape C04: Commercial Electric Heating

COINCIDENCE FACTOR

Assumed to be 0.

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

HTHV units may increase the facility's electric energy consumption due to high pressure motors that supply air at higher velocity.

$$\Delta \text{kWh} = - \text{kWh}/\text{HDD} * \text{HDD}$$

Where:

kWh/HDD = increase in electric energy consumption due to HTHV fan motor
 $= 1.04^{587}$

HDD = heating degree days

Zone	City	HDD55 ⁵⁸⁸	ΔkWh
1	Rockford	4,272	(4,443)
2	Chicago	4,029	(4,190)
3	Springfield	3,406	(3,542)
4	Belleville	2,515	(2,616)
5	Marion	2,546	(2,648)

Although HTHV fan motors have a higher power draw, they also result in decreased heating equipment operating time, potentially offsetting some of the increase in electrical energy consumption. Therefore, if replacing heating equipment other than unit heaters, a custom evaluation may be necessary to determine if there is an increase in electrical energy consumption.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

Since HTHV units operate during the winter (non-peak) season, there are no demand savings associated with this measure.

NATURAL GAS SAVINGS

Custom calculation below, otherwise use a deemed savings factor from the table that follows.

$$\Delta \text{Therms} = (\text{FLH}_{\text{base}} * \text{Cap}_{\text{base}} / (\eta_{\text{base}} * 100)) - (\text{FLH}_{\text{eff}} * \text{Cap}_{\text{eff}} / (\eta_{\text{eff}} * 100))$$

Where:

⁵⁸⁷ Based on data collected in "Field Demonstration of High Efficiency Gas Heaters", prepared for Better Buildings Alliance, US. DOE, Jim Young, Navigant Consulting, 2014. This study replaced four standard unit heaters with HTHV units, and the electrical energy increased from 0.4 kWh/HDD to 1.44 kWh/HDD. Therefore savings are assumed to be 1.04 kWh /HDD.

⁵⁸⁸ 30-year normals from the National Climactic Data Center (NCDC), assuming base temperature 55.

$$FLH_{base} = LF_{base} * \text{Hours}$$

$$FLH_{eff} = LF_{eff} * \text{Hours}$$

Hours = Annual operating hours of the unit, calculated as total number of hours when outside air temperature is less than 55°F. This can be adjusted based on the facility's occupancy schedule.

$$LF_{base} = \text{load factor of baseline unit heater}$$

$$= (Q_{inf,base} + Q_{w,base} + Q_{r,base}) / (Cap_{base} * 100)$$

$$LF_{eff} = \text{load factor of HTHV heater}$$

$$= (Q_{inf,eff} + Q_{w,eff} + Q_{r,eff}) / (Cap_{eff} * 100)$$

Cap_{base} = existing heating unit input capacity (MBtu/hr)
= can be collected on site, or assumed to be the same as HTHV unit capacity, Cap_{eff}

Cap_{eff} = HTHV unit input capacity (MBtu/hr)
= can be collected on site or from specification sheets

η_{base} = efficiency of existing heating unit
= collected from equipment nameplate or assumed as 70% for steam unit heaters, 80% for gas fired unit heaters, and 84% for rooftop units⁵⁸⁹

η_{eff} = efficiency of HTHV unit
= collected from equipment nameplate or assumed as 92%

100 = converts MBtu to therms

See table below for savings inputs.

Parameter	Existing Unit	Proposed (Efficient) Unit
<u>Temperatures</u>		
Setpoint Temperature (°F)	T _{setpoint} = collected on site, or assumed as 65°F	
Ceiling Temperature ⁵⁹⁰ (°F)	Either collected on site when the existing unit is in operation with an infrared gun, or assumed as: T _{c,base} = T _{setpoint} + 0.53°F/ft * Height	Either collected on site when the proposed unit is in operation with an infrared gun, or assumed as: T _{c,eff} = T _{setpoint} + 2 to 4°F
Average Room Temperature (°F)	T _{r,base} = (T _{setpoint} + T _{c,base})/2	T _{r,eff} = (T _{setpoint} + T _{c,eff})/2
Outside Air Temperature (°F)	T _{OA} , from local weather data ⁵⁹¹	
<u>Heat Loads</u>		
Infiltration Load ⁵⁹² :	Q _{inf,base} = 0.04CFM/ft ² * (Wall Surface Area + Roof Surface Area) * 1.08 * (T _{r,base} - T _{OA})	Q _{inf,eff} = 0.04CFM/ft ² * (Wall Surface Area + Roof Surface Area) * 1.08 * (T _{r,eff} - T _{OA})

⁵⁸⁹ Efficiency of existing systems assumed from ASHRAE 90.1 – 2010 and manufacturer's specification sheets for various equipment. Steam unit heaters have a lower efficiency due to steam distribution losses.

⁵⁹⁰ Baseline stratification rate is based on data collected in "Field Demonstration of High Efficiency Gas Heaters", prepared for Better Buildings Alliance, US. DOE, Jim Young, Navigant Consulting, 2014. The study also verifies that the proposed ceiling temperature can be maintained within 2-4°F of the setpoint.

⁵⁹¹ Use Typical Meteorological Year (TMY3) data from NREL available here: http://rredc.nrel.gov/solar/old_data/nsrdb/1991-2005/tmy3/

⁵⁹² Typical infiltration rate assumed from Infiltration Modeling Guidelines for Commercial Building Energy Analysis, prepared for US. DOE by Pacific Northwestern National Laboratory, 2009

Parameter	Existing Unit	Proposed (Efficient) Unit
Wall Conduction Load ⁵⁹³ :	$Q_{w,base} = 1/R\text{-value}_{wall} * (\text{Wall Surface Area} * 1.08 * (T_{r,base} - T_{OA}))$ Where R-value _{wall} = the insulation value of the wall. It can be collected on site, or assumed as R-15.	$Q_{w,eff} = 1/R\text{-value}_{wall} * (\text{Wall Surface Area} * 1.08 * (T_{r,eff} - T_{OA}))$ Where R-value _{wall} = the insulation value of the wall. It can be collected on site, or assumed as R-15.
Roof Conduction Load:	$Q_{r,base} = 1/R\text{-value}_{roof} * (\text{Roof Surface Area} * 1.08 * (T_{r,base} - T_{OA}))$ Where R-value _{roof} = the insulation value of the roof. It can be collected on site, or assumed as R-20.	$Q_{r,eff} = 1/R\text{-value}_{roof} * (\text{Roof Surface Area} * 1.08 * (T_{r,eff} - T_{OA}))$ Where R-value _{roof} = the insulation value of the roof. It can be collected on site, or assumed as R-20.
Surface Areas		
Roof Surface Area:	Collected on site or assumed as: = facility area in sq.ft. If facility area is unknown, assume facility area ⁵⁹⁴ = 41.4 sq. ft./MBtu/hr * Cap _{eff}	
<u>Wall Surface Area:</u>	Collected on site or assumed as: = (Height * Length + Height * Width) * 2 Where: Length, Height and Width (feet) of the facility can be collected on site. If unknown, assume: Length = Width = (Facility Area) ^{1/2} and Height = 25 ft If facility area is unknown, assume facility area = 41.4 sq. ft./MBtu/hr * Cap _{eff}	

The default values from the table above were used to calculate the deemed savings values in the table below. Savings are provided for various rated input capacity ranges and weather stations.

Cap _{eff} (MBtu/hr)	Average Cap _{eff} (MBtu/hr)	Nearest Weather Station	ΔTherms (Baseline Equipment: Steam Fired Unit Heaters)	ΔTherms (Baseline Equipment: Gas Fired Unit Heaters)	ΔTherms (Baseline Equipment: Rooftop Units)
300 > Cap _{eff} ≥ 500	400	Rockford	3,120	1,996	1,620
500 > Cap _{eff} ≥ 900	757	Rockford	5,208	3,346	2,725
900 > Cap _{eff} ≥ 1,000	950	Rockford	6,280	4,047	3,297
1,000 > Cap _{eff} ≥ 1,400	1,200	Rockford	7,656	4,932	4,020
1,400 > Cap _{eff} ≥ 1,600	1,499	Rockford	9,249	5,966	4,872
1,600 > Cap _{eff} ≥ 2,100	1,850	Rockford	11,100	7,160	5,865
2,100 > Cap _{eff} ≥ 2,400	2,200	Rockford	12,914	8,338	6,820
Cap _{eff} ≥ 2,400	2,718	Rockford	15,547	10,084	8,236
300 > Cap _{eff} ≥ 500	400	Chicago	2,820	1,824	1,488
500 > Cap _{eff} ≥ 900	757	Chicago	4,709	3,058	2,506
900 > Cap _{eff} ≥ 1,000	950	Chicago	5,681	3,696	3,031
1,000 > Cap _{eff} ≥ 1,400	1,200	Chicago	6,924	4,512	3,696
1,400 > Cap _{eff} ≥ 1,600	1,499	Chicago	8,364	5,456	4,482
1,600 > Cap _{eff} ≥ 2,100	1,850	Chicago	10,046	6,549	5,384
2,100 > Cap _{eff} ≥ 2,400	2,200	Chicago	11,682	7,634	6,292
Cap _{eff} ≥ 2,400	2,718	Chicago	14,079	9,214	7,583
300 > Cap _{eff} ≥ 500	400	Springfield	2,452	1,588	1,300

⁵⁹³ Roof and Wall Insulation R-values are based on ASHRAE 90.1- 2010. (Jim Young 2014) (K. Gowri 2009)

⁵⁹⁴ Based on DOE's Commercial Prototype Modeled Warehouse building (in Chicago), found here:

<https://www.energycodes.gov/commercial-prototype-building-models>

Cap _{eff} (MBtu/hr)	Average Cap _{eff} (MBtu/hr)	Nearest Weather Station	ΔTherms (Baseline Equipment: Steam Fired Unit Heaters)	ΔTherms (Baseline Equipment: Gas Fired Unit Heaters)	ΔTherms (Baseline Equipment: Rooftop Units)
500 > Cap _{eff} ≥ 900	757	Springfield	4,095	2,665	2,188
900 > Cap _{eff} ≥ 1,000	950	Springfield	4,950	3,221	2,651
1,000 > Cap _{eff} ≥ 1,400	1,200	Springfield	6,024	3,936	3,240
1,400 > Cap _{eff} ≥ 1,600	1,499	Springfield	7,285	4,767	3,912
1,600 > Cap _{eff} ≥ 2,100	1,850	Springfield	8,732	5,717	4,718
2,100 > Cap _{eff} ≥ 2,400	2,200	Springfield	10,164	6,666	5,500
Cap _{eff} ≥ 2,400	2,718	Springfield	12,258	8,045	6,632
300 > Cap _{eff} ≥ 500	400	Belleville	2,456	1,604	1,320
500 > Cap _{eff} ≥ 900	757	Belleville	4,103	2,687	2,218
900 > Cap _{eff} ≥ 1,000	950	Belleville	4,950	3,249	2,689
1,000 > Cap _{eff} ≥ 1,400	1,200	Belleville	6,036	3,972	3,276
1,400 > Cap _{eff} ≥ 1,600	1,499	Belleville	7,300	4,812	3,972
1,600 > Cap _{eff} ≥ 2,100	1,850	Belleville	8,751	5,772	4,773
2,100 > Cap _{eff} ≥ 2,400	2,200	Belleville	10,186	6,732	5,566
Cap _{eff} ≥ 2,400	2,718	Belleville	12,285	8,127	6,713
300 > Cap _{eff} ≥ 500	400	Marion	2,180	1,444	1,200
500 > Cap _{eff} ≥ 900	757	Marion	3,649	2,430	2,021
900 > Cap _{eff} ≥ 1,000	950	Marion	4,408	2,936	2,442
1,000 > Cap _{eff} ≥ 1,400	1,200	Marion	5,364	3,576	2,988
1,400 > Cap _{eff} ≥ 1,600	1,499	Marion	6,491	4,332	3,613
1,600 > Cap _{eff} ≥ 2,100	1,850	Marion	7,789	5,217	4,348
2,100 > Cap _{eff} ≥ 2,400	2,200	Marion	9,064	6,072	5,082
Cap _{eff} ≥ 2,400	2,718	Marion	10,926	7,339	6,116

WATER AND OTHER NON-ENERGY IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-HVC-HTHV-V01-180101

REVIEW DEADLINE: 1/1/2023

4.5 Lighting End Use

The commercial lighting measures use a standard set of variables for hours of use, waste heat factors, coincident factors and HVAC interaction effects. This table has been developed based on information provided by the various stakeholders. For ease of review, the table is included here and referenced in each measure.

The building characteristics can be found in the reference table named “EFLH Building Descriptions Updated 2014-11-21.xlsx”.

Note where a measure installation is within a building or application that does not fit with any of the defined building types below, the user should apply custom assumptions where it is reasonable to estimate them, else the building of best fit should be utilized.

Building/Space Type	Fixture Annual Operating Hours ⁵⁹⁵	Screw based bulb Annual Operating hours ⁵⁹⁶	Waste Heat Cooling Energy WHFe ⁵⁹⁷	Waste Heat Cooling Demand WHFd	Coincident Factor CF ⁵⁹⁸	Waste Heat Gas Heating IFTherms ⁵⁹⁹	Waste Heat Electric Resistance Heating IFkWh ⁶⁰⁰	Waste Heat Electric Heat Pump Heating IFkWh
Assisted Living	7,862	5,950	1.14	1.30	0.66	0.035	0.823	0.358
Childcare/Pre-School	2,860	2,860	1.17	1.29	0.72	0.018	0.420	0.183
College	3,395	2,588	1.06	1.39	0.63	0.020	0.462	0.201
Convenience Store	4,672	3,650	1.09	1.26	0.76	0.035	0.828	0.360
Elementary School	3,038	2,118	1.17	1.29	0.72	0.018	0.420	0.183
Garage	3,401	3,540	1.00	1.00	0.92	0.000	0.000	0.000
Garage, 24/7 lighting	8,766	8,766	1.00	1.00	1.00	0.000	0.000	0.000
Grocery	4,650	3,650	1.05	1.22	0.73	0.022	0.511	0.222
Healthcare Clinic	3,890	4,207	1.40	1.85	0.65	0.006	0.144	0.063
High School	3,038	2,327	1.18	1.39	0.72	0.028	0.656	0.285
Hospital - CAV no econ	7,616	4,207	1.11	1.29	0.76	0.022	0.527	0.229
Hospital - CAV econ	7,616	4,207	1.06	1.27	0.75	0.023	0.533	0.232
Hospital - VAV econ	7,616	4,207	1.37	1.79	0.70	0.010	0.241	0.105
Hospital - FCU	7,616	4,207	1.38	1.29	0.73	0.001	0.033	0.015

⁵⁹⁵ Fixtures hours of use are based upon schedule assumptions used in the eQuest models, except for those building types where Illinois based metering results provide a statistically valid estimate (currently: College, Elementary School, High School, Manufacturing, Low and Mid rise Office, Retail Department Store and Warehouse). Miscellaneous is a weighted average of indoor spaces using the relative area of each building type in the region (CBECS).

⁵⁹⁶ Hours of use for screw based bulbs are derived from DEER 2008 by building type for cfls. Garage, exterior and multi-family common area values are from the Hours of Use Table in this document. Miscellaneous is an average of interior space values. Some building types are averaged when DEER has two values: these include office, restaurant and retail. Healthcare clinic uses the hospital value.

⁵⁹⁷ The Waste Heat Factor for Energy and is developed using EQuest models for various building types base on Chicago Illinois (closest to statewide average HDD and CDD). Exterior and garage values are 1, unknown is a weighted average of the other building types.

⁵⁹⁸ Coincident diversity factors are based on either combined IL evaluation results (College, Elementary School, High School, Manufacturing, Low and Mid rise Office, Retail Department Store and Warehouse) or based upon schedules defined in the eQuest models described (all others).

⁵⁹⁹ IF Therms value is developed using EQuest models consistent with methodology for Waste Heat Factor for Energy.

⁶⁰⁰ Electric heat penalty assumptions are based on converting the IFTherm multiplier value in to kWh and then applying relative heating system efficiencies. The gas efficiency was assumed to be 78% AFUE based upon standard TRM assumption for existing unit average efficiency, and the electric resistance is assumed to be 100%, for Heat Pump is assumed to be 2.3COP:

$IF_{ElectricHeat} = IF_{Therms} * 29.3 \text{ kWh/therm} * 78\% \text{ (Gas Heating Equipment Efficiency)} / 100\% \text{ (Electric Resistance Efficiency)}$

Building/Space Type	Fixture Annual Operating Hours ⁵⁹⁵	Screw based bulb Annual Operating hours ⁵⁹⁶	Waste Heat Cooling Energy WHFe ⁵⁹⁷	Waste Heat Cooling Demand WHFd	Coincidence Factor CF ⁵⁹⁸	Waste Heat Gas Heating IFTherms ⁵⁹⁹	Waste Heat Electric Resistance Heating IFkWh ⁶⁰⁰	Waste Heat Electric Heat Pump Heating IFkWh
Manufacturing Facility	4,618	2,629	1.02	1.04	0.81	0.012	0.270	0.117
MF - High Rise - Common	6,138	5,950	1.14	1.32	0.64	0.025	0.596	0.259
MF - Mid Rise - Common	6,138	5,950	1.14	1.32	0.64	0.025	0.596	0.259
Hotel/Motel - Guest	2,390	777	1.18	1.36	0.28	0.020	0.463	0.201
Hotel/Motel - Common	6,138	4,542	1.20	1.24	0.73	0.032	0.748	0.325
Movie Theater	3,506	5,475	1.11	1.38	0.53	0.029	0.673	0.293
Office - High Rise - CAV no econ	2,886	3,088	1.00	1.07	0.57	0.037	0.874	0.380
Office - High Rise - CAV econ	2,886	3,088	1.00	1.07	0.57	0.039	0.905	0.394
Office - High Rise - VAV econ	2,886	3,088	1.27	1.65	0.53	0.022	0.510	0.222
Office - High Rise - FCU	2,886	3,088	1.35	1.56	0.59	0.015	0.346	0.150
Office - Low Rise	2,698	3,088	1.11	1.31	0.52	0.016	0.371	0.161
Office - Mid Rise	3,068	3,088	1.26	1.61	0.52	0.024	0.557	0.242
Religious Building	2,085	1,664	1.12	1.37	0.48	0.015	0.356	0.155
Restaurant	5,571	4,784	1.17	1.31	0.68	0.021	0.491	0.213
Retail - Department Store	5,478	2,935	1.12	1.31	0.95	0.022	0.514	0.223
Retail - Strip Mall	4,093	2,935	1.12	1.29	0.71	0.019	0.450	0.196
Warehouse	5,242	4,293	1.00	1.22	0.68	0.011	0.257	0.112
Unknown	3,379	3,612	1.09	1.36	0.58	0.022	0.522	0.227
Exterior – dusk to dawn	4,903	4,903	1.00	1.00	0.00	0.000	0.000	0.000
Exterior – dusk to business close	See calculation below		1.00	1.00	0.00	0.000	0.000	0.000
Low-Use Small Business	2,954	2,954	1.31	1.53	0.66	0.023	0.524	0.262
Uncooled Building	Varies	varies	1.00	1.00	0.66	0.014	0.320	0.160
Refrigerated Cases	5,802	n/a	1.29	1.29	0.69	0.000	0.000	0.000
Freezer Cases	5,802	n/a	1.50	1.5	0.69	0.000	0.000	0.000

Exterior Lighting Hours – dusk to business close

$$\text{Hours} = (6.19 * \text{Days}) + (\% \text{Adj} * \text{Days})$$

Where:

6.19 = Average hours per day between dusk and midnight⁶⁰¹

Days = Days of business operation

= Actual

%Adj = Percent adjustment dependent on hour closing⁶⁰²

⁶⁰¹ Calculated using the eQuest model by finding the total number of hours of exterior lighting consumption between dusk and midnight and dividing by 365 (2261 / 365 = 6.19 hours per day).

⁶⁰² See "IL TRM Ext Lighting.xlsx" for calculation.

Business closes at	4pm	5pm	6pm	7pm	8pm	9pm	10pm	11pm	12pm	1am	2am	3am
%Adj	-619%	-604%	-564%	-500%	-400%	-300%	-200%	-100%	0%	100%	200%	300%

For example a business open until 8pm, 260 days per year, would assume:

$$\text{Hours} = (6.19 * 260) + (-400\% * 260) = 569.4 \text{ hours}$$

4.5.1 Commercial ENERGY STAR Compact Fluorescent Lamp (CFL)

DESCRIPTION

A low wattage qualified compact fluorescent screw-in bulb (CFL) is installed in place of a baseline screw-in bulb.

Note a new ENERGY STAR specification v2.0 becomes effective on 1/2/2017

(https://www.energystar.gov/products/spec/lamps_specification_version_2_0_pd). The efficacy requirements can not currently be met by Compact Fluorescent Lamps, and therefore this specification has been removed. ENERGY STAR will maintain a list on their website with the final qualifying list of products prior to this change and it is strongly recommended that programs continue to use this list as qualifying criteria for products in the programs.

This characterization assumes that the CFL is installed in a commercial location. If the implementation strategy does not allow for the installation location to be known a deemed split should be used. For Residential targeted programs (e.g. an upstream retail program), a deemed split of 95% Residential and 5% Commercial assumptions should be used⁶⁰³, and for Commercial targeted programs a deemed split of 4% Residential and 96% Commercial should be used⁶⁰⁴.

Federal legislation stemming from the Energy Independence and Security Act of 2007 (EISA) required all general-purpose light bulbs between 40W and 100W to be approximately 30% more energy efficient than current incandescent bulbs. Production of 100W, standard efficacy incandescent lamps ended in 2012 followed by restrictions on 75W in 2013 and 60W and 40W in 2014. The baseline for this measure has therefore become bulbs (improved incandescent or halogen) that meet the new standard.

Finally, a provision in the EISA regulations requires that by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, in essence making the baseline equivalent to a current day CFL. Therefore the measure life (number of years that savings should be claimed) should be reduced once the assumed lifetime of the bulb exceeds 2020. Due to expected delay in clearing retail inventory and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until 2021.

This measure was developed to be applicable to the following program types: TOS, NC, RF. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the high-efficiency equipment must be a standard qualified compact fluorescent lamp.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be an EISA qualified incandescent or halogen as provided in the table provided in the Electric Energy Savings section.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life (number of years that savings should be claimed) should be calculated by dividing the rated life of the bulb (10,000 hours⁶⁰⁵) by the run hours. For example using Miscellaneous at 3,612 hours would give 2.8 years. When the number of years exceeds 2021, the number of years to that date should be used.

⁶⁰³ RES v C&I split is based on a weighted (by sales volume) average of ComEd PY6, PY7 and PY8 and Ameren PY5, PY6 and PY8 in store intercept survey results. See 'RESvCI Split_112016.xls'.

⁶⁰⁴ Based upon final weighted (by sales volume) average of the BILD program (ComEd's commercial lighting program) for PY 4 and PY5 and PY6.

⁶⁰⁵ Energy Star bulbs have a rated life of at least 8000 hours. In commercial settings you expect significantly less on/off switching than residential and so a rated life assumption of 10,000 hours is used.

DEEMED MEASURE COST

The incremental capital cost assumption for all bulbs under 2600 lumens is \$1.20⁶⁰⁶.

For bulbs over 2600 lumens the assumed incremental capital cost is \$5.

LOADSHAPE

- Loadshape C06 - Commercial Indoor Lighting
- Loadshape C07 - Grocery/Conv. Store Indoor Lighting
- Loadshape C08 - Hospital Indoor Lighting
- Loadshape C09 - Office Indoor Lighting
- Loadshape C10 - Restaurant Indoor Lighting
- Loadshape C11 - Retail Indoor Lighting
- Loadshape C12 - Warehouse Indoor Lighting
- Loadshape C13 - K-12 School Indoor Lighting
- Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)
- Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)
- Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)
- Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)
- Loadshape C18 - Industrial Indoor Lighting
- Loadshape C19 - Industrial Outdoor Lighting
- Loadshape C20 - Commercial Outdoor Lighting

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on the location type. Values are provided for each building type in section 4.5.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * \text{WHFe}$$

Where:

WattsBase = Actual (if retrofit measure) or based on lumens of CFL bulb and program year installed:

Minimum Lumens	Maximum Lumens	Incandescent Equivalent Post-EISA 2007 (WattsBase)
5280	6209	300
3000	5279	200

⁶⁰⁶ Based upon field data collected by CLEAResult and provided by ComEd. See ComEd Pricing Projections 06302016.xlsx for analysis.

Minimum Lumens	Maximum Lumens	Incandescent Equivalent Post-EISA 2007 (WattsBase)
2601	2999	150
1490	2600	72
1050	1489	53
750	1049	43
310	749	29
250	309	25

WattsEE = Actual wattage of CFL purchased or installed

ISR = In Service Rate or the percentage of units rebated that get installed.
 =100%⁶⁰⁷ if application form completed with sign off that equipment is not placed into storage

If sign off form not completed assume the following 3 year ISR assumptions:

Weighted Average 1st year In Service Rate (ISR)	2nd year Installations	3rd year Installations	Final Lifetime In Service Rate
71.2% ⁶⁰⁸	14.5%	12.3%	98.0% ⁶⁰⁹

Hours = Average hours of use per year are provided in Reference Table in Section 4.5, Screw based bulb annual operating hours, for each building type⁶¹⁰. If unknown use the Miscellaneous value.

WHFe = Waste heat factor for energy to account for cooling energy savings from efficient lighting are provided below for each building type in Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, a 14W standard CFL is installed in an office and sign off form provided:

$$\begin{aligned}\Delta \text{kWh} &= (((43 - 14)/1000) * 1.0 * 3088 * 1.25 \\ &= 111.9 \text{ kWh}\end{aligned}$$

⁶⁰⁷ Illinois evaluation of PY1 through PY3 has not found that fixtures or lamps placed into storage to be a significant enough issue to warrant including an "In-Service Rate" when commercial customers complete an application form.

⁶⁰⁸ 1st year in service rate is based upon review of PY4-6 evaluations from ComEd's commercial lighting program (BILD) (see 'IL Commercial Lighting ISR_2014.xls' for more information. The average first year ISR was calculated weighted by the number of bulbs sold.

⁶⁰⁹ The 98% Lifetime ISR assumption is based upon review of two evaluations:

'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings. Note that this Final Install Rate does NOT account for leakage of purchased bulbs being installed outside of the utility territory. EM&V should assess how and if data from evaluation should adjust this final installation rate to account for this impact

⁶¹⁰ Based on ComEd analysis taking DEER 2008 values and averaging with PY1 and PY2 evaluation results.

HEATING PENALTY

If electrically heated building:

$$\Delta kWh_{\text{heatpenalty}}^{611} = (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * -\text{IFkWh})$$

Where:

IFkWh = Lighting-HVAC Interaction Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, a 14W standard CFL is installed in a heat pump heated office and sign off form provided:

$$\begin{aligned} \Delta kWh_{\text{heatpenalty}} &= (((43 - 14) / 1000) * 1.0 * 3088 * -0.183) \\ &= -16.4 \text{ kWh} \end{aligned}$$

DEFERRED INSTALLS

As presented above, if a sign off form is not completed the characterization assumes that a percentage of bulbs purchased are not installed until Year 2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

Year 1 (Purchase Year) installs: Characterized using assumptions provided above or evaluated assumptions if available.

Year 2 and 3 installs: Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.

The NTG factor for the Purchase Year should be applied.

For example, for a 14W CFL (60W standard incandescent and 43W EISA qualified incandescent/halogen) purchased and using miscellaneous hours assumption.

$$\begin{aligned} \Delta kWH_{1\text{st year installs}} &= ((43 - 14) / 1000) * 0.755 * 3612 * 1.06 \\ &= 83.8 \text{ kWh} \end{aligned}$$

$$\begin{aligned} \Delta kWH_{2\text{nd year installs}} &= ((43 - 14) / 1000) * 0.121 * 3612 * 1.06 \\ &= 13.4 \text{ kWh} \end{aligned}$$

Note: Here we assume no change in hours assumption. NTG value from Purchase year applied.

$$\begin{aligned} \Delta kWH_{3\text{rd year installs}} &= ((43 - 14) / 1000) * 0.103 * 3612 * 1.06 \\ &= 11.4 \text{ kWh} \end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{WHFd} * \text{CF}$$

Where:

⁶¹¹Negative value because this is an increase in heating consumption due to the efficient lighting.

WHFd = Waste heat factor for demand to account for cooling savings from efficient lighting in cooled buildings is provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value..

CF = Summer Peak Coincidence Factor for measure is provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value..

Other factors as defined above

For example, a 14W standard CFL is installed in an office and sign off form provided:

$$\begin{aligned}\Delta kW &= ((43 - 14)/1000) * 1.0 * 1.3 * 0.66 \\ &= 0.025kW\end{aligned}$$

NATURAL GAS ENERGY SAVINGS

Heating Penalty if fossil fuel heated building (or if heating fuel is unknown):

$$\Delta \text{Therms}^{612} = (((\text{WattsBase} - \text{WattsEE})/1000) * \text{ISR} * \text{Hours} * - \text{IFTherms})$$

Where:

IFTherms = Lighting-HVAC Interaction Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

Other factors as defined above

For example, a 14W standard CFL is installed in an office and sign off form provided:

$$\begin{aligned}\Delta \text{Therms} &= (((43 - 14)/1000) * 1.0 * 3088 * -0.016 \\ &= - 1.4 \text{ Therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

The O&M assumptions that should be used in cost effectiveness calculations are provided below:

Replacement Period (years) ⁶¹³	Replacement Cost ⁶¹⁴
= 1000 / Hours	\$1.25

⁶¹² Negative value because this is an increase in heating consumption due to the efficient lighting.

⁶¹³ Calculated by dividing assumed rated life of baseline bulb by hours of use. Assumed lifetime of EISA qualified Halogen/ Incandescent is 1000 hours. The manufacturers are simply using a regular incandescent lamp with halogen fill gas rather than Halogen Infrared to meet the standard (as provided by G. Arnold, NEEP and confirmed by N. Horowitz at NRDC).

⁶¹⁴ Based upon field data collected by CLEAResult and provided by ComEd. See ComEd Pricing Projections 06302016.xlsx for analysis.

It is important to note that for cost-effectiveness screening purposes, the O&M cost adjustments should only be applied in cases where the light bulbs are actually in service and so should be multiplied by the appropriate ISR.

MEASURE CODE: CI-LTG-CCFL-V07-180101

REVIEW DEADLINE: 1/1/2020

4.5.2 Fluorescent Delamping

DESCRIPTION

This measure addresses the permanent removal of existing 8', 4', 3' and 2' fluorescent lamps. Unused lamps, lamp holders, and ballasts must be permanently removed from the fixture. This measure is applicable when retrofitting from T12 lamps to T8 lamps or simply removing lamps from a T8 fixture. Removing lamps from a T12 fixture that is not being retrofitted with T8 lamps are not eligible for this incentive.

Customers are responsible for determining whether or not to use reflectors in combination with lamp removal in order to maintain adequate lighting levels. Lighting levels are expected to meet the Illuminating Engineering Society of North America (IESNA) recommended light levels. Unused lamps, lamp holders, and ballasts must be permanently removed from the fixture and disposed of in accordance with local regulations. A pre-approval application is required for lamp removal projects.

This measure was developed to be applicable to the following program types: RF. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

Savings are defined on a per removed lamp basis. The retrofit wattage (efficient conditioned) is therefore assumed to be zero. The savings numbers provided below are for the straight lamp removal measures, as well as the lamp removal and install reflector measures. The lamp installed/retrofit is captured in another measure.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is either a T12 or a T8 lamp with default wattages provided below. Note, if the program does not allow for the lamp type to be known, then a T12:T8 weighting of 80%:20% can be applied⁶¹⁵.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 11 years per DEER 2005.

DEEMED MEASURE COST

The incremental capital cost is provided in the table below:

Measure Category	Value	Source
8-Foot Lamp Removal	\$16.00	ComEd/KEMA regression ⁶¹⁶
4-Foot Lamp Removal	\$12.00	ICF Portfolio Plan
8-Foot Lamp Removal with reflector	\$30.00	KEMA Assumption
4-Foot Lamp Removal with reflector	\$25.00	KEMA Assumption
2-Foot or 3-Foot Removal	\$12.35	KEMA Assumption
2-Foot or 3-Foot Removal with reflector	\$25.70	KEMA Assumption

LOADSHAPE

Loadshape C06 - Commercial Indoor Lighting

Loadshape C07 - Grocery/Conv. Store Indoor Lighting

Loadshape C08 - Hospital Indoor Lighting

⁶¹⁵ Based on ComEd's estimate of lamp type saturation.

⁶¹⁶ Based on the assessment of active projects in the 2008-09 ComEd Smart Ideas Program. See files "ltg costs 12-10-10.xl." and "Lighting Unit Costs 102605.doc"

Loadshape C09 - Office Indoor Lighting
 Loadshape C10 - Restaurant Indoor Lighting
 Loadshape C11 - Retail Indoor Lighting
 Loadshape C12 - Warehouse Indoor Lighting
 Loadshape C13 - K-12 School Indoor Lighting
 Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)
 Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)
 Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)
 Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)
 Loadshape C18 - Industrial Indoor Lighting
 Loadshape C19 - Industrial Outdoor Lighting
 Loadshape C20 - Commercial Outdoor Lighting

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on the location type. Values are provided for each building type in the reference section below.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * \text{WHFe}$$

Where:

WattsBase = Assume wattage reduction of lamp removed

	Wattage of lamp removed ⁶¹⁷		Weighted average
	T8	T12	80% T12, 20% T8
8-ft T8	38.6	60.3	56.0
4-ft T8	19.4	33.7	30.8
3-ft T8	14.6	40.0	34.9
2-ft T8	9.8	28.0	24.4

WattsEE = 0

ISR = In Service Rate or the percentage of units rebated that get installed.

⁶¹⁷ Default wattage reduction is based on averaging the savings from moving from a 2 to 1, 3 to 2 and 4 to 3 lamp fixture, as provided in the Standard Performance Contract Procedures Manual: Appendix B: Table of Standard Fixture Wattages (http://www.sce.com/NR/rdonlyres/7A3455F0-A337-439B-9607-10A016D32D4B/0/spc_B_Std_Fixture_Watts.pdf). An adjustment is made to the T8 delamped fixture to account for the significant increase in ballast factor. See 'Delamping calculation.xls' for details.

=100% if application form completed with sign off that equipment permanently removed and disposed of.

Hours = Average hours of use per year are provided in Reference Table in Section 4.5. If unknown use the Miscellaneous value.

WHFe = Waste heat factor for energy to account for cooling energy savings from efficient lighting are provided below for each building type in Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, delamping a 4 ft T8 fixture in an office building:

$$\begin{aligned}\Delta \text{kWh} &= ((19.4 - 0)/1000) * 1.0 * 4439 * 1.25 \\ &= 107.6 \text{ kWh}\end{aligned}$$

HEATING PENALTY

If electrically heated building:

$$\Delta \text{kWh}_{\text{heatpenalty}}^{618} = (((\text{WattsBase} - \text{WattsEE})/1000) * \text{ISR} * \text{Hours} * -\text{IFkWh})$$

Where:

IFkWh = Lighting-HVAC Interaction Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, delamping a 4 ft T8 fixture in a heat pump heated office building:

$$\begin{aligned}\Delta \text{kWh}_{\text{heatpenalty}} &= ((19.4 - 0)/1000) * 1.0 * 4439 * -0.151 \\ &= -13.0 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = ((\text{WattsBase} - \text{WattsEE})/1000) * \text{ISR} * \text{WHFd} * \text{CF}$$

Where:

WHFd = Waste heat factor for demand to account for cooling savings from efficient lighting in cooled buildings is provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value..

CF = Summer Peak Coincidence Factor for measure is provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value..

Other factors as defined above

For example, delamping a 4 ft T8 fixture in an office building:

$$\begin{aligned}\Delta \text{kW} &= ((19.4 - 0)/1000) * 1.0 * 1.3 * 0.66 \\ &= 0.017 \text{ kW}\end{aligned}$$

⁶¹⁸Negative value because this is an increase in heating consumption due to the efficient lighting.

NATURAL GAS ENERGY SAVINGS

Heating Penalty if fossil fuel heated building (or if heating fuel is unknown):

$$\Delta\text{Therms}^{619} = (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * - \text{IFTherms}$$

Where:

IFTherms = Lighting-HVAC Interaction Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

Other factors as defined above

For example, delamping a 4 ft T8 fixture in an office building:

$$\begin{aligned}\Delta\text{Therms} &= ((19.4 - 0) / 1000) * 1.0 * 4439 * -0.016 \\ &= -1.4 \text{ therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-LTG-DLMP-V02-140601

REVIEW DEADLINE: 1/1/2021

⁶¹⁹ Negative value because this is an increase in heating consumption due to the efficient lighting.

4.5.3 High Performance and Reduced Wattage T8 Fixtures and Lamps

DESCRIPTION

This measure applies to “High Performance T8” (HPT8) lamp/ballast systems that have higher lumens per watt than standard T8 systems. This measure applies to the installation of new equipment with efficiencies that exceed that of the equipment that would have been installed following standard market practices and is applicable to time of sale as well as retrofit measures. Retrofit measures may include new fixtures or relamp/reballast measures. In addition, options have been provided to allow for the “Reduced Wattage T8 lamps” or RWT8 lamps that result in re-lamping opportunities that produce equal or greater light levels than standard T8 lamps while using fewer watts.

If the implementation strategy does not allow for the installation location to be known, a deemed split of 99% Commercial and 1% Residential should be used⁶²⁰.

This measure was developed to be applicable to the following program types: TOS, RF, DI.

If applied to other program types, the measure savings should be verified.

The measure applies to all commercial HPT8 installations excluding new construction and major renovation or change of use measures (see lighting power density measure). Lookup tables have been provided to account for the different types of installations. Whenever possible, actual costs and hours of use should be utilized for savings calculations. Default new and baseline assumptions have been provided in the reference tables. Default component costs and lifetimes have been provided for Operating and Maintenance Calculations. Please see the Definition Table to determine applicability for each program. HPT8 configurations not included in the TRM may be included in custom program design using the provided algorithms as long as energy savings is achieved. The following table defines the applicability for different programs

Time of Sale (TOS)	Retrofit (RF) and Direct Install (DI)
<p>This measure relates to the installation of new equipment with efficiency that exceeds that of equipment that would have been installed following standard market practices. In general, the measure will include qualifying high efficiency low ballast factor ballasts paired with high efficiency long life lamps as detailed in the attached tables. High-bay applications use this system paired with qualifying high ballast factor ballasts and high performance 32 w lamps. Custom lighting designs can use qualifying low, normal or high ballast factor ballasts and qualifying lamps in lumen equivalent applications where total system wattage is reduced when calculated using the Calculation of Savings Algorithms.</p>	<p>This measure relates to the replacement of existing equipment with new equipment with efficiency that exceeds that of the existing equipment. In general, the retrofit will include qualifying high efficiency low ballast factor ballasts paired with high efficiency long life lamps as detailed in the attached tables. Custom lighting designs can use qualifying low, normal or high ballast factor ballasts and qualifying lamps in lumen equivalent applications where total system wattage is reduced when calculated using the Calculation of Savings Algorithms.</p> <p>High efficiency troffers (new/or retrofit) utilizing HPT8 technology can provide even greater savings. When used in a high-bay application, high-performance T8 fixtures can provide equal light to HID high-bay fixtures, while using fewer watts; these systems typically utilize high ballast factor ballasts, but qualifying low and normal ballast factor ballasts may be used when appropriate light levels are provided and overall wattage is reduced.</p>

⁶²⁰ Based on weighted average of Final ComEd’s BILD program data from PY5 and PY6. For Residential installations, hours of use assumptions from ‘5.5 Interior Hardwired Compact Fluorescent Lamp (CFL) Fixture’ measure should be used.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient conditions for all applications are a qualifying HP or RWT8 fixture and lamp/ballast combinations listed on the CEE website under qualifying HP T8 products⁶²¹ and qualifying RWT8 products⁶²².

The definition of efficient equipment varies based on the program and is defined below:

Time of Sale (TOS)	Retrofit (RF) and Direct Install (DI)
<p>High efficiency troffers combined with high efficiency lamps and ballasts allow for fewer lamps to be used to provide a given lumen output. High efficiency troffers must have a fixture efficiency of 80% or greater to qualify. Default values are given for a 2 lamp HPT8 fixture replacing a 3 lamp standard efficiency T8 fixture, but other configurations may qualify and the Calculation of savings algorithm used to account for base watts being replaced with EE watts.</p> <p>High bay fixtures must have fixture efficiencies of 85% or greater.</p> <p>RWT8 lamps: 2', 3' and 8' lamps must meet the wattage requirements specified in the RWT8 new and baseline assumptions table. This measure assumes a lamp only purchase.</p>	<p>High efficiency troffers (new or retrofit kits) combined with high efficiency lamps and ballasts allow for fewer lamps to be used to provide a given lumen output. High efficiency troffers must have a fixture efficiency of 80% or greater to qualify. Default values are given for a 2 lamp HPT8 fixture replacing a 3 lamp standard efficiency T8 fixture, but other configurations may qualify and the Calculation of savings algorithm used to account for base watts being replaced with EE watts.</p> <p>High bay fixtures will have fixture efficiencies of 85% or greater.</p> <p>RWT8: 2', 3' and 8' lamps must meet the wattage requirements specified in the RWT8 new and baseline assumptions table.</p>

DEFINITION OF BASELINE EQUIPMENT

The definition of baseline equipment varies based on the program and is defined below:

Time of Sale (TOS)	Retrofit (RF) and Direct Install (DI)
<p>The baseline is standard efficiency T8 systems that would have been installed. The baseline for high-bay fixtures is pulse start metal halide fixtures, the baseline for a 2 lamp high efficiency troffer is a 3 lamp standard efficiency troffer.</p>	<p>The baseline is the existing system.</p> <p>In July 14, 2012, Federal Standards were enacted that were expected to eliminate T-12s as an option for linear fluorescent fixtures. Through v3.0 of the TRM, it was assumed that the T-12 would no longer be baseline for retrofits from 1/1/2016. However, due to significant loopholes in the legislation, T-12 compliant product is still freely available and in Illinois T-12s continue to hold a significant share of the existing and replacement lamp market. Therefore the timing of the sunset of T-12s as a viable baseline has been pushed back in v6.0 until 1/1/2019 and will be revisited in future update sessions.</p> <p>There will be a baseline shift applied to all measures installed before 2019. See table C-1.</p>

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The deemed lifetime of efficient equipment varies based on the program and is defined below:

⁶²¹ <http://library.cee1.org/content/cee-high-performance-t8-specification>

⁶²² <http://library.cee1.org/content/reduced-wattage-t8-specification>

Time of Sale (TOS)	Retrofit (RF) and Direct Install (DI)
<p>Fixture lifetime is 15 years⁶²³. Fixture retrofits which utilize RWT8 lamps have a lifetime equivalent to the life of the lamp, capped at 15 years. There is no guarantee that a reduced wattage lamp will be installed at time of burnout, but if one is, savings will be captured in the RWT8 measure below. RWT8 lifetime is the life of the product, at the reported operating hours (lamp life in hours divided by operating hours per year – see reference table "RWT8 Component Costs and Lifetime"), capped at 15 years.⁶²⁴</p>	<p>Fixture lifetime is 15 years. As per explanation above, for existing T12 fixtures, a mid life baseline shift should be applied in 2019 as described in table C-1. Note, since the fixture lifetime is deemed at 15 years, the replacement cost of both the lamp and ballast should be incorporated in to the O&M calculation.</p>

DEEMED MEASURE COST

The deemed measure cost is found in the reference table at the end of this characterization.

LOADSHAPE

- Loadshape C06 - Commercial Indoor Lighting
- Loadshape C07 - Grocery/Conv. Store Indoor Lighting
- Loadshape C08 - Hospital Indoor Lighting
- Loadshape C09 - Office Indoor Lighting
- Loadshape C10 - Restaurant Indoor Lighting
- Loadshape C11 - Retail Indoor Lighting
- Loadshape C12 - Warehouse Indoor Lighting
- Loadshape C13 - K-12 School Indoor Lighting
- Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)
- Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)
- Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)
- Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)
- Loadshape C18 - Industrial Indoor Lighting
- Loadshape C19 - Industrial Outdoor Lighting
- Loadshape C20 - Commercial Outdoor Lighting

⁶²³ 15 years from GDS Measure Life Report, June 2007

⁶²⁴ ibid

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = (Watts_{base} - Watts_{EE}) / 1000 * Hours * WHF_e * ISR$$

Where:

Watts_{base} = Input wattage of the existing system which depends on the baseline fixture configuration (number and type of lamp) and number of fixtures. Value can be selected from the appropriate reference table as shown below, or a custom value can be entered if the configurations in the tables is not representative of the existing system.

Program	Reference Table
Time of Sale	A-1: HPT8 New and Baseline Assumptions
Retrofit	A-2: HPT8 New and Baseline Assumptions
Reduced Wattage T8, time of sale or retrofit	A-3: RWT8 New and Baseline Assumptions

Watt_{SEE} = New Input wattage of EE fixture which depends on new fixture configuration (number of lamps) and ballast factor and number of fixtures. Value can be selected from the appropriate reference table as shown below, or a custom value can be entered if the configurations in the tables is not representative of the existing system.

Program	Reference Table
Time of Sale	A-1: HPT8 New and Baseline Assumptions
Retrofit	A-2: HPT8 New and Baseline Assumptions
Reduced Wattage T8, time of sale or retrofit	A-3: RWT8 New and Baseline Assumptions

Hours = Average hours of use per year as provided by the customer or selected from the Reference Table in Section 4.5, Fixture annual operating hours. If hours or building type are unknown, use the Miscellaneous value.

WHF_e = Waste heat factor for energy to account for cooling energy savings from efficient lighting is selected from the Reference Table in Section 4.5 for each building type. If building is un-cooled, the value is 1.0.

ISR = In Service Rate or the percentage of units rebated that get installed.
=100%⁶²⁵ if application form completed with sign off that equipment is not placed into storage

If sign off form not completed assume the following 3 year ISR assumptions:

⁶²⁵ Illinois evaluation of PY1 through PY3 has not found that fixtures or lamps placed into storage to be a significant enough issue to warrant including an "In-Service Rate" when commercial customers complete an application form.

Weighted Average 1st year In Service Rate (ISR)	2nd year Installations	3rd year Installations	Final Lifetime In Service Rate
98% ⁶²⁶	0%	0%	98.0% ⁶²⁷

HEATING PENALTY

If electrically heated building:

$$\Delta \text{Wh}_{\text{heatpenalty}}^{628} = (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * -\text{IFkWh})$$

Where:

IFkWh = Lighting-HVAC Interaction Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

SUMMER COINCIDENT DEMAND SAVINGS

$$\Delta \text{kW} = ((\text{Watts}_{\text{base}} - \text{Watts}_{\text{EE}}) / 1000) * \text{WHF}_d * \text{CF} * \text{ISR}$$

Where:

WHF_d = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is selected from the Reference Table in Section 4.5 for each building type. If the building is not cooled WHF_d is 1.

CF = Summer Peak Coincidence Factor for measure is selected from the Reference Table in Section 4.5 for each building type. If the building type is unknown, use the Miscellaneous value of 0.66.

Other factors as defined above

NATURAL GAS SAVINGS

$$\Delta \text{Therms}^{629} = (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * -\text{IFTherms})$$

Where:

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Please select from the Reference Table in Section 4.5 for each building type.

⁶²⁶ 1st year in service rate is based upon review of PY5-6 evaluations from ComEd's commercial lighting program (BILD) (see 'IL Commercial Lighting ISR_2014.xls' for more information)

⁶²⁷ The 98% Lifetime ISR assumption is based upon review of two evaluations:

'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings. Note that this Final Install Rate does NOT account for leakage of purchased bulbs being installed outside of the utility territory. EM&V should assess how and if data from evaluation should adjust this final installation rate to account for this impact

⁶²⁸ Negative value because this is an increase in heating consumption due to the efficient lighting.

⁶²⁹ Negative value because this is an increase in heating consumption due to the efficient lighting.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

Actual operation and maintenance costs will vary by specific equipment installed/replaced. See Reference tables for Operating and Maintenance Values;

Program	Reference Table
Time of Sale	B-1: HPT8 Component Costs and Lifetime
Retrofit	B-2: HPT8 Component Costs and Lifetime
Reduced Wattage T8, time of sale or retrofit	B-3: HPT8 Component Costs and Lifetime

REFERENCE TABLES

See following page

A-1: Time of Sale: HPT8 New and Baseline Assumptions⁶³⁰

EE Measure Description	Nominal Watts	Watts _{EE}	Baseline Description	Nominal Watt	Watts _{BASE}	Incremental Cost	Watts _{SAVE}
4-Lamp HPT8 w/ High-BF Ballast High-Bay	128	147.2	200 Watt Pulse Start Metal-Halide	200	232	\$75	84.80
4-Lamp HPT8 w/ High-BF Ballast High-Bay	128	147.2	250 Watt Metal Halide	250	295	\$75	147.80
6-Lamp HPT8 w/ High-BF Ballast High-Bay	192	220.8	320 Watt Pulse Start Metal-Halide	320	348.8	\$75	128.00
6-Lamp HPT8 w/ High-BF Ballast High-Bay	192	220.8	400 Watt Pulse Start Metal Halide	400	455	\$75	234.20
8-Lamp HPT8 w/ High-BF Ballast High-Bay	256	294.4	Proportionally Adjusted according to 6-Lamp HPT8 Equivalent to 320 PSMH	320	476	\$75	181.60
8-Lamp HPT8 w/ High-BF Ballast High-Bay	256	292.4	Proportionally Adjusted according to 6-Lamp HPT8 Equivalent to 400 W Metal Halide	400	618	75	323.60
1-Lamp HPT8-high performance 32 w lamp	32	24.64	1-Lamp Standard F32T8 w/ Elec. Ballast	32	28.16	\$15	3.52
1-Lamp HPT8-high performance 28 w lamp	28	21.56	1-Lamp Standard F32T8 w/ Elec. Ballast	32	28.16	\$15	6.60
1-Lamp HPT8-high performance 25 w lamp	25	19.25	1-Lamp Standard F32T8 w/ Elec. Ballast	32	28.16	\$15	8.91
2-Lamp HPT8 -high performance 32 w lamp	64	49.28	2-Lamp Standard F32T8 w/ Elec. Ballast	64	56.32	\$18	7.04
2-Lamp HPT8-high performance 28 w lamp	56	43.12	2-Lamp Standard F32T8 w/ Elec. Ballast	64	56.32	\$18	13.20
2-Lamp HPT8-high performance 25 w lamp	50	38.5	2-Lamp Standard F32T8 w/ Elec. Ballast	64	56.32	\$18	17.82
3-Lamp HPT8-high performance 32 w lamp	96	73.92	3-Lamp Standard F32T8 w/ Elec. Ballast	96	84.48	\$20	10.56
3-Lamp HPT8-high performance 28 w lamp	84	64.68	3-Lamp Standard F32T8 w/ Elec. Ballast	96	84.48	\$20	19.80
3-Lamp HPT8-high performance 25 w lamp	75	57.75	3-Lamp Standard F32T8 w/ Elec. Ballast	96	84.48	\$20	26.73
4-Lamp HPT8 -high performance 32 w lamp	128	98.56	4-Lamp Standard F32T8 w/ Elec. Ballast	128	112.64	\$23	14.08
4-Lamp HPT8-high performance 28 w lamp	112	86.24	4-Lamp Standard F32T8 w/ Elec. Ballast	128	112.64	\$23	26.40
4-Lamp HPT8-high performance 25 w lamp	100	77	4-Lamp Standard F32T8 w/ Elec. Ballast	128	112.64	\$23	35.64
2-lamp High-Performance HPT8 Troffer	64	49.28	3-Lamp F32T8 w/ Elec. Ballast	96	84.48	\$100	35.20

Table developed using a constant ballast factor of .77 for troffers/linear HPT8 and 1.15 for HPT8 highbay, 1.0 for all MH/MHPS, and 0.95 for T12 and 0.88 for standard T8. Input wattages are an average of manufacturer inputs that account for ballast efficacy

⁶³⁰ Watt, lumen, lamp life, and ballast factor assumptions for efficient measures are based upon Consortium for Energy Efficiency (CEE) Commercial Lighting Qualifying Product Lists. Watt, lumen, lamp life, and ballast factor assumptions for baseline fixtures are based upon manufacturer specification sheets. Baseline and efficient measure cost data comes from lighting suppliers, past Efficiency Vermont projects, and professional judgment.

A-2: Retrofit HPT8 New and Baseline Assumptions⁶³¹

EE Measure Description	Nominal Watts	Ballast Factor	WattsEE	Baseline Description	Nominal Watts	WattsBASE	WattsSAVE	Full Measure Cost
4-Lamp HPT8 w/ High-BF Ballast High-Bay	128	1.15	147.2	200 Watt Pulse Start Metal-Halide	200	232	84.80	\$200
4-Lamp HPT8 w/ High-BF Ballast High-Bay	128	1.15	147.2	250 Watt Metal Halide	250	295	147.80	\$200
6-Lamp HPT8 w/ High-BF Ballast High-Bay	192	1.15	220.8	320 Watt Pulse Start Metal-Halide	320	348.8	128.00	\$225
6-Lamp HPT8 w/ High-BF Ballast High-Bay	192	1.15	220.8	400 Watt Pulse Start Metal Halide	400	455	234.20	\$225
8-Lamp HPT8 w/ High-BF Ballast High-Bay	256	1.15	294.4	Proportionally Adjusted according to 6-Lamp HPT8 Equivalent to 320 PSMH	320	476	181.60	\$250
8-Lamp HPT8 w/ High-BF Ballast High-Bay	256	1.15	294.4	Proportionally Adjusted according to 6-Lamp HPT8 Equivalent to 400 W Metal Halide	400	618	323.60	\$250
1-Lamp Relamp/Reballast T12 to HPT8	32	0.77	24.64	1-Lamp F34T12 w/ EEMag Ballast	34	42	17.36	\$50
2-Lamp Relamp/Reballast T12 to HPT8	64	0.77	49.28	2-Lamp F34T12 w/ EEMag Ballast	68	67	17.72	\$55
3-Lamp Relamp/Reballast T12 to HPT8	96	0.77	73.92	3-Lamp F34T12 w/ EEMag Ballast	102	104	30.08	\$60
4-Lamp Relamp/Reballast T12 to HPT8	128	0.77	98.56	4-Lamp F34T12 w/ EEMag Ballast	136	144	45.44	\$65
1-Lamp Relamp/Reballast T12 to HPT8	32	0.77	24.64	1-Lamp F40T12 w/ EEMag Ballast	40	41	16.36	\$50
2-Lamp Relamp/Reballast T12 to HPT8	64	0.77	49.28	2-Lamp F40T12 w/ EEMag Ballast	80	87	37.72	\$55
3-Lamp Relamp/Reballast T12 to HPT8	96	0.77	73.92	3-Lamp F40T12 w/ EEMag Ballast	120	141	67.08	\$60
4-Lamp Relamp/Reballast T12 to HPT8	128	0.77	98.56	4-Lamp F40T12 w/ EEMag Ballast	160	172	73.44	\$65
1-Lamp Relamp/Reballast T12 to HPT8	32	0.77	24.64	1-Lamp F40T12 w/ Mag Ballast	40	51	26.36	\$50
2-Lamp Relamp/Reballast T12 to HPT8	64	0.77	49.28	2-Lamp F40T12 w/ Mag Ballast	80	97	47.72	\$55
3-Lamp Relamp/Reballast T12 to HPT8	96	0.77	73.92	3-Lamp F40T12 w/ Mag Ballast	120	135	61.08	\$60
4-Lamp Relamp/Reballast T12 to HPT8	128	0.77	98.56	4-Lamp F40T12 w/ Mag Ballast	160	175	76.44	\$65
1-Lamp Relamp/Reballast T8 to HPT8	32	0.77	24.64	1-Lamp F32T8 w/ Elec. Ballast	32	28.16	3.52	\$50
2-Lamp Relamp/Reballast T8 to HPT8	64	0.77	49.28	2-Lamp F32T8 w/ Elec. Ballast	64	56.32	7.04	\$55
3-Lamp Relamp/Reballast T8 to HPT8	96	0.77	73.92	3-Lamp F32T8 w/ Elec. Ballast	96	84.48	10.56	\$60
4-Lamp Relamp/Reballast T8 to HPT8	128	0.77	98.56	4-Lamp F32T8 w/ Elec. Ballast	128	112.64	14.08	\$65

⁶³¹ Watt, lumen, lamp life, and ballast factor assumptions for efficient measures are based upon Consortium for Energy Efficiency (CEE) Commercial Lighting Qualifying Product Lists. Watt, lumen, lamp life, and ballast factor assumptions for baseline fixtures are based upon manufacturer specification sheets. Baseline and efficient measure cost data comes from lighting suppliers, past Efficiency Vermont projects, Xcel Energy Lighting Efficiency Input Wattage Guide and professional judgment.

EE Measure Description	Nominal Watts	Ballast Factor	Watts _{EE}	Baseline Description	Nominal Watts	Watts _{BASE}	Watts _{SAVE}	Full Measure Cost
2-lamp High-Performance HPT8 Troffer or high efficiency retrofit troffer	64	0.77	49.28	3-Lamp F32T8 w/ Elec. Ballast	96	84.48	35.20	\$100

Table developed using a constant ballast factor of 0.77 for troffers/linear HPT8 and 1.15 for HPT8 highbay, 1.0 for all MH/MHPS, and 0.95 for T12 and 0.88 for standard T8. Input wattages are an average of manufacturer inputs that account for ballast efficacy.

A- 3: RWT8 New and Baseline Assumptions

Table developed using a constant ballast factor of 0.88 for RWT8 and Standard T8.

EE Measure Description	Nominal Watts	Watts _{EE}	EE Lamp Cost	Baseline Description	Base Lamp Cost	Nominal Watts	Watts _{BASE}	Watts _{SAVE}	Measure Cost
RW T8 - F28T8 Lamp	28	24.64	\$4.50	F32 T8 Standard Lamp	\$2.50	32	28.16	3.52	\$2.00
RWT8 F2T8 Extra Life Lamp	28	24.64	\$4.50	F32 T8 Standard Lamp	\$2.50	32	28.16	3.52	\$2.00
RWT8 - F32/25W T8 Lamp	25	22.00	\$4.50	F32 T8 Standard Lamp	\$2.50	32	28.16	6.16	\$2.00
RWT8 - F32/25W T8 Lamp Extra Life	25	22.00	\$4.50	F32 T8 Standard Lamp	\$2.50	32	28.16	6.16	\$2.00
RWT8 F17T8 Lamp - 2 ft	16	14.08	\$4.80	F17 T8 Standard Lamp - 2ft	\$2.80	17	14.96	0.88	\$2.00
RWT8 F25T8 Lamp - 3 ft	23	20.24	\$5.10	F25 T8 Standard Lamp - 3ft	\$3.10	25	22.00	1.76	\$2.00
RWT8 F30T8 Lamp - 6' Utube	30	26.40	\$11.31	F32 T8 Standard Utube	\$9.31	32	28.16	1.76	\$2.00
RWT8 F29T8 Lamp - Utube	29	25.52	\$11.31	F32 T8 Standard Utube	\$9.31	32	28.16	2.64	\$2.00
RWT8 F96T8 Lamp - 8 ft	65	57.20	\$9.00	F96 T8 Standard Lamp - 8 ft	\$7.00	70	61.60	4.40	\$2.00

B-1: Time of Sale T8 Component Costs and Lifetime⁶³²

EE Measure Description	EE Lamp Cost	EE Lamp Life (hrs)	EE Lamp Rep. Labor Cost per lamp	EE Ballast Cost	EE Ballast Life (hrs)	EE Ballast Rep. Labor Cost	Baseline Description	Base Lamp Cost	Base Lamp Life (hrs)	Base Lamp Rep. Labor Cost	Base Ballast Cost	Base Ballast Life (hrs)	Base Ballast Rep. Labor Cost
4-Lamp HPT8 w/ High-BF Ballast High-Bay	\$5.00	24000	\$6.67	\$32.50	70000	\$15.00	200 Watt Pulse Start Metal-Halide	\$21.00	10000	\$6.67	\$87.75	40000	\$22.50
6-Lamp HPT8 w/ High-BF Ballast High-Bay	\$5.00	24000	\$6.67	\$32.50	70000	\$15.00	320 Watt Pulse Start Metal-Halide	\$21.00	20000	\$6.67	\$109.35	40000	\$22.50
8-Lamp HPT8 w/ High-BF Ballast High-Bay	\$5.00	24000	\$6.67	\$32.50	70000	\$15.00	Lamp HPT8 Equivalent to 320 PSMH	\$21.00	20000	\$6.67	\$109.35	40000	\$22.50
1-Lamp HPT8 – all qualifying lamps	\$5.00	24000	\$2.67	\$32.50	70000	\$15.00	1-Lamp Standard F32T12 w/ Elec Ballast	\$2.50	20000	\$2.67	\$15.00	70000	\$15.00
2-Lamp HPT8 – all qualifying lamps	\$5.00	24000	\$2.67	\$32.50	70000	\$15.00	2-Lamp Standard F32T12 w/ Elec Ballast	\$2.50	20000	\$2.67	\$15.00	70000	\$15.00
3-Lamp HPT8 – all qualifying lamps	\$5.00	24000	\$2.67	\$32.50	70000	\$15.00	3-Lamp Standard F32T8 w/ Elec. Ballast	\$2.50	20000	\$2.67	\$15.00	70000	\$15.00
4-Lamp HPT8 – all qualifying lamps	\$5.00	24000	\$2.67	\$32.50	70000	\$15.00	4-Lamp Standard F32T8 w/ Elec. Ballast	\$2.50	20000	\$2.67	\$15.00	70000	\$15.00
				\$32.50									
2-lamp High-Performance HPT8 Troffer	\$5.00	24000	\$2.67	\$32.50	70000	\$15.00	3-Lamp F32T8 w/ Elec. Ballast	\$2.50	20000	\$2.67	\$15.00	70000	\$15.00

⁶³² Watt, lumen, lamp life, and ballast factor assumptions for efficient measures are based upon Consortium for Energy Efficiency (CEE) Commercial Lighting Qualifying Product Lists. Watt, lumen, lamp life, and ballast factor assumptions for baseline fixtures are based upon manufacturer specification sheets. Baseline and efficient measure cost data comes from lighting suppliers, past Efficiency Vermont projects, and professional judgment

B-2: T8 Retrofit Component Costs and Lifetime⁶³³

EE Measure Description	EE Lamp Cost	EE Lamp Life (hrs)	EE Lamp Rep. Labor Cost per lamp	EE Ballast Cost	EE Ballast Life (hrs)	EE Ballast Rep. Labor Cost	Baseline Description	Base Lamp Cost	Base Lamp Life (hrs)	Base Lamp Rep. Labor Cost	Base Ballast Cost	Base Ballast Life (hrs)	Base Ballast Rep. Labor Cost
4-Lamp HPT8 w/ High-BF Ballast High-Bay	\$5.00	24000	\$6.67	\$32.50	70000	\$15.00	200 Watt Pulse Start Metal-Halide	\$29.00	12000	\$6.67	\$87.75	40000	\$22.50
6-Lamp HPT8 w/ High-BF Ballast High-Bay	\$5.00	24000	\$6.67	\$32.50	70000	\$15.00	320 Watt Pulse Start Metal-Halide	\$72.00	20000	\$6.67	\$109.35	40000	\$22.50
8-Lamp HPT8 w/ High-BF Ballast High-Bay	\$5.00	24000	\$6.67	\$32.50	70000	\$15.00	Proportionally Adjusted according to 6-Lamp HPT8 Equivalent to 320 PSMH	\$17.00	20000	\$6.67	\$109.35	40000	\$22.50
1-Lamp Relamp/Reballast T12 to HPT8	\$5.00	24000	\$2.67	\$32.50	70000	\$15.00	1-Lamp F34T12 w/ EEMag Ballast	\$2.70	20000	\$2.67	\$20.00	40000	\$15.00
2-Lamp Relamp/Reballast T12 to HPT8	\$5.00	24000	\$2.67	\$32.50	70000	\$15.00	2-Lamp F34T12 w/ EEMag Ballast	\$2.70	20000	\$2.67	\$20.00	40000	\$15.00
3-Lamp Relamp/Reballast T12 to HPT8	\$5.00	24000	\$2.67	\$32.50	70000	\$15.00	3-Lamp F34T12 w/ EEMag Ballast	\$2.70	20000	\$2.67	\$20.00	40000	\$15.00
4-Lamp Relamp/Reballast T12 to HPT8	\$5.00	24000	\$2.67	\$32.50	70000	\$15.00	4-Lamp F34T12 w/ EEMag Ballast	\$2.70	20000	\$2.67	\$20.00	40000	\$15.00
1-Lamp Relamp/Reballast T8 to HPT8	\$5.00	24000	\$2.67	\$32.50	70000	\$15.00	1-Lamp F32T8 w/ Elec. Ballast	\$2.70	20000	\$2.67	\$20.00	70000	\$15.00
2-Lamp Relamp/Reballast T8 to HPT8	\$5.00	24000	\$2.67	\$32.50	70000	\$15.00	2-Lamp F32T8 w/ Elec. Ballast	\$2.70	20000	\$2.67	\$20.00	70000	\$15.00
3-Lamp Relamp/Reballast T8 to HPT8	\$5.00	24000	\$2.67	\$32.50	70000	\$15.00	3-Lamp F32T8 w/ Elec. Ballast	\$2.70	20000	\$2.67	\$20.00	70000	\$15.00
4-Lamp Relamp/Reballast T8 to HPT8	\$5.00	24000	\$2.67	\$32.50	70000	\$15.00	4-Lamp F32T8 w/ Elec. Ballast	\$2.70	20000	\$2.67	\$20.00	70000	\$15.00
2-lamp High-Performance HPT8 Troffer	\$5.00	24000	\$2.67	\$32.50	70000	\$15.00	3-Lamp F32T8 w/ Elec. Ballast	\$2.50	20000	\$2.67	\$15.00	70000	\$15.00

⁶³³ Cost assumptions for baseline fixtures are based upon manufacturer specification sheets. Baseline and efficient measure cost data comes from lighting suppliers, past Efficiency Vermont projects, and professional judgment

B-3: Reduced Wattage T8 Component Costs and Lifetime⁶³⁴

EE measure description	EE Lamp Cost	EE Lamp Life (hrs)	Baseline Description	Base Lamp Cost	Base Lamp Life (hrs)	Base Lamp Rep. Labor Cost
RW T8 - F28T8 Lamp	\$4.50	30000	F32 T8 Standard Lamp	\$2.50	15000	\$2.67
RWT8 F2T8 Extra Life Lamp	\$4.50	36000	F32 T8 Standard Lamp	\$2.50	15000	\$2.67
RWT8 - F32/25W T8 Lamp	\$4.50	30000	F32 T8 Standard Lamp	\$2.50	15000	\$2.67
RWT8 - F32/25W T8 Lamp Extra Life	\$4.50	36000	F32 T8 Standard Lamp	\$2.50	15000	\$2.67
RWT8 F17T8 Lamp - 2 ft	\$4.80	18000	F17 T8 Standard Lamp - 2ft	\$2.80	15000	\$2.67
RWT8 F25T8 Lamp - 3 ft	\$5.10	18000	F25 T8 Standard Lamp - 3ft	\$3.10	15000	\$2.67
RWT8 F30T8 Lamp - 6' Utube	\$11.31	24000	F32 T8 Standard Utube	\$9.31	15000	\$2.67
RWT8 F29T8 Lamp - Utube	\$11.31	24000	F32 T8 Standard Utube	\$9.31	15000	\$2.67
RWT8 F96T8 Lamp - 8 ft	\$9.00	24000	F96 T8 Standard Lamp - 8 ft	\$7.00	15000	\$2.67

⁶³⁴ Cost assumptions for baseline fixtures are based upon manufacturer specification sheets. Baseline and efficient measure cost data comes from lighting suppliers, past Efficiency Vermont projects, and professional judgment.

C-1: T12 Baseline Adjustment:

For measures installed up to 1/1/2019, the full savings (as calculated above in the Algorithm section) will be claimed up to 1/1/2019. A savings adjustment will be applied to the annual savings for the remainder of the measure life. The adjustment to be applied for each measure is listed in the reference table below.

Savings Adjustment Factors

EE Measure Description	Savings Adjustment T12 EEmag ballast and 34 w lamps to HPT8	Savings Adjustment T12 EEmag ballast and 40 w lamps to HPT8	Savings Adjustment T12 mag ballast and 40 w lamps to HPT8
1-Lamp Relamp/Reballast T12 to HPT8	47%	30%	20%
2-Lamp Relamp/Reballast T12 to HPT8	53%	30%	22%
3-Lamp Relamp/Reballast T12 to HPT8	42%	38%	21%
4-Lamp Relamp/Reballast T12 to HPT8	44%	29%	23%

Measures installed in 2018 will claim full savings for one year. Savings adjustment factors will be applied to the full savings for savings starting in 1/1/2019 and for the remainder of the measure life. The savings adjustment is equal to the ratio between wattage reduction from T8 baseline to HPT8 and wattage reduction from T12 EE ballast with 40 w lamp baseline from the table 'T8 New and Baseline Assumptions'.⁶³⁵

Example: 2 lamp T8 to 2 lamp HPT8 retrofit saves 10 watts, while the T12 EE with 40 w lamp to HPT8 saves 33 watts. Thus the ratio of wattage reduced is 30%.

MEASURE CODE: CI-LTG-T8FX-V06-160601

REVIEW DEADLINE: 1/1/2019

⁶³⁵ See "HPRWT8_reference.xlsx" for more information.

EPE Program Downloads. Web accessed <http://www.electricefficiency.com/downloads.asp?section=ci> download Copy of LSF_2012_v4.04_250rows.xls.

Kuiken et al, Focus on Energy Evaluation. Business Programs: Deemed Savings Manual v1.0, Kema, march 22, 2010 available at http://www.focusonenergy.com/files/Document_Management_System/Evaluation/bpdeemedsavingsmanuav10_evaluationreport.pdf Based on ComEd's BILD program data from PY4 and PY5. For Residential installations, hours of use assumptions from '5.5.6 LED Downlights' should be used for LED fixtures and '5.5.8 LED Screw Based Omnidirectional Bulbs' should be used for LED bulbs.

4.5.4 LED Bulbs and Fixtures

DESCRIPTION

This characterization provides savings assumptions for a variety of LED lamps including Omnidirectional (e.g. A-Type lamps), Decorative (e.g. Globes and Torpedoes) and Directional (PAR Lamps, Reflectors, MR16), and fixtures including refrigerated case, recessed and outdoor/garage fixtures.

If the implementation strategy does not allow for the installation location to be known, for Residential targeted programs (e.g. an upstream retail program), a deemed split of 95% Residential and 5% Commercial assumptions should be used⁶³⁶, and for Commercial targeted programs a deemed split of 96% Commercial and 4% Residential should be used⁶³⁷.

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, new lamps must be ENERGY STAR labeled. Note a new ENERGY STAR specification v2.0 becomes effective on 1/2/2017

(https://www.energystar.gov/products/spec/lamps_specification_version_2_0_pd).

Lamps and fixtures should be found in the reference tables below. Fixtures must be ENERGY STAR labeled or on the Design Lights Consortium qualifying fixture list.

DEFINITION OF BASELINE EQUIPMENT

Refer to the baseline tables. In 2012, Federal legislation stemming from the Energy Independence and Security Act of 2007 (EIAS) required all general-purpose light bulbs between 40 watts and 100 watts to have ~30% increased efficiency, essentially phasing out standard incandescent technology. In 2012, the 100 w lamp standards apply; in 2013 the 75 w lamp standards will apply, followed by restrictions on the 60 w and 40 w lamps in 2014.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

Lifetime is the life of the product, at the reported operating hours (lamp life in hours divided by operating hours per year – see reference table "LED component Costs and Lifetime." The analysis period is the same as the lifetime, capped at 15 years. (15 years from GDS Measure Life Report, June 2007).

DEEMED MEASURE COST

Wherever possible, actual incremental costs should be used. Refer to reference table "LED component Cost & Lifetime" for defaults.

LOADSHAPE

Loadshape C06 - Commercial Indoor Lighting

Loadshape C07 - Grocery/Conv. Store Indoor Lighting

Loadshape C08 - Hospital Indoor Lighting

Loadshape C09 - Office Indoor Lighting

Loadshape C10 - Restaurant Indoor Lighting

⁶³⁶ RES v C&I split is based on a weighted (by sales volume) average of ComEd PY6, PY7 and PY8 and Ameren PY5, PY6 and PY8 in store intercept survey results. See 'RESvCI Split_112016.xls'.

⁶³⁷ Based on final ComEd's BILD program data from PY4, PY5 and PY6. For Residential installations, hours of use assumptions from '5.5.6 LED Downlights' should be used for LED fixtures and '5.5.8 LED Screw Based Omnidirectional Bulbs' should be used for LED bulbs.

Loadshape C11 - Retail Indoor Lighting
 Loadshape C12 - Warehouse Indoor Lighting
 Loadshape C13 - K-12 School Indoor Lighting
 Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)
 Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)
 Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)
 Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)
 Loadshape C18 - Industrial Indoor Lighting
 Loadshape C19 - Industrial Outdoor Lighting
 Loadshape C20 - Commercial Outdoor Lighting

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on the location type. Values are provided for each building type in the reference section below.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = ((Watts_{base} - Watts_{EE}) / 1000) * Hours * WHF_e * ISR$$

Where:

$Watts_{base}$ = Input wattage of the existing or baseline system. Reference the “LED New and Baseline Assumptions” table for default values.

$Watts_{EE}$ = Actual wattage of LED purchased / installed. If unknown, use default provided below:

For ENERGY STAR rated lamps the following lumen equivalence tables should be used:

Omnidirectional Lamps - ENERGY STAR Minimum Luminous Efficacy = 80lm/W for <90 CRI lamps and 70lm/W for ≥90 CRI lamps.

Minimum Lumens	Maximum Lumens	Lumens used to calculate LED Wattage (midpoint)	LED Wattage ⁶³⁸ (WattsEE)	Baseline 2014-2019 (WattsBase)	Delta Watts 2014-2019 (WattsEE)	Baseline Post EISA 2020 requirement ⁶³⁹ (WattsBase)	Delta Watts Post 2020 (WattsEE)
5280	6209	5745	72.9	300.0	227.1	300.0	227.1
3000	5279	4140	52.5	200.0	147.5	200.0	147.5
2601	2999	2800	35.5	150.0	114.5	150.0	114.5

⁶³⁸ Based on ENERGY STAR V2.0 specs – for omnidirectional <90CRI: 80 lm/W and for omnidirectional ≥90 CRI: 70 lm/W. To weight these two criteria, the ENERGY STAR qualified list was reviewed and found to contain 87.8% lamps <90CRI and 12.2% ≥90CRI.

⁶³⁹ Calculated as 45lm/W for all EISA non-exempt bulbs.

Minimum Lumens	Maximum Lumens	Lumens used to calculate LED Wattage (midpoint)	LED Wattage ⁶³⁸ (WattsEE)	Baseline 2014-2019 (WattsBase)	Delta Watts 2014-2019 (WattsEE)	Baseline Post EISA 2020 requirement ⁶³⁹ (WattsBase)	Delta Watts Post 2020 (WattsEE)
1490	2600	2045	26.0	72.0	46.0	45.4	19.5
1050	1489	1270	16.1	53.0	36.9	28.2	12.1
750	1049	900	11.4	43.0	31.6	20.0	8.6
310	749	530	6.7	29.0	22.3	11.8	5.0
250	309	280	3.5	25.0	21.5	25.0	21.5

Decorative Lamps - ENERGY STAR Minimum Luminous Efficacy = 65Lm/W for all lamps

Nominal wattage of lamp to be replaced (Watts _{base})	Minimum initial light output of LED lamp (lumens)	LED Wattage (Watts _{EE})	Delta Watts
10	70	1.08	8.92
15	90	1.38	13.6
25	150	2.31	22.7
40	300	4.62	35.4
60	500	7.69	52.3

Directional Lamps - ENERGY STAR Minimum Luminous Efficacy = 70Lm/W for <90 CRI lamps and 61 Lm/W for ≥90CRI lamps.

For Directional R, BR, and ER lamp types:

Bulb Type	Lower Lumen Range	Upper Lumen Range	Watts _{Base}	Lumens used to calculate LED Wattage (midpoint)	LED Wattage (Watts _{EE})	Delta Watts
R, ER, BR with medium screw bases w/ diameter >2.25" (*see exceptions below)	420	472	40	446	6.6	33.4
	473	524	45	499	7.3	37.7
	525	714	50	620	9.1	40.9
	715	937	65	826	12.1	52.9
	938	1259	75	1099	16.2	58.8
	1260	1399	90	1330	19.6	70.4
	1400	1739	100	1570	23.1	76.9
	1740	2174	120	1957	28.8	91.2
	2175	2624	150	2400	35.3	114.7
	2625	2999	175	2812	41.3	133.7
*R, BR, and ER with medium screw bases w/	3000	4500	200	3750	55.1	144.9
	400	449	40	425	6.2	33.8
	450	499	45	475	7.0	38.0
	500	649	50	575	8.5	41.5
	650	1199	65	925	13.6	51.4

Bulb Type	Lower Lumen Range	Upper Lumen Range	Watts _{Base}	Lumens used to calculate LED Wattage (midpoint)	LED Wattage (Watts _{EE})	Delta Watts
diameter <=2.25"						
*ER30, BR30, BR40, or ER40	400	449	40	425	6.2	33.8
	450	499	45	475	7.0	38.0
	500	649	50	575	8.5	41.5
*BR30, BR40, or ER40	650	1419	65	1035	15.2	49.8
*R20	400	449	40	425	6.2	33.8
	450	719	45	585	8.6	36.4
*All reflector lamps below lumen ranges specified above	200	299	20	250	3.7	16.3
	300	399	30	350	5.1	24.9

Directional lamps are exempt from EISA regulations.

For PAR, MR, and MRX Lamps Types:

For these highly focused directional lamp types, it is necessary to have Center Beam Candle Power (CBCP) and beam angle measurements to accurately estimate the equivalent baseline wattage. The formula below is based on the Energy Star Center Beam Candle Power tool.⁶⁴⁰ If CBCP and beam angle information are not available or if the equation below returns a negative value (or undefined), use the manufacturer's recommended baseline wattage equivalent.⁶⁴¹

Watts_{base} =

$$375.1 - 4.355(D) - \sqrt{227,800 - 937.9(D) - 0.9903(D^2) - 1479(BA) - 12.02(D * BA) + 14.69(BA^2) - 16,720 * \ln(CBCP)}$$

Where:

D = Bulb diameter (e.g. for PAR20 D = 20)

BA = Beam angle

CBCP = Center beam candle power

The result of the equation above should be rounded DOWN to the nearest wattage established by Energy Star:

Diameter	Permitted Wattages
16	20, 35, 40, 45, 50, 60, 75
20	50
30S	40, 45, 50, 60, 75
30L	50, 75
38	40, 45, 50, 55, 60, 65, 75, 85, 90, 100, 120, 150, 250

⁶⁴⁰ <http://energystar.supportportal.com/link/portal/23002/23018/Article/32655/>

⁶⁴¹ The Energy Star Center Beam Candle Power tool does not accurately model baseline wattages for lamps with certain bulb characteristic combinations – specifically for lamps with very high CBCP.

Hours	= Average hours of use per year are provided in the Reference Table in Section 4.5, Screw based bulb annual operating hours, for each building type. If unknown, use the Miscellaneous value.
WHFe	= Waste heat factor for energy to account for cooling energy savings from efficient lighting are provided below for each building type in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.
ISR	= In Service Rate -the percentage of units rebated that actually get installed. =100% ⁶⁴² if application form completed with sign off that equipment is not placed into storage. If sign off form not completed assume the following 3 year ISR assumptions:

Weighted Average 1st year In Service Rate (ISR)	2nd year Installations	3rd year Installations	Final Lifetime In Service Rate
95.7% ⁶⁴³	1.2%	1.1%	98.0% ⁶⁴⁴

HEATING PENALTY

If electrically heated building:

$$\Delta \text{kWh}_{\text{heatpenalty}}^{645} = (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * -\text{IFkWh})$$

Where:

IFkWh	= Lighting-HVAC Interaction Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.
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For example, For example, a 9W LED lamp, 450 lumens, is installed in a heat pump heated office in 2014 and sign off form provided:

$$\begin{aligned} \Delta \text{kWh}_{\text{heatpenalty}} &= ((29 - 6.7) / 1000) * 1.0 * 3088 * -0.151 \\ &= - 10.4 \text{ kWh} \end{aligned}$$

⁶⁴² Illinois evaluation of PY1 through PY3 has not found that fixtures or lamps placed into storage to be a significant enough issue to warrant including an "In-Service Rate" when commercial customers complete an application form.

⁶⁴³ Based on ComEd's BILD program data from PY5 and PY6, see "IL Commercial Lighting ISR_2014.xls".

⁶⁴⁴ In the absence of any data for LEDs specifically it is assumed that the same proportion of bulbs eventually get installed as for CFLs. The 98% CFL assumption is based upon review of two evaluations:

'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings. Note that this Final Install Rate does NOT account for leakage of purchased bulbs being installed outside of the utility territory. EM&V should assess how and if data from evaluation should adjust this final installation rate to account for this impact

⁶⁴⁵ Negative value because this is an increase in heating consumption due to the efficient lighting.

DEFERRED INSTALLS

As presented above, if a sign off form is not completed the characterization assumes that a percentage of bulbs purchased are not installed until Year 2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

Year 1 (Purchase Year) installs: Characterized using assumptions provided above or evaluated assumptions if available.

Year 2 and 3 installs: Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.

The NTG factor for the Purchase Year should be applied.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = ((\text{Watts}_{\text{base}} - \text{Watts}_{\text{EE}}) / 1000) * \text{ISR} * \text{WHF}_d * \text{CF}$$

Where:

WHF_d = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is provided in Referecne Table in Section 4.5. If unknown, use the Miscellaneous value.

CF = Summer Peak Coincidence Factor for measure is provided in the Referecne Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, a 9W LED lamp, 450 lumens, is installed in an office in 2014 and sign off form provided:

$$\begin{aligned} \Delta kW &= ((29 - 6.7) / 1000) * 1.0 * 1.3 * 0.66 \\ &= 0.019 \text{ kW} \end{aligned}$$

NATURAL GAS ENERGY SAVINGS

Heating Penalty if fossil fuel heated building (or if heating fuel is unknown):

$$\Delta \text{Therms} = (((\text{Watts}_{\text{Base}} - \text{Watts}_{\text{EE}}) / 1000) * \text{ISR} * \text{Hours} * - \text{IFTherms})$$

Where:

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Referecne Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, For example, a 9W LED lamp, 450 lumens, is installed in an office in 2014 and sign off form provided:

$$\begin{aligned} \Delta \text{Therms} &= ((29 - 6.7) / 1000) * 1.0 * 3088 * -0.016 \\ &= -1.10 \text{ therms} \end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

For all measures except Standard Omnidirectional lamps (which have an EISA baseline shift) the individual component lifetimes and costs are provided in the reference table section below⁶⁴⁶.

In order to account for the falling EISA Qualified bulb replacement cost provided above, an equivalent annual levelized baseline replacement cost over the lifetime of the LED bulb (assumed to be 15,000/3612 = 4.2 years) is calculated⁶⁴⁷. The key assumptions used in this calculation are documented below⁶⁴⁸:

	Std Inc.	EISA Compliant Halogen	CFL
2017	\$0.43	\$1.25	N/A
2018	\$0.43	\$1.25	N/A
2019	\$0.43	\$1.25	N/A
2020 & after	\$0.43	N/A	\$2.45

The NPV for replacement lamps and annual levelized replacement costs using the societal real discount rate of 0.46% are presented below. It is important to note that for cost-effectiveness screening purposes, the O&M cost adjustments should only be applied in cases where the light bulbs are actually in service and so should be multiplied by the appropriate ISR:

Location	Lumen Level	NPV of replacement costs for period			Levelized annual replacement cost savings		
		2018	2019	2020	2018	2019	2020
Commercial	Lumens <310 or >2600 (EISA exempt)	\$6.02	\$6.02	\$6.02	\$1.47	\$1.47	\$1.47
	Lumens ≥ 310 and ≤ 2600 (EISA compliant)	\$13.22	\$9.64	\$6.04	\$3.22	\$2.35	\$1.47
Multi Family Common Areas	Lumens <310 or >2600 (non-EISA compliant)	\$5.92	\$5.92	\$5.92	\$2.37	\$2.37	\$2.37
	Lumens ≥ 310 and ≤ 2600 (EISA compliant)	\$17.20	\$14.25	\$8.32	\$6.88	\$5.70	\$3.33

For halogen bulbs, we assume the same replacement cycle as incandescent bulbs.⁶⁴⁹ The replacement cycle is based on the miscellaneous hours of use. Both incandescent and halogen lamps are assumed to last for 1,000 hours before needing replacement and CFLs after 10,000 hours.

REFERENCE TABLES**LED Bulb Assumptions**

Wherever possible, actual incremental costs should be used. If unavailable assume the following incremental costs⁶⁵⁰:

⁶⁴⁶ See "LED Lighting Systems TRM Reference Tables" for breakdown of component cost assumptions.

⁶⁴⁷ See C&I OmniDirectional LED O&M Calc_012017.xls" for more information. The values assume the non-residential average hours assumption of 3612.

⁶⁴⁸ Based upon pricing forecast developed by Applied Proactive Technologies Inc (APT) based on industry input and provided to Ameren.

⁶⁴⁹ The manufacturers of the new minimally compliant EISA Halogens are using regular incandescent lamps with halogen fill gas rather than halogen infrared to meet the standard and so the component rated life is equal to the standard incandescent.

⁶⁵⁰ Baseline and LED lamp costs are based on field data collected by CLEAResult and provided by ComEd. See ComEd Pricing Projections 06302016.xlsx for analysis. Given LED prices are expected to continue declining assumed costs should be reassessed on an annual basis and replaced with IL specific LED program information when available.

Bulb Type	Year	LED	Incandescent	Incremental Cost
Omnidirectional	2017	\$3.21	\$1.25	\$1.96
	2018	\$3.21		\$1.96
	2019	\$3.11		\$1.86
Directional	2017	\$6.24	\$3.53	\$2.71
	2018-2019	\$5.18		\$1.65
Decorative and Globe	2017	\$3.50	\$1.60	\$1.90
	2018-2019	\$3.40	\$1.74	\$1.66

Directional and Decorative O&M; apply incandescent cost assumption provided above with a frequency calculated by dividing the assumed rated life of the baseline bulb (1000 hours) by the building specific hours of use assumption

LED Fixture Wattage and Incremental Cost Assumptions⁶⁵¹

LED Category	EE Measure Description	Watts _{EE}	Baseline Description	Watts _{BASE}	Incremental Cost	Mid Life Savings Adjustment (2018)
LED Downlight Fixtures	LED Recessed, Surface, Pendant Downlights	17.6	Baseline LED Recessed, Surface, Pendant Downlights	54.3	\$27	N/A
LED Interior Directional	LED Track Lighting	12.2	Baseline LED Track Lighting	60.4	\$59	N/A
	LED Wall-Wash Fixtures	8.3	Baseline LED Wall-Wash Fixtures	17.7	\$59	N/A
LED Display Case	LED Display Case Light Fixture	7.1 per ft	Baseline LED Display Case Light Fixture	36.2 per ft	\$11/ft	N/A
	LED Undercabinet Shelf-Mounted Task Light Fixtures	7.1 per ft	Baseline LED Undercabinet Shelf-Mounted Task Light Fixtures	36.2 per ft	\$11/ft	N/A
	LED Refrigerated Case Light, Horizontal or Vertical	7.6 per ft	Baseline LED Refrigerated Case Light, Horizontal or Vertical (per foot)	15.2 per ft	\$11/ft	N/A
	LED Freezer Case Light, Horizontal or Vertical	7.7 per ft	Baseline LED Freezer Case Light, Horizontal or Vertical (per foot)	18.7 per ft	\$11/ft	N/A
LED Linear Replacement Lamps	LED 4' Linear Replacement Lamp	18.7	80:20 T12:T8; Lamp Only 32w T8:34w T12	33.6	\$24	89%
	LED 2' Linear Replacement Lamp	9.7	80:20 T12:T8; Lamp Only 17w T8:20w T12	19.4	\$13	75%
LED Troffers	LED 2x2 Recessed Light Fixture, 2000-3500 lumens	34.1	80:20 T12:Standard T8 2-Lamp 32w T8, 2-Lamp 34w T12	61.0	\$48	85%
	LED 2x2 Recessed Light Fixture, 3501-5000 lumens	42.8	80:20 T12:Standard T8 3-Lamp 32w T8, 3-Lamp 34w T12	103.3	\$91	69%
	LED 2x4 Recessed Light Fixture, 3000-4500 lumens	37.9	80:20 T12:Standard T8 2-Lamp 32w T8, 2-Lamp 34w T12	61.0	\$62	83%

⁶⁵¹ Watt, lumen, lamp life, and ballast factor assumptions for efficient measures are based upon Consortium for Energy Efficiency (CEE) Commercial Lighting Qualifying Product Lists alongside past Efficiency Vermont projects and PGE refrigerated case study. Watt, lumen, lamp life, and ballast factor assumptions for baseline fixtures are based upon manufacturer specification sheets. Baseline cost data comes from lighting suppliers, past Efficiency Vermont projects, and professional judgment. Efficient cost data comes from 2012 DOE "Energy Savings Potential of Solid-State Lighting in General Illumination Applications", Table A.1. See "LED Lighting Systems TRM Reference Tables.xlsx" for more information and specific product links.

LED Category	EE Measure Description	Watts _{EE}	Baseline Description	Watts _{BASE}	Incremental Cost	Mid Life Savings Adjustment (2018)
	LED 2x4 Recessed Light Fixture, 4501-6000 lumens	54.3	80:20 T12:Standard T8 3-Lamp 32w T8, 3-Lamp 34w T12	103.3	\$99	62%
	LED 2x4 Recessed Light Fixture, 6001-7500 lumens	72.7	80:20 T12:Standard T8 4-Lamp 32w T8, 4-Lamp 34w T12	137.7	\$150	61%
	LED 1x4 Recessed Light Fixture, 1500-3000 lumens	18.1	80:20 T12:Standard T8 1-Lamp 32w T8, 1-Lamp 34w T12	30.6	\$36	88%
	LED 1x4 Recessed Light Fixture, 3001-4500 lumens	39.6	80:20 T12:Standard T8 2-Lamp 32w T8, 2-Lamp 34w T12	61.0	\$76	81%
	LED 1x4 Recessed Light Fixture, 4501-6000 lumens	53.1	80:20 T12:Standard T8 3-Lamp 32w T8, 3-Lamp 34w T12	103.3	\$130	62%
LED Linear Ambient Fixtures	LED Surface & Suspended Linear Fixture, <= 3000 lumens	19.7	80:20 T12:Standard T8 1-Lamp 32w T8, 1-Lamp 34w T12	30.6	\$54	86%
	LED Surface & Suspended Linear Fixture, 3001-4500 lumens	37.8	80:20 T12:Standard T8 2-Lamp 32w T8, 2-Lamp 34w T12	61.0	\$104	83%
	LED Surface & Suspended Linear Fixture, 4501-6000 lumens	55.9	80:20 T12:Standard T8 3-Lamp 32w T8, 3-Lamp 34w T12	103.3	\$158	60%
	LED Surface & Suspended Linear Fixture, 6001-7500 lumens	62.6	T5HO 2L-F54T5HO - 4'	120.0	\$215	N/A
	LED Surface & Suspended Linear Fixture, > 7500 lumens	95.4	T5HO 3L-F54T5HO - 4'	180.0	\$374	N/A
LED High & Low Bay Fixtures	LED Low-Bay Fixtures, <= 10,000 lumens	90.3	3-Lamp T8HO Low-Bay	157.0	\$191	N/A
	LED High-Bay Fixtures, 10,001-15,000 lumens	127.5	4-Lamp T8HO High-Bay	196.0	\$331	N/A
	LED High-Bay Fixtures, 15,001-20,000 lumens	191.0	6-Lamp T8HO High-Bay	294.0	\$482	N/A
	LED High-Bay Fixtures, > 20,000 lumens	249.7	8-Lamp T8HO High-Bay	392.0	\$818	N/A
LED Agricultural Interior Fixtures	LED Ag Interior Fixtures, <= 2,000 lumens	17.0	25% 73 Watt EISA Inc, 75% 1L T8	42.0	\$33	N/A
	LED Ag Interior Fixtures, 2,001-4,000 lumens	27.8	25% 146 Watt EISA Inc, 75% 2L T8	81.0	\$54	N/A
	LED Ag Interior Fixtures, 4,001-6,000 lumens	51.2	25% 217 Watt EISA Inc, 75% 3L T8	121.0	\$125	N/A
	LED Ag Interior Fixtures, 6,001-8,000 lumens	71.7	25% 292 Watt EISA Inc, 75% 4L T8	159.0	\$190	N/A
	LED Ag Interior Fixtures, 8,001-12,000 lumens	103.5	200W Pulse Start Metal Halide	227.3	\$298	N/A
	LED Ag Interior Fixtures, 12,001-16,000 lumens	143.8	320W Pulse Start Metal Halide	363.6	\$450	N/A
	LED Ag Interior Fixtures, 16,001-20,000 lumens	183.3	350W Pulse Start Metal Halide	397.7	\$595	N/A

LED Category	EE Measure Description	Watts _{EE}	Baseline Description	Watts _{BASE}	Incremental Cost	Mid Life Savings Adjustment (2018)
	LED Ag Interior Fixtures, > 20,000 lumens	305.0	(2) 320W Pulse Start Metal Halide	727.3	\$998	N/A
LED Exterior Fixtures	LED Exterior Fixtures, <= 5,000 lumens	42.6	100W Metal Halide	113.6	\$190	N/A
	LED Exterior Fixtures, 5,001-10,000 lumens	68.2	175W Pulse Start Metal Halide	198.9	\$287	N/A
	LED Exterior Fixtures, 10,001-15,000 lumens	122.5	250W Pulse Start Metal Halide	284.1	\$391	N/A
	LED Exterior Fixtures, > 15,000 lumens	215.0	400W Pulse Start Metal Halide	454.5	\$793	N/A

LED Fixture Component Costs & Lifetime⁶⁵²

LED Category	EE Measure Description	EE Measure				Baseline			
		Lamp Life (hrs)	Total Lamp Replacement Cost	LED Driver Life (hrs)	Total LED Driver Replacement Cost	Lamp Life (hrs)	Total Lamp Replacement Cost	Ballast Life (hrs)	Total Ballast Replacement Cost
LED Downlight Fixtures	LED Recessed, Surface, Pendant Downlights	50,000	\$30.75	70,000	\$47.50	2,500	\$8.86	40,000	\$14.40
LED Interior Directional	LED Track Lighting	50,000	\$39.00	70,000	\$47.50	2,500	\$12.71	40,000	\$11.00
	LED Wall-Wash Fixtures	50,000	\$39.00	70,000	\$47.50	2,500	\$9.17	40,000	\$27.00
LED Display Case	LED Display Case Light Fixture	50,000	\$9.75/ft	70,000	\$11.88/ft	2,500	\$6.70	40,000	\$5.63
	LED Undercabinet Shelf-Mounted Task Light Fixtures	50,000	\$9.75/ft	70,000	\$11.88/ft	2,500	\$6.70	40,000	\$5.63
	LED Refrigerated Case Light, Horizontal or Vertical	50,000	\$8.63/ft	70,000	\$9.50/ft	15,000	\$1.13	40,000	\$8.00
	LED Freezer Case Light, Horizontal or Vertical	50,000	\$7.88/ft	70,000	\$7.92/ft	12,000	\$0.94	40,000	\$6.67
LED Linear Replacement Lamps	LED 4' Linear Replacement Lamp	50,000	\$8.57	70,000	\$13.67	24,000	\$6.17	40,000	\$11.96
	LED 2' Linear Replacement Lamp	50,000	\$5.76	70,000	\$13.67	24,000	\$6.17	40,000	\$11.96
LED Troffers	LED 2x2 Recessed Light Fixture, 2000-3500 lumens	50,000	\$46.68	70,000	\$40.00	24,000	\$26.33	40,000	\$35.00

⁶⁵² Note some measures have blended baselines (T12:T8 80:20). All values are provided to enable calculation of appropriate O&M impacts. Total costs include lamp, labor and disposal cost assumptions where applicable, see "IL LED Lighting Systems TRM Tables" for more information.

LED Category	EE Measure Description	EE Measure				Baseline			
		Lamp Life (hrs)	Total Lamp Replacement Cost	LED Driver Life (hrs)	Total LED Driver Replacement Cost	Lamp Life (hrs)	Total Lamp Replacement Cost	Ballast Life (hrs)	Total Ballast Replacement Cost
	LED 2x2 Recessed Light Fixture, 3501-5000 lumens	50,000	\$56.31	70,000	\$40.00	24,000	\$39.50	40,000	\$35.00
	LED 2x4 Recessed Light Fixture, 3000-4500 lumens	50,000	\$49.58	70,000	\$40.00	24,000	\$12.33	40,000	\$35.00
	LED 2x4 Recessed Light Fixture, 4501-6000 lumens	50,000	\$57.76	70,000	\$40.00	24,000	\$18.50	40,000	\$35.00
	LED 2x4 Recessed Light Fixture, 6001-7500 lumens	50,000	\$68.89	70,000	\$40.00	24,000	\$24.67	40,000	\$35.00
	LED 1x4 Recessed Light Fixture, 1500-3000 lumens	50,000	\$43.43	70,000	\$40.00	24,000	\$6.17	40,000	\$35.00
	LED 1x4 Recessed Light Fixture, 3001-4500 lumens	50,000	\$52.31	70,000	\$40.00	24,000	\$12.33	40,000	\$35.00
	LED 1x4 Recessed Light Fixture, 4501-6000 lumens	50,000	\$63.86	70,000	\$40.00	24,000	\$18.50	40,000	\$35.00
LED Linear Ambient Fixtures	LED Surface & Suspended Linear Fixture, <= 3000 lumens	50,000	\$45.01	70,000	\$40.00	24,000	\$6.17	40,000	\$35.00
	LED Surface & Suspended Linear Fixture, 3001-4500 lumens	50,000	\$58.73	70,000	\$40.00	24,000	\$12.33	40,000	\$35.00
	LED Surface & Suspended Linear Fixture, 4501-6000 lumens	50,000	\$73.50	70,000	\$40.00	24,000	\$18.50	40,000	\$35.00
	LED Surface & Suspended Linear Fixture, 6001-7500 lumens	50,000	\$88.69	70,000	\$40.00	30,000	\$26.33	40,000	\$60.00
	LED Surface & Suspended Linear Fixture, > 7500 lumens	50,000	\$123.91	70,000	\$40.00	30,000	\$39.50	40,000	\$60.00
LED High & Low Bay Fixtures	LED Low-Bay Fixtures, <= 10,000 lumens	50,000	\$90.03	70,000	\$62.50	18,000	\$64.50	40,000	\$92.50
	LED High-Bay Fixtures, 10,001-15,000 lumens	50,000	\$122.59	70,000	\$62.50	18,000	\$86.00	40,000	\$92.50

LED Category	EE Measure Description	EE Measure				Baseline			
		Lamp Life (hrs)	Total Lamp Replacement Cost	LED Driver Life (hrs)	Total LED Driver Replacement Cost	Lamp Life (hrs)	Total Lamp Replacement Cost	Ballast Life (hrs)	Total Ballast Replacement Cost
	LED High-Bay Fixtures, 15,001-20,000 lumens	50,000	\$157.22	70,000	\$62.50	18,000	\$129.00	40,000	\$117.50
	LED High-Bay Fixtures, > 20,000 lumens	50,000	\$228.52	70,000	\$62.50	18,000	\$172.00	40,000	\$142.50
LED Agricultural Interior Fixtures	LED Ag Interior Fixtures, <= 2,000 lumens	50,000	\$37.00	70,000	\$40.00	18,250	\$1.23	40,000	\$26.25
	LED Ag Interior Fixtures, 2,001-4,000 lumens	50,000	\$44.96	70,000	\$40.00	18,250	\$1.43	40,000	\$26.25
	LED Ag Interior Fixtures, 4,001-6,000 lumens	50,000	\$63.02	70,000	\$40.00	18,250	\$1.62	40,000	\$26.25
	LED Ag Interior Fixtures, 6,001-8,000 lumens	50,000	\$79.78	70,000	\$40.00	18,250	\$1.81	40,000	\$26.25
	LED Ag Interior Fixtures, 8,001-12,000 lumens	50,000	\$119.91	70,000	\$62.50	15,000	\$63.00	40,000	\$112.50
	LED Ag Interior Fixtures, 12,001-16,000 lumens	50,000	\$151.89	70,000	\$62.50	15,000	\$68.00	40,000	\$122.50
	LED Ag Interior Fixtures, 16,001-20,000 lumens	50,000	\$184.62	70,000	\$62.50	15,000	\$73.00	40,000	\$132.50
	LED Ag Interior Fixtures, > 20,000 lumens	50,000	\$285.75	70,000	\$62.50	15,000	\$136.00	40,000	\$202.50
LED Exterior Fixtures	LED Exterior Fixtures, <= 5,000 lumens	50,000	\$86.92	70,000	\$62.50	15,000	\$58.00	40,000	\$102.50
	LED Exterior Fixtures, 5,001-10,000 lumens	50,000	\$111.81	70,000	\$62.50	15,000	\$63.00	40,000	\$112.50
	LED Exterior Fixtures, 10,001-15,000 lumens	50,000	\$138.32	70,000	\$62.50	15,000	\$68.00	40,000	\$122.50
	LED Exterior Fixtures, > 15,000 lumens	50,000	\$223.67	70,000	\$62.50	15,000	\$73.00	40,000	\$132.50

MEASURE CODE: CI-LTG-LEDB-V06-180101

REVIEW DEADLINE: 1/1/2019

4.5.5 Commercial LED Exit Signs

DESCRIPTION

This measure characterizes the savings associated with installing a Light Emitting Diode (LED) exit sign in place of a fluorescent or incandescent exit sign in a Commercial building. Light Emitting Diode exit signs have a string of very small, typically red or green, glowing LEDs arranged in a circle or oval. The LEDs may also be arranged in a line on the side, top or bottom of the exit sign. LED exit signs provide the best balance of safety, low maintenance, and very low energy usage compared to other exit sign technologies.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is assumed to be an exit sign illuminated by LEDs.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be a fluorescent or incandescent model.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 16 years⁶⁵³.

DEEMED MEASURE COST

The incremental cost for this measure is assumed to be \$30⁶⁵⁴.

LOADSHAPE

Loadshape C53 - Flat

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 100%⁶⁵⁵.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{HOURS} * \text{WHF}_e$$

Where:

WattsBase = Actual wattage if known, if unknown assume the following:

Baseline Type	WattsBase
Incandescent	35W ⁶⁵⁶

⁶⁵³ 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, "Effective/Remaining Useful Life Values", California Public Utilities Commission, December 16, 2008.

⁶⁵⁴ NYSERDA Deemed Savings Database, Labor cost assumes 25 minutes @ \$18/hr.

⁶⁵⁵ Assuming continuous operation of an LED exit sign, the Summer Peak Coincidence Factor is assumed to equal 1.0.

⁶⁵⁶ Based on review of available product.

Baseline Type	WattsBase
Fluorescent	11W ⁶⁵⁷
Unknown (e.g. time of sale)	23W ⁶⁵⁸

WattsEE = Actual wattage if known, if unknown assume 2W⁶⁵⁹

HOURS = Annual operating hours
= 8766

WHF_e = Waste heat factor for energy to account for cooling energy savings from efficient lighting are provided for each building type in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, replacing incandescent fixture in an office

$$\Delta kWH = (35 - 2)/1000 * 8766 * 1.25$$

$$= 362 \text{ kWh}$$

For example, replacing fluorescent fixture in a hospital

$$\Delta kWH = (11 - 2)/1000 * 8766 * 1.35$$

$$= 106.5 \text{ kWh}$$

HEATING PENALTY

If electrically heated building:

$$\Delta kWH_{\text{heatpenalty}}^{660} = (((\text{WattsBase} - \text{WattsEE})/1000) * \text{Hours} * -\text{IFkWh})$$

Where:

IFkWh = Lighting-HVAC Interaction Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, replacing incandescent fixture in a heat pump heated office

$$\Delta kWH_{\text{heatpenalty}} = (35 - 2)/1000 * 8766 * -0.151$$

$$= -43.7 \text{ kWh}$$

For example, replacing fluorescent fixture in a heat pump heated hospital

$$\Delta kWH_{\text{heatpenalty}} = (11 - 2)/1000 * 8766 * -0.104$$

$$= -8.2 \text{ kWh}$$

⁶⁵⁷ Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, February, 19, 2010

⁶⁵⁸ ComEd has been using a weighted baseline of 70 percent incandescent and 30 percent compact fluorescent, reflecting program experience and a limited sample of evaluation verification findings that we consider to be reasonable (Navigant, through comment period February 2013)

⁶⁵⁹ Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, February, 19, 2010

⁶⁶⁰ Negative value because this is an increase in heating consumption due to the efficient lighting.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{WHF}_d * \text{CF}$$

Where:

WHF_d = Waste heat factor for demand to account for cooling savings from efficient lighting in cooled buildings is provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value..

CF = Summer Peak Coincidence Factor for measure
= 1.0

For example, replacing incandescent fixture in an office

$$\begin{aligned}\Delta kW &= (35 - 2)/1000 * 1.3 * 1.0 \\ &= 0.043 \text{ kW}\end{aligned}$$

For example, replacing fluorescent fixture in a hospital

$$\begin{aligned}\Delta kW &= (11 - 2)/1000 * 1.69 * 1.0 \\ &= 0.015 \text{ kW}\end{aligned}$$

NATURAL GAS SAVINGS

Heating Penalty if natural gas heated building (or if heating fuel is unknown):

$$\Delta \text{therms} = (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{Hours} * \text{IFTherms})$$

Where:

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

For example, replacing incandescent fixture in an office

$$\begin{aligned}\Delta \text{Therms} &= (35 - 2)/1000 * 8766 * -0.016 \\ &= -4.63 \text{ Therms}\end{aligned}$$

For example, replacing fluorescent fixture in a hospital

$$\begin{aligned}\Delta \text{Therms} &= (11 - 2)/1000 * 8766 * -0.011 \\ &= -0.87 \text{ Therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

The annual O&M Cost Adjustment savings should be calculated using the following component costs and lifetimes.

Component	Baseline Measures	
	Cost	Life (yrs)

	Baseline Measures	
Lamp	\$7.00 ⁶⁶¹	1.37 years ⁶⁶²

MEASURE CODE: CI-LTG-LEDE-V02-140601

REVIEW DEADLINE: 1/1/2021

⁶⁶¹ Consistent with assumption for a Standard CFL bulb with an estimated labor cost of \$4.50 (assuming \$18/hour and a task time of 15 minutes).

⁶⁶² Assumes a lamp life of 12,000 hours and 8766 run hours $12000/8766 = 1.37$ years.

4.5.6 LED Traffic and Pedestrian Signals

DESCRIPTION

Traffic and pedestrian signals are retrofitted to be illuminated with light emitting diodes (LED) instead of incandescent lamps. Incentive applies for the replacement or retrofit of existing incandescent traffic signals with new LED traffic and pedestrian signal lamps. Each lamp can have no more than a maximum LED module wattage of 25. Incentives are not available for spare lights. Lights must be hardwired and single lamp replacements are not eligible, with the exception of pedestrian hand signals. Eligible lamps must meet the Energy Star Traffic Signal Specification and the Institute for Transportation Engineers specification for traffic signals.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

Refer to the Table titled 'Traffic Signals Technology Equivalencies' for efficient technology wattage and savings assumptions.

DEFINITION OF BASELINE EQUIPMENT

Refer to the Table titled 'Traffic Signals Technology Equivalencies' for baseline efficiencies and savings assumptions.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of an LED traffic signal is 100,000 hours (manufacturer's estimate), capped at 10 years.⁶⁶³ The life in years is calculated by dividing 100,000 hrs by the annual operating hours for the particular signal type.

DEEMED MEASURE COST

The actual measure installation cost should be used (including material and labor).

LOADSHAPE

Loadshape C24 - Traffic Signal - Red Balls, always changing or flashing

Loadshape C25 - Traffic Signal - Red Balls, changing day, off night

Loadshape C26 - Traffic Signal - Green Balls, always changing

Loadshape C27 - Traffic Signal - Green Balls, changing day, off night

Loadshape C28 - Traffic Signal - Red Arrows

Loadshape C29 - Traffic Signal - Green Arrows

Loadshape C30 - Traffic Signal - Flashing Yellows

Loadshape C31 - Traffic Signal - "Hand" Don't Walk Signal

Loadshape C32 - Traffic Signal - "Man" Walk Signal

Loadshape C33 - Traffic Signal - Bi-Modal Walk/Don't Walk

COINCIDENCE FACTOR⁶⁶⁴

The summer peak coincidence factor (CF) for this measure is dependent on lamp type as below:

Lamp Type	CF
Red Round, always changing or flashing	0.55

⁶⁶³ ACEEE, (1998) A Market Transformation Opportunity Assessment for LED Traffic Signals, <http://www.cee1.org/gov/led/led-ace3/ace3led.pdf>

⁶⁶⁴ Ibid

Lamp Type	CF
Red Arrows	0.90
Green Arrows	0.10
Yellow Arrows	0.03
Green Round, always changing or flashing	0.43
Flashing Yellow	0.50
Yellow Round, always changing	0.02
"Hand" Don't Walk Signal	0.75
"Man" Walk Signal	0.21

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = (W_{\text{base}} - W_{\text{eff}}) \times \text{HOURS} / 1000$$

Where:

W_{base}	=The connected load of the baseline equipment = see Table 'Traffic Signals Technology Equivalencies'
W_{eff}	=The connected load of the baseline equipment = see Table 'Traffic Signals Technology Equivalencies'
EFLH	= annual operating hours of the lamp = see Table 'Traffic Signals Technology Equivalencies'
1000	= conversion factor (W/kW)

EXAMPLE

For example, an 8 inch red, round signal:

$$\begin{aligned} \Delta \text{kWh} &= ((69 - 7) \times 4818) / 1000 \\ &= 299 \text{ kWh} \end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = (W_{\text{base}} - W_{\text{eff}}) \times \text{CF} / 1000$$

Where:

W_{base}	=The connected load of the baseline equipment = see Table 'Traffic Signals Technology Equivalencies'
W_{eff}	=The connected load of the efficient equipment = see Table 'Traffic Signals Technology Equivalencies'
CF	= Summer Peak Coincidence Factor for measure

EXAMPLE

For example, an 8 inch red, round signal:

$$\begin{aligned}\Delta kW &= ((69 - 7) \times 0.55) / 1000 \\ &= 0.0341 \text{ kW}\end{aligned}$$

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

REFERENCE TABLES

Traffic Signals Technology Equivalencies⁶⁶⁵

Traffic Fixture Type	Fixture Size and Color	Efficient Lamps	Baseline Lamps	HOURS	Efficient Fixture Wattage	Baseline Fixture Wattage	Energy Savings (in kWh)
Round Signals	8" Red	LED	Incandescent	4818	7	69	299
Round Signals	12" Red	LED	Incandescent	4818	6	150	694
Flashing Signal ⁶⁶⁶	8" Red	LED	Incandescent	4380	7	69	272
Flashing Signal	12" Red	LED	Incandescent	4380	6	150	631
Flashing Signal	8" Yellow	LED	Incandescent	4380	10	69	258
Flashing Signal	12" Yellow	LED	Incandescent	4380	13	150	600
Round Signals	8" Yellow	LED	Incandescent	175	10	69	10
Round Signals	12" Yellow	LED	Incandescent	175	13	150	24
Round Signals	8" Green	LED	Incandescent	3767	9	69	266
Round Signals	12" Green	LED	Incandescent	3767	12	150	520
Turn Arrows	8" Yellow	LED	Incandescent	701	7	116	76
Turn Arrows	12" Yellow	LED	Incandescent	701	9	116	75
Turn Arrows	8" Green	LED	Incandescent	701	7	116	76
Turn Arrows	12" Green	LED	Incandescent	701	7	116	76
Pedestrian Sign	12" Hand/Man	LED	Incandescent	8766	8	116	946

Reference specifications for above traffic signal wattages are from the following manufacturers:

1. 8" Incandescent traffic signal bulb: General Electric Traffic Signal Model 17325-69A21/TS
2. 12" Incandescent traffic signal bulb: General Electric Signal Model 35327-150PAR46/TS
3. Incandescent Arrows & Hand/Man Pedestrian Signs: General Electric Traffic Signal Model 19010-116A21/TS
4. 8" and 12" LED traffic signals: Leotek Models TSL-ES08 and TSL-ES12
5. 8" LED Yellow Arrow: General Electric Model DR4-YTA2-01A
6. 8" LED Green Arrow: General Electric Model DR4-GCA2-01A
7. 12" LED Yellow Arrow: Dialight Model 431-3334-001X
8. 12" LED Green Arrow: Dialight Model 432-2324-001X
9. LED Hand/Man Pedestrian Sign: Dialight 430-6450-001X

⁶⁶⁵ Technical Reference Manual for Pennsylvania Act 129 Energy Efficiency and Conservation Program and Act 213 Alternative Energy Portfolio Standards. Pennsylvania Public Utility Commission. May 2009

⁶⁶⁶ Technical Reference Manual for Ohio, August 6, 2010

MEASURE CODE: CI-LTG-LEDT-V01-120601

REVIEW DEADLINE: 1/1/2019

4.5.7 Lighting Power Density

DESCRIPTION

This measure relates to installation of efficient lighting systems in new construction or substantial renovation of commercial buildings excluding low rise (three stories or less) residential buildings. Substantial renovation is when two or more building systems are renovated, such as shell and heating, heating and lighting, etc. State Energy Code specifies a lighting power density level by building type for both the interior and the exterior. Either the Building Area Method or Space by Space method as defined in IECC 2012 or 2015, depending on the IECC in effect on the date of the building permit (if unknown assume IECC 2015), can be used for calculating the Interior Lighting Power Density⁶⁶⁷. The measure consists of a design that is more efficient (has a lower lighting power density in watts/square foot) than code requires. The IECC applies to both new construction and renovation.

This measure was developed to be applicable to the following program types: NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the lighting system must be more efficient than the baseline Energy Code lighting power density in watts/square foot for either the interior space or exterior space.

DEFINITION OF BASELINE EQUIPMENT

The baseline is assumed to be a lighting power density that meets IECC 2012 or 2015, depending on the IECC in effect on the date of the building permit (if unknown assume IECC 2015)..

DEEMED CALCULATION FOR THIS MEASURE

Annual kWh Savings

$$\Delta \text{kWh} = (\text{WSF}_{\text{base}} - \text{WSF}_{\text{eff}}) / 1000 * \text{SF} * \text{Hours} * \text{WHF}_e$$

Summer Coincident Peak kW Savings

$$\Delta \text{kW} = (\text{WSF}_{\text{base}} - \text{WSF}_{\text{eff}}) / 1000 * \text{SF} * \text{CF} * \text{WHF}_d$$

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years⁶⁶⁸

DEEMED MEASURE COST

The actual incremental cost over a baseline system will be collected from the customer if possible or developed on a fixture by fixture basis.

LOADSHAPE

Loadshape C06 - Commercial Indoor Lighting

Loadshape C07 - Grocery/Conv. Store Indoor Lighting

Loadshape C08 - Hospital Indoor Lighting

Loadshape C09 - Office Indoor Lighting

Loadshape C10 - Restaurant Indoor Lighting

⁶⁶⁷ Refer to the referenced code documents for specifics on calculating lighting power density using either the whole building method (IECC) or the Space by Space method (ASHRAE 90.1).

⁶⁶⁸ Measure Life Report, Residential and Commercial/Industrial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

Loadshape C11 - Retail Indoor Lighting
 Loadshape C12 - Warehouse Indoor Lighting
 Loadshape C13 - K-12 School Indoor Lighting
 Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)
 Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)
 Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)
 Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)
 Loadshape C18 - Industrial Indoor Lighting
 Loadshape C19 - Industrial Outdoor Lighting
 Loadshape C20 - Commercial Outdoor Lighting

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on the building type.

Algorithm

CALCULATION OF SAVINGS

ENERGY SAVINGS

$$\Delta \text{kWh} = (\text{WSF}_{\text{base}} - \text{WSF}_{\text{effic}}) / 1000 * \text{SF} * \text{Hours} * \text{WHF}_e$$

Where:

WSF_{base} = Baseline lighting watts per square foot or linear foot as determined by building or space type. Whole building analysis values are presented in the Reference Tables below.⁶⁶⁹

$\text{WSF}_{\text{effic}}$ = The actual installed lighting watts per square foot or linear foot.

SF = Provided by customer based on square footage of the building area applicable to the lighting design for new building.

Hours = Annual site-specific hours of operation of the lighting equipment collected from the customer. If not available, use building area type as provided in the Reference Table in Section 4.5, Fixture annual operating hours.

WHF_e = Waste Heat Factor for Energy to account for cooling savings from efficient lighting is as provided in the Reference Table in Section 4.5 by building type. If building is not cooled WHF_e is 1.

HEATING PENALTY

If electrically heated building:

$$\Delta \text{kWh}_{\text{heatpenalty}}^{670} = (\text{WSF}_{\text{base}} - \text{WSF}_{\text{effic}}) / 1000 * \text{SF} * \text{Hours} * -\text{IFkWh}$$

Where:

IFkWh = Lighting-HVAC Interaction Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected

⁶⁶⁹See IECC 2012 and 2015 - Reference Code documentation for additional information.

⁶⁷⁰Negative value because this is an increase in heating consumption due to the efficient lighting.

by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = (WSF_{base} - WSF_{effic}) / 1000 * SF * CF * WHF_d$$

Where:

WHF_d = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is as provided in the Reference Table in Section 4.5 by building type. If building is not cooled WHF_d is 1.

CF = Summer Peak Coincidence Factor for measure is as provided in the Reference Table in Section 4.5 by building type. If the building type is unknown, use the Miscellaneous value of 0.66.

Other factors as defined above

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms} = (WSF_{base} - WSF_{effic}) / 1000 * SF * \text{Hours} * \text{IFTerms}$$

Where:

IFTerms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting. This value is provided in the Reference Table in Section 4.5 by building type.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

REFERENCE TABLES

Lighting Power Density Values from IECC 2012 and 2015 for Interior Commercial New Construction and Substantial Renovation Building Area Method:

Building Area Type ⁶⁷¹	IECC 2012 Lighting Power Density (w/ft ²)	IECC 2015 Lighting Power Density (w/ft ²)
Automotive Facility	0.9	0.80
Convention Center	1.2	1.01
Court House	1.2	1.01
Dining: Bar Lounge/Leisure	1.3	1.01
Dining: Cafeteria/Fast Food	1.4	0.9
Dining: Family	1.6	0.95
Dormitory	1.0	0.57
Exercise Center	1.0	0.84
Fire station	0.8	0.67
Gymnasium	1.1	0.94

⁶⁷¹ In cases where both a general building area type and a more specific building area type are listed, the more specific building area type shall apply.

Building Area Type ⁶⁷¹	IECC 2012 Lighting Power Density (w/ft ²)	IECC 2015 Lighting Power Density (w/ft ²)
Healthcare – clinic	1.0	0.90
Hospital	1.2	1.05
Hotel	1.0	0.87
Library	1.3	1.19
Manufacturing Facility	1.3	1.17
Motel	1.0	0.87
Motion Picture Theater	1.2	0.76
Multifamily	0.7	0.51
Museum	1.1	1.02
Office	0.9	0.82
Parking Garage	0.3	0.21
Penitentiary	1.0	0.81
Performing Arts Theater	1.6	1.39
Police Station	1.0	0.87
Post Office	1.1	0.87
Religious Building	1.3	1.0
Retail ⁶⁷²	1.4	1.26
School/University	1.2	0.87
Sports Arena	1.1	0.91
Town Hall	1.1	0.89
Transportation	1.0	0.70
Warehouse	0.6	0.66
Workshop	1.4	1.19

⁶⁷² Where lighting equipment is specified to be installed to highlight specific merchandise in addition to lighting equipment specified for general lighting and is switched or dimmed on circuits different from the circuits for general lighting, the small of the actual wattage of the lighting equipment installed specifically for merchandise, or additional lighting power as determined below shall be added to the interior lighting power determined in accordance with this line item.

Lighting Power Density Values from IECC 2012 for Interior Commercial New Construction and Substantial Renovation Space by Space Method:

COMMERCIAL ENERGY EFFICIENCY

**TABLE C405.5.2(2)
INTERIOR LIGHTING POWER ALLOWANCES:
SPACE-BY-SPACE METHOD**

COMMON SPACE-BY-SPACE TYPES	LPD (w/ft ²)
Atrium – First 40 feet in height	0.03 per ft. ht.
Atrium – Above 40 feet in height	0.02 per ft. ht.
Audience/seating area – permanent	
For auditorium	0.9
For performing arts theater	2.6
For motion picture theater	1.2
Classroom/lecture/training	1.30
Conference/meeting/multipurpose	1.2
Corridor/transition	0.7
Dining area	
Bar/lounge/leisure dining	1.40
Family dining area	1.40
Dressing/fitting room performing arts theater	1.1
Electrical/mechanical	1.10
Food preparation	1.20
Laboratory for classrooms	1.3
Laboratory for medical/industrial/research	1.8
Lobby	1.10
Lobby for performing arts theater	3.3
Lobby for motion picture theater	1.0
Locker room	0.80
Lounge recreation	0.8
Office – enclosed	1.1
Office – open plan	1.0
Restroom	1.0
Sales area	1.6 ^a
Stairway	0.70
Storage	0.8
Workshop	1.60
Courthouse/police station/penitentiary	
Courtroom	1.90
Confinement cells	1.1
Judge chambers	1.30
Penitentiary audience seating	0.5
Penitentiary classroom	1.3
Penitentiary dining	1.1
BUILDING SPECIFIC SPACE-BY-SPACE TYPES	
Automotive – service/repair	0.70
Bank/office – banking activity area	1.5
Dormitory living quarters	1.10
Gymnasium/fitness center	
Fitness area	0.9
Gymnasium audience/seating	0.40
Playing area	1.40

(continued)

**TABLE C405.5.2(2)—continued
INTERIOR LIGHTING POWER ALLOWANCES:
SPACE-BY-SPACE METHOD**

COMMON SPACE-BY-SPACE TYPES	LPD (w/ft ²)
Healthcare clinic/hospital	
Corridors/transition	1.00
Exam/treatment	1.70
Emergency	2.70
Public and staff lounge	0.80
Medical supplies	1.40
Nursery	0.9
Nurse station	1.00
Physical therapy	0.90
Patient room	0.70
Pharmacy	1.20
Radiology/imaging	1.3
Operating room	2.20
Recovery	1.2
Lounge/recreation	0.8
Laundry – washing	0.60
Hotel	
Dining area	1.30
Guest rooms	1.10
Hotel lobby	2.10
Highway lodging dining	1.20
Highway lodging guest rooms	1.10
Library	
Stacks	1.70
Card file and cataloguing	1.10
Reading area	1.20
Manufacturing	
Corridors/transition	0.40
Detailed manufacturing	1.3
Equipment room	1.0
Extra high bay (> 50-foot floor-ceiling height)	1.1
High bay (25- – 50-foot floor-ceiling height)	1.20
Low bay (< 25-foot floor-ceiling height)	1.2
Museum	
General exhibition	1.00
Restoration	1.70
Parking garage – garage areas	0.2
Convention center	
Exhibit space	1.50
Audience/seating area	0.90
Fire stations	
Engine room	0.80
Sleeping quarters	0.30
Post office	
Sorting area	0.9
Religious building	
Fellowship hall	0.60
Audience seating	2.40
Worship pulpit/choir	2.40
Retail	
Dressing/fitting area	0.9
Mall concourse	1.6
Sales area	1.6 ^a

(continued)

TABLE C405.5.2(2)—continued
INTERIOR LIGHTING POWER ALLOWANCES:
SPACE-BY-SPACE METHOD

BUILDING SPECIFIC SPACE-BY-SPACE TYPES	LPD (w/ft²)
Sports arena	
Audience seating	0.4
Court sports area – Class 4	0.7
Court sports area – Class 3	1.2
Court sports area – Class 2	1.9
Court sports area – Class 1	3.0
Ring sports area	2.7
Transportation	
Air/train/bus baggage area	1.00
Airport concourse	0.60
Terminal – ticket counter	1.50
Warehouse	
Fine material storage	1.40
Medium/bulky material	0.60

Lighting Power Density Values from IECC 2015 for Interior Commercial New Construction and Substantial Renovation
Space by Space Method:TABLE C405.4.2(2)
INTERIOR LIGHTING POWER ALLOWANCES:
SPACE-BY-SPACE METHOD

COMMON SPACE TYPES ^a	LPD (watts/sq.ft)
Atrium	
Less than 40 feet in height	0.03 per foot in total height
Greater than 40 feet in height	$0.40 + 0.02$ per foot in total height
Audience seating area	
In an auditorium	0.63
In a convention center	0.82
In a gymnasium	0.65
In a motion picture theater	1.14
In a penitentiary	0.28
In a performing arts theater	2.43
In a religious building	1.53
In a sports arena	0.43
Otherwise	0.43
Banking activity area	1.01
Breakroom (See Lounge/Breakroom)	
Classroom/lecture hall/training room	
In a penitentiary	1.34
Otherwise	1.24
Conference/meeting/multipurpose room	1.23
Copy/print room	0.72
Corridor	
In a facility for the visually impaired (and not used primarily by the staff) ^b	0.92
In a hospital	0.79
In a manufacturing facility	0.41
Otherwise	0.66
Courtroom	1.72
Computer room	1.71
Dining area	
In a penitentiary	0.96
In a facility for the visually impaired (and not used primarily by the staff) ^b	1.9
In bar/lounge or leisure dining	1.07
In cafeteria or fast food dining	0.65
In family dining	0.89
Otherwise	0.65
Electrical/mechanical room	0.95
Emergency vehicle garage	0.56

(continued)

TABLE C405.4.2(2)—continued
INTERIOR LIGHTING POWER ALLOWANCES:
SPACE-BY-SPACE METHOD

COMMON SPACE TYPES ^a	LPD (watts/sq.ft)
Food preparation area	1.21
Guest room	0.47
Laboratory	
In or as a classroom	1.43
Otherwise	1.81
Laundry/washing area	0.6
Loading dock, interior	0.47
Lobby	
In a facility for the visually impaired (and not used primarily by the staff) ^b	1.8
For an elevator	0.64
In a hotel	1.06
In a motion picture theater	0.59
In a performing arts theater	2.0
Otherwise	0.9
Locker room	0.75
Lounge/breakroom	
In a healthcare facility	0.92
Otherwise	0.73
Office	
Enclosed	1.11
Open plan	0.98
Parking area, interior	0.19
Pharmacy area	1.68
Restroom	
In a facility for the visually impaired (and not used primarily by the staff) ^b	1.21
Otherwise	0.98
Sales area	1.59
Seating area, general	0.54
Stairway (See space containing stairway)	
Stairwell	0.69
Storage room	0.63
Vehicular maintenance area	0.67
Workshop	1.59
BUILDING TYPE SPECIFIC SPACE TYPES ^a	LPD (watts/sq.ft)
Facility for the visually impaired ^b	
In a chapel (and not used primarily by the staff)	2.21
In a recreation room (and not used primarily by the staff)	2.41
Automotive (See Vehicular Maintenance Area above)	
Convention Center—exhibit space	1.45
Dormitory—living quarters	0.38
Fire Station—sleeping quarters	0.22
Gymnasium/fitness center	
In an exercise area	0.72
In a playing area	1.2

(continued)

**TABLE C405.4.2(2)—continued
INTERIOR LIGHTING POWER ALLOWANCES:
SPACE-BY-SPACE METHOD**

BUILDING TYPE SPECIFIC SPACE TYPES ^a	LPD (watts/sq.ft)
healthcare facility	
In an exam/treatment room	1.66
In an imaging room	1.51
In a medical supply room	0.74
In a nursery	0.88
In a nurse's station	0.71
In an operating room	2.48
In a patient room	0.62
In a physical therapy room	0.91
In a recovery room	1.15
Library	
In a reading area	1.06
In the stacks	1.71
Manufacturing facility	
In a detailed manufacturing area	1.29
In an equipment room	0.74
In an extra high bay area (greater than 50' floor-to-ceiling height)	1.05
In a high bay area (25-50' floor-to-ceiling height)	1.23
In a low bay area (less than 25' floor-to-ceiling height)	1.19
Museum	
In a general exhibition area	1.05
In a restoration room	1.02
Performing arts theater—dressing room	0.61
Post Office—Sorting Area	0.94
Religious buildings	
In a fellowship hall	0.64
In a worship/pulpit/choir area	1.53
Retail facilities	
In a dressing/fitting room	0.71
In a mall concourse	1.1
Sports arena—playing area	
For a Class I facility	3.68
For a Class II facility	2.4
For a Class III facility	1.8
For a Class IV facility	1.2
Transportation facility	
In a baggage/carousel area	0.53
In an airport concourse	0.36
At a terminal ticket counter	0.8
Warehouse—storage area	
For medium to bulky, palletized items	0.58
For smaller, hand-carried items	0.95

- a. In cases where both a common space type and a building area specific space type are listed, the building area specific space type shall apply.
- b. A 'Facility for the Visually Impaired' is a facility that is licensed or will be licensed by local or state authorities for senior long-term care, adult daycare, senior support or people with special visual needs.

The exterior lighting design will be based on the building location and the applicable “Lighting Zone” as defined in IECC 2015 Table C405.5.2(1) which follows. This table is identical to IECC 2012 Table C405.62(1).

TABLE C405.5.2(1)
EXTERIOR LIGHTING ZONES

LIGHTING ZONE	DESCRIPTION
1	Developed areas of national parks, state parks, forest land, and rural areas
2	Areas predominantly consisting of residential zoning, neighborhood business districts, light industrial with limited nighttime use and residential mixed-use areas
3	All other areas not classified as lighting zone 1, 2 or 4
4	High-activity commercial districts in major metropolitan areas as designated by the local land use planning authority

The lighting power density savings will be based on reductions below the allowable design levels as specified in IECC 2012 Table C405.6.2(2) or IECC 2015 Table C405.5.2(2).

Allowable Design Levels from IECC 2012

TABLE C405.6.2(2)
INDIVIDUAL LIGHTING POWER ALLOWANCES FOR BUILDING EXTERIORS

		LIGHTING ZONES			
		Zone 1	Zone 2	Zone 3	Zone 4
Base Site Allowance (Base allowance is usable in tradable or nontradable surfaces.)		500 W	600 W	750 W	1300 W
Tradable Surfaces (Lighting power densities for uncovered parking areas, building grounds, building entrances and exits, canopies and overhangs and outdoor sales areas are tradable.)	Uncovered Parking Areas				
	Parking areas and drives	0.04 W/ft ²	0.06 W/ft ²	0.10 W/ft ²	0.13 W/ft ²
	Building Grounds				
	Walkways less than 10 feet wide	0.7 W/linear foot	0.7 W/linear foot	0.8 W/linear foot	1.0 W/linear foot
	Walkways 10 feet wide or greater, plaza areas, special feature areas	0.14 W/ft ²	0.14 W/ft ²	0.16 W/ft ²	0.2 W/ft ²
	Stairways	0.75 W/ft ²	1.0 W/ft ²	1.0 W/ft ²	1.0 W/ft ²
	Pedestrian tunnels	0.15 W/ft ²	0.15 W/ft ²	0.2 W/ft ²	0.3 W/ft ²
	Building Entrances and Exits				
	Main entries	20 W/linear foot of door width	20 W/linear foot of door width	30 W/linear foot of door width	30 W/linear foot of door width
	Other doors	20 W/linear foot of door width	20 W/linear foot of door width	20 W/linear foot of door width	20 W/linear foot of door width
	Entry canopies	0.25 W/ft ²	0.25 W/ft ²	0.4 W/ft ²	0.4 W/ft ²
	Sales Canopies				
	Free-standing and attached	0.6 W/ft ²	0.6 W/ft ²	0.8 W/ft ²	1.0 W/ft ²
	Outdoor Sales				
	Open areas (including vehicle sales lots)	0.25 W/ft ²	0.25 W/ft ²	0.5 W/ft ²	0.7 W/ft ²
	Street frontage for vehicle sales lots in addition to "open area" allowance	No allowance	10 W/linear foot	10 W/linear foot	30 W/linear foot
Nontradable Surfaces (Lighting power density calculations for the following applications can be used only for the specific application and cannot be traded between surfaces or with other exterior lighting. The following allowances are in addition to any allowance otherwise permitted in the "Tradable Surfaces" section of this table.)	Building facades	No allowance	0.1 W/ft ² for each illuminated wall or surface or 2.5 W/linear foot for each illuminated wall or surface length	0.15 W/ft ² for each illuminated wall or surface or 3.75 W/linear foot for each illuminated wall or surface length	0.2 W/ft ² for each illuminated wall or surface or 5.0 W/linear foot for each illuminated wall or surface length
	Automated teller machines and night depositories	270 W per location plus 90 W per additional ATM per location	270 W per location plus 90 W per additional ATM per location	270 W per location plus 90 W per additional ATM per location	270 W per location plus 90 W per additional ATM per location
	Entrances and gatehouse inspection stations at guarded facilities	0.75 W/ft ² of covered and uncovered area	0.75 W/ft ² of covered and uncovered area	0.75 W/ft ² of covered and uncovered area	0.75 W/ft ² of covered and uncovered area
	Loading areas for law enforcement, fire, ambulance and other emergency service vehicles	0.5 W/ft ² of covered and uncovered area	0.5 W/ft ² of covered and uncovered area	0.5 W/ft ² of covered and uncovered area	0.5 W/ft ² of covered and uncovered area
	Drive-up windows/doors	400 W per drive-through	400 W per drive-through	400 W per drive-through	400 W per drive-through
	Parking near 24-hour retail entrances	800 W per main entry	800 W per main entry	800 W per main entry	800 W per main entry

For SI: 1 foot = 304.8 mm, 1 watt per square foot = W/0.0929 m².

Allowable Design Levels from IECC 2015

TABLE C405.5.2(2)
INDIVIDUAL LIGHTING POWER ALLOWANCES FOR BUILDING EXTERIORS

		LIGHTING ZONES			
		Zone 1	Zone 2	Zone 3	Zone 4
Base Site Allowance (Base allowance is usable in tradable or nontradable surfaces.)		500 W	600 W	750 W	1300 W
Tradable Surfaces (Lighting power densities for uncovered parking areas, building grounds, building entrances and exits, canopies and overhangs and outdoor sales areas are tradable.)	Uncovered Parking Areas				
	Parking areas and drives	0.04 W/ft ²	0.06 W/ft ²	0.10 W/ft ²	0.13 W/ft ²
	Building Grounds				
	Walkways less than 10 feet wide	0.7 W/linear foot	0.7 W/linear foot	0.8 W/linear foot	1.0 W/linear foot
	Walkways 10 feet wide or greater, plaza areas special feature areas	0.14 W/ft ²	0.14 W/ft ²	0.16 W/ft ²	0.2 W/ft ²
	Stairways	0.75 W/ft ²	1.0 W/ft ²	1.0 W/ft ²	1.0 W/ft ²
	Pedestrian tunnels	0.15 W/ft ²	0.15 W/ft ²	0.2 W/ft ²	0.3 W/ft ²
	Building Entrances and Exits				
	Main entries	20 W/linear foot of door width	20 W/linear foot of door width	30 W/linear foot of door width	30 W/linear foot of door width
	Other doors	20 W/linear foot of door width	20 W/linear foot of door width	20 W/linear foot of door width	20 W/linear foot of door width
	Entry canopies	0.25 W/ft ²	0.25 W/ft ²	0.4 W/ft ²	0.4 W/ft ²
	Sales Canopies				
	Free-standing and attached	0.6 W/ft ²	0.6 W/ft ²	0.8 W/ft ²	1.0 W/ft ²
	Outdoor Sales				
	Open areas (including vehicle sales lots)	0.25 W/ft ²	0.25 W/ft ²	0.5 W/ft ²	0.7 W/ft ²
	Street frontage for vehicle sales lots in addition to "open area" allowance	No allowance	10 W/linear foot	10 W/linear foot	30 W/linear foot
Nontradable Surfaces (Lighting power density calculations for the following applications can be used only for the specific application and cannot be traded between surfaces or with other exterior lighting. The following allowances are in addition to any allowance otherwise permitted in the "Tradable Surfaces" section of this table.)	Building facades	No allowance	0.075 W/ft ² of gross above-grade wall area	0.113 W/ft ² of gross above-grade wall area	0.15 W/ft ² of gross above-grade wall area
	Automated teller machines (ATM) and night depositories	270 W per location plus 90 W per additional ATM per location	270 W per location plus 90 W per additional ATM per location	270 W per location plus 90 W per additional ATM per location	270 W per location plus 90 W per additional ATM per location
	Entrances and gatehouse inspection stations at guarded facilities	0.75 W/ft ² of covered and uncovered area	0.75 W/ft ² of covered and uncovered area	0.75 W/ft ² of covered and uncovered area	0.75 W/ft ² of covered and uncovered area
	Loading areas for law enforcement, fire, ambulance and other emergency service vehicles	0.5 W/ft ² of covered and uncovered area	0.5 W/ft ² of covered and uncovered area	0.5 W/ft ² of covered and uncovered area	0.5 W/ft ² of covered and uncovered area
	Drive-up windows/doors	400 W per drive-through	400 W per drive-through	400 W per drive-through	400 W per drive-through
	Parking near 24-hour retail entrances	800 W per main entry	800 W per main entry	800 W per main entry	800 W per main entry

For SI: 1 foot = 304.8 mm, 1 watt per square foot = W/0.0929 m².
W = watts.

MEASURE CODE: CI-LTG-LPDE-V03-160601

REVIEW DEADLINE: 1/1/2020

4.5.8 Miscellaneous Commercial/Industrial Lighting

DESCRIPTION

This measure is designed to calculate savings from energy efficient lighting upgrades that are not captured in other measures within the TRM. If a lighting project fits the measure description in sections 4.5.1-4.5.4, then those criteria, definitions, and calculations should be used.

Unlike other lighting measures this one applies only to RF applications (because there is no defined baseline for TOS or NC applications).

DEFINITION OF EFFICIENT EQUIPMENT

A lighting fixture that replaces an existing fixture to provide the same or greater lumen output at a lower kW consumption.

DEFINITION OF BASELINE EQUIPMENT

The definition of baseline equipment is the existing lighting fixture.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The deemed lifetime of the efficient equipment fixture, regardless of program type is 15 years⁶⁷³.

DEEMED MEASURE COST

The actual cost of the efficient light fixture should be used.

LOADSHAPE

- Loadshape C06 - Commercial Indoor Lighting
- Loadshape C07 - Grocery/Conv. Store Indoor Lighting
- Loadshape C08 - Hospital Indoor Lighting
- Loadshape C09 - Office Indoor Lighting
- Loadshape C10 - Restaurant Indoor Lighting
- Loadshape C11 - Retail Indoor Lighting
- Loadshape C12 - Warehouse Indoor Lighting
- Loadshape C13 - K-12 School Indoor Lighting
- Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)
- Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)
- Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)
- Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)
- Loadshape C18 - Industrial Indoor Lighting
- Loadshape C19 - Industrial Outdoor Lighting
- Loadshape C20 - Commercial Outdoor Lighting

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on the location type. Values are provided for each building type in section 4.5.

⁶⁷³ 15 years from GDS Measure Life Report, June 2007

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta kWh = ((Watts_{base} - Watts_{EE}) / 1000) * Hours * WHF_e * ISR$$

Where:

Watts_{base} = Input wattage of the existing system which depends on the baseline fixture configuration (number and type of lamp) and ballast factor (if applicable) and number of fixtures.

=Actual

Watts_{EE} = New Input wattage of EE fixture which depends on new fixture configuration (number of lamps) and ballast factor (if applicable) (if applicable) and number of fixtures.

= Actual

Hours = Average hours of use per year as provided by the customer or selected from the Reference Table in Section 4.5, Fixture annual operating hours, by building type. If hours or building type are unknown, use the Miscellaneous value.

WHF_e = Waste heat factor for energy to account for cooling energy savings from efficient lighting is selected from the Reference Table in Section 4.5 for each building type. If building is un-cooled, the value is 1.0.

ISR = In Service Rate or the percentage of units rebated that get installed.

=100%⁶⁷⁴ if application form completed with sign off that equipment is not placed into storage. If sign off form not completed assume the following 3 year ISR assumptions:

Weighted Average 1 st year In Service Rate (ISR)	2 nd year Installations	3 rd year Installations	Final Lifetime In Service Rate
75.5% ⁶⁷⁵	12.1%	10.3%	98.0% ⁶⁷⁶

HEATING PENALTY

If electrically heated building:

$$\Delta kWh_{heatpenalty}^{677} = (((Watts_{Base} - Watts_{EE}) / 1000) * ISR * Hours * -IFkWh$$

Where:

⁶⁷⁴Illinois evaluation of PY1 through PY3 has not found that fixtures or lamps placed into storage to be a significant enough issue to warrant including an "In-Service Rate" when commercial customers complete an application form.

⁶⁷⁵ 1st year in service rate is based upon review of PY4-5 evaluations from ComEd's commercial lighting program (BILD) (see 'IL Commercial Lighting ISR.xls' for more information. The average first year ISR was calculated weighted by the number of bulbs sold.

⁶⁷⁶ The 98% Lifetime ISR assumption is based upon review of two evaluations:

'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings.

⁶⁷⁷Negative value because this is an increase in heating consumption due to the efficient lighting.

IFkWh = Lighting-HVAC Interaction Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

DEFERRED INSTALLS

As presented above, if a sign off form is not completed the characterization assumes that a percentage of bulbs purchased are not installed until Year 2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

Year 1 (Purchase Year) installs: Characterized using assumptions provided above or evaluated assumptions if available.

Year 2 and 3 installs: Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.

The NTG factor for the Purchase Year should be applied.

SUMMER COINCIDENT DEMAND SAVINGS

$$\Delta kW = ((\text{Watts}_{\text{base}} - \text{Watts}_{\text{EE}}) / 1000) * \text{WHF}_d * \text{CF} * \text{ISR}$$

Where:

WHFd = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is selected from the Reference Table in Section 4.5 for each building type. If the building is not cooled WHFd is 1.

CF = Summer Peak Coincidence Factor for measure is selected from the Reference table in Section 4.5 for each building type. If the building type is unknown, use the Miscellaneous value of 0.66.

Other factors as defined above

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms}^{678} = (((\text{Watts}_{\text{Base}} - \text{Watts}_{\text{EE}}) / 1000) * \text{ISR} * \text{Hours} * - \text{IFTherms}$$

Where:

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting. This value is selected from the Reference Table in Section 6.5 for each building type.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

If there are differences between the maintenance of the efficient and baseline lighting system then they should be evaluated on a project-by-project basis.

⁶⁷⁸Negative value because this is an increase in heating consumption due to the efficient lighting.

MEASURE CODE: CI-LTG-MSCI-V02-140601

REVIEW DEADLINE: 1/1/2021

4.5.9 Multi-Level Lighting Switch

DESCRIPTION

This measure relates to the installation new multi-level lighting switches on an existing lighting system.

This measure can only relate to the adding of a new control in an existing building, since multi-level switching is required in the Commercial new construction building energy code (IECC 2012/2015).

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient system is assumed to be a lighting system controlled by multi-level lighting controls.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be an uncontrolled lighting system where all lights in a given area are on the same circuit or all circuits come on at the same time.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life for all lighting controls is assumed to be 8 years⁶⁷⁹.

DEEMED MEASURE COST

When available, the actual cost of the measure shall be used. When not available, the incremental capital cost for this measure is assumed to be \$274⁶⁸⁰.

LOADSHAPE

- Loadshape C06 - Commercial Indoor Lighting
- Loadshape C07 - Grocery/Conv. Store Indoor Lighting
- Loadshape C08 - Hospital Indoor Lighting
- Loadshape C09 - Office Indoor Lighting
- Loadshape C10 - Restaurant Indoor Lighting
- Loadshape C11 - Retail Indoor Lighting
- Loadshape C12 - Warehouse Indoor Lighting
- Loadshape C13 - K-12 School Indoor Lighting
- Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)
- Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)
- Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)
- Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)
- Loadshape C18 - Industrial Indoor Lighting
- Loadshape C19 - Industrial Outdoor Lighting
- Loadshape C20 - Commercial Outdoor Lighting

⁶⁷⁹ Consistent with Occupancy Sensor control measure.

⁶⁸⁰ Goldberg et al, State of Wisconsin Public Service Commission of Wisconsin, Focus on Energy Evaluation, Business Programs: Incremental Cost Study, KEMA, October 28, 2009.

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on the location type. Values are provided for each building type in the reference section below.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta kWh = KW_{Controlled} * Hours * ESF * WHF_e$$

Where:

$KW_{Controlled}$ = Total lighting load connected to the control in kilowatts.

= Actual

Hours = total operating hours of the controlled lighting circuit before the lighting controls are installed. This number should be collected from the customer. Average hours of use per year are provided in the Reference Table in Section 4.5, Fixture annual operating hours, for each building type if customer specific information is not collected. If unknown building type, use the Miscellaneous value.

ESF = Energy Savings factor (represents the percentage reduction to the $KW_{controlled}$ due to the use of multi-level switching).

= Dependent on building type⁶⁸¹:

Building Type	Energy Savings Factor (ESF)
Private Office	21.6%
Open Office	16.0%
Retail	14.8%
Classrooms	8.3%
Unknown, average	15%

WHF_e = Waste heat factor for energy to account for cooling energy savings from efficient lighting is provided in the Reference Table in Section 4.5 for each building type. If building is un-cooled, the value is 1.0.

HEATING PENALTY

If electrically heated building:

$$\Delta kWh_{heatpenalty}^{682} = KW_{Controlled} * Hours * ESF * -IFkWh$$

Where:

IFkWh = Lighting-HVAC Interaction Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected

⁶⁸¹ Based on results from "Lighting Controls Effectiveness Assessment: Final Report on Bi-Level Lighting Study" published by the California Public Utilities Commission (CPUC), prepared by ADM Associates.

<http://lightingcontrolsassociation.org/bi-level-switching-study-demonstrates-energy-savings/>

⁶⁸² Negative value because this is an increase in heating consumption due to the efficient lighting.

by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = KW_{\text{controlled}} * ESF * WHF_d * CF$$

Where:

WHF_d = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is provided in the Reference Table in Section 4.5. If the building is un-cooled WHF_d is 1.

CF = Summer Peak Coincidence Factor for measure is provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value of 0.66⁶⁸³.

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{therms} = KW_{\text{controlled}} * \text{Hours} * ESF * - IF\text{Therms}$$

Where:

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting and provided in the Reference Table in Section 4.5 by building type.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-LTG-MLLC-V03-160601

REVIEW DEADLINE: 1/1/2021

⁶⁸³ By applying the ESF and the same coincidence factor for general lighting savings we are in essence assuming that the savings from multi-level switching are as likely during peak periods as any other time. In the absence of better information this seems like a reasonable assumption and if anything may be on the conservative side since you might expect the peak periods to be generally sunnier and therefore more likely to have lower light levels. It is also consistent with the control type reducing the wattage lighting load, the same as the general lighting measures.

4.5.10 Occupancy Sensor Lighting Controls

DESCRIPTION

This measure relates to the installation of new occupancy sensors on a new or existing lighting system. Lighting control types covered by this measure include wall, ceiling or fixture mounted occupancy sensors. Passive infrared, ultrasonic detectors and fixture-mounted sensors or sensors with a combination thereof are eligible. Lighting controls required by state energy codes are not eligible. This must be a new installation and may not replace an existing lighting occupancy sensor control.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the existing system is assumed to be manually controlled or an uncontrolled lighting system which is being controlled by one of the lighting controls systems listed above. This measure is intended for controlling interior lighting only.

A subset of occupancy sensors are those that are programmed as “vacancy” sensors. To qualify as a vacancy sensor, the control must be configured such that manual input is required to turn on the controlled lighting and the control automatically turns the lighting off. Additional savings are achieved compared to standard occupancy sensors because lighting does not automatically turn on and occupants may decide to not turn it on. Note that vacancy sensors are not a viable option for many applications where standard occupancy sensors should be used instead.

DEFINITION OF BASELINE EQUIPMENT

The baseline is assumed to be a lighting system uncontrolled by occupancy.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life for all lighting controls is assumed to be 8 years⁶⁸⁴.

DEEMED MEASURE COST

When available, the actual cost of the measure shall be used. When not available, the following default values are provided:

Lighting control type	Cost ⁶⁸⁵
Full cost of wall mounted occupancy sensor	\$51
Full cost of ceiling or remote mounted occupancy sensor	\$102
Full cost of fixture-mounted occupancy sensor	\$91.83
Full cost of fixture embedded occupancy sensor ⁶⁸⁶	\$54

⁶⁸⁴ DEER 2008

⁶⁸⁵ Taken from NEEP Commercial Lighting Controls, Incremental Cost Data Analysis, 2011 “NEEP Commercial Lighting Controls 2011_08_29.xlsx”

⁶⁸⁶ Fixture embedded Occupancy Sensors are included with the fixture and therefore no additional installation costs are incurred for these sensors. Therefore, it is assumed that the costs associated with Fixture-embedded Occupancy Sensors should not surpass those of the wall mounted due to the similarity in installation.

LOADSHAPE

Loadshape C06 - Commercial Indoor Lighting
 Loadshape C07 - Grocery/Conv. Store Indoor Lighting
 Loadshape C08 - Hospital Indoor Lighting
 Loadshape C09 - Office Indoor Lighting
 Loadshape C10 - Restaurant Indoor Lighting
 Loadshape C11 - Retail Indoor Lighting
 Loadshape C12 - Warehouse Indoor Lighting
 Loadshape C13 - K-12 School Indoor Lighting
 Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)
 Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)
 Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)
 Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)
 Loadshape C18 - Industrial Indoor Lighting
 Loadshape C19 - Industrial Outdoor Lighting
 Loadshape C20 - Commercial Outdoor Lighting

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on location.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = \text{KW}_{\text{Controlled}} * \text{Hours} * \text{ESF} * \text{WHF}_e$$

Where:

$\text{KW}_{\text{Controlled}}$ = Total lighting load connected to the control in kilowatts. Savings is per control. The total connected load per control should be collected from the customer or the default values presented below used;

Lighting Control Type	Default kw controlled ⁶⁸⁷
Wall mounted occupancy sensor (per control)	0.305
Remote mounted occupancy sensor (per control)	0.517
Fixture mounted sensor (per fixture)	0.180

Hours = total operating hours of the controlled lighting circuit before the lighting controls are installed. This number should be collected from the customer. Average hours of use per year are provided in the Reference Table in Section 4.5, Fixture annual operating hours, for each building type if customer specific information is not collected. If unknown building type, use the Miscellaneous value.

⁶⁸⁷ Based on EVT control data for Occupancy Sensor Costs 2009-2014.

ESF = Energy Savings factor (represents the percentage reduction to the operating Hours from the non-controlled baseline lighting system).

Lighting Control Type	Energy Savings Factor ⁶⁸⁸
Wall, Ceiling or Fixture-Mounted Occupancy Sensors	24%
Wall-Mounted Occupancy Sensors Configured as "Vacancy Sensors"	31% ⁶⁸⁹

WHF_e = Waste heat factor for energy to account for cooling energy savings from efficient lighting is provided in the Reference Table in Section 4.5 for each building type. If building is un-cooled, the value is 1.0.

HEATING PENALTY

If electrically heated building:

$$\Delta \text{kWh}_{\text{heatpenalty}}^{690} = \text{KW}_{\text{Controlled}} * \text{Hours} * \text{ESF} * -\text{IFkWh}$$

Where:

IFkWh = Lighting-HVAC Interaction Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \text{KW}_{\text{controlled}} * \text{WHF}_d * (\text{CF}_{\text{baseline}} - \text{CF}_o)$$

Where:

WHF_d = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is provided in the Reference Table in Section 4.5. If the building is un-cooled WHF_d is 1.

CF_{baseline} = Baseline Summer Peak Coincidence Factor for the lighting system without Occupancy Sensors installed selected from the Reference Table in Section 4.5 for each building type. If the building type is unknown, use the Miscellaneous value of 0.66

CF_o = Retrofit Summer Peak Coincidence Factor the lighting system with Occupancy Sensors installed is 0.15 regardless of building type.⁶⁹¹

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{therms} = \text{KW}_{\text{Controlled}} * \text{Hours} * \text{ESF} * -\text{IFTherms}$$

Where:

⁶⁸⁸ Lawrence Berkeley National Laboratory. A Meta-Analysis of Energy Savings from Lighting Controls in Commercial Buildings. Page & Associates Inc. 2011

⁶⁸⁹ Papamichael, Konstantions, Bi-Level Switching in Office Spaces, California Lighting Technology Center, February 1,2010.

Note: See Figure 8 on page 10 for relevant study results. The study shows a 30% extra savings above a typical occupancy sensor; 24% * 1.3 = 31%.

⁶⁹⁰ Negative value because this is an increase in heating consumption due to the efficient lighting.

⁶⁹¹ Coincidence Factor Study Residential and Commercial Industrial Lighting Measures, RLW Analytics, Spring 2007. Note, the connected load used in the calculation of the CF for occupancy sensor lights includes the average ESF.

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting and provided in the Reference Table in Section 4.5 by building type.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-LTG-OSLC-V04-160601

REVIEW DEADLINE: 1/1/2021

4.5.11 Solar Light Tubes

DESCRIPTION

A tubular skylight which is 10" to 21" in diameter with a prismatic or translucent lens is installed on the roof of a commercial facility. The lens reflects light captured from the roof opening through a highly specular reflective tube down to the mounted fixture height. When in use, a light tube fixture resembles a metal halide fixture. Uses include grocery, school, retail and other single story commercial buildings.

In order that the savings characterized below apply, the electric illumination in the space must be automatically controlled to turn off or down when the tube is providing enough light.

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is assumed to be a tubular skylight that concentrates and directs light from the roof to an area inside the facility.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment for this measure is a fixture with comparable luminosity. The specifications for the baseline lamp depend on the size of the Light Tube being installed.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The estimated useful life for a light tube commercial skylight is 10 years⁶⁹².

DEEMED MEASURE COST

If available, the actual incremental cost should be used. For analysis purposes, assume an incremental cost for a light tube commercial skylight is \$500².

LOADSHAPE

Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)⁶⁹³

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on location.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = \text{kW}_f * \text{HOURS} * \text{WHFe}$$

Where:

kW_f = Connected load of the fixture the solar tube replaces

⁶⁹² Equal to the manufacturers standard warranty

⁶⁹³ The savings from solar light tubes are only realized during the sunlight hours. It is therefore appropriate to apply the single shift (8/5) loadshape to this measure.

Size of Tube	Average Lumen output for Chicago Illinois (minimum) ⁶⁹⁴	Equivalent fixture	kW
21"	9,775 (4,179)	50% 3 x 2 32W lamp CFL (207W, 9915 lumens) 50% 4 lamp F32 w/Elec 4' T8 (114W, 8895 lumens)	0.161
14"	4,392 (1,887)	50% 2 42W lamp CFL (94W, 4406 lumens) 50% 2 lamp F32 w/Elec 4' T8 (59W, 4448 lumens)	0.077
10"	2,157 (911)	50% 1 42W lamp CFL (46W, 2203 lumens) 50% 1 lamp F32 w/Elec 4' T8 (32W, 2224 lumens)	0.039
		AVERAGE	0.092

HOURS = Equivalent full load hours
= 2400⁶⁹⁵

WHF_e = Waste heat factor for energy to account for cooling energy savings from efficient lighting is selected from the Reference Table in Section 4.5 for each building type. If building is un-cooled, the value is 1.0.

HEATING PENALTY

If electrically heated building:

$$\Delta kWh_{\text{heatpenalty}}^{696} = kW_f * \text{HOURS} * -IFkWh$$

Where:

IFkWh = Lighting-HVAC Interaction Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kW_f * WHF_d * CF$$

Where:

WHF_d = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is selected from the Reference Table in Section 4.5 for each building type. If the building is not cooled WHF_d is 1.

CF = Summer Peak Coincidence Factor for measure is selected from the Reference Table in Section 4.5 for each building type. If the building type is unknown, use the Miscellaneous value of 0.66.

NATURAL GAS SAVINGS

$$\Delta \text{Therms}^{697} = \Delta kW_f * \text{HOURS} * -IF_{\text{Therms}}$$

Where:

⁶⁹⁴ Solatube Test Report (2005). http://www.maine绿色建筑.com/files/file/solatube/stb_lumens_datasheet.pdf

⁶⁹⁵ Ibid. The lumen values presented in the kW table represent the average of the lightest 2400 hours.

⁶⁹⁶ Negative value because this is an increase in heating consumption due to the efficient lighting.

⁶⁹⁷ Negative value because this is an increase in heating consumption due to the efficient lighting.

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Please select from the Reference Table in Section 4.5 for each building type.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-LTG-STUB-V02-140601

REVIEW DEADLINE: 1/1/2020

4.5.12 T5 Fixtures and Lamps

DESCRIPTION

T5 Lamp/ballast systems have higher lumens per watt than a standard T8 or an existing T8 or T12 system. The smaller lamp diameter allows for better optical systems, and more precise control of lighting. These characteristics result in light fixtures that produce equal or greater light than standard T8 or T12 fixtures, while using fewer watts.

This measure applies to the installation of new equipment with efficiencies that exceed that of the equipment that would have been installed following standard market practices and is applicable to time of sale as well as retrofit measures.

If the implementation strategy does not allow for the installation location to be known, a deemed split of 99% Commercial and 1% Residential should be used⁶⁹⁸.

This measure was developed to be applicable to the following program types: TOS, RF, DI.

If applied to other program types, the measure savings should be verified.

The measure applies to all commercial T5 installations excluding new construction and substantial renovation or change of use measures (see lighting power density measure). Lookup tables have been provided to account for various installations. Actual existing equipment wattages should be compared to new fixture wattages whenever possible while maintaining lumen equivalent designs. Default new and baseline assumptions are provided if existing equipment cannot be determined. Actual costs and hours of use should be utilized when available. Default component costs and lifetimes have been provided for Operating and Maintenance Calculations. Please see the Definition Table to determine applicability for each program. Configurations not included in the TRM may be included in custom program design using the provided algorithms as long as energy savings is achieved. The following table defines the applicability for different programs:

Time of Sale (TOS)	Retrofit (RF) and DI
This program applies to installations where customer and location of equipment is not known, or at time of burnout of existing equipment. T5 Lamp/ballast systems have higher lumens per watt than a standard T8 system. The smaller lamp diameter allows for better optical systems, and more precise control of lighting. These characteristics result in light fixtures that produce equal or greater light than standard T8 fixtures, while using fewer watts.	For installations that upgrade installations before the end of their useful life. T5 Lamp/ballast systems have higher lumens per watt than a standard T8 or T12 system. The smaller lamp diameter allows for better optical systems, and more precise control of lighting. These characteristics result in light fixtures that produce equal or greater light than standard T8 or T12 fixtures, while using fewer watts and having longer life.

DEFINITION OF EFFICIENT EQUIPMENT

The definition of efficient equipment varies based on the program and is defined below:

Time of Sale (TOS)	Retrofit (RF) and DI
4' fixtures must use a T5 lamp and ballast configuration. 1' and 3' lamps are not eligible. High Performance Troffers must be 85% efficient or greater. T5 HO high bay fixtures must be 3, 4 or 6 lamps and 90% efficient or better.	4' fixtures must use a T5 lamp and ballast configuration. 1' and 3' lamps are not eligible. High Performance Troffers must be 85% efficient or greater. T5 HO high bay fixtures must be 3, 4 or 6 lamps and 90% efficient or better.

⁶⁹⁸ Based on weighted average of Final ComEd's BILD program data from PY5 and PY6. For Residential installations, hours of use assumptions from '5.5 Interior Hardwired Compact Fluorescent Lamp (CFL) Fixture' measure should be used.

DEFINITION OF BASELINE EQUIPMENT

The definition of baseline equipment varies based on the program and is defined below:

Time of Sale (TOS)	Retrofit (RF) and DI
<p>The baseline is T8 with equivalent lumen output. In high-bay applications, the baseline is pulse start metal halide systems.</p>	<p>The baseline is the existing system.</p> <p>In July 14, 2012, Federal Standards were enacted that were expected to eliminate T-12s as an option for linear fluorescent fixtures. Through v3.0 of the TRM, it was assumed that the T-12 would no longer be baseline for retrofits from 1/1/2016. However, due to significant loopholes in the legislation, T-12 compliant product is still freely available and in Illinois T-12s continue to hold a significant share of the existing and replacement lamp market. Therefore the timing of the sunset of T-12s as a viable baseline has been pushed back in v6.0 until 1/1/2019 and will be revisited in future update sessions.</p> <p>There will be a baseline shift applied to all measures installed before 2019 in years remaining in the measure life. See table C-1.</p>

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The deemed lifetime of the efficient equipment fixture, regardless of program type is Fixture lifetime is 15 years⁶⁹⁹.

LOADSHAPE

- Loadshape C06 - Commercial Indoor Lighting
- Loadshape C07 - Grocery/Conv. Store Indoor Lighting
- Loadshape C08 - Hospital Indoor Lighting
- Loadshape C09 - Office Indoor Lighting
- Loadshape C10 - Restaurant Indoor Lighting
- Loadshape C11 - Retail Indoor Lighting
- Loadshape C12 - Warehouse Indoor Lighting
- Loadshape C13 - K-12 School Indoor Lighting
- Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)
- Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)
- Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)
- Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)
- Loadshape C18 - Industrial Indoor Lighting
- Loadshape C19 - Industrial Outdoor Lighting
- Loadshape C20 - Commercial Outdoor Lighting

⁶⁹⁹ 15 years from GDS Measure Life Report, June 2007

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta kWh = ((Watts_{base} - Watts_{EE}) / 1000) * Hours * WHF_e * ISR$$

Where:

Watts_{base} = Input wattage of the existing system which depends on the baseline fixture configuration (number and type of lamp) and number of fixtures. Value can be selected from the appropriate reference table as shown below, or a custom value can be entered if the configurations in the tables is not representative of the existing system.

Watts_{EE} = New Input wattage of EE fixture which depends on new fixture configuration (number of lamps) and ballast factor and number of fixtures. Value can be selected from the appropriate reference table as shown below, or a custom value can be entered if the configurations in the tables is not representative of the existing system.

Program	Reference Table
Time of Sale	A-1: T5 New and Baseline Assumptions
Retrofit, DI	A-2: T5 New and Baseline Assumptions

Hours = Average hours of use per year as provided by the customer or selected from the Reference Table in Section 4.5, Fixture annual operating hours, by building type. If hours or building type are unknown, use the Miscellaneous value.

WHF_e = Waste heat factor for energy to account for cooling energy savings from efficient lighting is selected from the Reference Table in Section 4.5 for each building type. If building is un-cooled, the value is 1.0.

ISR = In Service Rate or the percentage of units rebated that get installed.
 = 100%⁷⁰⁰ if application form completed with sign off that equipment is not placed into storage. If sign off form not completed assume the following 3 year ISR assumptions:

Weighted Average 1 st year In Service Rate (ISR)	2 nd year Installations	3 rd year Installations	Final Lifetime In Service Rate
98% ⁷⁰¹	0%	0%	98.0% ⁷⁰²

⁷⁰⁰ Illinois evaluation of PY1 through PY3 has not found that fixtures or lamps placed into storage to be a significant enough issue to warrant including an "In-Service Rate" when commercial customers complete an application form.

⁷⁰¹ 1st year in service rate is based upon review of PY5-6 evaluations from ComEd's commercial lighting program (BILD) (see 'IL Commercial Lighting ISR_2014.xls' for more information)

⁷⁰² The 98% Lifetime ISR assumption is based upon review of two evaluations:

'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009" and 'KEMA Inc, Feb 2010, Final Evaluation Report; Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings.

HEATING PENALTY

If electrically heated building:

$$\Delta kWh_{\text{heatpenalty}}^{703} = (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * -\text{IFkWh})$$

Where:

IFkWh = Lighting-HVAC Interaction Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

SUMMER COINCIDENT DEMAND SAVINGS

$$\Delta kW = ((\text{Watts}_{\text{base}} - \text{Watts}_{\text{EE}}) / 1000) * \text{WHF}_d * \text{CF} * \text{ISR}$$

Where:

WHF_d = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is selected from the Reference Table in Section 4.5 for each building type. If the building is not cooled WHF_d is 1.

CF = Summer Peak Coincidence Factor for measure is selected from the Reference Table in Section 4.5 for each building type. If the building type is unknown, use the Miscellaneous value.

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms}^{704} = (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * -\text{IFTherms})$$

Where:

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting. This value is selected from the Reference Table in Section 4.5 for each building type.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

See Reference tables for Operating and Maintenance Values

Program	Reference Table
Time of Sale	B-1: T5 Component Costs and Lifetime
Retrofit, DI	B-2: T5 Component Costs and Lifetime

REFERENCE TABLES

See following page.

⁷⁰³Negative value because this is an increase in heating consumption due to the efficient lighting.

⁷⁰⁴Negative value because this is an increase in heating consumption due to the efficient lighting.

A-1: Time of Sale: T5 New and Baseline Assumptions⁷⁰⁵

EE Measure Description	EE Cost	Watts _{EE}	Baseline Description	Base Cost	Watts _{BASE}	Measure Cost	Watts _{SAVE}
2-Lamp T5 High-Bay	\$200.00	180	200 Watt Pulse Start Metal-Halide	\$100.00	232	\$100.00	52
3-Lamp T5 High-Bay	\$200.00	180	200 Watt Pulse Start Metal-Halide	\$100.00	232	\$100.00	52
4-Lamp T5 High-Bay	\$225.00	240	320 Watt Pulse Start Metal-Halide	\$125.00	350	\$100.00	110
6-Lamp T5 High-Bay	\$250.00	360	Proportionally Adjusted according to 6-Lamp HPT8 Equivalent to 320 PSMH	\$150.00	476	\$100.00	116
1-Lamp T5 Troffer/Wrap	\$100.00	32	Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8	\$60.00	44	\$40.00	12
2-Lamp T5 Troffer/Wrap	\$100.00	64	3-Lamp F32T8 Equivalent w/ Elec. Ballast	\$60.00	88	\$40.00	24
1-Lamp T5 Industrial/Strip	\$70.00	32	Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8	\$40.00	44	\$30.00	12
2-Lamp T5 Industrial/Strip	\$70.00	64	3-Lamp F32T8 Equivalent w/ Elec. Ballast	\$40.00	88	\$30.00	24
3-Lamp T5 Industrial/Strip	\$70.00	96	Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8	\$40.00	132	\$30.00	36
4-Lamp T5 Industrial/Strip	\$70.00	128	Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8	\$40.00	178	\$30.00	50
1-Lamp T5 Indirect	\$175.00	32	Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8	\$145.00	44	\$30.00	12
2-Lamp T5 Indirect	\$175.00	64	3-Lamp F32T8 Equivalent w/ Elec. Ballast	\$145.00	88	\$30.00	24

⁷⁰⁵ Adapted from Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, October 26, 2011.

A-2: Retrofit T5 New and Baseline Assumptions⁷⁰⁶

EE Measure Description	EE Cost	Watts _{EE}
3-Lamp T5 High-Bay	\$200.00	180
4-Lamp T5 High-Bay	\$225.00	234
6-Lamp T5 High-Bay	\$250.00	358
1-Lamp T5 Troffer/Wrap	\$100.00	32
2-Lamp T5 Troffer/Wrap	\$100.00	64
1-Lamp T5 Industrial/Strip	\$70.00	32
2-Lamp T5 Industrial/Strip	\$70.00	64
3-Lamp T5 Industrial/Strip	\$70.00	96
4-Lamp T5 Industrial/Strip	\$70.00	128
1-Lamp T5 Indirect	\$175.00	32
2-Lamp T5 Indirect	\$175.00	64

Baseline Description	Watts _{BASE}
200 Watt Pulse Start Metal-Halide	232
250 Watt Metal-Halide	295
320 Watt Pulse Start Metal-Halide	350
400 Watt Metal-Halide	455
400 Watt Pulse Start Metal-Halide	476
1-Lamp F34T12 w/ EEMag Ballast	40
2-Lamp F34T12 w/ EEMag Ballast	68
3-Lamp F34T12 w/ EEMag Ballast	110
4-Lamp F34T12 w/ EEMag Ballast	139
1-Lamp F40T12 w/ EEMag Ballast	48
2-Lamp F40T12 w/ EEMag Ballast	82
3-Lamp F40T12 w/ EEMag Ballast	122
4-Lamp F40T12 w/ EEMag Ballast	164
1-Lamp F40T12 w/ Mag Ballast	57
2-Lamp F40T12 w/ Mag Ballast	94
3-Lamp F40T12 w/ Mag Ballast	147
4-Lamp F40T12 w/ Mag Ballast	182
1-Lamp F32T8	32
2-Lamp F32T8	59
3-Lamp F32T8	88
4-Lamp F32T8	114

⁷⁰⁶Ibid.

B-1: Time of Sale T5 Component Costs and Lifetime⁷⁰⁷

EE Measure Description	EE Lamp Cost	EE Lamp Life (hrs)	EE Lamp Rep. Labor Cost per lamp	EE Ballast Cost	EE Ballast Life (hrs)	EE Ballast Rep. Labor Cost	Baseline Description	# Base Lamps	Base Lamp Cost	Base Lamp Life (hrs)	Base Lamp Rep. Labor Cost	# Base Ballasts	Base Ballast Cost	Base Ballast Life (hrs)	Base Ballast Rep. Labor Cost
3-Lamp T5 High-Bay	\$12.00	20000	\$6.67	\$52.00	70000	\$22.50	200 Watt Pulse Start Metal-Halide	1.00	\$21.00	10000	\$6.67	1.00	\$87.75	40000	\$22.50
4-Lamp T5 High-Bay	\$12.00	20000	\$6.67	\$52.00	70000	\$22.50	320 Watt Pulse Start Metal-Halide	1.00	\$21.00	20000	\$6.67	1.00	\$109.35	40000	\$22.50
6-Lamp T5 High-Bay	\$12.00	20000	\$6.67	\$52.00	70000	\$22.50	Adjusted according to 6-Lamp HPT8 Equivalent to 320	1.36	\$21.00	20000	\$6.67	1.50	\$109.35	40000	\$22.50
1-Lamp T5 Troffer/Wrap	\$12.00	20000	\$2.67	\$52.00	70000	\$15.00	Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8	1.50	\$2.50	20000	\$2.67	0.50	\$15.00	70000	\$15.00
2-Lamp T5 Troffer/Wrap	\$12.00	20000	\$2.67	\$52.00	70000	\$15.00	3-Lamp F32T8 Equivalent w/ Elec. Ballast	3.00	\$2.50	20000	\$2.67	1.00	\$15.00	70000	\$15.00
1-Lamp T5 Industrial/Strip	\$12.00	20000	\$2.67	\$52.00	70000	\$15.00	Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8	1.50	\$2.50	20000	\$2.67	0.50	\$15.00	70000	\$15.00
2-Lamp T5 Industrial/Strip	\$12.00	20000	\$2.67	\$52.00	70000	\$15.00	3-Lamp F32T8 Equivalent w/ Elec. Ballast	3.00	\$2.50	20000	\$2.67	1.00	\$15.00	70000	\$15.00
3-Lamp T5 Industrial/Strip	\$12.00	20000	\$2.67	\$52.00	70000	\$15.00	Proportionally adjusted according to 2-Lamp T5 Equivalent	4.50	\$2.50	20000	\$2.67	1.50	\$15.00	70000	\$15.00
4-Lamp T5 Industrial/Strip	\$12.00	20000	\$2.67	\$52.00	70000	\$15.00	Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8	6.00	\$2.50	20000	\$2.67	2.00	\$15.00	70000	\$15.00
1-Lamp T5 Indirect	\$12.00	20000	\$2.67	\$52.00	70000	\$15.00	Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8	1.50	\$2.50	20000	\$2.67	0.50	\$15.00	70000	\$15.00
2-Lamp T5 Indirect	\$12.00	20000	\$2.67	\$52.00	70000	\$15.00	3-Lamp F32T8 Equivalent w/ Elec. Ballast	3.00	\$2.50	20000	\$2.67	1.00	\$15.00	70000	\$15.00

⁷⁰⁷ Adapted from Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, October 26, 2011.

B-2: T5 Retrofit Component Costs and Lifetime⁷⁰⁸

	EE Lamp Cost	EE Lamp Life (hrs)	EE Lamp Rep. Labor Cost per lamp	EE Ballast Cost	EE Ballast Life (hrs)	EE Ballast Rep. Labor Cost		# Base Lamps	Base Lamp Cost	Base Lamp Life (hrs)	Base Lamp Rep. Labor Cost	# Base Ballast s	Base Ballast Cost	Base Ballast Life (hrs)	Base Ballast Rep. Labor Cost
EE Measure Description							Baseline Description								
3-Lamp T5 High-Bay	\$12.00	20000	\$6.67	\$52.00	70000	\$22.50	200 Watt Pulse Start Metal-Halide	1.00	\$21.00	10000	\$6.67	1.00	\$ 88	40000	\$22.50
							250 Watt Metal Halide	1.00	\$21.00	10000	\$6.67	1.00	\$ 92	40000	\$22.50
4-Lamp T5 High-Bay	\$12.00	20000	\$6.67	\$52.00	70000	\$22.50	320 Watt Pulse Start Metal-Halide	1.00	\$72.00	20000	\$6.67	1.00	\$ 109	40000	\$22.50
							400 Watt Metal Halide	1.00	\$17.00	20000	\$6.67	1.00	\$ 114	40000	\$22.50
6-Lamp T5 High-Bay	\$12.00	20000	\$6.67	\$52.00	70000	\$22.50	Proportionally Adjusted according to 6-Lamp HPT8 Equivalent to 320 PSMH	1.36	\$72.00	20000	\$6.67	1.50	\$ 109	40000	\$22.50
1-Lamp T5 Troffer/w/rap	\$12.00	20000	\$2.67	\$52.00	70000	\$15.00	Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8	1.50	\$2.50	20000	\$2.67	0.50	\$ 15	70000	\$15.00
2-Lamp T5 Troffer/w/rap	\$12.00	20000	\$2.67	\$52.00	70000	\$15.00	3-Lamp F32T8 Equivalent w/ Elec. Ballast	3.00	\$2.50	20000	\$2.67	1.00	\$ 15	70000	\$15.00
1-Lamp T5 Industrial/Strip	\$12.00	20000	\$2.67	\$52.00	70000	\$15.00	Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8	1.50	\$2.50	20000	\$2.67	0.50	\$ 15	70000	\$15.00
2-Lamp T5 Industrial/Strip	\$12.00	20000	\$2.67	\$52.00	70000	\$15.00	3-Lamp F32T8 Equivalent w/ Elec. Ballast	3.00	\$2.50	20000	\$2.67	1.00	\$ 15	70000	\$15.00
3-Lamp T5 Industrial/Strip	\$12.00	20000	\$2.67	\$52.00	70000	\$15.00	Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8	4.50	\$2.50	20000	\$2.67	1.50	\$ 15	70000	\$15.00
4-Lamp T5 Industrial/Strip	\$12.00	20000	\$2.67	\$52.00	70000	\$15.00	Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8	6.00	\$2.50	20000	\$2.67	2.00	\$ 15	70000	\$15.00
1-Lamp T5 Indirect	\$12.00	20000	\$2.67	\$52.00	70000	\$15.00	Proportionally adjusted according to 2-Lamp T5 Equivalent to 3-Lamp T8	1.50	\$2.50	20000	\$2.67	0.50	\$ 15	70000	\$15.00
2-Lamp T5 Indirect	\$12.00	20000	\$2.67	\$52.00	70000	\$15.00	3-Lamp F32T8 Equivalent w/ Elec. Ballast	3.00	\$2.50	20000	\$2.67	1.00	\$ 15	70000	\$15.00

⁷⁰⁸ Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, October 26, 2011
 EPE Program Downloads. Web accessed <http://www.electricefficiency.com/downloads.asp?section=ci> download Copy of LSF_2012_v4.04_250rows.xls.
 Kuiken et al, Focus on Energy Evaluation. Business Programs: Deemed Savings Manual v1.0, Kema, march 22, 2010 available at
http://www.focusonenergy.com/files/Document_Management_System/Evaluation/bpdeemedavingsmanuav10_evaluationreport.pdf

C-1: T12 Baseline Adjustment:

Savings Adjustment Factors

	watts	Equivalent T12 watts adjusted for lumen equivalency-34 w and 40 w with EEMag ballast	Equivalent T12 watts adjusted for lumen equivalency-40 w with EEMag ballast	Equivalent T12 watts adjusted for lumen equivalency-40 w with Mag ballast	Prportionally Adjusted for Lumens wattage for T8 equivalent
1-Lamp T5 Industrial/Strip	32	61	73	82	44
2-Lamp T5 Industrial/Strip	64	103	125	135	88
3-Lamp T5 Industrial/Strip	96	167	185	211	132
4-Lamp T5 Industrial/Strip	128	211	249	226	178
		Savings Factor Adjustment to the T8 baseline	Savings Factor Adjustment to the T8 baseline	Savings Factor Adjustment to the T8 baseline	
1-Lamp T5 Industrial/Strip		42%	29%	24%	
2-Lamp T5 Industrial/Strip		61%	40%	34%	
3-Lamp T5 Industrial/Strip		51%	40%	31%	
4-Lamp T5 Industrial/Strip		60%	41%	51%	

Measures installed in 2018 will claim full savings for one year. Savings adjustment factors based on a T8 baseline will be applied to the full savings for savings starting in 2019 and for the remainder of the measure life. The adjustment to be applied for each measure is listed in the reference table above and is based on equivalent lumens.

MEASURE CODE: CI-LTG-T5FX-V05-180101

REVIEW DEADLINE: 1/1/2019

4.5.13 Occupancy Controlled Bi-Level Lighting Fixtures

DESCRIPTION

This measure relates to replacing existing uncontrolled continuous lighting fixtures with new bi-level lighting fixtures. This measure can only relate to replacement in an existing building, since multi-level switching is required in the Commercial new construction building energy code (IECC 2012/2015).

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient system is assumed to be an occupancy controlled lighting fixture that reduces light level during unoccupied periods.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be an uncontrolled lighting system on continuously, e.g. in stairwells and corridors for health and safety reasons.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life for all lighting controls is assumed to be 8 years⁷⁰⁹.

DEEMED MEASURE COST

When available, the actual cost of the measure shall be used. When not available, the assumed measure cost is \$274⁷¹⁰.

LOADSHAPE

- Loadshape C06 - Commercial Indoor Lighting
- Loadshape C07 - Grocery/Conv. Store Indoor Lighting
- Loadshape C08 - Hospital Indoor Lighting
- Loadshape C09 - Office Indoor Lighting
- Loadshape C10 - Restaurant Indoor Lighting
- Loadshape C11 - Retail Indoor Lighting
- Loadshape C12 - Warehouse Indoor Lighting
- Loadshape C13 - K-12 School Indoor Lighting
- Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)
- Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)
- Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)
- Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)
- Loadshape C18 - Industrial Indoor Lighting
- Loadshape C19 - Industrial Outdoor Lighting
- Loadshape C20 - Commercial Outdoor Lighting

⁷⁰⁹ DEER 2008.

⁷¹⁰ Consistent with the Multi-level Fixture measure with reference to Goldberg et al, State of Wisconsin Public Service Commission of Wisconsin, Focus on Energy Evaluation, Business Programs: Incremental Cost Study, KEMA, October 28, 2009. Also consistent with field experience of about \$250 per fixture and \$25 install labor.

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on the location type. Values are provided for each building type in the reference section below.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta kWh = (KW_{Baseline} - (KW_{Controlled} * (1 - ESF))) * Hours * WHF_e$$

Where:

$KW_{Baseline}$ = Total baseline lighting load of the existing/baseline fixture

= Actual

Note that if the existing fixture is only being retrofit with bi-level occupancy controls and not being replaced $KW_{Baseline}$ will equal $KW_{Controlled}$.

$KW_{Controlled}$ = Total controlled lighting load at full light output of the new bi-level fixture

= Actual

Hours = Number of hours lighting is on. This measure is limited to 24/7 operation.

= 8,766

ESF = Energy Savings factor (represents the percentage reduction to the $KW_{Controlled}$ due to the occupancy control).

= % Standby Mode * (1 - % Full Light at Standby Mode)

% Standby Mode = Represents the percentage of the time the fixture is operating in standby (i.e. low-wattage) mode.

% Full Light at Standby Mode = Represents the assumed wattage consumption during standby mode relative to the full wattage consumption. Can be achieved either through dimming or a stepped control strategy.

= Dependent on application. If participant provided or metered data is available for both or either of these inputs a custom savings factor should be calculated. If not defaults are provided below:

Application	% Standby Mode	% Full Light at Standby Mode	Energy Savings Factor (ESF)
Stairwells	78.5% ⁷¹¹	50%	39.3%
		33%	52.6%
		10%	70.7%
		5%	74.6%
Corridors		50%	25.0%

⁷¹¹ Average found from the four buildings in the State of California Energy Commission Lighting Research Program Bi-Level Stairwell Fixture Performance Final Report:

http://www.archenergy.com/lrp/lightingperf_standards/project_5_1_reports.htm

Application	% Standby Mode	% Full Light at Standby Mode	Energy Savings Factor (ESF)
	50.0% ⁷¹²	33%	33.5%
		10%	45.0%
		5%	47.5%
Other 24/7 Space Type	50.0% ⁷¹³	50%	25.0%
		33%	33.5%
		10%	45.0%
		5%	47.5%

WHF_e = Waste heat factor for energy to account for cooling energy savings from efficient lighting is provided in the Reference Table in Section 4.5 for each building type. If building is un-cooled, the value is 1.0.

HEATING PENALTY

If electrically heated building:

$$\Delta \text{kWh}_{\text{heating penalty}}^{714} = (\text{KW}_{\text{Baseline}} - (\text{KW}_{\text{Controlled}} * (1 - \text{ESF}))) * \text{Hours} * -\text{IFkWh}$$

Where:

IFkWh = Lighting-HVAC Interaction Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = (\text{KW}_{\text{Baseline}} - (\text{KW}_{\text{Controlled}} * (1 - \text{ESF}))) * \text{WHF}_d * (\text{CF}_{\text{baseline}} - \text{CF}_{\text{os}})$$

Where:

WHF_d = Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is provided in the Reference Table in Section 4.5. If the building is un-cooled WHF_d is 1.

CF_{baseline} = Baseline Summer Peak Coincidence Factor for the lighting system without Occupancy Sensors installed selected from the Reference Table in Section 4.5 for each building type. If the building type is unknown, use the Miscellaneous value of 0.66

CF_{os} = Retrofit Summer Peak Coincidence Factor for the lighting system with Occupancy Sensors installed is 0.15 regardless of building type.⁷¹⁵

⁷¹² Value determined from the Pacific Gas and Electric Company: Bi-Level Lighting Control Credits study for Interior Corridors of Hotels, Motels and High Rise Residential.

http://www.energy.ca.gov/title24/2005standards/archive/documents/2002-07-18_workshop/2002-07-18_BILEVEL_LIGHTING.PDF

⁷¹³ Conservative estimate.

⁷¹⁴ Negative value because this is an increase in heating consumption due to the efficient lighting.

⁷¹⁵ Coincidence Factor Study Residential and Commercial Industrial Lighting Measures, RLW Analytics, Spring 2007. Note, the connected load used in the calculation of the CF for occupancy sensor lights includes the average ESF.

NATURAL GAS HEATING PENALTY

If natural gas heating:

$$\Delta \text{therms} = (\text{KW}_{\text{Baseline}} - (\text{KW}_{\text{Controlled}} * (1 - \text{ESF}))) * \text{Hours} * \text{IFTherms}$$

Where:

IFTherms = Lighting-HVAC Integration Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting and provided in the Reference Table in Section 4.5 by building type.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-LTG-OCBL-V02-160601

REVIEW DEADLINE: 1/1/2021

4.5.14 Commercial ENERGY STAR Specialty Compact Fluorescent Lamp (CFL)

DESCRIPTION

A qualified specialty compact fluorescent bulb is installed in place of an incandescent specialty bulb in a commercial location.

Note a new ENERGY STAR specification v2.0 becomes effective on 1/2/2017

(https://www.energystar.gov/products/spec/lamps_specification_version_2_0_pd). The efficacy requirements can not currently be met by Compact Fluorescent Lamps, and therefore this specification has been removed. ENERGY STAR will maintain a list on their website with the final qualifying list of products prior to this change and it is strongly recommended that programs continue to use this list as qualifying criteria for products in the programs.

If the implementation strategy does not allow for the installation location to be known a deemed split should be used. For Residential targeted programs (e.g. an upstream retail program), a deemed split of 95% Residential and 5% Commercial assumptions should be used⁷¹⁶, and for Commercial targeted programs a deemed split of 4% Residential and 96% Commercial should be used⁷¹⁷.

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the high-efficiency equipment must be a qualified specialty compact fluorescent lamp.

DEFINITION OF BASELINE EQUIPMENT

The baseline is a specialty incandescent light bulb including those exempt of the EISA 2007 standard: three-way, plant light, daylight bulb, bug light, post light, globes G40 (<40W), candelabra base (<60W), vibration service bulb, decorative candle with medium or intermediate base (<40W), shatter resistant and reflector bulbs and standard bulbs greater than 2601 lumens, and those non-exempt from EISA 2007: dimmable, globes (less than 5" diameter and >40W), candle (shapes B, BA, CA >40W, candelabra base lamps (>60W) and intermediate base lamps (>40W).

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life (number of years that savings should be claimed) should be calculated by dividing the rated life of the bulb (10,000 hours⁷¹⁸) by the run hours. For example using Miscellaneous at 3612 hours would give 2.8 years. For non-exempt bulbs, when the number of years exceeds 2021, the number of years to that date should be used.

DEEMED MEASURE COST

For the Retail (Time of Sale) measure, the incremental capital cost for this measure is \$5⁷¹⁹.

For the Retrofit measures, the full cost of \$8.50 should be used plus \$5 labor⁷²⁰ for a total of \$13.50. However actual program delivery costs should be utilized if available.

⁷¹⁶ RES v C&I split is based on a weighted (by sales volume) average of ComEd PY6, PY7 and PY8 and Ameren PY5, PY6 and PY8 in store intercept survey results. See 'RESvCI Split_112015.xls'.

⁷¹⁷ Based upon final weighted (by sales volume) average of the BILD program (ComEd's commercial lighting program) for PY 4 and PY5 and PY6.

⁷¹⁸ Energy Star bulbs have a rated life of at least 8000 hours. In commercial settings you expect significantly less on/off switching than residential and so a rated life assumption of 10,000 hours is used.

⁷¹⁹ NEEP Residential Lighting Survey, 2011

⁷²⁰ Based on 15 minutes at \$20 per hour.

LOADSHAPE

Loadshape C06 - Commercial Indoor Lighting
 Loadshape C07 - Grocery/Conv. Store Indoor Lighting
 Loadshape C08 - Hospital Indoor Lighting
 Loadshape C09 - Office Indoor Lighting
 Loadshape C10 - Restaurant Indoor Lighting
 Loadshape C11 - Retail Indoor Lighting
 Loadshape C12 - Warehouse Indoor Lighting
 Loadshape C13 - K-12 School Indoor Lighting
 Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)
 Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)
 Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)
 Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)
 Loadshape C18 - Industrial Indoor Lighting
 Loadshape C19 - Industrial Outdoor Lighting
 Loadshape C20 - Commercial Outdoor Lighting

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on the location type. Values are provided for each building type in section 4.5.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * \text{WHFe}$$

Where:

WattsBase = Actual wattage equivalent of incandescent specialty bulb, use the tables below to obtain the incandescent bulb equivalent wattage⁷²¹; use 60W if unknown⁷²²

EISA exempt bulb types:

Bulb Type	Lower Lumen Range	Upper Lumen Range	WattsBase
Standard Spirals >=2601	2601	2999	150
	3000	5279	200
	5280	6209	300

⁷²¹ Based upon the draft ENERGY STAR specification for lamps

(http://energystar.gov/products/specs/sites/products/files/ENERGY_STAR_Lamps_V1_0_Draft%203.pdf) and the Energy Policy and Conservation Act of 2012.

⁷²² A 2006-2008 California Upstream Lighting Evaluation found an average incandescent wattage of 61.7 Watts (KEMA, Inc, The Cadmus Group, Itron, Inc, PA Consulting Group, Jai J. Mitchell Analytics, Draft Evaluation Report: Upstream Lighting Program. Prepared for the California Public Utilities Commission, Energy Division. December 10, 2009)

Bulb Type	Lower Lumen Range	Upper Lumen Range	WattsBase
3-Way	250	449	25
	450	799	40
	800	1099	60
	1100	1599	75
	1600	1999	100
	2000	2549	125
	2550	2999	150
Globe (medium and intermediate bases less than 750 lumens)	90	179	10
	180	249	15
	250	349	25
	350	749	40
Decorative (Shapes B, BA, C, CA, DC, F, G, medium and intermediate bases less than 750 lumens)	70	89	10
	90	149	15
	150	299	25
	300	749	40
Globe (candelabra bases less than 1050 lumens)	90	179	10
	180	249	15
	250	349	25
	350	499	40
	500	1049	60
Decorative (Shapes B, BA, C, CA, DC, F, G, candelabra bases less than 1050 lumens)	70	89	10
	90	149	15
	150	299	25
	300	499	40
	500	1049	60

EISA non-exempt bulb types:

Bulb Type	Lower Lumen Range	Upper Lumen Range	Incandescent Equivalent Post-EISA 2007 (WattsBase)
Dimmable Twist, Globe (less than 5" in diameter and > 749 lumens), candle (shapes B, BA, CA > 749 lumens), Candelabra Base Lamps (>1049 lumens), Intermediate Base Lamps (>749 lumens)	310	749	29
	750	1049	43
	1050	1489	53
	1490	2600	72

Directional Lamps - ENERGY STAR Minimum Luminous Efficacy = 40Lm/W for lamps with rated wattages less than 20W and 50 Lm/W for lamps with rated wattages ≥ 20 watts⁷²³.

For Directional R, BR, and ER lamp types⁷²⁴:

Bulb Type	Lower Lumen Range	Upper Lumen Range	Watts _{Base}
R, ER, BR with medium screw bases w/ diameter >2.25" (*see exceptions below)	420	472	40
	473	524	45
	525	714	50
	715	937	65
	938	1259	75
	1260	1399	90
	1400	1739	100
	1740	2174	120
	2175	2624	150
	2625	2999	175
*R, BR, and ER with medium screw bases w/ diameter ≤ 2.25"	3000	4500	200
	400	449	40
	450	499	45
	500	649	50
*ER30, BR30, BR40, or ER40	650	1199	65
	400	449	40
	450	499	45
*BR30, BR40, or ER40	500	649	50
	650	1419	65
*R20	400	449	40
	450	719	45
*All reflector lamps below lumen ranges specified above	200	299	20
	300	399	30

Directional lamps are exempt from EISA regulations.

For PAR, MR, and MRX Lamps Types:

For these highly focused directional lamp types, it is necessary to have Center Beam Candle Power (CBCP) and beam angle measurements to accurately estimate the equivalent baseline wattage. The formula below is based on the Energy Star Center Beam Candle Power tool.⁷²⁵ If CBCP and beam angle information are not

⁷²³ From pg 10 of the Energy Star Specification for lamps v1.1

⁷²⁴ From pg 11 of the Energy Star Specification for lamps v1.1

⁷²⁵ <http://energystar.supportportal.com/link/portal/23002/23018/Article/32655/>

available or if the equation below returns a negative value (or undefined), use the manufacturer's recommended baseline wattage equivalent.⁷²⁶

Wattsbase =

$$375.1 - 4.355(D) - \sqrt{227,800 - 937.9(D) - 0.9903(D^2) - 1479(BA) - 12.02(D * BA) + 14.69(BA^2) - 16,720 * \ln(CBCP)}$$

Where:

D = Bulb diameter (e.g. for PAR20 D = 20)

BA = Beam angle

CBCP = Center beam candle power

The result of the equation above should be rounded DOWN to the nearest wattage established by Energy Star:

Diameter	Permitted Wattages
16	20, 35, 40, 45, 50, 60, 75
20	50
30S	40, 45, 50, 60, 75
30L	50, 75
38	40, 45, 50, 55, 60, 65, 75, 85, 90, 100, 120, 150, 250

EISA non-exempt bulb types:

Bulb Type	Lower Lumen Range	Upper Lumen Range	Incandescent Equivalent Post-EISA 2007 (WattsBase)
Dimmable Twist, Globe (less than 5" in diameter and > 749 lumens), candle (shapes B, BA, CA > 749 lumens), Candelabra Base Lamps (>1049 lumens), Intermediate Base Lamps (>749 lumens)	310	749	29
	750	1049	43
	1050	1489	53
	1490	2600	72

WattsEE = Actual wattage of energy efficient specialty bulb purchased, use 15W if unknown⁷²⁷

ISR = In Service Rate or the percentage of units rebated that get installed.
=100%⁷²⁸ if application form completed with sign off that equipment is not placed into storage

⁷²⁶ The Energy Star Center Beam Candle Power tool does not accurately model baseline wattages for lamps with certain bulb characteristic combinations – specifically for lamps with very high CBCP.

⁷²⁷ An evaluation (Energy Efficiency / Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: Residential Energy Star® Lighting

http://ilsag.org/yahoo_site_admin/assets/docs/ComEd_Res_Lighting_PY2_Evaluation_Report_2010-12-21_Final.12113928.pdf) reported 13-17W as the most common specialty CFL wattage (69% of program bulbs). 2009 California data also reported an average CFL wattage of 15.5 Watts (KEMA, Inc, The Cadmus Group, Itron, Inc, PA Consulting Group, Jai J. Mitchell Analytics, Draft Evaluation Report: Upstream Lighting Program, Prepared for the California Public Utilities Commission, Energy Division. December 10, 2009).

⁷²⁸ Illinois evaluation of PY1 through PY3 has not found that fixtures or lamps placed into storage to be a significant enough issue to warrant including an "In-Service Rate" when commercial customers complete an application form.

If sign off form not completed assume the following 3 year ISR assumptions:

Weighted Average 1 st year In Service Rate (ISR)	2 nd year Installations	3 rd year Installations	Final Lifetime In Service Rate
71.2% ⁷²⁹	14.5%	12.3%	98.0% ⁷³⁰

Hours = Average hours of use per year are provided in Reference Table in Section 4.5, Screw based bulb annual operating hours, for each building type⁷³¹. If unknown use the Miscellaneous value.

WHFe = Waste heat factor for energy to account for cooling energy savings from efficient lighting are provided below for each building type in Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

DEFERRED INSTALLS

As presented above, the characterization assumes that a percentage of bulbs purchased are not installed until Year 2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

Year 1 (Purchase Year) installs: Characterized using assumptions provided above or evaluated assumptions if available.

Year 2 and 3 installs: Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.

The NTG factor for the Purchase Year should be applied.

EXAMPLE

For example, for a 14W 500 lumen R20 reflector lamp is installed in an office and sign off form provided.

$$\begin{aligned}\Delta\text{kWh} &= (((45 - 14)/1000) * 1.0 * 3088 * 1.25 \\ &= 119.7 \text{ kWh}\end{aligned}$$

HEATING PENALTY

If electrically heated building:

⁷²⁹ 1st year in service rate is based upon review of PY4-6 evaluations from ComEd's commercial lighting program (BILD) (see 'IL Commercial Lighting ISR_2014.xls' for more information. The average first year ISR was calculated weighted by the number of bulbs sold.

⁷³⁰ The 98% Lifetime ISR assumption is based upon review of two evaluations:

'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings. Note that this Final Install Rate does NOT account for leakage of purchased bulbs being installed outside of the utility territory. EM&V should assess how and if data from evaluation should adjust this final installation rate to account for this impact

⁷³¹ Based on ComEd analysis taking DEER 2008 values and averaging with PY1 and PY2 evaluation results.

$$\Delta kWh_{\text{heatpenalty}}^{732} = (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * -\text{IFkWh})$$

Where:

IFkWh = Lighting-HVAC Interaction Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

EXAMPLE

For example, for a 14W 500 lumen R20 reflector lamp is installed in a heat pump heated office and sign off form provided.

$$\begin{aligned}\Delta kWh_{\text{heatpenalty}} &= (((45 - 14) / 1000) * 1.0 * 3088 * -0.183) \\ &= -17.5 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{WHFd} * \text{CF}$$

Where:

WHFd = Waste heat factor for demand to account for cooling savings from efficient lighting in cooled buildings is provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value..

CF = Summer Peak Coincidence Factor for measure is provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value..

Other factors as defined above

EXAMPLE

For example, for a 14W 500 lumen R20 reflector lamp is installed in an office and sign off form provided.

$$\begin{aligned}\Delta kW &= ((45 - 14) / 1000) * 1.0 * 1.3 * 0.66 \\ &= 0.027 \text{ kW}\end{aligned}$$

NATURAL GAS SAVINGS

Heating Penalty if fossil fuel heated building (or if heating fuel is unknown):

$$\Delta \text{Therms}^{733} = (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * -\text{IFTherms})$$

Where:

⁷³²Negative value because this is an increase in heating consumption due to the efficient lighting.

⁷³³ Negative value because this is an increase in heating consumption due to the efficient lighting.

IFTherms = Lighting-HVAC Interaction Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

Other factors as defined above

EXAMPLE

For example, for a 14W 500 lumen R20 reflector lamp is installed in a gas heated office and sign off form provided.

$$\begin{aligned}\Delta\text{Therms} &= (((45 - 14)/1000) * 1.0 * 3088 * -0.016 \\ &= - 1.5 \text{ Therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

The following O&M assumptions should be used: Life of the baseline bulb is assumed to be (1000/HOURS) year; baseline replacement cost is assumed to be \$3.5 for those bulbs types exempt from EISA and \$5 for non-exempt EISA bulb types defined above⁷³⁴. It is important to note that for cost-effectiveness screening purposes, the O&M cost adjustments should only be applied in cases where the light bulbs are actually in service and so should be multiplied by the appropriate ISR.

MEASURE CODE: CI-LTG-SCFL-V03-180101

REVIEW DEADLINE: 1/1/2020

⁷³⁴ NEEP Residential Lighting Survey, 2011

4.5.15 LED Open Sign

DESCRIPTION

LED open signs must replace an existing neon open sign. LED drivers can be either electronic switching or linear magnetic, with the electronic switching supplies being the most efficient. The on/off power switch may be found on either the power line or load side of the driver, with the line side location providing significantly lower standby losses when the sign is turned off and is not operating. All new open signs must meet UL-84 (UL-844) requirements.

Replacement signs cannot use more than 20% of the input power of the sign that is being replaced.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient product is an LED type illuminated open sign.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is a neon type illuminated open sign.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The estimated useful life is 15 years.⁷³⁵

DEEMED MEASURE COST

The actual measure installation cost should be used (including material and labor).

LOADSHAPE

- Loadshape C06 - Commercial Indoor Lighting
- Loadshape C07 - Grocery/Conv. Store Indoor Lighting
- Loadshape C08 - Hospital Indoor Lighting
- Loadshape C09 - Office Indoor Lighting
- Loadshape C10 - Restaurant Indoor Lighting
- Loadshape C11 - Retail Indoor Lighting
- Loadshape C12 - Warehouse Indoor Lighting
- Loadshape C13 - K-12 School Indoor Lighting
- Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)
- Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)
- Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)
- Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)
- Loadshape C18 - Industrial Indoor Lighting
- Loadshape C19 - Industrial Outdoor Lighting
- Loadshape C20 - Commercial Outdoor Lighting

⁷³⁵ 15 years from GDS Measure Life Report, June 2007

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is dependent on the location type. Values are provided for each building type in the reference section in Section 4.5.

Algorithm

CALCULATION OF ENERGY SAVINGS**ELECTRIC ENERGY SAVINGS**

The following equation was used to determine the energy savings from installing LED open signs:

$$\Delta kWh = (Watts_{base} - Watts_{ee}) / 1,000 * Hours * WHFe$$

Where:

Watts _{base}	= Wattage of neon sign with magnetic high voltage transformer = Actual; if unknown use 46.0W ⁷³⁶
Watts _{ee}	= Wattage of LED sign with low voltage transformer = Actual; if unknown use 14.9W ⁷³⁷
Hours	= Annual hours of operation, assumed to be consistent with operating hours. Values are provided in the Reference Table in Section 4.5.
WHFe	= Waste heat factor for energy to account for cooling energy savings from efficient lighting are provided below for each building type in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

HEATING PENALTY

If electrically heated building:

$$\Delta kWh_{heatpenalty}^{738} = ((Watts_{base} - Watts_{ee}) / 1000) * Hours * -IFkWh$$

Where:

IFkWh	= Lighting-HVAC Interaction Factor for electric heating impacts; this factor represents the increased electric space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.
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DEMAND SAVINGS

$$\Delta kW = ((Watts_{base} - Watts_{ee}) / 1000) * CF * WHFd$$

Where:

WHFd	= Waste Heat Factor for Demand to account for cooling savings from efficient lighting in cooled buildings is provided in Reference Table in Section 4.5. If unknown, use the Miscellaneous value.
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⁷³⁶ Measured average demand data. Southern California Edison, "Replace Neon Open Sign with LED Open Sign", Workpaper SCE13LG070, Revision 2, October 2015. Pg. 10

⁷³⁷ Ibid.

⁷³⁸ Negative value because this is an increase in heating consumption due to the efficient lighting.

CF = Summer Peak Coincidence Factor for measure is provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

Other variables as provided above.

Based on defaults provided above, the deemed energy savings are provided below:

Electric Energy and Coincident Peak Demand Savings

Building Types ⁷³⁹	Energy Savings (kWh)	$\Delta kWh_{\text{heatpenalty}}$ (if electric heat)	Coincident Demand Savings (kW)
Convenience Store	158	-120	0.0298
Grocery	152	-74	0.0277
Healthcare Clinic	169	-17	0.0374
Hotel/Motel - Common	229	-143	0.0282
Movie Theater	121	-73	0.0227
Restaurant	203	-85	0.0277
Retail - Department Store	191	-88	0.0387
Miscellaneous	115	-55	0.0245

NATURAL GAS SAVINGS

Heating Penalty if fossil fuel heated building (or if heating fuel is unknown):

$$\Delta \text{Therms}^{740} = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{Hours} * \text{IFTherms}$$

Where:

IFTherms = Lighting-HVAC Interaction Factor for gas heating impacts; this factor represents the increased gas space heating requirements due to the reduction of waste heat rejected by the efficient lighting. Values are provided in the Reference Table in Section 4.5. If unknown, use the Miscellaneous value.

Other factors as defined above

Based on defaults provided above, the deemed penalty is provided below:

Building Type	$\Delta \text{Therms}_{\text{heatpenalty}}$ (if gas heat)
Convenience Store	-5.1
Grocery	-3.2
Healthcare Clinic	-0.7
Hotel/Motel - Common	-6.1
Movie Theater	-3.2
Restaurant	-3.6
Retail - Department Store	-3.7
Miscellaneous	-2.3

WATER AND OTHER NON-ENERGY IMPACT DESCRIPTIONS AND CALCULATION

N/A

⁷³⁹ Savings can be calculated for additional building types using the default values provided in the Reference Table in Section 4.5.

⁷⁴⁰ Negative value because this is an increase in heating consumption due to the efficient lighting.

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-LTG-OPEN-V01-180101

REVIEW DEADLINE: 1/1/2022

4.6 Refrigeration End Use

4.6.1 Automatic Door Closer for Walk-In Coolers and Freezers

DESCRIPTION

This measure is for installing an auto-closer to the main insulated opaque door(s) of a walk-in cooler or freezer. The auto-closer must firmly close the door when it is within 1 inch of full closure.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

This measure consists of the installation of an automatic, hydraulic-type door closer on main walk-in cooler or freezer doors. These closers save energy by reducing the infiltration of warm outside air into the refrigeration itself.

DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline condition is assumed to be a walk in cooler or freezer without an automatic closure.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The deemed measure life is 8 years.⁷⁴¹

DEEMED MEASURE COST

The deemed measure cost is \$156.82 for a walk-in cooler or freezer.⁷⁴²

LOADSHAPE

Loadshape C22 - Commercial Refrigeration

COINCIDENCE FACTOR

The measure has deemed kW savings therefore a coincidence factor does not apply.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Savings calculations are based on values from through PG&E's Workpaper PGECOREF110.1 – Auto-Closers for Main Cooler or Freezer Doors. Savings are averaged across all California climate zones and vintages⁷⁴³.

Annual Savings	kWh
Walk in Cooler	943
Walk in Freezer	2307

⁷⁴¹ Source: DEER 2008

⁷⁴² Ibid.

⁷⁴³ Measure savings from ComEd TRM developed by KEMA. June 1, 2010

SUMMER COINCIDENT PEAK DEMAND SAVINGS

Annual Savings	kW
Walk in Cooler	0.137
Walk in Freezer	0.309

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-RFG-ATDC-V01-120601

REVIEW DEADLINE: 1/1/2019

4.6.2 Beverage and Snack Machine Controls

DESCRIPTION

This measure relates to the installation of new controls on refrigerated beverage vending machines, non-refrigerated snack vending machines, and glass front refrigerated coolers. Controls can significantly reduce the energy consumption of vending machine and refrigeration systems. Qualifying controls must power down these systems during periods of inactivity but, in the case of refrigerated machines, must always maintain a cool product that meets customer expectations. This measure relates to the installation of a new control on a new or existing unit. This measure should **not** be applied to ENERGY STAR qualified vending machines, as they already have built-in controls.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a standard efficiency refrigerated beverage vending machine, non-refrigerated snack vending machine, or glass front refrigerated cooler with a control system capable of powering down lighting and refrigeration systems during periods of inactivity.

DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline equipment is assumed to be a standard efficiency refrigerated beverage vending machine, non-refrigerated snack vending machine, or glass front refrigerated cooler without a control system capable of powering down lighting and refrigeration systems during periods of inactivity.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 5 years⁷⁴⁴.

DEEMED MEASURE COST

The actual measure installation cost should be used (including material and labor), but the following can be assumed for analysis purposes⁷⁴⁵:

Refrigerated Vending Machine and Glass Front Cooler: \$180.00

Non-Refrigerated Vending Machine: \$80.00

LOADSHAPE

Loadshape C52 - Beverage and Snack Machine Controls

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 0⁷⁴⁶.

⁷⁴⁴ Measure Life Study, prepared for the Massachusetts Joint Utilities, Energy & Resource Solutions, November 2005.

⁷⁴⁵ ComEd workpapers, 8—15-11.pdf

⁷⁴⁶ Assumed that the peak period is coincident with periods of high traffic diminishing the demand reduction potential of occupancy based controls.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = \text{WATTSbase} / 1000 * \text{HOURS} * \text{ESF}$$

Where:

WATTSbase = connected W of the controlled equipment; see table below for default values by connected equipment type:

Equipment Type	WATTSbase ⁷⁴⁷
Refrigerated Beverage Vending Machines	400
Non-Refrigerated Snack Vending Machines	85
Glass Front Refrigerated Coolers	460

1000 = conversion factor (W/kW)

HOURS = operating hours of the connected equipment; assumed that the equipment operates 24 hours per day, 365.25 days per year
= 8766

ESF = Energy Savings Factor; represents the percent reduction in annual kWh consumption of the equipment controlled; see table below for default values:

Equipment Type	Energy Savings Factor (ESF) ⁷⁴⁸
Refrigerated Beverage Vending Machines	46%
Non-Refrigerated Snack Vending Machines	46%
Glass Front Refrigerated Coolers	30%

EXAMPLE

For example, adding controls to a refrigerated beverage vending machine:

$$\begin{aligned} \Delta \text{kWh} &= \text{WATTSbase} / 1000 * \text{HOURS} * \text{ESF} \\ &= 400 / 1000 * 8766 * 0.46 \\ &= 1613 \text{ kWh} \end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

⁷⁴⁷ USA Technologies Energy Management Product Sheets, July 2006; cited September 2009. <http://www.usatech.com/energy_management/energy_productsheets.php>

⁷⁴⁸ Ibid.

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-RFG-BEVM-V02-150601

REVIEW DEADLINE: 1/1/2019

4.6.3 Door Heater Controls for Cooler or Freezer

DESCRIPTION

By installing a control device to turn off door heaters when there is little or no risk of condensation, one can realize significant energy savings. There are two commercially available control strategies that achieve “on-off” control of door heaters based on either (1) the relative humidity of the air in the store or (2) the “conductivity” of the door (which drops when condensation appears). In the first strategy, the system activates your door heaters when the relative humidity in your store rises above a specific setpoint, and turns them off when the relative humidity falls below that setpoint. In the second strategy, the sensor activates the door heaters when the door conductivity falls below a certain setpoint, and turns them off when the conductivity rises above that setpoint.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a door heater control on a commercial glass door cooler or refrigerator utilizing humidity or conductivity control.

DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline condition is assumed to be a commercial glass door cooler or refrigerator with a standard heated door with no controls installed.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years⁷⁴⁹.

DEEMED MEASURE COST

The incremental capital cost for a humidity-based control is \$300 per circuit regardless of the number of doors controlled. The incremental cost for conductivity-based controls is \$200⁷⁵⁰.

LOADSHAPE

Loadshape C51 - Door Heater Control

COINCIDENCE FACTOR⁷⁵¹

The summer peak coincidence factor for this measure is assumed to be 0%⁷⁵².

⁷⁴⁹ 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, “Effective/Remaining Useful Life Values”, California Public Utilities Commission, December 16, 2008.

⁷⁵⁰ Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, February, 19, 2010

⁷⁵¹ Source partial list from DEER 2008

⁷⁵² Based on the assumption that humidity levels will most likely be relatively high during the peak period, reducing the likelihood of demand savings from door heater controls.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWH} = \text{kWbase} * \text{NUMdoors} * \text{ESF} * \text{BF} * 8766$$

Where:

kWbase^{753} = connected load kW for typical reach-in refrigerator or freezer door and frame with a heater.

= If actual kWbase is unknown, assume 0.195 kW for freezers and 0.092 kW for coolers.

NUMdoors = number of reach-in refrigerator or freezer doors controlled by sensor

= Actual installed

ESF^{754} = Energy Savings Factor; represents the percentage of hours annually that the door heater is powered off due to the controls.

= assume 55% for humidity-based controls, 70% for conductivity-based controls

BF^{755} = Bonus Factor; represents the increased savings due to reduction in cooling load inside the cases, and the increase in cooling load in the building space to cool the additional heat generated by the door heaters.

Definition	Representative Evaporator Temperature Range, °F ⁷⁵⁶	Typical Uses	BF
Low	-35 to 0	Freezers for times such as frozen pizza, ice cream, etc.	1.36
Medium	0 – 20	Coolers for items such as meat, milk, dairy, etc	1.22
High	20 – 45	Coolers for items such as floral, produce and meat preparation rooms	1.15

8766 = annual hours of operation

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

N/A

⁷⁵³ A review of TRM methodologies from Vermont, New York, Wisconsin, and Connecticut reveals several different sources for this factor. Connecticut requires site-specific information, whereas New York's characterization does not explicitly identify the kWbase. Connecticut and Vermont provide values that are very consistent, and the simple average of these two values has been used for the purposes of this characterization.

⁷⁵⁴ A review of TRM methodologies from Vermont, New York, Wisconsin, and Connecticut reveals several different estimates of ESF. Vermont is the only TRM that provides savings estimates dependent on the control type. Additionally, these estimates are the most conservative of all TRMs reviewed. These values have been adopted for the purposes of this characterization.

⁷⁵⁵ Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, February, 19, 2010

⁷⁵⁶ Energy Efficiency Supermarket Refrigeration, Wisconsin Electric Power Company, July 23, 1993

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-RFG-DHCT-V01-120601

REVIEW DEADLINE: 1/1/2019

4.6.4 Electronically Commutated Motors (ECM) for Walk-in and Reach-in Coolers / Freezers

DESCRIPTION

This measure is applicable to the replacement of an existing, uncontrolled, and continuously operating standard-efficiency shaded-pole evaporator fan motor in refrigerated display cases or fan coil in walk-ins.

This measure achieves savings by installing a more efficient motor, the result of which produces less waste heat that the cooling system must reject.

If applicable, savings from this measure may be claimed in combination with measure 4.6.6 Evaporator Fan Control for Electrically Commutated Motors.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

This measure applies to the replacement of an existing standard-efficiency shaded-pole evaporator fan motor in refrigerated display cases or fan coil in walk-ins. The replacement unit must be an electronically commutated motor (ECM) with a minimum efficiency of 66%. If controls are added as part of the motor upgrade to reduce annual run time, additional savings may potentially be claimed using measure 4.6.6 Evaporator Fan Control.

DEFINITION OF BASELINE EQUIPMENT

The baseline is the existing shaded-pole motor(s) with no fan control operating 8760 hours continuously in a refrigerated display case or fan coil unit of a walk-in cooling unit.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years⁷⁵⁷

DEEMED MEASURE COST

The measure cost is assumed to be \$177 per motor for a walk in cooler and walk in freezer. ⁷⁵⁸

LOADSHAPE

Loadshape C53 - Flat

COINCIDENCE FACTOR

The peak kW coincidence factor is 100%.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = \text{Savings per motor} * \text{motors}$$

Where:

⁷⁵⁷ DEER

⁷⁵⁸ Difference in the fully installed cost (\$468) for ECM motor and controller, listed in Work Paper PGE3PREF126, "ECM for Walk-In Evaporator with Fan Controller," June 20,2012, and the measure cost specified in 4.6.6 (\$291)

Savings per motor = based on the motor rating of the ECM motor:

Evaporator Fan Motor Rating (of ECM)	Annual kWh Savings/motor
16W	408
1/15 - 1/20HP	1,064
1/5HP	1,409
1/3HP	1,994
1/2HP	2,558
3/4HP	2,782

motors = number of fan motors replaced

SUMMER COINCIDENT PEAK DEMAND SAVINGS

ΔkW = $\Delta kWh / \text{Hours} * CF * \text{motors}$

Where:

ΔkWh = Gross customer annual kWh savings for the measure, as listed above

Hours = Full Load hours per year

= 8760

CF = Summer Peak Coincident Factor

= 1.0

Other variables as defined above.

The following table provides the resulting kW savings (per motor)

Evaporator Fan Motor Rating (of ECM)	Peak kW Savings/motor
16W	0.047
1/15 - 1/20HP	0.121
1/5HP	0.161
1/3HP	0.228
1/2HP	0.292
3/4HP	0.318

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-RFG-ECMF-V02-180101

REVIEW DEADLINE: 1/1/2022

4.6.5 ENERGY STAR Refrigerated Beverage Vending Machine

DESCRIPTION

ENERGY STAR qualified new and rebuilt vending machines incorporate more efficient compressors, fan motors, and lighting systems as well as low power mode option that allows the machine to be placed in low-energy lighting and/or low-energy refrigeration states during times of inactivity.

This measure was developed to be applicable to the following program types: TOS, NC .

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The refrigerated vending machine can be new or rebuilt but must meet the ENERGY STAR specifications which include low power mode.

DEFINITION OF BASELINE EQUIPMENT

The baseline vending machine is a standard unit

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The deemed lifetime of this measure is 14 years⁷⁵⁹

DEEMED MEASURE COST

The incremental cost of this measure is \$500⁷⁶⁰

LOADSHAPE

Loadshape C22 - Commercial Refrigeration

COINCIDENCE FACTOR

It is assumed that controls are only effective during off-peak hours and so have no peak-kW savings.

Algorithm

CALCULATION OF SAVINGS

Beverage machine savings are taken from the ENERGY STAR savings calculator and summarized in the following table. ENERGY STAR provides savings numbers for machines with and without control software. The average savings are calculated here.

ELECTRIC ENERGY SAVINGS

ENERGY STAR Vending Machine Savings⁷⁶¹

Vending Machine Capacity (cans)	kWh Savings Per Machine w/o software	kWh Savings Per Machine w/ software
<500	1,099	1,659

⁷⁵⁹ ENERGY STAR

⁷⁶⁰ ENERGY STAR

⁷⁶¹ Savings from Vending Machine Calculator:

http://www.energystar.gov/index.cfm?fuseaction=find_a_product.showProductGroup&pgw_code=VMC

Vending Machine Capacity (cans)	kWh Savings Per Machine w/o software	kWh Savings Per Machine w/ software
500 - 599	1,754	2,231
600 - 699	1,242	1,751
700 - 799	1,741	2,283
800+	713	1,288
Average	1,310	1,842

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-RFG-ESVE-V02-150601

REVIEW DEADLINE: 1/1/2019

4.6.6 Evaporator Fan Control for Electrically Commutated Motors

DESCRIPTION

This measure is for the installation of controls for Electronically Commutated Motors in existing medium temperature walk-in coolers. The controller reduces airflow of the evaporator fans when there is no refrigerant flow.

This measure achieves savings by controlling the motor(s) to run at lower speeds (or shut off entirely) when there is no refrigerant flow, the result of which produces less waste heat that the cooling system must reject.

If eligible, this measure may be claimed in combination with 4.6.4 Electronically Commutated Motors (ECM) for Walk-in and Reach-in Coolers / Freezers.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The measure must control a minimum of 1/20 HP where fans operate continuously at full speed. The measure also must reduce fan motor power by at least 75% during the off cycle. This measure is not applicable if any of the following conditions apply:

- The compressor runs more than 4380 hours annually
- The evaporator fan does not run at full speed all the time
- The evaporator fan motor runs on poly-phase power
- Evaporator does not use off-cycle or time-off defrost.

DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the existing condition must be a reach-in or walk-in freezer or cooler with continuously running evaporator fans driven by Electrically Commutated Motors

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 16 years⁷⁶²

DEEMED MEASURE COST

The measure cost is assumed to be \$291⁷⁶³

LOADSHAPE

Loadshape C46 - Evaporator Fan Control

COINCIDENCE FACTOR

The measure has deemed kW savings therefore a coincidence factor does not apply.

⁷⁶² Source: DEER

⁷⁶³ Source: DEER

Algorithm

CALCULATION OF SAVINGS

Savings are based on a measure created by Energy & Resource Solutions for the California Municipal Utilities Association⁷⁶⁴ and supported by a PGE workpaper. Note that climate differences across all California climate zones result in negligible savings differences, which indicates that the average savings for the California study should apply equally as well to Illinois. Savings found in the aforementioned source are presented in combination with savings from an ECM upgrade, however for the purposes of this measure only those associated with the controller are considered.

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = \text{Savings per motor} * \text{motors}$$

Where:

Savings per motor = based on the motor rating of the ECM motor:

Evaporator Fan Motor Rating (of ECM)	Annual kWh Savings/motor
16W	212
1/15 - 1/20HP	315
1/5HP	920
1/3HP	1,524
1/2HP	2,283
3/4HP	3,444

motors = number of fan motors controlled

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \text{Peak kW savings per motor (as listed in the table below)} * \text{motors (as defined above)}$$

Evaporator Fan Motor Rating (of ECM)	Peak kW Savings/motor
16W	0.024
1/15 - 1/20HP	0.036
1/5HP	0.105
1/3HP	0.174
1/2HP	0.261
3/4HP	0.393

NATURAL GAS ENERGY SAVINGS

N/A

⁷⁶⁴ See 'EC_motor_with_controller_182014.xlsx'.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-RFG-EVPF-V03-180101

REVIEW DEADLINE: 1/1/2024

4.6.7 Strip Curtain for Walk-in Coolers and Freezers

DESCRIPTION

This commercial measure pertains to the installation of infiltration barriers (strip curtains) on walk-in coolers or freezers. Strip curtains impede heat transfer from adjacent warm and humid spaces into walk-ins when the main door is opened, thereby reducing the cooling load. As a result, compressor run time and energy consumption are reduced. The engineering assumption is that the walk-in door is open for varying durations per day based on facility type, and the strip curtain covers the entire door frame. All assumptions are based on values that were determined by direct measurement and monitoring of over 100 walk-in units in the 2006-2008 evaluation for the CA Public Utility Commission.⁷⁶⁵

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is a strip curtain at least 0.06 inches thick⁷⁶⁶ added to a walk-in cooler or freezer. The new strip curtain must cover the entire area of the doorway when the door is opened.

DEFINITION OF BASELINE EQUIPMENT

The baseline assumption is a walk-in cooler or freezer that previously had either no strip curtain installed or an old, ineffective strip curtain installed.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 4 years⁷⁶⁷.

DEEMED MEASURE COST

The incremental capital cost for this measure is \$10.22/sq ft of door opening⁷⁶⁸

LOADSHAPE

Loadshape C22 - Commercial Refrigeration

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is 100%⁷⁶⁹.

⁷⁶⁵ The scale factors have been determined with tracer gas measurements on over 100 walk-in refrigeration units during the California Public Utility Commission's evaluation of the 2006-2008 CA investor owned utility energy efficiency programs. The door-open and close times, and temperatures of the infiltrating and refrigerated airs are taken from shortterm monitoring of over 100 walk-in units. http://www.calmac.org/publications/ComFac_Evaluation_V1_Final_Report_02-18-2010.pdf.

⁷⁶⁶ Pennsylvania Public Utility Commission TRM, chapter 3.5.9 Strip Curtains for Walk-in Freezers and Coolers.

⁷⁶⁷ DEER 2014 Effective Useful Life.

⁷⁶⁸ The reference for incremental cost is \$10.22 per square foot of door opening (includes material and labor). 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, "Cost Values and Summary Documentation", California Public Utilities Commission, December 16, 2008.

⁷⁶⁹ The summer coincident peak demand reduction is assumed as the total annual savings divided by the total number of hours per year, effectively assuming the average demand reduction is realized during the peak period. This is a reasonable assumption for refrigeration savings.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS⁷⁷⁰**

$$\Delta \text{kWh} = \Delta \text{kWh/sq ft} * A$$

Where:

$\Delta \text{kWh/sq ft}$ = Average annual kWh savings per square foot of infiltration barrier. Values can be found in Table 4.6.7 - 1.

A = Doorway area. If the actual doorway area in square feet is unknown, then use the values found in Table 4.6.7 - 2.

Table 4.6.7 - 1: Default Energy Savings and for Strip Curtains⁷⁷¹

Type	Pre-Existing Curtains	Energy Savings $\Delta \text{kWh/sq ft}$
Supermarket - Cooler	Yes	37
Supermarket - Cooler	No	108
Supermarket - Freezer	Yes	119
Supermarket - Freezer	No	349
Convenience Store - Cooler	Yes	5
Convenience Store - Cooler	No	20
Convenience Store - Freezer	Yes	8
Convenience Store - Freezer	No	27
Restaurant - Cooler	Yes	8
Restaurant - Cooler	No	30
Restaurant - Freezer	Yes	34
Restaurant - Freezer	No	119
Refrigerated Warehouse	Yes	254
Refrigerated Warehouse	No	729

Table 4.6.7 - 2: Default Doorway Area by Facility Type⁷⁷²

Facility Type	Doorway Area (sq ft)
Supermarket - Cooler	35
Supermarket - Freezer	35
Convenience Store - Cooler	21
Convenience Store - Freezer	21
Restaurant - Cooler	21
Restaurant - Freezer	21
Refrigerated Warehouse	80

⁷⁷⁰ The source algorithm from which the savings per square foot values are determined is based on Tamm's equation (an application of Bernoulli's equation) [Kaltverluste durch kuhlraumoffnungen. Tamm W., Kaltetechnik-Klimatisierung 1966;18;142-144;] and the ASHRAE handbook [American Society of Heating, Refrigeration, and Air-Conditioning Engineers (ASHRAE). 2010. ASHRAE Handbook, Refrigeration: 13.4, 13.6].

⁷⁷¹ Table 3-114 Default Energy Savings and Demand Reductions for Strip Curtains in Pennsylvania Public Utility Commission TRM, chapter 3.5.9 Strip Curtains for Walk-in Freezers and Coolers.

⁷⁷² Assumed Doorway area for four different facility types including supermarket, convenience store, restaurant and refrigerated warehouse. Pennsylvania Public Utility Commission 2016 TRM, chapter 3.5.9 Strip Curtains for Walk-in Freezers and Coolers.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh / 8766 * CF$$

Where:

8766 = hours per year

CF = Summer Peak Coincidence Factor for the measure
= 1.0

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-RFG-CRTN-V04-180101

REVIEW DEADLINE: 1/1/2022

4.6.8 Refrigeration Economizers

DESCRIPTION

This measure applies to commercial walk in refrigeration systems and includes two components, outside air economizers and evaporator fan controllers. Economizers save energy by bringing in outside air when weather conditions allow, rather than operating the compressor. Walk-in refrigeration systems evaporator fans run almost all the time; 24 hrs/day, 365 days/yr. This is because they must run constantly to provide cooling when the compressor is running, and to provide air circulation when the compressor is not running. However, evaporator fans are a very inefficient method of providing air circulation. Installing an evaporator fan control system will turn off evaporator fans while the compressor is not running, and instead turn on an energy-efficient 35 watt fan to provide air circulation, resulting in significant energy savings. This measure allows for economizer systems with evaporator fan controls plus a circulation fan and without a circulation fan.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure an economizer is installed on a walk in refrigeration system.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is a walk-in refrigeration system without an economizer

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The estimated life of this measure is 15 years⁷⁷³.

DEEMED MEASURE COST

The installation cost for an economizer is \$2,558.⁷⁷⁴

LOADSHAPE

Loadshape C22 - Commercial Refrigeration

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 0%⁷⁷⁵.

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

Electric energy savings is calculated based on whether evaporator fans run all

With Fan Control Installed

$$\Delta \text{kWh} = [\text{HP} * \text{kWhCond}] + [((\text{kWEvap} * \text{nFans}) - \text{kWCirc}) * \text{Hours} * \text{DCComp} * \text{BF}] - [\text{kWEcon} * \text{DCEcon} * \text{Hours}]$$

⁷⁷³ Estimated life from Efficiency Vermont TRM

⁷⁷⁴ Based on average of costs from Freeaire, Natural Cool, and Cooltrol economizer systems.

⁷⁷⁵ Based on the assumption that humidity levels will most likely be relatively high during the peak period, reducing the likelihood of demand savings.

Without Fan Control Installed

$$\Delta \text{kWh} = [\text{HP} * \text{kWhCond}] - [\text{kWEcon} * \text{DCEcon} * \text{Hours}]$$

Where:

HP = Horsepower of Compressor
= actual installed

kWhCond = Condensing unit savings, per hp. (value from savings table) ⁷⁷⁶

	Hermetic / Semi-Hermetic	Scroll	Discus
kWh/HP	1,256	1,108	1,051

Hours = Number of annual hours that economizer operates ⁷⁷⁷.

Region (city)	Hours
1 (Rockford)	2,376
2 (Chicago/O'Hare)	1,968
3 (Springfield)	1,728
4 (Bellevue)	1,488
5 (Marion)	1,224

DCComp = Duty cycle of the compressor
= 50% ⁷⁷⁸

kWEvap = Connected load kW of each evaporator fan,
= If known, actual installed. Otherwise assume 0.123 kW ⁷⁷⁹

kWCirc = Connected load kW of the circulating fan
= If known, actual installed. Otherwise assume 0.035 kW ⁷⁸⁰

nFans = Number of evaporator fans
= actual number of evaporator fans

DCEcon = Duty cycle of the economizer fan on days that are cool enough for the economizer to be working
= If known, actual installed. Otherwise assume 63% ⁷⁸¹

⁷⁷⁶ Savings table uses Economizer Calc.xls. Assume 5HP compressor size used to develop kWh/Hp value. No floating head pressure controls and compressor is located outdoors

⁷⁷⁷ In the source TRM (VT) this value was 2,996 hrs based on 38° F cooler setpoint, Burlington VT weather data, and 5 degree economizer deadband. The IL numbers were calculated by using weather bin data for each location (number of hours < 38F at each location is the Hours value).

⁷⁷⁸ A 50% duty cycle is assumed based on examination of duty cycle assumptions from Richard Travers (35%-65%), Cooltrol (35%-65%), Natural Cool (70%), Pacific Gas & Electric (58%). Also, manufacturers typically size equipment with a built-in 67% duty factor and contractors typically add another 25% safety factor, which results in a 50% overall duty factor. (as referenced by the Efficiency Vermont, Technical Reference User Manual)

⁷⁷⁹ Based on an a weighted average of 80% shaded pole motors at 132 watts and 20% PSC motors at 88 watts

⁷⁸⁰ Wattage of fan used by Freeaire and Cooltrol. This fan is used to circulate air in the cooler when the evaporator fan is turned off. As such, it is not used when fan control is not present

⁷⁸¹ Average of two manufacturer estimates of 50% and 75%.

BF = Bonus factor for reduced cooling load from running the evaporator fan less or (1.3)⁷⁸²
 kW_{Econ} = Connected load kW of the economizer fan
 = If known, actual installed. Otherwise assume 0.227 kW.⁷⁸³

EXAMPLE

For example, adding an outdoor air economizer and fan controls in Rockford to a 5 hp walk in refrigeration unit with 3 evaporator fans would save:

$$\begin{aligned}\Delta \text{kWh} &= [\text{HP} * \text{kWhCond}] + [((\text{kWEvap} * \text{nFans}) - \text{kWCirc}) * \text{Hours} * \text{DCComp} * \text{BF}] - [\text{kWEcon} * \text{DCEcon} * \text{Hours}] \\ &= [5 * 1256] + [((0.123 * 3) - 0.035) * 2376 * 0.5 * 1.3] - [0.227 * 0.63 * 2376] \\ &= 6456 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \Delta \text{kWh} / \text{Hours}$$

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-RFG-ECON-V05-150601

REVIEW DEADLINE: 1/1/2020

⁷⁸² Bonus factor (1+ 1/3.5) assumes COP of 3.5, based on the average of standard reciprocating and discus compressor efficiencies with a Saturated Suction Temperature of 20°F and a condensing temperature of 90°F

⁷⁸³ The 227 watts for an economizer is calculated from the average of three manufacturers: Freeaire (186 Watts), Cooltrol (285 Watts), and Natural Cool (218 Watts).

4.6.9 Night Covers for Open Refrigerated Display Cases

DESCRIPTION

This measure is the installation of fitted covers on existing open-type refrigerated and freezer display cases that are deployed during the facility unoccupied hours. Night covers are designed to reduce refrigeration energy consumption by reducing the work done by the compressor. Night covers reduce the heat and moisture entry into the refrigerated space through various heat transfer mechanisms. By fully or partially covering the case opening, night covers reduce the convective heat transfer into the case through reduced air infiltration. Additionally, they provide a measure of insulation, reducing conduction into the case, and also decrease radiation into the case by blocking radiated heat from entering the refrigerated space.

DEFINITION OF EFFICIENT EQUIPMENT

Curtains or covers on top of open refrigerated or freezer display cases that are applied at least six hours (during off-hours) in a 24-hour period.

DEFINITION OF BASELINE EQUIPMENT

Refrigerated and freezer, open-type display case in vertical, semi-vertical, and horizontal displays, with no night cover.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is 5 years, based on DEER 2014.⁷⁸⁴

DEEMED MEASURE COST

The incremental capital cost for this measure is \$42 per linear foot of cover installed including material and labor.⁷⁸⁵

LOADSHAPE

Loadshape 22: Commercial Refrigeration

COINCIDENCE FACTOR

N/A – savings occur at night only.

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = \text{ES} * \text{L}$$

Where:

ES = the energy savings ($\Delta \text{kWh/ft}$) found in table below:

⁷⁸⁴ 2014 Database for Energy-Efficiency Resources (DEER), Version 2014, "Cost Values and Summary Documentation", California Public Utilities Commission, January, 2014.

⁷⁸⁵ 2014 Database for Energy-Efficiency Resources (DEER), Version 2014, "Cost Values and Summary Documentation", California Public Utilities Commission, January, 2014.

Display Case Description	Case Temperature Range (°F)	Annual Electricity Use kWh/ft ⁷⁸⁶	ES ΔkWh/ft reduction (= 9% reduction of electricity use ^{787,788})
Vertical Open, Remote Condensing, Medium Temperature	35°F to 55°F	1453	131
Vertical Open, Remote Condensing, Low Temperature	0°F to 30°F	3292	296
Vertical Open, Self-Contained Medium Temperature	35°F to 55°F	2800	252
Horizontal Open, Remote Condensing, Medium Temperature	35°F to 55°F	439	40
Horizontal Open, Remote Condensing, Low Temperature	0°F to 30°F	1007	91
Horizontal Open, Self-Contained, Medium Temperature	35°F to 55°F	1350	121
Horizontal Open, Self-Contained, Low Temperature	0°F to 30°F	2749	247

L = the length of the refrigerated case in linear feet
= Actual

SUMMER COINCIDENT PEAK DEMAND SAVINGS

Peak savings are null because savings occur at night only.

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

⁷⁸⁶ Energy Conservation Standards for Commercial Refrigeration Equipment: Technical Support Document, U.S. Department of Energy, September 2013. The information required to estimate annual energy savings for refrigerated display cases is taken from the 2013-2014 U.S. Department of Energy (DOE) energy conservation standard rulemaking for Commercial Refrigerated Equipment. During the rulemaking process, DOE estimates the energy savings specific to night covers through extensive simulation and energy models that are validated by both manufacturers of night covers and refrigerated cases. The information is also referenced from a study done by Southern California Edison and testing by Technischer Überwachungs-Verein Rheinland, which are used by DOE for the rulemaking process.

⁷⁸⁷ Southern California Edison Refrigeration Technology and Test Center. Effects of the Low Emissivity Shields on Performance and Power Use of a Refrigerated Display Case. 1997. Southern California Edison, Rancho Cucamonga, CA.

⁷⁸⁸ Technischer Überwachungs-Verein Rheinland E.V. Laboratory test results for energy savings on refrigerated dairy case, conducted for Econofrost.

MEASURE CODE: CI-RFG-NCOV-V01-150601

REVIEW DEADLINE: 1/1/2024

4.6.10 High Speed Rollup Doors

DESCRIPTION

This measure entails the installation of High Speed Doors in refrigerated warehouses. High speed doors can save energy by lowering infiltration through a reduction in time that cooled spaces are exposed to ambient outdoor conditions. This in turn can lower the demand on refrigeration systems.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a High Speed Door installed on the loading dock doorway of a refrigerated space. The high speed door is assumed to act as a primary door. It should be noted that for high-traffic applications (about 45 door passages per hour, using the defaults for this measure) a custom analysis is necessary to ensure that high-speed rollup doors will provide savings, because strip curtains may outperform the high speed door, if no other open-door protection device is installed.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is existing strip curtains on doorways to a loading dock. During times of traffic, primary doors are left open, leaving just the strip curtains as open-doorway protection.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years.⁷⁸⁹

DEEMED MEASURE COST

The incremental measure cost is \$150/sq.ft.⁷⁹⁰

LOADSHAPE

Loadshape C22 - Commercial Refrigeration

COINCIDENCE FACTOR

The coincidence factor is assumed to be 1.00.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Electric savings consider the change in loading on the refrigeration system as well as the consumption of the drive on the high speed door. The following algorithms are based heavily on those derived and described in chapter 24 Refrigerated-Facility Loads of the ASHRAE Refrigeration Handbook.

$$\Delta kWh = (0.00008333 * q * D_f * \eta * [D_{tB}(1 - E_B) - D_{tE}(1 - E_E)] - D_{tM}M) * t$$

Where:

⁷⁸⁹ DEER 2008, Consulted from similar refrigerated warehouse measures.

⁷⁹⁰ Rite Hite – Industrial High Speed Doors

0.00008333 = conversion from Btu/h to tons

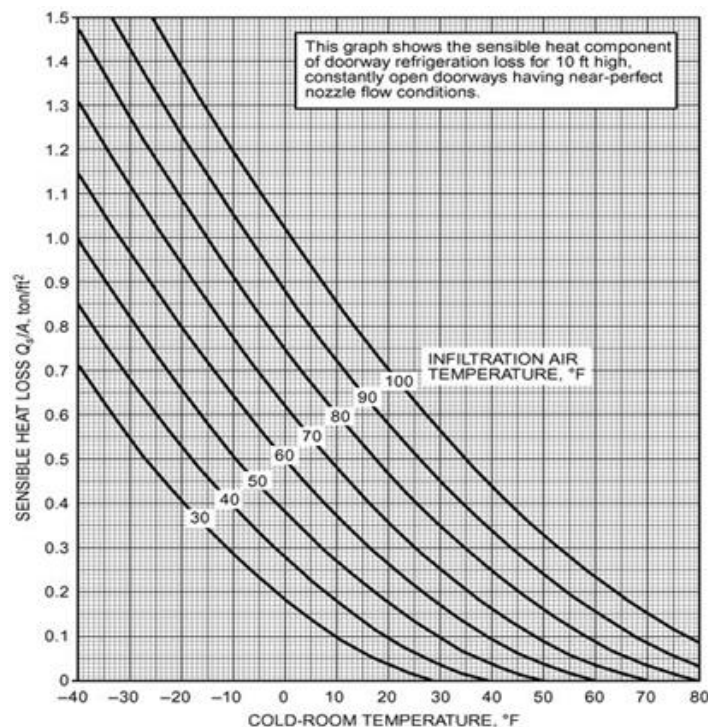
q = sensible and latent refrigeration load for fully established flow, Btu/h

$$= 3790 * W * H^{1.5} * \left(\frac{Q_s}{A}\right) * \left(\frac{1}{R_s}\right)$$

W = width of doorway, in feet. Custom input.

H = height of doorway, in feet. Custom input.

$\frac{Q_s}{A}$ = Sensible heat load of infiltration air per square foot of door way opening, as read from the following figure and dependent on infiltration air temperature and cooled space temperature. If unknown, infiltration temperature can be assumed to be 50°F⁷⁹¹, cooler temperature 35°F and freezer temperature -10°F⁷⁹², resulting in values of 0.06 for a cooler and 0.5 for a freezer.



R_s = Sensible heat ratio of the infiltration air heat gain, as read or interpolated from the chart below or from a psychrometric chart, dependent on temperature and relative humidity of infiltration air and cooled space temperature. If unknown, use the same assumptions as previously with a warm space relative humidity value of 70%⁷⁹³, resulting in values of 0.685 (interpolated) for coolers and 0.73 (interpolated) for freezers.

⁷⁹¹ Taken to represent the overall annual average temperature in Illinois. TMY3 data for the five weather regions defined by the TRM indicate averages that fall within the range of 47.6 (Rockford) to 55.9 (Marion).

⁷⁹² Refrigerated Warehouse, 2013 California Building Energy Standards, CODES AND STANDARDS ENHANCEMENT INITIATIVE (CASE), March 2011

⁷⁹³ Taken to represent the overall annual average in Illinois. TMY3 data for the five weather regions defined by the TRM indicate averages that fall within the range of 69.1 (Springfield) to 72.1 (Rockford).

Warm Space		Cold Space at 90% rh									
Temp.	rh	Dry-Bulb Temperature, °F									
°F	%	-40	-30	-20	-10	0	10	20	30	40	50
70	100	0.60	0.58	0.56	0.53	0.50	0.47	0.44	0.41	0.37	0.34
	80	0.66	0.64	0.61	0.59	0.56	0.53	0.50	0.48	0.46	0.44
	60	0.72	0.70	0.68	0.66	0.63	0.61	0.59	0.58	0.59	0.64
	40	0.79	0.78	0.76	0.75	0.73	0.72	0.71	0.73	0.80	—
60	100	0.66	0.64	0.62	0.59	0.56	0.52	0.49	0.45	0.41	0.35
	80	0.71	0.69	0.67	0.64	0.62	0.59	0.56	0.53	0.52	0.53
	60	0.77	0.75	0.73	0.71	0.69	0.67	0.65	0.65	0.70	—
	40	0.83	0.82	0.81	0.79	0.78	0.77	0.78	0.83	—	—
50	100	0.72	0.70	0.67	0.64	0.61	0.57	0.53	0.49	0.43	—
	80	0.76	0.74	0.72	0.70	0.67	0.64	0.61	0.59	0.62	—
	60	0.81	0.80	0.78	0.76	0.74	0.72	0.71	0.75	—	—
	40	0.87	0.86	0.84	0.83	0.82	0.82	0.85	—	—	—
40	100	0.77	0.75	0.72	0.69	0.66	0.62	0.57	0.51	—	—
	80	0.81	0.79	0.77	0.74	0.72	0.69	0.66	0.67	—	—
	60	0.85	0.84	0.82	0.80	0.78	0.77	0.79	0.99	—	—
	40	0.90	0.89	0.88	0.87	0.86	0.88	0.97	—	—	—
30	100	0.82	0.80	0.77	0.74	0.70	0.66	0.59	—	—	—
	80	0.85	0.83	0.81	0.79	0.76	0.73	0.73	—	—	—
	60	0.88	0.87	0.86	0.84	0.83	0.83	0.94	—	—	—
	40	0.92	0.91	0.90	0.90	0.91	0.96	—	—	—	—
20	100	0.86	0.84	0.82	0.79	0.75	0.69	—	—	—	—
	80	0.89	0.87	0.85	0.83	0.81	0.80	—	—	—	—
	60	0.91	0.90	0.89	0.88	0.88	0.95	—	—	—	—
	40	0.94	0.94	0.93	0.94	0.97	—	—	—	—	—
10	100	0.90	0.88	0.86	0.83	0.78	—	—	—	—	—
	80	0.92	0.90	0.89	0.87	0.86	—	—	—	—	—
	60	0.94	0.93	0.92	0.92	0.96	—	—	—	—	—
	40	0.96	0.96	0.96	0.98	—	—	—	—	—	—
0	100	0.92	0.91	0.89	0.85	—	—	—	—	—	—
	80	0.94	0.93	0.92	0.91	—	—	—	—	—	—
	60	0.96	0.95	0.95	0.97	—	—	—	—	—	—
	40	0.97	0.97	0.98	—	—	—	—	—	—	—

D_f = doorway flow factor. Equal to 0.8 for a doorway between a freezer and a dock and 1.1 for a doorway between a cooler and a dock⁷⁹⁴.

η = Efficiency of refrigeration system (kW/ton). Custom input, if unknown assume 1.6 kW/ton for coolers and 2.4 kW/ton⁷⁹⁵ for freezers.

D_{tB} = decimal portion of time doorway is open in the baseline condition. If during facility operating hours, the primary doors are left open, leaving only open-doorway protective devices (e.g., strip curtains) as a barrier, this is considered 1.0. If primary doors are actively operated and do not remain open for the entire time the facility is in operation, refer to the following calculation.

$$D_{tB} = \frac{(P \theta_{pB} + 60 \theta_{oB})}{3600 \theta_d}$$

P = Number of passages through doorway per hour.

θ_{pB} = Door open to close time in seconds.

θ_{oB} = Time door remains open in minutes.

θ_d = Period of time considered in hours, 1 hr.

D_{tE} = decimal portion of time doorway is open in the efficient condition.

⁷⁹⁴ ASHRAE, "Refrigerated –Facility Loads", in Refrigeration Handbook 2014: ASHRAE, 2014, 24.7

⁷⁹⁵ Professional judgement, in alignment with typical freezer and cooler performance found in the Michigan Energy Measures Database (MEMD).

$$D_{tE} = \frac{(P \theta_{pE} + 60 \theta_{oE})}{3600 \theta_d}$$

P = Number of passages through doorway per hour. Custom input, assume 5.9⁷⁹⁶ if unknown.

θ_{pE} = Door open to close time in seconds. Custom input, assume 7.5 seconds⁷⁹⁷ if unknown.

θ_{oE} = Time door remains open in minutes. Custom input, assume 3 minutes⁷⁹⁸ if unknown.

θ_d = Period of time considered in hours, 1 hr.

D_{tM} = decimal portion of time high speed door motor is operational.

$$D_{tM} = \frac{P \theta_{pE}}{3600 \theta_d}$$

Variables defined above.

E_B = effectiveness of baseline open-doorway protective device (strip curtains). Equal to 0.85⁷⁹⁹.

E_E = effectiveness of efficient open-doorway protective device. Equal to 0, unless an additional protective device exists to limit infiltration during times when the high-speed door is open.

M = operating input power of the high speed door motor, in kW.

= Custom input, assume 1.49kW⁸⁰⁰ if unknown.

t = hours per year when primary doors to the cooled space are open.

= Custom input, assume 2,959 hrs/yr⁸⁰¹ if unknown.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = (\Delta kWh / t) * CF$$

Where

CF = Summer peak coincidence factor for this measure

= 1.0

All other variables as defined above.

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

⁷⁹⁶ ASHRAE, "Refrigerated –Facility Loads", in Refrigeration Handbook 2014: ASHRAE, 2014, 24.11

⁷⁹⁷ ASHRAE, "Refrigerated –Facility Loads", in Refrigeration Handbook 2014: ASHRAE, 2014, 24.6

⁷⁹⁸ Professional judgement

⁷⁹⁹ ASHRAE, "Refrigerated –Facility Loads", in Refrigeration Handbook 2014: ASHRAE, 2014, 24.7

⁸⁰⁰ Rite Hite – Industrial High Speed Doors, product line commonly uses 2HP drives.

⁸⁰¹ Based on a ComEd survey that obtained the number of hours per week certain building types operate. Warehouses had an average response of 55.6 and industrials had 58.2. Calculated by taking the simple average of the two and multiplying by 52 weeks/yr.

DEEMED O&M COST ADJUSTMENT CALCULATION

Manufacturers suggest annual inspection and maintenance (such as patching tears) of high speed doors. At a minimum, greasing of fittings and oil top-off should be carried out annually. This is estimated at a cost of \$150 per year⁸⁰².

MEASURE CODE: CI-RFG-HSRD-V01-180101

REVIEW DEADLINE: 1/1/2022

⁸⁰² Assumes approximately 1 hour of maintenance, based on manufacturer product spec sheets.

4.7 Compressed Air

4.7.1 VSD Air Compressor

DESCRIPTION

This measure relates to the installation of an air compressor with a variable frequency drive, load/no load controls or variable displacement control. The baseline compressors defined choke off the inlet air to modulate the compressor output, which is not efficient. Efficient compressors use a variable speed drive on the motor to match output to the load. Savings are calculated using representative baseline and efficient demand numbers for compressor capacities according to the facility's load shape, and the number of hours the compressor runs at that capacity. Demand curves are as per DOE data for a Variable Speed compressor versus a Modulating compressor. This measure applies only to an individual compressor ≤ 40 hp

This measure was developed to be applicable to the following program types: TOS.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The high efficiency equipment is a compressor ≤ 40 hp with variable speed control.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is a modulating compressor with blow down ≤ 40 hp

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

10 years.

DEEMED MEASURE COST

$$\text{IncrementalCost (\$)} = (127 \times \text{hp}_{\text{compressor}}) + 1446$$

Where:

127 and 1446⁸⁰³ = compressor motor nominal hp to incremental cost conversion factor and offset

hp_{compressor} = compressor motor nominal

DEEMED O&M COST ADJUSTMENTS

N/A

LOADSHAPE

Loadshape C35 - Industrial Process

COINCIDENCE FACTOR

The coincidence factor equals 0.95

⁸⁰³ Conversion factor and offset based on a linear regression analysis of the relationship between air compressor motor nominal horsepower and incremental cost. Several Vermont vendors were surveyed to determine the cost of equipment. See "Compressed Air Analysis.xls" and "Compiled Data ReQuest Results.xls" for incremental cost details.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = 0.9 \times \text{hp}_{\text{compressor}} \times \text{HOURS} \times (\text{CF}_b - \text{CF}_e)$$

Where:

 ΔkWh = gross customer annual kWh savings for the measure $\text{hp}_{\text{compressor}}$ = compressor motor nominal hp 0.9^{804} = compressor motor nominal hp to full load kW conversion factor

HOURS = compressor total hours of operation below depending on shift

Shift	Hours
Single shift (8/5)	1976 hours 7 AM – 3 PM, weekdays, minus some holidays and scheduled down time
2-shift (16/5)	3952 hours 7AM – 11 PM, weekdays, minus some holidays and scheduled down time
3-shift (24/5)	5928 hours 24 hours per day, weekdays, minus some holidays and scheduled down time
4-shift (24/7)	8320 hours 24 hours per day, 7 days a week minus some holidays and scheduled down time

CF_b = baseline compressor factor⁸⁰⁵
=0.890

CF_e = efficient compressor⁸⁰⁶
=0.705

EXAMPLE

For example a VFD compressor with 10 HP operating in a 1 shift facility would save

$$\begin{aligned} \Delta \text{kWh} &= 0.9 \times 10 \times 1976 \times (0.890 - 0.705) \\ &= 3290 \text{ kWh} \end{aligned}$$

⁸⁰⁴ Conversion factor based on a linear regression analysis of the relationship between air compressor motor nominal horsepower and full load kW from power measurements of 72 compressors at 50 facilities on Long Island. See "BHP Weighted Compressed Air Load Profiles v2.xls".

⁸⁰⁵ Compressor factors were developed using DOE part load data for different compressor control types as well as load profiles from 50 facilities employing air compressors less than or equal to 40 hp. "See "BHP Weighted Compressed Air Load Profiles.xls" for source data and calculations (The "variable speed drive" compressor factor has been adjusted up from the 0.675 presented in the analysis to 0.705 to account for the additional power draw of the VSD).

⁸⁰⁶ Ibid.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh / \text{HOURS} * CF$$

EXAMPLE

For example a VFD compressor with 10 HP operating in a 1 shift facility would save

$$\begin{aligned}\Delta kW &= 3290/1976 * .95 \\ &= 1.58 \text{ kW}\end{aligned}$$

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-MSC-VSDA-V01-120601

REVIEW DEADLINE: 1/1/2019

4.7.2 Compressed Air Low Pressure Drop Filters

DESCRIPTION

Low pressure drop filters remove solids and aerosols from compressed air systems with a longer life and lower pressure drop than standard coalescing filters, resulting in better efficiencies.

This measure was developed to be applicable to the following program types: RF. If applied to other program types, the measure savings should be verified

DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is a low pressure drop filter with pressure drop not exceeding 1 psid when new and 3 psid at element change.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is a standard coalescing filter with a pressure drop of 3 psid when new and 5 or more at element change

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

5 years

DEEMED MEASURE COST

The incremental cost for this measure is estimated to be \$1000 Incremental cost per filter⁸⁰⁷

LOADSHAPE

Loadshape C35 - Industrial Process

COINCIDENCE FACTOR

The coincidence factor equals 0.95

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = (\text{kW}_{\text{typical}} \times \Delta P \times \text{SF} \times \text{Hours} / \text{HP}_{\text{typical}}) \times \text{HP}_{\text{real}}$$

Where:

$\text{kW}_{\text{typical}}$ = Adjusted compressor power (kW) based on typical compressor loading and operating profile. Use actual compressor control type if known:

⁸⁰⁷ Incremental cost research found in LPDF Costs. xlsx

Compressor kW_{typical}

Control Type	kW _{typical} ⁸⁰⁸
Reciprocating - On/off Control	70.2
Reciprocating - Load/Unload	74.8
Screw - Load/Unload	82.3
Screw - Inlet Modulation	82.5
Screw - Inlet Modulation w/ Unloading	82.5
Screw - Variable Displacement	73.2
Screw - VFD	70.8

= If the actual compressor control type is not known, then use a weighted average based on the following market assumptions:

Control Type	Share %	kW _{typical} ⁸⁰⁹
Market share estimation for load/unload control compressors	40%	74.8
Market share estimation for modulation w/unloading control compressors	40%	82.5
Market share estimation for variable displacement control compressors	20%	73.2
Weighted Average		77.6

ΔP = Reduced filter loss (psi)

=2 psi⁸¹⁰

SF =1% reduction in power per 2 psi reduction in system pressure is equal to 0.5% reduction per 1 psi, or a Savings Factor of 0.005⁸¹¹

Hours = depending on shifts

Single shift (8/5) – 1976 hours (7 AM – 3 PM, weekdays, minus some holidays and scheduled down time) + 500 hrs maintenance = 2476 hrs

2-shift (16/5) – 3952 hours (7AM – 11 PM, weekdays, minus some holidays and scheduled down time) + 500 hrs maintenance = 4452 hrs

3-shift (24/5) – 5928 hours (24 hours per day, weekdays, minus some holidays and scheduled down time) + 500 hrs maintenance = 6428 hrs

4-shift (24/7) – 8320 hours (24 hours per day, 7 days a week minus some holidays and scheduled down time)

HP_{typical} = Nominal HP for typical compressor = 100 hp⁸¹²

⁸⁰⁸ See "Industrial System Standard Deemed Saving Analysis.xls"

⁸⁰⁹ See "Industrial System Standard Deemed Saving Analysis.xls"

⁸¹⁰ Assumed pressure will be reduced from a roughly 3 psi pressure drop through a filter to less than 1 psi, for a 2 psi savings

⁸¹¹ "Optimizing pneumatic systems for extra savings," 10, 2010, <http://www.compressedairchallenge.org/library/articles/2010-10-CABP.pdf>

⁸¹² Industrial System Standard Deemed Saving Analysis.xls

HP_{real} = Total HP of real compressors distributing air through filter. This should include the total horsepower of the compressors that normally run through the filter, but not backup compressors

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh / \text{HOURS} * CF$$

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-MSC-CALPDF-V01-140601

REVIEW DEADLINE: 1/1/2020

4.7.3 Compressed Air No-Loss Condensate Drains

DESCRIPTION

No-loss condensate drains remove condensate as needed without venting compressed air, resulting in less air demand and consequently better efficiency. Replacement or upgrades of existing no-loss drains are not eligible for the incentive.

This measure was developed to be applicable to the following program types: RF. If applied to other program types, the measure savings should be verified

DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is installation of no-loss condensate drains.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is installation of standard condensate drains (open valve, timer, or both)

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

10 years

DEEMED MEASURE COST

\$700 per drain ⁸¹³

LOADSHAPE

Loadshape C35 - Industrial Process

COINCIDENCE FACTOR

The coincidence factor equals 0.95

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = \text{CFM}_{\text{reduced}} \times \text{kW}_{\text{CFM}} \times \text{Hours}$$

Where:

$$\begin{aligned} \text{CFM}_{\text{reduced}} &= \text{Reduced air consumption (CFM) per drain} \\ &= 3 \text{ CFM}^{814} \end{aligned}$$

$$\text{kW}_{\text{CFM}} = \text{System power reduction per reduced air demand (kw/CFM) depending on the type of compressor control:}$$

⁸¹³ Based on empirical project data from ComEd Comprehensive Compressed Air Study program and VEIC review of pricing data found in CAS Cost Data.xls

⁸¹⁴ Reduced CFM consumption is based on an a timer drain opening for 10 seconds every 300 seconds as the baseline. See "Industrial System Standard Deemed Saving Analysis.xls"

System Power Reduction per Reduced Air Demand⁸¹⁵

Control Type	kW / CFM
Reciprocating - On/off Control	0.184
Reciprocating - Load/Unload	0.136
Screw - Load/Unload	0.152
Screw - Inlet Modulation	0.055
Screw - Inlet Modulation w/ Unloading	0.055
Screw - Variable Displacement	0.153
Screw - VFD	0.178

Or if compressor control type is unknown, then a weighted average based on market share can be used:

Control Type	Share %	kW / CFM
Market share estimation for load/unload control compressors	40%	0.136
Market share estimation for modulation w/unloading control compressors	40%	0.055
Market share estimation for variable displacement control compressors	20%	0.153
Weighted Average		0.107

Hours = Compressed air system pressurized hours
=6136 hours⁸¹⁶

SUMMER COINCIDENT PEAK DEMAND SAVINGS

ΔkW = $\Delta kWh / \text{HOURS} * CF$

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-MSC-CANCLD-V01-140601

REVIEW DEADLINE: 1/1/2020

⁸¹⁵ Calculated based on the type of compressor control. This assumes the compressor will be between 40% and 100% capacity before and after the changes to the system demand. See "Industrial System Standard Deemed Saving Analysis.xls"

⁸¹⁶ US DOE, Evaluation of the Compressed Air Challenge® Training Program, Page 19

4.7.4 Efficient Compressed Air Nozzles

DESCRIPTION

This measure is for the replacement of standard air nozzle with high-efficiency air nozzle used in a compressed air system. High-efficiency air nozzles reduce the amount of air required to blow off parts or for drying. These nozzles utilize the Coandă effect to pull in free air to accomplish tasks with significantly less compressed air. High-efficiency nozzles often replace simple copper tubes. These nozzles have the added benefits of noise reduction and improved safety in systems with greater than 30 psig.

DEFINITION OF EFFICIENT EQUIPMENT

The high-efficiency air nozzle must meet the following specifications:

1. High-efficiency air nozzle must replace continuous open blow-offs
2. High-efficiency air nozzle must meet SCFM rating at 80psig less than or equal to: 1/8" 11 SCFM, 1/4" 29 SCFM, 5/16" 56 SCFM, 1/2" 140 SCFM.
3. Manufacturer's specification sheet of the high-efficiency air nozzle must be provided along with the make and model

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is a standard air nozzle

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is 15 years⁸¹⁷

DEEMED MEASURE COST

The estimated incremental measure costs are presented in the following table⁸¹⁸

Nozzle Diameter	1/8"	1/4"	5/16"	1/2"
Average IMC	\$42	\$57	\$87	\$121

LOADSHAPE

Loadshape C35 - Industrial Process

COINCIDENCE FACTOR

The coincidence factor equals 0.95

⁸¹⁷ PA Consulting Group (2009). Business Programs: Measure Life Study. Prepared for State of Wisconsin Public Service Commission.

⁸¹⁸ Costs are from EXAIR's website and are an average of nozzles that meet the flow requirements. Models include Atto Super, Pico Super, Nano Super, Micro Super, Mini Super, Super and Large Super nozzles. www.exair.com. Accessed March 20, 2014

Algorithm

CALCULATION OF ENERGY SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = (\text{SCFM} * \text{SCFM}\% \text{Reduced}) * \text{kW}/\text{CFM} * \% \text{USE} * \text{HOURS}$$

Where:

SCFM = Air flow through standard nozzle. Use actual rated flow at 80 psi if known. If unknown, the table below includes the CFM by orifice diameter^{819, 820}.

Orifice Diameter	SCFM
1/8"	21
1/4"	58
5/16"	113
1/2"	280

SCFM%Reduced = Percent in reduction of air loss per nozzle. Estimated at 50%⁸²¹

kW/CFM = System power reduction per air demand (kW/CFM) depending on the type of air compressor found in table below⁸²²

Air Compressor Type	ΔkW/CFM
Reciprocating – On/off Control	0.18
Reciprocating – Load/Unload	0.14
Screw – Load/Unload	0.15
Screw – Inlet Modulation	0.06
Screw – Inlet Modulation w/ Unloading	0.06
Screw – Variable Displacement	0.15
Screw - VFD	0.18

%USE = percent of the compressor total operating hours that the nozzle is in use

= Custom, if unknown assume 5%⁸²³

Hours = Compressed air system pressurized hours.

= Use actual hours if known, otherwise assume values in table below:

⁸¹⁹ Review of manufacturer's information

⁸²⁰ Technical Reference Manual (TRM) for Ohio Senate Bill 221 "Energy Efficiency and Conservation Program" and 09-512-GE-UNC, October 15, 2009. Pgs 170-171

⁸²¹ Conservative estimate based on average values provided by the Compressed Air Challenge Training Program, Machinery's Handbook 25th Edition, and manufacturers' catalog.

⁸²² Calculated based on the type of compressor control. This assumes the compressor will be between 40% and 100% capacity before and after the changes to the system demand. See "Industrial System Standard Deemed Saving Analysis.xls"

⁸²³ Assumes 50% handheld air guns and 50% stationary air nozzles. Manual air guns tend to be used less than stationary air nozzles, and a conservative estimate of 1 second of blow-off per minute of compressor run time is assumed. Stationary air nozzles are commonly more wasteful as they are often mounted on machine tools and can be manually operated resulting in the possibility of a long term open blow situation. An assumption of 5 seconds of blow-off per minute of compressor run time is used.

Shift	Hours
Single Shift	1976
Two Shifts	3952
Three Shifts	5928
Four Shifts or Continual Operation	8320
Unknown / Weighted average ⁸²⁴	5702

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh / \text{HOURS} * CF$$

Where:

ΔkWh = As calculated above

CF = 0.95

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE CI-MSC-CNOZ-V01-150601

REVIEW DEADLINE: 1/1/2022

⁸²⁴ Weighting of 16% single shift, 23% two shift, 25% three shift and 36% continual based on DOE evaluation of the Compressed Air Challenge, section 2.1.5 Facility Operating Schedules

4.7.5 Efficient Refrigerated Compressed Air Dryer

DESCRIPTION

An air dryer is an essential component in a compressed air system that prevents condensate from being deposited in the compressed air supply lines of a facility. If the warm, saturated compressed air is supplied directly into the plant, excess condensate will form in the compressed air supply lines. Uncontrolled condensate can damage demand-side tools and process equipment. Secondly, in an oil-flooded rotary screw compressor, the residual oil from compression can be carried along the supply lines potentially damaging process equipment. Industries that use compressed air for processes make use of various types of dryers including refrigerated dryers (both cycling and non-cycling). For this measure, three types of refrigerated air dryers will be considered: thermal mass, variable speed and digital scroll. All of these technologies offer better part load performance compared to non-cycling refrigerated dryers, thereby offering energy savings during periods when the dryer is not operating at peak capacity.

This measure was developed to be applicable to the following program types: TOS.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

A new, high efficiency thermal mass dryer, variable speed dryer, or digital scroll dryer.

DEFINITION OF BASELINE EQUIPMENT

A standard non-cycling refrigerated compressed air dryer of comparable capacity.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

THE MEASURE LIFE IS 10 YEARS⁸²⁵.

DEEMED MEASURE COST

The incremental capital cost for this measure is \$6 per CFM.⁸²⁶

LOADSHAPE

Loadshape C35 – Industrial Process

COINCIDENCE FACTOR

The coincidence factor equals 0.95

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = P_s \times (\text{EC50}_{\text{baseline}} - \text{EC50}_{\text{efficient}}) \times \text{HOURS} \times \text{CFM}$$

⁸²⁵ State of Wisconsin Public Service Commission, Focus on Energy Evaluation, Business Programs: Measure Life Study, August 25, 2009.

⁸²⁶ Analysis of material cost between cycling and non-cycling dryers according to prices from Grainger. Cost provided is the average incremental cost when comparing non-cycling and cycling dryers of the same CFM capacity. <http://www.grainger.com/category/refrigerated-compressed-air-dryers/compressed-air-treatment/pneumatics/ecatalog/N-kk5?bc=y>

Where:

- P_s = Full flow specific power of the dryer
 = 0.007 kW/CFM⁸²⁷ (for both baseline and efficient equipment)
- $EC50_{baseline}$ = Energy consumption ratio of baseline dryer at 50%⁸²⁸ inlet load capacity as compared to fully loaded operating conditions.⁸²⁹
 = 0.843
- $ECF50_{efficient}$ = Energy consumption ratio of efficient dryer at 50% inlet load capacity as compared to fully loaded operating conditions.
 = Dependent on efficient dryer type, refer to the following table⁸³⁰:

Dryer Type	$EC50_{efficient}$
Thermal-Mass	0.729
VSD	0.501
Digital Scroll	0.551

- HOURS = Compressed air system pressurized hours, depending on shift. If unknown, use weighted average. This value is the weighted average of facility owner responses from the DOE evaluation of the Compressed Air Challenge. Facility owners with compressed air systems were surveyed detailing the number of shifts their facilities operated.

Shift	Hours	Distribution of Facilities by Hours of Operation ⁸³¹	Weighted Hours
Single Shift 7 AM – 3 PM, weekdays, minus some holidays and scheduled down time	1,976	16%	316
Two Shifts 7AM – 11 PM, weekdays, minus some holidays and scheduled down time	3,952	23%	909
Three Shifts 24 hours per day, weekdays, minus some holidays and scheduled down time	5,928	25%	1,482
Four Shifts or Continual Operation 24 hours per day, 7 days a week minus some holidays and scheduled down time	8,320	36%	2,995
Total weighted average			5,702

- CFM = Cubic feet per minute, rate of airflow through the dryer.

⁸²⁷ Compressed Air Challenge: Compressed Air Best Practice; “Cycling Air Dryers – Are Savings Significant?” Fox, Timothy J. and Marshall, Ron. <http://www.compressedairchallenge.org/library/articles/2011-11-CABP.pdf>

⁸²⁸ Engineering judgement, based on the assumption that on average, compressed air systems will operate at 50% capacity.

⁸²⁹ Compressed Air Challenge: Compressed Air Best Practice; “Cycling Air Dryers – Are Savings Significant?” Fox, Timothy J. and Marshall, Ron. <http://www.compressedairchallenge.org/library/articles/2011-11-CABP.pdf>

⁸³⁰ Compressed Air Challenge: Compressed Air Best Practice; “Cycling Air Dryers – Are Savings Significant?” Fox, Timothy J. and Marshall, Ron. <http://www.compressedairchallenge.org/library/articles/2011-11-CABP.pdf>

⁸³¹ DOE evaluation of the Compressed Air Challenge, section 2.1.5 Facility Operating Schedules.

= Assume 50% of actual rated capacity.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh / \text{HOURS} * CF$$

Where:

$$CF = 0.95$$

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-CPA-CADR-V01-160601

REVIEW DEADLINE: 1/1/2023

4.8 Miscellaneous End Use

4.8.1 Pump Optimization

DESCRIPTION

Pump improvements can be done to optimize the design and control of centrifugal water pumping systems, including water solutions with freeze protection up to 15% concentration by volume. Other fluid and gas pumps cannot use this measure calculation. The measurement of energy and demand savings for commercial and industrial applications will vary with the type of pumping technology, operating hours, efficiency, and existing and proposed controls. Depending on the specific application slowing the pump, trimming or replacing the impeller may be suitable options for improving pumping efficiency. Pumps up to 40 HP are allowed to use this energy savings calculation. Larger motors should use a custom calculation (which may result in larger savings that this measure would claim).

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is proven to be an optimized centrifugal pumping system meeting the applicable program efficiency requirements:

- Pump balancing valves no more than 15% throttled
- Balancing valves on at least one load 100% open.

DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, the baseline equipment is assumed to be the existing pumping system including existing controls and sequence of operations.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years⁸³²

DEEMED MEASURE COST

The incremental capital cost for this measure can vary considerably depending upon the strategy employed to achieve the required efficiency levels and should be determined on a site-specific basis.

DEEMED O&M COST ADJUSTMENTS

N/A

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 38%⁸³³

⁸³² Martin, N. et al., Emerging Energy-Efficient Industrial Technologies: New York State Edition, American Council for an Energy Efficient Economy (ACEEE), March 2001 (as stated in the OH State TRM, page 269)

⁸³³ Summer Peak Coincidence Factor has been preserved from the "Technical Reference Manual" (TRM) for Ohio Senate Bill 221 Energy Efficiency and Conservation Program and 09-512-GE-UNC," October 15, 2009. This is likely a conservative estimate, but is recommended for further study (as stated in the OH State TRM, page 269)

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta kWh = (HP_{motor} * 0.746 * LF / \eta_{motor}) * HOURS * ESF$$

Where:

HP_{motor} = Installed nameplate motor horsepower
= Actual

0.746 = Conversion factor from horse-power to kW (kW/hp)

LF / η_{motor} = Combined as a single factor since efficiency is a function of load
= 0.65⁸³⁴

Where:

LF = Load Factor; Ratio of the peak running load to the nameplate rating of the motor

η_{motor} = Motor efficiency at pump operating conditions

$HOURS$ = Annual operating hours of the pump
= Actual

ESF = Energy Savings Factor; assume a value of 15%⁸³⁵.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = (HP_{motor} * 0.746 * (LF / \eta_{motor})) * (ESF) * CF$$

Where:

CF = Summer Coincident Peak Factor for measure

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

⁸³⁴ "Measured Loading of Energy Efficient Motors - the Missing Link in Engineering Estimates of Savings," ACEEE 1994 Summer Study Conference, Asilomar, CA.

⁸³⁵ Published estimates of typical pumping efficiency improvements range from 5 to 40%. For analysis purposes, assume 15%. United States Industrial Electric Motor Systems Market Opportunities Assessment December 2002, Table E-7, Page 18, https://www1.eere.energy.gov/manufacturing/tech_assistance/pdfs/mtrmkt.pdf

MEASURE CODE: CI-MSC-PMPO-V01-150601

REVIEW DEADLINE: 1/1/2022

4.8.2 Roof Insulation for C&I Facilities

DESCRIPTION

Energy and demand saving are realized through reductions in the building cooling and heating loads. This measure was developed to be applicable to the following program types: RF and NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is above code and should be determined by the program.

DEFINITION OF BASELINE EQUIPMENT

The retrofit baseline condition is adopted from Ohio Energy Technical Reference Manual and expanded to cover all type of commercial buildings in the state of Illinois as follows.

For retrofits, the R-value for the entire assembly:

Building Type	Retrofit Assembly R-Value
Assembly	13.5
Assisted Living	13.5
College	13.5
Convenience Store	13.5
Elementary School	13.5
Garage	13.5
Grocery	13.5
Healthcare Clinic	13.5
High School	13.5
Hospital	13.5
Hotel/Motel	13.5
Manufacturing Facility	12
MF - High Rise	13.5
MF - Mid Rise	13.5
Movie Theater	13.5
Office - High Rise	13.5
Office - Low Rise	13.5
Office - Mid Rise	13.5
Religious Building	13.5
Restaurant	13.5
Retail - Department Store	13.5
Retail - Strip Mall	13.5
Warehouse	12
Unknown	13.5

For new construction use R-value from IECC 2012 or ASHRAE – 90.1 – 2010, or use IECC 2015 or ASHRAE – 90.1 – 2013, depending on the IECC in effect on the date of the building permit (if unknown assume IECC 2015)..

R-Values: ASHRAE – 90.1 – 2010

		IL TRM Zones 1, 2, & 3 [ASHRAE/IECC Climate Zone 5 (A, B, C)]			
		Nonresidential		Semiheated	
		Assembly Maximum	Insulation Min. R-Value	Assembly Maximum	Insulation Min. R-Value
Insulation Above Deck	Entirely	0.048	R-20 c.i.	U-0.119	R-7.6 c.i.
Metal Building (Roof)		0.055	R-13.0 + R-13.0	U-0.083	R-13.0
Attic and Other		0.027	R-38.0	U-0.053	R-19.0

		IL TRM Zones 4 & 5 [ASHRAE/IECC Climate Zone 4 (A, B, C)]			
		Nonresidential		Semiheated	
		Assembly Maximum	Insulation Min. R-Value	Assembly Maximum	Insulation Min. R-Value
Insulation Above Deck	Entirely	0.048	R-20.0 c.i.	0.173	R-5.0 c.i.
Metal Building (Roof)		0.055	R-13.0 + R-13.0	0.097	R-10.0
Attic and Other		0.027	R-38.0	0.053	R-19.0

Table Notes
c.i. = continuous insulation

R-Values: ASHRAE – 90.1 – 2010

		IL TRM Zones 1, 2, & 3 [ASHRAE/IECC Climate Zone 5 (A, B, C)]			
		Nonresidential		Semiheated	
		Assembly Maximum	Insulation Min. R-Value	Assembly Maximum	Insulation Min. R-Value
Insulation Entirely Above Deck		0.032	R-30.0 c.i.	0.063	R-15 c.i.
Metal Building (Roof)		0.037	R-19 + R-11 Ls or R-25 + R-8 Ls	0.082	R-19
Attic and Other		0.021	R-49	0.034	R-30

		IL TRM Zones 4 & 5 [ASHRAE/IECC Climate Zone 4 (A, B, C)]			
		Nonresidential		Semiheated	
		Assembly Maximum	Insulation Min. R-Value	Assembly Maximum	Insulation Min. R-Value
Insulation Entirely Above Deck		0.032	R-30.0 c.i.	0.093	R-10 c.i.
Metal Building (Roof)		0.037	R-19 + R-11 Ls or R-25 + R-8 Ls	0.082	R-19
Attic and Other		0.021	R-49	0.034	R-30

Table Notes
c.i. = continuous insulation

Ls = linear system, a continuous vapor barrier liner installed below the purlins and uninterrupted by framing members

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure expected useful life (EUL) is assumed to be 20 years per DEER 2008. This is consistent with SDG&E's 9th Year Measure Retrofit Study (1996 & 1997 Residential Weatherization Programs), CPUC's Energy Efficiency Policy Manual v.2, and GDS's Measure Life Report Residential and Commercial/Industrial Lighting and HVAC Measures (June 2007).

DEEMED MEASURE COST

Per the W017 Itron California Measure Cost Study⁸³⁶, the material cost for R-30 insulation is \$0.59 per square foot. The installation cost is \$0.81 per square foot. The total measure cost, therefore, is \$1.40 per square foot of insulation installed. However, the actual cost should be used when available.

LOADSHAPE

Loadshape C03: Commercial Cooling

COINCIDENCE FACTOR

CF_{SSP} = Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour)
= 91.3%⁸³⁷

CF_{PJM} = PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period)
= 47.8%⁸³⁸

Algorithm

CALCULATION OF ENERGY SAVINGS**ELECTRIC ENERGY SAVINGS**

Electric energy savings is calculated as the sum of energy saved when cooling the building and energy saved when heating the building

$$\Delta kWh = \Delta kWh_{cooling} + \Delta kWh_{heating}$$

If central cooling, the electric energy saved in annual cooling due to the added insulation is

$$\Delta kWh_{cooling} = ((1/R_{existing}) - (1/R_{new})) * Area * EFLH_{cooling} * \Delta T_{AVG,cooling} / 1,000 / \eta_{cooling}$$

Where:

$R_{existing}$ = Roof heat loss coefficient with existing insulation [(hr-°F-ft²)/Btu]

R_{new} = Roof heat loss coefficient with new insulation [(hr-°F-ft²)/Btu]

Area = Area of the roof surface in square feet. Assume 1000 sq ft for planning.

$EFLH_{cooling}$ = Equivalent Full Load Hours for Cooling [hr] are provided in Section 4.4, HVAC end use

$\Delta T_{AVG,cooling}$ = Average temperature difference [°F] during cooling season between outdoor air temperature and assumed 75°F indoor air temperature

Climate Zone (City based upon)	$OA_{AVG,cooling}$ [°F] ⁸³⁹	$\Delta T_{AVG,cooling}$ [°F]
1 (Rockford)	81	6

⁸³⁶ Measure costs are from the W017 Itron California Measure Cost Study, accessed via

<http://www.energydataweb.com/cpuc/search.aspx>. The data is provided in a file named "MCS Results Matrix – Volume I".

⁸³⁷ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

⁸³⁸ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

⁸³⁹ National Solar Radiation Data Base -- 1991- 2005 Update: Typical Meteorological Year 3

http://rredc.nrel.gov/solar/old_data/nsrdb/1991-2005/tmy3/by_state_and_city.html

Climate Zone (City based upon)	OA _{AVG,cooling} [°F] ⁸³⁹	ΔT _{AVG,cooling} [°F]
2 (Chicago)	81	6
3 (Springfield)	81	6
4 (Belleville)	82	7
5 (Marion)	82	7

1,000 = Conversion from Btu to kBtu

$\eta_{cooling}$ = Seasonal energy efficiency ratio (SEER) of cooling system (kBtu/kWh). Use actual if possible, if unknown and for planning purposes assume the following:

Year Equipment was Installed	SEER estimate
Before 2006	10
After 2006	13

If the building is heated with electric heat (resistance or heat pump), the electric energy saved in annual heating due to the added insulation is

$$\Delta kWh_{heating} = [(1/R_{existing}) - (1/R_{new})] * Area * EFLH_{heating} * \Delta T_{AVG,heating} / 3,412 / \eta_{heating}$$

Where:

$EFLH_{heating}$ = Equivalent Full Load Hours for Heating [hr] are provided in Section 4.4, HVAC end use

$\Delta T_{AVG,heating}$ = Average temperature difference [°F] during heating season between outdoor air temperature and assumed 55°F heating base temperature

Climate Zone (City based upon)	OA _{AVG,heating} [°F] ⁸⁴⁰	ΔT _{AVG,heating} [°F]
1 (Rockford)	32	23
2 (Chicago)	34	21
3 (Springfield)	35	20
4 (Belleville)	36	19
5 (Marion)	39	16

3,142 = Conversion from Btu to kWh.

$\eta_{heating}$ = Efficiency of heating system. Use actual efficiency. If not available refer to default table below.

System Type	Age of Equipment	HSPF Estimate	η_{Heat} (Effective COP Estimate) (HSPF/3.413)*0.85
Heat Pump	Before 2006	6.8	1.7
	After 2006	7.7	1.92
Resistance	N/A	N/A	1

If the building is heated with a gas furnace, there will be some electric savings in heating the building attributed to extra insulation since the furnace fans will run less.

⁸⁴⁰ National Solar Radiation Data Base -- 1991- 2005 Update: Typical Meteorological Year 3
http://rredc.nrel.gov/solar/old_data/nsrdb/1991-2005/tmy3/by_state_and_city.html

$$\Delta \text{kWh}_{\text{heating}} = \Delta \text{Therms} * \text{Fe} * 29.3$$

Where:

ΔTherms	= Gas savings calculated with equation below.
Fe	= Percentage of heating energy consumed by fans, assume 3.14%
29.3	= Conversion from therms to kWh

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = (\Delta \text{kWh}_{\text{cooling}} / \text{EFLH}_{\text{cooling}}) * \text{CF}$$

Where:

$\text{EFLH}_{\text{cooling}}$	= Equivalent full load hours of air conditioning are provided in Section 4.4, HVAC end use
CF_{SSP}	= Summer System Peak Coincidence Factor for Commercial cooling (during system peak hour) = 91.3% ⁸⁴¹
CF_{PJM}	= PJM Summer Peak Coincidence Factor for Commercial cooling (average during peak period) = 47.8% ⁸⁴²

NATURAL GAS SAVINGS

If building uses a gas furnace, the savings resulting from the insulation is calculated with the following formula.

$$\Delta \text{Therms} = ((1/\text{R}_{\text{existing}}) - (1/\text{R}_{\text{new}})) * \text{Area} * \text{EFLH}_{\text{heating}} * \Delta \text{T}_{\text{AVG,heating}} / 100,000 / \eta_{\text{heat}}$$

Where:

$\text{R}_{\text{existing}}$	= Roof heat loss coefficient with existing insulation [(hr-°F-ft²)/Btu]
R_{new}	= Roof heat loss coefficient with new insulation [(hr-°F-ft²)/Btu]
Area	= Area of the roof surface in square feet. Assume 1000 sq ft for planning.
$\text{EFLH}_{\text{heating}}$	= Equivalent Full Load Hours for Heating are provided in Section 4.4, HVAC end use
$\Delta \text{T}_{\text{AVG,heating}}$	= Average temperature difference [°F] during heating season (see above)
100,000	= Conversion from BTUs to Therms
η_{heat}	= Efficiency of existing furnace. Assume 0.78 for planning purposes.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

⁸⁴¹ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The AC load during the utility's peak hour is divided by the maximum AC load during the year.

⁸⁴² Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year

MEASURE CODE: CI-MSC-RINS-V02-160601

REVIEW DEADLINE: 1/1/2021

4.8.3 Computer Power Management Software

DESCRIPTION

Computer power management software is installed on a network of computers. This is software which monitors and records computer and monitor usage, as well as allows centralized control of computer power management settings.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is defined by the requirements listed below:

- Allow centralized control and override of computer power management settings of workstations which include both a computer monitor and CPU (i.e. a desktop or laptop computer on a distributed network)
- Be able to control on/off/sleep states on both the CPU and monitor according to the Network Administrator-defined schedules and apply power management policies to network groups
- Have capability to allow networked workstations to be remotely wakened from power-saving mode (e.g. for system maintenance or power/setting adjustments)
- Have capability to detect and monitor power management performance and generate energy savings reports
- Have capability to produce system reports to confirm the inventory and performance of equipment on which the software is installed.

This measure was developed to be applicable to the following program types: Retrofit. If applied to other program types, the measure savings should be verified.

DEFINITION OF BASELINE EQUIPMENT

Baseline is defined as a computer network without software enforcing the power management capabilities in existing computers and monitors.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is five years.⁸⁴³

DEEMED MEASURE COST

The deemed measure cost is \$29 per networked computer, including labor.⁸⁴⁴

LOADSHAPE

Loadshape C21: Commercial Office Equipment.

COINCIDENCE FACTOR

N/A

Algorithm

⁸⁴³ The following reference uses 10 years, however, given the rapid changes in the technology industry, there is quite a lot of uncertainty about the measure life and a more conservative value was used (i.e. half the published measure life): Table VI.1: Dimetrosky, S., Luedtke, J. S., & Seiden, K. (2005). Surveyor Network Energy Manager: Market Progress Evaluation Report, No. 2 (Northwest Energy Efficiency Alliance report #E05-136). Portland, OR: Quantec LLC;).

⁸⁴⁴ Work Paper WPSCNROE0003 Revision 1, Power Management Software for Networked Computers. Southern California Edison

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = W_{\text{savings}} * W$$

Where:

W_{savings}	= annual energy savings per workstation = 200 kWh ⁸⁴⁵ for desktops, 50 kWh for laptops ⁸⁴⁶ = If unknown assume 161 kWh (based on 74% desktop and 26% laptop ⁸⁴⁷)
W	= number of desktop or laptop workstations controlled by the power management software

SUMMER COINCIDENT PEAK DEMAND SAVINGS

NA

NATURAL GAS SAVING

NA

WATER IMPACT DESCRIPTIONS AND CALCULATION

NA

DEEMED O&M COST ADJUSTMENT CALCULATION

Assumed to be \$2/unit⁸⁴⁸

MEASURE CODE: CI-MSC-CPMS-V01-150601

REVIEW DEADLINE: 1/1/2020

⁸⁴⁵ Based on average energy savings/computer from the following sources:

South California Edison, Work Paper WPSCNROE0003 (200k Wh)

Surveyor Network Energy Manager Evaluation Report , NEEA (68, 100, and 128kWh)

Regional Technical Forum <http://rtf.nwcouncil.org/measures/measure.asp?id=95> (200 kWh)

EnergySTAR Computer Power Management Savings Calculator (~190 kWh for a mix of laptop/desktop and assuming 30% are already turned off at night)

http://www.energystar.gov/ia/products/power_mgt/LowCarbonITSavingsCalc.xlsx?78c1-120e&78c1-120e

Power Management for Networked Computers: A Review of Utility Incentive Programs J. Michael Walker, Beacon Consultants Network Inc., 2009 ACEEE Summer Study on Energy Efficiency in Industry (330 kWh)

⁸⁴⁶ Power Management for Networked Computers: A Review of Utility Incentive Programs J. Michael Walker, Beacon Consultants Network Inc., 2009 ACEEE Summer Study on Energy Efficiency in Industry

⁸⁴⁷ Based on PY6 ComEd Computer Software Program data showing a split of 74% desktop to 26% laptop.

⁸⁴⁸ Based on Dimetrosky, S., Luedtke, J. S., & Seiden, K. (2005). Surveyor Network Energy Manager: Market Progress Evaluation Report, No. 2 (Northwest Energy Efficiency Alliance report #E05-136). Portland, OR: Quantec LLC and review of CLEARResult document providing Qualifying Software Providers for ComEd program and their licensing fees; "Qualifying Vendor Software Comparison.pdf".

4.8.4 Modulating Commercial Gas Clothes Dryer

DESCRIPTION

This measure relates to the installation of a two-stage modulating gas valve retrofit kit on a standard commercial non-modulating gas dryer. Commercial gas clothes dryers found in coin-operated laundromats or on-premise laundromats (hospitals, hotels, health clubs, etc.) traditionally have a single firing rate which is sized properly for highest heat required in initial drying stages but is oversized for later drying stages requiring lesser heat. This causes the burner to cycle on/off frequently, resulting in less efficient drying and wasted gas. Replacing the single stage gas valve with a two-stage gas valve allows the firing rate to adjust to the changing heat demand, thereby reducing overall gas consumption.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

A 30 to 250 pound capacity commercial gas dryer retrofitted with a two-stage modulating gas valve kit.

DEFINITION OF BASELINE EQUIPMENT

A 30 to 250 pound capacity commercial gas dryer with no modulating capabilities.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The deemed measure life for the retrofit kit is 14 years, assumed to be equal to that of a commercial gas dryer⁸⁴⁹.

DEEMED MEASURE COST

The full retrofit cost is assumed to be \$700, including the material cost for the basic modulating gas valve retrofit kit (\$600) and the associated of labor for installation (\$100)⁸⁵⁰.

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

⁸⁴⁹ Zhang, Yanda, and Julianna Wei. *Commerical Clothes Dryers, CASE Initiative for PY2013: Title 20 Standards Development*. California Public Utilities Commission, 2013.

⁸⁵⁰ Engineering judgement, based on observed costs during Nicor Gas pilot study. "Nicor Gas Emerging Technology Program, 1036: Commercial Dryer Modulation Retrofit Public Project Report." 2014.

NATURAL GAS ENERGY SAVINGS

Note: Accurately estimating dryer energy consumption is complicated and challenging due to a variety of factors that influence cycle times and characteristics and ultimately drying energy requirements. Clothing loads can vary by weight, volume, fiber composition, physical structure, and initial water content, meaning that for any given cycle drying energy requirements can differ. Additionally, dryer settings selected by the user as well as interactions with the site's HVAC systems are known to influence dryer performance. As better information becomes available, this characterization can be modified to allow for a more site-specific estimation of savings.

$$\Delta \text{Therms} = N_{\text{Cycles}} * SF$$

Where:

N_{Cycles} = Number of dryer cycles per year. Refer to the table below if this value is not directly available.

Application	Cycles per Year
Coin- Operated Laundromats ⁸⁵¹	1,483
Multi-family Dryers ⁸⁵²	1,074
On-Premise Laundromats ⁸⁵³	3,607

SF = Savings factor
= 0.18 therms/cycle⁸⁵⁴

If using default cycles the savings are as follows:

Application	ΔTherms
Coin- Operated Laundromats ⁸⁵⁵	267
Multi-family Dryers ⁸⁵⁶	193
On-Premise Laundromats ⁸⁵⁷	649

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-MSC-MODD-V01-160601

REVIEW DEADLINE: 1/1/2023

⁸⁵¹ From DOE's Federal Register Notices - found here: <http://energy.gov/eere/buildings/recent-federal-register-notice>

⁸⁵² Ibid.

⁸⁵³ Average value for dryer cycles in healthcare facility, hotels, drycleaners and laundromats from tests conducted in Nicor Gas Emerging Technology Program's Commercial Dryer Modulation Retrofit Public Project Report.

⁸⁵⁴ Based on Illinois weather data, and average dryer performance for laundromat (30 to 45lb) and hotel (75 to 170 lb) dryers. See GTI Analysis.xlsx for complete derivation.

⁸⁵⁵ From DOE's Federal Register Notices - found here: <http://energy.gov/eere/buildings/recent-federal-register-notice>

⁸⁵⁶ Ibid.

⁸⁵⁷ Average value for dryer cycles in healthcare facility, hotels, drycleaners and laundromats from tests conducted in Nicor Gas Emerging Technology Program's Commercial Dryer Modulation Retrofit Public Project Report.

4.8.5 High Speed Clothes Washer

DESCRIPTION

This measure applies to the installation of clothes washers with extraction speeds of 200 g or greater, which is significantly higher than traditional hard-mount washers. Standard washer extractors in laundromats operate at speeds of 70-80⁸⁵⁸ g. The high-speed extraction process in the wash cycle removes more water from each compared to standard washers, reducing operating time and gas consumption of clothes dryers. Heat exposure and mechanical action are also reduced, resulting in less linen wear.

This measure was developed to be applicable to the following program types: TOS, NC, EREP. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is assumed to be a clothes washer with an extraction speed of 200 g or greater, installed in a commercial laundromat.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be a clothes washer with an extraction speed of 100 g or less, installed in a commercial laundromat.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure lifetime is assumed to be the typical lifetime of a commercial clothes washer: 7 years⁸⁵⁹.

For early replacement measures it is assumed the existing unit would last another 2.3 years⁸⁶⁰

DEEMED MEASURE COST⁸⁶¹

The incremental cost for time of sale is \$9.70/lb capacity.

The full cost of the high speed washer for early replacement applications is \$164.89/lb capacity. The deferred replacement cost of the baseline unit is \$155.19/lb capacity. This future cost should be discounted to present value using the real discount rate:

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

⁸⁵⁸ "The Real Size of a Front Load Washer" http://laundromat123.com/Laundromat_Washer_Comparison.html

⁸⁵⁹ "Assessment of Water Savings for Commercial Washers: Report on the Monitoring and Assessment of Water Savings from the Coin-Operated Multi-Load Clothes Washers Voucher Initiative Program." San Diego County Water Authority October 2016. http://www.coinwash.com/temp/coinwash_Assessment_Water_%20Savings_Commercial_Clothes_Washers.pdf

⁸⁶⁰ Third of expected measure life.

⁸⁶¹ Measure costs are based on data from a quote provided by a commercial washer distributor to Franklin Energy Services.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS SAVINGS

$$\Delta \text{Therms} = (\text{Ncycles} * \text{Days} * \text{Capacity} * \text{RMC} * h_e / \eta_{\text{dryer}} / 100,000) * \text{DryerUse} * \text{LF}$$

Where:

Ncycles = Average number of washer cycles per day

= Use values from table below, depending on application

Application	Ncycles
Coin-operated Laundromats	4.3 ⁸⁶²
Multi-family	3.4 ⁸⁶³
Hotel/Motel/Hospital	10.4 ⁸⁶⁴

Days = Days per year of commercial laundromat operation

= Actual, or if unknown, assume 360 days⁸⁶⁵Capacity = Clothes washer rated capacity (lb/cycle)⁸⁶⁶

= Actual

RMC = Retained Moisture Content (%)⁸⁶⁷ reduction from replacing a low extraction speed washer= Assume 25%⁸⁶⁸

⁸⁶² "2014-2015 State of the Self-Service Laundry Industry Report." Carlo Calma, April 13, 2015. <https://americancoinop.com/articles/2014-2015-state-self-service-laundry-industry-report-conclusion>

⁸⁶³ "Assessment of Water Savings for Commercial Washers: Report on the Monitoring and Assessment of Water Savings from the Coin-Operated Multi-Load Clothes Washers Voucher Initiative Program." San Diego County Water Authority October 2016. http://www.coinwash.com/temp/coinwash_Assessment_Water_%20Savings_Commercial_Clothes_Washers.pdf

⁸⁶⁴ "Laundry Planning Guide." EDRO, January 2015. <http://www.edrocorp.com/downloads/Laundry-Planning-Guide%202015.pdf>

⁸⁶⁵ Based on professional judgement, assuming closed on holidays.

⁸⁶⁶ Clothes washer capacity is based on weight of dry clothing.

⁸⁶⁷ The EDRO "Laundry Planning Guide" describes moisture retention as "the ratio of retained moisture weight to clean dry textile weight." The pounds of water retained by clothing at the end of a wash cycle is calculated by multiplying Capacity (lbs of dry clothing per cycle) by RMC.

⁸⁶⁸ Using chart provided (Figure 1) and assuming a 100% nominal cotton load, the retained moisture drops from approximately 90% to 65% when a 100 g washer is replaced with a 200 g washer. Chart from "Laundry Planning Guide." EDRO, January 2015.

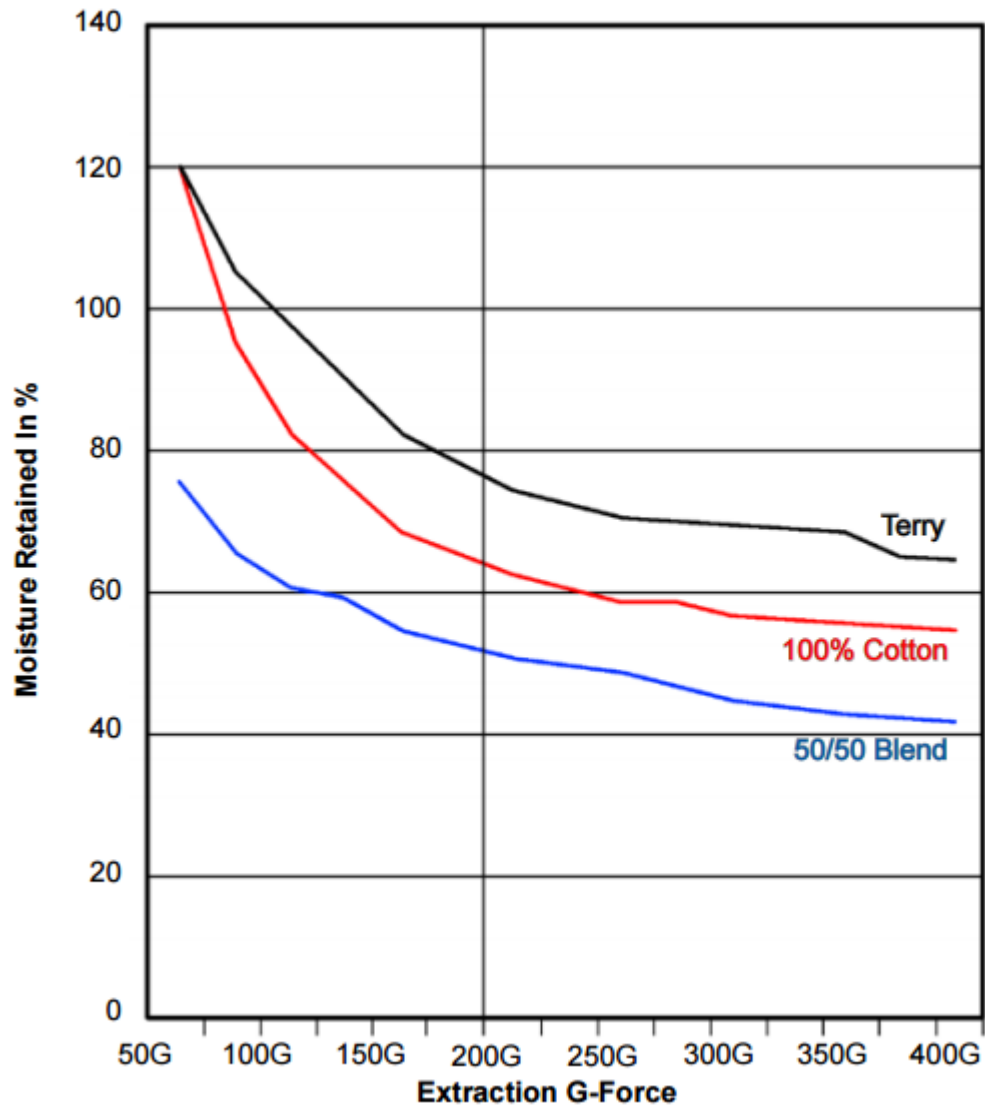


Figure 1

h_e = Heat required by a dryer to evaporate 1 lb of water

= Assume 1,200 Btu/lb⁸⁶⁹

η_{dryer} = Efficiency of the clothes dryer

= Actual, or if unknown, assume 60%⁸⁷⁰

100,000 = Converts Btus to therms

DryerUse = % of washer loads dried in the field

= Assume 91%⁸⁷¹

⁸⁶⁹ "Laundry Planning Guide." EDRO, January 2015.

⁸⁷⁰ "Are We Missing Energy Savings in Clothes Dryers?" Paul Bendt (Ecos), 2010

<http://aceee.org/files/proceedings/2010/data/papers/2206.pdf>

⁸⁷¹ "Dryer Field Study." Northwest Energy Efficiency Alliance, November 20, 2014. <http://www.ecotope.com/wp/wp-content/uploads/2014/04/nea-clothes-dryer-field-study.pdf>

LF = Load Factor (%) to account for the pounds per washer load, as a percentage of rated capacity
 = Assume 66%⁸⁷²

EXAMPLE

For example, a clothes washer with a 14 lb/cycle capacity and installed at a coin-operated laundromat, using default assumptions, would save:

$$\begin{aligned}\Delta\text{Therms} &= (\text{Ncycles} * \text{Days} * \text{Capacity} * \text{RMC} * h_e / \eta_{\text{dryer}} / 100,000) * \text{DryerUse} * \text{LF} \\ &= (4.3 * 360 * 14 * 0.25 * 1,200 / 0.60 / 100,000) * 0.91 * 0.66 \\ &= 65 \text{ therms}\end{aligned}$$

WATER AND OTHER NON-ENERGY IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-MSC-HSCW-V01-180101

REVIEW DEADLINE: 1/1/2021

⁸⁷²"Assessment of Water Savings for Commercial Washers: Report on the Monitoring and Assessment of Water Savings from the Coin-Operated Multi-Load Clothes Washers Voucher Initiative Program." San Diego County Water Authority October 2016.

4.8.6 ENERGY STAR Computers

DESCRIPTION

This measure estimates savings for a desktop computer with ENERGY STAR (ES) Version 6.0 rating, ES 6.0 +20%, ES 6.0 with 80 PLUS Gold PSUs, and ES 6.0 with 80 PLUS Platinum PSUs.

This measure was developed to be applicable to the following program types: TOS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient product is a desktop with a rating of ENERGY STAR Version 6.0 rating, ES 6.0 +20%, ES 6.0 with 80 PLUS Gold PSUs, or ES 6.0 with 80 PLUS Platinum PSUs.

DEFINITION OF BASELINE EQUIPMENT

Non ENERGY STAR qualified equipment with standard efficiency power supply

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The life of this measure is 4 years.⁸⁷³

DEEMED MEASURE COST⁸⁷⁴

The incremental cost for an 80 Plus Desktop PSU is \$5.

The incremental cost for an ENERGY STAR desktop PSU is \$20.

LOADSHAPE

C21 Commercial Office Equipment

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS⁸⁷⁵

$$\Delta \text{kWh} = 8760/1000 * (((\text{Watts}_{\text{Base,Off}} * \% \text{Time}_{\text{Off}}) + (\text{Watts}_{\text{Base,Sleep}} * \% \text{Time}_{\text{Sleep}}) + (\text{Watts}_{\text{Base,Long}} * \% \text{Time}_{\text{Long}}) + (\text{Watts}_{\text{Base,Short}} * \% \text{Time}_{\text{Short}})) - ((\text{Watts}_{\text{Eff,Off}} * \% \text{Time}_{\text{Off}}) + (\text{Watts}_{\text{Eff,Sleep}} * \% \text{Time}_{\text{Sleep}}) + (\text{Watts}_{\text{Eff,Long}} * \% \text{Time}_{\text{Long}}) + (\text{Watts}_{\text{Eff,Short}} * \% \text{Time}_{\text{Short}})))$$

Where (see assumptions in table below):

8760/1000 = Converts W to kWh

Watts_{Base,Off} = baseline equipment power in off mode

⁸⁷³ Codes and Standards Enhancement (CASE) Initiative For PY 2013: Title 20 Standards Development, August 6, 2013 Page 6.

http://www.energy.ca.gov/appliances/2014-AAER-01/prerulemaking/documents/comments_12-AAER-2A/California_IOUs_Standards_Proposal_Addendum_Computers_2014-10-27_TN-73899.pdf

⁸⁷⁴ Research Into Action, 80 PLUS Market Progress Evaluation Report #5, November 26, 2013. Page 24.

⁸⁷⁵ Algorithm comes from ENERGY STAR Version 6.0 Guide

%Time _{Off}	= typical percent of time a desktop, integrated desktop or notebook is in off mode during the year
Watts _{Base,Sleep}	= baseline equipment power in sleep mode
%Time _{Sleep}	= typical percent time in sleep mode
Watts _{Base,Long}	= baseline equipment power in long idle mode
%Time _{Long}	= typical percent time in long idle mode
Watts _{Base,Short}	= baseline equipment power in short idle mode
%Time _{Short}	= typical percent time in short idle mode
Watts _{Eff,Off}	= efficient equipment power in off mode
Watts _{Eff,Sleep}	= efficient equipment power in sleep mode
Watts _{Eff,Long}	= efficient equipment power in long idle mode
Watts _{Eff,Short}	= efficient equipment power in short idle mode

Measure Annual Mode Time (%)	Off	Sleep	Long Idle	Short Idle
Duty cycle - Commercial ⁸⁷⁶	45%	5%	15%	35%

Measure Watt Draw in Mode (Watts)	Off	Sleep	Long Idle	Short Idle
Baseline ⁸⁷⁷	0.88	2.1	26.5	27.9
ES 6.0 Desktops ⁸⁷⁸	0.55	1.23	24.66	26.04
ES 6.0 +20% Desktops ⁸⁷⁹	0.52	1.63	21.33	22.58
ES 6.0 Desktops w/ 80 PLUS Gold PSUs ⁸⁸⁰	0.50	1.50	23.08	24.38
ES 6.0 Desktops w/ 80 PLUS Platinum PSUs ⁸⁸¹	0.50	1.50	22.19	23.44

Calculated energy consumption in each mode, and savings provided below:

Measure TEC by Mode (kWh) Commercial	Off	Sleep	Long Idle	Short Idle	TEC (kWh/yr)	Savings (kWh/yr)
Baseline	3.5	0.9	34.8	85.5	124.8	N/A
ES 6.0 Desktops	2.2	0.5	32.4	79.9	115.0	9.8
ES 6.0 +20% Desktops	2.0	0.7	28.0	69.2	100.0	24.7
ES 6.0 Desktops w/ 80 PLUS Gold PSUs	2.0	0.7	30.3	74.7	107.7	17.1
ES 6.0 Desktops w/ 80 PLUS Platinum PSUs	2.0	0.7	29.2	71.9	103.7	21.1

Savings calculations can be referenced in "ENERGY STAR Desktop Analysis.xlsx"

⁸⁷⁶ ECMA 283, Appendix B, Majority Profile Study; ENERGY STAR v6.0 duty cycle. See

<https://www.energystar.gov/sites/default/files/specs/Version%206%201%20Computers%20Final%20Program%20Requirements.pdf>.

⁸⁷⁷ Computer CASE Report, CA IOUs. http://www.energy.ca.gov/appliances/2013rulemaking/documents/proposals/12-AAER-2A_Consumer_Electronics/California_IOUs_Standards_Proposal_Computers_UPDATED_2013-08-06_TN-71813.pdf

⁸⁷⁸ Analysis of current DT I2 category desktops in ES v6.0 QPL, available at <http://www.energystar.gov/productfinder/product/certified-computers/results>.

⁸⁷⁹ Analysis of current DT I2 category desktops in ES v6.0 QPL, passing with > 20% margin.

⁸⁸⁰ 80 PLUS program savings calculator, additional 6.4% savings over ES v6.0 Bronze PSU levels. Based on program measurements, available at <http://www.80plus.org>.

⁸⁸¹ 80 PLUS program savings calculator, additional 10% savings over ES v6.0 Bronze PSU levels.

SUMMER COINCIDENT PEAK DEMAND SAVINGS⁸⁸²

$$\Delta kW = (Watts_{Base} - Watts_{Eff}) / 1000 * CF$$

Where:

Watts_{Base} = Assumed average baseline wattage during peak period (see table below)

Watts_{Eff} = Assumed average efficient wattage during peak period (see table below)

CF = Summer Peak Coincidence Factor
= 1.0

Calculated average demand during peak period, and savings provided below:

Measure TEC by Mode (kWh) Commercial	TEC (watts)	Demand Savings
Baseline	25.2	N/A
ES 6.0 Desktops	23.4	0.0018
ES 6.0 +20% Desktops	20.3	0.0048
ES 6.0 Desktops w/ 80 PLUS Gold PSUs	21.9	0.0032
ES 6.0 Desktops w/ 80 PLUS Platinum PSUs	21.1	0.0041

Savings calculations can be referenced in "ENERGY STAR Desktop Analysis.xlsx"

NATURAL GAS SAVINGS

N/A

WATER AND OTHER NON-ENERGY IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-MSC-COMP-V01-180101

REVIEW DEADLINE: 1/1/2020

⁸⁸² It assumed that computers will not be off during peak period, and that the weighting of sleep, long idle and short idle during peak hours is consistent with the whole year. Wattage assumptions are weighted accordingly and coincidence factor is thus assumed to be 1.0 – see "ENERGY STAR Desktop Analysis.xlsx" for calculation.

4.8.7 Advanced Power Strip – Tier 1 Commercial

DESCRIPTION

This measure relates to Advanced Power Strips – Tier 1 which are multi-plug power strips with the ability to automatically disconnect specific connected loads depending upon the power draw of a control load, also plugged into the strip. Power is disconnected from the switched (controlled) outlets when the control load power draw is reduced below a certain adjustable threshold, thus turning off the appliances plugged into the switched outlets. By disconnecting, the standby load of the controlled devices, the overall load of a centralized group of equipment (e.g. a desk workstation) can be reduced. In a commercial office space, savings generally occur during off-hours, when connected equipment continues to consume electricity while in standby mode or when off. Uncontrolled outlets are also provided that are not affected by the control device and so are always providing power to any device plugged into it.

This measure was developed to be applicable to the following program types: DI.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient case is an advanced power strip with a load-sensing master plug and at least two controlled plugs.

DEFINITION OF BASELINE EQUIPMENT

The assumed baseline is a standard power strip with surge protection that does not control connected loads.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of the advanced power strip is 7 years.⁸⁸³

DEEMED MEASURE COST

For direct install the actual full install cost (including labor) and for kits the full equipment cost should be used.

LOADSHAPE

Loadshape C47 – Standby Losses – Commercial Office⁸⁸⁴

COINCIDENCE FACTOR

N/A due to no savings attributable to standby losses between 1 and 5 PM.

⁸⁸³ This is a consistent assumption with 5.2.2 Advanced Power Strip – Tier 2.

⁸⁸⁴ Loadshapes were calculated from empirical studies and compared to the existing loadshape in Volume 1, Table 3.5. The studies were:

Acker, Brad *et al*, "Office Space Plug Load Profiles and Energy Saving Interventions," 2012 ACEEE Summer Study on Energy Efficiency in Buildings.

Sheppy, M. *et al*, "Reducing Plug Loads in Office Spaces" Hawaii and Guam Energy Improvement Technology Demonstration Project, NREL/NAVFAC (January 2014).

Algorithm

CALCULATION OF ENERGY SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta kWh^{885} = ((kW_{wkd} * (hrs_{wkd} - hrs_{wkd-open}) + kW_{wkend} * (hrs_{wkend} - hrs_{wkend-open}) * weeks/year) * ISR$$

Where:

W_{wkd} = Standby power consumption of connected electronics on weekday off-hours. If unknown, assume 0.0315 kW.

kW_{wkend} = Standby power consumption of connected electronics on weekend off-hours. If unknown, assume 0.00617 kW.

hrs_{wkd} = total hours during the work week (Monday 7:30 AM to Friday 5:30 PM)
= 106

hrs_{wkend} = total hours during the weekend (Friday 5:30 PM to Monday 7:30 AM)
= 62

$hrs_{wkd-open}$ = hours the office is open during the work week. If unknown, assume 50 hours.

$hrs_{wkend-open}$ = hours the office is open during the weekend. If unknown, assume 0 hours.

$weeks/year$ = number of weeks per year
= 52.2

ISR = In Service Rate
= Assume 0.969 for commercial Direct Install application⁸⁸⁶

For example, an office open 9 hours per day (45 hours per week) on weekdays and 4 hours on Saturday:

$$\begin{aligned} \Delta kWh &= ((0.0315 * (106 - 45) + 0.00617 * (62 - 4) * 52.2) * 0.969 \\ &= 115 kWh \end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A due to no savings attributable to standby losses between 1 and 5 PM.

NATURAL GAS SAVINGS

N/A

WATER AND OTHER NON-ENERGY IMPACT DESCRIPTIONS AND CALCULATION

N/A

⁸⁸⁵ Savings algorithm reconstructed from weekday and weekend savings information in Sheppy *et. al*, and verified against savings in Acker *et. al* and savings in: BPA, "Smart Power Strip Energy Savings Evaluation: Ross Complex," (2011). Office stations are assumed to have zero or minimal standby losses during normal operating hours. Method shown in "Commercial Tier 1 APS Calculations – IL TRM.xlsx".

⁸⁸⁶ Based upon review of the PY2 and PY3 ComEd Direct Install Residential program surveys. This value could be modified based upon commercial application evaluation.

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-MSC-APSC-V01-180101

REVIEW DEADLINE: 1/1/2020

4.8.8 High Efficiency Transformer

DESCRIPTION

Distribution transformers are used in commercial and industrial applications to step down power from distribution voltage to be used in HVAC or process loads (220V or 480V) or to serve plug loads (120V).

Distribution transformers that are more efficient than the required minimum federal standard efficiency qualify for this measure. If there is no specific standard efficiency requirement, the transformer does not qualify (because we cannot define a reasonable baseline). For example, although the federal standards increased the minimum required efficiency in 2016, most transformers with a NEMA premium or CEE Tier 2 rating will still achieve energy conservation. Standards are defined for low-voltage dry-type distribution transformers (up to 333kVA single-phase and 1000kVA 3-phase), liquid-immersed distribution transformers (up to 833kVA single-phase and 2500kVA 3-phase), and medium-voltage dry-type distribution transformers (up to 833kVA single-phase and 2500kVA 3-phase).

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

Any transformer that is more efficient than the federal minimum standard. This includes CEE Tier II (single or three phase) and most NEMA premium efficiency rated products.

DEFINITION OF BASELINE EQUIPMENT

A transformer that meets the minimum federal efficiency requirement should be used as the baseline to calculate savings. Standards are developed by the Department of Energy and published in the Federal Register 10CFR 431⁸⁸⁷.

(a) Low-Voltage Dry-Type Distribution Transformers.

(2) The efficiency of a low-voltage dry-type distribution transformer manufactured on or after January 1, 2016, shall be no less than that required for their kVA rating in the table below. Low-voltage dry-type distribution transformers with kVA ratings not appearing in the table shall have their minimum efficiency level determined by linear interpolation of the kVA and efficiency values immediately above and below that kVA rating.

Single-phase		Three-phase	
kVA	Efficiency (%)	kVA	Efficiency (%)
15	97.70	15	97.89
25	98.00	30	98.23
37.5	98.20	45	98.40
50	98.30	75	98.60
75	98.50	112.5	98.74
100	98.60	150	98.83
167	98.70	225	98.94
250	98.80	300	99.02
333	98.90	500	99.14
		750	99.23
		1000	99.28

(b) Liquid-Immersed Distribution Transformers.

⁸⁸⁷ US Department of Energy, "Energy Conservation Program: Energy Conservation Standards for Distribution Transformers; Final Rule", 10 CFR Part 431, Published April 18, 2013, Compliance effective as of January 1, 2016.

(2) The efficiency of a liquid-immersed distribution transformer manufactured on or after January 1, 2016, shall be no less than that required for their kVA rating in the table below. Liquid-immersed distribution transformers with kVA ratings not appearing in the table shall have their minimum efficiency level determined by linear interpolation of the kVA and efficiency values immediately above and below that kVA rating.

Single-phase		Three-phase	
kVA	Efficiency (%)	kVA	Efficiency (%)
10	98.70	15	98.65
15	98.82	30	98.83
25	98.95	45	98.92
37.5	99.05	75	99.03
50	99.11	112.5	99.11
75	99.19	150	99.16
100	99.25	225	99.23
167	99.33	300	99.27
250	99.39	500	99.35
333	99.43	750	99.40
500	99.49	1000	99.43
667	99.52	1500	99.48
833	99.55	2000	99.51
		2500	99.53

(c) Medium-Voltage Dry-Type Distribution Transformers.

(2) The efficiency of a medium-voltage dry-type distribution transformer manufactured on or after January 1, 2016, shall be no less than that required for their kVA and BIL rating in the table below. Medium-voltage dry-type distribution transformers with kVA ratings not appearing in the table shall have their minimum efficiency level determined by linear interpolation of the kVA and efficiency values immediately above and below that kVA rating.

Single-phase				Three-phase			
kVA	BIL*			kVA	BIL		
	20-45 kV	46-95 kV	≥96 kV		20-45 kV	46-95 kV	≥96 kV
	Efficiency (%)	Efficiency (%)	Efficiency (%)		Efficiency (%)	Efficiency (%)	Efficiency (%)
15	98.10	97.86		15	97.50	97.18	
25	98.33	98.12		30	97.90	97.63	
37.5	98.49	98.30		45	98.10	97.86	
50	98.60	98.42		75	98.33	98.13	
75	98.73	98.57	98.53	112.5	98.52	98.36	
100	98.82	98.67	98.63	150	98.65	98.51	
167	98.96	98.83	98.80	225	98.82	98.69	98.57
250	99.07	98.95	98.91	300	98.93	98.81	98.69
333	99.14	99.03	98.99	500	99.09	98.99	98.89
500	99.22	99.12	99.09	750	99.21	99.12	99.02

Single-phase				Three-phase			
kVA	BIL*			kVA	BIL		
	20-45 kV	46-95 kV	≥96 kV		20-45 kV	46-95 kV	≥96 kV
	Efficiency (%)	Efficiency (%)	Efficiency (%)		Efficiency (%)	Efficiency (%)	Efficiency (%)
667	99.27	99.18	99.15	1000	99.28	99.20	99.11
833	99.31	99.23	99.20	1500	99.37	99.30	99.21
				2000	99.43	99.36	99.28
				2500	99.47	99.41	99.33

DEEMED LIFETIME OF EFFICIENT EQUIPMENT30 years⁸⁸⁸**DEEMED MEASURE COST**

Actual incremental costs should be used.

LOADSHAPE

Use custom loadshape based on application; default loadshape is Loadshape C53 – Flat.

COINCIDENCE FACTOR

Coincidence Factor for distribution transformers is 1.0 by definition. By including the load factor in the demand savings calculation, the load profile is accounted for.

Algorithm

CALCULATION OF ENERGY SAVINGS

Savings are determined by metering equipment

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = \text{Losses}_{\text{base}} - \text{Losses}_{\text{EE}}$$

Where:

$$\text{Losses}_{\text{base}} = \text{PowerRating} * \text{LF} * \text{PF} * \left(\frac{1}{\text{EFF}_{\text{base}}} - 1 \right) * 8766$$

$$\text{Losses}_{\text{EE}} = \text{PowerRating} * \text{LF} * \text{PF} * \left(\frac{1}{\text{EFF}_{\text{EE}}} - 1 \right) * 8766$$

PowerRating = kVA rating of the transformer (in units of kVA)

EFF_{base} = baseline total efficiency rating of federal minimum standard transformer (refer to baseline tables above based on kVA, voltage, and type of transformer)

⁸⁸⁸ US DOE lists lifetime at 32 years. For consistency with efficiency measure evaluated lifetimes, 30 years is the recommended maximum deemed lifetime. US Department of Energy, "Energy Conservation Program: Energy Conservation Standards for Distribution Transformers; Final Rule", 10 CFR Part 431, Published April 18, 2013, Effective as of January 1, 2016.

EFF _{EE}	= actual total efficiency rating of the transformer as calculated by the appropriate DOE test method ⁸⁸⁹
LF	= Load Factor for the transformer. Ratio of average transformer load to peak load rating over a period of one year. Use actual load factor for the network segment served based on historical data. If unknown, use 22% for commercial load and 45% for industrial load. ⁸⁹⁰
PF	= Power Factor for the load being served by the transformer. Ratio of real power to apparent power supplied to the transformer. Use actual power factor for the network segment served. If unknown, use 1.0 (unity) by default. ⁸⁹¹

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \text{PowerRating} * LF * PF * \left(\frac{1}{Eff_{base}} - \frac{1}{Eff_{EE}} \right)$$

Variables as provided above.

NATURAL GAS SAVINGS

N/A

WATER AND OTHER NON-ENERGY IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: CI-MSC-TRNS-V01-180101

REVIEW DEADLINE: 1/1/2021

⁸⁸⁹ Energy Conservation Program: Test Procedures for Distribution Transformers; Final Rule. Effective May 30, 2006.
<https://www.regulations.gov/document?D=EERE-2006-TP-0090-0001>

⁸⁹⁰ Guidelines on The Calculation and Use of Loss Factors, Electric Authority, Te Mana Hiko, February 14, 2013

⁸⁹¹ Unity power factor for used as default value, as used in the test procedures provided by US DOE. Energy Conservation Program: Test Procedures for Distribution Transformers; Final Rule. Effective May 30, 2006.
<https://www.regulations.gov/document?D=EERE-2006-TP-0090-0001>

4.8.9 High Frequency Battery Chargers

DESCRIPTION

This measure applies to industrial high frequency battery chargers, used for industrial equipment such as fork lifts, replacing existing SCR (silicon controlled rectifier) or ferroresonant charging technology. High frequency battery chargers have a greater system efficiency.

This measure was developed to be applicable to the following program types: TOS, NC. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

High frequency battery charger systems with minimum Power Conversion Efficiency of 90% and a minimum 8-hour shift operation five days per week.

DEFINITION OF BASELINE EQUIPMENT

SCR or ferroresonant battery charger systems with minimum 8-hour shift operation five days per week.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

15 years⁸⁹²

DEEMED MEASURE COST

The deemed incremental measure cost is \$400⁸⁹³

LOADSHAPE

Loadshape C14 - Indust. 1-shift (8/5) (e.g., comp. air, lights)

Loadshape C15 - Indust. 2-shift (16/5) (e.g., comp. air, lights)

Loadshape C16 - Indust. 3-shift (24/5) (e.g., comp. air, lights)

Loadshape C17 - Indust. 4-shift (24/7) (e.g., comp. air, lights)

COINCIDENCE FACTOR

The coincidence factor is assumed to be 0.0 for 1 and 2-shift operation and 1.0 for 3 and 4-shift operation. ⁸⁹⁴

⁸⁹² Suzanne Foster Porter et al., "Analysis of Standards Options for Battery Charger Systems", (PG&E, 2010), 45

⁸⁹³ Suzanne Foster Porter et al., "Analysis of Standards Options for Battery Charger Systems", (PG&E, 2010), 42

⁸⁹⁴ Emerging Technologies Program Application Assessment Report #0808, Industrial Battery Charger Energy Savings Opportunities, Pacific Gas & Electric. May 29, 2009.

Algorithm

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = (\text{CAP} * \text{DOD}) * \text{CHG} * (\text{CR}_B / \text{PC}_B - \text{CR}_{EE} / \text{PC}_{EE})$$

Where:

- CAP** = Capacity of Battery
= Use actual battery capacity, otherwise use a default value of 35 kWh⁸⁹⁵
- DOD** = Depth of Discharge
= Use actual depth of discharge, otherwise use a default value of 80%.⁸⁹⁶
- CHG** = Number of Charges per year
= Use actual number of annual charges, if unknown use values below based on the type of operations⁸⁹⁷

Standard Operations	Number of Charges per year
1-shift (8 hrs/day – 5 days/week)	520
2-shift (16 hrs/day – 5 days/week)	1040
3-shift (24 hrs/day – 5 days/week)	1560
4-shift (24 hrs/day – 7 days/week)	2184

- CR_B** = Baseline Charge Return Factor
= 1.2485⁸⁹⁸
- PC_B** = Baseline Power Conversion Efficiency
= 0.84⁸⁹⁹
- CR_{EE}** = Efficient Charge Return Factor
= 1.107⁹⁰⁰
- PC_{EE}** = Efficient Power Conversion Efficiency
= 0.89⁹⁰¹

Default savings using defaults provided above are provided below:

Standard Operations	ΔkWh
1-shift (8 hrs/day – 5 days/week)	3,531
2-shift (16 hrs/day – 5 days/week)	7,061
3-shift (24 hrs/day – 5 days/week)	10,592

⁸⁹⁵ Jacob V. Renquist, Brian Dickman, and Thomas H. Bradley, "Economic Comparison of fuel cell powered forklifts to battery powered forklifts", International Journal of Hydrogen Energy Volume 37, Issue 17, (2012): 2

⁸⁹⁶ Ryan Matley, "Measuring Energy Efficiency Improvements in Industrial Battery Chargers", (ESL-IE-09-05-32, Energy Technology Conference, New Orleans, LA, May 12-15, 2009), 4

⁸⁹⁷ Number of charges is derived from the following reference and adjusted to the hours and days of the different types of shift operations. These values are based on an estimated 2-charge per 8-hour workday. See reference file Ryan Matley, "Measuring Energy Efficiency Improvements in Industrial Battery Chargers", (ESL-IE-09-05-32, Energy Technology Conference, New Orleans, LA, May 12-15, 2009), 4

⁸⁹⁸ Ryan Matley, "Measuring Energy Efficiency Improvements in Industrial Battery Chargers", (ESL-IE-09-05-32, Energy Technology Conference, New Orleans, LA, May 12-15, 2009), 4 (average of SCR and Ferroresonant)

⁸⁹⁹ Ibid.

⁹⁰⁰ Ibid.

⁹⁰¹ Ibid.

Standard Operations	ΔkWh
4-shift (24 hrs/day – 7 days/week)	14,829

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = (PF_B/PC_B - PF_{EE}/PC_{EE}) * Volts_{DC} * Amps_{DC} / 1000 * CF$$

Where:

PF_B = Power factor of baseline charger
= 0.9095⁹⁰²

PF_{EE} = Power factor of high frequency charger
= 0.9370⁹⁰³

$Volts_{DC}$ = Actual DC rated voltage of charger (assumed baseline charger is replaced with same rated high frequency unit)
= Use actual battery DC voltage rating, otherwise use a default value of 48 volts.⁹⁰⁴

$Amps_{DC}$ = Actual DC rated amperage of charger (assumed baseline charger is replaced with same rated high frequency unit)
= Use actual battery DC ampere rating, otherwise use a default value of 81 amps.⁹⁰⁵

1,000 = watt to kilowatt conversion factor

CF = Summer Coincident Peak Factor for this measure
= 0.0 (for 1 and 2-shift operation)⁹⁰⁶
= 1.0 (for 3 and 4-shift operation)⁹⁰⁷

Other variables as provided above.

Default savings using defaults provided above are provided below:

Standard Operations	ΔkW
1-shift (8 hrs/day – 5 days/week)	0
2-shift (16 hrs/day – 5 days/week)	0
3-shift (24 hrs/day – 5 days/week)	0.1165
4-shift (24 hrs/day – 7 days/week)	0.1165

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

⁹⁰² Ibid.

⁹⁰³ Ibid.

⁹⁰⁴ Voltage rating based on the assumption of 35kWh battery with a normalized average amp-hour capacity of 760 Ah charged over a 7.5 hour charge cycle. Pacific Gas & Electric, "Emerging Technologies Program Application Assessment Report #0808", Industrial Battery Charger Energy Savings Opportunities. May 29, 2009. Page 8, Table 3.

⁹⁰⁵ Ampere rating based on the assumption of 35kWh battery with a normalized average amp-hour capacity of 760 Ah charged over a 7.5 hour charge cycle. Pacific Gas & Electric, "Emerging Technologies Program Application Assessment Report #0808", Industrial Battery Charger Energy Savings Opportunities. May 29, 2009. Page 8, Table 3.

⁹⁰⁶ Emerging Technologies Program Application Assessment Report #0808, Industrial Battery Charger Energy Savings Opportunities, Pacific Gas & Electric. May 29, 2009.

⁹⁰⁷ Ibid.

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

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Volume 3: Residential Measures

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5.1 Appliances End Use

5.1.1 ENERGY STAR Air Purifier/Cleaner

DESCRIPTION

An air purifier (cleaner) meeting the efficiency specifications of ENERGY STAR is purchased and installed in place of a model meeting the current federal standard.

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is defined as an air purifier meeting the efficiency specifications of ENERGY STAR as provided below.

- Must produce a minimum 50 Clean Air Delivery Rate (CADR) for Dust¹ to be considered under this specification.
- Minimum Performance Requirement: = 2.0 CADR/Watt (Dust)
- Standby Power Requirement: = 2.0 Watts Qualifying models that perform secondary consumer functions (e.g. clock, remote control) must meet the standby power requirement.
- UL Safety Requirement: Models that emit ozone as a byproduct of air cleaning must meet UL Standard 867 (ozone production must not exceed 50ppb)

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be a conventional unit².

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 9 years³.

DEEMED MEASURE COST

The incremental cost for this measure is \$70.⁴

LOADSHAPE

Loadshape C53 - Flat

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 100 % (the unit is assumed to be always on).

¹ Measured according to the latest ANSI/AHAM AC-1 (AC-1) Standard

² As defined as the average of non-ENERGY STAR products found in EPA research, 2011, ENERGY STAR Qualified Room Air Cleaner Calculator.

³ ENERGY STAR Qualified Room Air Cleaner Calculator.

⁴ Ibid

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = \text{kWh}_{\text{Base}} - \text{kWh}_{\text{ESTAR}}$$

Where:

 kWh_{BASE} = Baseline kWh consumption per year⁵

= see table below

 $\text{kWh}_{\text{ESTAR}}$ = ENERGY STAR kWh consumption per year⁶

= see table below

Clean Air Delivery Rate (CADR)	CADR used in calculation (midpoint)	Baseline Energy Consumption (kWh/year)	Unit	ENERGY STAR Unit Energy Consumption (kWh/year)	ΔkWh
CADR 51-100	75	441		148	293
CADR 101-150	125	733		245	488
CADR 151-200	175	1025		342	683
CADR 201-250	225	1317		440	877
CADR Over 250	300	1755		586	1169

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \Delta \text{kWh} / \text{Hours} * \text{CF}$$

Where:

 ΔkWh = Gross customer annual kWh savings for the measure

Hours = Average hours of use per year

= 5844 hours⁷

CF = Summer Peak Coincidence Factor for measure

= 66.7%⁸

Clean Air Delivery Rate	ΔkW
CADR 51-100	0.033
CADR 101-150	0.056
CADR 151-200	0.078
CADR 201-250	0.100
CADR Over 250	0.133

⁵ ENERGY STAR Qualified Room Air Cleaner Calculator.
⁶ Ibid.⁷ Consistent with ENERGY STAR Qualified Room Air Cleaner Calculator assumption of 16 hours per day (16 * 365.25 = 5844).⁸ Assumes that the purifier usage is evenly spread throughout the year, therefore coincident peak is calculated as 5844/8766 = 66.7%.

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

There are no operation and maintenance cost adjustments for this measure.⁹

MEASURE CODE: RS-APL-ESAP-V02-160601

REVIEW DEADLINE: 1/1/2023

⁹ Some types of room air cleaners require filter replacement or periodic cleaning, but this is likely to be true for both efficient and baseline units and so no difference in cost is assumed.

5.1.2 ENERGY STAR and ENERGY STAR Most Efficient Clothes Washers

DESCRIPTION

This measure relates to the installation of a clothes washer meeting the ENERGY STAR, or ENERGY STAR Most Efficient minimum qualifications. Note if the DHW and dryer fuels of the installations are unknown (for example through a retail program) savings should be based on a weighted blend using RECS data (the resultant values (kWh, therms and gallons of water) are provided). The algorithms can also be used to calculate site specific savings where DHW and dryer fuels are known.

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

Clothes washer must meet the ENERGY STAR or ENERGY STAR Most Efficient minimum qualifications, as required by the program.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is a standard sized clothes washer meeting the minimum federal baseline as of March 2015¹⁰.

Efficiency Level	Top loading >2.5 Cu ft	Front Loading >2.5 Cu ft
Federal Standard	1.29 IMEF, 8.4 IWF	1.84 IMEF, 4.7 IWF
ENERGY STAR	2.06 IMEF, 4.3 IWF	2.38 IMEF, 3.7 IWF
ENERGY STAR Most Efficient	2.76 IMEF, 3.5 IWF	2.74 IMEF, 3.2IWF

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 14 years¹¹.

DEEMED MEASURE COST

The incremental cost for an ENERGY STAR unit is assumed to be \$65 and for an ENERGY STAR Most Efficient unit it is \$210¹².

DEEMED O&M COST ADJUSTMENTS

N/A

¹⁰ See http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/39.

¹¹ Based on DOE Life-Cycle Cost and Payback Period Excel-based analytical tool, available online at: http://www1.eere.energy.gov/buildings/appliance_standards/residential/clothes_washers_support_stakeholder_negotiations.html

¹² Cost estimates are based on Navigant analysis for the Department of Energy (see CW Analysis_09092014.xls). This analysis looked at incremental cost and shipment data from manufacturers and the Association of Home Appliance Manufacturers and attempts to find the costs associated only with the efficiency improvements. The ENERGY STAR level in this analysis was made the baseline (as it is now equivalent), the CEE Tier 3 level was made ENERGY STAR and ENERGY STAR Most efficient was extrapolated based on equal rates. Note these assumptions should be reviewed as qualifying product becomes available.

LOADSHAPE

Loadshape R01 - Residential Clothes Washer

COINCIDENCE FACTOR

The coincidence factor for this measure is 3.8%¹³.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

1. Calculate clothes washer savings based on Modified Energy Factor (MEF).

The Modified Energy Factor (MEF) includes unit operation, water heating and drying energy use: *"MEF is the quotient of the capacity of the clothes container, C, divided by the total clothes washer energy consumption per cycle, with such energy consumption expressed as the sum of the machine electrical energy consumption, M, the hot water energy consumption, E, and the energy required for removal of the remaining moisture in the wash load, D"* ¹⁴.

The hot water and dryer savings calculated here assumes electric DHW and Dryer (this will be separated in Step 2).

$$\text{IMEFsavings}^{15} = \text{Capacity} * (1/\text{IMEFbase} - 1/\text{IMEFeff}) * \text{Ncycles}$$

Where

Capacity	= Clothes Washer capacity (cubic feet) = Actual. If capacity is unknown assume 3.45 cubic feet ¹⁶
IMEFbase	= Integrated Modified Energy Factor of baseline unit = 1.66 ¹⁷
IMEFeff	= Integrated Modified Energy Factor of efficient unit = Actual. If unknown assume average values provided below.
Ncycles	= Number of Cycles per year = 295 ¹⁸

¹³ Calculated from Itron eShapes, 8760 hourly data by end use for Missouri, as provided by Ameren.

¹⁴ Definition provided on the Energy star website.

¹⁵ IMEFsavings represents total kWh only when water heating and drying are 100% electric.

¹⁶ Based on the average clothes washer volume of all units that pass the new Federal Standard on the California Energy Commission (CEC) database of Clothes Washer products accessed on 08/28/2014. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used.

¹⁷ Weighted average IMEF of Federal Standard rating for Front Loading and Top Loading units. Weighting is based upon the relative top v front loading percentage of available non-ENERGY STAR product in the CEC database.

¹⁸ Weighted average of 295 clothes washer cycles per year (based on 2009 Residential Energy Consumption Survey (RECS) national sample survey of housing appliances section, state of IL: <http://www.eia.gov/consumption/residential/data/2009/> If utilities have specific evaluation results providing a more appropriate assumption for single-family or multi-family homes, in a particular market, or geographical area then that should be used.

IMEFSavings is provided below based on deemed values¹⁹:

Efficiency Level	IMEF	IMEFSavings (kWh)
Federal Standard	1.66	0.0
ENERGY STAR	2.26	163
ENERGY STAR Most Efficient	2.74	242

2. Break out savings calculated in Step 1 for electric DHW and electric dryer

$$\Delta \text{kWh} = [\text{Capacity} * 1/\text{IMEF}_{\text{base}} * \text{Ncycles} * (\% \text{CW}_{\text{base}} + (\% \text{DHW}_{\text{base}} * \% \text{Electric_DHW}) + (\% \text{Dryer}_{\text{base}} * \% \text{Electric_Dryer}))] - [\text{Capacity} * 1/\text{IMEF}_{\text{eff}} * \text{Ncycles} * (\% \text{CW}_{\text{eff}} + (\% \text{DHW}_{\text{eff}} * \% \text{Electric_DHW}) + (\% \text{Dryer}_{\text{eff}} * \% \text{Electric_Dryer}))]$$

Where:

%CW = Percentage of total energy consumption for Clothes Washer operation (different for baseline and efficient unit – see table below)

%DHW = Percentage of total energy consumption used for water heating (different for baseline and efficient unit – see table below)

%Dryer = Percentage of total energy consumption for dryer operation (different for baseline and efficient unit – see table below)

	Percentage of Total Energy Consumption ²⁰		
	%CW	%DHW	%Dryer
Baseline	7.6%	31.2%	61.2%
ENERGY STAR	8.1%	23.4%	68.5%
ENERGY STAR Most Efficient	13.6%	10%	76.3%

%Electric_DHW = Percentage of DHW savings assumed to be electric

DHW fuel	%Electric_DHW
Electric	100%
Natural Gas	0%
Unknown	16% ²¹

%Electric_Dryer = Percentage of dryer savings assumed to be electric

Dryer fuel	%Electric_Dryer
Electric	100%

¹⁹ IMEF values are the weighted average of the new ENERGY STAR specifications. Weighting is based upon the relative top v front loading percentage of available ENERGY STAR and ENERGY STAR Most Efficient product in the CEC database. See “CW Analysis_01142016.xls” for the calculation.

²⁰ The percentage of total energy consumption that is used for the machine, heating the hot water or by the dryer is different depending on the efficiency of the unit. Values are based on a weighted average of top loading and front loading units based on data from DOE Life-Cycle Cost and Payback Period Excel-based analytical tool. See “CW Analysis_01142016.xls” for the calculation.

²¹ Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

Dryer fuel	%Electric_Dryer
Natural Gas	0%
Unknown	36% ²²

Using the default assumptions provided above, the prescriptive savings for each configuration are presented below:

	ΔkWH								
	Electric DHW Electric Dryer	Gas DHW Electric Dryer	Electric DHW Gas Dryer	Gas DHW Gas Dryer	Electric DHW Unknown Dryer	Gas DHW Unknown Dryer	Unknown DHW Electric Dryer	Unknown DHW Gas Dryer	Unknown DHW Unknown Dryer
ENERGY STAR	162.7	77.0	96.0	10.2	120.0	34.3	90.7	24.0	48.0
ENERGY STAR Most Efficient	242.1	88.2	149.9	-4.0	183.1	29.2	112.8	20.6	53.8

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh / \text{Hours} * CF$$

Where:

ΔkWh = Energy Savings as calculated above

Hours = Assumed Run hours of Clothes Washer

= 295 hours²³

CF = Summer Peak Coincidence Factor for measure.

= 0.038²⁴

Using the default assumptions provided above, the prescriptive savings for each configuration are presented below:

	ΔkW								
	Electric DHW Electric Dryer	Gas DHW Electric Dryer	Electric DHW Gas Dryer	Gas DHW Gas Dryer	Electric DHW Unknown Dryer	Gas DHW Unknown Dryer	Unknown DHW Electric Dryer	Unknown DHW Gas Dryer	Unknown DHW Unknown Dryer
ENERGY STAR	0.0210	0.0099	0.0124	0.0013	0.0155	0.0044	0.0117	0.0031	0.0062
ENERGY STAR Most Efficient	0.0312	0.0114	0.0193	-0.0005	0.0236	0.0038	0.0145	0.0027	0.0069

NATURAL GAS SAVINGS

Break out savings calculated in Step 1 of electric energy savings (MEF savings) and extract Natural Gas DHW and Natural Gas dryer savings from total savings:

$$\Delta \text{Therm} = [(\text{Capacity} * 1 / \text{IMEF}_{\text{base}} * \text{Ncycles} * ((\% \text{DHW}_{\text{base}} * \% \text{Natural Gas_DHW} * R_{\text{eff}}) + (\% \text{Dryer}_{\text{base}} * \% \text{Gas_Dryer}))) - (\text{Capacity} * 1 / \text{IMEF}_{\text{eff}} * \text{Ncycles} * ((\% \text{DHW}_{\text{eff}} * \% \text{Natural Gas_DHW} * R_{\text{eff}}) +$$

²² Default assumption for unknown is based on percentage of homes with electric dryer from EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used.

²³ Based on a weighted average of 295 clothes washer cycles per year assuming an average load runs for one hour (2009 Residential Energy Consumption Survey (RECS) national sample survey of housing appliances section:

<http://www.eia.gov/consumption/residential/data/2009/>)

²⁴ Calculated from Itron eShapes, 8760 hourly data by end use for Missouri, as provided by Ameren.

$$(\%Dryereff * \%Gas_Dryer))]] * Therm_convert$$

Where:

Therm_convert = Conversion factor from kWh to Therm
= 0.03413

R_eff = Recovery efficiency factor
= 1.26²⁵

%Natural Gas_DHW = Percentage of DHW savings assumed to be Natural Gas

DHW fuel	%Natural Gas_DHW
Electric	0%
Natural Gas	100%
Unknown	84% ²⁶

%Gas_Dryer = Percentage of dryer savings assumed to be Natural Gas

Dryer fuel	%Gas_Dryer
Electric	0%
Natural Gas	100%
Unknown	58% ²⁷

Other factors as defined above

Using the default assumptions provided above, the prescriptive savings for each configuration are presented below:

	ΔTherms								
	Electric DHW Electric Dryer	Gas DHW Electric Dryer	Electric DHW Gas Dryer	Gas DHW Gas Dryer	Electric DHW Unknown Dryer	Gas DHW Unknown Dryer	Unknown DHW Electric Dryer	Unknown DHW Gas Dryer	Unknown DHW Unknown Dryer
ENERGY STAR	0.00	3.7	2.3	6.0	1.3	5.0	3.1	5.4	4.4
ENERGY STAR Most Efficient	0.00	6.6	3.1	9.8	1.8	8.4	5.6	8.7	7.4

WATER IMPACT DESCRIPTIONS AND CALCULATION

$$\Delta Water \text{ (gallons)} = Capacity * (IWFbase - IWFeff) * Ncycles$$

Where

IWFbase = Integrated Water Factor of baseline clothes washer
= 5.92²⁸

²⁵ To account for the different efficiency of electric and Natural Gas hot water heaters (gas water heater: recovery efficiencies ranging from 0.74 to 0.85 (0.78 used), and electric water heater with 0.98 recovery efficiency (http://www.energystar.gov/ia/partners/bldrs_lenders_raters/downloads/Waste_Water_Heat_Recovery_Guidelines.pdf). Therefore a factor of 0.98/0.78 (1.26) is applied.

²⁶ Default assumption for unknown fuel is based on percentage of homes with gas dryer from EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used
²⁷ Ibid.

²⁸ Weighted average IWF of Federal Standard rating for Front Loading and Top Loading units. Weighting is based upon the relative top v front loading percentage of available non-ENERGY STAR product in the CEC database.

IWF_{eff} = Water Factor of efficient clothes washer

= Actual. If unknown assume average values provided below.

Using the default assumptions provided above, the prescriptive water savings for each efficiency level are presented below:

Efficiency Level	IWF ²⁹	ΔWater (gallons per year)
Federal Standard	5.92	0.0
ENERGY STAR	3.93	2024
ENERGY STAR Most Efficient	3.21	2760

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-APL-ESCL-V04-160601

REVIEW DEADLINE: 1/1/2021

²⁹ IWF values are the weighted average of the new ENERGY STAR specifications. Weighting is based upon the relative top v front loading percentage of available ENERGY STAR and ENERGY STAR Most Efficient product in the CEC database. See “CW Analysis_01142016.xls” for the calculation.

5.1.3 ENERGY STAR Dehumidifier

DESCRIPTION

A dehumidifier meeting the minimum qualifying efficiency standard established by the current ENERGY STAR Version 3.0 (effective 10/1/2012) is purchased and installed in a residential setting in place of a unit that meets the minimum federal standard efficiency.

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure, the new dehumidifier must meet the ENERGY STAR standards as defined below:

Capacity (pints/day)	ENERGY STAR Criteria (L/kWh)
<75	≥1.85
75 to ≤185	≥2.80

Qualifying units shall be equipped with an adjustable humidistat control or shall require a remote humidistat control to operate.

DEFINITION OF BASELINE EQUIPMENT

The baseline for this measure is defined as a new dehumidifier that meets the Federal Standard efficiency standards. The Federal Standard for Dehumidifiers as of October 2012 is defined below:

Capacity (pints/day)	Federal Standard Criteria (L/kWh)
Up to 35	≥1.35
> 35 to ≤45	≥1.50
> 45 to ≤ 54	≥1.60
> 54 to ≤ 75	≥1.70
> 75 to ≤ 185	≥2.50

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of the measure is 12 years³⁰.

DEEMED MEASURE COST

The assumed incremental capital cost for this measure is \$60³¹.

LOADSHAPE

Loadshape R12 - Residential - Dehumidifier

³⁰ EPA Research, 2012; ENERGY STAR Dehumidifier Calculator

³¹ Based on extrapolating available data from the Department of Energy's Life Cycle Cost analysis spreadsheet and weighting based on volume of units available:

[See 'DOE life cycle cost_dehumidifier.xls' for calculation.](#)

COINCIDENCE FACTOR

The coincidence factor is assumed to be 37% ³².

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = (((\text{Avg Capacity} * 0.473) / 24) * \text{Hours}) * (1 / (\text{L/kWh}_{\text{Base}}) - 1 / (\text{L/kWh}_{\text{Eff}}))$$

Where:

Avg Capacity	= Average capacity of the unit (pints/day)
	= Actual, if unknown assume capacity in each capacity range as provided in table below, or if capacity range unknown assume average.
0.473	= Constant to convert Pints to Liters
24	= Constant to convert Liters/day to Liters/hour
Hours	= Run hours per year
	= 1632 ³³
L/kWh	= Liters of water per kWh consumed, as provided in tables above

Annual kWh results for each capacity class are presented below:

Capacity Range (pints/day)	Capacity Used (pints/day)	Federal Standard Criteria (≥ L/kWh)	ENERGY STAR Criteria (≥ L/kWh)	Annual kWh		
				Federal Standard	ENERGY STAR	Savings
≤25	20	1.35	1.85	477	348	129
> 25 to ≤35	30	1.35	1.85	715	522	193
> 35 to ≤45	40	1.5	1.85	858	695	162
> 45 to ≤ 54	50	1.6	1.85	1005	869	136
> 54 to ≤ 75	65	1.7	1.85	1230	1130	100
> 75 to ≤ 185	130	2.5	2.8	1673	1493	179
Average ³⁴						140

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \Delta \text{kWh} / \text{Hours} * \text{CF}$$

³² Assume usage is evenly distributed day vs. night, weekend vs. weekday and is used between April through the end of September (4392 possible hours). 1632 operating hours from ENERGY STAR Dehumidifier Calculator. Coincidence peak during summer peak is therefore 1632/4392 = 37.2%

³³ ENERGY STAR Dehumidifier Calculator; 24 hour operation over 68 days of the year.

³⁴ The relative weighting of each product class is based on number of units on the ENERGY STAR certified list. See “Dehumidifier Calcs.xls.”

Where:

Hours = Annual operating hours
 = 1632 hours ³⁵

CF = Summer Peak Coincidence Factor for measure
 = 0.37 ³⁶

Summer coincident peak demand results for each capacity class are presented below:

Capacity (pints/day) Range	Annual Summer peak kW Savings
≤25	0.029
> 25 to ≤35	0.044
> 35 to ≤45	0.037
> 45 to ≤ 54	0.031
> 54 to ≤ 75	0.023
> 75 to ≤ 185	0.041
Average	0.032

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-APL-ESDH-V03-160601

REVIEW DEADLINE: 1/1/2019

³⁵ Based on 68 days of 24 hour operation; ENERGY STAR Dehumidifier Calculator

³⁶ Assume usage is evenly distributed day vs. night, weekend vs. weekday and is used between April through the end of September (4392 possible hours). 1632 operating hours from ENERGY STAR Dehumidifier Calculator. Coincidence peak during summer peak is therefore 1632/4392 = 37.2%

5.1.4 ENERGY STAR Dishwasher

DESCRIPTION

A dishwasher meeting the efficiency specifications of ENERGY STAR is installed in place of a model meeting the federal standard. This measure is only for standard dishwashers, not compact dishwashers. A compact dishwasher is a unit that holds less than eight place settings with six serving pieces.

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is defined as a dishwasher meeting the efficiency specifications of ENERGY STAR. The Energy Star standard is presented in the table below:

Dishwasher Type	Maximum kWh/year	Maximum gallons/cycle
Standard (≥ 8 place settings + six serving pieces)	270	3.5
Standard with Connected Functionality ³⁷	283	
Compact (< 8 place settings + six serving pieces)	203	3.1

DEFINITION OF BASELINE EQUIPMENT

The Baseline reflects the minimum federal efficiency standards for dishwashers effective May 30, 2013, as presented in the table below³⁸.

Dishwasher Type	Maximum kWh/year	Maximum gallons/cycle
Standard	307	5.0
Compact	222	3.5

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of the measure is 13 years³⁹.

DEEMED MEASURE COST

The incremental cost for this measure is \$50⁴⁰.

³⁷ The new ENERGY STAR specification “establishes optional connected criteria for dishwashers. ENERGY STAR certified dishwashers with connected functionality offer favorable attributes for demand response programs to consider, since their peak energy consumption is relatively high, driven by water heating. ENERGY STAR certified dishwashers with connected functionality will offer consumers new convenience and energy-saving features, such as alerts for cycle completion and/or recommended maintenance, as well as feedback on the energy use of the product”. See ‘ENERGY STAR Residential Dishwasher Final Version 6.0 Cover Memo.pdf’. Calculated as per Version 6.0 specification; “ENERGY STAR Residential Dishwasher Version 6.0 Final Program Requirements.pdf”. Note that the potential for demand response and additional peak savings from units with Connected Functionality have not been explored. This could be a potential addition in a future version.

³⁸ http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/67

³⁹ Koomey, Jonathan et al. (Lawrence Berkeley National Lab), Projected Regional Impacts of Appliance Efficiency Standards for the U.S. Residential Sector, February 1998.

⁴⁰ Estimate based on review of Energy Star stakeholder documents

LOADSHAPE

Loadshape R02 - Residential Dish Washer

COINCIDENCE FACTORThe coincidence factor is assumed to be 2.6%⁴¹.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh}^{42} = ((\text{kWh}_{\text{Base}} - \text{kWh}_{\text{ESTAR}}) * (\% \text{kWh}_{\text{op}} + (\% \text{kWh}_{\text{heat}} * \% \text{Electric_DHW})))$$

Where:

 kWh_{BASE} = Baseline kWh consumption per year

Dishwasher Type	Maximum kWh/year
Standard	307
Compact	222

 $\text{kWh}_{\text{ESTAR}}$ = ENERGY STAR kWh annual consumption

Dishwasher Type	Maximum kWh/year
Standard	270
Standard with Connected Functionality	283
Compact	203

$\% \text{kWh}_{\text{op}}$ = Percentage of dishwasher energy consumption used for unit operation
 = 1 - 56%⁴³
 = 44%

$\% \text{kWh}_{\text{heat}}$ = Percentage of dishwasher energy consumption used for water heating
 = 56%⁴⁴

 $\% \text{Electric_DHW}$ = Percentage of DHW savings assumed to be electric

DHW fuel	$\% \text{Electric_DHW}$
Electric	100%
Natural Gas	0%

⁴¹ Calculated from Itron eShapes, 8760 hourly data by end use for Missouri, as provided by Ameren.⁴² The Federal Standard and ENERGY STAR annual consumption values include electric consumption for both the operation of the machine and for heating the water that is used by the machine.⁴³ ENERGY STAR Dishwasher Calculator(http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/CalculatorConsumerDishwasher.xls)⁴⁴ Ibid.

DHW fuel	%Electric_DHW
Unknown	16% ⁴⁵

Dishwasher Type	ΔkWh		
	With Electric DHW	With Gas DHW	With Unknown DHW
ENERGY STAR Standard	37.0	16.3	19.6
ENERGY STAR Standard with Connected Functionality	24.0	10.6	12.7
ENERGY STAR Compact	19.0	8.4	10.1

SUMMER COINCIDENT PEAK DEMAND SAVINGS⁴⁶

$$\Delta kW = \Delta kWh / \text{Hours} * CF$$

Where:

Hours = Annual operating hours⁴⁷

= 252 hours

CF = Summer Peak Coincidence Factor

= 2.6%⁴⁸

Dishwasher Type	ΔkW		
	With Electric DHW	With Gas DHW	With Unknown DHW
ENERGY STAR Standard	0.0038	0.0017	0.0020
ENERGY STAR Standard with Connected Functionality	0.0025	0.0011	0.0013
ENERGY STAR Compact	0.0020	0.0009	0.0010

NATURAL GAS SAVINGS

$$\Delta \text{Therm} = (kWh_{\text{Base}} - kWh_{\text{ESTAR}}) * \%kWh_{\text{heat}} * \% \text{Natural Gas_DHW} * R_{\text{eff}} * 0.03413$$

Where

%kWh_{heat} = % of dishwasher energy used for water heating

= 56%

%Natural Gas_{DHW} = Percentage of DHW savings assumed to be Natural Gas

DHW fuel	%Natural Gas_DHW
Electric	0%
Natural Gas	100%

⁴⁵ Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used.

⁴⁶ Note that the potential for demand response and additional peak savings from units with Connected Functionality have not been explored. This could be a potential addition in a future version.

⁴⁷ Assuming one and a half hours per cycle and 168 cycles per year therefore 252 operating hours per year; 168 cycles per year is based on a weighted average of dishwasher usage in Illinois derived from the 2009 RECs data;
<http://205.254.135.7/consumption/residential/data/2009/>

⁴⁸ End use data from Ameren representing the average DW load during peak hours/peak load.

DHW fuel	%Natural Gas_DHW
Unknown	84% ⁴⁹

R_eff = Recovery efficiency factor

= 1.26⁵⁰

0.03413 = factor to convert from kWh to Therm

Dishwasher Type	ΔTherms		
	With Electric DHW	With Gas DHW	With Unknown DHW
ENERGY STAR Standard	0.00	0.89	0.75
ENERGY STAR Standard with Connected Functionality	0.00	0.58	0.49
ENERGY STAR Compact	0.00	0.46	0.38

WATER IMPACT DESCRIPTIONS AND CALCULATION

$$\Delta \text{Water} = \text{Water}_{\text{Base}} - \text{Water}_{\text{EFF}}$$

Where

Water_{Base} = water consumption of conventional unit

Dishwasher Type	Water _{Base} (gallons) ⁵¹
Standard	840
Compact	588

Water_{EFF} = annual water consumption of efficient unit:

Dishwasher Type	Water _{EFF} (gallons) ⁵²
Standard	588
Compact	521

Dishwasher Type	ΔWater (gallons) ⁵³
Standard	252
Compact	67

⁴⁹ Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used.

⁵⁰ To account for the different efficiency of electric and Natural Gas hot water heaters (gas water heater: recovery efficiencies ranging from 0.74 to 0.85 (0.78 used), and electric water heater with 0.98 recovery efficiency (http://www.energystar.gov/ia/partners/bldrs_lenders_raters/downloads/Waste_Water_Heat_Recovery_Guidelines.pdf). Therefore a factor of 0.98/0.78 (1.26) is applied.

⁵¹ Assuming maximum allowed from specifications and 168 cycles per year based on a weighted average of dishwasher usage in Illinois derived from the 2009 RECS data; <http://205.254.135.7/consumption/residential/data/2009/>

⁵² Assuming maximum allowed from specifications and 168 cycles per year based on a weighted average of dishwasher usage in Illinois derived from the 2009 RECS data; <http://205.254.135.7/consumption/residential/data/2009/>

⁵³ Assuming maximum allowed from specifications and 168 cycles per year based on a weighted average of dishwasher usage in Illinois derived from the 2009 RECS data; <http://205.254.135.7/consumption/residential/data/2009/>

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-APL-ESDI-V03-160601

REVIEW DEADLINE: 6/1/2018

5.1.5 ENERGY STAR Freezer

DESCRIPTION

A freezer meeting the efficiency specifications of ENERGY STAR is installed in place of a model meeting the federal standard (NAECA). Energy usage specifications are defined in the table below (note, AV is the freezer Adjusted Volume and is calculated as $1.73 \times \text{Total Volume}$):

Product Category	Volume (cubic feet)	Assumptions up to September 2014		Assumptions after September 2014	
		Federal Baseline Maximum Energy Usage in kWh/year ⁵⁴	ENERGY STAR Maximum Energy Usage in kWh/year ⁵⁵	Federal Baseline Maximum Energy Usage in kWh/year ⁵⁶	ENERGY STAR Maximum Energy Usage in kWh/year ⁵⁷
Upright Freezers with Manual Defrost	7.75 or greater	$7.55 \times \text{AV} + 258.3$	$6.795 \times \text{AV} + 232.47$	$5.57 \times \text{AV} + 193.7$	$5.01 \times \text{AV} + 174.3$
Upright Freezers with Automatic Defrost	7.75 or greater	$12.43 \times \text{AV} + 326.1$	$11.187 \times \text{AV} + 293.49$	$8.62 \times \text{AV} + 228.3$	$7.76 \times \text{AV} + 205.5$
Chest Freezers and all other Freezers except Compact Freezers	7.75 or greater	$9.88 \times \text{AV} + 143.7$	$8.892 \times \text{AV} + 129.33$	$7.29 \times \text{AV} + 107.8$	$6.56 \times \text{AV} + 97.0$
Compact Upright Freezers with Manual Defrost	< 7.75 and 36 inches or less in height	$9.78 \times \text{AV} + 250.8$	$7.824 \times \text{AV} + 200.64$	$8.65 \times \text{AV} + 225.7$	$7.79 \times \text{AV} + 203.1$
Compact Upright Freezers with Automatic Defrost	< 7.75 and 36 inches or less in height	$11.40 \times \text{AV} + 391$	$9.12 \times \text{AV} + 312.8$	$10.17 \times \text{AV} + 351.9$	$9.15 \times \text{AV} + 316.7$
Compact Chest Freezers	< 7.75 and 36 inches or less in height	$10.45 \times \text{AV} + 152$	$8.36 \times \text{AV} + 121.6$	$9.25 \times \text{AV} + 136.8$	$8.33 \times \text{AV} + 123.1$

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is defined as a freezer meeting the efficiency specifications of ENERGY STAR, as defined below and calculated above:

Equipment	Volume	Criteria
Full Size Freezer	7.75 cubic feet or greater	At least 10% more energy efficient than the minimum federal government standard (NAECA).
Compact Freezer	Less than 7.75 cubic feet and 36 inches or less in height	At least 20% more energy efficient than the minimum federal government standard (NAECA).

⁵⁴ http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/43

⁵⁵ http://www.energystar.gov/ia/products/appliances/refrig/NAECA_calculation.xls?c827-f746

⁵⁶ http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/43

⁵⁷ <http://www.energystar.gov/products/specs/sites/products/files/ENERGY%20STAR%20Final%20Version%205.0%20Residential%20Refrigerators%20and%20Freezers%20Specification.pdf>

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be a model that meets the federal minimum standard for energy efficiency. The standard varies depending on the size and configuration of the freezer (chest freezer or upright freezer, automatic or manual defrost) and is defined in the table above.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 11 years⁵⁸.

DEEMED MEASURE COST

The incremental cost for this measure is \$35⁵⁹.

LOADSHAPE

Loadshape R04 - Residential Freezer

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 95%⁶⁰.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS:**

$$\Delta \text{kWh} = \text{kWh}_{\text{Base}} - \text{kWh}_{\text{ESTAR}}$$

Where:

kWh_{BASE} = Baseline kWh consumption per year as calculated in algorithm provided in table above.

$\text{kWh}_{\text{ESTAR}}$ = ENERGY STAR kWh consumption per year as calculated in algorithm provided in table above.

For example for a 7.75 cubic foot Upright Freezers with Manual Defrost purchased after September 2014:

$$\begin{aligned} \Delta \text{kWh} &= (5.57 * (7.75 * 1.73) + 193.7) - (5.01 * (7.75 * 1.73) + 174.3) \\ &= 268.4 - 241.5 \\ &= 26.9 \text{ kWh} \end{aligned}$$

If volume is unknown, use the following default values:

⁵⁸ Energy Star Freezer Calculator;

http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Consumer_Residential_Freezer_Sav_Calc.xls?570a-f000

⁵⁹ Based on review of data from the Northeast Regional ENERGY STAR Consumer Products Initiative; “2009 ENERGY STAR Appliances Practices Report”, submitted by Lockheed Martin, December 2009.

⁶⁰ Based on eShapes Residential Freezer load data as provided by Ameren.

Product Category	Volume Used ⁶¹	Assumptions up to September 2014			Assumptions after September 2014		
		kWh _{BASE}	kWh _{ESTAR}	kWh Savings	kWh _{BASE}	kWh _{ESTAR}	kWh Savings
Upright Freezers with Manual Defrost	27.9	469.1	422.2	46.9	349.2	314.2	35.0
Upright Freezers with Automatic Defrost	27.9	673.2	605.9	67.3	469.0	422.2	46.8
Chest Freezers and all other Freezers except Compact Freezers	27.9	419.6	377.6	42.0	311.4	280.2	31.2
Compact Upright Freezers with Manual Defrost	10.4	352.3	281.9	70.5	467.2	420.6	46.6
Compact Upright Freezers with Automatic Defrost	10.4	509.3	407.5	101.9	635.9	572.2	63.7
Compact Chest Freezers	10.4	260.5	208.4	52.1	395.1	355.7	39.4

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh / \text{Hours} * CF$$

Where:

ΔkWh = Gross customer annual kWh savings for the measure

Hours = Full Load hours per year

$$= 5890^{62}$$

CF = Summer Peak Coincident Factor

$$= 0.95^{63}$$

For example for a 7.75 cubic foot Upright Freezers with Manual Defrost:

$$\Delta kW = 26.9 / 5890 * 0.95$$

$$= 0.0043 \text{ kW}$$

If volume is unknown, use the following default values:

Product Category	Assumptions up to September 2014	Assumptions after September 2014
	kW Savings	kW Savings
Upright Freezers with Manual Defrost	0.0076	0.0057
Upright Freezers with Automatic Defrost	0.0109	0.0076
Chest Freezers and all other Freezers except Compact Freezers	0.0068	0.0050
Compact Upright Freezers with Manual Defrost	0.0114	0.0075

⁶¹ Volume is based on ENERGY STAR Calculator assumption of 16.14 ft³ average volume, converted to Adjusted volume by multiplying by 1.73.

⁶² Calculated from eShapes Residential Freezer load data as provided by Ameren by dividing total annual load by the maximum kW in any one hour.

⁶³ Based on eShapes Residential Freezer load data as provided by Ameren.

Product Category	Assumptions up to September 2014	Assumptions after September 2014
	kW Savings	kW Savings
Compact Upright Freezers with Automatic Defrost	0.0164	0.0103
Compact Chest Freezers	0.0084	0.0064

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-APL-ESFR-V02-140601

REVIEW DEADLINE: 1/1/2021

5.1.6 ENERGY STAR and CEE Tier 2 Refrigerator

DESCRIPTION

This measure relates to:

- a) Time of Sale: the purchase and installation of a new refrigerator meeting either ENERGY STAR or CEE TIER 2 specifications.
- b) Early Replacement: the early removal of an existing residential inefficient Refrigerator from service, prior to its natural end of life, and replacement with a new ENERGY STAR or CEE Tier 2 qualifying unit. Savings are calculated between existing unit and efficient unit consumption during the remaining life of the existing unit, and between new baseline unit and efficient unit consumption for the remainder of the measure life.

Energy usage specifications are defined in the table below (note, Adjusted Volume is calculated as the fresh volume + (1.63 * Freezer Volume):

Product Category	Existing Unit	Assumptions up to September 2014		Assumptions after September 2014	
	Based on Refrigerator Recycling algorithm	Federal Baseline Maximum Energy Usage in kWh/year ⁶⁴	ENERGY STAR Maximum Energy Usage in kWh/year ⁶⁵	Federal Baseline Maximum Energy Usage in kWh/year ⁶⁶	ENERGY STAR Maximum Energy Usage in kWh/year ⁶⁷
1. Refrigerators and Refrigerator-freezers with manual defrost	Use Algorithm in 5.1.8 Refrigerator and Freezer Recycling measure to estimate existing unit consumption	$8.82 \cdot AV + 248.4$	$7.056 \cdot AV + 198.72$	$6.79AV + 193.6$	$6.11 \cdot AV + 174.2$
2. Refrigerator-Freezer--partial automatic defrost		$8.82 \cdot AV + 248.4$	$7.056 \cdot AV + 198.72$	$7.99AV + 225.0$	$7.19 \cdot AV + 202.5$
3. Refrigerator-Freezers--automatic defrost with top-mounted freezer without through-the-door ice service and all-refrigerators--automatic defrost		$9.80 \cdot AV + 276$	$7.84 \cdot AV + 220.8$	$8.07AV + 233.7$	$7.26 \cdot AV + 210.3$
4. Refrigerator-Freezers--automatic defrost with side-mounted freezer without through-the-door ice service		$4.91 \cdot AV + 507.5$	$3.928 \cdot AV + 406$	$8.51AV + 297.8$	$7.66 \cdot AV + 268.0$
5. Refrigerator-Freezers--automatic defrost with bottom-mounted freezer		$4.60 \cdot AV + 459$	$3.68 \cdot AV + 367.2$	$8.85AV + 317.0$	$7.97 \cdot AV + 285.3$

⁶⁴ http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/43

⁶⁵ http://www.energystar.gov/ia/products/appliances/refrig/NAECA_calculation.xls?c827-f746

⁶⁶ http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/43

⁶⁷ <http://www.energystar.gov/products/specs/sites/products/files/ENERGY%20STAR%20Final%20Version%205.0%20Residential%20Refrigerators%20and%20Freezers%20Specification.pdf>

Product Category	Existing Unit	Assumptions up to September 2014		Assumptions after September 2014	
	Based on Refrigerator Recycling algorithm	Federal Baseline Maximum Energy Usage in kWh/year ⁶⁴	ENERGY STAR Maximum Energy Usage in kWh/year ⁶⁵	Federal Baseline Maximum Energy Usage in kWh/year ⁶⁶	ENERGY STAR Maximum Energy Usage in kWh/year ⁶⁷
without through-the-door ice service					
5A Refrigerator-freezer—automatic defrost with bottom-mounted freezer with through-the-door ice service		N/A	N/A	$9.25AV + 475.4$	$8.33 * AV + 436.3$
6. Refrigerator-Freezers--automatic defrost with top-mounted freezer with through-the-door ice service		$10.20*AV+356$	$8.16*AV+284.8$	$8.40AV + 385.4$	$7.56 * AV + 355.3$
7. Refrigerator-Freezers--automatic defrost with side-mounted freezer with through-the-door ice service		$10.10*AV+406$	$8.08*AV+324.8$	$8.54AV + 432.8$	$7.69 * AV + 397.9$

Note CEE Tier 2 standard criteria is 25% less consumption than a new baseline unit. It is assumed that after September 2014 when the Federal Standard and ENERGY STAR specifications change, the CEE Tier 2 will remain set at 25% less than the new baseline assumption.

This measure was developed to be applicable to the following program types: TOS, NC, EREP.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is defined as a refrigerator meeting the efficiency specifications of ENERGY STAR or CEE Tier 2 (defined as requiring $\geq 20\%$ or $\geq 25\%$ less energy consumption than an equivalent unit meeting federal standard requirements respectively). The ENERGY STAR standard varies according to the size and configuration of the unit, as shown in table above.

DEFINITION OF BASELINE EQUIPMENT

Time of Sale: baseline is a new refrigerator meeting the minimum federal efficiency standard for refrigerator efficiency. The current federal minimum standard varies according to the size and configuration of the unit, as shown in table above. Note also that this federal standard will be increased for units manufactured after September 1, 2014.

Early Replacement: the baseline is the existing refrigerator for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 12 years.⁶⁸

Remaining life of existing equipment is assumed to be 4 years⁶⁹

DEEMED MEASURE COST

Time of Sale: The incremental cost for this measure is assumed to be \$40⁷⁰ for an ENERGY STAR unit and \$140⁷¹ for a CEE Tier 2 unit.

Early Replacement: The measure cost is the full cost of removing the existing unit and installing a new one. The actual program cost should be used. If unavailable assume \$451 for ENERGY STAR unit and \$551 for CEE Tier 2 unit⁷².

The avoided replacement cost (after 4 years) of a baseline replacement refrigerator is \$413⁷³. This cost should be discounted to present value using the nominal societal discount rate.

LOADSHAPE

Loadshape R05 - Residential Refrigerator

COINCIDENCE FACTOR

A coincidence factor is not used to calculate peak demand savings for this measure, see below.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS:

Time of Sale: $\Delta \text{kWh} = \text{UEC}_{\text{BASE}} - \text{UEC}_{\text{EE}}$

Early Replacement:

ΔkWh for remaining life of existing unit (1st 4 years) = $\text{UEC}_{\text{EXIST}} - \text{UEC}_{\text{EE}}$

ΔkWh for remaining measure life (next 8 years) = $\text{UEC}_{\text{BASE}} - \text{UEC}_{\text{EE}}$

Where:

$\text{UEC}_{\text{EXIST}}$ = Annual Unit Energy Consumption of existing unit as calculated in algorithm from 5.1.8 Refrigerator and Freezer Recycling measure.

UEC_{BASE} = Annual Unit Energy Consumption of baseline unit as calculated in algorithm provided in

⁶⁸ From ENERGY STAR calculator:

http://www.energystar.gov/buildings/sites/default/uploads/files/appliance_calculator.xlsx?7224-046c=&7224-__046ceiling_fan_calculator_xlsx=&f7d8-39dd&f7d8-39dd

⁶⁹ Standard assumption of one third of effective useful life.

⁷⁰ From ENERGY STAR calculator linked above.

⁷¹ Based on weighted average of units participating in Efficiency Vermont program and retail cost data provided in Department of Energy, "TECHNICAL REPORT: Analysis of Amended Energy Conservation Standards for Residential Refrigerator-Freezers", October 2005; http://www1.eere.energy.gov/buildings/appliance_standards/pdfs/refrigerator_report_1.pdf

⁷² ENERGY STAR full cost is based upon IL PHA Efficient Living Program data on sample size of 910 replaced units finding average cost of \$430 plus an average recycling/removal cost of \$21. The CEE Tier 2 estimate uses the delta from the Time of Sale estimate.

⁷³ Calculated using incremental cost from Time of Sale measure and applying inflation rate of 1.91%.

table above.

UEC_{EE} = Annual Unit Energy Consumption of ENERGY STAR unit as calculated in algorithm provided in table above.

For CEE Tier 2, unit consumption is calculated as 25% lower than baseline.

If volume is unknown, use the following defaults, based on an assumed Adjusted Volume of 25.8⁷⁴:

Assumptions prior to standard changes on September 1st, 2014:

Product Category	Existing Unit UEC _{EXIST} 75	New Baseline UEC _{BASE}	New Efficient UEC _{EE}		Early Replacement (1 st 4 years) ΔkWh		Time of Sale and Early Replacement (last 8 years) ΔkWh	
			ENERGY STAR	CEE T2	ENERGY STAR	CEE T2	ENERGY STAR	CEE T2
1. Refrigerators and Refrigerator-freezers with manual defrost	1027.7	475.7	380.5	356.8	647.2	671.0	95.1	118.9
2. Refrigerator-Freezer--partial automatic defrost	1027.7	475.7	380.5	356.8	647.2	671.0	95.1	118.9
3. Refrigerator-Freezers--automatic defrost with top-mounted freezer without through-the-door ice service and all-refrigerators--automatic defrost	814.5	528.5	422.8	396.4	391.7	418.1	105.7	132.1
4. Refrigerator-Freezers--automatic defrost with side-mounted freezer without through-the-door ice service	1241.0	634.0	507.2	475.5	733.7	765.4	126.8	158.5
5. Refrigerator-Freezers--automatic defrost with bottom-mounted freezer without through-the-door ice service	814.5	577.5	462.0	433.2	352.5	381.4	115.5	144.4
6. Refrigerator-Freezers--automatic defrost with top-mounted freezer with through-the-door ice service	814.5	618.8	495.1	464.1	319.5	350.4	123.8	154.7
7. Refrigerator-Freezers--automatic defrost with side-mounted freezer with through-the-door ice service	1241.0	666.3	533.0	499.7	707.9	741.3	133.3	166.6

Assumptions after standard changes on September 1st, 2014:

⁷⁴ Volume is based on the ENERGY STAR calculator average assumption of 14.75 ft³ fresh volume and 6.76 ft³ freezer volume.

⁷⁵ Estimates of existing unit consumption are based on using the 5.1.8 Refrigerator and Freezer Recycling algorithm and the inputs described here: Age = 10 years, Pre-1990 = 0, Size = 21.5 ft³ (from ENERGY STAR calc and consistent with AV of 25.8), Single Door = 0, Side by side = 1 for classifications stating side by side, 0 for classifications stating top/bottom, and 0.5 for classifications that do not distinguish, Primary appliances = 1, unconditioned = 0, Part use factor = 0.

Product Category	Existing Unit UEC _{EXIST} ⁷⁶	New Baseline UEC _{BASE}	New Efficient UEC _{EE}		Early Replacement (1 st 4 years) ΔkWh		Time of Sale and Early Replacement (last 8 years) ΔkWh	
			ENERGY STAR	CEE T2	ENERGY STAR	CEE T2	ENERGY STAR	CEE T2
1. Refrigerators and Refrigerator-freezers with manual defrost	1027.7	368.6	331.6	276.4	696.1	751.3	36.9	92.1
2. Refrigerator-Freezer--partial automatic defrost	1027.7	430.9	387.8	323.2	640.0	704.6	43.1	107.7
3. Refrigerator-Freezers--automatic defrost with top-mounted freezer without through-the-door ice service and all-refrigerators--automatic defrost	814.5	441.7	397.4	331.2	417.2	483.3	44.3	110.4
4. Refrigerator-Freezers--automatic defrost with side-mounted freezer without through-the-door ice service	1241.0	517.1	465.4	387.8	775.6	853.1	51.7	129.3
5. Refrigerator-Freezers--automatic defrost with bottom-mounted freezer without through-the-door ice service	814.5	545.1	490.7	408.8	323.9	405.8	54.4	136.3
5A Refrigerator-freezer—automatic defrost with bottom-mounted freezer with through-the-door ice service	814.5	713.8	651.0	535.3	163.6	279.2	62.8	178.4
6. Refrigerator-Freezers--automatic defrost with top-mounted freezer with through-the-door ice service	814.5	601.9	550.1	451.4	264.4	363.2	51.7	150.5
7. Refrigerator-Freezers--automatic defrost with side-mounted freezer with through-the-door ice service	1241.0	652.9	596.1	489.6	644.9	751.3	56.8	163.2

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = (\Delta kWh / 8766) * TAF * LSAF$$

Where:

TAF = Temperature Adjustment Factor

⁷⁶ Estimates of existing unit consumption are based on using the 5.1.8 Refrigerator and Freezer Recycling algorithm and the inputs described here: Age = 10 years, Pre-1990 = 0, Size = 21.5 ft³ (from ENERGY STAR calc and consistent with AV of 25.8), Single Door = 0, Side by side = 1 for classifications stating side by side, 0 for classifications stating top/bottom, and 0.5 for classifications that do not distinguish, Primary appliances = 1, unconditioned = 0, Part use factor = 0.

$$= 1.25^{77}$$

LSAF = Load Shape Adjustment Factor

$$= 1.057^{78}$$

If volume is unknown, use the following defaults:

Product Category	Assumptions prior to September 2014 standard change Δ kW				Assumptions after September 2014 standard change Δ kW			
	Early Replacement (1 st 4 years)		Time of Sale and Early Replacement (last 8 years)		Early Replacement (1 st 4 years)		Time of Sale and Early Replacement (last 8 years)	
	ENERGY STAR	CEE T2	ENERGY STAR	CEE T2	ENERGY STAR	CEE T2	ENERGY STAR	CEE T2
1. Refrigerators and Refrigerator-freezers with manual defrost	0.098	0.101	0.014	0.018	0.105	0.113	0.006	0.014
2. Refrigerator-Freezer--partial automatic defrost	0.098	0.101	0.014	0.018	0.096	0.106	0.006	0.016
3. Refrigerator-Freezers--automatic defrost with top-mounted freezer without through-the-door ice service and all-refrigerators--automatic defrost	0.059	0.063	0.016	0.020	0.063	0.073	0.007	0.017
4. Refrigerator-Freezers--automatic defrost with side-mounted freezer without through-the-door ice service	0.111	0.115	0.019	0.024	0.117	0.129	0.008	0.019
5. Refrigerator-Freezers--automatic defrost with bottom-mounted freezer without through-the-door ice service	0.053	0.057	0.017	0.022	0.049	0.061	0.008	0.021
5A Refrigerator-freezer—automatic defrost with bottom-mounted freezer with through-the-door ice service	n/a	n/a	n/a	n/a	0.025	0.042	0.009	0.027
6. Refrigerator-Freezers--automatic defrost with top-mounted freezer with through-the-door ice service	0.048	0.053	0.019	0.023	0.040	0.055	0.008	0.023
7. Refrigerator-Freezers--automatic defrost with side-mounted freezer with through-the-door ice service	0.107	0.112	0.020	0.025	0.097	0.113	0.009	0.025

NATURAL GAS SAVINGS

N/A

⁷⁷ Average temperature adjustment factor (to account for temperature conditions during peak period as compared to year as a whole) based on Blasnik, Michael, "Measurement and Verification of Residential Refrigerator Energy Use, Final Report, 2003-2004 Metering Study", July 29, 2004 (p. 47). It assumes 90 °F average outside temperature during peak period, 71°F average temperature in kitchens and 65°F average temperature in basement, and uses assumption that 66% of homes in Illinois have central cooling (CAC saturation: "Table HC7.9 Air Conditioning in Homes in Midwest Region, Divisions, and States, 2009 from Energy Information Administration", 2009 Residential Energy Consumption Survey; <http://www.eia.gov/consumption/residential/data/2009/xls/HC7.9%20Air%20Conditioning%20in%20Midwest%20Region.xls>)

⁷⁸ Daily load shape adjustment factor (average load in peak period /average daily load) also based on Blasnik, Michael, "Measurement and Verification of Residential Refrigerator Energy Use, Final Report, 2003-2004 Metering Study", July 29, 2004 (p. 48, using the average Existing Units Summer Profile for hours 13 through 17)

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-APL-ESRE-V05-180101

REVIEW DEADLINE: 1/1/2021

5.1.7 ENERGY STAR Room Air Conditioner

DESCRIPTION

This measure relates to:

- a) Time of Sale the purchase and installation of a room air conditioning unit that meets ENERGY STAR version 4.0 which is effective October 26th 2015), in place of a baseline unit. The baseline is based on the Federal Standard effective June 1st, 2014.

Product Type and Class (Btu/hr)		Federal Standard with louvered sides (CEER) ⁷⁹	Federal Standard without louvered sides (CEER)	ENERGY STAR v4.0 with louvered sides (CEER) ⁸⁰	ENERGY STAR v4.0 without louvered sides (CEER)
Without Reverse Cycle	< 8,000	11.0	10.0	11.5	10.5
	8,000 to 10,999	10.9	9.6	11.4	10.1
	11,000 to 13,999	10.9	9.5	11.4	10.0
	14,000 to 19,999	10.7	9.3	11.2	9.7
	20,000 to 24,999	9.4	9.4	9.8	9.8
	>=25,000	9.0	9.4	9.4	9.8
With Reverse Cycle	<14,000	9.8	9.3	10.3	9.7
	14,000 to 19,999	9.8	8.7	10.3	9.1
	>=20,000	9.3	8.7	9.7	9.1
Casement only		9.5		10.0	
Casement-Slider		10.4		10.8	

Side louvers extend from a room air conditioner model in order to position the unit in a window. A model without louvered sides is placed in a built-in wall sleeve and are commonly referred to as "through-the-wall" or "built-in" models.

Casement-only refers to a room air conditioner designed for mounting in a casement window of a specific size.

Casement-slider refers to a room air conditioner with an encased assembly designed for mounting in a sliding or casement window of a specific size.

Reverse cycle refers to the heating function found in certain room air conditioner models.

- b) Early Replacement: the early removal of an existing residential inefficient Room AC unit from service, prior to its natural end of life, and replacement with a new ENERGY STAR qualifying unit. Savings are calculated between existing unit and efficient unit consumption during the remaining life of the existing unit, and between new baseline unit and efficient unit consumption for the remainder of the measure life.

This measure was developed to be applicable to the following program types: TOS, NC, EREP.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the new room air conditioning unit must meet the ENERGY STAR version 4.0 (effective October 26th 2015)⁸¹ efficiency standards presented above.

⁷⁹ See DOE's Appliance and Equipment Standards for Room AC;
https://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/41

⁸⁰ ENERGY STAR Version 4.0 Room Air Conditioners Program Requirements

⁸¹ ENERGY STAR Version 4.0 Room Air Conditioners Program Requirements

DEFINITION OF BASELINE EQUIPMENT

Time of Sale: the baseline assumption is a new room air conditioning unit that meets the Federal Standard (effective June 1st, 2014)⁸² efficiency standards as presented above.

Early Replacement: the baseline is the existing Room AC for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 12 years⁸³.

Remaining life of existing equipment is assumed to be 4 years⁸⁴

DEEMED MEASURE COST

Time of Sale: The incremental cost for this measure is assumed to be \$40 for a ENERGY STAR unit⁸⁵.

Early Replacement: The measure cost is the full cost of removing the existing unit and installing a new one. The actual program cost should be used. If unavailable assume \$448 for ENERGY STAR unit⁸⁶.

The avoided replacement cost (after 4 years) of a baseline replacement unit is \$432.⁸⁷ This cost should be discounted to present value using the nominal societal discount rate.

LOADSHAPE

Loadshape R08 - Residential Cooling

COINCIDENCE FACTOR

The coincidence factor for this measure is assumed to be 0.3⁸⁸.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

Time of Sale: $\Delta \text{kWh} = (\text{FLH}_{\text{RoomAC}} * \text{Btu/H} * (1/\text{CEER}_{\text{base}} - 1/\text{CEER}_{\text{ee}}))/1000$

Early Replacment:

$\Delta \text{kWh for remaining life of existing unit (1st 4 years) = (\text{FLH}_{\text{RoomAC}} * \text{Btu/H} * (1/(\text{EER}_{\text{exist}}/1.01) -$

⁸² See DOE's Appliance and Equipment Standards for Room AC;

https://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/41

⁸³ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf

⁸⁴ Standard assumption of one third of effective useful life.

⁸⁵ Incremental cost based on field study conducted by Efficiency Vermont.

⁸⁶ Based on IL PHA Efficient Living Program Data for 810 replaced units showing \$416 per unit plus \$32 average recycling/removal cost.

⁸⁷ Estimate based upon Time of Sale incremental costs and applying inflation rate of 1.91%.

⁸⁸ Consistent with coincidence factors found in: RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008

(http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RA_C.pdf)

$$1/CEER_{ee})/1000$$

$$\Delta kWh \text{ for remaining measure life (next 8 years)} = (FLH_{RoomAC} * Btu/H * (1/CEER_{base} - 1/CEER_{ee}))/1000$$

Where:

FLH_{RoomAC} = Full Load Hours of room air conditioning unit
= dependent on location⁸⁹:

Climate Zone (City based upon)	FLH_{RoomAC}
1 (Rockford)	220
2 (Chicago)	210
3 (Springfield)	319
4 (Belleville)	428
5 (Marion)	374
Weighted Average ⁹⁰	248

Btu/H = Size of rebated unit
= Actual. If unknown assume 8500 Btu/hr⁹¹

EER_{exist} = Efficiency of existing unit
= Actual. If unknown assume 7.7⁹²

1.01 = Factor to convert EER to CEER (CEER includes standby and off power consumption)⁹³.

$CEER_{base}$ = Combined Energy Efficiency Ratio of baseline unit
= As provided in tables above

$CEER_{ee}$ = Combined Energy Efficiency Ratio of ENERGY STAR unit
= Actual. If unknown assume minimum qualifying standard as provided in tables above

⁸⁹ Full load hours for room AC is significantly lower than for central AC. The average ratio of FLH for Room AC (provided in RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008) to FLH for Central Cooling for the same location (provided by AHRI: http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls) is 31%. This ratio is applied to those IL cities that have FLH for Central Cooling provided in the Energy Star calculator. For other cities this is extrapolated using the FLH assumptions VEIC have developed for Central AC. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

⁹⁰ Weighted based on number of residential occupied housing units in each zone.

⁹¹ Based on maximum capacity average from the RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008

⁹² Based on Nexus Market Research Inc, RLW Analytics, December 2005; "Impact, Process, and Market Study of the Connecticut Appliance Retirement Program: Overall Report."

⁹³ Since the existing unit will be rated in EER, this factor is used to appropriately compare with the new CEER rating. Version 3.0 of the ENERGY STAR specification provided equivalent EER and CEER ratings and for the most popular size band the EER rating is approximately 1% higher than the CEER. See 'ENERGY STAR Version 3.0 Room Air Conditioners Program Requirements'.

Time of Sale:

For example for an 8,500 Btu/H capacity unit, with louvered sides, in an unknown location:

$$\begin{aligned}\Delta\text{kWh}_{\text{CEE TIER 1}} &= (248 * 8500 * (1/10.9 - 1/11.4)) / 1000 \\ &= 8.5 \text{ kWh}\end{aligned}$$

Early Replacement:

A 7.7EER, 9000Btu/h unit is removed from a home in Springfield and replaced with an ENERGY STAR unit with louvered sides:

$$\begin{aligned}\Delta\text{kWh for remaining life of existing unit (1}^{\text{st}} \text{ 4 years)} &= (319 * 9000 * (1/(7.7/1.01) - 1/11.4))/1000 \\ &= 124.7 \text{ kWh}\end{aligned}$$

$$\begin{aligned}\Delta\text{kWh for remaining measure life (next 8 years)} &= (319 * 9000 * (1/10.9 - 1/11.4))/1000 \\ &= 11.6 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

Time of Sale: $\Delta\text{kW} = \text{Btu/H} * ((1/(\text{CEER}_{\text{base}} * 1.01) - 1/(\text{CEER}_{\text{ee}} * 1.01)))/1000) * \text{CF}$

Early Replacement: $\Delta\text{kW} = \text{Btu/H} * ((1/\text{EER}_{\text{exist}} - 1/(\text{CEER}_{\text{ee}} * 1.01)))/1000) * \text{CF}$

Where:

CF = Summer Peak Coincidence Factor for measure
= 0.3⁹⁴

1.01 = Factor to convert CEER to EER (CEER includes standby and off power consumption)⁹⁵.
Other variable as defined above

⁹⁴ Consistent with coincidence factors found in: RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008

⁹⁵ Since the new CEER rating includes standby and off power consumption, for peak calculations it is more appropriate to apply the EER rating, but it appears as though new units will only be rated with a CEER rating. Version 3.0 of the ENERGY STAR specification provided equivalent EER and CEER ratings and for the most popular size band the EER rating is approximately 1% higher than the CEER. See 'ENERGY STAR Version 3.0 Room Air Conditioners Program Requirements'.

Time of Sale:

For example for an 8,500 Btu/H capacity unit, with louvered sides, for an unknown location:

$$\begin{aligned}\Delta kW_{CEE \text{ TIER } 1} &= (8500 * (1/(10.9 * 1.01) - 1/(11.4 * 1.01))) / 1000 * 0.3 \\ &= 0.010 \text{ kW}\end{aligned}$$

Early Replacement:

A 7.7 EER, 9000Btu/h unit is removed from a home in Springfield and replaced with an ENERGY STAR unit with louvered sides:

$$\begin{aligned}\Delta kW \text{ for remaining life of existing unit (1}^{st} \text{ 4 years)} &= (9000 * (1/7.7 - 1/(11.4 * 1.01)))/1000 * 0.3 \\ &= 0.12 \text{ kW}\end{aligned}$$

$$\begin{aligned}\Delta kW \text{ for remaining measure life (next 8 years)} &= (9000 * (1/(10.9 * 1.01) - 1/(11.4 * 1.01)))/1000 * 0.3 \\ &= 0.011 \text{ kW}\end{aligned}$$

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-APL-ESRA-V06-180101

REVIEW DEADLINE: 1/1/2021

5.1.8 Refrigerator and Freezer Recycling

DESCRIPTION

This measure describes savings from the retirement and recycling of inefficient but operational refrigerators and freezers. Savings are provided based on a 2013 workpaper provided by Cadmus that used data from a 2012 ComEd metering study and metering data from a Michigan study, to develop a regression equation that uses key inputs describing the retired unit. The savings are equivalent to the Unit Energy Consumption of the retired unit and should be claimed for the assumed remaining useful life of that unit. A part use factor is applied to account for those secondary units that are not in use throughout the entire year. The reader should note that the regression algorithm is designed to provide an accurate portrayal of savings for the population as a whole and includes those parameters that have a significant effect on the consumption. The precision of savings for individual units will vary.

The Net to Gross factor applied to these units should incorporate adjustments that account for:

- Those participants who would have removed the unit from the grid anyway (e.g. customers replacing their refrigerator via a big box store and using the pick-up option, customers taking their unit to the landfill or recycling station);
- Those participants who decided, based on the incentive provided by the Appliance Recycling program alone, to replace their existing inefficient unit with a new unit. This segment of participants is expected to be very small and documentation of their intentions will be gathered via telephone surveys (i.e., primary data sources). For such customers, the consumption of the new unit should be subtracted from the retired unit consumption and savings claimed for the remaining life of the existing unit. Note that participants who were already planning to replace their unit, and the incentive just ensured that the retired unit was recycled and not placed on the secondary market, should not be included in this adjustment.

This measure was developed to be applicable to the following program types: ERET.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

N/A

DEFINITION OF BASELINE EQUIPMENT

The existing inefficient unit must be operational and have a capacity of between 10 and 30 cubic feet.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The estimated remaining useful life of the recycling units is 8 years ⁹⁶.

DEEMED MEASURE COST

Measure cost includes the customer's value placed on their lost amenity, any customer transaction costs, and the cost of pickup and recycling of the refrigerator/freezer and should be based on actual costs of running the program. The payment (bounty) a Program Administrator makes to the customer serves as a proxy for the value the customer places on their lost amenity and any customer transaction costs. If unknown assume \$170⁹⁷ per unit.

⁹⁶ KEMA "Residential refrigerator recycling ninth year retention study", 2004

⁹⁷ The \$170 default assumption is based on \$120 cost of pickup and recycling per unit and \$50 proxy for customer transaction costs and value customer places on their lost amenity. \$120 is cost of pickup and recycling based on similar Efficiency Vermont program. \$50 is bounty, based on Ameren and ComEd program offerings as of 7/27/15.

LOADSHAPE

Loadshape R05 - Residential Refrigerator

COINCIDENCE FACTOR

The coincidence factor is assumed to be 0.00012.

Algorithm

CALCULATION OF SAVINGS**ENERGY SAVINGS⁹⁸**

Refrigerators:

Energy savings for refrigerators are based upon a linear regression model using the following coefficients⁹⁹:

Independent Variable Description	Estimate Coefficient
Intercept	83.324
Age (years)	3.678
Pre-1990 (=1 if manufactured pre-1990)	485.037
Size (cubic feet)	27.149
Dummy: Side-by-Side (= 1 if side-by-side)	406.779
Dummy: Primary Usage Type (in absence of the program) (= 1 if primary unit)	161.857
Interaction: Located in Unconditioned Space x CDD/365.25	15.366
Interaction: Located in Unconditioned Space x HDD/365.25	-11.067

$$\Delta \text{kWh} = [83.32 + (\text{Age} * 3.68) + (\text{Pre-1990} * 485.04) + (\text{Size} * 27.15) + (\text{Side-by-side} * 406.78) + (\text{Proportion of Primary Appliances} * 161.86) + (\text{CDD}/365.25 * \text{unconditioned} * 15.37) + (\text{HDD}/365.25 * \text{unconditioned} * -11.07)] * \text{Part Use Factor}$$

Where:

Age	= Age of retired unit
Pre-1990	= Pre-1990 dummy (=1 if manufactured pre-1990, else 0)
Size	= Capacity (cubic feet) of retired unit
Side-by-side	= Side-by-side dummy (= 1 if side-by-side, else 0)
Primary Usage	= Primary Usage Type (in absence of the program) dummy (= 1 if Primary, else 0)
Interaction: Located in Unconditioned Space x CDD/365.25	

⁹⁸ Based on the specified regression, a small number of units may have negative energy and demand consumption. These are a function of the unit size and age, and should comprise a very small fraction of the population. While on an individual basis this result is counterintuitive it is important that these negative results remain such that as a population the average savings is appropriate.

⁹⁹ Energy savings are based on an average 30-year TMY temperature of 51.1 degrees. Coefficients provided in July 30, 2014 memo from Cadmus: "Appliance Recycling Update no single door July 30 2014".

(=1 * CDD/365.25 if in unconditioned space)

CDD = Cooling Degree Days

= Dependent on location¹⁰⁰:

Climate Zone (City based upon)	CDD 65	CDD/365.25
1 (Rockford)	820	2.25
2 (Chicago)	842	2.31
3 (Springfield)	1,108	3.03
4 (Belleville)	1,570	4.30
5 (Marion)	1,370	3.75

Interaction: Located in Unconditioned Space x HDD/365.25

(=1 * HDD/365.25 if in unconditioned space)

HDD = Heating Degree Days

= Dependent on location:¹⁰¹

Climate Zone (City based upon)	HDD 65	HDD/365.25
1 (Rockford)	6,569	17.98
2 (Chicago)	6,339	17.36
3 (Springfield)	5,497	15.05
4 (Belleville)	4,379	11.99
5 (Marion)	4,476	12.25

Part Use Factor = To account for those units that are not running throughout the entire year. The most recent part-use factor participant survey results available at the start of the current program year shall be used¹⁰². For illustration purposes, this example uses 0.93.¹⁰³

For example, the program averages for AIC's ARP in PY4 produce the following equation:

$$\begin{aligned}
 \Delta \text{kWh} &= [83.32 + (22.81 * 3.68) + (0.45 * 485.04) + (18.82 * 27.15) + (0.17 * 406.78) \\
 &\quad + (0.34 * 161.86) + (1.29 * 15.37) + (6.49 * -11.07)] * 0.93 \\
 &= 969 * 0.93 \\
 &= 900.9 \text{ kWh}
 \end{aligned}$$

Freezers:

Energy savings for freezers are based upon a linear regression model using the following

¹⁰⁰ National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 65°F.

¹⁰¹ National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 65°F.

¹⁰² For example, the part-use factor that shall be applied to the current program year t (PYt) for savings verification purposes should be determined through the PYt-2 participant surveys conducted in the respective utility's service territory, if available. If an evaluation was not performed in PYt-2 the latest available evaluation should be used.

¹⁰³ Most recent refrigerator part-use factor from Ameren Illinois PY5 evaluation.

coefficients¹⁰⁴:

Independent Variable Description	Estimate Coefficient
Intercept	132.122
Age (years)	12.130
Pre-1990 (=1 if manufactured pre-1990)	156.181
Size (cubic feet)	31.839
Chest Freezer Configuration (=1 if chest freezer)	-19.709
Interaction: Located in Unconditioned Space x CDD/365.25	9.778
Interaction: Located in Unconditioned Space x HDD/365.25	-12.755

$$\Delta \text{kWh} = [132.12 + (\text{Age} * 12.13) + (\text{Pre-1990} * 156.18) + (\text{Size} * 31.84) + (\text{Chest Freezer} * -19.71) + (\text{CDDs} * \text{unconditioned} * 9.78) + (\text{HDDs} * \text{unconditioned} * -12.75)] * \text{Part Use Factor}$$

Where:

Age = Age of retired unit

Pre-1990 = Pre-1990 dummy (=1 if manufactured pre-1990, else 0)

Size = Capacity (cubic feet) of retired unit

Chest Freezer = Chest Freezer dummy (= 1 if chest freezer, else 0)

Interaction: Located in Unconditioned Space x CDD/365.25

(=1 * CDD/365.25 if in unconditioned space)

CDD = Cooling Degree Days (see table above)

Interaction: Located in Unconditioned Space x HDD/365.25

(=1 * HDD/365.25 if in unconditioned space)

HDD = Heating Degree Days (see table above)

Part Use Factor = To account for those units that are not running throughout the entire year. The most recent part-use factor participant survey results available at the start of the current program year shall be used¹⁰⁵. For illustration purposes, the example uses 0.85.¹⁰⁶

The program averages for AIC's ARP PY4 program are used as an example.

$$\begin{aligned} \Delta \text{kWh} &= [132.12 + (26.92 * 12.13) + (0.6 * 156.18) + (15.9 * 31.84) + (0.48 * -19.71) \\ &\quad + (6.61 * 9.78) + (1.3 * -12.75)] * 0.825 \\ &= 977 * 0.825 \\ &= 905 \text{ kWh} \end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \text{kWh}/8766 * \text{CF}$$

¹⁰⁴ Energy savings are based on an average 30-year TMY temperature of 51.1 degrees. Coefficients provided in January 31, 2013 memo from Cadmus: "Appliance Recycling Update".

¹⁰⁵ For example, the part-use factor that shall be applied to the current program year t (PYt) for savings verification purposes should be determined through the PYt-2 participant surveys conducted in the respective utility's service territory, if available. If an evaluation was not performed in PYt-2 the latest available evaluation should be used.

¹⁰⁶ Most recent freezer part-use factor from Ameren Illinois Company PY5 evaluation.

Where:

kWh = Savings provided in algorithm above
CF = Coincident factor defined as summer kW/average kW
= 1.081 for Refrigerators
= 1.028 for Freezers¹⁰⁷

For example, the program averages for AIC's ARP in PY4 produce the following equation:

$$\begin{aligned}\Delta kW &= 806/8766 * 1.081 \\ &= 0.099 \text{ kW}\end{aligned}$$

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-APL-RFRC-V06-160601

REVIEW DEADLINE: 1/1/2022

¹⁰⁷ Cadmus memo, February 12, 2013; "Appliance Recycling Update"

5.1.9 Room Air Conditioner Recycling

DESCRIPTION

This measure describes the savings resulting from running a drop off service taking existing residential, inefficient Room Air Conditioner units from service, prior to their natural end of life. This measure assumes that though a percentage of these units will be replaced this is not captured in the savings algorithm since it is unlikely that the incentive made someone retire a unit that they weren't already planning to retire. The savings therefore relate to the unit being taken off the grid as opposed to entering the secondary market. The Net to Gross factor applied to these units should incorporate adjustments that account for those participants who would have removed the unit from the grid anyway.

This measure was developed to be applicable to the following program types: ERET.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

N/A. This measure relates to the retiring of an existing inefficient unit.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is the existing inefficient room air conditioning unit.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed remaining useful life of the existing room air conditioning unit being retired is 4 years¹⁰⁸.

DEEMED MEASURE COST

The actual implementation cost for recycling the existing unit should be used.

LOADSHAPE

Loadshape R08 - Residential Cooling

COINCIDENCE FACTOR

The coincidence factor for this measure is assumed to be 30%¹⁰⁹.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = ((\text{FLH}_{\text{RoomAC}} * \text{Btu/hr} * (1/\text{EER}_{\text{exist}}))/1000)$$

Where:

¹⁰⁸ A third of assumed measure life for Room AC.

¹⁰⁹ Consistent with coincidence factors found in: RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008

(http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RA_C.pdf)

FLH_{RoomAC} = Full Load Hours of room air conditioning unit
 = dependent on location¹¹⁰:

Climate Zone (City based upon)	FLH_{RoomAC}
1 (Rockford)	220
2 (Chicago)	210
3 (Springfield)	319
4 (Belleville)	428
5 (Marion)	374
Weighted Average ¹¹¹	248

Btu/H = Size of retired unit
 = Actual. If unknown assume 8500 Btu/hr ¹¹²
 EER_{exist} = Efficiency of existing unit
 = 7.7¹¹³

For example for an 8500 Btu/h unit in Springfield:

$$\Delta kWh = ((319 * 8500 * (1/7.7)) / 1000)$$

$$= 352 \text{ kWh}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = (Btu/hr * (1/EER_{exist}))/1000 * CF$$

Where:

CF = Summer Peak Coincidence Factor for measure
 = 0.3¹¹⁴

¹¹⁰ The average ratio of FLH for Room AC (provided in RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008:

http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RAC.pdf) to FLH for Central Cooling for the same location (provided by AHRI:

http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls) is 31%. This ratio is applied to those IL cities that have FLH for Central Cooling provided in the Energy Star calculator. For other cities this is extrapolated using the FLH assumptions VEIC have developed for Central AC. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

¹¹¹ Weighted based on number of residential occupied housing units in each zone.

¹¹² Based on maximum capacity average from the RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008

¹¹³ Based on Nexus Market Research Inc, RLW Analytics, December 2005; "Impact, Process, and Market Study of the Connecticut Appliance Retirement Program: Overall Report."

¹¹⁴ Consistent with coincidence factors found in:

RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008

(http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RAC.pdf)

For example an 8500 Btu/h unit:

$$\begin{aligned}\Delta kW &= (8500 * (1/7.7)) / 1000 * 0.3 \\ &= 0.33 \text{ kW}\end{aligned}$$

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-APL-RARC-V01-120601

REVIEW DEADLINE: 1/1/2019

5.1.10 ENERGY STAR Clothes Dryer

DESCRIPTION

This measure relates to the installation of a residential clothes dryer meeting the ENERGY STAR criteria. ENERGY STAR qualified clothes dryers save energy through a combination of more efficient drying and reduced runtime of the drying cycle. More efficient drying is achieved through increased insulation, modifying operating conditions such as air flow and/or heat input rate, improving air circulation through better drum design or booster fans, and improving efficiency of motors. Reducing the runtime of dryers through automatic termination by temperature and moisture sensors is believed to have the greatest potential for reducing energy use in clothes dryers¹¹⁵. ENERGY STAR provides criteria for both gas and electric clothes dryers.

This measure was developed to be applicable to the following program types: TOS, NC. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

Clothes dryer must meet the ENERGY STAR criteria, as required by the program.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is a clothes dryer meeting the minimum federal requirements for units manufactured on or after January 1, 2015.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 14 years¹¹⁶.

DEEMED MEASURE COST

The incremental cost for an ENERGY STAR clothes dryer is assumed to be \$152¹¹⁷

LOADSHAPE

N/A

COINCIDENCE FACTOR

The coincidence factor for this measure is 3.8%¹¹⁸.

¹¹⁵ ENERGY STAR Market & Industry Scoping Report. Residential Clothes Dryers. Table 8. November 2011.

http://www.energystar.gov/ia/products/downloads/ENERGY_STAR_Scoping_Report_Residential_Clothes_Dryers.pdf

¹¹⁶ Based on an average estimated range of 12-16 years. ENERGY STAR Market & Industry Scoping Report. Residential Clothes Dryers. November 2011.

http://www.energystar.gov/ia/products/downloads/ENERGY_STAR_Scoping_Report_Residential_Clothes_Dryers.pdf

¹¹⁷ Based on the difference in installed cost for an efficient dryer (\$716) and standard dryer (\$564).

<http://www.aceee.org/files/proceedings/2012/data/papers/0193-000286.pdf>

¹¹⁸ Based on coincidence factor of 3.8% for clothes washers

Algorithm

CALCULATION OF ENERGY SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = (\text{Load}/\text{CEF}_{\text{base}} - \text{Load}/\text{CEF}_{\text{eff}}) * \text{Ncycles} * \% \text{Electric}$$

Where:

Load = The average total weight (lbs) of clothes per drying cycle. If dryer size is unknown, assume standard.

Dryer Size	Load (lbs) ¹¹⁹
Standard	8.45
Compact	3

CEF_{base} = Combined energy factor (CEF) (lbs/kWh) of the baseline unit is based on existing federal standards energy factor and adjusted to CEF as performed in the ENERGY STAR analysis¹²⁰. If product class unknown, assume electric, standard.

Product Class	CEF (lbs/kWh)
Vented Electric, Standard ($\geq 4.4 \text{ ft}^3$)	3.11
Vented Electric, Compact (120V) ($< 4.4 \text{ ft}^3$)	3.01
Vented Electric, Compact (240V) ($< 4.4 \text{ ft}^3$)	2.73
Ventless Electric, Compact (240V) ($< 4.4 \text{ ft}^3$)	2.13
Vented Gas	2.84 ¹²¹

CEF_{eff} = CEF (lbs/kWh) of the ENERGY STAR unit based on ENERGY STAR requirements.¹²² If product class unknown, assume electric, standard.

Product Class	CEF (lbs/kWh)
Vented or Ventless Electric, Standard ($\geq 4.4 \text{ ft}^3$)	3.93
Vented or Ventless Electric, Compact (120V) ($< 4.4 \text{ ft}^3$)	3.80
Vented Electric, Compact (240V) ($< 4.4 \text{ ft}^3$)	3.45
Ventless Electric, Compact (240V) ($< 4.4 \text{ ft}^3$)	2.68
Vented Gas	3.48 ¹²³

Ncycles = Number of dryer cycles per year. Use actual data if available. If unknown, use 283 cycles per year.¹²⁴

¹¹⁹ Based on ENERGY STAR test procedures. https://www.energystar.gov/index.cfm?c=clothesdry.pr_crit_clothes_dryers

¹²⁰ ENERGY STAR Draft 2 Version 1.0 Clothes Dryers Data and Analysis

¹²¹ Federal standards report CEF for gas clothes dryers in terms of lbs/kWh. To determine gas savings, this number is later converted to therms.

¹²² ENERGY STAR Clothes Dryers Key Product Criteria.

https://www.energystar.gov/index.cfm?c=clothesdry.pr_crit_clothes_dryers

¹²³ Federal standards report CEF for gas clothes dryers in terms of lbs/kWh. To determine gas savings, this number is later converted to therms.

¹²⁴ Appendix D to Subpart B of Part 430 – Uniform Test Method for Measuring the Energy Consumption of Dryers.

%Electric = The percent of overall savings coming from electricity
 = 100% for electric dryers, 16% for gas dryers¹²⁵

EXAMPLE

Time of Sale: For example, a standard, vented, electric clothes dryer:

$$\Delta \text{kWh} = ((8.45/3.11 - 8.45/3.93) * 283 * 100\%)$$

$$= 160 \text{ kWh}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \Delta \text{kWh}/\text{Hours} * \text{CF}$$

Where:

ΔkWh = Energy Savings as calculated above

Hours = Annual run hours of clothes dryer. Use actual data if available. If unknown, use 283 hours per year.¹²⁶

CF = Summer Peak Coincidence Factor for measure
 = 3.8%¹²⁷

EXAMPLE

Time of Sale: For example, a standard, vented, electric clothes dryer:

$$\Delta \text{kW} = 160/283 * 3.8\%$$

$$= 0.0215 \text{ kW}$$

NATURAL GAS SAVINGS

Natural gas savings only apply to ENERGY STAR vented gas clothes dryers.

$$\Delta \text{Therm} = (\text{Load}/\text{EF}_{\text{base}} - \text{Load}/\text{CE}_{\text{eff}}) * \text{Ncycles} * \text{Therm_convert} * \% \text{Gas}$$

Where:

Therm_convert = Conversion factor from kWh to Therm
 = 0.03413

%Gas = Percent of overall savings coming from gas
 = 0% for electric units and 84% for gas units¹²⁸

¹²⁵ %Electric accounts for the fact that some of the savings on gas dryers comes from electricity (motors, controls, etc). 16% was determined using a ratio of the electric to total savings from gas dryers given by ENERGY STAR Draft 2 Version 1.0 Clothes Dryers Data and Analysis.

¹²⁶ ENERGY STAR qualified dryers have a maximum test cycle time of 80 minutes. Assume one hour per dryer cycle.

¹²⁷ Based on coincidence factor of 3.8% for clothes washers.

¹²⁸ %Gas accounts for the fact that some of the savings on gas dryers comes from electricity (motors, controls, etc). 84% was determined using a ratio of the gas to total savings from gas dryers given by ENERGY STAR Draft 2 Version 1.0 Clothes Dryers Data and Analysis.

EXAMPLE

Time of Sale: For example, a standard, vented, gas clothes dryer:

$$\begin{aligned}\Delta\text{Therm} &= (8.45/2.84 - 8.45/3.48) * 283 * 0.03413 * 0.84 \\ &= 4.44 \text{ therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-APL-ESDR-V01-150601

REVIEW DEADLINE: 1/1/2021

5.1.11 ENERGY STAR Water Coolers

DESCRIPTION

Water coolers are a home appliance that offer consumers the ability to enjoy hot and/or cold water on demand. This measure is the characterization of the purchasing and use of an ENERGY STAR certified water cooler in place of a conventional water cooler.

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The high efficiency equipment is an ENERGY STAR certified water cooler meeting the ENERGY STAR 2.0 efficiency criteria.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is a standard or conventional, non-ENERGY STAR certified water cooler.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The estimated useful life for a water cooler is 10 years¹²⁹.

DEEMED MEASURE COST

The incremental cost for this measure is estimated at \$17¹³⁰.

LOADSHAPE

Loadshape C53: Flat

COINCIDENCE FACTOR

The summer peak coincidence factor is assumed to be 1.0.

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = (\text{kWh}_{\text{base}} - \text{kWh}_{\text{ee}}) * \text{Days}$$

Where:

kWh_{base} = Daily energy use (kWh/day) for baseline water cooler¹³¹

Type of Water Cooler	kWh_{base}
Hot and Cold Water – Storage	1.090
Hot and Cold Water – On Demand	0.330

¹²⁹ Savings Calculator for ENERGY STAR Certified Water Coolers, last updated 2009.

¹³⁰ Ameren Missouri PY3 Evaluation Report.

¹³¹ Savings Calculator for ENERGY STAR Certified Water Coolers, last updated 2009.

Type of Water Cooler	kWh _{base}
Cold Water Only	0.290

kWh_{ee} = Daily energy use (kWh/day) for ENERGY STAR water cooler¹³²

Type of Water Cooler	kWh _{ee}
Hot and Cold Water – Storage	0.747
Hot and Cold Water – On Demand	0.170
Cold Water Only	0.157

Days = Number of days per year that the water cooler is in use
= 365.25 days¹³³

Energy Savings:

Type of Water Cooler	ΔkWh
Hot and Cold Water – Storage	125.4
Hot and Cold Water – On Demand	58.4
Cold Water Only	48.7

DEMAND SAVINGS

$$\Delta kW = \Delta kWh / \text{Hours} * CF$$

Where:

Hours = Number of hours per year water cooler is in use
= 8766 hours¹³⁴

CF = Summer Peak Coincidence Factor for measure
= 1.0

Demand Savings:

Type of Water Cooler	ΔkW
Hot and Cold Water - Storage	0.0143
Hot and Cold Water – On Demand	0.0067
Cold Water Only	0.0056

NATURAL GAS SAVINGS

N/A

WATER AND OTHER NON-ENERGY IMPACT DESCRIPTIONS AND CALCULATION

N/A

¹³² Average kWh/day for from the ENERGY STAR efficient product database.

¹³³ Savings Calculator for ENERGY STAR Certified Water Coolers, last updated 2009.

¹³⁴ Assumed 365 days per year and 24 hours per day as utilized in daily energy consumption from ENERGY STAR Program Requirements Product Specification for Water Coolers Test Method.

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-APL-WTCL-V01-180101

REVIEW DEADLINE: 1/1/2024

5.2 Consumer Electronics End Use

5.2.1 Advanced Power Strip – Tier 1

DESCRIPTION

This measure relates to Advanced Power Strips – Tier 1 which are multi-plug surge protector power strips with the ability to automatically disconnect specific connected loads depending upon the power draw of a control load, also plugged into the strip. Power is disconnected from the switched (controlled) outlets when the control load power draw is reduced below a certain adjustable threshold, thus turning off the appliances plugged into the switched outlets. By disconnecting the standby load of the controlled devices, the overall load of a centralized group of equipment (i.e. entertainment centers and home office) can be reduced. Uncontrolled outlets are also provided that are not affected by the control device and so are always providing power to any device plugged into it. This measure characterization provides savings for a 5-plug strip and a 7-plug strip.

This measure was developed to be applicable to the following program types: TOS, NC, DI, KITS.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient case is the use of a 5 or 7-plug advanced power strip.

DEFINITION OF BASELINE EQUIPMENT

For time of sale or new construction applications, the assumed baseline is a standard power strip that does not control connected loads.

For direct install and kits, the baseline is the existing equipment utilized in the home.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of the advanced power strip is 7 years¹³⁵.

DEEMED MEASURE COST

For time of sale or new construction the incremental cost of an advanced Tier 1 power strip over a standard power strip with surge protection is assumed to be \$10¹³⁶.

For direct install the actual full install cost (including labor) and for kits the full equipment cost should be used.

LOADSHAPE

Loadshape R13 - Residential Standby Losses – Entertainment

Loadshape R14 - Residential Standby Losses - Home Office

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 80%¹³⁷.

¹³⁵ This is a consistent assumption with 5.2.2 Advanced Power Strip – Tier 2.

¹³⁶ Price survey performed by Illume Advising LLC for IL TRM workpaper, see “Current Surge Protector Costs and Comparison 7-2016” spreadsheet.

¹³⁷ Efficiency Vermont 2016 TRM coincidence factor for advanced power strip measure –in the absence of empirical evaluation data, this was based on assumptions of the typical run pattern for televisions and computers in homes.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = \text{kWh} * \text{ISR}$$

Where:

$$\begin{aligned} \text{kWh} &= \text{Assumed annual kWh savings per unit} \\ &= 56.5 \text{ kWh for 5-plug units or } 103 \text{ kWh for 7-plug units}^{138} \end{aligned}$$

$$\text{ISR} = \text{In Service Rate, dependent on delivery mechanism}$$

Delivery Mechanism	ISR
Energy Efficiency Kit	69% ¹³⁹
All other delivery mechanisms	100%

Using assumptions above:

# Plugs	Delivery Mechanism	ΔkWh
5- plug	Energy Efficiency Kit	39.0
	All other delivery mechanisms	56.5
7-plug	Energy Efficiency Kit	71.1
	All other delivery mechanisms	103.0
Unknown ¹⁴⁰	Energy Efficiency Kit	55.0
	All other delivery mechanisms	80.0

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \Delta \text{kWh} / \text{Hours} * \text{CF}$$

Where:

$$\begin{aligned} \text{Hours} &= \text{Annual number of hours during which the controlled standby loads are turned off by the Tier 1 Advanced power Strip.} \\ &= 7,129^{141} \end{aligned}$$

¹³⁸ NYSERDA Measure Characterization for Advanced Power Strips. Study based on review of: Smart Strip Electrical Savings and Usability, Power Smart Engineering, October 27, 2008.

Final Field Research Report, Ecos Consulting, October 31, 2006. Prepared for California Energy Commission's PIER Program. Developing and Testing Low Power Mode Measurement Methods, Lawrence Berkeley National Laboratory (LBNL), September 2004. Prepared for California Energy Commission's Public Interest Energy Research (PIER) Program.

2005 Intrusive Residential Standby Survey Report, Energy Efficient Strategies, March, 2006.

Smart Strip Portfolio of the Future, Navigant Consulting for San Diego G&E, March 31, 2009.

"Smart strip" in this context refers to the category of Advanced Power Strips, does not specifically signify Smart Strip® from BITS Limited, and was used without permission. Smart Strip® is a registered trademark of BITS Smart Strip, LLC.

¹³⁹ Average of Ameren Missouri, Potomac Edison, and PPL Electric ISR for smart strips in kits.

Cadmus, "Ameren Missouri RebateSavers Impact and Process Evaluation: Program Year 2013" p. 75.

Cadmus, "Process Evaluation Report, PPL Electric EE&C Plan, Program Year Five." p. 94

"Smart strip" in this context refers to the category of Advanced Power Strips, does not specifically signify Smart Strip® from BITS Limited, and was used without permission. Smart Strip® is a registered trademark of BITS Smart Strip, LLC.

¹⁴⁰ Calculated as average of 5 and 7 plug savings assumptions.

¹⁴¹ Average of hours for controlled TV and computer from; NYSERDA Measure Characterization for Advanced Power Strips

CF = Summer Peak Coincidence Factor for measure
= 0.8¹⁴²

# Plugs	Delivery Mechanism	ΔkW
5- plug	Energy Efficiency Kit	0.0044
	All other delivery mechanisms	0.0063
7-plug	Energy Efficiency Kit	0.0080
	All other delivery mechanisms	0.0116
Unknown ¹⁴³	Energy Efficiency Kit	0.0062
	All other delivery mechanisms	0.0090

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-CEL-SSTR-V03-180101

REVIEW DEADLINE: 1/1/2021

¹⁴² Efficiency Vermont 2016 TRM coincidence factor for advanced power strip measure –in the absence of empirical evaluation data, this was based on assumptions of the typical run pattern for televisions and computers in homes.

¹⁴³ Calculated as average of 5 and 7 plug savings assumptions.

5.2.2 Tier 2 Advanced Power Strips (APS) – Residential Audio Visual

DESCRIPTION

This measure relates to the installation of a Tier 2 Advanced Power Strip / surge protector for household audio visual environments (Tier 2 AV APS). Tier 2 AV APS are multi-plug power strips that remove power from audio visual equipment through intelligent control and monitoring strategies.

By utilizing advanced control strategies such as a countdown timer, external sensors (e.g. of infra-red remote usage and/or occupancy sensors, true RMS (Root Mean Square) power sensing; both active power loads and standby power loads of controlled devices are managed by Tier 2 AV APS devices¹⁴⁴. Monitoring and controlling both active and standby power loads of controlled devices will reduce the overall load of a centralized group of electrical equipment (i.e. the home entertainment center). This more intelligent sensing and control process has been demonstrated to deliver increased energy savings and demand reduction compared with 'Tier 1 Advanced Power Strips'.

The Tier 2 APS market is a relatively new and developing one. With several new Tier 2 APS products coming to market, it is important that energy savings are clearly demonstrated through independent field trials. The IL Technical Advisory Committee have developed a protocol whereby product manufacturers must submit independent field trial evidence of the Energy Reduction Percentage of their particular product either to the TRM Administrator for consideration during the TRM update process (August – December), or engage with a Program Administrator's independent evaluation team to review at other times. The product will be assigned a Product Class (A-H) corresponding to the proven savings and all products in a class will claim consistent savings. The IL TRM Administrator will maintain a list of eligible product and class on the IL TRM Sharepoint site. If a mid-year review has taken place, supporting information should be posted on the Sharepoint site such that other program administrators can review.

Due to the inherent variance day to day and week to week for hours of use of AV systems, it is critical that field trial studies effectively address the variability in usage patterns. There is significant discussion in the EM&V and academic domain on the optimal methodology for controlling for these factors and in submitting evidence of energy savings, it is critical that it is demonstrated that these issues are adequately addressed.

This measure was developed to be applicable to the following program types: DI. If applied to other program delivery types, the installation characteristics including the number of AV devices under control and an appropriate in service rate should be verified through evaluation.

Current evaluation is limited to Direct Install applications. Through a Direct Install program it can be assured that the APS is appropriately set up and the customer is knowledgeable about its function and benefit. It is encouraged that additional implementation strategies are evaluated to provide an indication of whether the units are appropriately set up, used with AV equipment and that the customer is knowledgeable about its function and benefit. This will then facilitate a basis for broadening out the deployment methods of the APS technology category beyond Direct Install.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient case is the use of a Tier 2 AV APS in a residential AV (home entertainment) environment that includes control of at least 2 AV devices with one being the television¹⁴⁵.

Only Tier 2 AV APS products that have independent demonstrated energy savings via field trials are eligible.

The minimum product specifications for Tier 2 AV APS are:

¹⁴⁴ Tier 2 AV APS identify when people are not engaged with their AV equipment and then remove power, for example a TV and its peripheral devices that are unintentionally left on when a person leaves the house or for instance where someone falls asleep while watching television.

¹⁴⁵ Given this requirement, an AV environment consisting of a television and DVD player or a TV and home theater would be eligible for a Tier 2 AV APS installation.

Safety & longevity

- Product and installation instructions shall comply with 2012 International Fire Code and 2000 NFPA 101 Life Safety Code (IL Fire Code).
- Third party tested to all applicable UL Standards.
- Contains a resettable circuit breaker
- Incorporates power switching electromechanical relays rated for 100,000 switching cycles at full 15 amp load (equivalent to more than 10 years of use).

Energy efficiency functionality

- Calculates real power as the time average of the instantaneous power, where instantaneous power is the product of instantaneous voltage and current.
- Delivers a warning when the countdown timer begins before an active power down event and maintains the warning until countdown is concluded or reset by use of the remote or other specified signal
- Uses an automatically adjustable power switching threshold.

DEFINITION OF BASELINE EQUIPMENT

The assumed baseline equipment is the existing equipment being used in the home (e.g. a standard power strip or wall socket) that does not control loads of connected AV equipment.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The default deemed lifetime value for Tier 2 AV APS is assumed to be 7 years¹⁴⁶.

DEEMED MEASURE COST

Direct Installation: The actual installed cost (including labor) of the new Tier 2 AV APS equipment should be used.

LOADSHAPE

Loadshape R13 - Residential Standby Losses – Entertainment

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 80%¹⁴⁷

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = \text{ERP} * \text{BaselineEnergy}_{\text{AV}} * \text{ISR}$$

Where:

¹⁴⁶ There is little evaluation to base a lifetime estimate upon. Based on review of assumptions from other jurisdictions and the relative treatment of In Service Rates and persistence, an estimate of 7 years was agreed by the Technical Advisory Committee, but further evaluation is recommended.

¹⁴⁷ In the absence of empirical evaluation data, this was based on assumptions of the typical run pattern for televisions and computers in homes.

ERP = Energy Reduction Percentage of qualifying Tier2 AV APS product range as provided below. See reference documents for Product Classification memo.

BaselineEnergy_{AV} = 432 kWh¹⁴⁸

Product Class	Field trial ERP range	ERP used	BaselineEnergy _{AV} (kWh)
A	55 – 60%	55%	238
B	50 – 54%	50%	216
C	45 – 49%	45%	194
D	40 – 44%	40%	173
E	35 – 39%	35%	151
F	30 – 34%	30%	130
G	25 – 29%	25%	108
H	20 – 24%	20%	86

ISR = In Service Rate. See reference documents for Product Classification memo.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh / \text{Hours} * CF$$

Where:

ΔkWh = Energy savings as calculated above

Hours = Annual number of hours during which the APS provides savings.
= 4,380¹⁴⁹

CF = Summer Peak Coincidence Factor for measure
= 0.8¹⁵⁰

NATURAL GAS SAVINGS

N/A¹⁵¹

WATER AND OTHER NON-ENERGY IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

¹⁴⁸ AESC, Inc, "Energy Savings of Tier 2 Advanced Power Strips in Residential AC Systems", p28. Note that this load represents the average *controlled* AV devices only and will likely be lower than total AC usage.

¹⁴⁹ This is estimate based on assumption that approximately half of savings are during active hours (supported by AESC study) (assumed to be 5.3 hrs/day, 1936 per year (NYSERDA 2011. "Advanced Power Strip Research Report")) and half during standby hours (8760-1936 = 6824 hours). The weighted average is 4380.

¹⁵⁰ In the absence of empirical evaluation data, this was based on assumptions of the typical run pattern for televisions and computers in homes. This appears to be supported by the Average Weekday AV Demand Profile and Reduction charts in the AESC study (p33-34). These show that the average demand reduction is relatively flat.

¹⁵¹ Interactive effects of Tier 2 APS on space conditioning loads has not yet been adequately studied.

MEASURE CODE: RS-CEL-APS2-V02-180101

REVIEW DEADLINE: 1/1/2019

5.3 HVAC End Use

5.3.1 Air Source Heat Pump

DESCRIPTION

A heat pump provides heating or cooling by moving heat between indoor and outdoor air.

This measure characterizes:

a) Time of Sale:

- a. The installation of a new residential sized ($\leq 65,000$ Btu/hr) air source heat pump that is more efficient than required by federal standards. This could relate to the replacement of an existing unit at the end of its useful life, or the installation of a new system in a new home.

b) Early Replacement:

The early removal of functioning electric heating and cooling (SEER 10 or under if present) systems from service, prior to its natural end of life, and replacement with a new high efficiency air source heat pump unit.

Early Replacement determination will be based on meeting the following conditions:

- The existing unit is operational when replaced, or
- The existing unit requires minor repairs ($< \$276$ per ton)¹⁵².
- All other conditions will be considered Time of Sale.

The Baseline SEER of the existing unit replaced:

- If the SEER of the existing unit is known and ≤ 10 , the Baseline SEER is the actual SEER value of the unit replaced. If the SEER is > 10 , the Baseline SEER = 14.
- If the SEER of the existing unit is unknown use assumptions in variable list below (SEER_exist and HSPF_exist).
- If the operational status or repair cost of the existing unit is unknown, use time of sale assumptions.

A weighted average early replacement rate is provided for use when the actual baseline early replacement rates are unknown.¹⁵³

Deemed Early Replacement Rates For ASHP

	Deemed Early Replacement Rate
Early Replacement Rate for ASHP participants	7%

This measure was developed to be applicable to the following program types: TOS, NC, EREP. If applied to other program types, the measure savings should be verified.

¹⁵² The Technical Advisory Committee agreed that if the cost of repair is less than 20% of the new baseline replacement cost it can be considered early replacement. Note the non-inflated cost is used as this would be a cost consideration in the program year.

¹⁵³ Based upon research from “Home Energy Efficiency Rebate Program GPY2 Evaluation Report” which outlines early replacement rates for both primary and secondary central air cooling (CAC) and residential furnaces. This is used as a reasonable proxy for ASHP installations since ASHP specific data is not available. Report presented to Nicor Gas Company February 27, 2014, available at <http://www.ilsag.info/evaluation-documents.html>.

DEFINITION OF EFFICIENT EQUIPMENT

A new residential sized ($\leq 65,000$ Btu/hr) air source heat pump with specifications to be determined by program.

DEFINITION OF BASELINE EQUIPMENT

A new residential sized ($\leq 65,000$ Btu/hr) air source heat pump meeting federal standards.

The baseline for the Time of Sale measure is based on the current Federal Standard efficiency level as of January 1st 2015; 14 SEER and 8.2HSPF.

The baseline for the early replacement measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 18 years.¹⁵⁴

Remaining life of existing ASHP/CAC equipment is assumed to be 6 years¹⁵⁵ and 18 years for electric resistance.

DEEMED MEASURE COST

Time of sale: The incremental capital cost for this measure is dependent on the efficiency of the new unit¹⁵⁶.

Efficiency (SEER)	Incremental Cost (\$/unit)
14.5	\$123
15	\$303
16	\$438
17	\$724
18	\$724

Early replacement: The full install cost for this measure is the actual cost of removing the existing unit and installing the new one. If this is unknown, assume the following (note these costs are per ton of unit capacity)¹⁵⁷:

Efficiency (SEER)	Full Retrofit Cost (including labor)
14.5	\$1,381 / ton + \$123
15	\$1,381 / ton + \$303
16	\$1,381 / ton + \$438
17	\$1,381 / ton + \$724
18	\$1,381 / ton + \$724

Assumed deferred cost (after 6 years) of replacing existing equipment with new baseline unit is assumed to be \$1,518 per ton of capacity¹⁵⁸. This cost should be discounted to present value using the nominal societal discount rate.

¹⁵⁴ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007, <http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf>

¹⁵⁵ Assumed to be one third of effective useful life

¹⁵⁶ Based on incremental cost results from Cadmus "HVAC Program: Incremental Cost Analysis Update", December 19, 2016.

¹⁵⁷ Baseline cost per ton derived from DEER 2008 Database Technology and Measure Cost Data (www.deeresources.com). See 'ASHP_Revised DEER Measure Cost Summary.xls' for calculation. Efficiency cost increment consistent with Cadmus study results.

¹⁵⁸ Ibid. \$1381 per ton inflated using rate of 1.91%.

LOADSHAPE

Loadshape R10 - Residential Electric Heating and Cooling

COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

CF _{SSP SF}	= Summer System Peak Coincidence Factor for Heat Pumps in single-family homes (during utility peak hour) = 72% ¹⁵⁹
CF _{PJM SF}	= PJM Summer Peak Coincidence Factor for Heat Pumps in single-family homes (average during PJM peak period) = 46.6% ¹⁶⁰
CF _{SSP, MF}	= Summer System Peak Coincidence Factor for Heat Pumps in multi-family homes (during system peak hour) = 67% ¹⁶¹
CF _{PJM, MF}	= PJM Summer Peak Coincidence Factor for Heat Pumps in multi-family homes (average during peak period) = 28.5%

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Time of sale:

$$\Delta \text{kWh} = ((\text{FLH}_{\text{cooling}} * \text{Capacity}_{\text{cooling}} * (1/\text{SEER}_{\text{base}} - 1/\text{SEER}_{\text{ee}})) / 1000) + ((\text{FLH}_{\text{heat}} * \text{Capacity}_{\text{heating}} * (1/\text{HSPF}_{\text{base}} - 1/\text{HSPF}_{\text{ee}})) / 1000)$$

Early replacement¹⁶²:

$$\begin{aligned} &\Delta \text{kWh for remaining life of existing unit (1st 6 years for replacing an ASHP, 18 years for replacing electric resistance):} \\ &= ((\text{FLH}_{\text{cooling}} * \text{Capacity}_{\text{cooling}} * (1/\text{SEER}_{\text{exist}} - 1/\text{SEER}_{\text{ee}})) / 1000) + ((\text{FLH}_{\text{heat}} * \end{aligned}$$

¹⁵⁹ Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

¹⁶⁰ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

¹⁶¹ Multifamily coincidence factors both from; *All-Electric Homes PY6 Metering Results: Multifamily HVAC Systems*, Cadmus, October 2015

¹⁶² The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input which would be the (new base to efficient savings)/(existing to efficient savings).

$$\text{Capacity_heating} * (1/\text{HSPF_exist} - 1/\text{HSFP_ee}) / 1000$$

ΔkWH for remaining measure life (next 12 years if replacing an ASHP):

$$= ((\text{FLH_cooling} * \text{Capacity_cooling} * (1/\text{SEER_base} - 1/\text{SEER_ee})) / 1000) + ((\text{FLH_heat} * \text{Capacity_heating} * (1/\text{HSPF_base} - 1/\text{HSFP_ee})) / 1000)$$

Where:

FLH_cooling = Full load hours of air conditioning

= dependent on location:

Climate Zone (City based upon)	FLH_cooling (single family) ¹⁶³	FLH_cooling (general multi family) ¹⁶⁴	FLH_cooling (weatherized multi family) ¹⁶⁵
1 (Rockford)	512	467	299
2 (Chicago)	570	506	324
3 (Springfield)	730	663	425
4 (Belleville)	1,035	940	603
5 (Marion)	903	820	526
Weighted Average ¹⁶⁶	629	564	362

Capacity_cooling = Cooling Capacity of Air Source Heat Pump (Btu/hr)

= Actual (1 ton = 12,000Btu/hr)

SEER_exist = Seasonal Energy Efficiency Ratio of existing cooling system (kBtu/kWh)

= Use actual SEER rating where it is possible to measure or reasonably estimate.

Existing Cooling System	SEER_exist ¹⁶⁷
Air Source Heat Pump	9.12
Central AC	8.60
No central cooling ¹⁶⁸	Make '1/SEER_exist' = 0

SEER_base = Seasonal Energy Efficiency Ratio of baseline Air Source Heat Pump (kBtu/kWh)

= 14 ¹⁶⁹

SEER_ee = Seasonal Energy Efficiency Ratio of efficient Air Source Heat Pump (kBtu/kWh)

¹⁶³ Full load hours for Chicago, Moline and Rockford are provided in “Final Evaluation Report: Central Air Conditioning Efficiency Services (CACES), 2010, Navigant Consulting”, p.33. An average FLH/Cooling Degree Day (from NCDC) ratio was calculated for these locations and applied to the CDD of the other locations in order to estimate FLH. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

¹⁶⁴ Ibid.

¹⁶⁵ *All-Electric Homes PY6 Metering Results: Multifamily HVAC Systems*, Cadmus, October 2015. The multifamily units within this study had undergone significant shell improvements (air sealing and insulation) and therefore this set of assumptions is only appropriate for units that have recently participated in a weatherization or other shell program. Note that the FLHcool where recalculated based on existing efficiencies consistent with the TRM rather than from the metering study.

¹⁶⁶ Weighted based on number of occupied residential housing units in each zone.

¹⁶⁷ Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

¹⁶⁸ If there is no central cooling in place but the incentive encourages installation of a new ASHP with cooling, the added cooling load should be subtracted from any heating benefit.

¹⁶⁹ Based on Minimum Federal Standard effective 1/1/2015;

<http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf>.

= Actual

FLH_heat = Full load hours of heating

= Dependent on location and home type:

Climate Zone (City based upon)	FLH_heat (single family and general multi family) ¹⁷⁰	FLH heat (weatherized multi family) ¹⁷¹
1 (Rockford)	1,969	748
2 (Chicago)	1,840	699
3 (Springfield)	1,754	667
4 (Belleville)	1,266	481
5 (Marion)	1,288	489
Weighted Average ¹⁷²	1,821	692

Capacity_heating = Heating Capacity of Air Source Heat Pump (Btu/hr)

= Actual (1 ton = 12,000Btu/hr)

HSPF_exist = Heating System Performance Factor¹⁷³ of existing heating system (kBtu/kWh)

= Use actual HSPF rating where it is possible to measure or reasonably estimate. If not available use:

Existing Heating System	HSPF_exist
Air Source Heat Pump	5.44 ¹⁷⁴
Electric Resistance	3.41 ¹⁷⁵

HSPF_base = Heating System Performance Factor of baseline Air Source Heat Pump (kBtu/kWh)

= 8.2¹⁷⁶

HSFP_ee = Heating System Performance Factor of efficient Air Source Heat Pump
(kBtu/kWh)

¹⁷⁰ Full load heating hours for heat pumps are provided for Rockford, Chicago and Springfield in the Energy Star Calculator. Estimates for the other locations were calculated based on the FLH to Heating Degree Day (from NCDC) ratio. VEIC consider Energy Star estimates to be high due to oversizing not being adequately addressed. Using average Illinois billing data (from <http://www.icc.illinois.gov/ags/consumereducation.aspx>) VEIC estimated the average gas heating load and used this to estimate the average home heating output (using 83% average gas heat efficiency). Dividing this by a typical 36,000 Btu/hr ASHP gives an estimate of average ASHP FLH_heat of 1821 hours. We used the ratio of this value to the average of the locations using the Energy Star data (1994 hours) to scale down the Energy Star estimates. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

¹⁷¹ All-Electric Homes PY6 Metering Results: Multifamily HVAC Systems, Cadmus, October 2015.

¹⁷² Weighted based on number of occupied residential housing units in each zone.

¹⁷³ HSPF ratings for Heat Pumps account for the seasonal average efficiency of the units and are based on testing within zone 4 which encompasses most of Illinois. Furthermore, a recent Cadmus/Opinion Dynamics metering study, "Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)", found no significant variance between metered performance and that presented in the TRM

¹⁷⁴ This is estimated based on finding the average HSPF/SEER ratio from the AHRI directory data (using the least efficient models – SEER 12 and SEER 13) – 0.596, and applying to the average nameplate SEER rating of all Early Replacement qualifying equipment in Ameren PY3-PY4. This estimation methodology appears to provide a result within 10% of actual HSPF.

¹⁷⁵ Electric resistance has a COP of 1.0 which equals 1/0.293 = 3.41 HSPF.

¹⁷⁶ Based on Minimum Federal Standard effective 1/1/2015;

<http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf>

= Actual

Time of Sale:

For example, a three ton, 15 SEER, 12EER, 9 HSPF Air Source Heat Pump installed in a single family home in Marion:

$$\begin{aligned}\Delta \text{kWh} &= ((903 * 36,000 * (1/14 - 1/15)) / 1000) + ((1,288 * 36,000 * (1/8.2 - 1/9)) / 1000) \\ &= 657 \text{ kWh}\end{aligned}$$

Early Replacement:

For example, a three ton, 15 SEER, 12EER, 9 HSPF Air Source Heat Pump replaces an existing working Air Source Heat Pump with unknown efficiency ratings in a single family home in Marion:

ΔkWh for remaining life of existing unit (1st 6 years):

$$\begin{aligned}&= ((903 * 36,000 * (1/9.12 - 1/15)) / 1000) + ((1,288 * 36,000 * (1/5.44 - 1/9)) / 1000) \\ &= 4769 \text{ kWh}\end{aligned}$$

ΔkWh for remaining measure life (next 12 years):

$$\begin{aligned}&= ((903 * 36,000 * (1/14 - 1/15)) / 1000) + ((1,288 * 36,000 * (1/8.2 - 1/9)) / 1000) \\ &= 657 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

Time of sale:

$$\Delta \text{kW} = (\text{Capacity_cooling} * (1/\text{EER_base} - 1/\text{EER_ee})) / 1000 * \text{CF}$$

Early replacement¹⁷⁷:

ΔkW for remaining life of existing unit (1st 6 years for replacing an ASHP, 18 years for replacing electric resistance):

$$= ((\text{Capacity_cooling} * (1/\text{EER}_{\text{exist}} - 1/\text{EER}_{\text{ee}})) / 1000 * \text{CF});$$

ΔkW for remaining measure life (next 12 years if replacing an ASHP):

$$= ((\text{Capacity_cooling} * (1/\text{EER}_{\text{base}} - 1/\text{EER}_{\text{ee}})) / 1000 * \text{CF})$$

Where:

$\text{EER}_{\text{exist}}$ = Energy Efficiency Ratio of existing cooling system (kBtu/hr / kW)

= Use actual EER rating where it is possible to measure or reasonably estimate. If EER unknown but SEER available convert using the equation:

$$\text{EER}_{\text{base}} = (-0.02 * \text{SEER}_{\text{exist}}^2) + (1.12 * \text{SEER}_{\text{exist}}) \quad ^{178}$$

If SEER or EER rating unavailable use:

¹⁷⁷ The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a “number of years to adjustment” and “savings adjustment” input which would be the (new base to efficient savings)/(existing to efficient savings).

¹⁷⁸ From Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder.

Existing Cooling System	EER_exist ¹⁷⁹
Air Source Heat Pump	8.55
Central AC	8.15
No central cooling ¹⁸⁰	Make '1/EER_exist' = 0

EER_base	= Energy Efficiency Ratio of baseline Air Source Heat Pump (kBtu/hr / kW) = 11.8 ¹⁸¹
EER_ee	= Energy Efficiency Ratio of efficient Air Source Heat Pump (kBtu/hr / kW) = Actual, If not provided convert SEER to EER using this formula: ¹⁸² = $(-0.02 * SEER_{ee}^2) + (1.12 * SEER_{ee})$
CF _{SSP SF}	= Summer System Peak Coincidence Factor for Heat Pumps in single-family homes (during system peak hour) = 72% ¹⁸³
CF _{PJM SF}	= PJM Summer Peak Coincidence Factor for Heat Pumps in single-family homes (average during peak period) = 46.6% ¹⁸⁴
CF _{SSP, MF}	= Summer System Peak Coincidence Factor for Heat Pumps in multi-family homes (during system peak hour) = 67% ¹⁸⁵
CF _{PJM, MF}	= PJM Summer Peak Coincidence Factor for Heat Pumps in multi-family homes (average during peak period) = 28.5% ³⁵

¹⁷⁹ Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

¹⁸⁰ If there is no central cooling in place but the incentive encourages installation of a new ASHP with cooling, the added cooling load should be subtracted from any heating benefit.

¹⁸¹ The Federal Standard does not include an EER requirement, so it is approximated with this formula: $(-0.02 * SEER^2) + (1.12 * SEER)$ Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder. Note this is appropriate for single speed units only.

¹⁸² Based on Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder. Note this is appropriate for single speed units only.

¹⁸³ Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

¹⁸⁴ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

¹⁸⁵ All-Electric Homes PY6 Metering Results: Multifamily HVAC Systems, Cadmus, October 2015

Time of Sale:

For example, a three ton, 15 SEER, 12EER, 9 HSPF Air Source Heat Pump installed in single-family home in Marion:

$$\begin{aligned}\Delta kW_{SSP} &= ((36,000 * (1/11.8 - 1/12)) / 1000) * 0.72 \\ &= 0.037 \text{ kW}\end{aligned}$$

$$\begin{aligned}\Delta kW_{PJM} &= ((36,000 * (1/11.8 - 1/12)) / 1000) * 0.466 \\ &= 0.024 \text{ kW}\end{aligned}$$

Early Replacement:

For example, a three ton, 15 SEER, 12EER, 9 HSPF Air Source Heat Pump replaces an existing working Air Source Heat Pump with unknown efficiency ratings in single-family home in Marion:

$$\begin{aligned}\Delta kW_{SSP} \text{ for remaining life of existing unit (1st 6 years):} \\ &= ((36,000 * (1/8.55 - 1/12)) / 1000) * 0.72 \\ &= 0.872 \text{ kW}\end{aligned}$$

$$\begin{aligned}\Delta kW_{SSP} \text{ for remaining measure life (next 12 years):} \\ &= ((36,000 * (1/11.8 - 1/12)) / 1000) * 0.72 \\ &= 0.037 \text{ kW}\end{aligned}$$

$$\begin{aligned}\Delta kW_{PJM} \text{ for remaining life of existing unit (1st 6 years):} \\ &= ((36,000 * (1/8.55 - 1/12)) / 1000) * 0.466 \\ &= 0.564 \text{ kW}\end{aligned}$$

$$\begin{aligned}\Delta kW_{PJM} \text{ for remaining measure life (next 12 years):} \\ &= ((36,000 * (1/11.8 - 1/12)) / 1000) * 0.466 \\ &= 0.024 \text{ kW}\end{aligned}$$

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-HVC-ASHP-V07-180101

REVIEW DEADLINE: 1/1/2021

5.3.2 Boiler Pipe Insulation

DESCRIPTION

This measure describes adding insulation to un-insulated boiler pipes in un-conditioned basements or crawlspaces.

This measure was developed to be applicable to the following program types: TOS, RNC, RF, DI.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient case is installing pipe wrap insulation to a length of boiler pipe.

DEFINITION OF BASELINE EQUIPMENT

The baseline is an un-insulated boiler pipe.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 15 years¹⁸⁶.

DEEMED MEASURE COST

The measure cost including material and installation is assumed to be \$3 per linear foot¹⁸⁷.

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS SAVINGS

$$\Delta \text{Therm} = (((1/R_{\text{exist}} * C_{\text{exist}}) - (1/R_{\text{new}} * C_{\text{new}})) * \text{FLH_heat} * L * \Delta T) / \eta_{\text{Boiler}} / 100,000$$

Where:

$$R_{\text{exist}} = \text{Pipe heat loss coefficient of uninsulated pipe (existing) [(hr-°F-ft²)/Btu]}$$

¹⁸⁶ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.
<http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf>

¹⁸⁷ Consistent with DEER 2008 Database Technology and Measure Cost Data (www.deeresources.com).

$$= 0.5^{188}$$

R_{new} = Pipe heat loss coefficient of insulated pipe (new) [(hr-°F-ft²)/Btu]

= Actual (0.5 + R value of insulation)

FLH_heat = Full load hours of heating

= Dependent on location¹⁸⁹:

Climate Zone (City based upon)	FLH_heat
1 (Rockford)	1,969
2 (Chicago)	1,840
3 (Springfield)	1,754
4 (Belleville)	1,266
5 (Marion)	1,288
Weighted Average ¹⁹⁰	1,821

L = Length of boiler pipe in unconditioned space covered by pipe wrap (ft)

= Actual

C_{exist} = Circumference of bare pipe (ft) (Diameter (in) * $\pi/12$)

= Actual (0.5" pipe = 0.131ft, 0.75" pipe = 0.196ft)

C_{new} = Circumference of pipe with insulation (ft) ([Diameter of pipe (in)] + ([Thickness of Insulation (in)]*2)) * $\pi/12$)

= Actual

ΔT = Average temperature difference between circulated heated water and unconditioned space air temperature (°F)¹⁹¹

Pipes in unconditioned basement:

Outdoor reset controls	ΔT (°F)
Boiler without reset control	110
Boiler with reset control	70

Pipes in crawl space:

¹⁸⁸ Assumption based on data obtained from the 3E Plus heat loss calculation software provided by the NAIMA (North American Insulation Manufacturer Association) and derived from Table 15 and Table 16 of 2009 ASHRAE Fundamentals Handbook, Chapter 23 Insulation for Mechanical Systems, page 23.17.

¹⁸⁹ Full load heating hours for heat pumps are provided for Rockford, Chicago and Springfield in the Energy Star Calculator. Estimates for the other locations were calculated based on the FLH to Heating Degree Day (from NCDC) ratio. VEIC consider Energy Star estimates to be high due to oversizing not being adequately addressed. Using average Illinois billing data (from <http://www.icc.illinois.gov/ags/consumereducation.aspx>) VEIC estimated the average gas heating load and used this to estimate the average home heating output (using 83% average gas heat efficiency). Dividing this by a typical 36,000 Btu/hr ASHP gives an estimate of average ASHP FLH_heat of 1821 hours. We used the ratio of this value to the average of the locations using the Energy Star data (1994 hours) to scale down the Energy Star estimates. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

¹⁹⁰ Weighted based on number of occupied residential housing units in each zone.

¹⁹¹ Assumes 160°F water temp for a boiler without reset control, 120°F for a boiler with reset control, and 50°F air temperature for pipes in unconditioned basements and the following average heating season outdoor temperatures as the air temperature in crawl spaces: Zone 1 – 33.1, Zone 2 – 34.4, Zone 3 – 37.7, Zone 4 – 40.0, Zone 5 – 39.8, Weighted Average – 35.3 (NCDC 1881-2010 Normals, average of monthly averages Nov – Apr for zones 1-3 and Nov-March for zones 4 and 5).

Climate Zone (City based upon)	ΔT (°F)	
	Boiler without reset control	Boiler with reset control
1 (Rockford)	127	87
2 (Chicago)	126	86
3 (Springfield)	122	82
4 (Belleville)	120	80
5 (Marion)	120	80
Weighted Average ¹⁹²	125	85

η_{Boiler} = Efficiency of boiler
= 0.819¹⁹³

For example, insulating 10 feet of 0.75" pipe with R-3 wrap (0.75" thickness) in a crawl space of a Marion home with a boiler without reset control:

$$\begin{aligned}\Delta \text{Therm} &= (((1/0.5 * 0.196) - (1/3.5 * 0.589)) * 10 * 120 * 1288) / 0.819 / 100,000 \\ &= 4.2 \text{ therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-HVC-PINS-V02-160601

REVIEW DEADLINE: 1/1/2022

¹⁹² Weighted based on number of occupied residential housing units in each zone.

¹⁹³ Average efficiency of boiler units found in Ameren PY3-PY4 data.

5.3.3 Central Air Conditioning

DESCRIPTION

This measure characterizes:

- a) Time of Sale:
 - a. The installation of a new residential sized ($\leq 65,000$ Btu/hr) Central Air Conditioning ducted split system meeting ENERGY STAR efficiency standards presented below. This could relate to the replacement of an existing unit at the end of its useful life, or the installation of a new system in a new home.

- b) Early Replacement:

Early Replacement determination will be based on meeting the following conditions:

- The existing unit is operational when replaced, or
- The existing unit requires minor repairs ($< \$190$ per ton)¹⁹⁴.
- All other conditions will be considered Time of Sale.

The Baseline SEER of the existing Central Air Conditioning unit replaced:

- If the SEER of the existing unit is known and ≤ 10 , the Baseline SEER is the actual SEER value of the unit replaced. If the SEER is > 10 , the Baseline SEER = 13.
- If the SEER of the existing unit is unknown, use assumptions in variable list below (SEER_exist).
- If the operational status or repair cost of the existing unit is unknown, use time of sale assumptions.

A weighted average early replacement rate is provided for use when the actual baseline early replacement rate is unknown¹⁹⁵.

Deemed Early Replacement Rates For CAC Units in Combined System Replacement (CSR) Projects

Replacement Scenario for the CAC Unit	Deemed Early Replacement Rate
Early Replacement Rate for a CAC unit when the CAC unit is the Primary unit in a CSR project	14%
Early Replacement Rate for a CAC unit when the CAC unit is the Secondary unit in a CSR project	40%

This measure was developed to be applicable to the following program types: TOS, NC, EREP. If applied to other program types, the measure savings should be verified.

¹⁹⁴ The Technical Advisory Committee agreed that if the cost of repair is less than 20% of the new baseline replacement cost it can be considered early replacement. Note the non-inflated cost is used as this would be a cost consideration in the program year.

¹⁹⁵ Based upon research from "Home Energy Efficiency Rebate Program GPY2 Evaluation Report" which outlines early replacement rates for both primary and secondary central air cooling (CAC) and residential furnaces. The unit (furnace or CAC unit) that initially caused the customer to contact a trade ally is defined as the "primary unit". The furnace or CAC unit that was also replaced but did not initially prompt the customer to contact a trade ally is defined as the "secondary unit". This evaluation used different criteria for early replacement due to the availability of data after the fact; cost of any repairs $< \$550$ and age of unit < 20 years. Report presented to Nicor Gas Company February 27, 2014, available at <http://www.ilsag.info/evaluation-documents.html>.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a ducted split central air conditioning unit meeting the minimum ENERGY STAR efficiency level standards; 14.5 SEER and 12 EER.

DEFINITION OF BASELINE EQUIPMENT

The baseline for the Time of Sale measure is based on the current Federal Standard efficiency level; 13 SEER and 11 EER.

The baseline for the early replacement measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and the new baseline as defined above¹⁹⁶ for the remainder of the measure life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 18 years¹⁹⁷.

Remaining life of existing equipment is assumed to be 6 years¹⁹⁸.

DEEMED MEASURE COST

Time of sale: The incremental capital cost for this measure is dependent on efficiency. Assumed incremental costs are provided below¹⁹⁹:

Efficiency Level (SEER)	Incremental Cost
14	\$0
15	\$108
16	\$221
17	\$620
18	\$620

Early replacement: The full install cost for this measure is the actual cost of removing the existing unit and installing the new one. If this is unknown, assume defaults below²⁰⁰.

Efficiency Level (SEER)	Full Retrofit Cost (including labor)
14	\$952 / ton + \$0
15	\$952 / ton + \$108
16	\$952 / ton + \$221
17	\$952 / ton + \$620
18	\$952 / ton + \$620

¹⁹⁶ Baseline SEER and EER should be updated when new minimum federal standards become effective.

¹⁹⁷ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

<http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf>

The "lifespan" of a central air conditioner is about 15 to 20 years (US DOE:

http://www.energysavers.gov/your_home/space_heating_cooling/index.cfm/mytopic=12440).

¹⁹⁸ Assumed to be one third of effective useful life

¹⁹⁹ Based on incremental cost results from Cadmus "HVAC Program: Incremental Cost Analysis Update", December 19, 2016.

²⁰⁰ Based on 3 ton initial cost estimate for a conventional unit from ENERGY STAR Central AC calculator, \$2,857

(http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls). Efficiency cost increment consistent with Cadmus study results.

Assumed deferred cost (after 6 years) of replacing existing equipment with new baseline unit is assumed to be \$3,140²⁰¹. This cost should be discounted to present value using the nominal societal discount rate.

LOADSHAPE

Loadshape R08 - Residential Cooling

COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

CF_{SPP} = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)
= 68%²⁰²

CF_{PJM} = PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)
= 46.6%²⁰³

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Time of sale:

$$\Delta \text{kWH} = (\text{FLHcool} * \text{Btu/hr} * (1/\text{SEERbase} - 1/\text{SEERee}))/1000$$

Early replacement²⁰⁴:

ΔkWH for remaining life of existing unit (1st 6 years):

$$=((\text{FLHcool} * \text{Capacity} * (1/\text{SEERexist} - 1/\text{SEERee}))/1000);$$

ΔkWH for remaining measure life (next 12 years):

$$= ((\text{FLHcool} * \text{Capacity} * (1/\text{SEERbase} - 1/\text{SEERee}))/1000)$$

Where:

FLHcool = Full load cooling hours

= dependent on location and building type²⁰⁵:

²⁰¹ Based on 3 ton initial cost estimate for a conventional unit from ENERGY STAR Central AC calculator, \$2,857, and applying inflation rate of 1.91% (http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls). While baselines are likely to shift in the future, there is currently no good indication of what the cost of a new baseline unit will be in 6 years. In the absence of this information, assuming a constant federal baseline cost is within the range of error for this prescriptive measure.

²⁰² Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

²⁰³ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

²⁰⁴ The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input which would be the (new base to efficient savings)/(existing to efficient savings).

²⁰⁵ Full load hours for Chicago, Moline and Rockford are provided in "Final Evaluation Report: Central Air Conditioning Efficiency

Climate Zone (City based upon)	FLHcool (single family)	FLHcool (multi family)
1 (Rockford)	512	467
2 (Chicago)	570	506
3 (Springfield)	730	663
4 (Belleville)	1035	940
5 (Marion)	903	820
Weighted Average ²⁰⁶	629	564

- Capacity = Size of new equipment in Btu/hr (note 1 ton = 12,000Btu/hr)
= Actual installed, or if actual size unknown 33,600Btu/hr for single-family buildings²⁰⁷
- SEERbase = Seasonal Energy Efficiency Ratio of baseline unit (kBtu/kWh)
= 13²⁰⁸
- SEERexist = Seasonal Energy Efficiency Ratio of existing unit (kBtu/kWh)
= Use actual SEER rating where it is possible to measure or reasonably estimate. If unknown assume 10.0²⁰⁹.
- SEERee = Seasonal Energy Efficiency Ratio of ENERGY STAR unit (kBtu/kWh)
= Actual installed or 14.5 if unknown

Time of sale example: a 3 ton unit with SEER rating of 14.5, in unknown location:

$$\Delta \text{kWh} = (629 * 36,000 * (1/13 - 1/14.5)) / 1000$$

$$= 180 \text{ kWh}$$

Early replacement example: a 3 ton unit, with SEER rating of 14.5 replaces an existing unit in unknown location:

$$\Delta \text{kWh}(\text{for first 6 years}) = (629 * 36,000 * (1/10 - 1/14.5)) / 1000$$

$$= 702 \text{ kWh}$$

$$\Delta \text{kWh}(\text{for next 12 years}) = (629 * 36,000 * (1/13 - 1/14.5)) / 1000$$

$$= 180 \text{ kWh}$$

Therefore savings adjustment of 26% (180/702) after 6 years.

Services (CACES), 2010, Navigant Consulting”, p.33. An average FLH/Cooling Degree Day (from NCDC) ratio was calculated for these locations and applied to the CDD of the other locations in order to estimate FLH. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

²⁰⁶ Weighted based on number of residential occupied housing units in each zone.

²⁰⁷ Actual unit size required for multi-family building, no size assumption provided because the unit size and resulting savings can vary greatly depending on the number of units.

²⁰⁸ Based on Minimum Federal Standard;

http://www1.eere.energy.gov/buildings/appliance_standards/residential/residential_cac_hp.html.

²⁰⁹ VEIC estimate based on Department of Energy Federal Standard between 1992 and 2006. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

Time of sale:

$$\Delta kW = (\text{Capacity} * (1/\text{EER}_{\text{base}} - 1/\text{EER}_{\text{ee}}))/1000 * CF$$

Early replacement²¹⁰:

ΔkW for remaining life of existing unit (1st 6 years):

$$= ((\text{Capacity} * (1/\text{EER}_{\text{exist}} - 1/\text{EER}_{\text{ee}}))/1000 * CF);$$

ΔkW for remaining measure life (next 12 years):

$$= ((\text{Capacity} * (1/\text{EER}_{\text{base}} - 1/\text{EER}_{\text{ee}}))/1000 * CF)$$

Where:

EER_{base} = EER Efficiency of baseline unit

$$= 11.2^{211}$$

$\text{EER}_{\text{exist}}$ = EER Efficiency of existing unit

= Actual EER of unit should be used, if EER is unknown, use 9.2²¹²

EER_{ee} = EER Efficiency of ENERGY STAR unit

= Actual installed or 12 if unknown

CF_{SSP} = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)

$$= 68\%^{213}$$

CF_{PJM} = PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)

$$= 46.6\%^{214}$$

²¹⁰ The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a “number of years to adjustment” and “savings adjustment” input which would be the (new base to efficient savings)/(existing to efficient savings).

²¹¹ The federal Standard does not currently include an EER component. The value is approximated based on the SEER standard (13) and equals EER 11.2. To perform this calculation we are using this formula: $(-0.02 * \text{SEER}^2) + (1.12 * \text{SEER})$ (from Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder).

²¹² Based on SEER of 10,0, using formula above to give 9.2 EER.

²¹³ Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

²¹⁴ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

Time of sale example: a 3 ton unit with EER rating of 12:

$$\begin{aligned}\Delta kW_{SSP} &= (36,000 * (1/11.2 - 1/12)) / 1000 * 0.68 \\ &= 0.146 \text{ kW}\end{aligned}$$

$$\begin{aligned}\Delta kW_{PJM} &= (36,000 * (1/11.2 - 1/12)) / 1000 * 0.466 \\ &= 0.100 \text{ kW}\end{aligned}$$

Early replacement example: a 3 ton unit with EER rating of 12 replaces an existing unit:

$$\begin{aligned}\Delta kW_{SSP} \text{ (for first 6 years)} &= (36,000 * (1/9.2 - 1/12)) / 1000 * 0.68 \\ &= 0.621 \text{ kW}\end{aligned}$$

$$\begin{aligned}\Delta kW_{SSP} \text{ (for next 12 years)} &= (36,000 * (1/11.2 - 1/12)) / 1000 * 0.68 \\ &= 0.146 \text{ kW}\end{aligned}$$

$$\begin{aligned}\Delta kW_{PJM} \text{ (for first 6 years)} &= (36,000 * (1/9.2 - 1/12)) / 1000 * 0.466 \\ &= 0.425 \text{ kW}\end{aligned}$$

$$\begin{aligned}\Delta kW_{PJM} \text{ (for next 12 years)} &= (36,000 * (1/11.2 - 1/12)) / 1000 * 0.466 \\ &= 0.100 \text{ kW}\end{aligned}$$

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-HVC-CAC1-V07-180101

REVIEW DEADLINE: 1/1/2021

5.3.4 Duct Insulation and Sealing

DESCRIPTION

This measure describes evaluating the savings associated with performing duct sealing using mastic sealant or metal tape to the distribution system of homes with either central air conditioning or a ducted heating system.

Two methodologies for estimating the savings associated from sealing the ducts are provided. The first preferred method requires the use of a blower door and the second requires careful inspection of the duct work.

1. **Modified Blower Door Subtraction** – this technique is described in detail on p.44 of the Energy Conservatory Blower Door Manual; which can be found on the Energy Conservatory website (As of Oct 2014: http://www.energyconservatory.com/sites/default/files/documents/mod_3-4_dg700_-_new_flow_rings_-_cr_-_tpt_-_no_fr_switch_manual_ce_0.pdf)
2. **Evaluation of Distribution Efficiency** – this methodology requires the evaluation of three duct characteristics below, and use of the Building Performance Institutes ‘Distribution Efficiency Look-Up Table’;
<http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf>
 - a. Percentage of duct work found within the conditioned space
 - b. Duct leakage evaluation
 - c. Duct insulation evaluation

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is sealed duct work throughout the unconditioned or semi-conditioned space in the home. A non-conditioned space is defined as a space outside of the thermal envelope of the building that is not intentionally heated for occupancy (crawl space, roof attic, etc). A semi-conditioned space is defined as a space within the thermal envelop that is not intentionally heated for occupancy (unfinished basement)²¹⁵.

DEFINITION OF BASELINE EQUIPMENT

The existing baseline condition is leaky duct work within the unconditioned or semi-conditioned space in the home.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of this measure is 20 years²¹⁶.

DEEMED MEASURE COST

The actual duct sealing measure cost should be used.

LOADSHAPE

Loadshape R08 - Residential Cooling

Loadshape R09 - Residential Electric Space Heat

²¹⁵ Definition matches Regain factor discussed in Home Energy Services Impact Evaluation, prepared for the Massachusetts Residential Retrofit and Low Income Program Area Evaluation, Cadmus Group, Inc., August 2012

²¹⁶ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.
http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf

Loadshape R10 - Residential Electric Heating and Cooling (Shell Measures)

COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

$$\begin{aligned} CF_{SSP} &= \text{Summer System Peak Coincidence Factor for Central A/C (during utility peak hour)} \\ &= 68\%^{217} \end{aligned}$$

$$\begin{aligned} CF_{PJM} &= \text{PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)} \\ &= 46.6\%^{218} \end{aligned}$$

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS****Methodology 1: Modified Blower Door Subtraction**

- a) Determine Duct Leakage rate before and after performing duct sealing:
 Duct Leakage (CFM50_{DL}) = (CFM50_{Whole House} – CFM50_{Envelope Only}) * SCF

Where:

CFM50 _{Whole House}	= Standard Blower Door test result finding Cubic Feet per Minute at 50 Pascal pressure differential
CFM50 _{Envelope Only}	= Blower Door test result finding Cubic Feet per Minute at 50 Pascal pressure differential with all supply and return registers sealed.
SCF	= Subtraction Correction Factor to account for underestimation of duct leakage due to connections between the duct system and the home. Determined by measuring pressure in duct system with registers sealed and using look up table provided by Energy Conservatory.

- b) Calculate duct leakage reduction, convert to CFM25_{DL} and factor in Supply and Return Loss Factors
 Duct Leakage Reduction (ΔCFM25_{DL}) = (Pre CFM50_{DL} – Post CFM50_{DL}) * 0.64 * (SLF + RLF)

Where:

0.64	= Converts CFM50 to CFM25 ²¹⁹
SLF	= Supply Loss Factor

²¹⁷ Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

²¹⁸ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

²¹⁹ 25 Pascals is the standard assumption for typical pressures experienced in the duct system under normal operating conditions. To convert CFM50 to CFM25 you multiply by 0.64 (inverse of the "Can't Reach Fifty" factor for CFM25; see Energy Conservatory Blower Door Manual).

= % leaks sealed located in Supply ducts * 1²²⁰

Default = 0.5²²¹

RLF = Return Loss Factor

= % leaks sealed located in Return ducts * 0.5²²²

Default = 0.25²²³

c) Calculate Electric Energy Savings:

$\Delta kWh = \Delta kWh_{cooling} + \Delta kWh_{Fan}$

$\Delta kWh_{cooling} = ((\Delta CFM25_{DL} / ((CapacityCool / 12,000) * 400)) * FLHcool * CapacityCool * TRFcool) / 1000 / \eta_{Cool}$

$\Delta kWh_{Fan} = (\Delta Therms * F_e * 29.3)$

Where:

$\Delta CFM25_{DL}$ = Duct leakage reduction in CFM25

= calculated above

CapacityCool = Capacity of Air Cooling system (Btu/hr)

= Actual

12,000 = Converts Btu/H capacity to tons

400 = Converts capacity in tons to CFM (400CFM / ton)²²⁴

FLHcool = Full load cooling hours

= Dependent on location as below²²⁵:

Climate Zone (City based upon)	FLHcool Single Family	FLHcool Multifamily
1 (Rockford)	512	467
2 (Chicago)	570	506
3 (Springfield)	730	663
4 (Belleville)	1,035	940

²²⁰ Assumes that for each percent of supply air loss there is one percent annual energy penalty. This assumes supply side leaks are direct losses to the outside and are not recaptured back to the house. This could be adjusted downward to reflect regain of usable energy to the house from duct leaks. For example, during the winter some of the energy lost from supply leaks in a crawlspace will probably be regained back to the house (sometimes 1/2 or more may be regained). More information provided in "Appendix E Estimating HVAC System Loss From Duct Airtightness Measurements" from <http://www.energyconservatory.com/download/dbmanual.pdf>

²²¹ Assumes 50% of leaks are in supply ducts.

²²² Assumes that for each percent of return air loss there is a half percent annual energy penalty. Note that this assumes that return leaks contribute less to energy losses than do supply leaks. This value could be adjusted upward if there was reason to suspect that the return leaks contribute significantly more energy loss than "average" (e.g. pulling return air from a super heated attic), or can be adjusted downward to represent significantly less energy loss (e.g. pulling return air from a moderate temperature crawl space). More information provided in "Appendix E Estimating HVAC System Loss From Duct Airtightness Measurements" from <http://www.energyconservatory.com/download/dbmanual.pdf>

²²³ Assumes 50% of leaks are in return ducts.

²²⁴ This conversion is an industry rule of thumb; e.g. see

<http://www.hvacsalesandsupply.com/Linked%20Documents/Tech%20Tips/61-Why%20400%20CFM%20per%20ton.pdf>

²²⁵ Based on Full Load Hours from ENERGY Star with adjustments made in a Navigant Evaluation, other cities were scaled using those results and CDD. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

Climate Zone (City based upon)	FLHcool Single Family	FLHcool Multifamily
5 (Marion)	903	820
Weighted Average ²²⁶	629	564

TRFcool = Thermal Regain Factor for cooling by space type
= 1.0 for Unconditioned Spaces
= 0.0 for Semi-Conditioned Spaces²²⁷

1000 = Converts Btu to kBtu

η Cool = Efficiency (SEER) of Air Conditioning equipment (kBtu/kWh)
= Actual. If unknown assume the following²²⁸:

Age of Equipment	SEER Estimate
Before 2006	10
After 2006 - 2014	13
Central AC After 1/1/2015	13
Heat Pump After 1/1/2015	14

Δ Therms = Therm savings as calculated in Natural Gas Savings

F_e = Furnace Fan energy consumption as a percentage of annual fuel consumption
= 3.14%²²⁹

29.3 = kWh per therm

²²⁶ Weighted based on number of occupied residential housing units in each zone.

²²⁷ Thermal regain (i.e. the potential for conditioned air escaping from ducts not being lost to the atmosphere) for residential pipe insulation measures is discussed in Home Energy Services Impact Evaluation, prepared for the Massachusetts Residential Retrofit and Low Income Program Area Evaluation, Cadmus Group, Inc., August 2012.

²²⁸ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

²²⁹ F_e is not one of the AHRI certified ratings provided for residential furnaces, but can be reasonably estimated from a calculation based on the certified values for fuel energy (E_f in MMBtu/yr) and E_{ae} (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the Energy Star version 3 criteria for 2% F_e . See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.

For example, duct sealing in unconditioned space a single family house in Springfield with a 36,000 Btu/H, SEER 11 central air conditioning, an 80% AFUE, 105,000 Btu/H natural gas furnace and the following blower door test results:

Before: $CFM_{50\text{Whole House}} = 4800 \text{ CFM}_{50}$

$CFM_{50\text{Envelope Only}} = 4500 \text{ CFM}_{50}$

House to duct pressure of 45 Pascals. = 1.29 SCF (Energy Conservatory look up table)

After: $CFM_{50\text{Whole House}} = 4600 \text{ CFM}_{50}$

$CFM_{50\text{Envelope Only}} = 4500 \text{ CFM}_{50}$

House to duct pressure of 43 Pascals = 1.39 SCF (Energy Conservatory look up table)

Duct Leakage:

$CFM_{50\text{DL before}} = (4800 - 4500) * 1.29$
 $= 387 \text{ CFM}$

$CFM_{50\text{DL after}} = (4600 - 4500) * 1.39$
 $= 139 \text{ CFM}$

Duct Leakage reduction at CFM25:

$\Delta CFM_{25\text{DL}} = (387 - 139) * 0.64 * (0.5 + 0.25)$
 $= 119 \text{ CFM}_{25}$

Energy Savings:

$\Delta kWh_{\text{cooling}} = [((119 / ((36,000/12,000) * 400)) * 730 * 36,000 * 1) / 1000 / 11] + (212 * 0.0314 * 29.3)$
 $= 237 + 195$
 $= 432 \text{ kWh}$

Heating savings for homes with electric heat:

$$\Delta kWh_{\text{heating}} = ((\Delta CFM_{25\text{DL}} / ((\text{OutputCapacityHeat} / 12,000) * 400)) * \text{FLH}_{\text{heat}} * \text{OutputCapacityHeat} * \text{TRF}_{\text{heat}}) / \eta_{\text{Heat}} / 3412$$

Where:

$\text{OutputCapacityHeat}$ = Heating output capacity (Btu/hr) of electric heat
 =Actual

FLH_{heat} = Full load heating hours
 = Dependent on location as below²³⁰:

Climate Zone (City based upon)	FLH_heat
1 (Rockford)	1,969
2 (Chicago)	1,840
3 (Springfield)	1,754

²³⁰ Heating EFLH based on ENERGY Star EFLH for Rockford, Chicago, and Springfield and on NCDC/NOAA HDD for the other two cities. In all cases, the hours were adjusted based on average natural gas heating consumption in IL.

Climate Zone (City based upon)	FLH_heat
4 (Belleville)	1,266
5 (Marion)	1,288
Weighted Average ²³¹	1,821

TRFheat = Thermal Regain Factor for heating by space type

= 0.40 for Semi-Conditioned Spaces

= 1.0 for Unconditioned Spaces²³²

ηHeat = Efficiency in COP of Heating equipment

= Actual. If not available use²³³:

System Type	Age of Equipment	HSPF Estimate	COP Estimate
Heat Pump	Before 2006	6.8	2.00
	After 2006 - 2014	7.7	2.26
	2015 on	8.2	2.40
Resistance	N/A	N/A	1.00

3412 = Converts Btu to kWh

For example, duct sealing in unconditioned space in a 36,000 Btu/H 2.5 COP heat pump heated single family house in Springfield with the blower door results described above:

$$\Delta \text{kWh}_{\text{heating}} = ((119 / ((36,000 / 12,000) * 400)) * 1,754 * 36,000 * 1) / 2.5 / 3412$$

$$= 734 \text{ kWh}$$

Methodology 2: Evaluation of Distribution Efficiency

Determine Distribution Efficiency by evaluating duct system before and after duct sealing using Building Performance Institute “Distribution Efficiency Look-Up Table”

$$\Delta \text{kWh} = (((DE_{\text{after}} - DE_{\text{before}}) / DE_{\text{after}}) * FLH_{\text{cool}} * \text{CapacityCool} * TRF_{\text{cool}}) / 1000 / \eta_{\text{Cool}} + (\Delta \text{Therms} * F_e * 29.3)$$

Where:

DE_{after} = Distribution Efficiency after duct sealing

DE_{before} = Distribution Efficiency before duct sealing

²³¹ Weighted based on number of occupied residential housing units in each zone.

²³² Thermal regain (i.e. the potential for conditioned air escaping from ducts not being lost to the atmosphere) for residential pipe insulation measures is discussed in Home Energy Services Impact Evaluation, prepared for the Massachusetts Residential Retrofit and Low Income Program Area Evaluation, Cadmus Group, Inc., August 2012.

²³³ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

FLHcool = Full load cooling hours
 = Dependent on location as below²³⁴:

Climate Zone (City based upon)	FLHcool Single Family	FLHcool Multifamily
1 (Rockford)	512	467
2 (Chicago)	570	506
3 (Springfield)	730	663
4 (Belleville)	1,035	940
5 (Marion)	903	820
Weighted Average ²³⁵	629	564

CapacityCool = Capacity of Air Cooling system (Btu/hr)
 =Actual

TRFcool = Thermal Regain Factor for cooling by space type
 = 1.0 for Unconditioned Spaces
 = 0.0 for Semi-Conditioned Spaces²³⁶

1000 = Converts Btu to kBtu

η Cool = Efficiency (SEER) of Air Conditioning equipment (kBtu/kWh)
 = Actual. If unknown assume²³⁷:

Age of Equipment	SEER Estimate
Before 2006	10
After 2006 - 2014	13
Central AC After 1/1/2015	13
Heat Pump After 1/1/2015	14

²³⁴ Based on Full Load Hours from ENERGY Star with adjustments made in a Navigant Evaluation, other cities were scaled using those results and CDD. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

²³⁵ Weighted based on number of occupied residential housing units in each zone.

²³⁶ Thermal regain for residential pipe insulation measures is discussed in Home Energy Services Impact Evaluation, prepared for the Massachusetts Residential Retrofit and Low Income Program Area Evaluation, Cadmus Group, Inc., August 2012.

²³⁷ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

For example, duct sealing in unconditioned space in a single family house in Springfield, with 36,000 Btu/H SEER 11 central air conditioning, an 80% AFUE, 105,000 Btu/H natural gas furnace and the following duct evaluation results:

$$DE_{\text{before}} = 0.85$$

$$DE_{\text{after}} = 0.92$$

Energy Savings:

$$\begin{aligned} \Delta kWh_{\text{cooling}} &= (((0.92 - 0.85)/0.92) * 730 * 36,000 * 1) / 1000 / 11 + (212 * 0.0314 * 29.3) \\ &= 182 + 195 \\ &= 377 \text{ kWh} \end{aligned}$$

Heating savings for homes with electric heat:

$$\Delta kWh_{\text{heating}} = ((DE_{\text{after}} - DE_{\text{before}}) / DE_{\text{after}}) * FLH_{\text{heat}} * \text{OutputCapacityHeat} * TRF_{\text{heat}} / \eta_{\text{Heat}} / 3412$$

Where:

OutputCapacityHeat = Heating output capacity (Btu/hr) of the electric heat
=Actual

FLHheat = Full load heating hours
= Dependent on location as below²³⁸:

Climate Zone (City based upon)	FLH_heat
1 (Rockford)	1,969
2 (Chicago)	1,840
3 (Springfield)	1,754
4 (Belleville)	1,266
5 (Marion)	1,288
Weighted Average ²³⁹	1,821

TRFheat = Thermal Regain Factor for heating by space type
= 0.40 for Semi-Conditioned Spaces
= 1.0 for Unconditioned Spaces²⁴⁰

COP = Coefficient of Performance of electric heating system²⁴¹
= Actual. If not available use²⁴²:

²³⁸ Heating EFLH based on ENERGY Star EFLH for Rockford, Chicago, and Springfield and on NCEC/NOAA HDD for the other two cities. In all cases, the hours were adjusted based on average natural gas heating consumption in IL.

²³⁹ Weighted based on number of occupied residential housing units in each zone.

²⁴⁰ Thermal regain for residential pipe insulation measures is discussed in Home Energy Services Impact Evaluation, prepared for the Massachusetts Residential Retrofit and Low Income Program Area Evaluation, Cadmus Group, Inc., August 2012.

²⁴¹ Note that the HSPF of a heat pump is equal to the COP * 3.413.

²⁴² These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for

System Type	Age of Equipment	HSPF Estimate	COP Estimate
Heat Pump	Before 2006	6.8	2.00
	After 2006 - 2014	7.7	2.26
	2015 on	8.2	2.40
Resistance	N/A	N/A	1.00

For example, duct sealing in unconditioned space in a 36,000 Btu/H, 2.5 COP heat pump heated single family house in Springfield with the following duct evaluation results:

$$DE_{\text{after}} = 0.92$$

$$DE_{\text{before}} = 0.85$$

Energy Savings:

$$\begin{aligned}\Delta kWh_{\text{heating}} &= ((0.92 - 0.85)/0.92) * 1,754 * 36,000 * 1) / 2.5) / 3412 \\ &= 563 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh_{\text{cooling}} / FLH_{\text{cool}} * CF$$

Where:

FLH_{cool} = Full load cooling hours:

= Dependent on location as below²⁴³:

Climate Zone (City based upon)	FLH _{cool} Single Family	FLH _{cool} Multifamily
1 (Rockford)	512	467
2 (Chicago)	570	506
3 (Springfield)	730	663
4 (Belleville)	1,035	940
5 (Marion)	903	820
Weighted Average ²⁴⁴	629	564

$$\begin{aligned}CF_{\text{SSP}} &= \text{Summer System Peak Coincidence Factor for Central A/C (during system peak hour)} \\ &= 68\%^{245}\end{aligned}$$

$$\begin{aligned}CF_{\text{PJM}} &= \text{PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)} \\ &= 46.6\%^{246}\end{aligned}$$

Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

²⁴³ Based on Full Load Hours from ENERGY Star with adjustments made in a Navigant Evaluation, other cities were scaled using those results and CDD. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

²⁴⁴ Weighted based on number of occupied residential housing units in each zone.

²⁴⁵ Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

²⁴⁶ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

NATURAL GAS SAVINGS

For homes with Natural Gas Heating:

Methodology 1: Modified Blower Door Subtraction

$$\Delta \text{Therm} = (((\Delta \text{CFM}_{25\text{DL}} / (\text{InputCapacityHeat} * 0.0123)) * \text{FLHheat} * \text{InputCapacityHeat} * \text{TRFheat} * (\eta_{\text{Equipment}} / \eta_{\text{System}})) / 100,000$$

Where:

$\Delta \text{CFM}_{25\text{DL}}$ = Duct leakage reduction in CFM25

InputCapacityHeat = Heating input capacity (Btu/hr)

=Actual

0.0123 = Conversion of Capacity to CFM (0.0123CFM / Btu/hr)²⁴⁷

FLHheat = Full load heating hours

=Dependent on location as below²⁴⁸:

Climate Zone (City based upon)	FLH_heat
1 (Rockford)	1,969
2 (Chicago)	1,840
3 (Springfield)	1,754
4 (Belleville)	1,266
5 (Marion)	1,288
Weighted Average ²⁴⁹	1,821

TRFheat = Thermal Regain Factor for heating by space type

= 0.40 for Semi-Conditioned Spaces

= 1.0 for Unconditioned Spaces²⁵⁰

100,000 = Converts Btu to therms

$\eta_{\text{Equipment}}$ = Heating Equipment Efficiency

²⁴⁷ Based on Natural Draft Furnaces requiring 100 CFM per 10,000 Btu, Induced Draft Furnaces requiring 130CFM per 10,000Btu and Condensing Furnaces requiring 150 CFM per 10,000 Btu (rule of thumb from http://contractingbusiness.com/enewsletters/cb_imp_43580/). Data provided by GAMA during the federal rule-making process for furnace efficiency standards, suggested that in 2000, 24% of furnaces purchased in Illinois were condensing units. Therefore a weighted average required airflow rate is calculated assuming a 50:50 split of natural v induced draft non-condensing furnaces, as 123 per 10,000Btu or 0.0123/Btu.

²⁴⁸ Heating EFLH based on ENERGY Star EFLH for Rockford, Chicago, and Springfield and on NCDC/NOAA HDD for the other two cities. In all cases, the hours were adjusted based on average natural gas heating consumption in IL.

²⁴⁹ Weighted based on number of occupied residential housing units in each zone.

²⁵⁰ Thermal regain for residential pipe insulation measures is discussed in Home Energy Services Impact Evaluation, prepared for the Massachusetts Residential Retrofit and Low Income Program Area Evaluation, Cadmus Group, Inc., August 2012.

	= Actual ²⁵¹ . If not available use 83% ²⁵²
η_{System}	= Pre duct sealing Heating System Efficiency (Equipment Efficiency * Pre Distribution Efficiency) ²⁵³
	= Actual. If not available use 70% ²⁵⁴

²⁵¹ The Equipment Efficiency can be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test.

If there are more than one heating systems, the weighted (by consumption) average efficiency should be used.

If the heating system or distribution is being upgraded within a package of measures together with the insulation upgrade, the new average heating system efficiency should be used.

²⁵² This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: <http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls>)) In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:

$$(0.24 \times 0.92) + (0.76 \times 0.8) = 0.829$$

²⁵³ The Distribution Efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute: (<http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf>) or by performing duct blaster testing.

²⁵⁴ Estimated as follows: $0.829 \times (1 - 0.15) = 0.70$

For example, duct sealing in unconditioned space in a house in Springfield with an 80% AFUE, 105,000 Btu/H (input capacity) natural gas furnace and the following blower door test results:

Before: $CFM_{50\text{Whole House}} = 4800$ CFM_{50}

$CFM_{50\text{Envelope Only}} = 4500$ CFM_{50}

House to duct pressure of 45 Pascals = 1.29 SCF (Energy Conservatory look up table)

After: $CFM_{50\text{Whole House}} = 4600$ CFM_{50}

$CFM_{50\text{Envelope Only}} = 4500$ CFM_{50}

House to duct pressure of 43 Pascals = 1.39 SCF (Energy Conservatory look up table)

Duct Leakage:

$CFM_{50\text{DL before}} = (4800 - 4500) * 1.29$
 $= 387$ CFM

$CFM_{50\text{DL after}} = (4600 - 4500) * 1.39$
 $= 119$ CFM

Duct Leakage reduction at CFM25:

$\Delta CFM_{25\text{DL}} = (387 - 119) * 0.64 * (0.5 + 0.25)$
 $= 119$ CFM_{25}

Energy Savings:

Pre Distribution Efficiency $= 1 - (387/4800) = 92\%$

$\eta_{\text{System}} = 80\% * 92\% = 74\%$

$\Delta\text{Therm} = ((119 / (105,000 * 0.0123)) * 1,754 * 105,000 * 1 * (0.8/0.74)) / 100,000$
 $= 183$ therms

Methodology 2: Evaluation of Distribution Efficiency

$$\Delta\text{Therm} = ((DE_{\text{after}} - DE_{\text{before}}) / DE_{\text{after}}) * FLH_{\text{heat}} * \text{InputCapacityHeat} * TRF_{\text{heat}} * (\eta_{\text{Equipment}} / \eta_{\text{System}}) / 100,000$$

Where:

DE_{after} = Distribution Efficiency after duct sealing

DE_{before} = Distribution Efficiency before duct sealing

Other variables as defined above

For example, duct sealing in unconditioned space in a house in Springfield an 80% AFUE, 105,000 Btu/H (input capacity) natural gas furnace and the following duct evaluation results:

$$DE_{\text{after}} = 0.92$$

$$DE_{\text{before}} = 0.85$$

Energy Savings:

$$\eta_{\text{System}} = 80\% * 85\% = 68\%$$

$$\begin{aligned} \Delta \text{Therm} &= ((0.92 - 0.85)/0.92) * 1,754 * 105,000 * 1 * (0.8/0.68) / 100,000 \\ &= 164 \text{ therm} \end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-HVC-DINS-V06-160601

REVIEW DEADLINE: 1/1/2022

5.3.5 Furnace Blower Motor

DESCRIPTION

A new furnace with a brushless permanent magnet (BPM) blower motor is installed instead of a new furnace with a lower efficiency motor. This measure characterizes only the electric savings associated with the fan and could be coupled with gas savings associated with a more efficient furnace. Savings decrease sharply with static pressure so duct improvements, and clean, low pressure drop filters can maximize savings. Savings improve when the blower is used for cooling as well and when it is used for continuous ventilation, but only if the non-BPM motor would have been used for continuous ventilation too. If the resident runs the BPM blower continuously because it is a more efficient motor and would not run a non-BPM motor that way, savings are near zero and possibly negative. This characterization uses a 2009 Focus on Energy study of BPM blower motor savings in Wisconsin, which accounted for the effects of this behavioral impact.

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

A furnace with a brushless permanent magnet (BPM) blower motor, also known by the trademark ECM, BLDC, and other names.

DEFINITION OF BASELINE EQUIPMENT

A furnace with a non-BPM blower motor.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 20 years²⁵⁵.

DEEMED MEASURE COST

The capital cost for this measure is assumed to be \$97²⁵⁶.

LOADSHAPE

Loadshape R08 - Residential Cooling

Loadshape R09 - Residential Electric Space Heat

Loadshape R10 - Residential Electric Heating and Cooling

COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

$$\begin{aligned} CF_{SSP} &= \text{Summer System Peak Coincidence Factor for Central A/C (during utility peak hour)} \\ &= 68\%^{257} \end{aligned}$$

²⁵⁵ Consistent with assumed life of a new gas furnace. Table 8.3.3 The Technical support documents for federal residential appliance standards: http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/chapter_8.pdf

²⁵⁶ Adapted from Tables 8.2.3 and 8.2.13 in http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/hvac_ch_08_lcc_2011-06-24.pdf

²⁵⁷ Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

$$\begin{aligned} CF_{PJM} &= \text{PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)} \\ &= 46.6\%^{258} \end{aligned}$$

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = \text{Heating Savings} + \text{Cooling Savings} + \text{Shoulder Season Savings}$$

Where:

$$\begin{aligned} \text{Heating Savings} &= \text{Blower motor savings during heating season} \\ &= 418 \text{ kWh}^{259} \end{aligned}$$

$$\begin{aligned} \text{Cooling Savings} &= \text{Blower motor savings during cooling season} \\ \text{If Central AC} &= 263 \text{ kWh} \\ \text{If No Central AC} &= 175 \text{ kWh} \\ \text{If unknown (weighted average)} & \\ &= 241 \text{ kWh}^{260} \end{aligned}$$

$$\begin{aligned} \text{Shoulder Season Savings} &= \text{Blower motor savings during shoulder seasons} \\ &= 51 \text{ kWh} \end{aligned}$$

For example, a blower motor in a home where Central AC presence is unknown:

$$\begin{aligned} \Delta kWh &= \text{Heating Savings} + \text{Cooling Savings} + \text{Shoulder Season Savings} \\ &= 418 + 263 + 51 \\ &= 732 \text{ kWh} \end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \text{Cooling Savings} / \text{FLH}_{\text{cooling}} * CF$$

Where:

$$\begin{aligned} \text{FLH}_{\text{cooling}} &= \text{Full load hours of air conditioning} \\ &= \text{Dependent on location}^{261}; \end{aligned}$$

²⁵⁸ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

²⁵⁹ To estimate heating, cooling and shoulder season savings for Illinois, VEIC adapted results from a 2009 Focus on Energy study of BPM blower motor savings in Wisconsin. This study included effects of behavior change based on the efficiency of new motor greatly increasing the amount of people that run the fan continuously. The savings from the Wisconsin study were adjusted to account for different run hour assumptions (average values used) for Illinois. See: FOE to IL Blower Savings.xlsx.

²⁶⁰ The weighted average value is based on assumption that 75% of homes installing BPM furnace blower motors have Central AC. 66% of IL housing units have CAC and 66% have gas furnaces. It is logical these two groups overlap to a large extent (like the 95% in the FOE study above).

²⁶¹ Full load hours for Chicago, Moline and Rockford are provided in "Final Evaluation Report: Central Air Conditioning Efficiency

Climate Zone (City based upon)	FLH_cooling
1 (Rockford)	512
2 (Chicago)	570
3 (Springfield)	730
4 (Belleville)	1,035
5 (Marion)	903
Weighted Average ²⁶²	629

CF_{SSP} = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)
= 68%²⁶³

CF_{PJM} = PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)
= 46.6%²⁶⁴

For example, a blower motor in a home of unknown location where Central AC prevalence is unknown:

$$\Delta kW_{SSP} = 251 / 629 * 0.68$$

$$= 0.271 \text{ kW}$$

$$\Delta kW_{SSP} = 251 / 629 * 0.466$$

$$= 0.186 \text{ kW}$$

NATURAL GAS SAVINGS

$$\Delta \text{therms}^{265} = - \text{Heating Savings} * 0.03412 / \text{AFUE}$$

Where:

0.03412 = Converts kWh to therms

AFUE = Efficiency of the Furnace

= Actual. If unknown assume 95%²⁶⁶ if in new furnace or 64.4 AFUE%²⁶⁷ if in existing furnace

Using defaults:

$$\text{For new Furnace} = - (418 * 0.03412) / 0.95$$

$$= - 15.0 \text{ therms}$$

$$\text{For existing Furnace} = - (418 * 0.03412) / 0.644$$

Services (CACES), 2010, Navigant Consulting”, p.33. An average FLH/Cooling Degree Day (from NCDC) ratio was calculated for these locations and applied to the CDD of the other locations in order to estimate FLH. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

²⁶² Weighted based on number of occupied residential housing units in each zone.

²⁶³ Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

²⁶⁴ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

²⁶⁵ The blower fan is in the heating duct so all, or very nearly all, of its waste heat is delivered to the conditioned space.

Negative value since this measure will increase the heating load due to reduced waste heat.

²⁶⁶ Minimum ENERGY STAR efficiency after 2.1.2012.

²⁶⁷ Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

= - 22.1 therms

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-HVC-FBMT-V03-150601

REVIEW DEADLINE: 1/1/2020

5.3.6 Gas High Efficiency Boiler

DESCRIPTION

High efficiency boilers achieve most gas savings through the utilization of a sealed combustion chamber and multiple heat exchangers that remove a significant portion of the waste heat from flue gasses. Because multiple heat exchangers are used to remove waste heat from the escaping flue gasses, some of the flue gasses condense and must be drained.

This measure characterizes:

a) Time of Sale:

- a. The installation of a new high efficiency, gas-fired hot water boiler in a residential location. This could relate to the replacement of an existing unit at the end of its useful life, or the installation of a new system in a new home.

b) Early Replacement:

Early Replacement determination will be based on meeting the following conditions:

- The existing unit is operational when replaced, or
- The existing unit requires minor repairs (<\$709)²⁶⁸.
- All other conditions will be considered Time of Sale.

The Baseline AFUE of the existing unit replaced:

- If the AFUE of the existing unit is known and $\leq 75\%$, the Baseline AFUE is the actual AFUE value of the unit replaced. If the AFUE is $>75\%$, the Baseline AFUE = 82%.
- If the AFUE of the existing unit is unknown, use assumptions in variable list below (AFUE(exist)).
- If the operational status or repair cost of the existing unit is unknown, use time of sale assumptions.

A weighted average early replacement rate is provided for use when the actual baseline early replacement rates are unknown²⁶⁹.

Deemed Early Replacement Rates For Boilers

	Deemed Early Replacement Rate
Early Replacement Rate for Boiler participants	7%

This measure was developed to be applicable to the following program types: TOS, NC, EREP. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed Boiler must be ENERGY STAR qualified (AFUE rated at or greater than 85% and input capacity less than 300,000 Btu/hr).

²⁶⁸ The Technical Advisory Committee agreed that if the cost of repair is less than 20% of the new baseline replacement cost it can be considered early replacement. Note the non-inflated cost is used as this would be a cost consideration in the program year.

²⁶⁹ Based upon research from “Home Energy Efficiency Rebate Program GPY2 Evaluation Report” which outlines early replacement rates for both primary and secondary central air cooling (CAC) and residential furnaces. This is used as a reasonable proxy for boiler installations since boiler specific data is not available. Report presented to Nicor Gas Company February 27, 2014, available at <http://www.ilsag.info/evaluation-documents.html>.

DEFINITION OF BASELINE EQUIPMENT

Time of sale: The baseline equipment for this measure is a new, gas-fired, standard-efficiency water boiler. The current Federal Standard minimum is 82% AFUE.

Early replacement: The baseline for this measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 25 years²⁷⁰.

Early replacement: Remaining life of existing equipment is assumed to be 8 years²⁷¹.

DEEMED MEASURE COST

Time of sale: The incremental install cost for this measure is dependent on tier²⁷²:

Measure Type	Installation Cost	Incremental Install Cost
AFUE 82%	\$3543	n/a
AFUE 85% (Energy Star Minimum)	\$4268	\$725
AFUE 90%	\$4815	\$1,272
AFUE 95%	\$5328	\$1,785

Early Replacement: The full installation cost is provided in the table above. The assumed deferred cost (after 8 years) of replacing existing equipment with a new baseline unit is assumed to be \$4,045²⁷³. This cost should be discounted to present value using the nominal discount rate.

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

N/A

²⁷⁰ Table 8.3.3 The Technical support documents for federal residential appliance standards:

http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/chapter_8.pdf

²⁷¹ Assumed to be one third of effective useful life

²⁷² Based on data provided in Appendix E of the Appliance Standards Technical Support Documents including equipment cost and installation labor

(http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/appendix_e.pdf). Where efficiency ratings are not provided, the values are interpolated from those that are.

²⁷³ \$3543 inflated using 1.91% rate.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS SAVINGS

Time of Sale:

$$\Delta \text{Therms} = \text{Gas_Boiler_Load} * \text{HF} * (1/\text{AFUE}(\text{base}) - 1/\text{AFUE}(\text{eff}))$$

Early replacement²⁷⁴:

Δ Therms for remaining life of existing unit (1st 8 years):

$$= \text{Gas_Boiler_Load} * \text{HF} * (1/\text{AFUE}(\text{exist}) - 1/\text{AFUE}(\text{eff}))$$

Δ Therms for remaining measure life (next 17 years):

$$= \text{Gas_Boiler_Load} * \text{HF} * (1/\text{AFUE}(\text{base}) - 1/\text{AFUE}(\text{eff}))$$

Where:

Gas_Boiler_Load²⁷⁵ = Estimate of annual household Load for gas boiler heated single-family homes. If location is unknown, assume the average below²⁷⁶.

= or Actual if informed by site-specific load calculations, ACCA Manual J or equivalent²⁷⁷.

Climate Zone (City based upon)	Gas_Boiler Load (therms)
1 (Rockford)	1275
2 (Chicago)	1218
3 (Springfield)	1043
4 (Belleville)	805
5 (Marion)	819
Average	1158

HF = Household factor, to adjust heating consumption for non-single-family households.

Household Type	HF
Single-Family	100%

²⁷⁴ The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a “number of years to adjustment” and “savings adjustment” input which would be the (new base to efficient savings)/(existing to efficient savings).

²⁷⁵ Boiler consumption values are informed by an evaluation which did not identify any fraction of heating load due to domestic hot water (DHW) provided by the boiler. Thus these values are an average of both homes with boilers only providing heat, and homes with boilers that also provide DHW. Heating load is used to describe the household heating need, which is equal to (gas heating consumption * AFUE)

²⁷⁶ Values are based on household heating consumption values and inferred average AFUE results from Table 3-4, Program Sample Analysis, *Nicor R29 Res Rebate Evaluation Report 092611_REV FINAL to Nicor*. Adjusting to a statewide average using relative HDD values to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city’s HDD.

²⁷⁷ The Air Conditioning Contractors of America Manual J, Residential Load Calculation 8th Edition produces equipment sizing loads for Single Family, Multi-single, and Condominiums using input characteristics of the home. A best practice for equipment selection and installation of Heating and Air Conditioning, load calculations should be completed by contractors during the selection process and may be readily available for program data purposes.

Household Type	HF
Multi-Family	65% ²⁷⁸
Actual	Custom ²⁷⁹

AFUE(exist) = Existing Boiler Annual Fuel Utilization Efficiency Rating
 = Use actual AFUE rating where it is possible to measure or reasonably estimate.
 If unknown, assume 61.6 AFUE%²⁸⁰.

AFUE(base) = Baseline Boiler Annual Fuel Utilization Efficiency Rating
 = 82%

AFUE(eff) = Efficient Boiler Annual Fuel Utilization Efficiency Rating
 = Actual. If unknown, use defaults dependent²⁸¹ on tier as listed below:

Measure Type	AFUE(eff)
ENERGY STAR®	87.5%
AFUE 90%	92.5%
AFUE 95%	95%

Time of Sale:

For example, a default sized ENERGY STAR boiler purchased and installed near Springfield

$$\begin{aligned}\Delta\text{Therms} &= 1043 * (1/0.82 - 1/0.875) \\ &= 80.0 \text{ Therms}\end{aligned}$$

Early Replacement:

For example, an existing function boiler with unknown efficiency is replaced with an ENERGY STAR boiler purchased and installed in Springfield.

Δ Therms for remaining life of existing unit (1st 8 years):

$$\begin{aligned}&= 1043 * (1/0.616 - 1/0.875) \\ &= 501 \text{ Therms}\end{aligned}$$

Δ Therms for remaining measure life (next 17 years):

$$\begin{aligned}&= (1043) * (1/0.82 - 1/0.875) \\ &= 80.0 \text{ Therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

²⁷⁸ Multifamily household heating consumption relative to single-family households is affected by overall household square footage and exposure to the exterior. This 65% factor is applied to MF homes based on professional judgment that average household size, and heat loads of MF households are smaller than single-family homes

²⁷⁹ Program-specific household factors may be utilized on the basis of sufficiently validated program evaluations.

²⁸⁰ Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

²⁸¹ Default values per tier selected based upon the average AFUE value for the tier range except for the top tier where the minimum is used due to proximity to the maximum possible.

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-HVC-GHEB-V06-180101

REVIEW DEADLINE: 1/1/2021

5.3.7 Gas High Efficiency Furnace

DESCRIPTION

High efficiency furnace features may include improved heat exchangers and modulating multi-stage burners.

This measure characterizes:

- a) Time of sale:
 - a. The installation of a new high efficiency, gas-fired condensing furnace in a residential location. This could relate to the replacement of an existing unit at the end of its useful life, or the installation of a new system in a new home.
- b) Early Replacement:

Early Replacement determination will be based on meeting the following conditions:

- The existing unit is operational when replaced, or
- The existing unit requires minor repairs (<\$528)²⁸².
- All other conditions will be considered Time of Sale.

The Baseline AFUE of the existing unit replaced:

- If the AFUE of the existing unit is known and $\leq 75\%$, the Baseline AFUE is the actual AFUE value of the unit replaced. If the AFUE is $> 75\%$, the Baseline AFUE = 80%.
- If the AFUE of the existing unit is unknown, use assumptions in variable list below (AFUE(exist)).
- If the operational status or repair cost of the existing unit is unknown, use time of sale assumptions.

A weighted average early replacement rate is provided for use when the actual baseline early replacement rate is unknown²⁸³.

Deemed Early Replacement Rates For Furnaces

Replacement Scenario for the Furnace	Deemed Early Replacement Rate
Early Replacement Rate for Furnace-only participants	7%
Early Replacement Rate for a furnace when the furnace is the Primary unit in a Combined System Replacement (CSR) project	14%
Early Replacement Rate for a furnace when the furnace is the Secondary unit in a CSR project	46%

Verified Quality Installation

This approach uses in-field measurement and interpretation of static pressures, identification and plotting of airflow, airflow measurement, temperature measurement and diagnostics, pressure measurements and duct design, and

²⁸² The Technical Advisory Committee agreed that if the cost of repair is less than 20% of the new baseline replacement cost it can be considered early replacement. Note the non-inflated cost is used as this would be a cost consideration in the program year.

²⁸³ Based upon research from “Home Energy Efficiency Rebate Program GPY2 Evaluation Report” which outlines early replacement rates for both primary and secondary central air cooling (CAC) and residential furnaces. The unit (furnace or CAC unit) that initially caused the customer to contact a trade ally is defined as the “primary unit”. The furnace or CAC unit that was also replaced but did not initially prompt the customer to contact a trade ally is defined as the “secondary unit”. This evaluation used different criteria for early replacement due to the availability of data after the fact; cost of any repairs < \$550 and age of unit < 20 years. Report presented to Nicor Gas Company February 27, 2014, available at <http://www.ilsag.info/evaluation-documents.html>.

BTU measurement to ensure that newly installed equipment is operating according to manufacturers' published potential performance. Installed equipment operating efficiency is largely dependent on the efficiency rating of the equipment, the skill of the installation contractor, the degree to which the equipment has aged or drifted from initial settings, and the system level constraints. When one or more of these key dependencies are operating sub-optimally, the overall efficiency of the equipment is degraded. A Verified Quality Install identifies sub-optimal performance and prescribes a solution during furnace installation.

This measure was developed to be applicable to the following program types: TOS, NC, EREP. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a residential sized (input energy less than 225,000 Btu/hr) natural gas fired furnace with an Annual Fuel Utilization Efficiency (AFUE) rating exceeding the program requirements.

DEFINITION OF BASELINE EQUIPMENT

Time of Sale: The current Federal Standard for gas furnaces is an AFUE rating of 80%. The baseline will be adjusted when the Federal Standard is updated.

Early replacement: The baseline for this measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and a new baseline unit for the remainder of the measure life. We estimate that the new baseline unit that could be purchased in the year the existing unit would have needed replacing is 90%.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 20 years²⁸⁴.

For early replacement: Remaining life of existing equipment is assumed to be 6 years²⁸⁵.

DEEMED MEASURE COST

Time of sale: The incremental installed cost (retail equipment cost plus installation cost) for this measure depends on efficiency as listed below²⁸⁶:

AFUE	Installed Cost	Incremental Installed Cost
80%	\$2011	n/a
90%	\$2641	\$630
91%	\$2727	\$716
92%	\$2813	\$802
93%	\$3025	\$1014
94%	\$3237	\$1226
95%	\$3449	\$1438
96%	\$3661	\$1650

Early Replacement: The full installed cost is provided in the table above. The assumed deferred cost (after 6 years)

²⁸⁴ Table 8.3.3 The Technical support documents for federal residential appliance standards:

http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/chapter_8.pdf

²⁸⁵ Assumed to be one third of effective useful life

²⁸⁶ Based on data from Table E.1.1 of Appendix E of the Appliance Standards Technical Support Documents including equipment cost and installation

labor. (http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/appendix_e.pdf). Where efficiency ratings are not provided, the values are interpolated from those that are. Note that ECM furnace fan cost (refer to other measure in TRM) has been deducted from the 93%-96% AFUE values to avoid double counting.

of replacing existing equipment with a new 90% baseline unit is assumed to be \$2903²⁸⁷. This cost should be discounted to present value using the nominal discount rate.

Verified Quality Installation: The additional design and installation work associated with verified quality installation has been estimated to take 1-2 hours (Tim Hanes, ESI). At \$40/hr, VQI adds \$60 to the installed cost.

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Electrical energy savings from the more fan-efficient (typically using brushless permanent magnet (BPM) blower motor) should also be claimed, please refer to “Furnace Blower Motor” characterization for details.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

If the blower motor is also used for cooling, coincident peak demand savings should also be claimed, please refer to “Furnace Blower Motor” characterization for savings details.

NATURAL GAS SAVINGS

Time of Sale:

$$\Delta \text{Therms} = \text{Gas_Furnace_Heating_Load} * \text{HF} * \left(\frac{1}{\text{AFUE}(\text{base}) * (1 - \text{Derating}(\text{base}))} \right) - \left(\frac{1}{\text{AFUE}(\text{eff}) * (1 - \text{Derating}(\text{eff}))} \right)$$

Early replacement²⁸⁸:

Δ Therms for remaining life of existing unit (1st 6 years):

$$= \text{Gas_Furnace_Heating_Load} * \text{HF} * \left(\frac{1}{\text{AFUE}(\text{exist}) * (1 - \text{Derating}(\text{exist}))} \right) - \left(\frac{1}{\text{AFUE}(\text{eff}) * (1 - \text{Derating}(\text{eff}))} \right)$$

Δ Therms for remaining measure life (next 14 years):

$$= \text{Gas_Furnace_Heating_Load} * \text{HF} * \left(\frac{1}{\text{AFUE}(\text{base}) * (1 - \text{Derating}(\text{base}))} \right) - \left(\frac{1}{\text{AFUE}(\text{eff}) * (1 - \text{Derating}(\text{eff}))} \right)$$

Where:

Gas_Furnace_Heating_Load

²⁸⁷ \$2641 inflated using 1.91% rate.

²⁸⁸ The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a “number of years to adjustment” and “savings adjustment” input which would be the (new base to efficient savings)/(existing to efficient savings).

= Estimate of annual household heating load ²⁸⁹ for gas furnace heated single-family homes. If location is unknown, assume the average below²⁹⁰.

= Actual if informed by site-specific load calculations, ACCA Manual J or equivalent²⁹¹.

Climate Zone (City based upon)	Gas_Furnace_Heating_Load (therms)
1 (Rockford)	873
2 (Chicago)	834
3 (Springfield)	714
4 (Belleville)	551
5 (Marion)	561
Average	793

HF = Household factor, to adjust heating consumption for non-single-family households.

Household Type	HF
Single-Family	100%
Multi-Family	65% ²⁹²
Actual	Custom ²⁹³

AFUE(exist) = Existing Furnace Annual Fuel Utilization Efficiency Rating

= Use actual AFUE rating where it is possible to measure or reasonably estimate.

If unknown, assume 64.4 AFUE% ²⁹⁴.

AFUE(base) = Baseline Furnace Annual Fuel Utilization Efficiency Rating

= Dependent on program type as listed below²⁹⁵:

Program Year	AFUE(base)
Time of Sale	80%
Early Replacement ²⁹⁶	90%

²⁸⁹ Heating load is used to describe the household heating need, which is equal to (gas consumption * AFUE)

²⁹⁰ Values are based on household heating consumption values and inferred average AFUE results from Table 2-1, *Energy Efficiency / Demand Response Nicor Gas Plan Year 1 (6/1/2011-5/31/2012) Research Report: Furnace Metering Study* (August 1, 2013) (prepared by Navigant Consulting, Inc.) and adjusting to a statewide average using relative HDD values to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

²⁹¹ The Air Conditioning Contractors of America Manual J, Residential Load Calculation 8th Edition produces equipment sizing loads for Single Family, Multi-single, and Condominiums using input characteristics of the home. A best practice for equipment selection and installation of Heating and Air Conditioning, load calculations are commonly completed by contractors during the selection process and may be readily available for program data purposes.

²⁹² Multifamily household heating consumption relative to single-family households is affected by overall household square footage and exposure to the exterior. This 65% factor is applied to MF homes based on professional judgment that average household size, and heat loads of MF households are smaller than single-family homes

²⁹³ Program-specific household factors may be utilized on the basis of sufficiently validated program evaluations.

²⁹⁴ Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

²⁹⁵ Though the Federal Minimum AFUE is 78%, there were only 50 models listed in the AHRI database at that level. At AFUE 79% the total rises to 308. There are 3,548 active furnace models listed with AFUE ratings between 78 and 80.

²⁹⁶ We estimate that the new baseline unit that could be purchased in the year the existing unit would have needed replacing is 90%.

AFUE(eff)	= Efficient Furnace Annual Fuel Utilization Efficiency Rating = Actual. If unknown, assume 95% ²⁹⁷
Derating(base)	=Baseline furnace AFUE derating = 6.4% ²⁹⁸
Derating(eff)	=Efficient furnace AFUE derating =0% if verified quality installation is performed =6.4% if verified quality installation is not performed ²⁹⁹

Time of Sale:

For example, a 95% AFUE furnace purchased and installed with verified quality installation for an existing home near Rockford:

$$\Delta \text{Therms} = 873 * ((1/(0.8*(1-6.4\%))) - (1/(0.95*(1-0\%))))$$

$$= 247 \text{ therms}$$

For example, a 95% AFUE furnace purchased and installed without verified quality installation for an existing home near Rockford:

$$\Delta \text{Therms} = 873 * ((1/(0.8*(1-6.4\%))) - (1/(0.95*(1-6.4\%))))$$

$$= 184 \text{ therms}$$

Early Replacement:

For example, an existing functioning furnace with unknown efficiency is replaced with an 95% furnace using quality installation in Rockford:

$$\Delta \text{Therms for remaining life of existing unit (1st 6 years):}$$

$$= 873 * ((1/(0.644*(1-6.4\%))) - (1/(0.95*(1-0\%))))$$

$$= 529 \text{ therms}$$

$$\Delta \text{Therms for remaining measure life (next 14 years):}$$

$$= 873 * ((1/(0.9*(1-6.4\%))) - (1/(0.95*(1-0\%))))$$

$$= 117 \text{ therms}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

²⁹⁷ Minimum ENERGY STAR efficiency after 2.1.2012.

²⁹⁸ Brand, L., Yee, S., and Baker, J. "Improving Gas Furnace Performance: A Field and Laboratory Study at End of Life." Building Technologies Office. National Renewable Energy Laboratory. 2015
http://apps1.eere.energy.gov/buildings/publications/pdfs/building_america/improving-gas-furnace-performance.pdf, accessed September 6th, 2016, DOE/GO--102015-4624

²⁹⁹ Ibid

MEASURE CODE: RS-HVC-GHEF-V07-180101

REVIEW DEADLINE: 1/1/2021

5.3.8 Ground Source Heat Pump

DESCRIPTION

This measure characterizes the installation of a Ground Source Heat Pump under the following scenarios:

- a) New Construction:
 - i. The installation of a new residential sized Ground Source Heat Pump system meeting ENERGY STAR efficiency standards presented below in a new home.
 - ii. Note the baseline in this case should be determined via EM&V and the algorithms are provided to allow savings to be calculated from any baseline condition.
- b) Time of Sale:
 - i. The planned installation of a new residential sized Ground Source Heat Pump system meeting ENERGY STAR efficiency standards presented below to replace an existing system(s) that does not meet the criteria for early replacement described in section c below.
 - ii. Note the baseline in this case is an equivalent replacement system to that which exists currently in the home. The calculation of savings is dependent on whether an incentive for the installation has been provided by both a gas and electric utility, just an electric utility or just a gas utility.
 - iii. Additional DHW savings are calculated based upon the fuel and efficiency of the existing unit.
- c) Early Replacement/Retrofit:
 - i. The early removal of functioning either electric or gas space heating and/or cooling systems from service, prior to the natural end of life, and replacement with a new high efficiency Ground Source Heat Pump system.
 - ii. Note the baseline in this case is the existing equipment being replaced. The calculation of savings is dependent on whether an incentive for the installation has been provided by both a gas and electric utility, just an electric utility or just a gas utility.
 - iii. Additional DHW savings are calculated based upon the fuel and efficiency of the existing unit.
 - iv. Early Replacement determination will be based on meeting the following conditions:
 - The existing unit is operational when replaced, or
 - The existing unit requires minor repairs, defined as costing less than³⁰⁰:

Existing System	Maximum repair cost
Air Source Heat Pump	\$276 per ton
Central Air Conditioner	\$190 per ton
Boiler	\$709
Furnace	\$528
Ground Source Heat Pump	<\$249 per ton

 - All other conditions will be considered Time of Sale.
 - v. The Baseline efficiency of the existing unit replaced:
 - If the efficiency of the existing unit is less than the maximum shown below, the Baseline efficiency is the actual efficiency value of the unit replaced. If the efficiency is greater than the maximum, the Baseline efficiency is shown in the “New Baseline” column below:

Existing System	Maximum efficiency for Actual	New Baseline
Air Source Heat Pump	10 SEER	14 SEER

³⁰⁰ The Technical Advisory Committee agreed that if the cost of repair is less than 20% of the new baseline replacement cost it can be considered early replacement.

Existing System	Maximum efficiency for Actual	New Baseline
Central Air Conditioner	10 SEER	13 SEER
Boiler	75% AFUE	82% AFUE
Furnace	75% AFUE	80% AFUE
Ground Source Heat Pump	10 SEER	13 SEER

- If the efficiency of the existing unit is unknown, use assumptions in variable list below (SEER, HSPF or AFUE exist).
- If the operational status or repair cost of the existing unit is unknown use time of sale assumptions.

This measure was developed to be applicable to the following program types: TOS, NC, EREP. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment must be a Ground Source Heat Pump unit meeting the minimum ENERGY STAR efficiency level standards effective at the time of installation as detailed below:

ENERGY STAR Requirements (Effective January 1, 2012)

Product Type	Cooling EER	Heating COP
Water-to-air		
Closed Loop	17.1	3.6
Open Loop	21.1	4.1
Water-to-Water		
Closed Loop	16.1	3.1
Open Loop	20.1	3.5
DGX	16	3.6

DEFINITION OF BASELINE EQUIPMENT

For these products, baseline equipment includes Air Conditioning, Space Heating and Water Heating.

New Construction:

To calculate savings with an electric baseline, the baseline equipment is assumed to be an Air Source Heat Pump meeting the Federal Standard efficiency level; 14 SEER, 8.2 HSPF and 11.8³⁰¹ EER and a Federal Standard electric hot water heater.

To calculate savings with a furnace/central AC baseline, the baseline equipment is assumed to be an 80% AFUE Furnace and central AC meeting the Federal Standard efficiency level; 13 SEER, 11 EER. If a gas water heater, the Federal Standard baseline is calculated as follows³⁰²; for ≤55 gallon tanks = 0.675 – (0.0015 * storage size in gallons) and for tanks >55 gallon = 0.8012 – (0.00078 * storage size in gallons). For a 40-gallon storage water heater this would be 0.615 EF.

Time of Sale: The baseline for this measure is a new replacement unit of the same system type as the existing unit,

³⁰¹ The Federal Standard does not include an EER requirement, so it is approximated with this formula: $(-0.02 * SEER^2) + (1.12 * SEER)$ Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder.

³⁰² Minimum Federal Standard as of 4/1/2015;
<http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf>

meeting the baselines provided below.

Unit Type	Efficiency Standard
ASHP	14 SEER, 11.8 EER, 8.2 HSPF
Gas Furnace	80% AFUE
Gas Boiler	82% AFUE
Central AC	13 SEER, 11 EER

Early replacement / Retrofit: The baseline for this measure is the efficiency of the *existing* heating, cooling and hot water equipment for the assumed remaining useful life of the existing unit and a new baseline heating and cooling system for the remainder of the measure life (as provided in table above except for Gas Furnace where new baseline assumption is 90% due to pending standard change).

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 25 years³⁰³.

For early replacement, the remaining life of existing equipment is assumed to be 8 years³⁰⁴.

DEEMED MEASURE COST

New Construction and Time of Sale: The actual installed cost of the Ground Source Heat Pump should be used (default of \$3957 per ton³⁰⁵), minus the assumed installation cost of the baseline equipment (\$1381 per ton for ASHP³⁰⁶ or \$2011 for a new baseline 80% AFUE furnace or \$3543 for a new 82% AFUE boiler³⁰⁷ and \$952 per ton³⁰⁸ for new baseline Central AC replacement).

Early Replacement: The full installation cost of the Ground Source Heat Pump should be used (default provided above). The assumed deferred cost (after 8 years) of replacing existing equipment with a new baseline unit is assumed to be \$1,518 per ton for a new baseline Air Source Heat Pump, or \$2,903 for a new baseline 90% AFUE furnace or \$4,045 for a new 82% AFUE boiler and 1,047 per ton for new baseline Central AC replacement³⁰⁹. This future cost should be discounted to present value using the nominal societal discount rate.

LOADSHAPE

Loadshape R10 - Residential Electric Heating and Cooling (if replacing gas heat and central AC)³¹⁰
 Loadshape R09 - Residential Electric Space Heat (if replacing electric heat with no cooling)
 Loadshape R10 - Residential Electric Heating and Cooling (if replacing ASHP)

³⁰³ System life of indoor components as per DOE estimate <http://energy.gov/energysaver/articles/geothermal-heat-pumps>. The ground loop has a much longer life, but the compressor and other mechanical components are the same as an ASHP. http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf

³⁰⁴ Assumed to be one third of effective useful life

³⁰⁵ Based on data provided in 'Results of HomeE geothermal and air source heat pump rebate incentives documented by IL electric cooperatives'.

³⁰⁶ Based on data provided on Home Advisor website, providing national average ASHP cost based on 2465 cost submittals. <http://www.homeadvisor.com/cost/heating-and-cooling/install-a-heat-pump/>

³⁰⁷ Furnace and boiler costs are based on data provided in Appendix E of the Appliance Standards Technical Support Documents including equipment cost and installation labor (http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/appendix_e.pdf). Where efficiency ratings are not provided, the values are interpolated from those that are.

³⁰⁸ Based on 3 ton initial cost estimate for a conventional unit from ENERGY STAR Central AC calculator (http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls).

³⁰⁹ All baseline replacement costs are consistent with their respective measures and include inflation rate of 1.91%.

³¹⁰ The baseline for calculating electric savings is an Air Source Heat Pump.

Note for purpose of cost effectiveness screening a fuel switch scenario, the heating kWh increase and cooling kWh decrease should be calculated separately such that the appropriate loadshape (i.e. Loadshape R09 - Residential Electric Space Heat and Loadshape R08 – Residential Cooling respectively) can be applied.

COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

$$\begin{aligned} CF_{SSP} &= \text{Summer System Peak Coincidence Factor for Heat Pumps (during utility peak hour)} \\ &= 72\%^{311} \end{aligned}$$

$$\begin{aligned} CF_{PJM} &= \text{PJM Summer Peak Coincidence Factor for Heat Pumps (average during PJM peak period)} \\ &= 46.6\%^{312} \end{aligned}$$

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

New Construction and Time of Sale (non-fuel switch only):

$$\begin{aligned} \Delta kWh &= [\text{Cooling savings}] + [\text{Heating savings}] + [\text{DHW savings}] \\ &= [\text{FLHcool} * \text{Capacity_cooling} * (1/\text{SEER}_{\text{base}} - 1/\text{EER}_{\text{PL}})/1000] + [\text{Elecheat} * \text{FLHheat} * \\ &\quad \text{Capacity_heating} * (1/\text{HSPF}_{\text{base}} - 1/(\text{COP}_{\text{PL}} * 3.412))/1000] + [\text{ElecDHW} * \% \text{DHWDisplaced} \\ &\quad * ((1/\text{EF}_{\text{ELEC}} * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412)] \end{aligned}$$

New Construction and Time of Sale (fuel switch only):

If measure is supported by gas utility only, $\Delta kWh = 0$

If measure is supported by gas and electric utility or electric utility only, electric utility claim savings calculated below:

$$\begin{aligned} \Delta kWh &= [\text{Cooling savings}] + [\text{Heating savings from base ASHP to GSHP}] + [\text{DHW savings}] \\ &= [\text{FLHcool} * \text{Capacity_cooling} * (1/\text{SEER}_{\text{base}} - 1/\text{EER}_{\text{PL}})/1000] + [\text{FLHheat} * \\ &\quad \text{Capacity_heating} * (1/\text{HSPF}_{\text{ASHP}} - 1/(\text{COP}_{\text{PL}} * 3.412))/1000] + [\text{ElecDHW} * \% \text{DHWDisplaced} \\ &\quad * ((1/\text{EF}_{\text{ELEC}} * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412)] \end{aligned}$$

Early replacement (non-fuel switch only)³¹³:

ΔkWh for remaining life of existing unit (1st 8 years):

³¹¹ Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'. <http://www.icc.illinois.gov/downloads/public/edocket/368522.pdf>

³¹² Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

³¹³ The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a "number of years to adjustment" and "savings adjustment" input which would be the (new base to efficient savings)/(existing to efficient savings).

$$= [\text{Cooling savings}] + [\text{Heating savings}] + [\text{DHW savings}]$$

$$= [\text{FLHcool} * \text{Capacity_cooling} * (1/\text{SEER}_{\text{exist}} - 1/\text{EER}_{\text{PL}})/1000] + [\text{ElecHeat} * \text{FLHheat} * \text{Capacity_heating} * (1/\text{HSPF}_{\text{exist}} - 1/(\text{COP}_{\text{PL}} * 3.412))/1000] + [\text{ElecDHW} * \% \text{DHWDisplaced} * ((1/\text{EF}_{\text{ELEC}} * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412)]$$

ΔkWh for remaining measure life (next 17 years):

$$= [\text{FLHcool} * \text{Capacity_cooling} * (1/\text{SEER}_{\text{base}} - 1/\text{EER}_{\text{PL}})/1000] + [\text{ElecHeat} * \text{FLHheat} * \text{Capacity_heating} * (1/\text{HSPF}_{\text{base}} - 1/(\text{COP}_{\text{PL}} * 3.412))/1000] + [\text{ElecDHW} * \% \text{DHWDisplaced} * ((1/\text{EF}_{\text{ELEC}} * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412)]$$

Early replacement - fuel switch only (see illustrative examples after Natural Gas section):

If measure is supported by gas utility only, $\Delta \text{kWh} = 0$

If measure is supported by gas and electric utility or electric utility only, electric utility claim savings calculated below:

ΔkWh for remaining life of existing unit (1st 8 years):

$$= [\text{Cooling savings}] + [\text{Heating savings from base ASHP to GSHP}] + [\text{DHW savings}]$$

$$= [\text{FLHcool} * \text{Capacity_cooling} * (1/\text{SEER}_{\text{exist}} - 1/\text{EER}_{\text{PL}})/1000] + [\text{FLHheat} * \text{Capacity_heating} * (1/\text{HSPF}_{\text{ASHP}} - 1/(\text{COP}_{\text{PL}} * 3.412))/1000] + [\text{ElecDHW} * \% \text{DHWDisplaced} * ((1/\text{EF}_{\text{ELEC}} * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412)]$$

ΔkWh for remaining measure life (next 17 years):

$$= [\text{Cooling savings}] + [\text{Heating savings from base ASHP to GSHP}] + [\text{DHW savings}]$$

$$= [\text{FLHcool} * \text{Capacity_cooling} * (1/\text{SEER}_{\text{base}} - 1/\text{EER}_{\text{PL}})/1000] + [\text{FLHheat} * \text{Capacity_heating} * (1/\text{HSPF}_{\text{ASHP}} - 1/(\text{COP}_{\text{PL}} * 3.412))/1000] + [\text{ElecDHW} * \% \text{DHWDisplaced} * ((1/\text{EF}_{\text{ELEC}} * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412)]$$

Where:

FLHcool = Full load cooling hours

Dependent on location as below³¹⁴:

Climate Zone (City based upon)	FLHcool Single Family	FLHcool Multifamily
1 (Rockford)	512	467
2 (Chicago)	570	506
3 (Springfield)	730	663
4 (Belleville)	1,035	940
5 (Marion)	903	820
Weighted Average ³¹⁵	629	564

Capacity_cooling = Cooling Capacity of Ground Source Heat Pump (Btu/hr)

= Actual (1 ton = 12,000Btu/hr)

SEERbase = SEER Efficiency of new replacement baseline unit

³¹⁴ Based on Full Load Hours from ENERGY STAR with adjustments made in a Navigant Evaluation, other cities were scaled using those results and CDD. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

³¹⁵ Weighted based on number of occupied residential housing units in each zone.

Existing Cooling System	SEERbase
Air Source Heat Pump	14 ³¹⁶
Central AC	13 ³¹⁷
No central cooling	13 ³¹⁸

SEER_{exist} = SEER Efficiency of existing cooling unit

= Use actual SEER rating where it is possible to measure or reasonably estimate, if unknown assume default provided below:

Existing Cooling System	SEER _{exist}
Air Source Heat Pump	9.12 ³¹⁹
Central AC	8.60 ³²⁰
No central cooling	13 ³²¹

SEER_{ASHP} = SEER Efficiency of new baseline Air Source Heat Pump unit (for fuel switch)

= 14³²²

EER_{PL} = Part Load EER Efficiency of efficient GSHP unit³²³

= Actual installed

ElecHeat = 1 if existing building is electrically heated

= 0 if existing building is not electrically heated

FLH_{heat} = Full load heating hours

Dependent on location as below³²⁴:

Climate Zone (City based upon)	FLH _{heat}
1 (Rockford)	1,969
2 (Chicago)	1,840
3 (Springfield)	1,754
4 (Belleville)	1,266
5 (Marion)	1,288
Weighted Average ³²⁵	1,821

³¹⁶ Minimum Federal Standard as of 1/1/2015;

<http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf>

³¹⁷ Minimum Federal Standard; Federal Register, Vol. 66, No. 14, Monday, January 22, 2001/Rules and Regulations, p. 7170-7200.

³¹⁸ Assumes that the decision to replace existing systems includes desire to add cooling.

³¹⁹ Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

³²⁰ Ibid.

³²¹ Assumes that the decision to replace existing systems includes desire to add cooling.

³²² Minimum Federal Standard as of 1/1/2015;

<http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf>

³²³ As per conversations with David Buss territory manager for Connor Co, the SEER and COP ratings of an ASHP equate most appropriately with the part load EER and COP of a GSHP.

³²⁴ Heating EFLH based on ENERGY STAR EFLH for Rockford, Chicago, and Springfield and on NCD/NOAA HDD for the other two cities. In all cases, the hours were adjusted based on average natural gas heating consumption in IL. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

³²⁵ Weighted based on number of occupied residential housing units in each zone.

Capacity_heating = Heating Capacity of Ground Source Heat Pump (Btu/hr)

= Actual (1 ton = 12,000Btu/hr)

HSPF_{base} = Heating System Performance Factor of new replacement baseline heating system (kBtu/kWh)

Existing Heating System	HSPF_base
Air Source Heat Pump	8.2
Electric Resistance	3.41 ³²⁶

HSPF_{exist} = Heating System Performance Factor of existing heating system (kBtu/kWh)

= Use actual HSPF rating where it is possible to measure or reasonably estimate. If unknown assume default:

Existing Heating System	HSPF_exist
Air Source Heat Pump	5.44
Electric Resistance	3.41

HSPF_{ASHP} = Heating Season Performance Factor for new ASHP baseline unit (for fuel switch)

= 8.2³²⁷

COP_{PL} = Part Load Coefficient of Performance of efficient unit³²⁸

= Actual Installed

3.412 = Constant to convert the COP of the unit to the Heating Season Performance Factor (HSPF).

ElecDHW = 1 if existing DHW is electrically heated

= 0 if existing DHW is not electrically heated

%DHWDIsplaced = Percentage of total DHW load that the GSHP will provide

= Actual if known

= If unknown and if desuperheater installed assume 44%³²⁹

= 0% if no desuperheater installed

EF_{ELEC} = Energy Factor (efficiency) of electric water heater

= Actual. If unknown or for new construction assume federal standard³³⁰:

For <=55 gallons: 0.96 – (0.0003 * rated volume in gallons)

For >55 gallons: 2.057 – (0.00113 * rated volume in gallons)

³²⁶ Electric resistance has a COP of 1.0 which equals 1/0.293 = 3.41 HSPF.

³²⁷ Minimum Federal Standard as of 1/1/2015;

<http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf>

³²⁸ As per conversations with David Buss territory manager for Connor Co, the SEER and COP ratings of an ASHP equate most appropriately with the part load EER and COP of a GSHP.

³²⁹ Assumes that the desuperheater can provide two thirds of hot water needs for eight months of the year (2/3 * 2/3 = 44%). Based on input from Doug Dougherty, Geothermal Exchange Organization.

³³⁰ Minimum Federal Standard as of 4/1/2015;

<http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf>

GPD = Gallons Per Day of hot water use per person
 = 45.5 gallons hot water per day per household/2.59 people per household³³¹
 = 17.6

Household = Average number of people per household

Household Unit Type	Household
Single-Family - Deemed	2.56 ³³²
Custom	Actual Occupancy or Number of Bedrooms ³³³

365.25 = Days per year

γ_{Water} = Specific weight of water
 = 8.33 pounds per gallon

T_{OUT} = Tank temperature
 = 125°F

T_{IN} = Incoming water temperature from well or municipal system
 = 54°F³³⁴

1.0 = Heat Capacity of water (1 Btu/lb*°F)

3412 = Conversion from Btu to kWh

³³¹ Deoreo, B., and P. Mayer. Residential End Uses of Water Study Update. Forthcoming. ©2015 Water Research Foundation. Reprinted With Permission.

³³² ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program citing 2006-2008 American Community Survey data from the US Census Bureau for Illinois cited on p. 17 of the PY2 Evaluation report. 2.75 * 93% evaluation adjustment

³³³ Bedrooms are suitable proxies for household occupancy, and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.

³³⁴ US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL
http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html

Illustrative Examples**New Construction using ASHP baseline:**

For example, a 3 ton unit with Part Load EER rating of 19 and Part Load COP of 4.4 with desuperheater is installed with a 50 gallon electric water heater in single family house in Springfield:

$$\Delta \text{kWh} = [\text{FLHcool} * \text{Capacity_cooling} * (1/\text{SEER}_{\text{base}} - 1/\text{EER}_{\text{PL}})/1000] + [\text{FLHheat} * \text{Capacity_heating} * (1/\text{HSPF}_{\text{base}} - 1/(\text{COP}_{\text{PL}} * 3.412))/1000] + [\text{ElecDHW} * \% \text{DHWDisplaced} * ((1/\text{EF}_{\text{ELEC EXIST}} * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412)]$$

$$\begin{aligned} \Delta \text{kWh} &= [730 * 36,000 * (1/14 - 1/19) / 1000] + [1754 * 36,000 * (1/8.2 - 1/(4.4 * 3.412)) / 1000] \\ &+ [1 * 0.44 * ((1/0.945 * 17.6 * 2.56 * 365.25 * 8.33 * (125-54) * 1)/3412)] \\ &= 494 + 3494 + 1328 \\ &= 5316 \text{ kWh} \end{aligned}$$

Early Replacement – non-fuel switch (see example after Natural gas section for Fuel switch):

For example, a 3 ton unit with Part Load EER rating of 19 and Part Load COP of 4.4 with desuperheater is installed in single family house in Springfield with a 50 gallon electric water heater replacing an existing working Air Source Heat Pump with unknown efficiency ratings:

ΔkWh for remaining life of existing unit (1st 8 years):

$$\begin{aligned} &= [730 * 36,000 * (1/9.12 - 1/19) / 1000] + [1754 * 36,000 * (1/5.44 - 1/(4.4 * 3.412)) / 1000] \\ &+ [0.44 * 1 * ((1/0.945 * 17.6 * 2.56 * 365.25 * 8.33 * (125-54) * 1)/3412)] \\ &= 1498 + 7401 + 1328 \\ &= 10,227 \text{ kWh} \end{aligned}$$

ΔkWh for remaining measure life (next 17 years):

$$\begin{aligned} &= (730 * 36,000 * (1/14 - 1/28) / 1000) + [1967 * 36,000 * (1/8.2 - 1/(4.4 * 3.412)) / 1000] \\ &+ [0.44 * 1 * ((1/0.945 * 17.6 * 2.56 * 365.25 * 8.33 * (125-54) * 1)/3412)] \\ &= 494 + 3494 + 1328 \\ &= 5316 \text{ kWh} \end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS**New Construction and Time of Sale:**

$$\Delta \text{kW} = (\text{Capacity_cooling} * (1/\text{EER}_{\text{base}} - 1/\text{EER}_{\text{FL}}))/1000 * \text{CF}$$

Early replacement:

ΔkW for remaining life of existing unit (1st 8 years):

$$= (\text{Capacity_cooling} * (1/\text{EER}_{\text{exist}} - 1/\text{EER}_{\text{FL}}))/1000 * \text{CF}$$

ΔkW for remaining measure life (next 17 years):

$$= (\text{Capacity_cooling} * (1/\text{EER}_{\text{base}} - 1/\text{EER}_{\text{FL}}))/1000 * \text{CF}$$

Where:

EER_{base} = EER Efficiency of new replacement unit

Existing Cooling System	EER_base
Air Source Heat Pump	11.8 ³³⁵
Central AC	11 ³³⁶
No central cooling	11 ³³⁷

EER_{exist} = Energy Efficiency Ratio of existing cooling unit (kBtu/hr / kW)
 = Use actual EER rating where it is possible to measure or reasonably estimate. If EER unknown but SEER available convert using the equation:

$$\text{EER}_{\text{exist}} = (-0.02 * \text{SEER}_{\text{exist}}^2) + (1.12 * \text{SEER}_{\text{exist}}) \quad 338$$

If SEER rating unavailable use:

Existing Cooling System	EER_exist
Air Source Heat Pump	8.55
Central AC	8.15 ³⁴⁰
No central cooling	11 ³⁴¹

EER_{FL} = Full Load EER Efficiency of ENERGY STAR GSHP unit ³⁴²

CF_{SSP} = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)
 = 72%³⁴³

CF_{PJM} = PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)
 = 46.6%³⁴⁴

³³⁵ The Federal Standard does not include an EER requirement, so it is approximated with the conversion formula from Wassmer, M. 2003 thesis referenced below.

³³⁶ Minimum Federal Standard; Federal Register, Vol. 66, No. 14, Monday, January 22, 2001/Rules and Regulations, p. 7170-7200.

³³⁷ Assumes that the decision to replace existing systems includes desire to add cooling.

³³⁸ From Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder.

³³⁹ Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

³⁴⁰ Ibid.

³⁴¹ Assumes that the decision to replace existing systems includes desire to add cooling.

³⁴² As per conversations with David Buss territory manager for Connor Co, the EER rating of an ASHP equate most appropriately with the full load EER of a GSHP.

³⁴³ Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

<http://www.icc.illinois.gov/downloads/public/edocket/368522.pdf>

³⁴⁴ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

New Construction or Time of Sale:

For example, a 3 ton unit with Full Load EER rating of 19:

$$\begin{aligned}\Delta kW_{SSP} &= (36,000 * (1/11.8 - 1/19))/1000 * 0.72 \\ &= 0.83 \text{ kW}\end{aligned}$$

$$\begin{aligned}\Delta kW_{PJM} &= (36,000 * (1/11 - 1/19))/1000 * 0.466 \\ &= 0.54 \text{ kW}\end{aligned}$$

Early Replacement:

For example, a 3 ton Full Load 19 EER replaces an existing working Air Source Heat Pump with unknown efficiency ratings in Marion:

$$\begin{aligned}\Delta kW_{SSP} \text{ for remaining life of existing unit (1st 8 years):} \\ &= (36,000 * (1/8.55 - 1/19))/1000 * 0.72 \\ &= 1.67 \text{ kW}\end{aligned}$$

$$\begin{aligned}\Delta kW_{SSP} \text{ for remaining measure life (next 17 years):} \\ &= (36,000 * (1/11.8 - 1/19))/1000 * 0.72 \\ &= 0.83 \text{ kW}\end{aligned}$$

$$\begin{aligned}\Delta kW_{PJM} \text{ for remaining life of existing unit (1st 8 years):} \\ &= (36,000 * (1/8.55 - 1/19))/1000 * 0.466 \\ &= 1.08 \text{ kW}\end{aligned}$$

$$\begin{aligned}\Delta kW_{PJM} \text{ for remaining measure life (next 17 years):} \\ &= (36,000 * (1/11.8 - 1/19))/1000 * 0.466 \\ &= 0.54 \text{ kW}\end{aligned}$$

NATURAL GAS SAVINGS

New Construction and Time of Sale with baseline gas heat and/or hot water:

If measure is supported by gas utility only, gas utility claim savings calculated below:

$$\begin{aligned}\Delta \text{Therms} &= [\text{Heating Savings}] + [\text{DHW Savings}] \\ &= [\text{Replaced gas consumption} - \text{therm equivalent of GSHP source kWh}] + [\text{DHW Savings}] \\ &= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUEbase}) - (\text{kWh to Therm} * \text{FLHheat} * \\ &\quad \text{Capacity_heating} * 1/(\text{COP}_{PL} * 3.412))/1000)] + [(1 - \text{ElecDHW}) * \% \text{DHWDisplaced} * (1/ \\ &\quad \text{EF}_{\text{GAS EXIST}} * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (\text{T}_{\text{OUT}} - \text{T}_{\text{IN}}) * 1.0) / 100,000]\end{aligned}$$

If measure is supported by electric utility only, $\Delta \text{Therms} = 0$

If measure is supported by gas and electric utility, gas utility claim savings calculated below, (electric savings is provided in Electric Energy Savings section):

$$\begin{aligned}\Delta \text{Therms} &= [\text{Heating Savings}] + [\text{DHW Savings}] \\ &= [\text{Replaced gas consumption} - \text{therm equivalent of base ASHP source kWh}] + [\text{DHW Savings}] \\ &= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUEbase}) - (\text{kWh to Therm} * \text{FLHheat} * \end{aligned}$$

$$\text{Capacity_heating} * 1/\text{HSPF}_{\text{ASHP}}/1000) + [(1 - \text{ElecDHW}) * \% \text{DHWD} \text{Displaced} * (1/ \text{EF}_{\text{GAS}} \text{EXIST} * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 100,000)]$$

Early replacement for homes with existing gas heat and/or hot water:

If measure is supported by gas utility only, gas utility claim savings calculated below:

Δ Therms for remaining life of existing unit (1st 8 years):

$$\begin{aligned} &= [\text{Heating Savings}] + [\text{DHW Savings}] \\ &= [\text{Replaced gas consumption} - \text{therm equivalent of GSHP source kWh}] + [\text{DHW Savings}] \\ &= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUE}_{\text{exist}}) - (\text{kWh to Therm} * \text{FLHheat} * \text{Capacity_heating} * 1/(\text{COP}_{\text{PL}} * 3.412))/1000)] + [(1 - \text{ElecDHW}) * \% \text{DHWD} \text{Displaced} * (1/ \text{EF}_{\text{GAS}} \text{EXIST} * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 100,000)] \end{aligned}$$

Δ Therms for remaining measure life (next 17 years):

$$= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUE}_{\text{baseER}}) - (\text{kWh to Therm} * \text{FLHheat} * \text{Capacity_heating} * 1/(\text{COP}_{\text{PL}} * 3.412))/1000)] + [(1 - \text{ElecDHW}) * \% \text{DHWD} \text{Displaced} * (1/ \text{EF}_{\text{GAS}} \text{EXIST} * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 100,000)]$$

If measure is supported by electric utility only, Δ Therms = 0

If measure is supported by gas and electric utility, gas utility claim savings calculated below:

Δ Therms for remaining life of existing unit (1st 8 years):

$$\begin{aligned} \Delta \text{Therms} &= [\text{Heating Savings}] + [\text{DHW Savings}] \\ &= [\text{Replaced gas consumption} - \text{therm equivalent of base ASHP source kWh}] + [\text{DHW Savings}] \\ &= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUE}_{\text{exist}}) - (\text{kWh to Therm} * \text{FLHheat} * \text{Capacity_heating} * 1/\text{HSPF}_{\text{ASHP}})/1000)] + [(1 - \text{ElecDHW}) * \% \text{DHWD} \text{Displaced} * (1/ \text{EF}_{\text{GAS}} \text{EXIST} * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 100,000)] \end{aligned}$$

Δ Therms for remaining measure life (next 17 years):

$$= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUE}_{\text{baseER}}) - (\text{kWh to Therm} * \text{FLHheat} * \text{Capacity_heating} * 1/\text{HSPF}_{\text{ASHP}})/1000)] + [(1 - \text{ElecDHW}) * \% \text{DHWD} \text{Displaced} * (1/ \text{EF}_{\text{GAS}} \text{EXIST} * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 100,000)]$$

Where:

ElecHeat = 1 if existing building is electrically heated
= 0 if existing building is not electrically heated

Gas_Heating_Load
= Estimate of annual household heating load ³⁴⁵ for gas furnace heated single-family homes. If location is unknown, assume the average below.
= Actual if informed by site-specific load calculations, ACCA Manual J or equivalent³⁴⁶.

³⁴⁵ Heating load is used to describe the household heating need, which is equal to (gas consumption * AFUE)

³⁴⁶ The Air Conditioning Contractors of America Manual J, Residential Load Calculation 8th Edition produces equipment sizing loads for Single Family, Multi-single, and Condominiums using input characteristics of the home. A best practice for equipment selection and installation of Heating and Air Conditioning, load calculations are commonly completed by contractors during the selection process and may be readily available for program data purposes.

Climate Zone (City based upon)	Gas_Heating_Load if Furnace (therms) ³⁴⁷	Gas_Heating_Load if Boiler (therms) ³⁴⁸
1 (Rockford)	873	1275
2 (Chicago)	834	1218
3 (Springfield)	714	1043
4 (Belleville)	551	805
5 (Marion)	561	819
Average	793	1158

AFUEbase = Baseline Annual Fuel Utilization Efficiency Rating

= 80% if furnace and 82% if boiler.

AFUEexist = Existing Annual Fuel Utilization Efficiency Rating

= Use actual AFUE rating where it is possible to measure or reasonably estimate.

If unknown, assume 64.4% if furnace and 61.6%³⁴⁹ if boiler.

AFUEbaseER = Baseline Annual Fuel Utilization Efficiency Rating for early replacement measure

= 90%³⁵⁰ if furnace and 82% if boiler.

kWhtoTherm = Converts source kWh to Therms

= $H_{grid} / 100000$

H_{grid} = Heat rate of the grid in btu/kWh based on the average fossil heat rate for the EPA eGRID subregion and includes a factor that takes into account T&D losses.

For systems operating less than 6,500 hrs per year:

Use the Non-baseload heat rate provided by EPA eGRID for RFC West region for ComEd territory (including independent providers connected to RFC West), and SERC Midwest region for Ameren territory (including independent providers connected to SERC Midwest)³⁵¹. Also include any line losses.

For systems operating more than 6,500 hrs per year:

³⁴⁷ Values are based on household heating consumption values and inferred average AFUE results from Table 2-1, *Energy Efficiency / Demand Response Nicor Gas Plan Year 1 (6/1/2011-5/31/2012) Research Report: Furnace Metering Study* (August 1, 2013) (prepared by Navigant Consulting, Inc.) and adjusting to a statewide average using relative HDD values to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

³⁴⁸ Boiler consumption values are informed by an evaluation which did not identify any fraction of heating load due to domestic hot water (DHW) provided by the boiler. Thus these values are an average of both homes with boilers only providing heat, and homes with boilers that also provide DHW. Values are based on household heating consumption values and inferred average AFUE results from Table 3-4, Program Sample Analysis, *Nicor R29 Res Rebate Evaluation Report 092611_REV FINAL to Nicor*. Adjusting to a statewide average using relative HDD values to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

³⁴⁹ Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

³⁵⁰ Assumes that Federal Standard will have been increased to 90% by the time the existing unit would have to have been replaced.

³⁵¹ Refer to EPA eGRID data http://www.epa.gov/chp/documents/fuel_and_co2_savings.pdf, page 24 and http://www.epa.gov/cleanenergy/documents/egridzips/eGRID_9th_edition_V1-0_year_2010_Summary_Tables.pdf, page 9. Current values are:

- Non-Baseload RFC West: 9,811 Btu/kWh * (1 + Line Losses)
- Non-Baseload SERC Midwest: 10,511 Btu/kWh * (1 + Line Losses)
- All Fossil Average RFC West: 10,038 Btu/kWh * (1 + Line Losses)
- All Fossil Average SERC Midwest: 10,364 Btu/kWh * (1 + Line Losses)

Use the All Fossil Average heat rate provided by EPA eGRID for RFC West region for ComEd territory, and SERC Midwest region for Ameren territory. Also include any line losses.

3.412 = Converts COP to HSPF

EF_{GAS EXIST} = Energy Factor (efficiency) of existing gas water heater

= Actual. If unknown assume federal standard³⁵²:

For ≤55 gallons: $0.675 - (0.0015 * \text{tank_size})$

For > 55 gallons $0.8012 - (0.00078 * \text{tank size})$

= If tank size unknown assume 40 gallons and EF_Baseline of 0.615

All other variables provided above

³⁵² Minimum Federal Standard as of 4/1/2015;
<http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf>

Illustrative Examples *[for illustrative purposes a Heat Rate of 10,000 Btu/kWh is used]*

New construction using gas furnace and central AC baseline, *supported by Gas utility only:*

For example, a 3 ton unit with Part Load EER rating of 19 and Part Load COP of 4.4 in single family house in Springfield with a 40 gallon gas water heater is installed in place of a natural gas furnace and 3 ton Central AC unit:

$$\begin{aligned}\Delta \text{kWh} &= 0 \\ \Delta \text{Therms} &= [\text{Heating Savings}] + [\text{DHW Savings}] \\ &= [\text{Replaced gas consumption} - \text{therm equivalent of GSHP source kWh}] + [\text{DHW Savings}] \\ &= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUEbase}) - (\text{kWh to Therm} * \text{FLHheat} * \\ &\quad \text{Capacity_heating} * 1/(\text{COP}_{\text{PL}} * 3.412)/1000))] + [(1 - \text{ElecDHW}) * \% \text{DHWD} \text{Displaced} * (1/ \text{EF}_{\text{GAS}} \\ &\quad \text{EXIST} * \text{GPD} * \text{Household} * 365.25 * \gamma \text{Water} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 100,000]] \\ &= [(1-0) * ((714/0.80) - (10000/100000 * 1754 * 36,000 * 1/(4.4 * 3.412))/1000)] + [(1 - 0) * \\ &\quad (0.44 * (1/ 0.615 * 17.6 * 2.56 * 365.25 * 8.33 * (125-54) * 1) / 100,000)] \\ &= 472 + 70 \\ &= 542 \text{ therms}\end{aligned}$$

Early Replacement fuel switch, *supported by gas and electric utility:*

For example, a 3 ton unit with Part Load EER rating of 19 and Part Load COP of 4.4 in single family house in Springfield with a 40 gallon gas water heater replaces an existing working natural gas furnace and 3 ton Central AC unit with unknown efficiency ratings:

$$\begin{aligned}\Delta \text{kWh for remaining life of existing unit (1st 8 years):} \\ &= [\text{Cooling savings}] + [\text{Heating savings from base ASHP to GSHP}] + [\text{DHW savings}] \\ &= [(\text{FLHcool} * \text{Capacity_cooling} * (1/\text{SEER}_{\text{EXIST}} - (1/\text{EER}_{\text{PL}})/1000) + [(\text{FLHheat} * \\ &\quad \text{Capacity_heating} * (1/\text{HSPF}_{\text{ASHP}} - (1/\text{COP}_{\text{PL}} * 3.412))/1000] + [\text{ElecDHW} * \% \text{DHWD} \text{Displaced} \\ &\quad * (((1/ \text{EF}_{\text{ELEC}}) * \text{GPD} * \text{Household} * 365.25 * \gamma \text{Water} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412))] \\ &= [(730 * 36,000 * (1/8.6 - 1/19)) / 1000] + [(1754 * 36,000 * (1/8.2 - 1/(4.4 * 3.412))) / 1000] \\ &\quad + [0 * 0.44 * (((1/0.904) * 17.6 * 2.56 * 365.25 * 8.33 * (125-54) * 1)/3412)] \\ &= 1673 + 3494 + 0 \\ &= 5167 \text{ kWh}\end{aligned}$$

Continued on next page.

Illustrative Example continued

Δ kWh for remaining measure life (next 17 years):

$$\begin{aligned}
 &= [\text{Cooling savings}] + [\text{Heating savings}] + [\text{DHW savings}] \\
 &= [(\text{FLHcool} * \text{Capacity_cooling} * (1/\text{SEER}_{\text{base}} - (1/\text{EER}_{\text{PL}})/1000) + [(\text{FLHheat} * \text{Capacity_heating} * (1/\text{HSPF}_{\text{ASHP}} - (1/\text{COP}_{\text{PL}} * 3.412))/1000] + [\text{ElecDHW} * \% \text{DHWD} \text{Displaced} * ((1/\text{EF}_{\text{ELEC}}) * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412)] \\
 &= [(730 * 36,000 * (1/13 - 1/19)) / 1000] + [1754 * 36,000 * (1/8.2 - 1/(4.4 * 3.412)) / 1000] + [0 * 0.44 * ((1/0.904) * 17.6 * 2.56 * 365.25 * 8.33 * (125-54) * 1) / 3412)] \\
 &= 638 + 3494 + 0 \\
 &= 4132 \text{ kWh}
 \end{aligned}$$

Δ Therms for remaining life of existing unit (1st 8 years):

$$\begin{aligned}
 &= [\text{Heating Savings}] + [\text{DHW Savings}] \\
 &= [\text{Replaced gas consumption} - \text{therm equivalent of base ASHP source kWh}] + [\text{DHW Savings}] \\
 &= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUE}_{\text{exist}}) - (\text{kWh to Therm} * \text{FLHheat} * \text{Capacity_heating} * 1/\text{HSPF}_{\text{ASHP}})/1000)] + [(1 - \text{ElecDHW}) * \% \text{DHWD} \text{Displaced} * (1/\text{EF}_{\text{GAS EXIST}} * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 100,000)] \\
 &= [(1-0) * ((714/0.644) - (10000/100000 * 1754 * 36,000 * 1/8.2)/1000)] + [(1 - 0) * (0.44 * (1/0.615 * 17.6 * 2.56 * 365.25 * 8.33 * (125-54) * 1) / 100,000)] \\
 &= 339 + 70 \\
 &= 408 \text{ therms}
 \end{aligned}$$

Δ Therms for remaining measure life (next 17 years):

$$\begin{aligned}
 &= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUE}_{\text{baseER}}) - (\text{kWh to Therm} * \text{FLHheat} * \text{Capacity_heating} * 1/\text{HSPF}_{\text{ASHP}})/1000)] + [(1 - \text{ElecDHW}) * \% \text{DHWD} \text{Displaced} * (1/\text{EF}_{\text{GAS EXIST}} * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 100,000)] \\
 &= [(1-0) * ((714/0.9) - (10000/100000 * 1754 * 36,000 * 1/8.2)/1000)] + [(1 - 0) * (0.44 * (1/0.615 * 17.6 * 2.56 * 365.25 * 8.33 * (125-54) * 1) / 100,000)] \\
 &= 23 + 70 \\
 &= 93 \text{ therms}
 \end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

COST EFFECTIVENESS SCREENING AND LOAD REDUCTION FORECASTING WHEN FUEL SWITCHING

This measure can involve fuel switching from gas to electric.

For the purposes of forecasting load reductions due to fuel switch GSHP projects per Section 16-111.5B, changes in

site energy use at the customer's meter (using ΔkWh algorithm below) adjusted for utility line losses (at-the-busbar savings), customer switching estimates, NTG, and any other adjustment factors deemed appropriate, should be used.

The inputs to cost effectiveness screening should reflect the actual impacts on the electric and fuel consumption at the customer meter and, for fuel switching measures, this will not match the output of the calculation/allocation methodology presented in the "Electric Energy Savings" and "Natural Gas Savings" sections above. Therefore in addition to the calculation of savings claimed, the following values should be used to assess the cost effectiveness of the measure.

$$\begin{aligned}\Delta \text{Therms} &= [\text{Heating Consumption Replaced}^{353}] + [\text{DHW Savings if gas}] \\ &= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUEbase}))] + [(1 - \text{ElecDHW}) * \% \text{DHWD} \text{Displaced} \\ &\quad * (1/\text{EF}_{\text{GAS EXIST}} * \text{GPD} * \text{Household} * 365.25 * \gamma \text{Water} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 100,000]] \\ \Delta kWh &= - [\text{GSHP heating consumption}] + [\text{Cooling savings}^{354}] + [\text{DHW savings if electric}] \\ &= - [(\text{FLHheat} * \text{Capacity_heating} * (1/\text{COP}_{\text{PL}} * 3.412))/1000] + [(\text{FLHcool} * \text{Capacity_cooling} * (1/\text{SEERbase} - 1/\text{EER}_{\text{PL}}))/1000] + [\text{ElecDHW} * \% \text{DHWD} \text{Displaced} * \\ &\quad ((1/\text{EF}_{\text{ELEC}} * \text{GPD} * \text{Household} * 365.25 * \gamma \text{Water} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412)]\end{aligned}$$

Illustrative Example of Cost Effectiveness Inputs for Fuel Switching

For example, a 3 ton unit with Part Load EER rating of 19 and Part Load COP of 4.4 in single family house in Springfield with a 40 gallon gas water heater replaces an existing working natural gas furnace and 3 ton Central AC unit with unknown efficiency ratings. [Note the calculation provides the annual savings for the first 8 years of the measure life, an additional calculation (not shown) would be required to calculate the annual savings for the remaining life (years 9-25)]:

$$\begin{aligned}\Delta \text{Therms} &= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUEexist}))] + [(1 - \text{ElecDHW}) * \% \text{DHWD} \text{Displaced} * (1/\text{EF}_{\text{GAS EXIST}} * \text{GPD} * \text{Household} * 365.25 * \gamma \text{Water} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 100,000]] \\ &= [(1-0) * (714/0.644)] + [((1 - 0) * 0.44 * (1/0.615 * 17.6 * 2.56 * 365.25 * 8.33 * (125-54) * 1) / 100,000)] \\ &= 1109 + 70 \\ &= 1179 \text{ therms} \\ \Delta kWh &= - [(\text{FLHheat} * \text{Capacity_heating} * (1/\text{COP}_{\text{PL}} * 3.412))/1000] + [(\text{FLHcool} * \text{Capacity_cooling} * (1/\text{SEERexist} - 1/\text{EER}_{\text{PL}}))/1000] + [\text{ElecDHW} * \% \text{DHWD} \text{Displaced} * (((1/\text{EF}_{\text{ELEC}}) * \text{GPD} * \text{Household} * 365.25 * \gamma \text{Water} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412)] \\ &= - [(1754 * 36,000 * (1/(4.4 * 3.412)))/1000] + [(730 * 36,000 * (1/(8.6 - 1/19)))/1000] + [0 * 0.44 * (((1/0.904) * 17.6 * 2.56 * 365.25 * 8.33 * (125-54) * 1)/3412)] \\ &= -4206 + 1673 + 0 \\ &= -2533 \text{ kWh}\end{aligned}$$

³⁵³ Note AFUEbase in the algorithm should be replaced with AFUEexist for early replacement measures.

³⁵⁴ Note SEERbase in the algorithm should be replaced with SEERexist for early replacement measures.

MEASURE CODE: RS-HVC-GSHP-V07-180101

REVIEW DEADLINE: 1/1/2021

5.3.9 High Efficiency Bathroom Exhaust Fan

DESCRIPTION

This market opportunity is defined by the need for continuous mechanical ventilation due to reduced air-infiltration from a tighter building shell. In retrofit projects, existing fans may be too loud, or insufficient in other ways, to be operated as required for proper ventilation. This measure assumes a fan capacity of 50 CFM rated at a sound level of less than 2.0 sones at 0.1 inches of water column static pressure. This measure may be applied to larger capacity, up to 130 CFM, efficient fans with bi-level controls because the savings and incremental costs are very similar. All eligible installations shall be sized to provide the mechanical ventilation rate indicated by ASHRAE 62.2.

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

New efficient (average CFM/watt of 8.3³⁵⁵) exhaust-only ventilation fan, quiet (< 2.0 sones) Continuous operation in accordance with recommended ventilation rate indicated by ASHRAE 62.2³⁵⁶

DEFINITION OF BASELINE EQUIPMENT

New standard efficiency (average CFM/Watt of 3.1³⁵⁷) exhaust-only ventilation fan, quiet (< 2.0 sones) operating in accordance with recommended ventilation rate indicated by ASHRAE 62.2³⁵⁸

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 19 years³⁵⁹.

DEEMED MEASURE COST

Incremental cost per installed fan is \$43.50 for quiet, efficient fans³⁶⁰.

LOADSHAPE

Loadshape R11 - Residential Ventilation

COINCIDENCE FACTOR

The summer Peak Coincidence Factor is assumed to be 100% because the fan runs continuously.

³⁵⁵ VEIC analysis looking at average efficient fan (i.e. Brushless Permanent Magnet) efficacies at static pressures of 0.1 and 0.25 inches of water column for quiet fans rated for 50 CFM.

³⁵⁶ Bi-level controls may be used by efficient fans larger than 50 CFM

³⁵⁷ VEIC analysis looking at average baseline fan (i.e. non-Brushless Permanent Magnet) efficacies at static pressures of 0.1 and 0.25 inches of water column for quiet fans rated for 50 CFM.

³⁵⁸ On/off cycling controls may be required of baseline fans larger than 50CFM.

³⁵⁹ Conservative estimate based upon GDS Associates Measure Life Report "Residential and C&I Lighting and HVAC measures" 25 years for whole-house fans, and 19 for thermostatically-controlled attic fans.

³⁶⁰ VEIC analysis using cost data collected from wholesale vendor; <http://www.westsidewholesale.com/>.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = (\text{CFM} * (1/\eta_{\text{BASELINE}} - 1/\eta_{\text{EFFICIENT}})/1000) * \text{Hours}$$

Where:

CFM = Nominal Capacity of the exhaust fan
= 50 CFM³⁶¹

η_{BASELINE} = Average efficacy for baseline fan
= 3.1 CFM/Watt³⁶²

$\eta_{\text{EFFICIENT}}$ = Average efficacy for efficient fan
= 8.3 CFM/Watt³⁶³

Hours = assumed annual run hours,
= 8766 for continuous ventilation.

$$\begin{aligned} \Delta \text{kWh} &= (50 * (1/3.1 - 1/8.3)/1000) * 8766 \\ &= 88.6 \text{ kWh} \end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = (\text{CFM} * (1/\eta_{\text{BASELINE}} - 1/\eta_{\text{EFFICIENT}})/1000) * \text{CF}$$

Where:

CF = Summer Peak Coincidence Factor
= 1.0 (continuous operation)

Other variables as defined above

$$\begin{aligned} \Delta \text{kW} &= (50 * (1/3.1 - 1/8.3)/1000) * 1.0 \\ &= 0.0101 \text{ kW} \end{aligned}$$

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

³⁶¹ 50CFM is the closest available fan size to ASHRAE 62.2 Section 4.1 Whole House Ventilation rates based upon typical square footage and bedrooms.

³⁶² VEIC analysis looking at average baseline fan (i.e. non-Brushless Permanent Magnet) efficacies at static pressures of 0.1 and 0.25 inches of water column for quiet fans rated for 50 CFM.

³⁶³ VEIC analysis looking at average efficient fan (i.e. Brushless Permanent Magnet) efficacies at static pressures of 0.1 and 0.25 inches of water column for quiet fans rated for 50 CFM.

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-HVC-BAFA-V01-120601

REVIEW DEADLINE: 1/1/2019

5.3.10 HVAC Tune Up (Central Air Conditioning or Air Source Heat Pump)

DESCRIPTION

This measure involves the measurement of refrigerant charge levels and airflow over the central air conditioning or heat pump unit coil, correction of any problems found and post-treatment re-measurement. Measurements must be performed with standard industry tools and the results tracked by the efficiency program.

Savings from this measure are developed using a reputable Wisconsin study. It is recommended that future evaluation be conducted in Illinois to generate a more locally appropriate characterization.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

N/A

DEFINITION OF BASELINE EQUIPMENT

This measure assumes that the existing unit being maintained is either a residential central air conditioning unit or an air source heat pump that has not been serviced for at least 3 years.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 2 years³⁶⁴.

DEEMED MEASURE COST

If the implementation mechanism involves delivering and paying for the tune up service, the actual cost should be used. If however the customer is provided a rebate and the program relies on private contractors performing the work, the measure cost should be assumed to be \$175³⁶⁵.

LOADSHAPE

Loadshape R08 - Residential Cooling

COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

CF_{SSP} = Summer System Peak Coincidence Factor for Central A/C (during utility peak hour)
= 68%³⁶⁶

CF_{SSP} = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)
= 72%³⁶⁷

³⁶⁴ Based on VEIC professional judgment.

³⁶⁵ Based on personal communication with HVAC efficiency program consultant Buck Taylor or Roltay Inc., 6/21/10, who estimated the cost of tune up at \$125 to \$225, depending on the market and the implementation details.

³⁶⁶ Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

³⁶⁷ Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

$$CF_{PJM} = \text{PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)}$$

$$= 46.6\%^{368}$$

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh_{\text{Central AC}} = (FLH_{\text{cool}} * \text{Capacity}_{\text{cooling}} * (1/SEER_{\text{CAC}}))/1000 * MFe$$

$$\Delta kWh_{\text{Air Source Heat Pump}} = ((FLH_{\text{cool}} * \text{Capacity}_{\text{cooling}} * (1/SEER_{\text{ASHP}}))/1000 * MFe) + (FLH_{\text{heat}} * \text{Capacity}_{\text{heating}} * (1/HSPF_{\text{ASHP}}))/1000 * MFe)$$

Where:

FLHcool = Full load cooling hours
Dependent on location as below:³⁶⁹

Climate Zone (City based upon)	FLHcool Single Family	FLHcool Multifamily
1 (Rockford)	512	467
2 (Chicago)	570	506
3 (Springfield)	730	663
4 (Belleville)	1,035	940
5 (Marion)	903	820
Weighted Average ³⁷⁰	629	564

Capacity_cooling = Cooling capacity of equipment in Btu/hr (note 1 ton = 12,000 Btu/hr)

= Actual

SEER_{CAC} = SEER Efficiency of existing central air conditioning unit receiving maintenance

= Actual. If unknown assume 10 SEER ³⁷¹

MFe = Maintenance energy savings factor

= 0.05³⁷²

SEER_{ASHP} = SEER Efficiency of existing air source heat pump unit receiving maintenance

= Actual. If unknown assume 10 SEER ³⁷³

³⁶⁸ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

³⁶⁹ Based on Full Load Hours from ENERGY Star with adjustments made in a Navigant Evaluation, other cities were scaled using those results and CDD. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

³⁷⁰ Weighted based on number of occupied residential housing units in each zone.

³⁷¹ Use actual SEER rating where it is possible to measure or reasonably estimate. Unknown default of 10 SEER is a VEIC estimate of existing unit efficiency, based on minimum federal standard between the years of 1992 and 2006.

³⁷² Energy Center of Wisconsin, May 2008; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research."

³⁷³ Use actual SEER rating where it is possible to measure or reasonably estimate. Unknown default of 10 SEER is a VEIC estimate of existing unit efficiency, based on minimum federal standard between the years of 1992 and 2006.

FLH_{heat} = Full load heating hours

Dependent on location:³⁷⁴

Climate Zone (City based upon)	FLH _{heat}
1 (Rockford)	2208
2 (Chicago)	2064
3 (Springfield)	1967
4 (Belleville)	1420
5 (Marion)	1445
Weighted Average ³⁷⁵	1821

Capacity_{heating} = Heating capacity of equipment in Btu/hr (note 1 ton = 12,000 Btu/hr)

= Actual

HSPF_{ASHP} = Heating Season Performance Factor of existing air source heat pump unit receiving maintenance

= Actual. If unknown assume 6.8 HSPF ³⁷⁶

For example, maintenance of a 3-ton, SEER 10 air conditioning unit in a single family house in Springfield:

$$\begin{aligned}\Delta \text{kWh}_{\text{CAC}} &= (730 * 36,000 * (1/10))/1000 * 0.05 \\ &= 131 \text{ kWh}\end{aligned}$$

For example, maintenance of a 3-ton, SEER 10, HSPF 6.8 air source heat pump unit in a single family house in Springfield:

$$\begin{aligned}\Delta \text{kWh}_{\text{ASHP}} &= ((730 * 36,000 * (1/10))/1000 * 0.05) + (1967 * 36,000 * (1/6.8))/1000 * 0.05 \\ &= 652 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \text{Capacity}_{\text{cooling}} * (1/\text{EER})/1000 * \text{MFd} * \text{CF}$$

Where:

EER = EER Efficiency of existing unit receiving maintenance in Btu/H/Watts

= Calculate using Actual SEER

$$= -0.02 * \text{SEER}^2 + 1.12 * \text{SEER} \quad ^{377}$$

³⁷⁴ Full load heating hours for heat pumps are provided for Rockford, Chicago and Springfield in the Energy Star Calculator. Estimates for the other locations were calculated based on the FLH to Heating Degree Day (from NCDC) ratio. VEIC consider Energy Star estimates to be high due to oversizing not being adequately addressed. Using average Illinois billing data (from <http://www.icc.illinois.gov/ags/consumereducation.aspx>) VEIC estimated the average gas heating load and used this to estimate the average home heating output (using 83% average gas heat efficiency). Dividing this by a typical 36,000 Btu/hr ASHP gives an estimate of average ASHP FLH_{heat} of 1821 hours. We used the ratio of this value to the average of the locations using the Energy Star data (1994 hours) to scale down the Energy Star estimates. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

³⁷⁵ Weighted based on number of occupied residential housing units in each zone.

³⁷⁶ Use actual HSPF rating where it is possible to measure or reasonably estimate. Unknown default of 6.8 HSPF is a VEIC estimate based on minimum Federal Standard between 1992 and 2006.

³⁷⁷ Based on Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy

MF _d	= Maintenance demand savings factor = 0.02 ³⁷⁸
CF _{SSP}	= Summer System Peak Coincidence Factor for Central A/C (during system peak hour) = 68% ³⁷⁹
CF _{SSP}	= Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour) = 72% ³⁸⁰
CF _{PJM}	= PJM Summer Peak Coincidence Factor for Central A/C and Heat Pumps (average during peak period) = 46.6% ³⁸¹

For example, maintenance of 3-ton, SEER 10 (equals EER 9.2) CAC unit:

$$\begin{aligned}\Delta kW_{SSP} &= 36,000 * 1/(9.2)/1000 * 0.02 * 0.68 \\ &= 0.0532 \text{ kW} \\ \Delta kW_{PJM} &= 36,000 * 1/(9.2)/1000 * 0.02 * 0.466 \\ &= 0.0365 \text{ kW}\end{aligned}$$

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

Conservatively not included.

MEASURE CODE: RS-HVC-TUNE-V03-160601

REVIEW DEADLINE: 1/1/2021

Calculations. Masters Thesis, University of Colorado at Boulder. Note this is appropriate for single speed units only.

³⁷⁸ Based on June 2010 personal conversation with Scott Pigg, author of Energy Center of Wisconsin, May 2008; “Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research” suggesting the average WI unit system draw of 2.8kW under peak conditions, and average peak savings of 50W.

³⁷⁹ Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

³⁸⁰ Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC’s 2010 system peak; ‘Impact and Process Evaluation of Ameren Illinois Company’s Residential HVAC Program (PY5)’.

³⁸¹ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

5.3.11 Programmable Thermostats

DESCRIPTION

This measure characterizes the household energy savings from the installation of a new or reprogramming of an existing Programmable Thermostat for reduced heating energy consumption through temperature set-back during unoccupied or reduced demand times. Because a literature review was not conclusive in providing a defensible source of prescriptive cooling savings from programmable thermostats, cooling savings from programmable thermostats are assumed to be zero for this version of the measure. It is not appropriate to assume a similar pattern of savings from setting a thermostat down during the heating season and up during the cooling season. Note that the EPA's EnergyStar program is developing a new specification for this project category, and if/when evaluation results demonstrate consistent cooling savings, subsequent versions of this measure will revisit this assumption³⁸². Energy savings are applicable at the household level; all thermostats controlling household heat should be programmable and installation of multiple programmable thermostats per home does not accrue additional savings.

This measure was developed to be applicable to the following program types: TOS, NC, RF, DI.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The criteria for this measure are established by replacement of a manual-only temperature control, with one that has the capability to adjust temperature setpoints according to a schedule without manual intervention. This category of equipment is broad and rapidly advancing in regards to the capability, and usability of the controls and their sophistication in setpoint adjustment and information display, but for the purposes of this characterization, eligibility is perhaps most simply defined by what it isn't: a manual only temperature control.

For the thermostat reprogramming measure, the auditor consults with the homeowner to determine an appropriate set back schedule, reprograms the thermostat and educates the homeowner on its appropriate use.

DEFINITION OF BASELINE EQUIPMENT

For new thermostats the baseline is a non-programmable thermostat requiring manual intervention to change temperature setpoint.

For the purpose of thermostat reprogramming, an existing programmable thermostat that an auditor determines is being used in override mode or otherwise effectively being operated like a manual thermostat.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life of a programmable thermostat is assumed to be 10 years³⁸³ based upon equipment life only³⁸⁴. For the purposes of claiming savings for a new programmable thermostat, this is reduced by a 50% persistence factor to give final measures life of 5 years. For reprogramming, this is reduced further to give a measure life of 2 years.

DEEMED MEASURE COST

Actual material and labor costs should be used if the implementation method allows. If unknown (e.g. through a

³⁸² The EnergyStar program discontinued its support for this measure category effective 12/31/09, and is presently developing a new specification for 'Residential Climate Controls'.

³⁸³ Table 1, HVAC Controls, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007

³⁸⁴ Future evaluation is strongly encouraged to inform the persistence of savings to further refine measure life assumption. As this characterization depends heavily upon a large scale but only 2-year study of the energy impacts of programmable thermostats, the longer term impacts should be assessed.

retail program) the capital cost for the new installation measure is assumed to be \$30³⁸⁵. The cost for reprogramming is assumed to be \$10 to account for the auditors time to reprogram and educate the homeowner.

LOADSHAPE

Loadshape R09 - Residential Electric Space Heat

COINCIDENCE FACTOR

N/A due to no savings attributable to cooling during the summer peak period.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh}^{386} = \% \text{ElectricHeat} * \text{Elec_Heating_Consumption} * \text{Heating_Reduction} * \text{HF} * \text{Eff_ISR} + (\Delta \text{Therms} * F_e * 29.3)$$

Where:

$\% \text{ElectricHeat}$ = Percentage of heating savings assumed to be electric

Heating fuel	$\% \text{ElectricHeat}$
Electric	100%
Natural Gas	0%
Unknown	6.5% ³⁸⁷

$\text{Elec_Heating_Consumption}$

= Estimate of annual household heating consumption for electrically heated single-family homes³⁸⁸. If location and heating type is unknown, assume 15,678 kWh³⁸⁹

³⁸⁵ Market prices vary significantly in this category, generally increasing with thermostat capability and sophistication. The basic functions required by this measure's eligibility criteria are available on units readily available in the market for the listed price.

³⁸⁶ Note the second part of the algorithm relates to furnace fan savings if the heating system is Natural Gas.

³⁸⁷ Assumes that half of the electric heat in the state is a heat pump able to be controlled by an advanced thermostat (consistent with Potential Study results from the state). Average value of 13% electric space heating from 2010 Residential Energy Consumption Survey for Illinois. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used.

³⁸⁸ Values in table are based on converting an average household heating load (834 therms) for Chicago based on 'Table E-1, Energy Efficiency/Demand Response Nicor Gas Plan Year 1: Research Report: Furnace Metering Study, Draft, Navigant, August 1 2013 to an electric heat load (divide by 0.03413) to electric resistance and ASHP heat load (resistance load reduced by 15% to account for distribution losses that occur in furnace heating but not in electric resistance while ASHP heat is assumed to suffer from similar distribution losses) and then to electric consumption assuming efficiencies of 100% for resistance and 200% for HP (see 'Household Heating Load Summary Calculations_11062013.xls'). Finally these values were adjusted to a statewide average using relative HDD assumptions to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

³⁸⁹ Assumption that 1/2 of electrically heated homes have electric resistance and 1/2 have Heat Pump, based on 2010 Residential Energy Consumption Survey for Illinois.

Climate Zone (City based upon)	Electric Resistance Elec_Heating_ Consumption (kWh)	Electric Heat Pump Elec_Heating_ Consumption (kWh)
1 (Rockford)	21,741	12,789
2 (Chicago)	20,771	12,218
3 (Springfield)	17,789	10,464
4 (Belleville)	13,722	8,072
5 (Marion)	13,966	8,215
Average	19,743	11,613

Heating_Reduction = Assumed percentage reduction in total household heating energy consumption due to programmable thermostat
= 6.2%³⁹⁰

HF = Household factor, to adjust heating consumption for non-single-family households.

Household Type	HF
Single-Family	100%
Multi-Family	65% ³⁹¹
Actual	Custom ³⁹²

Eff_ISR = Effective In-Service Rate, the percentage of thermostats installed and programmed effectively

Program Delivery	Eff_ISR
Direct Install	100%
Other, or unknown	56% ³⁹³

Δ Therms = Therm savings if Natural Gas heating system
= See calculation in Natural Gas section below

F_e = Furnace Fan energy consumption as a percentage of annual fuel consumption
= 3.14%³⁹⁴

³⁹⁰ The savings from programmable thermostats are highly susceptible to many factors best addressed, so far for this category, by a study that controlled for the most significant issues with a very large sample size. To the extent that the treatment group is representative of the program participants for IL, this value is suitable. Higher and lower values would be justified based upon clear dissimilarities due to program and product attributes. Future evaluation work should assess program specific impacts associated with penetration rates, baseline levels, persistence, and other factors which this value represents.

³⁹¹ Multifamily household heating consumption relative to single-family households is affected by overall household square footage and exposure to the exterior. This 65% factor is applied to MF homes based on professional judgment that average household size, and heat loads of MF households are smaller than single-family homes

³⁹² Program-specific household factors may be utilized on the basis of sufficiently validated program evaluations.

³⁹³ "Programmable Thermostats. Report to KeySpan Energy Delivery on Energy Savings and Cost Effectiveness," GDS Associates, Marietta, GA. 2002GDS

³⁹⁴ F_e is not one of the AHRI certified ratings provided for residential furnaces, but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the Energy Star version 3 criteria for 2% F_e. See

29.3 = kWh per therm

For example, a programmable thermostat directly installed in an electric resistance heated, single-family home in Springfield:

$$\Delta \text{kWh} = 1 * 17,789 * 0.062 * 100\% * 100\% + (0 * 0.0314 * 29.3)$$

$$= 1,103 \text{ kWh}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A due to no savings from cooling during the summer peak period.

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms} = \% \text{FossilHeat} * \text{Gas_Heating_Consumption} * \text{Heating_Reduction} * \text{HF} * \text{Eff_ISR}$$

Where:

$\% \text{FossilHeat}$ = Percentage of heating savings assumed to be Natural Gas

Heating fuel	$\% \text{FossilHeat}$
Electric	0%
Natural Gas	100%
Unknown	93.5% ³⁹⁵

$\text{Gas_Heating_Consumption}$

= Estimate of annual household heating consumption for gas heated single-family homes. If location is unknown, assume the average below³⁹⁶.

Climate Zone (City based upon)	$\text{Gas_Heating_Consumption}$ (therms)
1 (Rockford)	1,052
2 (Chicago)	1,005
3 (Springfield)	861
4 (Belleville)	664
5 (Marion)	676
Average	955

“Programmable Thermostats Furnace Fan Analysis.xlsx” for reference.

³⁹⁵ Assumes that half of the electric heat in the state is a heat pump able to be controlled by an advanced thermostat. Data from 2010 Residential Energy Consumption Survey for Illinois. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used.

³⁹⁶ Values are based on adjusting the average household heating load (834 therms) for Chicago based on ‘Table E-1, Energy Efficiency / Demand Response Nicor Gas Plan Year 1, Research Report: Furnace Metering Study’, divided by standard assumption of existing unit efficiency of 83% (estimate based on 24% of furnaces purchased in Illinois were condensing in 2000 (based on data from GAMA, provided to Department of Energy), assuming typical efficiencies: $(0.24 * 0.92) + (0.76 * 0.8) = 0.83$) to give 1005 therms. This Chicago value was then adjusted to a statewide average using relative HDD assumptions to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city’s HDD.

For example, a programmable thermostat directly-installed in a gas heated single-family home in Chicago:

$$\begin{aligned}\Delta\text{Therms} &= 1.0 * 1005 * 0.062 * 100\% * 100\% \\ &= 62.3 \text{ therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-HVC-PROG-V04-180101

REVIEW DEADLINE: 1/1/2021

5.3.12 Ductless Heat Pumps

DESCRIPTION

This measure is designed to calculate electric savings for the installation of a ductless mini-split heat pump (DMSHP). DMSHPs save energy in heating mode because they provide heat more efficiently than electric resistance heat and central ASHP systems. Additionally, DMSHPs use less fan energy to move heat and don't incur heat loss through a duct distribution system.

For cooling, the proposed savings calculations are aligned with those of typical replacement systems. DMSHPs save energy in cooling mode because they provide cooling capacity more efficiently than other types of unitary cooling equipment. A DMSHP installed in a home with a central ASHP system will save energy by offsetting some of the cooling energy of the ASHP. In order for this measure to apply, the control strategy for the heat pump is assumed to be chosen to maximize savings per installer recommendation.³⁹⁷

This measure characterizes the following scenarios:

- d) New Construction:
 - a. The installation of a new DMSHP meeting efficiency standards required by the program in a new home.
 - b. Note the baseline in this case should be determined via EM&V and the algorithms are provided to allow savings to be calculated from any baseline condition.
- e) Time of Sale:
 - a. The planned installation of a new DMSHP meeting efficiency standards required by the program to replace an existing system(s) that does not meet the criteria for early replacement described in section c below.
 - b. Note the baseline in this case is an equivalent replacement system to that which exists currently in the home. The calculation of savings is dependent on whether an incentive for the installation has been provided by both a gas and electric utility, just an electric utility or just a gas utility.
- f) Early Replacement/Retrofit:
 - a. The early removal or displacement of functioning either electric or gas space heating and/or cooling systems from service, prior to the natural end of life, and replacement with a new DMSHP.
 - b. Note the baseline in this case is the existing equipment being replaced/displaced. The calculation of savings is dependent on whether an incentive for the installation has been provided by both a gas and electric utility, just an electric utility or just a gas utility.
 - c. Early Replacement determination will be based on meeting the following conditions:
 - The existing unit is operational when replaced/displaced, or
 - The existing unit requires minor repairs, defined as costing less than³⁹⁸:

Existing System	Maximum repair cost
Air Source Heat Pump	\$276 per ton
Central Air Conditioner	\$190 per ton

³⁹⁷ The whole purpose of installing ductless heat pumps is to conserve energy, so the installer can be assumed to be capable of recommending an appropriate controls strategy. For most applications, the heating setpoint for the ductless heat pump should be at least 2F higher than any remaining existing system and the cooling setpoint for the ductless heat pump should be at least 2F cooler than the existing system (this should apply to all periods of a programmable schedule, if applicable). This helps ensure that the ductless heat pump will be used to meet as much of the load as possible before the existing system operates to meet the remaining load. Ideally, the new ductless heat pump controls should be set to the current comfort settings, while the existing system setpoints should be adjusted down (heating) and up (cooling) to capture savings.

³⁹⁸ The Technical Advisory Committee agreed that if the cost of repair is less than 20% of the new baseline replacement cost it can be considered early replacement.

Existing System	Maximum repair cost
Boiler	\$709
Furnace	\$528
Ground Source Heat Pump	<\$249 per ton

- All other conditions will be considered Time of Sale.
- d. The Baseline efficiency of the existing unit replaced:
- If the efficiency of the existing unit is less than the maximum shown below, the Baseline efficiency is the actual efficiency value of the unit replaced. If the efficiency is greater than the maximum, the Baseline efficiency is shown in the “New Baseline” column below:

Existing System	Maximum efficiency for Actual	New Baseline ³⁹⁹
Air Source Heat Pump	10 SEER	14 SEER
Central Air Conditioner	10 SEER	13 SEER
Boiler	75% AFUE	82% AFUE
Furnace	75% AFUE	80% AFUE
Ground Source Heat Pump	10 SEER	13 SEER

- If the efficiency of the existing unit is unknown, use assumptions in variable list below (SEER, HSPF or AFUE exist).
- If the operational status or repair cost of the existing unit is unknown use time of sale assumptions.

This measure was developed to be applicable to the following program types: RF, TOS, NC, EREP.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the new equipment must be a high-efficiency, variable-capacity (typically “inverter-driven” DC motor) ductless heat pump system that exceeds the program minimum efficiency requirements.

DEFINITION OF BASELINE EQUIPMENT

For these products, baseline equipment includes Air Conditioning and Space Heating:

New Construction:

To calculate savings with an electric baseline, the baseline equipment is assumed to be an Air Source Heat Pump meeting the Federal Standard efficiency level; 14 SEER, 8.2 HSPF and 11.8⁴⁰⁰ EER.

To calculate savings with a furnace/central AC baseline, the baseline equipment is assumed to be an 80% AFUE Furnace and central AC meeting the Federal Standard efficiency level; 13 SEER, 11 EER.

Time of Sale: The baseline for this measure is a new replacement unit of the same system type as the existing unit, meeting the baselines provided below.

Unit Type	Efficiency Standard
ASHP	14 SEER, 11.8 EER, 8.2 HSPF

³⁹⁹ Based on relevant Federal Standards.

⁴⁰⁰ The Federal Standard does not include an EER requirement, so it is approximated with this formula: $(-0.02 * SEER^2) + (1.12 * SEER)$ Wassmer, M. (2003). ‘A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations’ Masters Thesis, University of Colorado at Boulder.

Unit Type	Efficiency Standard
Gas Furnace	80% AFUE
Gas Boiler	82% AFUE
Central AC	13 SEER, 11 EER

Early replacement / Retrofit: The baseline for this measure is the efficiency of the *existing* heating and cooling equipment for the assumed remaining useful life of the existing unit and a new baseline heating and cooling system for the remainder of the measure life (as provided in table above except for Gas Furnace where new baseline assumption is 90% due to pending standard change). Note that in order to claim cooling savings, there must be an existing air conditioning system.

For multifamily buildings, each residence must have existing individual heating equipment. Multifamily residences with central heating do not qualify for this characterization.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 18 years⁴⁰¹.

For early replacement, the remaining life of existing equipment is assumed to be 6 years⁴⁰².

DEEMED MEASURE COST

New Construction and Time of Sale: The actual installed cost of the DMSHP should be used (defaults are provided below), minus the assumed installation cost of the baseline equipment (\$1,381 per ton for ASHP⁴⁰³ or \$2,011 for a new baseline 80% AFUE furnace or \$3,543 for a new 82% AFUE boiler⁴⁰⁴ and \$952 per ton⁴⁰⁵ for new baseline Central AC replacement).

Unit Size	Full Install Cost ⁴⁰⁶
1-Ton	\$3,000
1.5-Ton	\$3,750
2-Ton	\$4,500
2.5 – Ton	\$5,313
3-Ton	\$6,188

Early Replacement/retrofit (replacing existing equipment): The full installation cost of the DMSHP should be used (default provided above). The assumed deferred cost (after 8 years) of replacing existing equipment with a new baseline unit is assumed to be \$1,518 per ton for a new baseline Air Source Heat Pump, or \$2,903 for a new baseline 90% AFUE furnace or \$4,045 for a new 82% AFUE boiler and \$1,047 per ton for new baseline Central AC replacement⁴⁰⁷. This future cost should be discounted to present value using the nominal societal discount rate.

Where the DMSHP is a supplemental HVAC system, the full installation cost of the DMSHP should be used (default

⁴⁰¹ Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007

⁴⁰² Assumed to be one third of effective useful life

⁴⁰³ Based on data provided on Home Advisor website, providing national average ASHP cost based on 2465 cost submittals.
<http://www.homeadvisor.com/cost/heating-and-cooling/install-a-heat-pump/>

⁴⁰⁴ Furnace and boiler costs are based on data provided in Appendix E of the Appliance Standards Technical Support Documents including equipment cost and installation labor
http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/appendix_e.pdf). Where efficiency ratings are not provided, the values are interpolated from those that are.

⁴⁰⁵ Based on 3 ton initial cost estimate for a conventional unit from ENERGY STAR Central AC calculator
http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Calc_CAC.xls).

⁴⁰⁷ All baseline replacement costs are consistent with their respective measures and include inflation rate of 1.91%.

provided above) without a deferred replacement cost.

LOADSHAPE

Loadshape R10 - Residential Electric Heating and Cooling (if replacing gas heat and central AC)⁴⁰⁸

Loadshape R09 - Residential Electric Space Heat (if replacing electric heat with no cooling)

Loadshape R10 - Residential Electric Heating and Cooling (if replacing ASHP)

Note for purpose of cost effectiveness screening a fuel switch scenario, the heating kWh increase and cooling kWh decrease should be calculated separately such that the appropriate loadshape (i.e. Loadshape R09 - Residential Electric Space Heat and Loadshape R08 – Residential Cooling respectively) can be applied.

COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in four different ways below. The first two relate to the use of DMSHP to supplement existing cooling or provide limited zonal cooling, the second two relate to use of the DMSHP to provide whole house cooling. In each pair, the first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market. Both values provided are based on metering data for 40 DMSHPs in Ameren Illinois service territory⁴⁰⁹.

For supplemental or limited zonal cooling:

CF_{SSP} = Summer System Peak Coincidence Factor for DMSHP (during utility peak hour)
= 43.1%⁴¹⁰

CF_{PJM} = PJM Summer Peak Coincidence Factor for DMSHP (average during PJM peak period)
= 28.0%⁴¹¹

For whole house cooling:

CF_{SSP} = Summer System Peak Coincidence Factor for Heat Pumps (during utility peak hour)
= 72%⁴¹²

CF_{PJM} = PJM Summer Peak Coincidence Factor for Heat Pumps (average during PJM peak period)
= 46.6%⁴¹³

⁴⁰⁸ The baseline for calculating electric savings is an Air Source Heat Pump.

⁴⁰⁹ All-Electric Homes PY6 Metering Results: Multifamily HVAC Systems, Cadmus, October 2015

⁴¹⁰ Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

⁴¹¹ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

⁴¹² Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

<http://www.icc.illinois.gov/downloads/public/edocket/368522.pdf>

⁴¹³ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

Algorithms

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

New Construction and Time of Sale (non-fuel switch only):

$$\begin{aligned}\Delta \text{kWh} &= [\text{Heating Savings}] + [\text{Cooling Savings}] \\ &= [(\text{Elecheat} * \text{Capacity}_{\text{heat}} * \text{EFLH}_{\text{heat}} * (1/\text{HSPF}_{\text{Base}} - 1/\text{HSPF}_{\text{ee}})) / 1000] + [(\text{Capacity}_{\text{cool}} * \text{EFLH}_{\text{cool}} * (1/\text{SEER}_{\text{Base}} - 1/\text{SEER}_{\text{ee}})) / 1000]\end{aligned}$$

New Construction and Time of Sale (fuel switch only):

If measure is supported by gas utility only, $\Delta \text{kWh} = 0$

If measure is supported by gas and electric utility or electric utility only, electric utility claim savings calculated below:

$$\begin{aligned}\Delta \text{kWh} &= [\text{Heating Savings from base ASHP to DMSHP}] + [\text{Cooling Savings}] \\ &= [(\text{Capacity}_{\text{heat}} * \text{EFLH}_{\text{heat}} * (1/\text{HSPF}_{\text{ASHP}} - 1/\text{HSPF}_{\text{ee}})) / 1000] + [(\text{Capacity}_{\text{cool}} * \text{EFLH}_{\text{cool}} * (1/\text{SEER}_{\text{Base}} - 1/\text{SEER}_{\text{ee}})) / 1000]\end{aligned}$$

Early replacement (non-fuel switch only)⁴¹⁴:

ΔkWh for remaining life of existing unit (1st 6 years):

$$\begin{aligned}\Delta \text{kWh} &= [\text{Heating Savings}] + [\text{Cooling Savings}] \\ &= [(\text{Elecheat} * \text{Capacity}_{\text{heat}} * \text{EFLH}_{\text{heat}} * (1/\text{HSPF}_{\text{Exist}} - 1/\text{HSPF}_{\text{ee}})) / 1000] + [(\text{Capacity}_{\text{cool}} * \text{EFLH}_{\text{cool}} * (1/\text{SEER}_{\text{Exist}} - 1/\text{SEER}_{\text{ee}})) / 1000]\end{aligned}$$

ΔkWh for remaining measure life (next 12 years):

$$\begin{aligned}\Delta \text{kWh} &= [\text{Heating Savings}] + [\text{Cooling Savings}] \\ &= [(\text{Elecheat} * \text{Capacity}_{\text{heat}} * \text{EFLH}_{\text{heat}} * (1/\text{HSPF}_{\text{Base}} - 1/\text{HSPF}_{\text{ee}})) / 1000] + [(\text{Capacity}_{\text{cool}} * \text{EFLH}_{\text{cool}} * (1/\text{SEER}_{\text{Base}} - 1/\text{SEER}_{\text{ee}})) / 1000]\end{aligned}$$

Early replacement - fuel switch only :

If measure is supported by gas utility only, $\Delta \text{kWh} = 0$

If measure is supported by gas and electric utility or electric utility only, electric utility claim savings calculated below:

ΔkWh for remaining life of existing unit (1st 6 years):

$$\begin{aligned}\Delta \text{kWh} &= [\text{Heating Savings from base ASHP to DMSHP}] + [\text{Cooling Savings}] \\ &= [(\text{Capacity}_{\text{heat}} * \text{EFLH}_{\text{heat}} * (1/\text{HSPF}_{\text{ASHP}} - 1/\text{HSPF}_{\text{ee}})) / 1000] + [(\text{Capacity}_{\text{cool}} * \text{EFLH}_{\text{cool}} * (1/\text{SEER}_{\text{Exist}} - 1/\text{SEER}_{\text{ee}})) / 1000]\end{aligned}$$

ΔkWh for remaining measure life (next 12 years):

$$\Delta \text{kWh} = [\text{Heating Savings from base ASHP to DMSHP}] + [\text{Cooling Savings}]$$

⁴¹⁴ The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a “number of years to adjustment” and “savings adjustment” input which would be the (new base to efficient savings)/(existing to efficient savings).

$$= [(Capacity_{heat} * EFLH_{heat} * (1/HSPF_{ASHP} - 1/HSPF_{ee})) / 1000] + [(Capacity_{cool} * EFLH_{cool} * (1/SEER_{Base} - 1/SEER_{ee})) / 1000]$$

Where:

- ElecHeat** = 1 if existing building is electrically heated
= 0 if existing building is not electrically heated
- Capacity_{heat}** = Heating capacity of the ductless heat pump unit in Btu/hr
= Actual
- EFLH_{heat}** = Equivalent Full Load Hours for heating. Depends on location. See table below

Climate Zone (City based upon)	EFLH _{heat} ⁴¹⁵
1 (Rockford)	1,520
2 (Chicago)	1,421
3 (Springfield)	1,347
4 (Belleville)	977
5 (Marion)	994
Weighted Average	1,406

- HSPF_{base}** = Heating System Performance Factor of new replacement baseline heating system (kBtu/kWh)

Existing Heating System	HSPF _{base}
Air Source Heat Pump	8.2
Electric Resistance	3.41 ⁴¹⁶

- HSPF_{exist}** = HSPF rating of existing equipment (kbtu/kwh)
= Use actual HSPF rating where it is possible to measure or reasonably estimate. If unknown assume default:

Existing Equipment Type	HSPF _{exist}
Electric resistance heating	3.41 ⁴¹⁷
Air Source Heat Pump	5.44 ⁴¹⁸

- HSPF_{ASHP}** = Heating Season Performance Factor for new ASHP baseline unit (for fuel switch)
= 8.2⁴¹⁹

⁴¹⁵ All-Electric Homes PY6 Metering Results: Multifamily HVAC Systems, Cadmus, October 2015. FLH values are based on metering of multi-family units that were used as the primary heating source to the whole home, and in buildings that had received weatherization improvements. A DMSHP installed in a single-family home may be used more sporadically, especially if the DMSHP serves only a room, and buildings that have not been weatherized may require longer hours. Additional evaluation is recommended to refine the EFLH assumptions for the general population.

⁴¹⁶ Electric resistance has a COP of 1.0 which equals 1/0.293 = 3.41 HSPF.

⁴¹⁷ Electric resistance has a COP of 1.0 which equals 1/0.293 = 3.41 HSPF.

⁴¹⁸ This is from the ASHP measure which estimated HSPF based on finding the average HSPF/SEER ratio from the AHRI directory data (using the least efficient models – SEER 12 and SEER 13) – 0.596, and applying to the average nameplate SEER rating of all Early Replacement qualifying equipment in Ameren PY3-PY4. This estimation methodology appears to provide a result within 10% of actual HSPF.

⁴¹⁹ Minimum Federal Standard as of 1/1/2015;

<http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf>

$HSPF_{ee}$ = HSPF rating of new equipment (kbtu/kwh)

= Actual installed

$Capacity_{cool}$ = the cooling capacity of the ductless heat pump unit in Btu/hr⁴²⁰.

= Actual installed

$SEER_{base}$ = SEER Efficiency of new replacement baseline unit

Existing Cooling System	SEER _{base}
Air Source Heat Pump	14 ⁴²¹
Central AC	13 ⁴²²
No central cooling	13 ⁴²³

$SEER_{ee}$ = SEER rating of new equipment (kbtu/kwh)

= Actual installed⁴²⁴

$SEER_{exist}$ = SEER rating of existing equipment (kbtu/kwh)

= Use actual value. If unknown, see table below

Existing Cooling System	SEER _{exist} ⁴²⁵
Air Source Heat Pump	9.12
Central AC	8.60
Room AC	8.0 ⁴²⁶
No existing cooling ⁴²⁷	Make '1/SEER _{exist} ' = 0

$EFLH_{cool}$ = Equivalent Full Load Hours for cooling. Depends on location. See table below⁴²⁸.

Climate Zone (City based upon)	EFLH _{cool}
1 (Rockford)	323
2 (Chicago)	308
3 (Springfield)	468
4 (Belleville)	629
5 (Marion)	549

⁴²⁰ 1 Ton = 12 kbtu/hr

⁴²¹ Minimum Federal Standard as of 1/1/2015;

<http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf>

⁴²² Minimum Federal Standard; Federal Register, Vol. 66, No. 14, Monday, January 22, 2001/Rules and Regulations, p. 7170-7200.

⁴²³ Assumes that the decision to replace existing systems includes desire to add cooling.

⁴²⁴ Note that if only an EER rating is available, use the following conversion equation; $EER_{base} = (-0.02 * SEER_{base}^2) + (1.12 * SEER)$. From Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder.

⁴²⁵ Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

⁴²⁶ Estimated by converting the EER assumption using the conversion equation; $EER_{base} = (-0.02 * SEER_{base}^2) + (1.12 * SEER)$. From Wassmer, M. (2003). 'A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations', Masters Thesis, University of Colorado at Boulder.

⁴²⁷ If there is no existing cooling in place but the incentive encourages installation of a new DMSHP with cooling, the added cooling load should be subtracted from any heating benefit.

⁴²⁸ *All-Electric Homes PY6 Metering Results: Multifamily HVAC Systems*, Cadmus, October 2015. FLH values are based on metering of multi-family units, and in buildings that had received weatherization improvements. Additional evaluation is recommended to refine the EFLH assumptions for the general population.

Climate Zone (City based upon)	EFLH _{cool}
Weighted Average ⁴²⁹	364

For example, installing a 1.5-ton (heating and cooling capacity) ductless heat pump unit rated at 8 HSPF and 14 SEER in a single-family home in Chicago to displace electric baseboard heat and replace a window air conditioner of unknown efficiency, savings are:

$$\begin{aligned}\Delta \text{kWh}_{\text{heat}} &= (18000 * 1421 * (1/3.412 - 1/8))/1000 = 4,299 \text{ kWh} \\ \Delta \text{kWh}_{\text{cool}} &= (18000 * 308 * (1/8.0 - 1/14))/1000 = 297 \text{ kWh} \\ \Delta \text{kWh} &= 4,299 + 297 = 4,596 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

New Construction and Time of Sale:

$$\Delta \text{kW} = (\text{Capacity}_{\text{cool}} * (1/\text{EER}_{\text{base}} - 1/\text{EER}_{\text{ee}})) / 1000 * \text{CF}$$

Early replacement:

ΔkW for remaining life of existing unit (1st 6 years):

$$\Delta \text{kW} = (\text{Capacity}_{\text{cool}} * (1/\text{EER}_{\text{exist}} - 1/\text{EER}_{\text{ee}})) / 1000 * \text{CF}$$

ΔkW for remaining measure life (next 12 years):

$$\Delta \text{kW} = (\text{Capacity}_{\text{cool}} * (1/\text{EER}_{\text{base}} - 1/\text{EER}_{\text{ee}})) / 1000 * \text{CF}$$

Where:

EER_{base} = EER Efficiency of new replacement unit

Existing Cooling System	EER _{base}
Air Source Heat Pump	11.8 ⁴³⁰
Central AC	11 ⁴³¹
No central cooling	11 ⁴³²

EER_{exist} = Energy Efficiency Ratio of existing cooling system (kBtu/hr / kW)

= Use actual EER rating where it is possible to measure or reasonably estimate. If EER unknown but SEER available convert using the equation:

$$\text{EER}_{\text{exist}} = (-0.02 * \text{SEER}_{\text{exist}}^2) + (1.12 * \text{SEER}_{\text{exist}}) \quad 433$$

If SEER rating unavailable use:

⁴²⁹ Weighted based on number of residential occupied housing units in each zone.

⁴³⁰ The Federal Standard does not include an EER requirement, so it is approximated with the conversion formula from Wassmer, M. 2003 thesis referenced below.

⁴³¹ Minimum Federal Standard; Federal Register, Vol. 66, No. 14, Monday, January 22, 2001/Rules and Regulations, p. 7170-7200.

⁴³² Assumes that the decision to replace existing systems includes desire to add cooling.

⁴³³ From Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder.

Existing Cooling System	EER_exist
Air Source Heat Pump	8.55 ⁴³⁴
Central AC	8.15 ⁴³⁵
Room AC	7.7 ⁴³⁶
No existing cooling ⁴³⁷	Make '1/EER_exist' = 0

EER_ee = Energy Efficiency Ratio of new ductless Air Source Heat Pump (kBtu/hr / kW)
= Actual, If not provided convert SEER to EER using this formula: ⁴³⁸

$$= (-0.02 * SEER^2) + (1.12 * SEER)$$

For supplemental or limited zonal cooling:

CF_{SSP} = Summer System Peak Coincidence Factor for DMSHP (during utility peak hour)
= 43.1%⁴³⁹

CF_{PJM} = PJM Summer Peak Coincidence Factor for DMSHP (average during PJM peak period)
= 28.0%⁴⁴⁰

For whole house cooling:

CF_{SSP} = Summer System Peak Coincidence Factor for Heat Pumps (during utility peak hour)
= 72%⁴⁴¹

CF_{PJM} = PJM Summer Peak Coincidence Factor for Heat Pumps (average during PJM peak period)
= 46.6%⁴⁴²

NATURAL GAS SAVINGS

New Construction and Time of Sale with baseline gas heat:

If measure is supported by gas utility only, gas utility claim savings calculated below:

$\Delta\text{Therms} = [\text{Heating Savings}]$

$$= [\text{Replaced gas consumption} - \text{therm equivalent of DMSHP source kWh}]$$

$$= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUEbase}) - (\text{kWh to Therm} * \text{Capacity}_{\text{heat}} * \text{Capacity}_{\text{cool}}))]$$

⁴³⁴ Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

⁴³⁵ Ibid.

⁴³⁶ Same EER as Window AC recycling. Based on Nexus Market Research Inc, RLW Analytics, December 2005; "Impact, Process, and Market Study of the Connecticut Appliance Retirement Program: Overall Report."

⁴³⁷ If there is no central cooling in place but the incentive encourages installation of a new DMSHP with cooling, the added cooling load should be subtracted from any heating benefit.

⁴³⁸ Based on Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder. Note this is appropriate for single speed units only.

⁴³⁹ Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

⁴⁴⁰ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

⁴⁴¹ Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

<http://www.icc.illinois.gov/downloads/public/edocket/368522.pdf>

⁴⁴² Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

$$EFLH_{\text{heat}} * 1/(HSPF_{\text{ee}})/1000)]$$

If measure is supported by electric utility only, $\Delta\text{Therms} = 0$

If measure is supported by gas and electric utility, gas utility claim savings calculated below, (electric savings is provided in Electric Energy Savings section):

$$\begin{aligned}\Delta\text{Therms} &= [\text{Heating Savings}] \\ &= [\text{Replaced gas consumption} - \text{therm equivalent of base ASHP source kWh}] \\ &= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUE}_{\text{base}}) - (\text{kWh to Therm} * \text{Capacity}_{\text{heat}} * \\ &\quad EFLH_{\text{heat}} * 1/(HSPF_{\text{ASHP}})/1000)]\end{aligned}$$

Early replacement for homes with existing gas heat:

If measure is supported by gas utility only, gas utility claim savings calculated below:

$$\begin{aligned}\Delta\text{Therms for remaining life of existing unit (1st 6 years):} \\ &= [\text{Heating Savings}] \\ &= [\text{Replaced gas consumption} - \text{therm equivalent of DMSHP source kWh}] \\ &= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUE}_{\text{exist}}) - (\text{kWh to Therm} * \text{Capacity}_{\text{heat}} * \\ &\quad EFLH_{\text{heat}} * 1/(HSPF_{\text{ee}})/1000)] \\ \Delta\text{Therms for remaining measure life (next 12 years):} \\ &= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUE}_{\text{baseER}}) - (\text{kWh to Therm} * \text{Capacity}_{\text{heat}} * \\ &\quad EFLH_{\text{heat}} * 1/(HSPF_{\text{ee}})/1000)]\end{aligned}$$

If measure is supported by electric utility only, $\Delta\text{Therms} = 0$

If measure is supported by gas and electric utility, gas utility claim savings calculated below:

$$\begin{aligned}\Delta\text{Therms for remaining life of existing unit (1st 6 years):} \\ \Delta\text{Therms} &= [\text{Heating Savings}] \\ &= [\text{Replaced gas consumption} - \text{therm equivalent of base ASHP source kWh}] \\ &= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUE}_{\text{exist}}) - (\text{kWh to Therm} * \text{Capacity}_{\text{heat}} * \\ &\quad EFLH_{\text{heat}} * 1/(HSPF_{\text{ASHP}})/1000)] \\ \Delta\text{Therms for remaining measure life (next 12 years):} \\ &= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUE}_{\text{baseER}}) - (\text{kWh to Therm} * \text{Capacity}_{\text{heat}} * \\ &\quad EFLH_{\text{heat}} * 1/(HSPF_{\text{ASHP}})/1000)]\end{aligned}$$

Where:

ElecHeat = 1 if existing building is electrically heated
= 0 if existing building is not electrically heated

Gas_Heating_Load
= Estimate of annual household heating load ⁴⁴³ for gas furnace heated single-family homes. If location is unknown, assume the average below.
= Actual if informed by site-specific load calculations, ACCA Manual J or equivalent⁴⁴⁴.

⁴⁴³ Heating load is used to describe the household heating need, which is equal to (gas consumption * AFUE)

⁴⁴⁴ The Air Conditioning Contractors of America Manual J, Residential Load Calculation 8th Edition produces equipment sizing

Climate Zone (City based upon)	Gas_Heating_Load if Furnace (therms) ⁴⁴⁵	Gas_Heating_Load if Boiler (therms) ⁴⁴⁶
1 (Rockford)	873	1275
2 (Chicago)	834	1218
3 (Springfield)	714	1043
4 (Belleville)	551	805
5 (Marion)	561	819
Average	793	1158

AFUEbase = Baseline Annual Fuel Utilization Efficiency Rating

= 80% if furnace and 82% if boiler.

AFUEexist = Existing Annual Fuel Utilization Efficiency Rating

= Use actual AFUE rating where it is possible to measure or reasonably estimate.

If unknown, assume 64.4% if furnace and 61.6% ⁴⁴⁷ if boiler.

AFUEbaseER = Baseline Annual Fuel Utilization Efficiency Rating for early replacement measure

= 90%⁴⁴⁸ if furnace and 82% if boiler.

kWhtoTherm = Converts source kWh to Therms

= $H_{grid} / 100000$

H_{grid} = Heat rate of the grid in btu/kWh based on the average fossil heat rate for the EPA eGRID subregion and includes a factor that takes into account T&D losses.

For systems operating less than 6,500 hrs per year:

Use the Non-baseload heat rate provided by EPA eGRID for RFC West region for ComEd territory (including independent providers connected to RFC West), and SERC Midwest region for Ameren territory (including independent providers connected to SERC Midwest)⁴⁴⁹. Also include any line losses.

loads for Single Family, Multi-single, and Condominiums using input characteristics of the home. A best practice for equipment selection and installation of Heating and Air Conditioning, load calculations are commonly completed by contractors during the selection process and may be readily available for program data purposes.

⁴⁴⁵ Values are based on household heating consumption values and inferred average AFUE results from Table 2-1, *Energy Efficiency / Demand Response Nicor Gas Plan Year 1 (6/1/2011-5/31/2012) Research Report: Furnace Metering Study* (August 1, 2013) (prepared by Navigant Consulting, Inc.) and adjusting to a statewide average using relative HDD values to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

⁴⁴⁶ Boiler consumption values are informed by an evaluation which did not identify any fraction of heating load due to domestic hot water (DHW) provided by the boiler. Thus these values are an average of both homes with boilers only providing heat, and homes with boilers that also provide DHW. Values are based on household heating consumption values and inferred average AFUE results from Table 3-4, Program Sample Analysis, *Nicor R29 Res Rebate Evaluation Report 092611_REV FINAL to Nicor*. Adjusting to a statewide average using relative HDD values to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

⁴⁴⁷ Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

⁴⁴⁸ Assumes that Federal Standard will have been increased to 90% by the time the existing unit would have to have been replaced.

⁴⁴⁹ These values are subject to regular updates so should be reviewed regularly to ensure the current assumptions are correct.

Refer to the latest EPA eGRID data. Current values, based on eGrid 2014 are:

- Non-Baseload RFC West: 9,346 Btu/kWh * (1 + Line Losses)
- Non-Baseload SERC Midwest: 9,157 Btu/kWh * (1 + Line Losses)
- All Fossil Average RFC West: 9,931 Btu/kWh * (1 + Line Losses)

For systems operating more than 6,500 hrs per year:

Use the All Fossil Average heat rate provided by EPA eGRID for RFC West region for ComEd territory, and SERC Midwest region for Ameren territory. Also include any line losses.

All other variables provided above

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

COST EFFECTIVENESS SCREENING AND LOAD REDUCTION FORECASTING WHEN FUEL SWITCHING

This measure can involve fuel switching from gas to electric.

For the purposes of forecasting load reductions due to fuel switch DMSHP projects per Section 16-111.5B, changes in site energy use at the customer's meter (using ΔkWh algorithm below) adjusted for utility line losses (at-the-busbar savings), customer switching estimates, NTG, and any other adjustment factors deemed appropriate, should be used.

The inputs to cost effectiveness screening should reflect the actual impacts on the electric and fuel consumption at the customer meter and, for fuel switching measures, this will not match the output of the calculation/allocation methodology presented in the "Electric Energy Savings" and "Natural Gas Savings" sections above. Therefore in addition to the calculation of savings claimed, the following values should be used to assess the cost effectiveness of the measure.

$$\begin{aligned}\Delta \text{Therms} &= [\text{Heating Consumption Replaced}]^{450} \\ &= [(1 - \text{ElecHeat}) * ((\text{Gas_Heating_Load}/\text{AFUE}_{\text{base}})] \\ \Delta kWh &= - [\text{DMSHP heating consumption}] + [\text{Cooling savings}]^{451} \\ &= - [(\text{Capacity}_{\text{heat}} * \text{EFLH}_{\text{heat}} * 1/\text{HSPFee})/1000] + [(\text{Capacity}_{\text{cool}} * \text{EFLH}_{\text{cool}} * (1/\text{SEER}_{\text{Base}} - 1/\text{SEER}_{\text{ee}})) / 1000]\end{aligned}$$

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REVIEW DEADLINE: 1/1/2020

- All Fossil Average SERC Midwest: 10,209 Btu/kWh * (1 + Line Losses)

⁴⁵⁰ Note AFUE_{base} in the algorithm should be replaced with AFUE_{exist} for early replacement measures.

⁴⁵¹ Note SEER_{base} in the algorithm should be replaced with SEER_{exist} for early replacement measures.

5.3.13 Residential Furnace Tune-Up

DESCRIPTION

This measure is for a natural gas Residential furnace that provides space heating. The tune-up will improve furnace performance by inspecting, cleaning and adjusting the furnace and appurtenances for correct and efficient operation. Additional savings maybe realized through a complete system tune-up.

Two savings algorithms are provided for tune-up programs: through the HVAC SAVE program and for other tune-up programs, the difference being how relative efficiencies are measured.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure an approved technician must complete the tune-up requirements⁴⁵² listed below:

- Measure combustion efficiency using an electronic flue gas analyzer
- Check and clean blower assembly and components per manufacturer's recommendations
- Where applicable Lubricate motor and inspect and replace fan belt if required
- Inspect for gas leaks
- Clean burner per manufacturer's recommendations and adjust as needed
- Check ignition system and safety systems and clean and adjust as needed
- Check and clean heat exchanger per manufacturer's recommendations
- Inspect exhaust/flue for proper attachment and operation
- Inspect control box, wiring and controls for proper connections and performance
- Check air filter and clean or replace per manufacturer's
- Inspect duct work connected to furnace for leaks or blockages
- Measure temperature rise and adjust flow as needed
- Check for correct line and load volts/amps
- Check thermostat operation is per manufacturer's recommendations(if adjustments made, refer to 'Residential Programmable Thermostat' measure for savings estimate)
- Perform Carbon Monoxide test and adjust heating system until results are within standard industry acceptable limits

Verified Quality Maintenance:

This approach uses in-field measurement and interpretation of static pressures, identification and plotting of airflow, airflow measurement, temperature measurement and diagnostics, pressure measurements and duct design, and BTU measurement to ensure that existing equipment is operating according to manufacturers' published potential performance. Installed equipment operating efficiency is largely dependent on the efficiency rating of the equipment, the skill of the installation contractor, the degree to which the equipment has aged or drifted from initial settings, and the system level constraints. When one or more of these key dependencies are operating sub-optimally, the overall efficiency of the equipment is degraded. A Verified Quality Maintenance identifies sub-optimal performance and prescribes a solution during furnace tune ups.

The HVAC SAVE program has its own certifications and requirements. In addition to the maintenance described above, the following are key activities that are provided through an HVAC SAVE Verified Quality Maintenance visit⁴⁵³:

- Measure pressure drops at return, filter, coil and supply.
- Determine equipment air flow using OEM blower data or measuring.
- Measure temperature rise across heat exchanger.

⁴⁵² American Standard Maintenance for Indoor Units: <http://www.americanstandardair.com/owner-support/maintenance.html>

⁴⁵³ As provided in ANSI approved ACCA 4 specification for Quality Maintenance

- Determine on-rate for a furnace by clocking the gas meter.
- Record outdoor temperature & elevation, and complete test-in.
- Clean evaporator coil to OEM pressure drop specification.
- Clean/replace/modify air filter to OEM pressure drop specification.
- Reset air flow based on up design parameter and updated pressure conditions.
- Adjust/modify gas pressure and venting to OEM specifications.
- Complete final test-out, compare before and after

DEFINITION OF BASELINE EQUIPMENT

The baseline is furnace assumed not to have had a tune-up in the past 2 years.

HVAC SAVE tune-ups are a one-time measure and cannot be performed more than once on the same piece of equipment.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life for the tune up is 2 years.⁴⁵⁴

An HVAC SAVE tune-up lasts the remaining life of the equipment because they come from adjustments to fans and ducts that remain effective through normal operation of the equipment. Assume 10 years.

DEEMED MEASURE COST

The incremental cost for this measure should be the actual cost of tune up.

DEEMED O&M COST ADJUSTMENTS

There are no expected O&M savings associated with this measure.

LOADSHAPE

Loadshape R09 - Residential Electric Space Heat

COINCIDENCE FACTOR

N/A

Algorithms

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = \Delta \text{Therms} * F_e * 29.3$$

Where:

ΔTherms = as calculated below

F_e = Furnace Fan energy consumption as a percentage of annual fuel consumption
= 3.14%⁴⁵⁵

⁴⁵⁴Act on Energy Commercial Technical Reference Manual No. 2010-4, 9.2.3 Gas Forced-Air Furnace Tune-up.

⁴⁵⁵ F_e is not one of the AHRI certified ratings provided for residential furnaces, but can be reasonably estimated from a

29.3 = kWh per therm

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS SAVINGS

1. Verified Quality Maintenance:

$$\Delta \text{Therms} = (\text{Gas_Furnace_Heating_Load} * \text{HF} * (1 / (\text{AFUE} * (1 - \text{Derating}_{\text{pre}})) - 1 / (\text{AFUE} * (1 - \text{Derating}_{\text{post}})))$$

Where:

Gas_Furnace_Heating_Load = Estimate of annual household heating load⁴⁵⁶ for gas furnace heated single-family homes. If location is unknown, assume the average below⁴⁵⁷.

= Actual if informed by site-specific load calculations, ACCA Manual J or equivalent⁴⁵⁸.

Climate Zone (City based upon)	Gas_Furnace_Heating_Load (therms)
1 (Rockford)	873
2 (Chicago)	834
3 (Springfield)	714
4 (Belleville)	551
5 (Marion)	561
Average	793

HF = Household factor, to adjust heating consumption for non-single-family households.

Household Type	HF
Single-Family	100%
Multi-Family	65% ⁴⁵⁹
Actual	Custom ⁴⁶⁰

AFUE = Furnace Annual Fuel Utilization Efficiency Rating

= Actual

calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the Energy Star version 3 criteria for 2% Fe. See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.

⁴⁵⁶ Heating load is used to describe the household heating need, which is equal to (gas consumption * AFUE)

⁴⁵⁷ Values are based on household heating consumption values and inferred average AFUE results from Table 2-1, Energy Efficiency / Demand Response Nicor Gas Plan Year 1 (6/1/2011-5/31/2012) Research Report: Furnace Metering Study (August 1, 2013) (prepared by Navigant Consulting, Inc.) and adjusting to a statewide average using relative HDD values to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

⁴⁵⁸ The Air Conditioning Contractors of America Manual J, Residential Load Calculation 8th Edition produces equipment sizing loads for Single Family, Multi-single, and Condominiums using input characteristics of the home.

⁴⁵⁹ Multifamily household heating consumption relative to single-family households is affected by overall household square footage and exposure to the exterior. This 65% factor is applied to MF homes, based on professional judgment that average household size, and heat loads of MF households are smaller than single-family homes

⁴⁶⁰ Program-specific household factors may be utilized on the basis of sufficiently validated program evaluations.

Derating_{pre} = Furnace AFUE Derating before HVAC SAVE tune-up
= 6.4%⁴⁶¹

Derating_{post} = Furnace AFUE Derating after HVAC SAVE tune-up
= 0%

2. Other Tune-Up Programs:

$$\Delta \text{Therms} = (\text{Gas_Furnace_Heating_Load} * \text{HF} * (1/ \text{Effbefore} - 1/ (\text{Effbefore} + \text{Ei})))$$

Where:

Gas_Furnace_Heating_Load = Estimate of annual household heating load⁴⁶² for gas furnace heated single-family homes. If location is unknown, assume the average below⁴⁶³.

= Actual if informed by site-specific load calculations, ACCA Manual J or equivalent⁴⁶⁴.

Climate Zone (City based upon)	Gas_Furnace_Heating_Load (therms)
1 (Rockford)	873
2 (Chicago)	834
3 (Springfield)	714
4 (Belleville)	551
5 (Marion)	561
Average	793

HF = Household factor, to adjust heating consumption for non-single-family households.

Household Type	HF
Single-Family	100%
Multi-Family	65% ⁴⁶⁵
Actual	Custom ⁴⁶⁶

Effbefore = Efficiency of the furnace before the tune-up
= Actual

Note: Contractors should select a mid-level firing rate that appropriately represents the average building operating condition over the course of the heating season and take readings at a consistent firing rate for pre and post tune-up.

⁴⁶¹ Based on findings from Building America, US Department of Energy, Brand, Yee and Baker "Improving Gas Furnace Performance: A Field and Laboratory Study at End of Life", February 2015.

⁴⁶² Heating load is used to describe the household heating need, which is equal to (gas consumption * AFUE)

⁴⁶³ Values are based on household heating consumption values and inferred average AFUE results from Table 2-1, Energy Efficiency / Demand Response Nicor Gas Plan Year 1 (6/1/2011-5/31/2012) Research Report: Furnace Metering Study (August 1, 2013) (prepared by Navigant Consulting, Inc.) and adjusting to a statewide average using relative HDD values to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

⁴⁶⁴ The Air Conditioning Contractors of America Manual J, Residential Load Calculation 8th Edition produces equipment sizing loads for Single Family, Multi-single, and Condominiums using input characteristics of the home.

⁴⁶⁵ Multifamily household heating consumption relative to single-family households is affected by overall household square footage and exposure to the exterior. This 65% factor is applied to MF homes, based on professional judgment that average household size, and heat loads of MF households are smaller than single-family homes

⁴⁶⁶ Program-specific household factors may be utilized on the basis of sufficiently validated program evaluations.

EI = Efficiency Improvement of the furnace tune-up measure
= Actual

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-HVC-FTUN-V03-180101

REVIEW DEADLINE: 1/1/2021

5.3.14 Boiler Reset Controls

DESCRIPTION

This measure relates to improving system efficiency by adding controls to residential heating boilers to vary the boiler entering water temperature relative to heating load as a function of the outdoor air temperature to save energy. The water can be run a little cooler during fall and spring, and a little hotter during the coldest parts of the winter. A boiler reset control has two temperature sensors - one outside the house and one in the boiler water. As the outdoor temperature goes up and down, the control adjusts the water temperature setting to the lowest setting that is meeting the house heating demand. There are also limits in the controls to keep a boiler from operating outside of its safe performance range.⁴⁶⁷

This measure was developed to be applicable to the following program types: RF.

DEFINITION OF EFFICIENT EQUIPMENT

Natural gas single family residential customer adding boiler reset controls capable of resetting the boiler supply water temperature in an inverse fashion with outdoor air temperature. The system must be set so that the minimum temperature is not more than 10 degrees above manufacturer's recommended minimum return temperature. This boiler reset measure is limited to existing condensing boilers serving a single family residence. Boiler reset controls for non-condensing boilers in single family residences should be implemented as a custom measure, and the cost-effectiveness should be confirmed.

DEFINITION OF BASELINE EQUIPMENT

Existing condensing boiler in a single family residential setting without boiler reset controls.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The life of this measure is 20 years⁴⁶⁸

DEEMED MEASURE COST

The cost of this measure is \$612⁴⁶⁹

LOADSHAPE

NA

COINCIDENCE FACTOR

N/A

⁴⁶⁷ Energy Solutions Center, a consortium of natural gas utilities, equipment manufacturers and vendors. Boiler Reset Control, accessed at http://naturalgasefficiency.org/residential/Boiler_Reset_Control.htm

⁴⁶⁸ CLEAR result references the Brooklyn Union Gas Company, High Efficiency Heating and Water and Controls, Gas Energy Efficiency Program Implementation Plan.

⁴⁶⁹ Nexant. Questar DSM Market Characterization Report. August 9, 2006.

Algorithm

CALCULATION OF ENERGY SAVINGS**ELECTRIC ENERGY SAVINGS**

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

NA

NATURAL GAS SAVINGS

$$\Delta \text{Therms} = \text{Gas_Boiler_Load} * (1/\text{AFUE}) * \text{Savings Factor}$$

Where:

Gas_Boiler_Load⁴⁷⁰

= Estimate of annual household Load for gas boiler heated single-family homes. If location is unknown, assume the average below⁴⁷¹.

= or Actual if informed by site-specific load calculations, ACCA Manual J or equivalent⁴⁷².

Climate Zone (City based upon)	Gas_Boiler Load (therms)
1 (Rockford)	1275
2 (Chicago)	1218
3 (Springfield)	1043
4 (Belleville)	805
5 (Marion)	819
Average	1158

AFUE = Existing Condensing Boiler Annual Fuel Utilization Efficiency Rating
= Actual.

SF = Savings Factor, 5%⁴⁷³

⁴⁷⁰ Boiler consumption values are informed by an evaluation which did not identify any fraction of heating load due to domestic hot water (DHW) provided by the boiler. Thus these values are an average of both homes with boilers only providing heat, and homes with boilers that also provide DHW. Heating load is used to describe the household heating need, which is equal to (gas heating consumption * AFUE)

⁴⁷¹ Values are based on household heating consumption values and inferred average AFUE results from Table 3-4, Program Sample Analysis, *Nicor R29 Res Rebate Evaluation Report 092611_REV FINAL to Nicor*). Adjusting to a statewide average using relative HDD values to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

⁴⁷² The Air Conditioning Contractors of America Manual J, Residential Load Calculation 8th Edition produces equipment sizing loads for Single Family, Multi-single, and Condominiums using input characteristics of the home. A best practice for equipment selection and installation of Heating and Air Conditioning, load calculations should be completed by contractors during the selection process and may be readily available for program data purposes.

⁴⁷³ Energy Solutions Center, a consortium of natural gas utilities, equipment manufacturers and vendors. Boiler Reset Control, accessed at http://naturalgasefficiency.org/residential/Boiler_Reset_Control.htm

EXAMPLE

For example, boiler reset controls on a 92.5 AFUE boiler at a household in Rockford, IL

$$\begin{aligned}\Delta\text{Therms} &= 1275 * (1/0.925) * 0.05 \\ &= 69 \text{ Therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-HVC-BREC-V01-150601

REVIEW DEADLINE: 1/1/2021

5.3.15 ENERGY STAR Ceiling Fan

DESCRIPTION

A ceiling fan/light unit meeting the efficiency specifications of ENERGY STAR is installed in place of a model meeting the federal standard. ENERGY STAR qualified ceiling fan/light combination units are over 60% more efficient than conventional fan/light units, and use improved motors and blade designs⁴⁷⁴.

Due to the savings from this measure being derived from more efficient ventilation and more efficient lighting, and the loadshape and measure life for each component being very different, the savings are split in to the component parts and should be claimed together. Lighting savings should be estimated utilizing the 5.5.1 ENERGY STAR Compact Fluorescent Lamp measure.

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is defined as an ENERGY STAR certified ceiling fan with integral CFL bulbs.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be a standard fan with efficient incandescent or halogen light bulbs. Production of 100W, standard efficacy incandescent lamps ended in 2012 followed by restrictions on 75W in 2013 and 60W and 40W in 2014, due to the Energy Independence and Security Act of 2007 (EISA). Finally, a provision in the EISA regulations requires that by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, in essence making the baseline equivalent to a current day CFL. Therefore the measure life (number of years that savings should be claimed) for the lighting portion of the savings should be reduced once the assumed lifetime of the bulb exceeds 2020. Due to expected delay in clearing retail inventory and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until 2021.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The fan savings measure life is assumed to be 10 years.²

The lighting savings measure life is assumed to be 3 years for lighting savings for units installed in 2018, and then for every subsequent year should be reduced by one year⁴⁷⁵ (see 5.5.1 ENERGY STAR Compact Fluorescent Lamp measure).

DEEMED MEASURE COST

Incremental cost of unit is \$46.⁴⁷⁶

LOADSHAPE

R06 - Residential Indoor Lighting

R11 - Residential Ventilation

⁴⁷⁴ <http://www.energystar.gov/products/certified-products/detail/ceiling-fans>

⁴⁷⁵ Since the replacement baseline bulb from 2020 on will be equivalent to a CFL, no additional savings should be claimed from that point. Due to expected delay in clearing stock from retail outlets and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until 2021.

⁴⁷⁶ ENERGY STAR Ceiling Fan Savings Calculator
http://www.energystar.gov/buildings/sites/default/uploads/files/light_fixture_ceiling_fan_calculator.xlsx?8178-e52c

COINCIDENCE FACTOR

The summer peak coincidence factor for the ventilation savings is assumed to be 30%.⁴⁷⁷

For lighting savings, see 5.5.1 ENERGY STAR Compact Fluorescent Lamp measure.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta kWh = \Delta kWh_{fan} + \Delta kWh_{light}$$

$$\Delta kWh_{fan} = [Days * FanHours * ((\%Low_{base} * WattsLow_{base}) + (\%Med_{base} * WattsMed_{base}) + (\%High_{base} * WattsHigh_{base}))/1000] - [Days * FanHours * ((\%Low_{ES} * WattsLow_{ES}) + (\%Med_{ES} * WattsMed_{ES}) + (\%High_{ES} * WattsHigh_{ES}))/1000]$$

$$\Delta kWh_{light} = \text{see 5.5.1 ENERGY STAR Compact Fluorescent Lamp measure.}$$

Where⁴⁷⁸:

Days	= Days used per year = Actual. If unknown use 365.25 days/year
FanHours	= Daily Fan “On Hours” = Actual. If unknown use 3 hours
%Low _{base}	= Percent of time spent at Low speed of baseline = 40%
WattsLow _{base}	= Fan wattage at Low speed of baseline = Actual. If unknown use 15 watts
%Med _{base}	= Percent of time spent at Medium speed of baseline = 40%
WattsMed _{base}	= Fan wattage at Medium speed of baseline = Actual. If unknown use 34 watts
%High _{base}	= Percent of time spent at High speed of baseline = 20%
WattsHigh _{base}	= Fan wattage at High speed of baseline = Actual. If unknown use 67 watts
%Low _{ES}	= Percent of time spent at Low speed of ENERGY STAR

⁴⁷⁷ Assuming that the CF same as a Room AC. Consistent with coincidence factors found in: RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008 (http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RA C.pdf)

⁴⁷⁸ All fan default assumptions are based upon assumptions provided in the ENERGY STAR Ceiling Fan Savings Calculator; http://www.energystar.gov/buildings/sites/default/uploads/files/light_fixture_ceiling_fan_calculator.xlsx?8178-e52c

	= 40%
WattsLow _{ES}	= Fan wattage at Low speed of ENERGY STAR
	= Actual. If unknown use 6 watts
%Med _{ES}	= Percent of time spent at Medium speed of ENERGY STAR
	= 40%
WattsMed _{ES}	= Fan wattage at Medium speed of ENERGY STAR
	= Actual. If unknown use 23 watts
%High _{ES}	= Percent of time spent at High speed of ENERGY STAR
	= 20%
WattsHigh _{ES}	= Fan wattage at High speed of ENERGY STAR
	= Actual. If unknown use 56 watts

For ease of reference, the fan assumptions are provided below in table form:

	Low Speed	Medium Speed	High Speed
Percent of Time at Given Speed	40%	40%	20%
Conventional Unit Wattage	15	34	67
ENERGY STAR Unit Wattage	6	23	56
ΔW	9	11	11

If the lighting WattsBase and WattsEE is unknown, assume the following

$$\text{WattsBase} = 3 \times 43 = 129 \text{ W}$$

$$\text{WattsEE} = 1 \times 42 = 42 \text{ W}$$

EXAMPLE

For example, a ceiling fan with three 43W bulb light fixtures, replaced with an ES ceiling fan with one 42W bulb light fixture, the savings are:

$$\begin{aligned} \Delta \text{kWh}_{\text{fan}} &= [365.25 \times 3 \times ((0.4 \times 15) + (0.4 \times 34) + (0.2 \times 67)) / 1000] - \\ &\quad [365.25 \times 3 \times ((0.4 \times 6) + (0.4 \times 23) + (0.2 \times 56)) / 1000] \\ &= 36.2 - 25.0 = 11.2 \text{ kWh} \end{aligned}$$

$$\begin{aligned} \Delta \text{kWh}_{\text{light}} &= ((129 - 42) / 1000) \times 759 \times 1.06 \\ &= 70.0 \text{ kWh} \end{aligned}$$

$$\begin{aligned} \Delta \text{kWh} &= 11.2 + 70 \\ &= 81.2 \text{ kWh} \end{aligned}$$

Using the default assumptions provided above, the deemed savings is 81.2 kWh.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \Delta \text{kW}_{\text{Fan}} + \Delta \text{kW}_{\text{light}}$$

$$\Delta \text{kW}_{\text{Fan}} = ((\text{WattsHigh}_{\text{base}} - \text{WattsHigh}_{\text{ES}}) / 1000) \times \text{CF}_{\text{fan}}$$

ΔkW_{Light} = see 5.5.1 ENERGY STAR Compact Fluorescent Lamp measure.

Where:

CF_{fan} = Summer Peak coincidence factor for ventilation savings
= 30%⁴⁷⁹

CF_{light} = Summer Peak coincidence factor for lighting savings
= 7.1%⁴⁸⁰

EXAMPLE

For example a ceiling fan with three 43W bulb light fixtures, replaced with an ES ceiling fan with one 42W bulb light fixture, the savings are:

$$\begin{aligned}\Delta kW_{\text{fan}} &= ((67-56)/1000) * 0.3 \\ &= 0.0033 \text{ kW} \\ \Delta kW_{\text{light}} &= ((129 - 42)/1000) * 1.11 * 0.071 \\ &= 0.0068 \text{ kW} \\ \Delta kW &= 0.0033 + 0.0068 \\ &= 0.010 \text{ kW}\end{aligned}$$

Using the default assumptions provided above, the deemed savings is 0.010kW.

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

See 5.5.1 ENERGY STAR Compact Fluorescent Lamp measure for bulb replacement costs.

MEASURE CODE: RS-HVC-CFAN-V02-180101

REVIEW DEADLINE: 1/1/2022

⁴⁷⁹ Assuming that the CF same as a Room AC. Consistent with coincidence factors found in: RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008 (http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RA_C.pdf)

⁴⁸⁰ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

5.3.16 Advanced Thermostats

DESCRIPTION

This measure characterizes the household energy savings from the installation of a new thermostat(s) for reduced heating and cooling consumption through a configurable schedule of temperature setpoints (like a programmable thermostat) *and* automatic variations to that schedule to better match HVAC system runtimes to meet occupant comfort needs. These schedules may be defaults, established through user interaction, and be changed manually at the device or remotely through a web or mobile app. Automatic variations to that schedule could be driven by local sensors and software algorithms, and/or through connectivity to an internet software service. Data triggers to automatic schedule changes might include, for example: occupancy/activity detection, arrival & departure of conditioned spaces, optimization based on historical or population-specific trends, weather data and forecasts.⁴⁸¹ This class of products and services are relatively new, diverse, and rapidly changing. Generally, the savings expected for this measure aren't yet established at the level of individual features, but rather at the system level and how it performs overall. Like programmable thermostats, it is not suitable to assume that heating and cooling savings follow a similar pattern of usage and savings opportunity, and so here too this measure treats these savings independently. Note that it is a very active area of ongoing study to better map features to savings value, and establish standards of performance measurement based on field data so that a standard of efficiency can be developed.⁴⁸² That work is not yet complete but does inform the treatment of some aspects of this characterization and recommendations. Energy savings are applicable at the household level; all thermostats controlling household heat should be programmable and installation of multiple advanced thermostats per home does not accrue additional savings.

Note that though these devices and service could potentially be used as part of a demand response program, the costs, delivery, impacts, and other aspects of DR-specific program delivery are not included in this characterization at this time, though they could be added in the future.

This measure was developed to be applicable to the following program types: TOS, NC, RF, DI. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The criteria for this measure are established by replacement of a manual-only or programmable thermostat, with one that has the default enabled capability—or the capability to automatically—establish a schedule of temperature setpoints according to driving device inputs above and beyond basic time and temperature data of conventional programmable thermostats. As summarized in the description, this category of products and services is broad and rapidly advancing in regards to their capability, usability, and sophistication, but at a minimum must be capable of two-way communication⁴⁸³ and exceed the typical performance of manual and conventional programmable thermostats through the automatic or default capabilities described above.

DEFINITION OF BASELINE EQUIPMENT

The baseline is either the actual type (manual or programmable) if it is known,⁴⁸⁴ or an assumed mix of these two

⁴⁸¹ For example, the capabilities of products and added services that use ultrasound, infrared, or geofencing sensor systems, automatically develop individual models of home's thermal properties through user interaction, and optimize system operation based on equipment type and performance traits based on weather forecasts demonstrate the type of automatic schedule change functionality that apply to this measure characterization.

⁴⁸² The ENERGY STAR program discontinued its support for basic programmable thermostats effective 12/31/09, and is presently developing a new specification for 'Residential Climate Controls'.

⁴⁸³ This measure recognizes that field data may be available, through this 2-way communication capability, to better inform characterization of efficiency criteria and savings calculations. It is recommended that program implementations incorporate this data into their planning and operation activities to improve understanding of the measure to manage risks and enhance savings results.

⁴⁸⁴ If the actual thermostat is programmable and it is found to be used in override mode or otherwise effectively being operated like a manual thermostat, then the baseline may be considered to be a manual thermostat

types based upon information available from evaluations or surveys that represent the population of program participants. This mix may vary by program, but as a default, 44% programmable and 56% manual thermostats may be assumed⁴⁸⁵.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life for advanced thermostats is assumed to be similar to that of a programmable thermostat 10 years⁴⁸⁶ based upon equipment life only.⁴⁸⁷

DEEMED MEASURE COST

For DI and other programs for which installation services are provided, the actual material, labor, and other costs should be used. For retail, Bring Your Own Thermostat (BYOT) programs⁴⁸⁸, or other program types actual costs are still preferable⁴⁸⁹ but if unknown then the average incremental cost for the new installation measure is assumed to be \$175⁴⁹⁰.

LOADSHAPE

ΔkWh → Loadshape R10 - Residential Electric Heating and Cooling
 $\Delta kWh_{heating}$ → Loadshape R09 - Residential Electric Space Heat
 $\Delta kWh_{cooling}$ → Loadshape R08 - Residential Cooling

COINCIDENCE FACTOR

In the absence of conclusive results from empirical studies on peak savings, the TAC agreed to a temporary assumption of 50% of the cooling coincidence factor, acknowledging that while the savings from the advanced Thermostat will track with the cooling load, the impact during peak periods may be lower. This is an assumption that could use future evaluation to improve these estimates.

CF_{SSP} = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)
= 34%⁴⁹¹

CF_{PJM} = PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)
= 23.3%⁴⁹²

⁴⁸⁵ Based on Opinion Dynamics Corporation, "ComEd Residential Saturation/End Use, Market Penetration & Behavioral Study", Appendix 3: Detailed Mail Survey Results, p34, April 2013.

⁴⁸⁶ Table 1, HVAC Controls, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007

⁴⁸⁷ Future evaluation is strongly encouraged to inform the persistence of savings to further refine measure life assumption. As this characterization depends heavily upon a number of savings studies that only lasted a single year or less, the longer term impacts should be assessed.

⁴⁸⁸ In contrast to program designs that utilize program affiliated contractors or other trade ally partners that support customer participation through thermostat distribution, installation and other services, BYOT programs enroll customers *after* the time of purchase through online rebate and program integration sign-ups.

⁴⁸⁹ Including any one-time software integration or annual software maintenance, and or individual device energy feature fees.

⁴⁹⁰ Market prices vary considerably in this category, generally increasing with thermostat capability and sophistication. The core suite of functions required by this measure's eligibility criteria are available on units readily available in the market roughly in the range of \$200 and \$250, excluding the availability of any wholesale or volume discounts. The assumed incremental cost is based on the middle of this range (\$225) minus a cost of \$50 for the baseline equipment blend of manual and programmable thermostats. Note that any add-on energy service costs, which may include one-time setup and/or annual per device costs are not included in this assumption.

⁴⁹¹ Assumes 50% of the cooling coincidence factor (based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory).

⁴⁹² Assumes 50% of the cooling coincidence factor (based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.)

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta kWh^{493} = \Delta kWh_{heating} + \Delta kWh_{cooling}$$

$$\Delta kWh_{heating} = \%ElectricHeat * Elec_Heating_Consumption * Heating_Reduction * HF * Eff_ISR + (\Delta Therms * F_e * 29.3)$$

$$\Delta kWh_{cool} = \%AC * ((FLH * Btu/hr * 1/SEER)/1000) * Cooling_Reduction * Eff_ISR$$

Where:

$\%ElectricHeat$ = Percentage of heating savings assumed to be electric

Heating fuel	$\%ElectricHeat$
Electric	100%
Natural Gas	0%
Unknown	6.5% ⁴⁹⁴

$Elec_Heating_Consumption$

= Estimate of annual household heating consumption for electrically heated single-family homes⁴⁹⁵. If location and heating type is unknown, assume 15,678 kWh⁴⁹⁶

Climate Zone (City based upon)	Electric Resistance $Elec_Heating_Consumption$ (kWh)	Electric Heat Pump $Elec_Heating_Consumption$ (kWh)
1 (Rockford)	21,741	12,789
2 (Chicago)	20,771	12,218
3 (Springfield)	17,789	10,464
4 (Belleville)	13,722	8,072
5 (Marion)	13,966	8,215
Average	19,743	11,613

$Heating_Reduction$ = Assumed percentage reduction in total household heating energy

⁴⁹³ Electrical savings are a function of both heating and cooling energy usage reductions. For heating this is a function of the percent of electric heat (heat pumps) and fan savings in the case of a natural gas furnace.

⁴⁹⁴ Assumes that half of the electric heat in the state is a heat pump able to be controlled by an advanced thermostat (consistent with Potential Study results from the state). Average value of 13% electric space heating from 2010 Residential Energy Consumption Survey for Illinois. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used.

⁴⁹⁵ Values in table are based on converting an average household heating load (834 therms) for Chicago based on 'Table E-1, Energy Efficiency/Demand Response Nicor Gas Plan Year 1: Research Report: Furnace Metering Study, Draft, Navigant, August 1 2013 to an electric heat load (divide by 0.03413) to electric resistance and ASHP heat load (resistance load reduced by 15% to account for distribution losses that occur in furnace heating but not in electric resistance while ASHP heat is assumed to suffer from similar distribution losses) and then to electric consumption assuming efficiencies of 100% for resistance and 200% for HP (see 'Household Heating Load Summary Calculations_11062013.xls'). Finally these values were adjusted to a statewide average using relative HDD assumptions to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

⁴⁹⁶ Assumption that 1/2 of electrically heated homes have electric resistance and 1/2 have Heat Pump, based on 2010 Residential Energy Consumption Survey for Illinois.

consumption due to advanced thermostat

Existing Thermostat Type	Heating_Reduction ⁴⁹⁷
Manual	8.8%
Programmable	5.6%
Unknown (Blended)	7.4%

HF = Household factor, to adjust heating consumption for non-single-family households.

Household Type	HF
Single-Family	100%
Multi-Family	65% ⁴⁹⁸
Actual	Custom ⁴⁹⁹

Eff_ISR = Effective In-Service Rate, the percentage of thermostats installed and configured effectively for 2-way communication. Note that retrospective adjustments should be made during evaluation verification activities through the use of a realization rate if the program design does not ensure that each advanced thermostat is actually installed and/or if the evaluation determines that the advanced thermostat is not actually installed in the Program Administrator's service territory.

Program Delivery	Eff_ISR
Direct Install	100%
Other	100% ⁵⁰⁰

Δ Therms = Therm savings if Natural Gas heating system
= See calculation in Natural Gas section below

F_e = Furnace Fan energy consumption as a percentage of annual fuel consumption
= 3.14%⁵⁰¹

29.3 = kWh per therm

%AC = Fraction of customers with thermostat-controlled air-conditioning

⁴⁹⁷ These values represent adjusted baseline savings values (8.8% for manual, and 5.6% for programmable thermostats) as presented in Navigant's PowerPoint on Impact Analysis from Preliminary Gas savings findings (slide 28 of 'IL SAG Smart Thermostat Preliminary Gas Impact Findings 2015-12-08 to IL SAG.ppt'). These values are used as the basis for the weighted average savings value when the type of existing thermostat is not known. Using the default assumption of 56% manual and 44% programmable as described in the baseline definition section above the 7.4% savings value is equal to the sum of proportional savings for manual and programmable thermostats: $8.8\% \times 0.56 + 5.6\% \times 0.44$. Further evaluation and regular review of this key assumption is encouraged.

⁴⁹⁸ Multifamily household heating consumption relative to single-family households is affected by overall household square footage and exposure to the exterior. This 65% reduction factor is applied to MF homes, based on professional judgment that average household size, and heat loads of MF households are smaller than single-family homes

⁴⁹⁹ Program-specific household factors may be utilized on the basis of sufficiently validated program evaluations.

⁵⁰⁰ As a function of the method for determining savings impact of these devices, in-service rate effects are already incorporated into the savings value for heating_reduction above.

⁵⁰¹ F_e is not one of the AHRI certified ratings provided for residential furnaces, but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBTU/yr) and Eae (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the Energy Star version 3 criteria for 2% F_e. See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.

Thermostat control of air conditioning?	%AC
Yes	100%
No	0%
Unknown	66% ⁵⁰²
Unknown Multi-Family	46% ⁵⁰³
Unknown Single Family	87% ⁵⁰⁴

FLH = Estimate of annual household full load cooling hours for air conditioning equipment based on location and home type. If location and cooling type are unknown, assume the weighted average.

Climate zone (city based upon)	FLH (single family) ⁵⁰⁵	FLH (general multifamily) ⁵⁰⁶	FLH_cooling (weatherized multi family) ⁵⁰⁷
1 (Rockford)	512	467	243
2 (Chicago)	570	506	263
3 (Springfield)	730	663	345
4 (Belleville)	1035	940	489
5 (Marion)	903	820	426
Weighted average ⁵⁰⁸	629	564	293

Btu/hr = Size of AC unit⁵⁰⁹. (Note: One refrigeration ton is equal to 12,000 Btu/hr.)

Program Delivery	Btu/hr
Direct Install (Single Family known, or MF)	Actual
Unknown (Single family home only)	33,600

SEER = the cooling equipment's Seasonal Energy Efficiency Ratio rating (kBtu/kWh)
 = Use actual SEER rating where it is possible to measure or reasonably estimate.

Cooling System	SEER ⁵¹⁰
Air Source Heat Pump	9.12
Central AC	8.60

1/1000 = kBtu per Btu

⁵⁰² 66% of homes in Illinois having central cooling ("Table HC7.9 Air Conditioning in Homes in Midwest Region, Divisions, and States, 2009 from Energy Information Administration", 2009 Residential Energy Consumption Survey;

⁵⁰³ Based on Opinion Dynamics Corporation, "ComEd Residential Saturation/End Use, Market Penetration & Behavioral Study", Appendix 3: Detailed Mail Survey Results, April 2013.

⁵⁰⁴ Ibid.

⁵⁰⁵ Full load hours for Chicago, Moline and Rockford are provided in "Final Evaluation Report: Central Air Conditioning Efficiency Services (CACES), 2010, Navigant Consulting", p.33. An average FLH/Cooling Degree Day (from NCDC) ratio was calculated for these locations and applied to the CDD of the other locations in order to estimate FLH. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

⁵⁰⁶ Ibid.

⁵⁰⁷ *All-Electric Homes PY6 Metering Results: Multifamily HVAC Systems*, Cadmus, October 2015

⁵⁰⁸ Weighted based on number of residential occupied housing units in each zone.

⁵⁰⁹ Actual unit size required for multi-family building, no size assumption provided because the unit size and resulting savings can vary greatly depending on the number of units.

⁵¹⁰ Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

$$\begin{aligned}\text{Cooling_Reduction} &= \text{Assumed percentage reduction in total household cooling energy consumption due to installation of advanced thermostat}^{511}: \\ &= 8.0\%^{512}\end{aligned}$$

For example, an advanced thermostat replacing a programmable thermostat directly installed in an electric heat pump heated, single-family home in Springfield with advanced thermostat-controlled air conditioning of a system of unknown size and seasonal efficiency rating:

$$\begin{aligned}\Delta \text{kWh} &= \Delta \text{kWh}_{\text{heating}} + \Delta \text{kWh}_{\text{cooling}} \\ &= 1 * 10,464 * 5.6\% * 100\% * 100\% + (0 * 0.0314 * 29.3) + 100\% * ((730 * 33,600 * (1/9.12))/1000) * 8\% * 100\% \\ &= 586\text{kWh} + 215 \text{ kWh} \\ &= 801 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \% \text{AC} * (\text{Cooling_Reduction} * \text{Btu/hr} * (1/\text{EER}))/1000 * \text{EFF_ISR} * \text{CF}$$

Where:

EER = Energy Efficiency Ratio of existing cooling system (kBtu/hr / kW)
= Use actual EER rating where it is possible to measure or reasonably estimate. If EER unknown but SEER available convert using the equation:

$$\text{EER} = (-0.02 * \text{SEER}_{\text{exist}}^2) + (1.12 * \text{SEER}_{\text{exist}})^{513}$$

If SEER or EER rating unavailable use:

Cooling System	EER ⁵¹⁴
Air Source Heat Pump	8.55
Central AC	8.15

CF_{SSP} = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)
= 34%⁵¹⁵

CF_{PJM} = PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)

⁵¹¹ The Technical Advisory Committee is currently collaborating on a plan for an upcoming evaluation that will relate to the exact population this measure represents, and results of that evaluation will be used to update this assumption when it is finalized.

⁵¹² This assumption is based upon the review of many evaluations from other regions in the US (see Navigant workpaper “Illinois Statewide TRM Workpaper_RES_Smart Thermostats_2015 11 02.docx” and VEIC summary “Studies informing the Illinois TRM Savings Characterization for Advanced Thermostats.docx”). These sources, are from different regions, products, and program delivery designs, but collectively form a sound basis, and directional guidance for the existence and magnitude of cooling savings. Because cooling savings are more volatile than those for heating due to variables in control behaviors, population, and product factors, conservatism is warranted and 8% was considered a conservative estimate based upon the array of results from these studies available at the time this value was developed. Further evaluation and regular review of this key assumption is encouraged.

⁵¹³ From Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder.

⁵¹⁴ Average nameplate efficiencies of all Early Replacement qualifying equipment in Ameren PY3-PY4.

⁵¹⁵ Assumes 50% of the cooling coincidence factor (based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.)

$$= 23.3\%^{516}$$

Other variables as provided above

For example, an advanced thermostat replacing a programmable thermostat directly installed in an electric resistance heated, single-family home in Springfield with advanced thermostat-controlled air conditioning of a system of unknown size and seasonal efficiency rating:

$$\Delta kW_{SSP} = 100\% * 8\% * 33,600 * (1/8.15)/1000 * 100\% * 34\%$$

$$= 0.11 \text{ kW}$$

$$\Delta kW_{PJM} = 100\% * 8\% * 33,600 * (1/8.15)/1000 * 100\% * 23.3\%$$

$$= 0.077 \text{ kW}$$

NATURAL GAS ENERGY SAVINGS

$$\Delta \text{Therms} = \% \text{FossilHeat} * \text{Gas_Heating_Consumption} * \text{Heating_Reduction} * \text{HF} * \text{Eff_ISR}$$

Where:

$\% \text{FossilHeat}$ = Percentage of heating savings assumed to be Natural Gas

Heating fuel	$\% \text{FossilHeat}$
Electric	0%
Natural Gas	100%
Unknown	93.5% ⁵¹⁷

$\text{Gas_Heating_Consumption}$

= Estimate of annual household heating consumption for gas heated single-family homes.
If location is unknown, assume the average below⁵¹⁸.

Climate Zone (City based upon)	$\text{Gas_Heating_Consumption}$ (therms)
1 (Rockford)	1,052
2 (Chicago)	1,005
3 (Springfield)	861
4 (Belleville)	664

⁵¹⁶ Assumes 50% of the cooling coincidence factor (based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.)

⁵¹⁷ Assumes that half of the electric heat in the state is a heat pump able to be controlled by an advanced thermostat. Data from 2010 Residential Energy Consumption Survey for Illinois. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used.

⁵¹⁸ Values are based on adjusting the average household heating consumption (849 therms) for Chicago based on 'Table 3-4, Program Sample Analysis, Nicor R29 Res Rebate Evaluation Report 092611_REV FINAL to Nicor', calculating inferred heating load by dividing by average efficiency of new in program units in the study (94.4%) and then applying standard assumption of existing unit efficiency of 83% (estimate based on 24% of furnaces purchased in Illinois were condensing in 2000 (based on data from GAMA, provided to Department of Energy), assuming typical efficiencies: $(0.24 * 0.92) + (0.76 * 0.8) = 0.83$). This Chicago value was then adjusted to a statewide average using relative HDD assumptions to adjust for the evaluation results focus on northern region. Values for individual cities are then calculated by comparing average HDD to the individual city's HDD.

Climate Zone (City based upon)	Gas_Heating_ Consumption (therms)
5 (Marion)	676
Average	955

Other variables as provided above

For example, an advanced thermostat replacing a programmable thermostat directly-installed in a gas heated single-family home in Chicago:

$$\begin{aligned}\Delta\text{Therms} &= 1.0 * 1005 * 5.6\% * 100\% * 100\% \\ &= 56.28 \text{ therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-HVC-ADTH-V02-180101

REVIEW DEADLINE: 1/1/2019

5.4 Hot Water End Use

5.4.1 Domestic Hot Water Pipe Insulation

DESCRIPTION

This measure describes adding insulation to un-insulated domestic hot water pipes. The measure assumes the pipe wrap is installed to the first length of both the hot and cold pipe up to the first elbow. This is the most cost effective section to insulate since the water pipes act as an extension of the hot water tank up to the first elbow which acts as a heat trap. Insulating this length therefore helps reduce standby losses. Default savings are provided per 3ft length and are appropriate up to 6ft of the hot water pipe and 3ft of the cold.

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient case is installing pipe wrap insulation to a length of hot water pipe.

DEFINITION OF BASELINE EQUIPMENT

The baseline is an un-insulated hot water pipe.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 15 years⁵¹⁹.

DEEMED MEASURE COST

The measure cost including material and installation is assumed to be \$3 per linear foot⁵²⁰.

LOADSHAPE

Loadshape C53 - Flat

COINCIDENCE FACTOR

This measure assumes a flat loadshape since savings relate to reducing standby losses and as such the coincidence factor is 1.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

For electric DHW systems:

$$\Delta \text{kWh} = ((1/R_{\text{exist}} - 1/R_{\text{new}}) * (L * C) * \Delta T * 8,766) / \eta_{\text{DHW}} / 3413$$

Where:

⁵¹⁹ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.
<http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf>

⁵²⁰ Consistent with DEER 2008 Database Technology and Measure Cost Data (www.deeresources.com).

Rexist	= Pipe heat loss coefficient of uninsulated pipe (existing) [(hr-°F-ft)/Btu] = 1.0 ⁵²¹
Rnew	= Pipe heat loss coefficient of insulated pipe (new) [(hr-°F-ft)/Btu] = Actual (1.0 + R value of insulation)
L	= Length of pipe from water heating source covered by pipe wrap (ft) = Actual
C	= Circumference of pipe (ft) (Diameter (in) * $\pi/12$) = Actual (0.5" pipe = 0.131ft, 0.75" pipe = 0.196ft)
ΔT	= Average temperature difference between supplied water and outside air temperature (°F) = 60°F ⁵²²
8,766	= Hours per year
η_{DHW}	= Recovery efficiency of electric hot water heater = 0.98 ⁵²³
3412	= Conversion from Btu to kWh

For example, insulating 5 feet of 0.75" pipe with R-5 wrap:

$$\begin{aligned}
 \Delta kWh &= ((1/R_{exist} - 1/R_{new}) * (L * C) * \Delta T * 8,766) / \eta_{DHW} / 3412 \\
 &= ((1/1 - 1/(1+5)) * (5 * 0.196) * 60 * 8766) / 0.98 / 3412 \\
 &= 128 \text{ kWh}
 \end{aligned}$$

If inputs above are not available the following default per 3ft R-5 length can be used for up to 6 ft length on the hot pipe and 3 ft on the cold pipe.

$$\begin{aligned}
 \Delta kWh &= ((1/R_{exist} - 1/R_{new}) * (L * C) * \Delta T * 8,766) / \eta_{DHW} / 3412 \\
 &= ((1/1 - 1/(1+5)) * (3 * 0.196) * 60 * 8766) / 0.98 / 3412 \\
 &= 77.1 \text{ kWh per 3ft length}
 \end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh / 8766$$

Where:

ΔkWh	= kWh savings from pipe wrap installation
8766	= Number of hours in a year (since savings are assumed to be constant over year).

⁵²¹ Navigant Consulting Inc., April 2009; "Measures and Assumptions for Demand Side Management (DSM) Planning; Appendix C Substantiation Sheets", p77.

⁵²² Assumes 125°F water leaving the hot water tank and average temperature of basement of 65°F.

⁵²³ Electric water heaters have recovery efficiency of 98%: <http://www.ahridirectory.org/ahridirectory/pages/home.aspx>

For example, insulating 5 feet of 0.75" pipe with R-5 wrap:

$$\begin{aligned}\Delta kW &= 128/8766 \\ &= 0.015kW\end{aligned}$$

If inputs above are not available the following default per 3ft R-4 length can be used for up to 6 ft length on the hot pipe and 3 ft on the cold pipe.

$$\begin{aligned}\Delta kW &= 77.1/8766 \\ &= 0.0088 kW\end{aligned}$$

NATURAL GAS SAVINGS

For Natural Gas DHW systems:

$$\Delta Therm = ((1/R_{exist} - 1/R_{new}) * (L * C) * \Delta T * 8,766) / \eta_{DHW} / 100,000$$

Where:

$$\begin{aligned}\eta_{DHW} &= \text{Recovery efficiency of gas hot water heater} \\ &= 0.78^{524} \\ &\text{Other variables as defined above}\end{aligned}$$

For example, insulating 5 feet of 0.75" pipe with R-5 wrap:

$$\begin{aligned}\Delta Therm &= ((1/1 - 1/(1+5)) * (5 * 0.196) * 60 * 8766) / 0.78 / 100,000 \\ &= 5.51 \text{ therms}\end{aligned}$$

If inputs above are not available the following default per 3ft R-4 length can be used for up to 6ft length on the hot pipe and 3ft on the cold pipe.

$$\begin{aligned}\Delta Therm &= ((1/R_{exist} - 1/R_{new}) * (L * C) * \Delta T * 8,766) / \eta_{DHW} / 100,000 \\ &= ((1/1 - 1/(1+5)) * (3 * 0.196) * 60 * 8766) / 0.78 / 100,000 \\ &= 3.30 \text{ therms per 3ft length}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-HWE-PINS-V02-150601

REVIEW DEADLINE: 1/1/2019

⁵²⁴ Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 78%

5.4.2 Gas Water Heater

DESCRIPTION

This measure characterizes:

- a) Time of sale or new construction:
The purchase and installation of a new efficient gas-fired water heater, in place of a Federal Standard unit in a residential setting. Savings are provided for power-vented, condensing storage, and whole-house tankless units meeting specific EF criteria.
- b) Early replacement:
The early removal of an existing functioning natural gas water heater from service, prior to its natural end of life, and replacement with a new high efficiency unit. Savings are calculated between existing unit and efficient unit consumption during the remaining life of the existing unit, and between new baseline unit and efficient unit consumption for the remainder of the measure life.

This measure was developed to be applicable to the following program types: TOS, NC, EREP.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the efficient equipment must be a water heater rated with the following minimum efficiency ratings:

Water Heater Type	Minimum Energy Factor
Gas Storage	0.67
Condensing gas storage	0.80
Tankless whole-house unit	0.82

DEFINITION OF BASELINE EQUIPMENT

Time of Sale or New Construction: The baseline condition is assumed to be a standard gas storage water heater of the same capacity as the efficient unit, rated at the federal minimum. For 20 to 55 gallon tanks the Federal Standard is calculated as $0.675 - (0.0015 * \text{storage size in gallons})$ and for tanks 55 - 100 gallon $0.8012 - (0.00078 * \text{storage size in gallons})$ ⁵²⁵. For a 40-gallon storage water heater this would be 0.615 EF.

Early replacement: The baseline for this measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and a new baseline unit for the remainder of the measure life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 13 years.⁵²⁶

For early replacement: Remaining life of existing equipment is assumed to be 4 years⁵²⁷.

⁵²⁵ Minimum Federal Standard as of 4/1/2015;

<http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf>

⁵²⁶ DOE, 2010 Residential Heating Products Final Rule Technical Support Document, Table 8.2.14

http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/htgp_finalrule_ch8.pdf Note: This source is used to support this category in aggregate. For all water heaters, life expectancy will depend on local variables such as water chemistry and homeowner maintenance. Some categories, including condensing storage and tankless water heaters do not yet have sufficient field data to support separate values. Preliminary data show lifetimes may exceed 20 years, though this has yet to be sufficiently demonstrated.

⁵²⁷ Assumed to be one third of effective useful life

DEEMED MEASURE COST

Time of Sale or New Construction:

The incremental capital cost for this measure is dependent on the type of water heater as listed below⁵²⁸.

Early Replacement: The full installed cost is provided in the table below. The assumed deferred cost (after 4 years) of replacing existing equipment with a new baseline unit is assumed to be \$650⁵²⁹. This cost should be discounted to present value using the nominal discount rate.

Water heater Type	Incremental Cost	Full Install Cost
Gas Storage	\$400	\$1014
Condensing gas storage	\$685	\$1299
Tankless whole-house unit	\$605	\$1219

LOADSHAPE

N/A

COINCIDENCE FACTOR

N/A

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

N/A

SUMMER COINCIDENT PEAK DEMAND SAVINGS

N/A

NATURAL GAS ENERGY SAVINGS

Time of Sale or New Construction:

$$\Delta \text{Therms} = (1/EF_{\text{BASE}} - 1/EF_{\text{EFFICIENT}}) * (\text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 100,000$$

Early replacement⁵³⁰:

ΔTherms for remaining life of existing unit (1st 4 years):

$$= (1/EF_{\text{EXISTING}} - 1/EF_{\text{EFFICIENT}}) * (\text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 100,000$$

ΔTherms for remaining measure life (next 9 years):

⁵²⁸ Source for cost info; DOE, 2010 Residential Heating Products Final Rule Technical Support Document, Table 8.2.14 (http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/htgp_finalrule_ch8.pdf)

⁵²⁹ The deemed install cost of a Gas Storage heater is based upon DCEO Efficient Living Program Data for a sample size of 157 gas water heaters, and applying inflation rate of 1.91%.

⁵³⁰ The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a “number of years to adjustment” and “savings adjustment” input which would be the (new base to efficient savings)/(existing to efficient savings).

$$= (1/EF_{BASE} - 1/EF_{EFFICIENT}) * (GPD * Household * 365.25 * \gamma_{Water} * (T_{OUT} - T_{IN}) * 1.0) / 100,000$$

Where:

EF_Baseline = Energy Factor rating for baseline equipment

For <=55 gallons: 0.675 – (0.0015 * tank_size)

For > 55 gallons: 0.8012 – (0.00078 * tank size)

= If tank size unknown assume 40 gallons and EF_Baseline of 0.615

EF_Efficient = Energy Factor Rating for efficient equipment

= Actual. If Tankless whole-house multiply rated efficiency by 0.91⁵³¹. If unknown assume values in look up in table below

Water Heater Type	EF_Efficient
Condensing Gas Storage	0.80
Gas Storage	0.67
Tankless whole-house	0.82 * 0.91 = 0.75

EF_Existing = Energy Factor rating for existing equipment

= Use actual EF rating where it is possible to measure or reasonably estimate.

= if unknown assume 0.52⁵³²

GPD = Gallons Per Day of hot water use per person

= 45.5 gallons hot water per day per household/2.59 people per household⁵³³

= 17.6

Household = Average number of people per household

Household Unit Type	Household
Single-Family - Deemed	2.56 ⁵³⁴
Multi-Family - Deemed	2.1 ⁵³⁵
Custom	Actual Occupancy or Number of Bedrooms ⁵³⁶

365.25 = Days per year, on average

⁵³¹ The disconnect between rated energy factor and in-situ energy consumption is markedly different for tankless units due to significantly higher contributions to overall household hot water usage from short draws. In tankless units the large burner and unit heat exchanger must fire and heat up for each draw. The additional energy losses incurred when the mass of the unit cools to the surrounding space in-between shorter draws was found to be 9% in a study prepared for Lawrence Berkeley National Laboratory by Davis Energy Group, 2006. "Field and Laboratory Testing of Tankless Gas Water Heater Performance" Due to the similarity (storage) between the other categories and the baseline, this derating factor is applied only to the tankless category.

⁵³² Based on DCEO Efficient Living Program Data for a sample size of 157 gas water heaters.

⁵³³ Deoreo, B., and P. Mayer. Residential End Uses of Water Study Update. Forthcoming. ©2015 Water Research Foundation. Reprinted With Permission.

⁵³⁴ ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program citing 2006-2008 American Community Survey data from the US Census Bureau for Illinois cited on p. 17 of the PY2 Evaluation report. 2.75 * 93% evaluation adjustment

⁵³⁵ Navigant, ComEd PY3 Multi-Family Home Energy Savings Program Evaluation Report Final, May 16, 2012.

⁵³⁶ Bedrooms are suitable proxies for household occupancy, and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.

γ_{Water}	= Specific Weight of water = 8.33 pounds per gallon
T_{OUT}	= Tank temperature = 125°F
T_{IN}	= Incoming water temperature from well or municipal system = 54°F ⁵³⁷
1.0	= Heat Capacity of water (1 Btu/lb*°F)

For example, a 40 gallon condensing gas storage water heater, with an energy factor of 0.80 in a single family house:

$$\Delta \text{Therms} = (1/0.615 - 1/0.8) * (17.6 * 2.56 * 365.25 * 8.33 * (125 - 54) * 1) / 100,000$$

$$= 36.6 \text{ therms}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-HWE-GWHT-V07-180101

REVIEW DEADLINE: 1/1/2021

⁵³⁷ US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL
http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html

5.4.3 Heat Pump Water Heaters

DESCRIPTION

The installation of a heat pump domestic hot water heater in place of a standard electric water heater in a home. Savings are presented dependent on the heating system installed in the home due to the impact of the heat pump water heater on the heating loads.

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a Heat Pump domestic water heater.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is assumed to be a new electric water heater meeting federal minimum efficiency standards⁵³⁸:

For ≤55 gallons:	$0.96 - (0.0003 * \text{rated volume in gallons})$
For >55 gallons:	$2.057 - (0.00113 * \text{rated volume in gallons})$

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 13 years.⁵³⁹

DEEMED MEASURE COST

The incremental capital cost for this measure is \$1,000, for a HPWH with an energy factor of 2.0. The full cost, applicable in a retrofit, is \$1,575. For a HPWH with an energy factor of 2.35, these costs are \$1,134 and \$1,703 respectively.⁵⁴⁰

LOADSHAPE

Loadshape R03 - Residential Electric DHW

COINCIDENCE FACTOR

The summer Peak Coincidence Factor is assumed to be 12%.⁵⁴¹

⁵³⁸ Minimum Federal Standard as of 4/1/2015;

<http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf>

⁵³⁹ DOE, 2010 Residential Heating Products Final Rule Technical Support Document, Page 8-52
http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/htgp_finalrule_ch8.pdf

⁵⁴⁰ DOE, 2010 Residential Heating Products Final Rule Technical Support Document, Table 8.2.14
http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/htgp_finalrule_ch8.pdf

⁵⁴¹ Calculated from Figure 8 "Combined six-unit summer weekday average electrical demand" in FEMP study; Field Testing of Pre-Production Prototype Residential Heat Pump Water Heaters
http://www1.eere.energy.gov/femp/pdfs/tir_heatpump.pdf as $(\text{average kW usage during peak period} * \text{hours in peak period}) / [(\text{annual kWh savings} / \text{FLH}) * \text{hours in peak period}] = (0.1 \text{ kW} * 5 \text{ hours}) / [(2100 \text{ kWh (default assumptions)} / 2533 \text{ hours}) * 5 \text{ hours}] = 0.12$

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = (((1/\text{EF}_{\text{BASE}} - 1/\text{EF}_{\text{EFFICIENT}}) * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412) + \text{kWh}_{\text{cooling}} - \text{kWh}_{\text{heating}}$$

Where:

EF_{BASE} = Energy Factor (efficiency) of standard electric water heater according to federal standards⁵⁴²:

For ≤55 gallons: $0.96 - (0.0003 * \text{rated volume in gallons})$

For >55 gallons: $2.057 - (0.00113 * \text{rated volume in gallons})$
 = 0.945 for a 50 gallon tank, the most common size for HPWH

$\text{EF}_{\text{EFFICIENT}}$ = Energy Factor (efficiency) of Heat Pump water heater
 = Actual

GPD = Gallons Per Day of hot water use per person
 = 45.5 gallons hot water per day per household/2.59 people per household⁵⁴³
 = 17.6

Household = Average number of people per household

Household Unit Type	Household
Single-Family - Deemed	2.56 ⁵⁴⁴
Multi-Family - Deemed	2.1 ⁵⁴⁵
Custom	Actual Occupancy or Number of Bedrooms ⁵⁴⁶

365.25 = Days per year

γ_{Water} = Specific weight of water
 = 8.33 pounds per gallon

T_{OUT} = Tank temperature
 = 125°F

T_{IN} = Incoming water temperature from well or municipal system

⁵⁴² Minimum Federal Standard as of 1/1/2015;

<http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf>

⁵⁴³ Deoreo, B., and P. Mayer. Residential End Uses of Water Study Update. Forthcoming. ©2015 Water Research Foundation. Reprinted With Permission.

⁵⁴⁴ ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program citing 2006-2008 American Community Survey data from the US Census Bureau for Illinois cited on p. 17 of the PY2 Evaluation report. 2.75 * 93% evaluation adjustment

⁵⁴⁵ Navigant, ComEd PY3 Multi-Family Home Energy Savings Program Evaluation Report Final, May 16, 2012.

⁵⁴⁶ Bedrooms are suitable proxies for household occupancy, and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.

$$= 54^{\circ}\text{F}^{547}$$

1.0 = Heat Capacity of water (1 Btu/lb*°F)

3412 = Conversion from Btu to kWh

$$\begin{aligned} \text{kWh}_{\text{cooling}}^{548} &= \text{Cooling savings from conversion of heat in home to water heat} \\ &= (((((\text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412) - \\ &\quad ((1 / \text{EF}_{\text{NEW}} * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412)) * \text{LF} \\ &\quad * 27\%) / \text{COP}_{\text{COOL}}) * \text{LM} \end{aligned}$$

Where:

LF = Location Factor

= 1.0 for HPWH installation in a conditioned space

= 0.5 for HPWH installation in an unknown location

= 0.0 for installation in an unconditioned space

27% = Portion of reduced waste heat that results in cooling savings⁵⁴⁹

COP_{COOL} = COP of central air conditioning

= Actual, if unknown, assume 3.08 (10.5 SEER / 3.412)

LM = Latent multiplier to account for latent cooling demand

= 1.33⁵⁵⁰

kWh_{heating} = Heating cost from conversion of heat in home to water heat (dependent on heating fuel)

$$= (((((\text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412) -$$

$$((1 / \text{EF}_{\text{NEW}} * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412)) * \text{LF} * 49\%) / \text{COP}_{\text{HEAT}}) * (1 - \% \text{NaturalGas})$$

Where:

49% = Portion of reduced waste heat that results in increased heating load⁵⁵¹

COP_{HEAT} = COP of electric heating system

= actual. If not available use⁵⁵²:

⁵⁴⁷ US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html

⁵⁴⁸ This algorithm calculates the heat removed from the air by subtracting the HPWH electric consumption from the total water heating energy delivered. This is then adjusted to account for location of the HP unit and the coincidence of the waste heat with cooling requirements, the efficiency of the central cooling and latent cooling demands.

⁵⁴⁹ REMRate determined percentage (27%) of lighting savings that result in reduced cooling loads (lighting is used as a proxy for hot water heating since load shapes suggest their seasonal usage patterns are similar).

⁵⁵⁰ A sensible heat ratio (SHR) of 0.75 corresponds to a latent multiplier of 4/3 or 1.33. SHR of 0.75 for typical split system from page 10 of "Controlling Indoor Humidity Using Variable-Speed Compressors and Blowers" by M. A. Andrade and C. W. Bullard, 1999: www.ideals.illinois.edu/bitstream/handle/2142/11894/TR151.pdf

⁵⁵¹ REMRate determined percentage (49%) of lighting savings that result in increased heating loads (lighting is used as a proxy for hot water heating since load shapes suggest their seasonal usage patterns are similar).

⁵⁵² These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. Note efficiency should include duct losses. Defaults provided assume 15% duct loss for heat pumps.

System Type	Age of Equipment	HSPF Estimate	COP _{HEAT} (COP Estimate) = (HSPF/3.413)*0.85
Heat Pump	Before 2006	6.8	1.7
	After 2006 - 2014	7.7	1.92
	2015 on	8.2	2.04
Resistance	N/A	N/A	1.00
Unknown ⁵⁵³	N/A	N/A	1.28

For example, a 2.0 EF heat pump water heater, in a conditioned space in a single family home with gas space heat and central air conditioning (SEER 10.5) in Belleville:

$$\Delta \text{kWh} = [(1 / 0.945 - 1 / 2.0) * 17.6 * 2.56 * 365.25 * 8.33 * (125 - 54)] / 3412 + 166.3 - 0$$

$$= 1759 \text{ kWh}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \Delta \text{kWh} / \text{Hours} * \text{CF}$$

Where:

$$\text{Hours} = \text{Full load hours of water heater}$$

$$= 2533^{554}$$

$$\text{CF} = \text{Summer Peak Coincidence Factor for measure}$$

$$= 0.12^{555}$$

For example, a 2.0 COP heat pump water heater, in a conditioned space in a single family home with gas space heat and central air conditioning in Belleville:

$$\text{kW} = 1759 / 2533 * 0.12$$

$$= 0.083 \text{ kW}$$

NATURAL GAS SAVINGS

$$\Delta \text{Therms} = - (((\text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412) - (((\text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) / 3412) / \text{EF}_{\text{EFFICIENT}})) * \text{LF} * 49\% * 0.03412) / \eta_{\text{Heat}} * \% \text{NaturalGas}$$

Where:

$$\Delta \text{Therms} = \text{Heating cost from conversion of heat in home to water heat for homes with Natural Gas}$$

⁵⁵³ Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see "HC6.9 Space Heating in Midwest Region.xls", using average for East North Central Region. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

⁵⁵⁴ Full load hours assumption based on Efficiency Vermont analysis of Itron eShapes.

⁵⁵⁵ Calculated from Figure 8 "Combined six-unit summer weekday average electrical demand" in FEMP study; Field Testing of Pre-Production Prototype Residential Heat Pump Water Heaters http://www1.eere.energy.gov/femp/pdfs/tir_heatpump.pdf as (average kW usage during peak period * hours in peak period) / [(annual kWh savings / FLH) * hours in peak period] = (0.1 kW * 5 hours) / [(2100 kWh / 2533 hours) * 5 hours] = 0.12

heat^{556}
0.03412 = conversion factor (therms per kWh)
 η_{Heat} = Efficiency of heating system
= Actual.⁵⁵⁷ If not available use 70%.⁵⁵⁸
%NaturalGas = Factor dependent on heating fuel:

Heating System	%NaturalGas
Electric resistance or heat pump	0%
Natural Gas	100%
Unknown heating fuel ⁵⁵⁹	87%

Other factors as defined above

For example, a 2.0 COP heat pump water heater in conditioned space, in a single family home with gas space heat (70% system efficiency):

$$\begin{aligned}
 \Delta \text{Therms} &= -(((17.6 * 2.56 * 365.25 * 8.33 * (125 - 54) * 1.0) / 3412) - (17.6 * 2.56 * 365.25 * 8.33 \\
 &\quad * (125 - 54) * 1.0 / 3412 / 2.0)) * 1 * 0.49 * 0.03412) / (0.7 * 1) \\
 &= - 34.1 \text{ therms}
 \end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-HWE-HPWH-V06-180101

REVIEW DEADLINE: 1/1/2021

⁵⁵⁶ This is the additional energy consumption required to replace the heat removed from the home during the heating season by the heat pump water heater. kWh_heating (electric resistance) is that additional heating energy for a home with electric resistance heat (COP 1.0). This formula converts the additional heating kWh for an electric resistance home to the MMBtu required in a Natural Gas heated home, applying the relative efficiencies.

⁵⁵⁷ Ideally, the System Efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. The Distribution Efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute: (<http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf>) or by performing duct blaster testing.

⁵⁵⁸ This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: <http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls>)) In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:

$(0.24 * 0.92) + (0.76 * 0.8) * (1 - 0.15) = 0.70$

⁵⁵⁹ 2010 American Community Survey.

5.4.4 Low Flow Faucet Aerators

DESCRIPTION

This measure relates to the installation of a low flow faucet aerator in a household kitchen or bath faucet fixture.

This measure may be used for units provided through Efficiency Kit's however the in service rate for such measures should be derived through evaluation results specifically for this implementation methodology.

This measure was developed to be applicable to the following program types: TOS, NC, RF, DI, KITS.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a low flow faucet aerator, for bathrooms rated at 1.5 gallons per minute (GPM) or less, or for kitchens rated at 2.2 GPM or less. Savings are calculated on an average savings per faucet fixture basis.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is assumed to be a standard bathroom faucet aerator rated at 2.25 GPM or greater, or a standard kitchen faucet aerator rated at 2.75 GPM or greater. Average measured flow rates are used in the algorithm and are lower, reflecting the penetration of previously installed low flow fixtures (and therefore the freerider rate for this measure should be 0), use of the faucet at less than full flow, debris buildup, and lower water system pressure than fixtures are rated at.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 9 years.⁵⁶⁰

DEEMED MEASURE COST

For time of sale or new construction the incremental cost for this measure is \$3⁵⁶¹ or program actual.

For faucet aerators provided through Direct Install or within Efficiency Kits, the actual program delivery costs (including labor if applicable) should be utilized. If unknown assume \$8⁵⁶² for Direct Install and \$3 for Efficiency Kits.

LOADSHAPE

Loadshape R03 - Residential Electric DHW

COINCIDENCE FACTOR

The coincidence factor for this measure is assumed to be 2.2%.⁵⁶³

⁵⁶⁰ Table C-6, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. "http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf"

⁵⁶¹ 2011, Market research average of \$3.

⁵⁶² Includes assess and install labor time of \$5 (20min @ \$15/hr)

⁵⁶³ Calculated as follows: Assume 18% aerator use takes place during peak hours (based on: Oreo et al, "The end uses of hot water in single family homes from flow trace analysis", 2001.) There are 65 days in the summer peak period, so the percentage of total annual aerator use in peak period is $0.18 \times 65 / 365 = 3.21\%$. The number of hours of recovery during peak periods is therefore assumed to be $3.21\% \times 180 = 5.8$ hours of recovery during peak period where 180 equals the average annual electric DHW recovery hours for faucet use including SF and MF homes. There are 260 hours in the peak period so the probability you will see savings during the peak period is $5.8 / 260 = 0.022$

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

Note these savings are *per* faucet retrofitted⁵⁶⁴ (unless faucet type is unknown, then it is per household).

$$\Delta \text{kWh} = \% \text{ElectricDHW} * ((\text{GPM}_{\text{base}} * \text{L}_{\text{base}} - \text{GPM}_{\text{low}} * \text{L}_{\text{low}}) * \text{Household} * 365.25 * \text{DF} / \text{FPH}) * \text{EPG}_{\text{electric}} * \text{ISR}$$

Where:

$\% \text{ElectricDHW}$ = proportion of water heating supplied by electric resistance heating

DHW fuel	$\% \text{ElectricDHW}$
Electric	100%
Natural Gas	0%
Unknown	16% ⁵⁶⁵

GPM_{base} = Average flow rate, in gallons per minute, of the baseline faucet “as-used.” This includes the effect of existing low flow fixtures and therefore the freerider rate for this measure should be 0.

= 1.39⁵⁶⁶ or custom based on metering studies⁵⁶⁷ or if measured during DI:

= Measured full throttle flow * 0.83 throttling factor⁵⁶⁸

GPM_{low} = Average flow rate, in gallons per minute, of the low-flow faucet aerator “as-used”

= 0.94⁵⁶⁹ or custom based on metering studies⁵⁷⁰ or if measured during DI:

= Rated full throttle flow * 0.95 throttling factor⁵⁷¹

⁵⁶⁴ This algorithm calculates the amount of energy saved per aerator by determining the fraction of water consumption savings for the upgraded fixture.

⁵⁶⁵ Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

⁵⁶⁶ Deoreo, B., and P. Mayer. Residential End Uses of Water Study Update. Forthcoming. ©2015 Water Research Foundation. Reprinted With Permission.

⁵⁶⁷ Measurement should be based on actual average flow consumed over a period of time rather than a onetime spot measurement for maximum flow. Studies have shown maximum flow rates do not correspond well to average flow rate due to occupant behavior which does not always use maximum flow.

⁵⁶⁸ 2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings. Page 1-265. www.seattle.gov/light/Conserve/Reports/paper_10.pdf

⁵⁶⁹ Average retrofit flow rate for kitchen and bathroom faucet aerators from sources 2, 4, 5, and 7 (see source table at end of characterization). This accounts for all throttling and differences from rated flow rates. Assumes all kitchen aerators at 2.2 gpm or less and all bathroom aerators at 1.5 gpm or less. The most comprehensive available studies did not disaggregate kitchen use from bathroom use, but instead looked at total flow and length of use for all faucets. This makes it difficult to reliably separate kitchen water use from bathroom water use. It is possible that programs installing low flow aerators lower than the 2.2 gpm for kitchens and 1.5 gpm for bathrooms will see a lower overall average retrofit flow rate.

⁵⁷⁰ Measurement should be based on actual average flow consumed over a period of time rather than a onetime spot measurement for maximum flow. Studies have shown maximum flow rates do not correspond well to average flow rate due to occupant behavior which does not always use maximum flow.

⁵⁷¹ 2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings. Page 1-265.

L_base = Average baseline daily length faucet use per capita for faucet of interest in minutes

= if available custom based on metering studies, if not use:

Faucet Type	L_base (min/person/day)
Kitchen	4.5 ⁵⁷²
Bathroom	1.6 ⁵⁷³
If location unknown (total for household): Single-Family	9.0 ⁵⁷⁴
If location unknown (total for household): Multi-Family	6.9 ⁵⁷⁵

L_low = Average retrofit daily length faucet use per capita for faucet of interest in minutes

= if available custom based on metering studies, if not use:

Faucet Type	L_low (min/person/day)
Kitchen	4.5 ⁵⁷⁶
Bathroom	1.6 ⁵⁷⁷
If location unknown (total for household): Single-Family	9.0 ⁵⁷⁸
If location unknown (total for household): Multi-Family	6.9 ⁵⁷⁹

Household = Average number of people per household

Household Unit Type	Household
Single-Family - Deemed	2.56 ⁵⁸⁰
Multi-Family - Deemed	2.1 ⁵⁸¹
Custom	Actual Occupancy or Number of Bedrooms ⁵⁸²

www.seattle.gov/light/Conserve/Reports/paper_10.pdf

⁵⁷² Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group. This study of 135 single and multi-family homes in Michigan metered energy parameters for efficient showerhead and faucet aerators.

⁵⁷³ Ibid.

⁵⁷⁴ One kitchen faucet plus 2.83 bathroom faucets. Based on findings from a 2009 ComEd residential survey of 140 sites, provided by Cadmus.

⁵⁷⁵ One kitchen faucet plus 1.5 bathroom faucets. Based on findings from a 2009 ComEd residential survey of 140 sites, provided by Cadmus.

⁵⁷⁶ Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group.

⁵⁷⁷ Ibid.

⁵⁷⁸ One kitchen faucet plus 2.83 bathroom faucets. Based on findings from a 2009 ComEd residential survey of 140 sites, provided by Cadmus.

⁵⁷⁹ One kitchen faucet plus 1.5 bathroom faucets. Based on findings from a 2009 ComEd residential survey of 140 sites, provided by Cadmus.

⁵⁸⁰ ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program citing 2006-2008 American Community Survey data from the US Census Bureau for Illinois cited on p. 17 of the PY2 Evaluation report. 2.75 * 93% evaluation adjustment

⁵⁸¹ Navigant, ComEd PY3 Multi-Family Home Energy Savings Program Evaluation Report Final, May 16, 2012.

⁵⁸² Bedrooms are suitable proxies for household occupancy, and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.

365.25 = Days in a year, on average.

DF = Drain Factor

Faucet Type	Drain Factor ⁵⁸³
Kitchen	75%
Bath	90%
Unknown	79.5%

FPH = Faucets Per Household

0	FPH
Kitchen Faucets Per Home (KFPH)	1
Bathroom Faucets Per Home (BFPH): Single-Family	2.83 ⁵⁸⁴
Bathroom Faucets Per Home (BFPH): Multi-Family	1.5 ⁵⁸⁵
If location unknown (total for household): Single-Family	3.83
If location unknown (total for household): Multi-Family	2.5

EPG_{electric} = Energy per gallon of water used by faucet supplied by electric water heater

$$= (8.33 * 1.0 * (\text{WaterTemp} - \text{SupplyTemp})) / (\text{RE}_{\text{electric}} * 3412)$$

$$= (8.33 * 1.0 * (86 - 54.1)) / (0.98 * 3412)$$

$$= 0.0795 \text{ kWh/gal (Bath)}, 0.0969 \text{ kWh/gal (Kitchen)}, 0.0919 \text{ kWh/gal (Unknown)}$$

8.33 = Specific weight of water (lbs/gallon)

1.0 = Heat Capacity of water (btu/lb-°F)

WaterTemp = Assumed temperature of mixed water
= 86F for Bath, 93F for Kitchen 91F for Unknown⁵⁸⁶

SupplyTemp = Assumed temperature of water entering house
= 54.1F⁵⁸⁷

RE_{electric} = Recovery efficiency of electric water heater

⁵⁸³ Because faucet usages are at times dictated by volume, only usage of the sort that would go straight down the drain will provide savings. VEIC is unaware of any metering study that has determined this specific factor and so through consensus with the Illinois Technical Advisory Group have deemed these values to be 75% for the kitchen and 90% for the bathroom. If the aerator location is unknown an average of 79.5% should be used which is based on the assumption that 70% of household water runs through the kitchen faucet and 30% through the bathroom $(0.7*0.75)+(0.3*0.9)=0.795$.

⁵⁸⁴Based on findings from a 2009 ComEd residential survey of 140 sites, provided by Cadmus.

⁵⁸⁵ Ibid.

⁵⁸⁶ Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group. If the aerator location is unknown an average of 91% should be used which is based on the assumption that 70% of household water runs through the kitchen faucet and 30% through the bathroom $(0.7*93)+(0.3*86)=0.91$.

⁵⁸⁷ US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL
http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html.

= 98%⁵⁸⁸

3412 = Converts Btu to kWh (btu/kWh)

ISR = In service rate of faucet aerators dependant on install method as listed in table below

Selection	ISR
Direct Install - Single Family	0.95 ⁵⁸⁹
Direct Install – Multi Family Kitchen	0.91 ⁵⁹⁰
Direct Install – Multi Family Bathroom	0.95 ⁵⁹¹
Efficiency Kit Bathroom Aerator	0.63 ⁵⁹²
Efficiency Kit Kitchen Aerator	0.60 ⁵⁹³
Distributed School Efficiency Kit Aerator	To be determined through evaluation

For example, a direct installed kitchen low flow faucet aerator in a single-family electric DHW home:

$$\Delta \text{kWh} = 1.0 * (((1.39 * 4.5 - 0.94 * 4.5) * 2.56 * 365.25 * 0.75) / 1) * 0.0969 * 0.95$$

$$= 131 \text{ kWh}$$

For example, a direct installed bath low flow faucet aerator in a multi-family electric DHW home:

$$\Delta \text{kWh} = 1.0 * (((1.39 * 1.6 - 0.94 * 1.6) * 2.1 * 365.25 * 0.90) / 1.5) * 0.0795 * 0.95$$

$$= 25.0 \text{ kWh}$$

For example, a direct installed low flow faucet aerator in unknown faucet in a single-family electric DHW home:

$$\Delta \text{kWh} = 1.0 * (((1.39 * 9.0 - 0.94 * 9.0) * 2.56 * 365.25 * 0.795) / 3.83) * 0.0919 * 0.95$$

$$= 68.6 \text{ kWh}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \Delta \text{kWh} / \text{Hours} * \text{CF}$$

Where:

⁵⁸⁸ Electric water heaters have recovery efficiency of 98%: <http://www.ahridirectory.org/ahridirectory/pages/home.aspx>

⁵⁸⁹ ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program Table 3-8

⁵⁹⁰ Navigant, ComEd-Nicor Gas EPY4/GPY1 Multi-Family Home Energy Savings Program Evaluation Report DRAFT 2013-01-28

⁵⁹¹ Ibid.

⁵⁹² From Navigant memo, "Nicor Gas energySMART Energy Saving Kits Program In Service Rate and Process Analysis", August 28, 2015.

⁵⁹³ Ibid.

ΔkWh = calculated value above

Hours = Annual electric DHW recovery hours for faucet use per faucet

$$= ((GPM_base * L_base) * Household / FPH * 365.25 * DF) * 0.545^{594} / GPH$$

Building Type	Faucet location	Calculation	Hours per faucet
Single Family	Kitchen	$((1.39 * 4.5) * 2.56 / 1 * 365.25 * 0.75) * 0.545 / 25.5$	94
	Bathroom	$((1.39 * 1.6) * 2.56 / 2.83 * 365.25 * 0.9) * 0.545 / 25.5$	14
	Unknown	$((1.39 * 9.0) * 2.56 / 3.83 * 365.25 * 0.795) * 0.545 / 25.5$	52
Multi Family	Kitchen	$((1.39 * 4.5) * 2.1 / 1 * 365.25 * 0.75) * 0.545 / 25.5$	77
	Bathroom	$((1.39 * 1.6) * 2.1 / 1.5 * 365.25 * 0.9) * 0.545 / 25.5$	22
	Unknown	$((1.39 * 6.9) * 2.1 / 2.5 * 365.25 * 0.795) * 0.545 / 25.5$	50

GPH = Gallons per hour recovery of electric water heater calculated for 70.9F temp rise (125-54.1), 98% recovery efficiency, and typical 4.5kW electric resistance storage tank.

$$= 25.5$$

CF = Coincidence Factor for electric load reduction

$$= 0.022^{595}$$

For example, a direct installed kitchen low flow faucet aerator in a single family electric DHW home:

$$\begin{aligned} \Delta kW &= 131 / 94 * 0.022 \\ &= 0.0306 \text{ kW} \end{aligned}$$

NATURAL GAS SAVINGS

$\Delta Therms$ = $\%FossilDHW * ((GPM_base * L_base - GPM_low * L_low) * Household * 365.25 * DF / FPH) * EPG_gas * ISR$

Where:

$\%FossilDHW$ = proportion of water heating supplied by Natural Gas heating

DHW fuel	$\%Fossil_DHW$
Electric	0%
Natural Gas	100%
Unknown	84% ⁵⁹⁶

EPG_gas = Energy per gallon of Hot water supplied by gas

⁵⁹⁴ 54.5% is the proportion of hot 120F water mixed with 54.1F supply water to give 90F mixed faucet water.

⁵⁹⁵ Calculated as follows: Assume 18% aerator use takes place during peak hours (based on: Oreo et al, "The end uses of hot water in single family homes from flow trace analysis", 2001.) There are 65 days in the summer peak period, so the percentage of total annual aerator use in peak period is $0.18 * 65 / 365 = 3.21\%$. The number of hours of recovery during peak periods is therefore assumed to be $3.21\% * 180 = 5.8$ hours of recovery during peak period where 180 equals the average annual electric DHW recovery hours for faucet use including SF and MF homes. There are 260 hours in the peak period so the probability you will see savings during the peak period is $5.8 / 260 = 0.022$

⁵⁹⁶ Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

$$\begin{aligned}
 &= (8.33 * 1.0 * (\text{WaterTemp} - \text{SupplyTemp})) / (\text{RE_gas} * 100,000) \\
 &= 0.00341 \text{ Therm/gal for SF homes (Bath)}, 0.00415 \text{ Therm/gal for SF homes (Kitchen)}, \\
 &\quad 0.00394 \text{ Therm/gal for SF homes (Unknown)} \\
 &= 0.00397 \text{ Therm/gal for MF homes (Bath)}, 0.00484 \text{ Therm/gal for MF homes (Kitchen)}, \\
 &\quad 0.00459 \text{ Therm/gal for MF homes (Unknown)} \\
 \text{RE_gas} &= \text{Recovery efficiency of gas water heater} \\
 &= 78\% \text{ For SF homes}^{597} \\
 &= 67\% \text{ For MF homes}^{598} \\
 100,000 &= \text{Converts Btus to Therms (btu/Therm)} \\
 &\text{Other variables as defined above.}
 \end{aligned}$$

For example, a direct-installed kitchen low flow faucet aerator in a fuel DHW single-family home:

$$\begin{aligned}
 \Delta\text{Therms} &= 1.0 * (((1.39 * 4.5 - 0.94 * 4.5) * 2.56 * 365.25 * 0.75) / 1) * 0.00415 * 0.95 \\
 &= 5.60 \text{ Therms}
 \end{aligned}$$

For example, a direct installed bath low flow faucet aerator in a fuel DHW multi-family home:

$$\begin{aligned}
 \Delta\text{Therms} &= 1.0 * (((1.39 * 1.6 - 0.94 * 1.6) * 2.1 * 365.25 * 0.90) / 1.5) * 0.003974 * 0.95 \\
 &= 1.25 \text{ Therms}
 \end{aligned}$$

For example, a direct installed low flow faucet aerator in unknown faucet in a fuel DHW single-family home:

$$\begin{aligned}
 \Delta\text{Therms} &= 1.0 * (((1.39 * 9.0 - 0.94 * 9.0) * 2.56 * 365.25 * 0.795) / 3.83) * 0.00394 * 0.95 \\
 &= 2.94 \text{ Therms}
 \end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

$$\begin{aligned}
 \Delta\text{gallons} &= ((\text{GPM_base} * \text{L_base} - \text{GPM_low} * \text{L_low}) * \text{Household} * 365.25 * \text{DF} / \text{FPH}) * \text{ISR} \\
 &\text{Variables as defined above}
 \end{aligned}$$

⁵⁹⁷ DOE Final Rule discusses Recovery Efficiency with an average around 0.76 for Gas Fired Storage Water heaters and 0.78 for standard efficiency gas fired tankless water heaters up to 0.95 for the highest efficiency gas fired condensing tankless water heaters. These numbers represent the range of new units however, not the range of existing units in stock. Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 78%.

⁵⁹⁸ Water heating in multi-family buildings is often provided by a larger central boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of 0.59 and the 0.75 for single family homes. An average efficiency of 0.67 is used for this analysis as a default for multi-family buildings.

For example, a direct-installed kitchen low flow aerator in a single family home

$$\Delta \text{gallons} = (((1.39 * 4.5 - 0.94 * 4.5) * 2.56 * 365.25 * 0.75) / 1) * 0.95$$

$$= 1350 \text{ gallons}$$

For example, a direct installed bath low flow faucet aerator in a multi-family home:

$$\Delta \text{gallons} = (((1.39 * 1.6 - 0.94 * 1.6) * 2.1 * 365.25 * 0.90) / 1.5) * 0.95$$

$$= 314 \text{ gallons}$$

For example, a direct installed low flow faucet aerator in unknown faucet in a single-family home:

$$\Delta \text{gallons} = (((1.39 * 9.0 - 0.94 * 9.0) * 2.56 * 365.25 * 0.795) / 3.83) * 0.95$$

$$= 747 \text{ gallons}$$

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

SOURCES

Source ID	Reference
1	2011, DeOreo, William. California Single Family Water Use Efficiency Study. April 20, 2011.
2	2000, Mayer, Peter, William DeOreo, and David Lewis. Seattle Home Water Conservation Study. December 2000.
3	1999, Mayer, Peter, William DeOreo. Residential End Uses of Water. Published by AWWA Research Foundation and American Water Works Association. 1999.
4	2003, Mayer, Peter, William DeOreo. Residential Indoor Water Conservation Study. Aquacraft, Inc. Water Engineering and Management. Prepared for East Bay Municipal Utility District and the US EPA. July 2003.
5	2011, DeOreo, William. Analysis of Water Use in New Single Family Homes. By Aquacraft. For Salt Lake City Corporation and US EPA. July 20, 2011.
6	2011, Aquacraft. Albuquerque Single Family Water Use Efficiency and Retrofit Study. For Albuquerque Bernalillo County Water Utility Authority. December 1, 2011.
7	2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings.

MEASURE CODE: RS-HWE-LFFA-V06-180101

REVIEW DEADLINE: 1/1/2021

5.4.5 Low Flow Showerheads

DESCRIPTION

This measure relates to the installation of a low flow showerhead in a single or multi-family household.

This measure may be used for units provided through Efficiency Kit's however the in service rate for such measures should be derived through evaluation results specifically for this implementation methodology.

This measure was developed to be applicable to the following program types: TOS, RF, NC, DI, KITS.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a low flow showerhead rated at 2.0 gallons per minute (GPM) or less. Savings are calculated on a per showerhead fixture basis.

DEFINITION OF BASELINE EQUIPMENT

For Direct-install programs, the baseline condition is assumed to be a standard showerhead rated at 2.5 GPM or greater.

For retrofit and time-of-sale programs, the baseline condition is assumed to be a representative average of existing showerhead flow rates of participating customers including a range of low flow showerheads, standard-flow showerheads, and high-flow showerheads.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years.⁵⁹⁹

DEEMED MEASURE COST

For time of sale or new construction the incremental cost for this measure is \$7⁶⁰⁰ or program actual.

For low flow showerheads provided through Direct Install or within Efficiency Kits, the actual program delivery costs (including labor if applicable) should be utilized. If unknown assume \$12⁶⁰¹ for Direct Install and \$7 for Efficiency Kits.

LOADSHAPE

Loadshape R03 - Residential Electric DHW

COINCIDENCE FACTOR

The coincidence factor for this measure is assumed to be 2.78%.⁶⁰²

⁵⁹⁹ Table C-6, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. Evaluations indicate that consumer dissatisfaction may lead to reductions in persistence, particularly in Multi-Family, ["http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf"](http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf)

⁶⁰⁰ Market research average of \$7.

⁶⁰¹ Includes assess and install labor time of \$5 (20min @ \$15/hr)

⁶⁰² Calculated as follows: Assume 11% showers take place during peak hours (based on: Oreo et al, "The end uses of hot water in single family homes from flow trace analysis", 2001.). There are 65 days in the summer peak period, so the percentage of total annual aerator use in peak period is $0.11 * 65 / 365 = 1.96\%$. The number of hours of recovery during peak periods is therefore assumed to be $1.96\% * 369 = 7.23$ hours of recovery during peak period, where 369 equals the average annual electric DHW recovery hours for showerhead use including SF and MF homes with Direct Install and Retrofit/TOS measures. There are 260 hours in the peak period so the probability you will see savings during the peak period is $7.23 / 260 = 0.0278$

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

Note these savings are per showerhead fixture

$$\Delta \text{kWh} = \% \text{ElectricDHW} * ((\text{GPM}_{\text{base}} * \text{L}_{\text{base}} - \text{GPM}_{\text{low}} * \text{L}_{\text{low}}) * \text{Household} * \text{SPCD} * 365.25 / \text{SPH}) * \text{EPG}_{\text{electric}} * \text{ISR}$$

Where:

$\% \text{ElectricDHW}$ = proportion of water heating supplied by electric resistance heating

DHW fuel	$\% \text{ElectricDHW}$
Electric	100%
Natural Gas	0%
Unknown	16% ⁶⁰³

GPM_{base} = Flow rate of the baseline showerhead

Program	GPM_{base}
Direct-install	2.67 ⁶⁰⁴
Retrofit, Efficiency Kits, NC or TOS	2.35 ⁶⁰⁵

GPM_{low} = As-used flow rate of the low-flow showerhead, which may, as a result of measurements of program evaluations deviate from rated flows, see table below:

Rated Flow
2.0 GPM
1.75 GPM
1.5 GPM
Custom or Actual ⁶⁰⁶

L_{base} = Shower length in minutes with baseline showerhead
= 7.8 min⁶⁰⁷

⁶⁰³ Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

⁶⁰⁴ Based on measured data from Ameren IL EM&V of Direct-Install program. Program targets showers that are rated 2.5 GPM or above.

⁶⁰⁵ Representative value from sources 1, 2, 4, 5, 6 and 7 (See Source Table at end of measure section) adjusted slightly upward to account for program participation which is expected to target customers with existing higher flow devices rather than those with existing low flow devices.

⁶⁰⁶ Note that actual values may be either a) program-specific minimum flow rate, or b) program-specific evaluation-based value of actual effective flow-rate due to increased duration or temperatures. The latter increases in likelihood as the rated flow drops and may become significant at or below rated flows of 1.5 GPM. The impact can be viewed as the inverse of the throttling described in the footnote for baseline flowrate.

⁶⁰⁷ Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group. This study of 135 single and multi-family homes in Michigan metered energy parameters

L_{low} = Shower length in minutes with low-flow showerhead
 = 7.8 min⁶⁰⁸

Household = Average number of people per household

Household Unit Type ⁶⁰⁹	Household
Single-Family - Deemed	2.56 ⁶¹⁰
Multi-Family - Deemed	2.1 ⁶¹¹
Custom	Actual Occupancy or Number of Bedrooms ⁶¹²

SPCD = Showers Per Capita Per Day
 = 0.6⁶¹³

365.25 = Days per year, on average.

SPH = Showerheads Per Household so that per-showerhead savings fractions can be determined

Household Type	SPH
Single-Family	1.79 ⁶¹⁴
Multi-Family	1.3 ⁶¹⁵
Custom	Actual

EPG_{electric} = Energy per gallon of hot water supplied by electric
 = $(8.33 * 1.0 * (\text{ShowerTemp} - \text{SupplyTemp})) / (\text{RE}_{\text{electric}} * 3412)$
 = $(8.33 * 1.0 * (101 - 54.1)) / (0.98 * 3412)$
 = 0.117 kWh/gal

8.33 = Specific weight of water (lbs/gallon)

1.0 = Heat Capacity of water (btu/lb-°)

ShowerTemp = Assumed temperature of water
 = 101F⁶¹⁶

for efficient showerhead and faucet aerators.

⁶⁰⁸ Ibid.

⁶⁰⁹ If household type is unknown, as may be the case for time of sale measures, then single family deemed value shall be used.

⁶¹⁰ ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program citing 2006-2008 American Community Survey data from the US Census Bureau for Illinois cited on p. 17 of the PY2 Evaluation report. 2.75 * 93% evaluation adjustment

⁶¹¹ ComEd PY3 Multi-Family Evaluation Report REVISED DRAFT v5 2011-12-08.docx

⁶¹² Bedrooms are suitable proxies for household occupancy, and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.

⁶¹³ Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group.

⁶¹⁴ Based on findings from a 2009 ComEd residential survey of 140 sites, provided by Cadmus.

⁶¹⁵ Ibid.

⁶¹⁶ Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group.

SupplyTemp	= Assumed temperature of water entering house = 54.1F ⁶¹⁷
RE_electric	= Recovery efficiency of electric water heater = 98% ⁶¹⁸
3412	= Converts Btu to kWh (btu/kWh)
ISR	= In service rate of showerhead = Dependant on program delivery method as listed in table below

Selection	ISR
Direct Install - Single Family	0.98 ⁶¹⁹
Direct Install – Multi Family	0.95 ⁶²⁰
Efficiency Kits--One showerhead kit	0.65 ⁶²¹
Efficiency Kits—Two showerhead kit	0.67 ⁶²²
Distributed School Efficiency Kit showerhead	To be determined through evaluation

For example, a direct-installed 1.5 GPM low flow showerhead in a single family home with electric DHW where the number of showers is not known:

$$\Delta \text{kWh} = 1.0 * ((2.67 * 7.8 - 1.5 * 7.8) * 2.56 * 0.6 * 365.25 / 1.79) * 0.117 * 0.98$$

$$= 328 \text{ kWh}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \Delta \text{kWh} / \text{Hours} * \text{CF}$$

Where:

ΔkWh = calculated value above

Hours = Annual electric DHW recovery hours for showerhead use

$$= ((\text{GPM_base} * \text{L_base}) * \text{Household} * \text{SPCD} * 365.25) * 0.712^{623} / \text{GPH}$$

= 302 for SF Direct Install; 248 for MF Direct Install

= 266 for SF Retrofit, Efficiency Kits, NC and TOS; 218 for MF Retrofit, Efficiency Kits, NC and TOS

GPH = Gallons per hour recovery of electric water heater calculated for 65.9F temp rise (120-54.1), 98% recovery efficiency, and typical 4.5kW electric resistance storage tank.

⁶¹⁷ US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html.

⁶¹⁸ Electric water heaters have recovery efficiency of 98%: <http://www.ahridirectory.org/ahridirectory/pages/home.aspx>

⁶¹⁹ Deemed values are from ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program Table 3-8. Alternative ISRs may be developed for program delivery methods based on evaluation results.

⁶²⁰ Navigant, ComEd-Nicor Gas EPY4/GPY1 Multi-Family Home Energy Savings Program Evaluation Report FINAL 2013-06-05

⁶²¹ From Navigant memo, "Nicor Gas energySMART Energy Saving Kits Program In Service Rate and Process Analysis", August 28, 2015.

⁶²² Ibid

⁶²³ 71.2% is the proportion of hot 120F water mixed with 54.1F supply water to give 101F shower water.

$$= 27.51$$

CF = Coincidence Factor for electric load reduction

$$= 0.0278^{624}$$

For example, a direct installed 1.5 GPM low flow showerhead in a single family home with electric DHW where the number of showers is not known:

$$\Delta kW = 328/302 * 0.0278$$

$$= 0.0302 \text{ kW}$$

NATURAL GAS SAVINGS

$$\Delta \text{Therms} = \% \text{FossilDHW} * ((\text{GPM}_{\text{base}} * L_{\text{base}} - \text{GPM}_{\text{low}} * L_{\text{low}}) * \text{Household} * \text{SPCD} * 365.25 / \text{SPH}) * \text{EPG}_{\text{gas}} * \text{ISR}$$

Where:

$\% \text{FossilDHW}$ = proportion of water heating supplied by Natural Gas heating

DHW fuel	$\% \text{Fossil_DHW}$
Electric	0%
Natural Gas	100%
Unknown	84% ⁶²⁵

EPG_{gas} = Energy per gallon of Hot water supplied by gas
 $= (8.33 * 1.0 * (\text{ShowerTemp} - \text{SupplyTemp})) / (\text{RE}_{\text{gas}} * 100,000)$
 $= 0.00501 \text{ Therm/gal for SF homes}$
 $= 0.00583 \text{ Therm/gal for MF homes}$

RE_{gas} = Recovery efficiency of gas water heater
 $= 78\% \text{ For SF homes}^{626}$
 $= 67\% \text{ For MF homes}^{627}$

⁶²⁴ Calculated as follows: Assume 11% showers take place during peak hours (based on: Oreo et al, "The end uses of hot water in single family homes from flow trace analysis", 2001.). There are 65 days in the summer peak period, so the percentage of total annual aerator use in peak period is $0.11 * 65 / 365 = 1.96\%$. The number of hours of recovery during peak periods is therefore assumed to be $1.96\% * 369 = 7.23$ hours of recovery during peak period where 369 equals the average annual electric DHW recovery hours for showerhead use including SF and MF homes with Direct Install and Retrofit/TOS measures. There are 260 hours in the peak period so the probability you will see savings during the peak period is $7.23 / 260 = 0.0278$

⁶²⁵ Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

⁶²⁶ DOE Final Rule discusses Recovery Efficiency with an average around 0.76 for Gas Fired Storage Water heaters and 0.78 for standard efficiency gas fired tankless water heaters up to 0.95 for the highest efficiency gas fired condensing tankless water heaters. These numbers represent the range of new units however, not the range of existing units in stock. Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 78%.

⁶²⁷ Water heating in multi-family buildings is often provided by a larger central boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of 0.59 and the 0.75 for single family homes. An average efficiency of 0.67 is used for this analysis as a default for multi-family buildings.

100,000 = Converts Btus to Therms (btu/Therm)

Other variables as defined above.

For example, a direct installed 1.5 GPM low flow showerhead in a gas fired DHW single family home where the number of showers is not known:

$$\Delta \text{Therms} = 1.0 * ((2.67 * 7.8 - 1.5 * 7.8) * 2.56 * 0.6 * 365.25 / 1.79) * 0.00501 * 0.98$$

$$= 14.0 \text{ therms}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

$$\Delta \text{gallons} = ((\text{GPM_base} * \text{L_base} - \text{GPM_low} * \text{L_low}) * \text{Household} * \text{SPCD} * 365.25 / \text{SPH}) * \text{ISR}$$

Variables as defined above

For example, a direct installed 1.5 GPM low flow showerhead in a single family home where the number of showers is not known:

$$\Delta \text{gallons} = ((2.67 * 7.8 - 1.5 * 7.8) * 2.56 * 0.6 * 365.25 / 1.79) * 0.98$$

$$= 2803 \text{ gallons}$$

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

SOURCES

Source ID	Reference
1	2011, DeOreo, William. California Single Family Water Use Efficiency Study. April 20, 2011.
2	2000, Mayer, Peter, William DeOreo, and David Lewis. Seattle Home Water Conservation Study. December 2000.
3	1999, Mayer, Peter, William DeOreo. Residential End Uses of Water. Published by AWWA Research Foundation and American Water Works Association. 1999.
4	2003, Mayer, Peter, William DeOreo. Residential Indoor Water Conservation Study. Aquacraft, Inc. Water Engineering and Management. Prepared for East Bay Municipal Utility District and the US EPA. July 2003.
5	2011, DeOreo, William. Analysis of Water Use in New Single Family Homes. By Aquacraft. For Salt Lake City Corporation and US EPA. July 20, 2011.
6	2011, Aquacraft. Albuquerque Single Family Water Use Efficiency and Retrofit Study. For Albuquerque Bernalillo County Water Utility Authority. December 1, 2011.
7	2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings.

MEASURE CODE: RS-HWE-LFSH-V05-180101

REVIEW DEADLINE: 1/1/2023

5.4.6 Water Heater Temperature Setback

DESCRIPTION

This measure was developed to be applicable to the following program types: NC, RF, DI, KITS.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

High efficiency is a hot water tank with the thermostat reduced to no lower than 120 degrees.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is a hot water tank with a thermostat setting that is higher than 120 degrees, typically systems with settings of 130 degrees or higher. Note if there are more than one DHW tanks in the home at or higher than 130 degrees and they are all turned down, then the savings per tank can be multiplied by the number of tanks.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of the measure is 2 years.

DEEMED MEASURE COST

The incremental cost of a setback is assumed to be \$5 for contractor time, or no cost if the measure is self-installed.

LOADSHAPE

Loadshape R03 - Residential Electric DHW

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 1.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

For homes with electric DHW tanks:

$$\Delta \text{kWh}^{628} = (U * A * (T_{\text{pre}} - T_{\text{post}}) * \text{Hours} * \text{ISR}) / (3412 * \text{RE}_{\text{electric}})$$

Where:

U	= Overall heat transfer coefficient of tank (Btu/Hr-°F-ft ²). = Actual if known. If unknown assume R-12, U = 0.083
A	= Surface area of storage tank (square feet)

⁶²⁸ Note this algorithm provides savings only from reduction in standby losses. The TAC considered avoided energy from not heating the water to the higher temperature but determined that dishwashers are likely to boost the temperature within the unit (roughly canceling out any savings), faucet and shower use is likely to be at the same temperature so there would need to be more lower temperature hot water being used (cancelling any savings) and clothes washers will only see savings if the water from the tank is taken without any temperature control. It was felt the potential impact was too small to be characterized.

= Actual if known. If unknown use table below based on capacity of tank. If capacity unknown assume 50 gal tank; A = 24.99ft²

Capacity (gal)	A (ft ²) ⁶²⁹
30	19.16
40	23.18
50	24.99
80	31.84

Tpre = Actual hot water setpoint prior to adjustment

Tpost = Actual new hot water setpoint, which may not be lower than 120 degrees

Default Hot Water Temperature Inputs	
Tpre	135
Tpost	120

Hours = Number of hours in a year (since savings are assumed to be constant over year).
= 8766

ISR = In service rate of measure

= Dependant on program delivery method as listed in table below

Delivery method	ISR
Instructions provided in a Kit	To be determined through evaluation
All other	1.0

3412 = Conversion from Btu to kWh

RE_electric = Recovery efficiency of electric hot water heater
= 0.98⁶³⁰

A deemed savings assumption, where site specific assumptions are not available would be as follows:

$$\begin{aligned}\Delta\text{kWh} &= (U * A * (T_{\text{pre}} - T_{\text{post}}) * \text{Hours} * \text{ISR}) / (3412 * \text{RE}_{\text{electric}}) \\ &= (((0.083 * 24.99) * (135 - 120) * 8766 * 1.0) / (3412 * 0.98)) \\ &= 81.6 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta\text{kW} = \Delta\text{kWh} / \text{Hours} * \text{CF}$$

Where:

Hours = 8766

CF = Summer Peak Coincidence Factor for measure
= 1

⁶²⁹ Assumptions from PA TRM. Area values were calculated from average dimensions of several commercially available units, with radius values measured to the center of the insulation.

⁶³⁰ Electric water heaters have recovery efficiency of 98%: <http://www.ahridirectory.org/ahridirectory/pages/home.aspx>

A deemed savings assumption, where site specific assumptions are not available would be as follows:

$$\begin{aligned}\Delta kW &= (81.6 / 8766) * 1 \\ \Delta kW \text{ default} &= 0.00931 \text{ kW}\end{aligned}$$

NATURAL GAS SAVINGS

For homes with gas water heaters:

$$\Delta \text{Therms} = (U * A * (T_{pre} - T_{post}) * \text{Hours} * \text{ISR}) / (100,000 * RE_{\text{gas}})$$

Where

$$100,000 = \text{Converts Btus to Therms (btu/Therm)}$$

$$\begin{aligned}RE_{\text{gas}} &= \text{Recovery efficiency of gas water heater} \\ &= 78\% \text{ For SF homes}^{631} \\ &= 67\% \text{ For MF homes}^{632}\end{aligned}$$

A deemed savings assumption, where site specific assumptions are not available would be as follows:

For Single Family homes:

$$\begin{aligned}\Delta \text{Therms} &= (U * A * (T_{pre} - T_{post}) * \text{Hours} * \text{ISR}) / (RE_{\text{gas}}) \\ &= (((0.083 * 24.99) * (135 - 120) * 8766 * 1.0) / (100,000 * 0.78)) \\ &= 3.5 \text{ Therms}\end{aligned}$$

For Multi Family homes:

$$\begin{aligned}\Delta \text{Therms} &= (U * A * (T_{pre} - T_{post}) * \text{Hours} * \text{ISR}) / (RE_{\text{gas}}) \\ &= (((0.083 * 24.99) * (135 - 120) * 8766 * 1.0) / (100,000 * 0.67)) \\ &= 4.1 \text{ Therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-HWE-TMPS-V05-160601

REVIEW DEADLINE: 1/1/2022

⁶³¹ DOE Final Rule discusses Recovery Efficiency with an average around 0.76 for Gas Fired Storage Water heaters and 0.78 for standard efficiency gas fired tankless water heaters up to 0.95 for the highest efficiency gas fired condensing tankless water heaters. These numbers represent the range of new units however, not the range of existing units in stock. Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 78%.

⁶³² Water heating in multi-family buildings is often provided by a larger central boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of 0.59 and the 0.75 for single family homes. An average efficiency of 0.67 is used for this analysis as a default for multi-family buildings.

5.4.7 Water Heater Wrap

DESCRIPTION

This measure relates to a Tank Wrap or insulation “blanket” that is wrapped around the outside of a hot water tank to reduce stand-by losses. This measure applies only for homes that have an electric water heater that is not already well insulated. Generally this can be determined based upon the appearance of the tank.⁶³³

This measure was developed to be applicable to the following program types: RF, DI.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The measure is a properly installed, R-8 or greater insulating tank wrap to reduce standby energy losses from the tank to the surrounding ambient area.

DEFINITION OF BASELINE EQUIPMENT

The baseline is a standard electric domestic hot water tank without an additional tank wrap. Gas storage water heaters are excluded due to the limitations of retrofit wrapping and the associated impacts on reduced savings and safety.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 5 years⁶³⁴.

DEEMED MEASURE COST

The incremental cost for this measure will be the actual material cost of procuring and labor cost of installing the tank wrap.

LOADSHAPE

Loadshape R03 - Residential Electric DHW

COINCIDENCE FACTOR

This measure assumes a flat loadshape and as such the coincidence factor is 1.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

For electric DHW systems:

$$\Delta \text{kWh} = ((A_{\text{base}} / R_{\text{base}} - A_{\text{insul}} / R_{\text{insul}}) * \Delta T * \text{Hours}) / (3412 * \eta_{\text{DHW}})$$

Where:

⁶³³ Visually determine whether it is insulated by foam (newer, rigid, and more effective) or fiberglass (older, gives to gently pressure, and not as effective)

⁶³⁴ This estimate assumes the tank wrap is installed on an existing unit with 5 years remaining life.

R_{base}	= Overall thermal resistance coefficient prior to adding tank wrap (Hr-°F-ft ² /BTU).
R_{insul}	= Overall thermal resistance coefficient after addition of tank wrap (Hr-°F-ft ² /BTU).
A_{base}	= Surface area of storage tank prior to adding tank wrap (square feet) ⁶³⁵
A_{insul}	= Surface area of storage tank after addition of tank wrap (square feet) ⁶³⁶
ΔT	= Average temperature difference between tank water and outside air temperature (°F) = 60°F ⁶³⁷
Hours	= Number of hours in a year (since savings are assumed to be constant over year). = 8766
3412	= Conversion from Btu to kWh
η_{DHW}	= Recovery efficiency of electric hot water heater = 0.98 ⁶³⁸

The following table has default savings for various tank capacity and pre and post R-VALUES.

Capacity (gal)	R_{base}	R_{insul}	A_{base} (ft ²) ⁶³⁹	A_{insul} (ft ²) ⁶⁴⁰	ΔkWh	ΔkW
30	8	16	19.16	20.94	171	0.0195
30	10	18	19.16	20.94	118	0.0135
30	12	20	19.16	20.94	86	0.0099
30	8	18	19.16	20.94	194	0.0221
30	10	20	19.16	20.94	137	0.0156
30	12	22	19.16	20.94	101	0.0116
40	8	16	23.18	25.31	207	0.0236
40	10	18	23.18	25.31	143	0.0164
40	12	20	23.18	25.31	105	0.0120
40	8	18	23.18	25.31	234	0.0268
40	10	20	23.18	25.31	165	0.0189
40	12	22	23.18	25.31	123	0.0140
50	8	16	24.99	27.06	225	0.0257
50	10	18	24.99	27.06	157	0.0179
50	12	20	24.99	27.06	115	0.0131
50	8	18	24.99	27.06	255	0.0291
50	10	20	24.99	27.06	180	0.0206
50	12	22	24.99	27.06	134	0.0153
80	8	16	31.84	34.14	290	0.0331
80	10	18	31.84	34.14	202	0.0231
80	12	20	31.84	34.14	149	0.0170
80	8	18	31.84	34.14	328	0.0374
80	10	20	31.84	34.14	232	0.0265
80	12	22	31.84	34.14	173	0.0198

⁶³⁵ Area includes tank sides and top to account for typical wrap coverage.

⁶³⁶ Ibid.

⁶³⁷ Assumes 125°F water leaving the hot water tank and average temperature of basement of 65°F.

⁶³⁸ Electric water heaters have recovery efficiency of 98%: <http://www.ahridirectory.org/ahridirectory/pages/home.aspx>

⁶³⁹ Assumptions from PA TRM. Area values were calculated from average dimensions of several commercially available units, with radius values measured to the center of the insulation. Area includes tank sides and top to account for typical wrap coverage.

⁶⁴⁰ Assumptions from PA TRM. A_{insul} was calculated by assuming that the water heater wrap is a 2" thick fiberglass material.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh / 8766 * CF$$

Where:

ΔkWh = kWh savings from tank wrap installation

8766 = Number of hours in a year (since savings are assumed to be constant over year).

CF = Summer Coincidence Factor for this measure
= 1.0

The table above has default kW savings for various tank capacity and pre and post R-values.

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-HWE-WRAP-V02-150601

REVIEW DEADLINE: 1/1/2022

5.4.8 Thermostatic Restrictor Shower Valve

DESCRIPTION

The measure is the installation of a thermostatic restrictor shower valve in a single or multi-family household. This is a valve attached to a residential showerhead which restricts hot water flow through the showerhead once the water reaches a set point (generally 95F or lower).

This measure was developed to be applicable to the following program types: RF, NC, DI. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a thermostatic restrictor shower valve installed on a residential showerhead.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is the residential showerhead without the restrictor valve installed.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years.⁶⁴¹

DEEMED MEASURE COST

The incremental cost of the measure should be the actual program cost (including labor if applicable) or \$30⁶⁴² plus \$20 labor⁶⁴³ if not available.

LOADSHAPE

Loadshape R03 - Residential Electric DHW

COINCIDENCE FACTOR

The coincidence factor for this measure is assumed to be 0.22%.⁶⁴⁴

⁶⁴¹ Assumptions based on NY TRM, Pacific Gas and Electric Company Work Paper PGECODHW113, and measure life of low-flow showerhead

⁶⁴² Based on actual cost of the SS-1002CP-SB Ladybug Water-Saving Shower-Head adapter from Evolve showerheads.

⁶⁴³ Estimate for contractor installation time.

⁶⁴⁴ Calculated as follows: Assume 11% showers take place during peak hours (based on: Oreo et al, "The end uses of hot water in single family homes from flow trace analysis", 2001.). There are 65 days in the summer peak period, so the percentage of total annual use in peak period is $0.11 * 65 / 365 = 1.96\%$. The number of hours of recovery during peak periods is therefore assumed to be $1.96\% * 29.5 = 0.577$ hours of recovery during peak period, where 29.5 equals the average annual electric DHW recovery hours for showerhead use prevented by the device including SF and MF homes with Direct Install and Retrofit/TOS measures. There are 260 hours in the peak period so the probability you will see savings during the peak period is $0.577 / 260 = 0.0022$

Algorithm

CALCULATION OF ENERGY SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = \% \text{ElectricDHW} * ((\text{GPM_base_S} * \text{L_showerdevice}) * \text{Household} * \text{SPCD} * 365.25 / \text{SPH}) * \text{EPG_electric} * \text{ISR}$$

Where:

%ElectricDHW = proportion of water heating supplied by electric resistance heating

DHW fuel	%ElectricDHW
Electric	100%
Natural Gas	0%
Unknown	16% ⁶⁴⁵

GPM_base_S = Flow rate of the basecase showerhead, or actual if available

Program	GPM
Direct-install, device only	2.67 ⁶⁴⁶
New Construction or direct install of device and low flow showerhead	Rated or actual flow of program-installed showerhead
Retrofit or TOS	2.35 ⁶⁴⁷

L_showerdevice = Hot water waste time avoided due to thermostatic restrictor valve
= 0.89 minutes⁶⁴⁸

Household = Average number of people per household

Household Unit Type ⁶⁴⁹	Household
Single-Family - Deemed	2.56 ⁶⁵⁰
Multi-Family - Deemed	2.1 ⁶⁵¹
Custom	Actual Occupancy or Number of Bedrooms ⁶⁵²

⁶⁴⁵ Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

⁶⁴⁶ Based on measured data from Ameren IL EM&V of Direct-Install program. Program targets showers that are rated 2.5 GPM or above. Assumes low flow showerhead not included in direct installation.

⁶⁴⁷ Representative value from sources 1, 2, 4, 5, 6 and 7 (See Source Table at end of measure section) adjusted slightly upward to account for program participation which is expected to target customers with existing higher flow devices rather than those with existing low flow devices.

⁶⁴⁸ Average of the following sources: ShowerStart LLC survey; "Identifying, Quantifying and Reducing Behavioral Waste in the Shower: Exploring the Savings Potential of ShowerStart", City of San Diego Water Department survey; "Water Conservation Program: ShowerStart Pilot Project White Paper", and PG&E Work Paper PGECODHW113.

⁶⁴⁹ If household type is unknown, as may be the case for time of sale measures, then single family deemed value shall be used.

⁶⁵⁰ ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program citing 2006-2008 American Community Survey data from the US Census Bureau for Illinois cited on p. 17 of the PY2 Evaluation report. 2.75 * 93% evaluation adjustment

⁶⁵¹ ComEd PY3 Multi-Family Evaluation Report REVISED DRAFT v5 2011-12-08.docx

⁶⁵² Bedrooms are suitable proxies for household occupancy, and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.

SPCD = Showers Per Capita Per Day

$$= 0.6^{653}$$

365.25 = Days per year, on average.

SPH = Showerheads Per Household so that per-showerhead savings fractions can be determined

Household Type	SPH
Single-Family	1.79 ⁶⁵⁴
Multi-Family	1.3 ⁶⁵⁵
Custom	Actual

EPG_{electric} = Energy per gallon of hot water supplied by electric

$$= (8.33 * 1.0 * (\text{ShowerTemp} - \text{SupplyTemp})) / (\text{RE}_{\text{electric}} * 3412)$$

$$= (8.33 * 1.0 * (101 - 54.1)) / (0.98 * 3412)$$

$$= 0.117 \text{ kWh/gal}$$

8.33 = Specific weight of water (lbs/gallon)

1.0 = Heat Capacity of water (btu/lb-°)

ShowerTemp = Assumed temperature of water

$$= 101\text{F}^{656}$$

SupplyTemp = Assumed temperature of water entering house

$$= 54.1\text{F}^{657}$$

RE_{electric} = Recovery efficiency of electric water heater

$$= 98\%^{658}$$

3412 = Converts Btu to kWh (btu/kWh)

ISR = In service rate of showerhead

= Dependent on program delivery method as listed in table below

Selection	ISR
Direct Install - Single Family	0.98 ⁶⁵⁹
Direct Install – Multi Family	0.95 ⁶⁶⁰
Efficiency Kits	To be determined through evaluation

⁶⁵³ Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group.

⁶⁵⁴ Based on findings from a 2009 ComEd residential survey of 140 sites, provided by Cadmus.

⁶⁵⁵ Ibid.

⁶⁵⁶ Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group.

⁶⁵⁷ US DOE Building America Program. Building America Analysis Spreadsheet. For Chicago, IL http://www1.eere.energy.gov/buildings/building_america/analysis_spreadsheets.html.

⁶⁵⁸ Electric water heaters have recovery efficiency of 98%: <http://www.ahridirectory.org/ahridirectory/pages/home.aspx>

⁶⁵⁹ Deemed values are from ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program Table 3-8. Alternative ISRs may be developed for program delivery methods based on evaluation results.

⁶⁶⁰ Navigant, ComEd-Nicor Gas EPY4/GPY1 Multi-Family Home Energy Savings Program Evaluation Report FINAL 2013-06-05

EXAMPLE

For example, a direct installed valve in a single-family home with electric DHW:

$$\begin{aligned}\Delta\text{kWh} &= 1.0 * (2.67 * 0.89 * 2.56 * 0.6 * 365.25 / 1.79) * 0.117 * 0.98 \\ &= 85 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta\text{kW} = \Delta\text{kWh}/\text{Hours} * \text{CF}$$

Where:

ΔkWh = calculated value above

Hours = Annual electric DHW recovery hours for wasted showerhead use prevented by device

$$= ((\text{GPM_base_S} * \text{L_showerdevice}) * \text{Household} * \text{SPCD} * 365.25) * 0.712^{661} / \text{GPH}$$

GPH = Gallons per hour recovery of electric water heater calculated for 65.9F temp rise (120-54.1), 98% recovery efficiency, and typical 4.5kW electric resistance storage tank.

$$= 27.51$$

$$= 34.4 \text{ for SF Direct Install; } 28.3 \text{ for MF Direct Install}$$

$$= 30.3 \text{ for SF Retrofit and TOS; } 24.8 \text{ for MF Retrofit and TOS}$$

CF = Coincidence Factor for electric load reduction

$$= 0.0022^{662}$$

EXAMPLE

For example, a direct installed thermostatic restrictor device in a single family home with electric DHW where the number of showers is not known.

$$\begin{aligned}\Delta\text{kW} &= 85.3/34.4 * 0.0022 \\ &= 0.0055 \text{ kW}\end{aligned}$$

NATURAL GAS SAVINGS

$$\Delta\text{Therms} = \% \text{FossilDHW} * ((\text{GPM_base_S} * \text{L_showerdevice}) * \text{Household} * \text{SPCD} * 365.25 / \text{SPH}) * \text{EPG_gas} * \text{ISR}$$

Where:

$\% \text{FossilDHW}$ = proportion of water heating supplied by Natural Gas heating

⁶⁶¹ 71.2% is the proportion of hot 120F water mixed with 54.1F supply water to give 101F shower water.

⁶⁶² Calculated as follows: Assume 11% showers take place during peak hours (based on: Oreo et al, "The end uses of hot water in single family homes from flow trace analysis", 2001.). There are 65 days in the summer peak period, so the percentage of total annual use in peak period is $0.11 * 65 / 365 = 1.96\%$. The number of hours of recovery during peak periods is therefore assumed to be $1.96\% * 29.5 = 0.577$ hours of recovery during peak period, where 29.5 equals the average annual electric DHW recovery hours for showerhead use prevented by the device including SF and MF homes with Direct Install and Retrofit/TOS measures. There are 260 hours in the peak period so the probability you will see savings during the peak period is $0.577 / 260 = 0.0022$

DHW fuel	%Fossil_DHW
Electric	0%
Natural Gas	100%
Unknown	84% ⁶⁶³

EPG_gas = Energy per gallon of Hot water supplied by gas

$$= (8.33 * 1.0 * (\text{ShowerTemp} - \text{SupplyTemp})) / (\text{RE_gas} * 100,000)$$

$$= 0.00501 \text{ Therm/gal for SF homes}$$

$$= 0.00583 \text{ Therm/gal for MF homes}$$

RE_gas = Recovery efficiency of gas water heater

$$= 78\% \text{ For SF homes}^{664}$$

$$= 67\% \text{ For MF homes}^{665}$$

100,000 = Converts Btus to Therms (btu/Therm)
 Other variables as defined above.

EXAMPLE

For example, a direct installed thermostatic restrictor device in a gas fired DHW single family home where the number of showers is not known:

$$\Delta\text{Therms} = 1.0 * ((2.67 * 0.89) * 2.56 * 0.6 * 365.25 / 1.79) * 0.00501 * 0.98$$

$$= 3.7 \text{ therms}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

$$\Delta\text{gallons} = ((\text{GPM_base_S} * \text{L_showerdevice}) * \text{Household} * \text{SPCD} * 365.25 / \text{SPH}) * \text{ISR}$$

Variables as defined above

⁶⁶³ Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

⁶⁶⁴ DOE Final Rule discusses Recovery Efficiency with an average around 0.76 for Gas Fired Storage Water heaters and 0.78 for standard efficiency gas fired tankless water heaters up to 0.95 for the highest efficiency gas fired condensing tankless water heaters. These numbers represent the range of new units however, not the range of existing units in stock. Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 78%.

⁶⁶⁵ Water heating in multi-family buildings is often provided by a larger central boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of 0.59 and the 0.75 for single family homes. An average efficiency of 0.67 is used for this analysis as a default for multi-family buildings.

EXAMPLE

For example, a direct installed thermostatic restrictor device in a single family home where the number of showers is not known:

$$\begin{aligned}\Delta \text{gallons} &= ((2.67 * 0.89) * 2.56 * 0.6 * 365.25 / 1.79) * 0.98 \\ &= 730 \text{ gallons}\end{aligned}$$

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

SOURCES

Source ID	Reference
1	2011, DeOreo, William. California Single Family Water Use Efficiency Study. April 20, 2011.
2	2000, Mayer, Peter, William DeOreo, and David Lewis. Seattle Home Water Conservation Study. December 2000.
3	1999, Mayer, Peter, William DeOreo. Residential End Uses of Water. Published by AWWA Research Foundation and American Water Works Association. 1999.
4	2003, Mayer, Peter, William DeOreo. Residential Indoor Water Conservation Study. Aquacraft, Inc. Water Engineering and Management. Prepared for East Bay Municipal Utility District and the US EPA. July 2003.
5	2011, DeOreo, William. Analysis of Water Use in New Single Family Homes. By Aquacraft. For Salt Lake City Corporation and US EPA. July 20, 2011.
6	2011, Aquacraft. Albuquerque Single Family Water Use Efficiency and Retrofit Study. For Albuquerque Bernalillo County Water Utility Authority. December 1, 2011.
7	2008, Schultdt, Marc, and Debra Tachibana. Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes. 2008 ACEEE Summer Study on Energy Efficiency in Buildings.
8	2011, Lutz, Jim. "Water and Energy Wasted During Residential Shower Events: Findings from a Pilot Field Study of Hot Water Distribution Systems", Energy Analysis Department Lawrence Berkeley National Laboratory, September 2011.
9	2008, Water Conservation Program: ShowerStart Pilot Project White Paper, City of San Diego, CA.
10	2012, Pacific Gas and Electric Company, Work Paper PGECODHW113, Low Flow Showerhead and Thermostatic Shower Restriction Valve, Revision # 4, August 2012.
11	2008, "Simply & Cost Effectively Reducing Shower Based Warm-Up Waste: Increasing Convenience & Conservation by Attaching ShowerStart to Existing Showerheads", ShowerStart LLC.
12	2014, New York State Record of Revision to the TRM, Case 07-M-0548, June 19, 2014.

MEASURE CODE: RS-HWE-TRVA-V03-180101

REVIEW DEADLINE: 1/1/2023

5.4.9 Shower Timer

DESCRIPTION

Shower Timers are designed to make it easy for people to consistently take short showers, resulting in water and energy savings.

The shower timer provides a reminder to participants on length of their shower visually or auditorily.

This measure was developed to be applicable to the following program type: KITS, DI.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The shower timer should provide a reminder to participants to keep showers to a length of 5 minutes or less.

DEFINITION OF BASELINE EQUIPMENT

The baseline is no shower timer.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The deemed lifetime is 2 years⁶⁶⁶.

DEEMED MEASURE COST

For shower timers provided in Efficiency Kits, the actual program delivery costs should be utilized.

LOADSHAPE

Loadshape R03 - Residential Electric DHW

COINCIDENCE FACTOR

The coincidence factor for this measure is assumed to be 2.78%.⁶⁶⁷

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = \% \text{Electric DHW} * \text{GPM} * (\text{L}_{\text{base}} - \text{L}_{\text{timer}}) * \text{Household} * \text{Days/yr} * \text{SPCD} * \text{UsageFactor} * \text{EPG}_{\text{Electric}}$$

Where:

⁶⁶⁶ Estimate of persistence of behavior change instigated by the shower timer.

⁶⁶⁷ Calculated as follows: Assume 11% showers take place during peak hours (based on: Oreo et al, "The end uses of hot water in single family homes from flow trace analysis", 2001.). There are 65 days in the summer peak period, so the percentage of total annual aerator use in peak period is $0.11 * 65 / 365 = 1.96\%$. The number of hours of recovery during peak periods is therefore assumed to be $1.96\% * 369 = 7.23$ hours of recovery during peak period, where 369 equals the average annual electric DHW recovery hours for showerhead use including SF and MF homes with Direct Install and Retrofit/TOS measures. There are 260 hours in the peak period so the probability you will see savings during the peak period is $7.23 / 260 = 0.0278$

%Electric DHW = Proportion of water heating supplied by electric resistance heating

DHW fuel	%ElectricDHW
Electric	100%
Natural Gas	0%
Unknown	16% ⁶⁶⁸

GPM = Flow rate of showerhead as used

= Custom, to be determined through evaluation. If data is not available use 1.93⁶⁶⁹

L_base = Number of minutes in shower without a shower timer

=7.8 minutes⁶⁷⁰

L_timer = Number of minutes in shower after shower timer

= Custom, to be determined through evaluation. If data is not available use 5.79⁶⁷¹

Household = Number in household using timer

Household Unit Type ⁶⁷²	Household
Single-Family - Deemed	2.56 ⁶⁷³
Multi-Family - Deemed	2.1 ⁶⁷⁴
Custom	Actual Occupancy or Number of Bedrooms ⁶⁷⁵

Days/yr = 365.25

SPCD = Showers Per Capita Per Day

= 0.6⁶⁷⁶

UsageFactor = How often each participant is using shower timer

=Custom, to be determined through evaluation. If data is not available use 0.34⁶⁷⁷

EPG_Electric = Energy per gallon of hot water supplied by electric

⁶⁶⁸ Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

⁶⁶⁹ Navigant Elementary Education GPY4 Evaluation Report, dated May 12, 2016. Average of all utilities.

⁶⁷⁰ Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group. This study of 135 single and multi-family homes in Michigan metered energy parameters for efficient showerhead and faucet aerators.

⁶⁷¹ Navigant Elementary Education GPY4 Evaluation Report, dated May 12, 2016. Average of all utilities.

⁶⁷² If household type is unknown, as may be the case for time of sale measures, then single family deemed value shall be used.

⁶⁷³ ComEd Energy Efficiency/ Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: All Electric Single Family Home Energy Performance Tune-Up Program citing 2006-2008 American Community Survey data from the US Census Bureau for Illinois cited on p. 17 of the PY2 Evaluation report. 2.75 * 93% evaluation adjustment

⁶⁷⁴ ComEd PY3 Multi-Family Evaluation Report REVISED DRAFT v5 2011-12-08.docx

⁶⁷⁵ Bedrooms are suitable proxies for household occupancy, and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.

⁶⁷⁶ Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group.

⁶⁷⁷ Navigant Elementary Education GPY4 Evaluation Report, dated May 12, 2016. Average of all utilities.

$$\begin{aligned}
 &= (8.33 * 1.0 * (\text{ShowerTemp} - \text{SupplyTemp})) / (\text{RE_electric} * 3412) \\
 &= (8.33 * 1.0 * (101 - 54.1)) / (0.98 * 3412) \\
 &= 0.117 \text{ kWh/gal}
 \end{aligned}$$

Based on default assumptions provided above, the savings for a single family home would be:

$$\begin{aligned}
 \Delta \text{kWh} &= \% \text{Electric DHW} * \text{GPM} * (\text{L_base} - \text{L_timer}) * \text{Household} * \text{Days/yr} * \text{SPCD} * \text{UsageFactor} \\
 &\quad * \text{EPG_Electric} \\
 &= 0.16 * 1.93 * (7.8 - 5.79) * 2.56 * 365.25 * 0.6 * 0.34 * 0.117 \\
 &= 13.9 \text{ kWh}
 \end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = \Delta \text{kWh/Hours} * \text{CF}$$

Where:

ΔkWh = calculated value above

Hours = Annual electric DHW recovery hours for showerhead use

$$= ((\text{GPM_base} * \text{L_base}) * \text{Household Users} * \text{SPCD} * 365.25) * 0.712 / \text{GPH}$$

GPH = Gallons per hour recovery of electric water heater calculated for 65.9F temp rise (120-54.1), 98% recovery efficiency, and typical 4.5kW electric resistance storage tank.
= 27.51

CF = Coincidence Factor for electric load reduction
= 0.0278⁶⁷⁸

Based on default assumptions provided above, the savings for a single family home would be:

$$\begin{aligned}
 \Delta \text{kW} &= \Delta \text{kWh/Hours} * \text{CF} \\
 &= 0.0013 \text{ kW}
 \end{aligned}$$

NATURAL GAS SAVINGS

$$\Delta \text{Therms} = \% \text{FossilDHW} * \text{GPM} * (\text{L_base} - \text{L_timer}) * \text{Household} * \text{Days/yr} * \text{SPCD} * \text{UsageFactor} * \text{EPG_Gas}$$

$\% \text{FossilDHW}$ = Proportion of water heating supplied by electric resistance heating

DHW fuel	$\% \text{FossilDHW}$
Electric	0%
Natural Gas	100%
Unknown	84% ⁶⁷⁹

⁶⁷⁸ Calculated as follows: Assume 11% showers take place during peak hours (based on: Oreo et al, "The end uses of hot water in single family homes from flow trace analysis", 2001.). There are 65 days in the summer peak period, so the percentage of total annual aerator use in peak period is $0.11 * 65 / 365 = 1.96\%$. The number of hours of recovery during peak periods is therefore assumed to be $1.96\% * 369 = 7.23$ hours of recovery during peak period where 369 equals the average annual electric DHW recovery hours for showerhead use including SF and MF homes with Direct Install and Retrofit/TOS measures. There are 260 hours in the peak period so the probability you will see savings during the peak period is $7.23 / 260 = 0.0278$

⁶⁷⁹ Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used

EPG_gas = Energy per gallon of Hot water supplied by gas
 = $(8.33 * 1.0 * (\text{ShowerTemp} - \text{SupplyTemp})) / (\text{RE_gas} * 100,000)$
 = 0.00501 Therm/gal for SF homes
 = 0.00583 Therm/gal for MF homes

RE_gas = Recovery efficiency of gas water heater
 = 78% For SF homes⁶⁸⁰
 = 67% For MF homes⁶⁸¹

100,000 = Converts Btus to Therms (btu/Therm)

Other variables as defined above.

Based on default assumptions provided above, the savings for a single family home would be:

$\Delta \text{ Therms} = \% \text{FossilDHW} * \text{GPM} * (\text{L_base} - \text{L_timer}) * \text{Household} * \text{Days/yr} * \text{SPCD} * \text{UsageFactor} * \text{EPG_Gas}$
 = $0.84 * 1.93 * (7.8 - 5.79) * 2.56 * 365.25 * 0.6 * 0.34 * 0.00501$
 = 3.1 Therms

WATER DESCRIPTIONS AND CALCULATION

$\Delta \text{Gallons} = \text{GPM} * (\text{L_base} - \text{L_timer}) * \text{Household} * \text{Days/yr} * \text{SPCD} * \text{UsageFactor}$

Variables as defined above

Based on default assumptions provided above, the savings for a single family home would be:

$\Delta \text{Gallons} = \text{GPM} * (\text{L_base} - \text{L_timer}) * \text{Household} * \text{Days/yr} * \text{SPCD} * \text{UsageFactor}$
 = $1.93 * (7.8 - 5.79) * 2.56 * 365.25 * 0.6 * 0.34$
 = 740.0 gallons

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-DHW-SHTM-V01-180101

REVIEW DEADLINE: 1/1/2019

⁶⁸⁰ DOE Final Rule discusses Recovery Efficiency with an average around 0.76 for Gas Fired Storage Water heaters and 0.78 for standard efficiency gas fired tankless water heaters up to 0.95 for the highest efficiency gas fired condensing tankless water heaters. These numbers represent the range of new units however, not the range of existing units in stock. Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 78%.

⁶⁸¹ Water heating in multi-family buildings is often provided by a larger central boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of 0.59 and the 0.75 for single family homes. An average efficiency of 0.67 is used for this analysis as a default for multi-family buildings.

5.5 Lighting End Use

5.5.1 Compact Fluorescent Lamp (CFL)

DESCRIPTION

A low wattage qualified compact fluorescent screw-in bulb (CFL) is installed in place of a baseline screw-in bulb.

Note a new ENERGY STAR specification v2.0 becomes effective on 1/2/2017

(https://www.energystar.gov/products/spec/lamps_specification_version_2_0_pd). The efficacy requirements cannot currently be met by Compact Fluorescent Lamps, and therefore this specification has been removed.

ENERGY STAR will maintain a list on their website with the final qualifying list of products prior to this change and it is strongly recommended that programs continue to use this list as qualifying criteria for products in the programs.

This characterization assumes that the CFL is installed in a residential location. If the implementation strategy does not allow for the installation location to be known (e.g. an upstream retail program), a deemed split of 95% Residential and 5% Commercial assumptions should be used⁶⁸².

Federal legislation stemming from the Energy Independence and Security Act of 2007 (EISA) required all general-purpose light bulbs between 40W and 100W to be approximately 30% more energy efficient than current incandescent bulbs. Production of 100W, standard efficacy incandescent lamps ended in 2012, followed by restrictions on 75W in 2013 and 60W and 40W in 2014. The baseline for this measure has therefore become bulbs (improved incandescent or halogen) that meet the new standard.

A provision in the EISA regulations requires that by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, in essence making the baseline equivalent to a current day CFL. Therefore the measure life (number of years that savings should be claimed) should be reduced once the assumed lifetime of the bulb exceeds 2020. Due to expected delay in clearing retail inventory and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until 2021.

This measure was developed to be applicable to the following program types: TOS, NC, DI, KITS. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the high-efficiency equipment must be a standard qualified compact fluorescent lamp.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be an EISA qualified incandescent or halogen as provided in the table provided in the Electric Energy Savings section.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life (number of years that savings should be claimed) for bulbs installed in 2018 is assumed to be 3 years and then for every subsequent year should be reduced by one year⁶⁸³.

⁶⁸² RES v C&I split is based on a weighted (by sales volume) average of ComEd PY6, PY7 and PY8 and Ameren PY5, PY6 and PY8 in store intercept survey results. See 'RESvCI Split_112016.xls'.

⁶⁸³ Since the replacement baseline bulb from 2020 on will be equivalent to a CFL, no additional savings should be claimed from that point. Due to expected delay in clearing stock from retail outlets and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until 2021.

DEEMED MEASURE COST

For the Retail (Time of Sale) measure, the incremental capital cost is \$1.20⁶⁸⁴.

For the Direct Install measure, the full cost of \$2.45 per bulb should be used, plus \$5 labor cost⁶⁸⁵ for a total of \$7.45 per bulb. However actual program delivery costs should be utilized if available.

For bulbs provided in Efficiency Kits, the actual program delivery costs should be utilized.

LOADSHAPE

Loadshape R06 - Residential Indoor Lighting

Loadshape R07 - Residential Outdoor Lighting

COINCIDENCE FACTOR

The summer peak coincidence factor is assumed to be 7.1% for Time of Sale Residential and in-unit Multi Family bulbs, 27.3% for exterior bulbs and 8.1% for unknown⁶⁸⁶ and 7.4% for Residential Direct Install⁶⁸⁷.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta kWh = ((WattsBase - WattsEE) / 1000) * ISR * (1 - Leakage) * Hours * WHFe$$

Where:

WattsBase = Based on lumens of CFL bulb and program year installed:

Minimum Lumens	Maximum Lumens	Incandescent Equivalent Post- EISA 2007 (WattsBase)
5280	6209	300
3000	5279	200
2601	2999	150
1490	2600	72
1050	1489	53
750	1049	43
310	749	29
250	309	25

WattsEE = Actual wattage of CFL purchased / installed

⁶⁸⁴ Based upon field data collected by CLEAResult and provided by ComEd. See ComEd Pricing Projections 06302016.xlsx for analysis.

⁶⁸⁵ Based on 15 minutes at \$20 an hour. Includes some portion of travel time to site.

⁶⁸⁶ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

⁶⁸⁷ Based on lighting logger study conducted as part of the PY5/PY6 ComEd Residential Lighting Program evaluation and excluding all logged bulbs installed in closets.

ISR = In Service Rate, the percentage of units rebated that are actually in service.

Program		Weighted Average 1st Year In Service Rate (ISR)	2nd year Installations	3rd year Installations	Final Lifetime In Service Rate
Retail (Time of Sale)		76.5% ⁶⁸⁸	11.6%	9.9%	98.0% ⁶⁸⁹
Direct Install		96.9% ⁶⁹⁰			
Efficiency Kits ⁶⁹¹	CFL Distribution ⁶⁹²	59%	13%	11%	83%
	School Kits ⁶⁹³	61%	13%	11%	86%
	Direct Mail Kits ⁶⁹⁴	66%	14%	12%	93%

Leakage = Adjustment to account for the percentage of program bulbs that move out (and in if deemed appropriate⁶⁹⁵) of the Utility Jurisdiction.

KITS programs = Determined through evaluation

Upstream (TOS) Lighting programs = Determined through evaluation or use deemed assumptions below⁶⁹⁶:

ComEd: 2.1%

Ameren: 13.1%

All other programs = 0

⁶⁸⁸ 1st year in service rate is based upon review of PY6-8 evaluations from ComEd and PY5,6 and 8 for Ameren (see 'IL RES Lighting ISR_112016.xls' for more information). The average first year ISR for each utility was calculated weighted by the number of bulbs in the each year's survey. This was then weighted by annual sales to give a statewide assumption.

⁶⁸⁹ The 98% Lifetime ISR assumption is based upon review of two evaluations:

'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings.

⁶⁹⁰ Based upon review of the PY2 and PY3 ComEd Direct Install program surveys. This value includes bulb failures in the 1st year to be consistent with the Commission approval of annualization of savings for first year savings claims. ComEd PY2 All Electric Single Family Home Energy Performance Tune-Up Program Evaluation, Navigant Consulting, December 21, 2010.

⁶⁹¹ In Service Rates provided are for the CFL bulb within a kit only. Given the significant differences in program design and the level of education provided through Efficiency Kits programs, the evaluators should apply the ISR estimated through evaluations (either past evaluations or the current program year evaluation) of the specific Efficiency Kits provided. In cases where program-specific evaluation results for an ISR are unavailable, the default ISR values for Efficiency Kits provided may be used.

⁶⁹² Free bulbs provided without request, with little or no education. Based on 'Impact and Process Evaluation of 2013 (PY6) Ameren Illinois Company Residential CFL Distribution Program', Report Table 11 and Appendix B.

⁶⁹³ Kits provided free to students through school, with education program. Based on 'Impact and Process Evaluation of 2013 (PY6) Ameren Illinois Company Residential Efficiency Kits Program', table 10. Final ISR assumptions are based upon comparing with CFL Distribution First year ISR and multiplying by the CFL Distribution Final ISR value, and second and third year estimates based on same proportion of future installs.

⁶⁹⁴ Opt-in program to receive kits via mail, with little or no education. Based on 'Impact and Process Evaluation of 2013 (PY6) Ameren Illinois Company Residential Efficiency Kits Program', table 10, as above.

⁶⁹⁵ Leakage in is only appropriate to credit to IL utility program savings if it is reasonably expected that the IL utility program marketing efforts played an important role in influencing customer to purchase the light bulbs. Furthermore, consideration that such customers might be free riders should be addressed. If leakage in is assessed, efforts should be made to ensure no double counting of savings occurs if the evaluation is estimating both leakage in and spillover savings of light bulbs.

⁶⁹⁶ Leakage rate is based upon review of PY6-8 evaluations from ComEd and PY5,6 and 8 for Ameren (see 'IL Leakage Rates_112016.xls' for more information).

Hours = Average hours of use per year

Program Delivery	Installation Location	Hours ⁶⁹⁷
Retail (Time of Sale) and Efficiency Kits	Residential Interior and in-unit Multi Family	759
	Exterior	2,475 ⁶⁹⁸
	Unknown	847 ⁶⁹⁹
Direct Install	Residential Interior and in-unit Multi Family	793
	Exterior	2,475

WHFe = Waste heat factor for energy to account for cooling energy savings from efficient lighting

Bulb Location	WHFe
Interior single family or unknown location	1.06 ⁷⁰⁰
Multi family in unit	1.04 ⁷⁰¹
Exterior or uncooled location	1.0

DEFERRED INSTALLS

As presented above, the characterization assumes that a percentage of bulbs purchased are not installed until Year 2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

Year 1 (Purchase Year) installs: Characterized using assumptions provided above or evaluated assumptions if available.

Year 2 and 3 installs: Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.

The NTG factor for the Purchase Year should be applied.

⁶⁹⁷ Except where noted, based on lighting logger study conducted as part of the PY5/PY6 ComEd Residential Lighting Program evaluation. Direct Install value excludes all logged bulbs installed in closets.

⁶⁹⁸ Based on secondary research conducted as part of the PY5/PY6 ComEd Residential Lighting Program evaluation.

⁶⁹⁹ Assumes 5% exterior lighting, based on PYPY5/PY6 ComEd Residential Lighting Program evaluation.

⁷⁰⁰ The value is estimated at 1.06 (calculated as $1 + (0.66 * (0.27 / 2.8))$). Based on cooling loads decreasing by 27% of the lighting savings (average result from REMRate modeling of several different configurations and IL locations of homes), assuming typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm $(-0.02 * SEER2) + (1.12 * SEER)$ (from Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = $EER/3.412 = 2.8COP$) and 66% of homes in Illinois having central cooling ("Table HC7.9 Air Conditioning in Homes in Midwest Region, Divisions, and States, 2009 from Energy Information Administration", 2009 Residential Energy Consumption Survey)

⁷⁰¹ As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average)

For example, for a 14W CFL (60W standard incandescent and 43W EISA qualified incandescent/halogen):

$$\Delta kWh_{1st \text{ year installs}} = ((43 - 14) / 1000) * 0.765 * 847 * 1.06$$

$$= 19.9 \text{ kWh}$$

$$\Delta kWh_{2nd \text{ year installs}} = ((43 - 14) / 1000) * 0.116 * 847 * 1.06$$

$$= 3.0 \text{ kWh}$$

Note: Here we assume no change in hours assumption. NTG value from Purchase year applied.

$$\Delta kWh_{3rd \text{ year installs}} = ((43 - 14) / 1000) * 0.099 * 847 * 1.06$$

$$= 2.6 \text{ kWh}$$

HEATING PENALTY

If electric heated home (if heating fuel is unknown assume gas, see Natural Gas section):

$$\Delta kWh^{702} = - (((WattsBase - WattsEE) / 1000) * ISR * Hours * HF) / \eta_{Heat}$$

Where:

- HF = Heating Factor or percentage of light savings that must be heated
 = 49%⁷⁰³ for interior or unknown location
 = 0% for exterior or unheated location
- η_{Heat} = Efficiency in COP of Heating equipment
 = actual. If not available use⁷⁰⁴:

System Type	Age of Equipment	HSPF Estimate	COP _{HEAT} (COP Estimate) = (HSPF/3.413)*0.85
Heat Pump	Before 2006	6.8	1.7
	After 2006 - 2014	7.7	1.92
	2015 on	8.2	2.04
Resistance	N/A	N/A	1.00
Unknown ⁷⁰⁵	N/A	N/A	1.28

⁷⁰² Negative value because this is an increase in heating consumption due to the efficient lighting.

⁷⁰³ This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

⁷⁰⁴ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. Note efficiency should include duct losses. Defaults provided assume 15% duct loss for heat pumps.

⁷⁰⁵ Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see "HC6.9 Space Heating in Midwest Region.xls", using average for East North Central Region. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

For example, a 14W standard CFL is purchased and installed in home with 2.0 COP (including duct loss) Heat Pump:

$$\begin{aligned}\Delta\text{kWh}_{1\text{st year}} &= - ((43 - 14) / 1000) * 0.765 * 759 * 0.49 / 2.0 \\ &= - 4.2 \text{ kWh}\end{aligned}$$

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta\text{kW} = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{WHFd} * \text{CF}$$

Where:

WHFd = Waste heat factor for demand to account for cooling savings from efficient lighting.

Bulb Location	WHFd
Interior single family or unknown location	1.11 ⁷⁰⁶
Multi family in unit	1.07 ⁷⁰⁷
Exterior or uncooled location	1.0

CF = Summer Peak Coincidence Factor for measure.

Program Delivery	Bulb Location	CF ⁷⁰⁸
Retail(Time of Sale)	Interior single family or Multi Family in unit	7.1%
	Exterior	27.3%
	Unknown location	8.1%
Direct Install	Residential	7.4%

Other factors as defined above

For example, a 14W standard CFL is purchased and installed in a single family interior location:

$$\begin{aligned}\Delta\text{kW} &= ((43 - 14) / 1000) * 0.765 * 1.11 * 0.071 \\ &= 0.0017 \text{ kW}\end{aligned}$$

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year.

⁷⁰⁶ The value is estimated at 1.11 (calculated as $1 + (0.66 * 0.466 / 2.8)$). See footnote relating to WHFe for details. Note the 46.6% factor represents the average Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load.

⁷⁰⁷ As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average)

⁷⁰⁸ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation. Direct Install value is based on resut excluding all logged bulbs installed in closets.

NATURAL GAS SAVINGS

Heating Penalty if Natural Gas heated home (or if heating fuel is unknown):

$$\Delta \text{Therms}^{709} = - (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * \text{HF} * 0.03412) / \eta \text{Heat}$$

Where:

HF	= Heating Factor or percentage of light savings that must be heated
	= 49% ⁷¹⁰ for interior or unknown location
	= 0% for exterior or unheated location
0.03412	= Converts kWh to Therms
ηHeat	= Efficiency of heating system
	= 70% ⁷¹¹

For example, a 14 standard CFL is purchased and installed in a home:

$$\begin{aligned} \Delta \text{Therms} &= - (((43 - 14) / 1000) * 0.765 * 759 * 0.49 * 0.03412) / 0.7 \\ &= - 0.40 \text{ Therms} \end{aligned}$$

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

The O&M assumptions that should be used in cost effectiveness calculations are provided below:

⁷⁰⁹ Negative value because this is an increase in heating consumption due to the efficient lighting.

⁷¹⁰ This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

⁷¹¹ This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey) In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:

$$(0.24 * 0.92) + (0.76 * 0.8) * (1 - 0.15) = 0.70$$

Program Delivery	Installation Location	Replacement Period (years) ⁷¹²	Replacement Cost ⁷¹³
Retail (Time of Sale) and Efficiency Kits	Residential Interior and in-unit Multi Family	1.3	\$1.25
	Exterior	0.4	
	Unknown	1.2	
Direct Install	Residential Interior and in-unit Multi Family	1.3	
	Exterior	0.4	

It is important to note that for cost-effectiveness screening purposes, the O&M cost adjustments should only be applied in cases where the light bulbs are actually in service and so should be multiplied by the appropriate ISR.

MEASURE CODE: RS-LTG-ESCF-V06-180101

REVIEW DEADLINE: 1/1/2020

⁷¹² Calculated by dividing assumed rated life of baseline bulb by hours of use. Assumed lifetime of EISA qualified Halogen/Incandescents is 1000 hours. The manufacturers are simply using a regular incandescent lamp with halogen fill gas rather than Halogen Infrared to meet the standard (as provided by G. Arnold, NEEP and confirmed by N. Horowitz at NRDC).

⁷¹³ Based upon field data collected by CLEAResult and provided by ComEd. See ComEd Pricing Projections 06302016.xlsx for analysis.

5.5.2 ENERGY STAR Specialty Compact Fluorescent Lamp (CFL)

DESCRIPTION

A qualified specialty compact fluorescent bulb is installed in place of an incandescent specialty bulb.

Note a new ENERGY STAR specification v2.0 becomes effective on 1/2/2017

(https://www.energystar.gov/products/spec/lamps_specification_version_2_0_pd). The efficacy requirements cannot currently be met by Compact Fluorescent Lamps, and therefore this specification has been removed. ENERGY STAR will maintain a list on their website with the final qualifying list of products prior to this change and it is strongly recommended that programs continue to use this list as qualifying criteria for products in the programs.

This characterization assumes that the specialty CFL is installed in a residential location. If the implementation strategy does not allow for the installation location to be known (e.g. an upstream retail program) a deemed split of 95% Residential and 5% Commercial assumptions should be used⁷¹⁴.

This measure was developed to be applicable to the following program types: TOS, NC, DI, KITS.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the high-efficiency equipment must be a qualified specialty compact fluorescent lamp.

DEFINITION OF BASELINE EQUIPMENT

The baseline is a specialty incandescent light bulb including those exempt of the EISA 2007 standard: three-way, plant light, daylight bulb, bug light, post light, globes G40 (<40W), candelabra base (<60W), vibration service bulb, decorative candle with medium or intermediate base (<40W), shatter resistant and reflector bulbs and standard bulbs greater than 2601 lumens, and those non-exempt from EISA 2007: dimmable, globes (less than 5" diameter and >40W), candle (shapes B, BA, CA >40W, candelabra base lamps (>60W) and intermediate base lamps (>40W).

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 6.8 year⁷¹⁵ for bulbs exempt from EISA, or 3 years for bulbs non-exempt installed in 2018 and then for every subsequent year should be reduced by one year⁷¹⁶.

DEEMED MEASURE COST

For the Retail (Time of Sale) measure, the incremental capital cost for this measure is \$5⁷¹⁷.

For the Direct Install measure, the full cost of \$8.50 should be used plus \$5 labor⁷¹⁸ for a total of \$13.50. However

⁷¹⁴ RES v C&I split is based on a weighted (by sales volume) average of ComEd PY6, PY7 and PY8 and Ameren PY5, PY6 and PY8 in store intercept survey results. See 'RESvCI Split_112015.xls'.

⁷¹⁵ The assumed measure life for the specialty bulb measure characterization was reported in "Residential Lighting Measure Life Study", Nexus Market Research, June 4, 2008 (measure life for markdown bulbs). Measure life estimate does not distinguish between equipment life and measure persistence. Measure life includes products that were installed and operated until failure (i.e., equipment life) as well as those that were retired early and permanently removed from service for any reason, be it early failure, breakage, or the respondent not liking the product (i.e., measure persistence).

⁷¹⁶ Since the replacement baseline bulb from 2020 on will be equivalent to a CFL, no additional savings should be claimed from that point. Due to expected delay in clearing stock from retail outlets and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until 2021.

⁷¹⁷ NEEP Residential Lighting Survey, 2011

⁷¹⁸ Based on 15 minutes at \$20 per hour.

actual program delivery costs should be utilized if available.

For bulbs provided in Efficiency Kits, the actual program delivery costs should be utilized.

LOADSHAPE

Loadshape R06 - Residential Indoor Lighting

Loadshape R07 - Residential Outdoor Lighting

COINCIDENCE FACTOR

Unlike standard CFLs that could be installed in any room, certain types of specialty CFLs are more likely to be found in specific rooms, which affects the coincident peak factor. Coincidence factors by bulb types are presented below⁷¹⁹

Bulb Type	Peak CF
Three-way	0.078 ⁷²⁰
Dimmable	0.078 ⁷²¹
Interior reflector (incl. dimmable)	0.091
Exterior reflector	0.273
Candelabra base and candle medium and intermediate base	0.121
Bug light	0.273
Post light (>100W)	0.273
Daylight	0.081
Plant light	0.081
Globe	0.075
Vibration or shatterproof	0.081
Standard spirals >= 2601 lumens, Residential, Multi-family in unit	0.071
Standard spirals >= 2601 lumens, unknown	0.081
Standard spirals >= 2601 lumens, exterior	0.273
Specialty - Generic	0.081

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * (1 - \text{Leakage}) * \text{Hours} * \text{WHFe}$$

Where:

WattsBase = Actual wattage equivalent of incandescent specialty bulb, use the tables below to obtain the incandescent bulb equivalent wattage⁷²²; use 60W if unknown⁷²³

⁷¹⁹ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

⁷²⁰ Based on average of bedroom, dining room, office and living room results from the lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

⁷²¹ Ibid

⁷²² Based upon the draft ENERGY STAR specification for lamps

(http://energystar.gov/products/specs/sites/products/files/ENERGY_STAR_Lamps_V1_0_Draft%203.pdf) and the Energy Policy and Conservation Act of 2012.

⁷²³ A 2006-2008 California Upstream Lighting Evaluation found an average incandescent wattage of 61.7 Watts (KEMA, Inc, The Cadmus Group, Itron, Inc, PA Consulting Group, Jai J. Mitchell Analytics, Draft Evaluation Report: Upstream Lighting Program. Prepared for the California Public Utilities Commission, Energy Division. December 10, 2009)

EISA exempt bulb types:

Bulb Type	Lower Lumen Range	Upper Lumen Range	WattsBase
Standard Spirals >=2601	2601	2999	150
	3000	5279	200
	5280	6209	300
3-Way	250	449	25
	450	799	40
	800	1099	60
	1100	1599	75
	1600	1999	100
	2000	2549	125
	2550	2999	150
Globe (medium and intermediate bases less than 750 lumens)	90	179	10
	180	249	15
	250	349	25
	350	749	40
Decorative (Shapes B, BA, C, CA, DC, F, G, medium and intermediate bases less than 750 lumens)	70	89	10
	90	149	15
	150	299	25
	300	749	40
Globe (candelabra bases less than 1050 lumens)	90	179	10
	180	249	15
	250	349	25
	350	499	40
	500	1049	60
Decorative (Shapes B, BA, C, CA, DC, F, G, candelabra bases less than 1050 lumens)	70	89	10
	90	149	15
	150	299	25
	300	499	40
	500	1049	60

Directional Lamps - ENERGY STAR Minimum Luminous Efficacy = 40Lm/W for lamps with rated wattages less than 20W and 50 Lm/W for lamps with rated wattages >= 20 watts⁷²⁴.

For Directional R, BR, and ER lamp types⁷²⁵:

Bulb Type	Lower Lumen Range	Upper Lumen Range	WattsBase
R, ER, BR with medium screw bases w/ diameter >2.25" (*see exceptions below)	420	472	40
	473	524	45
	525	714	50
	715	937	65
	938	1259	75
	1260	1399	90
	1400	1739	100
	1740	2174	120

⁷²⁴ From pg 10 of the Energy Star Specification for lamps v1.1

⁷²⁵ From pg 11 of the Energy Star Specification for lamps v1.1

Bulb Type	Lower Lumen Range	Upper Lumen Range	Watts _{Base}
	2175	2624	150
	2625	2999	175
	3000	4500	200
*R, BR, and ER with medium screw bases w/ diameter <=2.25"	400	449	40
	450	499	45
	500	649	50
	650	1199	65
*ER30, BR30, BR40, or ER40	400	449	40
	450	499	45
	500	649	50
*BR30, BR40, or ER40	650	1419	65
*R20	400	449	40
	450	719	45
*All reflector lamps below lumen ranges specified above	200	299	20
	300	399	30

Directional lamps are exempt from EISA regulations.

For PAR, MR, and MRX Lamps Types:

For these highly focused directional lamp types, it is necessary to have Center Beam Candle Power (CBCP) and beam angle measurements to accurately estimate the equivalent baseline wattage. The formula below is based on the Energy Star Center Beam Candle Power tool.⁷²⁶ If CBCP and beam angle information are not available, or if the equation below returns a negative value (or undefined), use the manufacturer's recommended baseline wattage equivalent.⁷²⁷

Watts_{base} =

$$375.1 - 4.355(D) - \sqrt{227,800 - 937.9(D) - 0.9903(D^2) - 1479(BA) - 12.02(D * BA) + 14.69(BA^2) - 16,720 * \ln(CBCP)}$$

Where:

D = Bulb diameter (e.g. for PAR20 D = 20)

BA = Beam angle

CBCP = Center beam candle power

The result of the equation above should be rounded DOWN to the nearest wattage established by Energy Star:

Diameter	Permitted Wattages
16	20, 35, 40, 45, 50, 60, 75
20	50
30S	40, 45, 50, 60, 75
30L	50, 75
38	40, 45, 50, 55, 60, 65, 75, 85, 90, 100, 120, 150, 250

EISA non-exempt bulb types:

⁷²⁶ <http://energystar.supportportal.com/link/portal/23002/23018/Article/32655/>

⁷²⁷ The Energy Star Center Beam Candle Power tool does not accurately model baseline wattages for lamps with certain bulb characteristic combinations – specifically for lamps with very high CBCP.

Bulb Type	Lower Lumen Range	Upper Lumen Range	Incandescent Equivalent Post-EISA 2007 (WattsBase)
Dimmable Twist, Globe (less than 5" in diameter and > 749 lumens), candle (shapes B, BA, CA > 749 lumens), Candelabra Base Lamps (>1049 lumens), Intermediate Base Lamps (>749 lumens)	310	749	29
	750	1049	43
	1050	1489	53
	1490	2600	72

WattsEE = Actual wattage of energy efficient specialty bulb purchased, use 15W if unknown⁷²⁸

ISR = In Service Rate, the percentage of units rebated that are actually in service.

Program		Weighted Average 1st year In Service Rate (ISR)	2nd year Installations	3rd year Installations	Final Lifetime In Service Rate
Retail (Time of Sale)		88.0% ⁷²⁹	5.4%	4.6%	98.0% ⁷³⁰
Direct Install		96.9% ⁷³¹			
Efficiency Kits ⁷³²	CFL Distribution ⁷³³	59%	13%	11%	83%
	School Kits ⁷³⁴	61%	13%	11%	86%
	Direct Mail Kits ⁷³⁵	66%	14%	12%	93%

Leakage = Adjustment to account for the percentage of program bulbs that move out (and in if

⁷²⁸ An evaluation (Energy Efficiency / Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: Residential Energy Star ® Lighting) reported 13-17W as the most common specialty CFL wattage (69% of program bulbs). 2009 California data also reported an average CFL wattage of 15.5 Watts (KEMA, Inc, The Cadmus Group, Itron, Inc, PA Consulting Group, Jai J. Mitchell Analytics, Draft Evaluation Report: Upstream Lighting Program, Prepared for the California Public Utilities Commission, Energy Division. December 10, 2009).

⁷²⁹ 1st year in service rate is based upon review of PY4-6 evaluations from ComEd and PY5-6 from Ameren (see 'IL RES Lighting ISR_122014.xls' for more information. The average first year ISR was calculated weighted by the number of bulbs in the each year's survey.

⁷³⁰ The 98% Lifetime ISR assumption is consistent with the assumption for standard CFLs (in the absence of evidence that it should be different for this bulb type) based upon review of two evaluations:

'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings.

⁷³¹ Consistent with assumption for standard CFLs (in the absence of evidence that it should be different for this bulb type). Based upon review of the PY2 and PY3 ComEd Direct Install program surveys. This value includes bulb failures in the 1st year to be consistent with the Commission approval of annualization of savings for first year savings claims. ComEd PY2 All Electric Single Family Home Energy Performance Tune-Up Program Evaluation, Navigant Consulting, December 21, 2010.

⁷³² In Service Rates provided are for the bulb within a kit only. Given the significant differences in program design and the level of education provided through Efficiency Kits programs, the evaluators should apply the ISR estimated through evaluations (either past evaluations or the current program year evaluation) of the specific Efficiency Kits program. In cases where program-specific evaluation results for an ISR are unavailable, the default ISR values for Efficiency Kits provide may be used.

⁷³³ Free bulbs provided without request, with little or no education. Consistent with Standard CFL assumptions.

⁷³⁴ Kits provided free to students through school, with education program. Consistent with Standard CFL assumptions.

⁷³⁵ Opt-in program to receive kits via mail, with little or no education. Consistent with Standard CFL assumptions.

deemed appropriate⁷³⁶) of the Utility Jurisdiction.

KITS programs = Determined through evaluation

Upstream (TOS) Lighting programs = Determined through evaluation

or use deemed assumptions below⁷³⁷:

ComEd: 2.1%

Ameren: 13.1%

All other programs = 0

Hours = Average hours of use per year, varies by bulb type as presented below:⁷³⁸

Bulb Type	Annual hours of use (HOU)
Three-way	850
Dimmable	850
Interior reflector (incl. dimmable)	861
Exterior reflector	2475
Candelabra base and candle medium and intermediate base	1190
Bug light	2475
Post light (>100W)	2475
Daylight	847
Plant light	847
Globe	639
Vibration or shatterproof	847
Standard Spiral >2601 lumens, Residential, Multi Family in-unit	759
Standard Spiral >2601 lumens, unknown	847
Standard Spiral >2601 lumens, Exterior	2475
Specialty - Generic	847

WHFe = Waste heat factor for energy to account for cooling savings from efficient lighting

Bulb Location	WHFe
Interior single family or unknown location	1.06 ⁷³⁹

⁷³⁶ Leakage in is only appropriate to credit to IL utility program savings if it is reasonably expected that the IL utility program marketing efforts played an important role in influencing customer to purchase the light bulbs. Furthermore, consideration that such customers might be free riders should be addressed. If leakage in is assessed, efforts should be made to ensure no double counting of savings occurs if the evaluation is estimating both leakage in and spillover savings of light bulbs.

⁷³⁷ Leakage rate is based upon review of PY6-8 evaluations from ComEd and PY5,6 and 8 for Ameren (see 'IL Leakage Rates_112016.xls' for more information).

⁷³⁸ Hours of use by specialty bulb type calculated using the average hours of use in locations or rooms where each type of specialty bulb is most commonly found. Values for Reflector, Decorative and Globe are taken directly from the lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation. All other hours have been updated based on the room specific hours of use from the PY5/PY6 logger study.

⁷³⁹ The value is estimated at 1.06 (calculated as $1 + (0.66 * (0.27 / 2.8))$). Based on cooling loads decreasing by 27% of the lighting savings (average result from REMRate modeling of several different configurations and IL locations of homes), assuming typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm $(-0.02 * SEER2) + (1.12 * SEER)$ (from Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP) and 66% of homes in Illinois having central cooling ("Table HC7.9 Air Conditioning in Homes in Midwest

Bulb Location	W _{HFe}
Multi family in unit	1.04 ⁷⁴⁰
Exterior or uncooled location	1.0

DEFERRED INSTALLS

As presented above, the characterization assumes that a percentage of bulbs purchased are not installed until Year 2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

Year 1 (Purchase Year) installs: Characterized using assumptions provided above or evaluated assumptions if available.

Year 2 and 3 installs: Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.

The NTG factor for the Purchase Year should be applied.

For example, for a 13W dimmable CFL impacted by EISA 2007 (60W standard incandescent and 43W EISA qualified incandescent/halogen).

$$\Delta \text{kWh}_{1\text{st year installs}} = ((60 - 13) / 1000) * 0.823 * 850 * 1.06$$

$$= 34.9 \text{ kWh}$$

$$\Delta \text{kWh}_{2\text{nd year installs}} = ((43 - 13) / 1000) * 0.085 * 850 * 1.06$$

$$= 2.3 \text{ kWh}$$

Note: Here we assume no change in hours assumption. NTG value from Purchase year applied.

$$\Delta \text{kWh}_{3\text{rd year installs}} = ((43 - 13) / 1000) * 0.072 * 850 * 1.06$$

$$= 1.9 \text{ kWh}$$

Note: delta watts is equivalent to install year. Here we assume no change in hours assumption.

HEATING PENALTY

If electric heated home (if heating fuel is unknown assume gas, see Natural Gas section):

$$\Delta \text{kWh}^{741} = - (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * \text{HF}) / \eta \text{Heat}$$

Where:

HF = Heating Factor or percentage of light savings that must be heated

= 49%⁷⁴² for interior or unknown location

= 0% for exterior location

ηHeat = Efficiency in COP of Heating equipment

Region, Divisions, and States, 2009 from Energy Information Administration", 2009 Residential Energy Consumption Survey)

⁷⁴⁰ As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average).

⁷⁴¹ Negative value because this is an increase in heating consumption due to the efficient lighting.

⁷⁴² This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

= actual. If not available use⁷⁴³:

System Type	Age of Equipment	HSPF Estimate	COP _{HEAT} (COP Estimate) = (HSPF/3.413)*0.85
Heat Pump	Before 2006	6.8	1.7
	After 2006 - 2014	7.7	1.92
	2015 on	8.2	2.04
Resistance	N/A	N/A	1.00
Unknown ⁷⁴⁴	N/A	N/A	1.28

For example, a 15W globe CFL replacing a 60W incandescent specialty bulb installed in home with 2.0 COP Heat Pump (including duct loss):

$$\Delta \text{kWh}_{1\text{st year}} = - ((60 - 15) / 1000) * 0.823 * 639 * 0.49) / 2.0$$

$$= - 5.8 \text{ kWh}$$

Second and third year savings should be calculated using the appropriate ISR.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{WHFd} * \text{CF}$$

Where:

WHFd = Waste heat factor for demand to account for cooling savings from efficient lighting.

Bulb Location	WHFd
Interior single family or unknown location	1.11 ⁷⁴⁵
Multi family in unit	1.07 ⁷⁴⁶
Exterior or uncooled location	1.0

CF = Summer Peak Coincidence Factor for measure. Coincidence factors by bulb types are presented below⁷⁴⁷

⁷⁴³ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. Note efficiency should include duct losses. Defaults provided assume 15% duct loss for heat pumps.

⁷⁴⁴ Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see "HC6.9 Space Heating in Midwest Region.xls", using average for East North Central Region. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

⁷⁴⁵ The value is estimated at 1.11 (calculated as $1 + (0.66 * 0.466 / 2.8)$). See footnote relating to WHFe for details. Note the 46.6% factor represents the average Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load.

⁷⁴⁶ As above but using estimate of 45% of multifamily buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average);

<http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls>.

⁷⁴⁷ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

Bulb Type	Peak CF
Three-way	0.078 ⁷⁴⁸
Dimmable	0.078 ⁷⁴⁹
Interior reflector (incl. dimmable)	0.091
Exterior reflector	0.273
Candelabra base and candle medium and intermediate base	0.121
Bug light	0.273
Post light (>100W)	0.273
Daylight	0.081
Plant light	0.081
Globe	0.075
Vibration or shatterproof	0.081
Standard Spiral >=2601 lumens, Residential, Multi-family in unit	0.071
Standard spirals >= 2601 lumens, unknown	0.081
Standard spirals >= 2601 lumens, exterior	0.273
Specialty - Generic	0.081

Other factors as defined above

For example, a 15W specialty CFL replacing a 60W incandescent specialty bulb:

$$\begin{aligned}\Delta kW_{1st\ year} &= ((60 - 15) / 1000) * 0.823 * 1.11 * 0.081 \\ &= 0.003\ kW\end{aligned}$$

Second and third year savings should be calculated using the appropriate ISR.

NATURAL GAS SAVINGS

Heating Penalty if Natural Gas heated home (or if heating fuel is unknown):

$$\Delta \text{Therms}^{750} = - (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * \text{HF} * 0.03412) / \eta \text{Heat}$$

Where:

$$\begin{aligned}\text{HF} &= \text{Heating Factor or percentage of light savings that must be heated} \\ &= 49\%^{751} \text{ for interior or unknown location} \\ &= 0\% \text{ for exterior location} \\ 0.03412 &= \text{Converts kWh to Therms} \\ \eta \text{Heat} &= \text{Efficiency of heating system} \\ &= 70\%^{752}\end{aligned}$$

⁷⁴⁸ Based on average of bedroom, dining room, office and living room results from the lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

⁷⁴⁹ Ibid

⁷⁵⁰ Negative value because this is an increase in heating consumption due to the efficient lighting.

⁷⁵¹ This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

⁷⁵² This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of IL. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular

For example, a 15W Globe specialty CFL replacing a 60W incandescent specialty bulb:

$$\begin{aligned}\Delta \text{Therms} &= - (((60 - 15) / 1000) * 0.823 * 639 * 0.49 * 0.03412) / 0.7 \\ &= - 0.57 \text{ Therms}\end{aligned}$$

Second and third year savings should be calculated using the appropriate ISR.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

The following O&M assumptions should be used: Life of the baseline bulb is assumed to be 1.32 year⁷⁵³; baseline replacement cost is assumed to be \$3.5⁷⁵⁴.

It is important to note that for cost-effectiveness screening purposes, the O&M cost adjustments should only be applied in cases where the light bulbs area actually in service and so should be multiplied by the appropriate ISR.

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market or geographical area then that should be used.)

In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:

$$(0.24 * 0.92) + (0.76 * 0.8) * (1 - 0.15) = 0.70$$

⁷⁵³ Assuming 1000 hour rated life for incandescent bulb: 1000/759 = 1.32

⁷⁵⁴ NEEP Residential Lighting Survey, 2011

5.5.3 ENERGY STAR Torchiere

DESCRIPTION

A high efficiency ENERGY STAR fluorescent torchiere is purchased in place of a baseline mix of halogen and incandescent torchieres and installed in a residential setting.

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the fluorescent torchiere must meet ENERGY STAR efficiency standards.

DEFINITION OF BASELINE EQUIPMENT

The baseline is based on a mix of halogen and incandescent torchieres.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The lifetime of the measure is assumed to be 8 years⁷⁵⁵.

DEEMED MEASURE COST

The incremental cost for this measure is assumed to be \$5⁷⁵⁶.

LOADSHAPE

Loadshape R06 - Residential Indoor Lighting

Loadshape R07 - Residential Outdoor Lighting

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is 7.1% for Residential and in-unit Multi Family bulbs and 8.1% for bulbs installed in unknown locations⁷⁵⁷.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = ((\Delta Watts) / 1000) * ISR * (1 - Leakage) * HOURS * WHFe$$

Where:

$$\begin{aligned} \Delta Watts &= \text{Average delta watts per purchased ENERGY STAR torchiere} \\ &= 115.8^{758} \end{aligned}$$

⁷⁵⁵ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

⁷⁵⁶ DEER 2008 Database Technology and Measure Cost Data (www.deeresources.com) and consistent with Efficiency Vermont TRM.

⁷⁵⁷ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

⁷⁵⁸ Nexus Market Research, "Impact Evaluation of the Massachusetts, Rhode Island and Vermont 2003 Residential Lighting

ISR = In Service Rate or percentage of units rebated that get installed.
= 0.86 ⁷⁵⁹

Leakage = Adjustment to account for the percentage of program bulbs that move out (and in if deemed appropriate⁷⁶⁰) of the Utility Jurisdiction.

KITS programs = Determined through evaluation

Upstream (TOS) Lighting programs = Determined through evaluation

or use deemed assumptions below⁷⁶¹:

ComEd: 2.1%

Ameren: 13.1%

All other programs = 0

HOURS = Average hours of use per year

Installation Location	Hours
Residential and in-unit Multi Family	1095 (3.0 hrs per day) ⁷⁶²

WHFe = Waste Heat Factor for Energy to account for cooling savings from efficient lighting

Bulb Location	WHFe
Interior single family or unknown location	1.06 ⁷⁶³
Multi family in unit	1.04 ⁷⁶⁴
Exterior or uncooled location	1.0

For single family buildings:

$$\Delta \text{kWh} = (115.8 / 1000) * 0.86 * 1095 * 1.06$$

$$= 116 \text{ kWh}$$

Programs", Final Report, October 1, 2004, p. 43 (Table 4-9)

⁷⁵⁹ Nexus Market Research, RLW Analytics "Impact Evaluation of the Massachusetts, Rhode Island, and Vermont 2003 Residential Lighting Programs" table 6-3 on p63 indicates that 86% torchieres were installed in year one.

http://publicservice.vermont.gov/energy/ee_files/efficiency/eval/marivtreportfinal100104.pdf

⁷⁶⁰ Leakage in is only appropriate to credit to IL utility program savings if it is reasonably expected that the IL utility program marketing efforts played an important role in influencing customer to purchase the light bulbs. Furthermore, consideration that such customers might be free riders should be addressed. If leakage in is assessed, efforts should be made to ensure no double counting of savings occurs if the evaluation is estimating both leakage in and spillover savings of light bulbs.

⁷⁶¹ Leakage rate is based upon review of PY6-8 evaluations from ComEd and PY5,6 and 8 for Ameren (see 'IL Leakage Rates_112016.xls' for more information).

⁷⁶² Nexus Market Research, "Impact Evaluation of the Massachusetts, Rhode Island and Vermont 2003 Residential Lighting Programs", Final Report, October 1, 2004, p. 104 (Table 9-7)

⁷⁶³ The value is estimated at 1.06 (calculated as $1 + (0.66 * (0.27 / 2.8))$). Based on cooling loads decreasing by 27% of the lighting savings (average result from REMRate modeling of several different configurations and IL locations of homes), assuming typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm $(-0.02 * \text{SEER}^2) + (1.12 * \text{SEER})$ (from Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = $\text{EER} / 3.412 = 2.8\text{COP}$) and 66% of homes in Illinois having central cooling ("Table HC7.9 Air Conditioning in Homes in Midwest Region, Divisions, and States, 2009 from Energy Information Administration", 2009 Residential Energy Consumption Survey)

⁷⁶⁴ As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average)

For multi family in unit:

$$\begin{aligned}\Delta\text{kWh} &= (115.8 / 1000) * 0.86 * 1095 * 1.04 \\ &= 113 \text{ kWh}\end{aligned}$$

HEATING PENALTY

If electric heated home (if heating fuel is unknown assume gas, see Natural Gas section):

$$\Delta\text{kWh}^{765} = - ((\Delta\text{Watts}) / 1000) * \text{ISR} * \text{HOURS} * \text{HF} / \eta\text{Heat}$$

Where:

- HF = Heating Factor or percentage of light savings that must be heated
= 49%⁷⁶⁶ for interior or unknown location
- ηHeat = Efficiency in COP of Heating equipment
= Actual. If not available use defaults provided below⁷⁶⁷:

System Type	Age of Equipment	HSPF Estimate	COP _{HEAT} (COP Estimate) = (HSPF/3.413)*0.85
Heat Pump	Before 2006	6.8	1.7
	After 2006 - 2014	7.7	1.92
	2015 on	8.2	2.04
Resistance	N/A	N/A	1.00
Unknown ⁷⁶⁸	N/A	N/A	1.28

For example, an ES torchiere installed in a house with a 2016 heat pump:

$$\begin{aligned}\Delta\text{kWh} &= - ((115.8) / 1000) * 0.86 * 1095 * 0.49 / 2.04 \\ &= - 26.2 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta\text{kW} = ((\Delta\text{Watts}) / 1000) * \text{ISR} * \text{WHFd} * \text{CF}$$

Where:

- WHFd = Waste Heat Factor for Demand to account for cooling savings from efficient lighting

⁷⁶⁵ Negative value because this is an increase in heating consumption due to the efficient lighting.

⁷⁶⁶ This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

⁷⁶⁷ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. Note efficiency should include duct losses. Defaults provided assume 15% duct loss for heat pumps.

⁷⁶⁸ Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see "HC6.9 Space Heating in Midwest Region.xls", using average for East North Central Region. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

Bulb Location	WHFd
Interior single family or unknown location	1.11 ⁷⁶⁹
Multi family in unit	1.07 ⁷⁷⁰
Exterior or uncooled location	1.0

CF = Summer Peak Coincidence Factor for measure

Bulb Location	CF ⁷⁷¹
Interior single family or Multi family in unit	7.1%
Unknown location	8.1%

For single family and multi-family in unit buildings:

$$\Delta kW = (115.8 / 1000) * 0.86 * 1.11 * 0.071$$

$$= 0.008kW$$

For unknown location:

$$\Delta kW = (115.8 / 1000) * 0.86 * 1.07 * 0.081$$

$$= 0.009 kW$$

NATURAL GAS SAVINGS

Heating penalty if Natural Gas heated home, or if heating fuel is unknown.

$$\Delta Therms_{SWH} = - (((\Delta Watts) / 1000) * ISR * HOURS * 0.03412 * HF) / \eta_{Heat}$$

Where:

$\Delta Therms_{SWH}$ = gross customer annual heating fuel increased usage for the measure from the reduction in lighting heat in therms.

0.03412 = conversion from kWh to therms

HF = Heating Factor or percentage of light savings that must be heated
= 49%⁷⁷²

η_{Heat} = average heating system efficiency
= 70%⁷⁷³

⁷⁶⁹ The value is estimated at 1.11 (calculated as $1 + (0.66 * 0.466 / 2.8)$). See footnote relating to WHFe for details. Note the 46.6% factor represents the average Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load.

⁷⁷⁰ As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average)

⁷⁷¹ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

⁷⁷² This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

⁷⁷³ This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey) In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:

$$\begin{aligned}\Delta\text{Therms}_{\text{WH}} &= - ((115.8 / 1000) * 0.86 * 1095 * 0.03412 * 0.49) / 0.70 \\ &= - 2.60 \text{ therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

Life of the baseline bulb is assumed to be 1.83 years⁷⁷⁴ for residential and multifamily in unit. Baseline bulb cost replacement is assumed to be \$6.⁷⁷⁵

It is important to note that for cost-effectiveness screening purposes, the O&M cost adjustments should only be applied in cases where the light bulbs area actually in service and so should be multiplied by the appropriate ISR.

MEASURE CODE: RS-LTG-ESTO-V04-180101

REVIEW DEADLINE: 1/1/2020

$(0.24 * 0.92) + (0.76 * 0.8) * (1 - 0.15) = 0.70$

⁷⁷⁴ Based on VEIC assumption of baseline bulb (mix of incandescent and halogen) average rated life of 2000 hours, $2000/1095 = 1.83$ years.

⁷⁷⁵ Derived from Efficiency Vermont TRM.

5.5.4 Exterior Hardwired Compact Fluorescent Lamp (CFL) Fixture

DESCRIPTION

An ENERGY STAR lighting fixture wired for exclusive use with pin-based compact fluorescent lamps is installed in an exterior residential setting. This measure could relate to either a fixture replacement or new installation (i.e. time of sale).

Federal legislation stemming from the Energy Independence and Security Act of 2007 required all general-purpose light bulbs between 40 and 100W to be approximately 30% more energy efficient than current incandescent bulbs. Production of 100W, standard efficacy incandescent lamps ends in 2012, followed by restrictions on 75W in 2013 and 60W and 40W in 2014. The baseline for this measure has therefore become bulbs (improved incandescent or halogen) that meet the new standard.

A provision in the EISA regulations requires that by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, in essence making the baseline equivalent to a current day CFL. Therefore the measure life (number of years that savings should be claimed) should be reduced once the assumed lifetime of the bulb exceeds 2020. Due to expected delay in clearing retail inventory and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until 2021.

This measure was developed to be applicable to the following program types: TOS, NC. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is an ENERGY STAR lighting exterior fixture for pin-based compact fluorescent lamps.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is a standard EISA qualified incandescent or halogen exterior fixture as provided in the table provided in the Electric Energy Savings section.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected life of an exterior fixture is 20 years⁷⁷⁶. However due to the backstop provision in the Energy Independence and Security Act of 2007 that requires by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, the baseline replacement would become a CFL in that year. The expected measure life for CFL fixtures installed in 2018 is therefore assumed to be 3 years. For bulbs installed in 2019, this would be reduced to 2 years⁷⁷⁷.

DEEMED MEASURE COST

The incremental cost for an exterior fixture is assumed to be \$32⁷⁷⁸.

LOADSHAPE

Loadshape R07 - Residential Outdoor Lighting

⁷⁷⁶ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007 (<http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf>) gives 20 years for an interior fluorescent fixture.

⁷⁷⁷ Due to expected delay in clearing stock from retail outlets and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until 2021.

⁷⁷⁸ ENERGY STAR Qualified Lighting Savings Calculator default incremental cost input for exterior fixture (http://www.energystar.gov/buildings/sites/default/uploads/files/light_fixture_ceiling_fan_calculator.xlsx?4349-303e=&b6b3-3efd&b6b3-3efd)

COINCIDENCE FACTOR

The summer peak coincidence factor is assumed to be 27.3%⁷⁷⁹.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta kWh = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * (1 - \text{Leakage}) * \text{Hours}$$

Where:

WattsBase = Based on lumens of CFL bulb and program year purchased:

Minimum Lumens	Maximum Lumens	Incandescent Equivalent Post-EISA 2007 (WattsBase)
5280	6209	300
3000	5279	200
2601	2999	150
1490	2600	72
1050	1489	53
750	1049	43
310	749	29
250	309	25

WattsEE = Actual wattage of CFL purchased

ISR = In Service Rate or the percentage of units rebated that get installed.

Program	Weighted Average 1st year In Service Rate (ISR)	2nd year Installations	3rd year Installations	Final Lifetime In Service Rate
Retail (Time of Sale)	87.5% ⁷⁸⁰	5.7%	4.8%	98.0% ⁷⁸¹
Direct Install	96.9 ⁷⁸²			

Leakage = Adjustment to account for the percentage of program bulbs that move out (and in if

⁷⁷⁹ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

⁷⁸⁰ 1st year in service rate is based upon review of PY2-3 evaluations from ComEd (see 'IL RES Lighting ISR.xls' for more information. The average first year ISR was calculated weighted by the number of bulbs in the each year's survey.

⁷⁸¹ The 98% Lifetime ISR assumption is consistent with the assumption for standard CFLs (in the absence of evidence that it should be different for this bulb type) based upon review of two evaluations:

'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report: Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings.

⁷⁸² In the absence of evaluation results for Direct Install Fixtures specifically, this is made consistent with the Direct Install CFL measure which is based upon review of the PY2 and PY3 ComEd Direct Install program surveys.

deemed appropriate⁷⁸³) of the Utility Jurisdiction.

KITS programs = Determined through evaluation

Upstream (TOS) Lighting programs = Determined through evaluation

or use deemed assumptions below⁷⁸⁴:

ComEd: 1.05%

Ameren: 6.55%

All other programs = 0

Hours = Average hours of use per year

=2475 (6.78 hrs per day)⁷⁸⁵

DEFERRED INSTALLS

As presented above, the characterization assumes that a percentage of bulbs purchased are not installed until Year 2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

Year 1 (Purchase Year) installs: Characterized using assumptions provided above or evaluated assumptions if available.

Year 2 and 3 installs: Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.

The NTG factor for the Purchase Year should be applied.

For example, for a 2 x 14W pin based CFL fixture (43W EISA qualified incandescent/halogen).

$$\Delta \text{kWh}_{1\text{st year installs}} = ((86 - 28) / 1000) * 0.875 * 2475$$

$$= 125.6 \text{ kWh}$$

$$\Delta \text{kWh}_{2\text{nd year installs}} = ((86 - 28) / 1000) * 0.057 * 2475$$

$$= 8.2 \text{ kWh}$$

Note: Here we assume no change in hours assumption. NTG value from Purchase year applied.

$$\Delta \text{kWh}_{3\text{rd year installs}} = ((86 - 28) / 1000) * 0.048 * 2475$$

$$= 6.9 \text{ kWh}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{CF}$$

Where:

⁷⁸³ Leakage in is only appropriate to credit to IL utility program savings if it is reasonably expected that the IL utility program marketing efforts played an important role in influencing customer to purchase the light bulbs. Furthermore, consideration that such customers might be free riders should be addressed. If leakage in is assessed, efforts should be made to ensure no double counting of savings occurs if the evaluation is estimating both leakage in and spillover savings of light bulbs.

⁷⁸⁴ Leakage rate is based upon TAC agreed 50% of the lamp leakage assumptions (based upon review of PY6-8 evaluations from ComEd and PY5,6 and 8 for Ameren (see 'IL Leakage Rates_112016.xls' for more information)).

⁷⁸⁵ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

CF = Summer Peak Coincidence Factor for measure.

= 27.3%⁷⁸⁶

Other factors as defined above

For example, a 2 x 14W pin-based CFL fixture:

$$\begin{aligned}\Delta kW_{1st\ year} &= ((86 - 28) / 1000) * 0.875 * 0.273 \\ &= 0.0142\ kW\end{aligned}$$

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year.

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

Life of the baseline bulb is assumed to be 0.4 years⁷⁸⁷ for exterior applications. Baseline bulb cost replacement is assumed to be \$1.25.⁷⁸⁸

It is important to note that for cost-effectiveness screening purposes, the O&M cost adjustments should only be applied in cases where the light bulbs are actually in service and so should be multiplied by the appropriate ISR.

MEASURE CODE: RS-LRG-EFOX-V06-180101

REVIEW DEADLINE: 1/1/2020

⁷⁸⁶ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

⁷⁸⁷ Calculated by dividing assumed rated life of baseline bulb by hours of use. Assumed lifetime of EISA qualified Halogen/Incandescents is 1000 hours. The manufacturers are simply using a regular incandescent lamp with halogen fill gas rather than Halogen Infrared to meet the standard (as provided by G. Arnold, NEEP and confirmed by N. Horowitz at NRDC).

⁷⁸⁸ Based upon field data collected by CLEAResult and provided by ComEd. See ComEd Pricing Projections 06302016.xlsx for analysis.

5.5.5 Interior Hardwired Compact Fluorescent Lamp (CFL) Fixture

DESCRIPTION

An ENERGY STAR lighting fixture wired for exclusive use with pin-based compact fluorescent lamps is installed in an interior residential setting. This measure could relate to either a fixture replacement or new installation (i.e. time of sale).

Federal legislation stemming from the Energy Independence and Security Act of 2007 required all general-purpose light bulbs between 40 and 100W to be approximately 30% more energy efficient than current incandescent bulbs. Production of 100W, standard efficacy incandescent lamps ends in 2012, followed by restrictions on 75W in 2013 and 60W and 40W in 2014. The baseline for this measure has therefore become bulbs (improved incandescent or halogen) that meet the new standard.

A provision in the EISA regulations requires that by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, in essence making the baseline equivalent to a current day CFL. Therefore the measure life (number of years that savings should be claimed) should be reduced once the assumed lifetime of the bulb exceeds 2020. Due to expected delay in clearing retail inventory and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until 2021.

This measure was developed to be applicable to the following program types: TOS, NC. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is an ENERGY STAR lighting interior fixture for pin-based compact fluorescent lamps.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is a standard EISA qualified incandescent or halogen interior fixture as provided in the table provided in the Electric Energy Savings section.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected life of an interior fixture is 20 years⁷⁸⁹. However due to the backstop provision in the Energy Independence and Security Act of 2007 that requires by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, the baseline replacement would become equivalent to a CFL in that year. The expected measure life for CFL fixtures installed in 2018 is therefore assumed to be 3 years. For bulbs installed in 2019, this would be reduced to 2 years and should be reduced each year⁷⁹⁰.

DEEMED MEASURE COST

The incremental cost for an interior fixture is assumed to be \$32⁷⁹¹.

LOADSHAPE

Loadshape R06 - Residential Indoor Lighting

⁷⁸⁹ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007 (<http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf>) gives 20 years for an interior fluorescent fixture.

⁷⁹⁰ Due to expected delay in clearing stock from retail outlets and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until 2021.

⁷⁹¹ ENERGY STAR Qualified Lighting Savings Calculator default incremental cost input for interior fixture (http://www.energystar.gov/buildings/sites/default/uploads/files/light_fixture_ceiling_fan_calculator.xlsx?4349-303e=&b6b3-3efd&b6b3-3efd)

COINCIDENCE FACTOR

The summer peak coincidence factor is assumed to be 7.1%⁷⁹² for Residential and in-unit Multi Family bulbs.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta \text{kWh} = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * (1 - \text{Leakage}) * \text{Hours} * \text{WHFe}$$

Where:

WattsBase = Based on lumens of CFL bulb and program year purchased:

Minimum Lumens	Maximum Lumens	Incandescent Equivalent Post-EISA 2007 (WattsBase)
5280	6209	300
3000	5279	200
2601	2999	150
1490	2600	72
1050	1489	53
750	1049	43
310	749	29
250	309	25

WattsEE = Actual wattage of CFL purchased

ISR = In Service Rate or the percentage of units rebated that get installed.

Program	Weighted Average 1st year In Service Rate (ISR)	2nd year Installations	3rd year Installations	Final Lifetime In Service Rate
Retail (Time of Sale)	87.5% ⁷⁹³	5.7%	4.8%	98.0% ⁷⁹⁴
Direct Install	96.9% ⁷⁹⁵			

⁷⁹² Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

⁷⁹³ 1st year in service rate is based upon review of PY2-3 evaluations from ComEd (see 'IL RES Lighting ISR.xls' for more information. The average first year ISR was calculated weighted by the number of bulbs in the each year's survey.

⁷⁹⁴ The 98% Lifetime ISR assumption is consistent with the assumption for standard CFLs (in the absence of evidence that it should be different for this bulb type) based upon review of two evaluations:

'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report:, Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings.

⁷⁹⁵ In the absence of evaluation results for Direct Install Fixtures specifically, this is made consistent with the Direct Install CFL measure which is based upon review of the PY2 and PY3 ComEd Direct Install program surveys.

Leakage = Adjustment to account for the percentage of program bulbs that move out (and in if deemed appropriate⁷⁹⁶) of the Utility Jurisdiction.

KITS programs = Determined through evaluation

Upstream (TOS) Lighting programs = Determined through evaluation

or use deemed assumptions below⁷⁹⁷:

ComEd: 1.05%

Ameren: 6.55%

All other programs = 0

Hours = Average hours of use per year

Installation Location	Hours
Residential and in-unit Multi Family	759 ⁷⁹⁸

WHFe = Waste heat factor for energy to account for cooling energy savings from efficient lighting

Bulb Location	WHFe
Interior single family or unknown location	1.06 ⁷⁹⁹
Multi family in unit	1.04 ⁸⁰⁰

DEFERRED INSTALLS

As presented above, the characterization assumes that a percentage of bulbs purchased are not installed until Year 2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

Year 1 (Purchase Year) installs: Characterized using assumptions provided above or evaluated assumptions if available.

Year 2 and 3 installs: Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.

The NTG factor for the Purchase Year should be applied.

⁷⁹⁶ Leakage in is only appropriate to credit to IL utility program savings if it is reasonably expected that the IL utility program marketing efforts played an important role in influencing customer to purchase the light bulbs. Furthermore, consideration that such customers might be free riders should be addressed. If leakage in is assessed, efforts should be made to ensure no double counting of savings occurs if the evaluation is estimating both leakage in and spillover savings of light bulbs.

⁷⁹⁷ Leakage rate is based upon TAC agreed 50% of the lamp leakage assumptions (based upon review of PY6-8 evaluations from ComEd and PY5,6 and 8 for Ameren (see 'IL Leakage Rates_112016.xls' for more information)).

⁷⁹⁸ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

⁷⁹⁹ The value is estimated at 1.06 (calculated as $1 + (0.66 * (0.27 / 2.8))$). Based on cooling loads decreasing by 27% of the lighting savings (average result from REMRate modeling of several different configurations and IL locations of homes), assuming typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm $(-0.02 * SEER2) + (1.12 * SEER)$ (from Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP) and 66% of homes in Illinois having central cooling ("Table HC7.9 Air Conditioning in Homes in Midwest Region, Divisions, and States, 2009 from Energy Information Administration", 2009 Residential Energy Consumption Survey)

⁸⁰⁰ As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average)

For example, for a 2 x 14W pin based CFL fixture (43W EISA qualified incandescent/halogen):

$$\Delta kWh_{1st \text{ year installs}} = ((86 - 28) / 1000) * 0.875 * 759 * 1.06$$

$$= 40.8 \text{ kWh}$$

$$\Delta kWh_{2nd \text{ year installs}} = ((86 - 28) / 1000) * 0.057 * 759 * 1.06$$

$$= 2.7 \text{ kWh}$$

Note: Here we assume no change in hours assumption. NTG value from Purchase year applied.

$$\Delta kWh_{3rd \text{ year installs}} = ((86 - 28) / 1000) * 0.048 * 759 * 1.06$$

$$= 2.2 \text{ kWh}$$

HEATING PENALTY

If electric heated building:

$$\Delta kWh^{801} = - (((WattsBase - WattsEE) / 1000) * ISR * Hours * HF) / \eta_{Heat}$$

Where:

- HF = Heating Factor or percentage of light savings that must be heated
 = 49%⁸⁰² for interior or unknown location
 = 0% for unheated location
- η_{Heat} = Efficiency in COP of Heating equipment
 = actual. If not available use⁸⁰³:

System Type	Age of Equipment	HSPF Estimate	COP ^{HEAT} (COP Estimate) = (HSPF/3.413)*0.85
Heat Pump	Before 2006	6.8	1.7
	After 2006 - 2014	7.7	1.92
	2015 on	8.2	2.04
Resistance	N/A	N/A	1.00
Unknown ⁸⁰⁴	N/A	N/A	1.28

⁸⁰¹ Negative value because this is an increase in heating consumption due to the efficient lighting.

⁸⁰² This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

⁸⁰³ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. Note efficiency should include duct losses. Defaults provided assume 15% duct loss for heat pumps.

⁸⁰⁴ Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see "HC6.9 Space Heating in Midwest Region.xls", using average for East North Central Region. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

For example, a 2 x 14W pin-based CFL fixture is purchased and installed in home with 2.0 COP (including duct loss) Heat Pump:

$$\begin{aligned}\Delta kWh_{1st\ year} &= - ((86 - 28) / 1000) * 0.875 * 759 * 0.49) / 2.0 \\ &= - 9.4\ kWh\end{aligned}$$

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = ((\text{WattsBase} - \text{WattsEE}) / 1\ 000) * \text{ISR} * \text{WHFd} * \text{CF}$$

Where:

WHFd = Waste heat factor for demand to account for cooling savings from efficient lighting.

Bulb Location	WHFd
Interior single family or unknown location	1.11 ⁸⁰⁵
Multi family in unit	1.07 ⁸⁰⁶
Exterior or uncooled location	1.0

CF = Summer Peak Coincidence Factor for measure.

Bulb Location	CF ⁸⁰⁷
Interior single family or unknown location	7.1%
Multi family in unit	7.1%

Other factors as defined above

For example, a 14W pin-based CFL fixture:

$$\begin{aligned}\Delta kW_{1st\ year} &= ((86 - 28) / 1000) * 0.875 * 1.11 * 0.071 \\ &= 0.004\ kW\end{aligned}$$

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year.

NATURAL GAS SAVINGS

$$\Delta \text{Therms}^{808} = - (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * \text{HF} * 0.03412) / \eta \text{Heat}$$

Where:

HF = Heating Factor or percentage of light savings that must be heated

⁸⁰⁵ The value is estimated at 1.11 (calculated as 1 + (0.66 * 0.466 / 2.8)). See footnote relating to WHFe for details. Note the 46.6% factor represents the average Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load.

⁸⁰⁶ As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average)

⁸⁰⁷ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

⁸⁰⁸ Negative value because this is an increase in heating consumption due to the efficient lighting.

	= 49% ⁸⁰⁹ for interior or unknown location
	= 0% for unheated location
0.03412	=Converts kWh to Therms
η_{Heat}	= Efficiency of heating system
	=70% ⁸¹⁰

For example, a 2 x 14W pin-based CFL fixture is purchased and installed in home with gas heat at 70% efficiency:

$$\Delta \text{Therms}_{\text{1st year}} = -((86 - 28) / 1000) * 0.875 * 759 * 0.49 * 0.03412 / 0.7$$

$$= - 0.9 \text{ Therms}$$

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

Life of the baseline bulb is assumed to be 1.3 years⁸¹¹ for interior applications. Baseline bulb cost replacement is assumed to be \$1.25.⁸¹²

It is important to note that for cost-effectiveness screening purposes, the O&M cost adjustments should only be applied in cases where the light bulbs are actually in service and so should be multiplied by the appropriate ISR.

MEASURE CODE: RS-LTG-IFIX-V06-180101

REVIEW DEADLINE: 1/1/2020

⁸⁰⁹ This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

⁸¹⁰ This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey). In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:

$(0.24 * 0.92) + (0.76 * 0.8) * (1 - 0.15) = 0.70$

⁸¹¹ Calculated by dividing assumed rated life of baseline bulb by hours of use. Assumed lifetime of EISA qualified Halogen/Incandescent is 1000 hours. The manufacturers are simply using a regular incandescent lamp with halogen fill gas rather than Halogen Infrared to meet the standard (as provided by G. Arnold, NEEP and confirmed by N. Horowitz at NRDC).

⁸¹² Based upon field data collected by CLEAResult and provided by ComEd. See ComEd Pricing Projections 06302016.xlsx for analysis.

5.5.6 LED Specialty Lamps

DESCRIPTION

This measure describes savings from a variety of specialty LED lamp types (including globe, decorative and downlights). This characterization assumes that the LED lamp or fixture is installed in a residential location. Where the implementation strategy does not allow for the installation location to be known (e.g. an upstream retail program) a deemed split of 95% Residential and 5% Commercial assumptions should be used⁸¹³.

This measure was developed to be applicable to the following program types: TOS, NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be an ENERGY STAR LED lamp or fixture. Note a new ENERGY STAR specification v2.0 becomes effective on 1/2/2017

(https://www.energystar.gov/products/spec/lamps_specification_version_2_0_pd).

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is assumed to be an incandescent/halogen lamp for all lamp types.

The baseline for the early replacement measure is the existing bulb being replaced.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

While LED rated lives are often 15,000 – 50,000 hours, all installations are assumed to be 10 years⁸¹⁴ except for recessed downlight and track lights at 15 years⁸¹⁵

For early replacement measures, if replacing a halogen or incandescent bulb, the remaining life is assumed to be 333 hours. For CFL's, the remaining life is 3,333 hours⁸¹⁶.

DEEMED MEASURE COST

The price of LED lamps is falling quickly. Where possible, the actual cost should be used and compared to the baseline cost provided below. If the incremental cost is unknown, assume the following⁸¹⁷:

Bulb Type	Year	Incandescent	LED	Incremental Cost
Recessed Downlight Luminaires	All	\$4.00	\$94.00	\$90.00
Track Lights	All	\$4.00	\$60.00	\$56.00
Directional	2017	\$3.53	\$6.24	\$2.71
	2018-2019		\$5.18	\$1.65
Decorative and Globe	2017	\$1.60	\$3.50	\$1.90
	2018-2019	\$1.74	\$3.40	\$1.66

⁸¹³ RES v C&I split is based on a weighted (by sales volume) average of ComEd PY6, PY7 and PY8 and Ameren PY5, PY6 and PY8 in store intercept survey results. See 'RESvCI Split_112016.xls'.

⁸¹⁴ Based on recommendation in the Dunskey Energy Consulting, Livingston Energy Innovations and Opinion Dynamics Corporation; NEEP Emerging Technology Research Report, p 6-18.

⁸¹⁵ Limited by persistence. NEEP EMV Emerging Technologies Research Report (December 2011)

⁸¹⁶ Representing a third of the expected lamp lifetime.

⁸¹⁷ Baseline and LED lamp costs for both directional and decorative and globe are based on field data collected by CLEAResult and provided by ComEd. See ComEd Pricing Projections 06302016.xlsx for analysis. Recessed downlight and track light costs are based on VEIC review of a year's worth of LED sales data through VEIC implemented programs and the retail cost averaged (see 2015 LED Sales Review.xls) and of price reports provided to Efficiency Vermont by a number of manufacturers and retailers. Baseline cost based on "2010-2012 WA017 Ex Ante Measure Cost Study Draft Report", Itron, February 28, 2014.

LOADSHAPE

Loadshape R06 - Residential Indoor Lighting

Loadshape R07 - Residential Outdoor Lighting

COINCIDENCE FACTOR

Unlike standard lamps that could be installed in any room, certain types of specialty lamps are more likely to be found in specific rooms, which affects the coincident peak factor. Coincidence factors by bulb types are presented below⁸¹⁸

Bulb Type	Peak CF
Three-way	0.078 ⁸¹⁹
Dimmable	0.078 ⁸²⁰
Interior reflector (incl. dimmable)	0.091
Exterior reflector	0.273
Unknown reflector	0.094
Candelabra base and candle medium and intermediate base	0.121
Bug light	0.273
Post light (>100W)	0.273
Daylight	0.081
Plant light	0.081
Globe	0.075
Vibration or shatterproof	0.081
Standard Spiral >=2601 lumens, Residential, Multi-family in unit	0.071
Standard spirals >= 2601 lumens, unknown	0.081
Standard spirals >= 2601 lumens, exterior	0.273
Specialty - Generic	0.081

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta kWh = ((WattsBase - WattsEE) / 1000) * ISR * (1 - Leakage) * Hours * WHFe$$

Where:

Watts_{base} = Input wattage of the existing or baseline system. Reference the table below for default values.

EISA exempt bulb types:

Bulb Type	Lower Lumen Range	Upper Lumen Range	WattsBase
Standard Spirals >=2601	2601	2999	150

⁸¹⁸ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

⁸¹⁹ Based on average of bedroom, dining room, office and living room results from the lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

⁸²⁰ Ibid

Bulb Type	Lower Lumen Range	Upper Lumen Range	WattsBase
	3000	5279	200
	5280	6209	300
3-Way	250	449	25
	450	799	40
	800	1099	60
	1100	1599	75
	1600	1999	100
	2000	2549	125
	2550	2999	150
Globe (medium and intermediate bases less than 750 lumens)	90	179	10
	180	249	15
	250	349	25
	350	749	40
Decorative (Shapes B, BA, C, CA, DC, F, G, medium and intermediate bases less than 750 lumens)	70	89	10
	90	149	15
	150	299	25
	300	749	40
Globe (candelabra bases less than 1050 lumens)	90	179	10
	180	249	15
	250	349	25
	350	499	40
	500	1049	60
Decorative (Shapes B, BA, C, CA, DC, F, G, candelabra bases less than 1050 lumens)	70	89	10
	90	149	15
	150	299	25
	300	499	40
	500	1049	60

Directional Lamps -

For Directional R, BR, and ER lamp types⁸²¹:

Bulb Type	Lower Lumen Range	Upper Lumen Range	WattsBase
R, ER, BR with medium screw bases w/ diameter >2.25" (*see exceptions below)	420	472	40
	473	524	45
	525	714	50
	715	937	65
	938	1259	75
	1260	1399	90
	1400	1739	100
	1740	2174	120
	2175	2624	150
	2625	2999	175
	3000	4500	200
	400	449	40

⁸²¹ From pg 11 of the Energy Star Specification for lamps v1.1

Bulb Type	Lower Lumen Range	Upper Lumen Range	Watts _{Base}
*R, BR, and ER with medium screw bases w/ diameter <=2.25"	450	499	45
	500	649	50
	650	1199	65
*ER30, BR30, BR40, or ER40	400	449	40
	450	499	45
	500	649	50
*BR30, BR40, or ER40	650	1419	65
*R20	400	449	40
	450	719	45
*All reflector lamps below lumen ranges specified above	200	299	20
	300	399	30

Directional lamps are exempt from EISA regulations.

For PAR, MR, and MRX Lamps Types:

For these highly focused directional lamp types, it is necessary to have Center Beam Candle Power (CBCP) and beam angle measurements to accurately estimate the equivalent baseline wattage. The formula below is based on the Energy Star Center Beam Candle Power tool.⁸²² If CBCP and beam angle information are not available or if the equation below returns a negative value (or undefined), use the manufacturer's recommended baseline wattage equivalent.⁸²³

Wattsbase =

$$375.1 - 4.355(D) - \sqrt{227,800 - 937.9(D) - 0.9903(D^2) - 1479(BA) - 12.02(D * BA) + 14.69(BA^2) - 16,720 * \ln(CBCP)}$$

Where:

D = Bulb diameter (e.g. for PAR20 D = 20)

BA = Beam angle

CBCP = Center beam candle power

The result of the equation above should be rounded DOWN to the nearest wattage established by Energy Star:

Diameter	Permitted Wattages
16	20, 35, 40, 45, 50, 60, 75
20	50
30S	40, 45, 50, 60, 75
30L	50, 75
38	40, 45, 50, 55, 60, 65, 75, 85, 90, 100, 120, 150, 250

⁸²² <http://energystar.supportportal.com/link/portal/23002/23018/Article/32655/>

⁸²³ The Energy Star Center Beam Candle Power tool does not accurately model baseline wattages for lamps with certain bulb characteristic combinations – specifically for lamps with very high CBCP.

EISA non-exempt bulb types:

Bulb Type	Lower Lumen Range	Upper Lumen Range	Incandescent Equivalent Post-EISA 2007 (WattsBase)
Dimmable Twist, Globe (less than 5" in diameter and > 749 lumens), candle (shapes B, BA, CA > 749 lumens), Candelabra Base Lamps (>1049 lumens), Intermediate Base Lamps (>749 lumens)	310	749	29
	750	1049	43
	1050	1489	53
	1490	2600	72

Watt_{SEE} = Actual wattage of LED purchased / installed.

ISR = In Service Rate or the percentage of units rebated that get installed

Program	Bulb Type	Weighted Average 1 st year In Service Rate (ISR)	2 nd year Installations	3 rd year Installations	Final Lifetime In Service Rate
Retail (Time of Sale)	Recessed downlight luminaries and Track Lights	100% ⁸²⁴			
	All other lamps	93.5% ⁸²⁵	2.4%	2.1%	98.0% ⁸²⁶
Direct Install	All lamps	96.9% ⁸²⁷			

Leakage = Adjustment to account for the percentage of program bulbs that move out (and in if deemed appropriate⁸²⁸) of the Utility Jurisdiction.

KITS programs = Determined through evaluation

Upstream (TOS) Lighting programs = Determined through evaluation

or use deemed assumptions below⁸²⁹:

⁸²⁴ NEEP EMV Emerging Technologies Research Report (December 2011)

⁸²⁵ 1st year in service rate is based upon analysis of ComEd PY7 and PY8 intercept data (see 'IL RES Lighting ISR_112016.xls' for more information).

⁸²⁶ The 98% Lifetime ISR assumption is based upon the standard CFL measure in the absence of any better reference. This value is based upon review of two evaluations:

'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report: Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings.

⁸²⁷ Consistent with assumption for standard CFLs (in the absence of evidence that it should be different for this bulb type). Based upon review of the PY2 and PY3 ComEd Direct Install program surveys. This value includes bulb failures in the 1st year to be consistent with the Commission approval of annualization of savings for first year savings claims. ComEd PY2 All Electric Single Family Home Energy Performance Tune-Up Program Evaluation, Navigant Consulting, December 21, 2010.

⁸²⁸ Leakage in is only appropriate to credit to IL utility program savings if it is reasonably expected that the IL utility program marketing efforts played an important role in influencing customer to purchase the light bulbs. Furthermore, consideration that such customers might be free riders should be addressed. If leakage in is assessed, efforts should be made to ensure no double counting of savings occurs if the evaluation is estimating both leakage in and spillover savings of light bulbs.

⁸²⁹ Leakage rate is based upon review of PY6-8 evaluations from ComEd and PY5,6 and 8 for Ameren (see 'IL Leakage Rates_112016.xls' for more information).

ComEd:	2.1%
Ameren:	13.1%
All other programs	= 0

Hours = Average hours of use per year ⁸³⁰

Bulb Type	Annual hours of use (HOU)
Three-way	850
Dimmable	850
Interior reflector (incl. dimmable)	861
Exterior reflector	2475
Unknown reflector	891
Candelabra base and candle medium and intermediate base	1190
Bug light	2475
Post light (>100W)	2475
Daylight	847
Plant light	847
Globe	639
Vibration or shatterproof	847
Standard Spiral >2601 lumens, Residential, Multi Family in-unit	759
Standard Spiral >2601 lumens, unknown	847
Standard Spiral >2601 lumens, Exterior	2475
Specialty – Generic Interior	847
Specialty – Generic Exterior	2475

WHFe = Waste heat factor for energy to account for cooling savings from efficient lighting

Bulb Location	WHFe
Interior single family or unknown location	1.06 ⁸³¹
Multi family in unit	1.04 ⁸³²
Exterior or uncooled location	1.0

For example, a 13W PAR20 LED is installed in place of a 750 lumen PAR20 incandescent screw-in lamp with medium

⁸³⁰ Hours of use by specialty bulb type calculated using the average hours of use in locations or rooms where each type of specialty bulb is most commonly found. Values for Reflector, Decorative and Globe are taken directly from the lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation. All other hours have been updated based on the room specific hours of use from the PY5/PY6 logger study.

⁸³¹ The value is estimated at 1.06 (calculated as $1 + (0.66 * (0.27 / 2.8))$). Based on cooling loads decreasing by 27% of the lighting savings (average result from REMRate modeling of several different configurations and IL locations of homes), assuming typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm $(-0.02 * SEER2) + (1.12 * SEER)$ (from Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP) and 66% of homes in Illinois having central cooling ("Table HC7.9 Air Conditioning in Homes in Midwest Region, Divisions, and States, 2009 from Energy Information Administration", 2009 Residential Energy Consumption Survey)

⁸³² As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average)

screw base, diameter >2.5", installed in single family interior location:

$$\begin{aligned}\Delta\text{kWh} &= ((45 - 13) / 1000) * 0.935 * 861 * 1.06 \\ &= 27.3 \text{ kWh}\end{aligned}$$

Mid Life Baseline Adjustment

For non-exempt lamps, an appropriate baseline adjustment should be included to account for the 2020 EISA backstop provision making replacement baseline lamps meet 45 lumens/watt. Due to expected delay in clearing retail inventory and to account for the operating life of a halogen incandescent potentially spanning over 2020, this shift is assumed not to occur until 2021.

Note for early replacement measures an additional baseline shift accounting for the replacement of the existing unit with a new baseline lamp should be accounted for.

HEATING PENALTY

If electric heated home (if heating fuel is unknown assume gas, see Natural Gas section):

$$\Delta\text{kWh}^{833} = - (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * \text{HF}) / \eta\text{Heat}$$

Where:

- HF = Heating Factor or percentage of light savings that must be heated
 = 49%⁸³⁴ for interior or unknown location
 = 0% for exterior location
- ηHeat = Efficiency in COP of Heating equipment
 = Actual. If not available use: ⁸³⁵:

System Type	Age of Equipment	HSPF Estimate	COP _{HEAT} (COP Estimate) = (HSPF/3.413)*0.85
Heat Pump	Before 2006	6.8	1.7
	After 2006 - 2014	7.7	1.92
	2015 on	8.2	2.04
Resistance	N/A	N/A	1.00
Unknown ⁸³⁶	N/A	N/A	1.28

⁸³³ Negative value because this is an increase in heating consumption due to the efficient lighting.

⁸³⁴ This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

⁸³⁵ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. Note efficiency should include duct losses. Defaults provided assume 15% duct loss for heat pumps.

⁸³⁶ Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see "HC6.9 Space Heating in Midwest Region.xls", using average for East North Central Region. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

For example, a 13W PAR20 LED is installed in place of a 750 lumen PAR20 incandescent screw-in lamp with medium screw base, diameter >2.5", installed in single family interior location with a 2016 heat pump:

$$\begin{aligned}\Delta kWh &= - ((45 - 13) / 1000) * 0.935 * 861 * 0.49) / 2.04 \\ &= - 6.19 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{WHFd} * \text{CF}$$

Where:

WHFd = Waste heat factor for demand to account for cooling savings from efficient lighting.

Bulb Location	WHFd
Interior single family or unknown location	1.11 ⁸³⁷
Multi family in unit	1.07 ⁸³⁸
Exterior or uncooled location	1.0

CF = Summer Peak Coincidence Factor for measure, see above for values. ⁸³⁹

Bulb Type	Peak CF
Three-way	0.078 ⁸⁴⁰
Dimmable	0.078 ⁸⁴¹
Interior reflector (incl. dimmable)	0.091
Exterior reflector	0.273
Unknown reflector	0.094
Candelabra base and candle medium and intermediate base	0.121
Bug light	0.273
Post light (>100W)	0.273
Daylight	0.081
Plant light	0.081
Globe	0.075
Vibration or shatterproof	0.081
Standard Spiral >=2601 lumens, Residential, Multi-family in unit	0.071
Standard spirals >= 2601 lumens, unknown	0.081
Standard spirals >= 2601 lumens, exterior	0.273
Specialty - Generic	0.081

Other factors as defined above

⁸³⁷ The value is estimated at 1.11 (calculated as $1 + (0.66 * 0.466 / 2.8)$). See footnote relating to WHFe for details. Note the 46.6% factor represents the average Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load.

⁸³⁸ As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average)

⁸³⁹ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

⁸⁴⁰ Based on average of bedroom, dining room, office and living room results from the lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

⁸⁴¹ Ibid

For example, a 13W PAR20 LED is installed in place of a 750 lumen PAR20 incandescent screw-in lamp with medium screw base, diameter >2.5", installed in single family interior location:

$$\begin{aligned}\Delta kW &= ((45 - 13) / 1000) * 0.935 * 1.11 * 0.091 \\ &= 0.0030 \text{ kW}\end{aligned}$$

NATURAL GAS SAVINGS

Heating penalty if Natural Gas heated home, or if heating fuel is unknown.

$$\Delta \text{therms} = - (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * \text{HF} * 0.03412) / \eta \text{Heat}$$

Where:

HF = Heating factor, or percentage of lighting savings that must be replaced by heating system.

= 49%⁸⁴² for interior or unknown location

= 0% for exterior location

0.03412 = Converts kWh to Therms

ηHeat = Average heating system efficiency.

= 0.70⁸⁴³

Other factors as defined above

For example, a 13W PAR20 LED is installed in place of a 750 lumen PAR20 incandescent screw-in lamp with medium screw base, diameter >2.5", installed in single family interior location with gas heating at 70% total efficiency:

$$\begin{aligned}\Delta \text{therms} &= - (((45 - 13) / 1000) * 0.935 * 861 * 0.49 * 0.03412) / 0.70 \\ &= - 0.62 \text{ therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

For those bulbs types exempt from EISA (except for reflectors) the following O&M assumptions should be used: Life of the baseline bulb is assumed to be 1.32 year⁸⁴⁴; baseline replacement cost is assumed to be \$4.0.

For reflectors the life of the baseline bulb and the cost of its replacement is presented in the following table:

⁸⁴² Average result from REMRate modeling of several different configurations and IL locations of homes

⁸⁴³ This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey) In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:

$(0.24 * 0.92) + (0.76 * 0.8) * (1 - 0.15) = 0.70$

⁸⁴⁴ Assuming 1000 hour rated life for incandescent bulb: $1000 / 759 = 1.32$

Lamp Type	Baseline Lamp Life (hours)	Baseline Life (Single Family and in unit Multifamily - 1010 hours)	Baseline Replacement Cost
PAR20, PAR30, PAR38 screw-in lamps	2000	2.0	\$4.00
MR16/PAR16 pin-based lamps	2000	2.0	\$3.00
Recessed downlight luminaries	2000	2.0	\$4.00
Track lights	2000	2.0	\$4.00

For non-exempt EISA bulb types defined above, in order to account for the shift in baseline due to the Energy Independence and Security Act of 2007, an equivalent annual levelized baseline replacement cost over the lifetime of the LED bulb is calculated. The key assumptions used in this calculation are documented below:

Bulb replacement costs assumed in the O&M calculations are provided below⁸⁴⁵.

	EISA Compliant Incandescent /Halogen (Decorative/Globe)	Specialty CFL
2017	\$1.74	N/A
2018	\$1.74	N/A
2019	\$1.74	N/A
2020 & after	N/A	\$3.40 ⁸⁴⁶

Installation Location	Omnidirectional LED Measure Hours	Hours of Use per year ⁸⁴⁷	Measure Life in Years (capped at 10)
Interior and Unknown	15,000	847	10
Exterior	15,000	2475	6.1

The NPV for replacement lamps and annual levelized replacement costs using the societal real discount rate of 0.46% are presented below⁸⁴⁸.

Location	EISA Compliant Bulb Type	NPV of replacement costs for period			Levelized annual replacement cost savings		
		2018	2019	2020	2018	2019	2020
Interior and Unknown	Dimmable Twist, Globe (less than 5" in diameter and > 749 lumens), candle (shapes B, BA, CA > 749 lumens), Candelabra Base Lamps (>1049 lumens), Intermediate Base Lamps (>749 lumens)	\$2.86	\$2.86	\$2.86	\$0.29	\$0.29	\$0.29
Exterior		\$5.96	\$5.96	\$5.96	\$0.61	\$0.61	\$0.61

It is important to note that for cost-effectiveness screening purposes, the O&M cost adjustments should only be applied in cases where the light bulbs are actually in service and so should be multiplied by the appropriate ISR.

⁸⁴⁵ Baseline costs are based on field data collected by CLEAResult and provided by ComEd. See ComEd Pricing Projections 06302016.xlsx for analysis.

⁸⁴⁶ Assumed consistent with LED cost.

⁸⁴⁷ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluations.

⁸⁴⁸ See "Specialty LED EISA compliant O&M Calc.xlsx" for calculation.

MEASURE CODE: RS-LTG-LEDD-V07-180101

REVIEW DEADLINE: 1/1/2020

5.5.7 LED Exit Signs

DESCRIPTION

This measure characterizes the savings associated with installing a Light Emitting Diode (LED) exit sign in place of a fluorescent or incandescent exit sign in a MultiFamily building. Light Emitting Diode exit signs have a string of very small, typically red or green, glowing LEDs arranged in a circle or oval. The LEDs may also be arranged in a line on the side, top or bottom of the exit sign. LED exit signs provide the best balance of safety, low maintenance, and very low energy usage compared to other exit sign technologies.

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient equipment is assumed to be an exit sign illuminated by LEDs.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be a fluorescent or incandescent model.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 16 years⁸⁴⁹.

DEEMED MEASURE COST

The incremental cost for this measure is assumed to be \$30⁸⁵⁰.

LOADSHAPE

Loadshape C53 - Flat

COINCIDENCE FACTOR

The summer peak coincidence factor for this measure is assumed to be 100%⁸⁵¹.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta \text{kWh} = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{HOURS} * \text{WHF}_e$$

Where:

WattsBase = Actual wattage if known, if unknown assume the following:

⁸⁴⁹ 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, "Effective/Remaining Useful Life Values", California Public Utilities Commission, December 16, 2008.

⁸⁵⁰ NYSERDA Deemed Savings Database, Labor cost assumes 25 minutes @ \$18/hr.

⁸⁵¹ Assuming continuous operation of an LED exit sign, the Summer Peak Coincidence Factor is assumed to equal 1.0.

Baseline Type	WattsBase
Incandescent	35W ⁸⁵²
Fluorescent	11W ⁸⁵³
Unknown (e.g. time of sale)	11W

WattsEE = Actual wattage if known, if unknown assume 2W⁸⁵⁴

HOURS = Annual operating hours
= 8766

WHF_e = Waste heat factor for energy; accounts for cooling savings from efficient lighting.
= 1.04⁸⁵⁵ for multi family buildings

Default if replacing incandescent fixture

$$\Delta \text{kWh} = (35 - 2) / 1000 * 8766 * 1.04$$

$$= 301 \text{ kWh}$$

Default if replacing fluorescent fixture

$$\Delta \text{kWh} = (11 - 2) / 1000 * 8766 * 1.04$$

$$= 82 \text{ kWh}$$

HEATING PENALTY

If electric heated building (if heating fuel is unknown assume gas, see Natural Gas section):

$$\Delta \text{kWh}^{856} = - ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{Hours} * \text{HF}) / \eta \text{Heat}$$

Where:

HF = Heating Factor or percentage of light savings that must be heated
= 49%⁸⁵⁷

ηHeat = Efficiency in COP of Heating equipment
= Actual. If not available use: ⁸⁵⁸:

⁸⁵² Based on review of available product.

⁸⁵³ Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, February, 19, 2010

⁸⁵⁴ Efficiency Vermont Technical Reference User Manual (TRM) Measure Savings Algorithms and Cost Assumptions, February, 19, 2010

⁸⁵⁵ The value is estimated at 1.04 (calculated as $1 + (0.45 * (0.27 / 2.8))$). Based on cooling loads decreasing by 27% of the lighting savings (average result from REMRate modeling of several different configurations and IL locations of homes), assuming typical cooling system operating efficiency of 3.1 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm $(-0.02 * \text{SEER}2) + (1.12 * \text{SEER})$ (from Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to $\text{COP} = \text{EER} / 3.412 = 2.8\text{COP}$) and estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average)

⁸⁵⁶ Negative value because this is an increase in heating consumption due to the efficient lighting.

⁸⁵⁷ This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

⁸⁵⁸ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. Note efficiency should include duct

System Type	Age of Equipment	HSPF Estimate	COP _{HEAT} (COP Estimate) = (HSPF/3.413)*0.85
Heat Pump	Before 2006	6.8	1.7
	After 2006 - 2014	7.7	1.92
	2015 on	8.2	2.04
Resistance	N/A	N/A	1.00
Unknown ⁸⁵⁹	N/A	N/A	1.28

For example, a 2.0COP (including duct loss) Heat Pump heated building:

$$\begin{aligned} \text{If incandescent fixture: } \Delta \text{kWh} &= -((35 - 2)/1000 * 8766 * 0.49) / 2 \\ &= -71 \text{ kWh} \end{aligned}$$

$$\begin{aligned} \text{If fluorescent fixture } \Delta \text{kWh} &= -((11 - 2)/1000 * 8766 * 0.49) / 2 \\ &= -19 \text{ kWh} \end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{WHF}_d * \text{CF}$$

Where:

WHF_d = Waste heat factor for demand to account for cooling savings from efficient lighting. The cooling savings are only added to the summer peak savings.

=1.07⁸⁶⁰ for multi family buildings

CF = Summer Peak Coincidence Factor for measure

= 1.0

Default if incandescent fixture

$$\begin{aligned} \Delta \text{kW} &= (35 - 2)/1000 * 1.07 * 1.0 \\ &= 0.035 \text{ kW} \end{aligned}$$

Default if fluorescent fixture

$$\begin{aligned} \Delta \text{kW} &= (11 - 2)/1000 * 1.07 * 1.0 \\ &= 0.0096 \text{ kW} \end{aligned}$$

NATURAL GAS SAVINGS

Heating penalty if Natural Gas heated building, or if heating fuel is unknown.

$$\Delta \text{therms} = -(((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{Hours} * \text{HF} * 0.03412) / \eta_{\text{Heat}}$$

losses. Defaults provided assume 15% duct loss for heat pumps.

⁸⁵⁹ Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see "HC6.9 Space Heating in Midwest Region.xls", using average for East North Central Region. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

⁸⁶⁰ The value is estimated at 1.11 (calculated as $1 + (0.45 * 0.466 / 2.8)$). See footnote relating to WHFe for details. Note the 46.6% factor represents the average Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load.

Where:

HF	= Heating factor, or percentage of lighting savings that must be replaced by heating system. = 49% ⁸⁶¹
0.03412	= Converts kWh to Therms
η_{Heat}	= Average heating system efficiency. = 0.70 ⁸⁶²

Other factors as defined above

Default if incandescent fixture

$$\Delta \text{therms} = - ((35 - 2) / 1000) * 8766 * 0.49 * 0.03412 / 0.70$$

$$= -6.9 \text{ therms}$$

Default if fluorescent fixture

$$\Delta \text{therms} = - ((11 - 2) / 1000) * 8766 * 0.49 * 0.03412 / 0.70$$

$$= -1.9 \text{ therms}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

The annual O&M Cost Adjustment savings should be calculated using the following component costs and lifetimes.

Component	Baseline Measures	
	Cost	Life (yrs)
Lamp	\$7.00 ⁸⁶³	1.37 years ⁸⁶⁴

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REVIEW DEADLINE: 1/1/2019

⁸⁶¹ Average result from REMRate modeling of several different configurations and IL locations of homes

⁸⁶² This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey) In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:

$$(0.24 * 0.92) + (0.76 * 0.8) * (1 - 0.15) = 0.70$$

⁸⁶³ Consistent with assumption for a Standard CFL bulb with an estimated labor cost of \$4.50 (assuming \$18/hour and a task time of 15 minutes).

⁸⁶⁴ Assumes a lamp life of 12,000 hours and 8766 run hours $12000/8766 = 1.37$ years.

5.5.8 LED Screw Based Omnidirectional Bulbs

DESCRIPTION

This characterization provides savings assumptions for LED Screw Based Omnidirectional (e.g. A-Type lamps) lamps within the residential and multifamily sectors. This characterization assumes that the LED lamp or fixture is installed in a residential location. Where the implementation strategy does not allow for the installation location to be known (e.g. an upstream retail program) a deemed split of 95% Residential and 5% Commercial assumptions should be used⁸⁶⁵.

This measure was developed to be applicable to the following program types: TOS, NC, EREP, KITS.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, new lamps must be ENERGY STAR labeled. Note a new ENERGY STAR specification v2.0 becomes effective on 1/2/2017

(https://www.energystar.gov/products/spec/lamps_specification_version_2_0_pd).

DEFINITION OF BASELINE EQUIPMENT

In 2012, Federal legislation stemming from the Energy Independence and Security Act of 2007 (EIAS) will require all general-purpose light bulbs between 40 watts and 100 watts to have ~30% increased efficiency, essentially phasing out standard incandescent technology. In 2012, the 100 w lamp standards apply; in 2013 the 75 w lamp standards will apply, followed by restrictions on the 60 w and 40 w lamps in 2014. Since measures installed under this TRM all occur after 2014, baseline equipment are the values after EISA. These are shown in the baseline table below.

The baseline for the early replacement measure is the existing bulb being replaced.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The deemed measure life is 6.1 years⁸⁶⁶ for exterior application. For all other applications, lifetimes are capped at 10 years⁸⁶⁷.

For early replacement measures, if replacing a halogen or incandescent bulb, the remaining life is assumed to be 333 hours. For CFL's, the remaining life is 3,333 hours⁸⁶⁸.

DEEMED MEASURE COST

The price of LED lamps is falling quickly. Where possible, the actual LED lamp cost should be used and compared to the baseline cost provided below. If the incremental cost is unknown, assume the following⁸⁶⁹:

Year	EISA Compliant Halogen	LED-A	Incremental Cost
2017	\$1.25	\$3.21	\$1.96
2018		\$3.21	\$1.96

⁸⁶⁵ RES v C&I split is based on a weighted (by sales volume) average of ComEd PY6, PY7 and PY8 and Ameren PY5, PY6 and PY8 in store intercept survey results. See 'RESvCI Split_112016.xls'.

⁸⁶⁶ ENERGY STAR v2.0 requires omnidirectional LED bulbs to be rated for at least 15,000 hours. 15000/2475 (exterior hours of use) = 6.1 years.

⁸⁶⁷ Based on recommendation in the Dunskey Energy Consulting, Livingston Energy Innovations and Opinion Dynamics Corporation; NEEP Emerging Technology Research Report, p 6-18.

⁸⁶⁸ Representing a third of the expected lamp lifetime.

⁸⁶⁹ Baseline and LED lamp costs are based on field data collected by CLEAResult and provided by ComEd. See ComEd Pricing Projections 06302016.xlsx for analysis.

Year	EISA Compliant Halogen	LED-A	Incremental Cost
2019		\$3.11	\$1.86

LOADSHAPE

Loadshape R06 – Residential Indoor Lighting

Loadshape R07 – Residential Outdoor Lighting

COINCIDENCE FACTOR

The summer peak coincidence factor is assumed to be 7.1% for Residential and in-unit Multi Family bulbs, 27.3% for exterior bulbs and 8.1% for unknown⁸⁷⁰.

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

$$\Delta kWh = ((Watts_{base} - Watts_{EE}) / 1000) * ISR * (1 - Leakage) * Hours * WHF_e$$

Where:

$Watts_{base}$ = Input wattage of the existing or baseline system. Reference the “LED New and Baseline Assumptions” table for default values.

$Watts_{EE}$ = Actual wattage of LED purchased / installed. If unknown, use default provided below:

LED New and Baseline Assumptions Table

Minimum Lumens	Maximum Lumens	Lumens used to calculate LED Wattage (midpoint)	LED Wattage ⁸⁷¹ (WattsEE)	Baseline 2014-2019 (WattsBase)	Delta Watts 2014-2019 (WattsEE)	Baseline Post EISA 2020 requirement ⁸⁷² (WattsBase)	Delta Watts Post 2020 (WattsEE)
5280	6209	5745	72.9	300.0	227.1	300.0	227.1
3000	5279	4140	52.5	200.0	147.5	200.0	147.5
2601	2999	2800	35.5	150.0	114.5	150.0	114.5
1490	2600	2045	26.0	72.0	46.0	45.4	19.5
1050	1489	1270	16.1	53.0	36.9	28.2	12.1
750	1049	900	11.4	43.0	31.6	20.0	8.6
310	749	530	6.7	29.0	22.3	11.8	5.0
250	309	280	3.5	25.0	21.5	25.0	21.5

ISR = In Service Rate, the percentage of units rebated that are actually in service.

⁸⁷⁰ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

⁸⁷¹ Based on ENERGY STAR V2.0 specs – for omnidirectional <90CRI: 80 lm/W and for omnidirectional ≥90 CRI: 70 lm/W. To weight these two criteria, the ENERGY STAR qualified list was reviewed and found to contain 87.8% lamps <90CRI and 12.2% ≥90CRI.

⁸⁷² Calculated as 45lm/W for all EISA non-exempt bulbs.

Program		Weighted Average 1 st year In Service Rate (ISR)	2 nd year Installations	3 rd year Installations	Final Lifetime In Service Rate
Retail (Time of Sale)		89.9% ⁸⁷³	4.3%	3.7%	98.0% ⁸⁷⁴
Direct Install		96.9% ⁸⁷⁵			
Efficiency Kits ⁸⁷⁶	CFL Distribution ⁸⁷⁷	59%	13%	11%	83%
	School Kits ⁸⁷⁸	61%	13%	11%	86%
	Direct Mail Kits ⁸⁷⁹	66%	14%	12%	93%

Leakage = Adjustment to account for the percentage of program bulbs that move out (and in if deemed appropriate⁸⁸⁰) of the Utility Jurisdiction.

KITS programs = Determined through evaluation

Upstream (TOS) Lighting programs = Determined through evaluation

or use deemed assumptions below⁸⁸¹:

ComEd: 2.1%

Ameren: 13.1%

All other programs = 0

Hours = Average hours of use per year

⁸⁷³ 1st year in service rate is based upon analysis of ComEd PY7 and PY8 and Ameren PY8 intercept data (see 'IL RES Lighting ISR_112016.xls' for more information).

⁸⁷⁴ The 98% Lifetime ISR assumption is based upon the standard CFL measure in the absence of any better reference. This value is based upon review of two evaluations:

'Nexus Market Research, RLW Analytics and GDS Associates study; "New England Residential Lighting Markdown Impact Evaluation, January 20, 2009' and 'KEMA Inc, Feb 2010, Final Evaluation Report; Upstream Lighting Program, Volume 1.' This implies that only 2% of bulbs purchased are never installed. The second and third year installations are based upon Ameren analysis of the Californian KEMA study showing that 54% of future installs occur in year 2 and 46% in year 3. The 2nd and 3rd year installations should be counted as part of those future program year savings.

⁸⁷⁵ Based upon Standard CFL assumption in the absence of better data, and is based upon review of the PY2 and PY3 ComEd Direct Install program surveys. This value includes bulb failures in the 1st year to be consistent with the Commission approval of annualization of savings for first year savings claims. ComEd PY2 All Electric Single Family Home Energy Performance Tune-Up Program Evaluation, Navigant Consulting, December 21, 2010.

⁸⁷⁶ In Service Rates provided are for the bulb within a kit only. Given the significant differences in program design and the level of education provided through Efficiency Kits programs, the evaluators should apply the ISR estimated through evaluations (either past evaluations or the current program year evaluation) of the specific Efficiency Kits program. In cases where program-specific evaluation results for an ISR are unavailable, the default ISR values for Efficiency Kits provide may be used.

⁸⁷⁷ Free bulbs provided without request, with little or no education. Consistent with Standard CFL assumptions.

⁸⁷⁸ Kits provided free to students through school, with education program. Consistent with Standard CFL assumptions.

⁸⁷⁹ Opt-in program to receive kits via mail, with little or no education. Consistent with Standard CFL assumptions.

⁸⁸⁰ Leakage in is only appropriate to credit to IL utility program savings if it is reasonably expected that the IL utility program marketing efforts played an important role in influencing customer to purchase the light bulbs. Furthermore, consideration that such customers might be free riders should be addressed. If leakage in is assessed, efforts should be made to ensure no double counting of savings occurs if the evaluation is estimating both leakage in and spillover savings of light bulbs.

⁸⁸¹ Leakage rate is based upon review of PY6-8 evaluations from ComEd and PY5,6 and 8 for Ameren (see 'IL Leakage Rates_112016.xls' for more information).

Installation Location	Hours ⁸⁸²
Residential and in-unit Multi Family	759
Exterior	2475
Unknown	847

WHFe = Waste heat factor for energy to account for cooling energy savings from efficient lighting

Bulb Location	WHFe
Interior single family or unknown location	1.06 ⁸⁸³
Multi family in unit	1.04 ⁸⁸⁴
Exterior or uncooled location	1.0

Mid Life Baseline Adjustment

During the lifetime of a standard Omnidirectional LED, the baseline incandescent/halogen bulb would need to be replaced multiple times. Since the baseline bulb changes over time (except for <300 and 2600+ lumen lamps) the annual savings claim must be reduced within the life of the measure to account for this baseline shift.

For example, for 60W equivalent bulbs installed in 2018, the full savings (as calculated above in the Algorithm) should be claimed for the first three years, but a reduced annual savings (calculated energy savings above multiplied by the adjustment factor in the table below) claimed for the remainder of the measure life.

Minimum Lumens	Maximum Lumens	LED Wattage (WattsEE)	Delta Watts 2014-2019 (WattsEE)	Delta Watts Post 2020 (WattsEE)	Mid Life adjustment (made from 01/2021) to first year savings
1490	2600	26.0	46.0	19.5	42.3%
1050	1489	16.1	36.9	12.1	32.8%
750	1049	11.4	31.6	8.6	27.1%
310	749	6.7	22.3	5.0	22.6%

⁸⁸² Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluation.

⁸⁸³ The value is estimated at 1.06 (calculated as $1 + (0.66 * (0.27 / 2.8))$). Based on cooling loads decreasing by 27% of the lighting savings (average result from REMRate modeling of several different configurations and IL locations of homes), assuming typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm $(-0.02 * SEER2) + (1.12 * SEER)$ (from Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = $EER/3.412 = 2.8COP$) and 66% of homes in Illinois having central cooling ("Table HC7.9 Air Conditioning in Homes in Midwest Region, Divisions, and States, 2009 from Energy Information Administration", 2009 Residential Energy Consumption Survey)

⁸⁸⁴ As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average)

For example, an 8W LED lamp, 450 lumens, is installed in the interior of a home. The customer purchased the lamp through an upstream program:

$$\begin{aligned}\Delta\text{kWh} &= ((29-6.7/1000) * 847 * 1.06 * 0.899 \\ &= 18.0 \text{ kWh}\end{aligned}$$

This value should be claimed for three years, i.e. 2018-2020, but from 2021 until the end of the measure life for that same bulb, savings should be reduced to $(18.0 * 0.226 =) 4.1 \text{ kWh}$ for the remainder of the measure life. Note these adjustments should be applied to kW and fuel impacts as well.

Note for early replacement measures an additional baseline shift accounting for the replacement of the existing unit with a new baseline lamp should be accounted for.

DEFERRED INSTALLS

As presented above, the characterization assumes that a percentage of bulbs purchased are not installed until Year 2 and Year 3 (see ISR assumption above). The Illinois Technical Advisory Committee has determined the following methodology for calculating the savings of these future installs.

Year 1 (Purchase Year) installs: Characterized using assumptions provided above or evaluated assumptions if available.

Year 2 and 3 installs: Characterized using delta watts assumption and hours of use from the Install Year i.e. the actual deemed (or evaluated if available) assumptions active in Year 2 and 3 should be applied.

The NTG factor for the Purchase Year should be applied.

Using the example from above, for an 8W LED, 450 Lumens purchased for the interior of a residential homes through an upstream program.

$$\begin{aligned}\Delta\text{kWh}_{1\text{st year installs}} &= ((29-6.7)/1000)*847*1.06*0.899 \\ &= 18.0 \text{ kWh}\end{aligned}$$

$$\begin{aligned}\Delta\text{kWh}_{2\text{nd year installs}} &= ((29-6.7)/1000)*847*1.06*0.043 \\ &= 0.9 \text{ kWh}\end{aligned}$$

Note: Here we assume no change in hours assumption. NTG value from Purchase year applied.

HEATING PENALTY

If electric heated home (if heating fuel is unknown assume gas, see Natural Gas section):

$$\Delta\text{kWh}^{885} = - (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * \text{HF}) / \eta_{\text{Heat}}$$

Where:

HF = Heating Factor or percentage of light savings that must be heated
 = 49%⁸⁸⁶ for interior or unknown location
 = 0% for exterior or unheated location

⁸⁸⁵ Negative value because this is an increase in heating consumption due to the efficient lighting.

⁸⁸⁶ This means that heating loads increase by 49% of the lighting savings. This is based on the average result from REMRate modeling of several different configurations and IL locations of homes.

η_{Heat} = Efficiency in COP of Heating equipment
= actual. If not available use⁸⁸⁷:

System Type	Age of Equipment	HSPF Estimate	COP ^{HEAT} (COP Estimate) = (HSPF/3.413)*0.85
Heat Pump	Before 2006	6.8	1.7
	After 2006 - 2014	7.7	1.92
	2015 on	8.2	2.04
Resistance	N/A	N/A	1.00
Unknown ⁸⁸⁸	N/A	N/A	1.28

Using the same 8 W LED that is installed in home with 2.0 COP Heat Pump (including duct loss):

$$\Delta \text{kWh}_{1\text{st year}} = - ((29-6.7) / 1000) * 0.899 * 759 * 0.49 / 2.0$$

$$= - 3.7 \text{ kWh}$$

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year. The appropriate baseline shift adjustment should then be applied to all installs.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta \text{kW} = ((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{WHFd} * \text{CF}$$

Where:

WHFd = Waste heat factor for demand to account for cooling savings from efficient lighting.

Bulb Location	WHFd
Interior single family or unknown location	1.11 ⁸⁸⁹
Multi family in unit	1.07 ⁸⁹⁰
Exterior or uncooled location	1.0

CF = Summer Peak Coincidence Factor for measure.

Bulb Location	CF ⁸⁹¹
Interior single family or unknown location or Multi family in unit	7.1%

⁸⁸⁷ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. Note efficiency should include duct losses. Defaults provided assume 15% duct loss for heat pumps.

⁸⁸⁸ Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see "HC6.9 Space Heating in Midwest Region.xls", using average for East North Central Region. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

⁸⁸⁹ The value is estimated at 1.11 (calculated as $1 + (0.66 * 0.466 / 2.8)$). See footnote relating to WHFe for details. Note the 46.6% factor represents the average Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load.

⁸⁹⁰ As above but using estimate of 45% of multi family buildings in Illinois having central cooling (based on data from "Table HC7.1 Air Conditioning in U.S. Homes, By Housing Unit Type, 2009" which is for the whole of the US, scaled to IL air conditioning prevalence compared to US average)

⁸⁹¹ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluations.

Bulb Location	CF ⁸⁹¹
Exterior	27.3%
Unknown	8.1%

Other factors as defined above

For the same 8 W LED that is installed in a single family interior location, the demand savings are:

$$\begin{aligned}\Delta kW &= ((29-6.7) / 1000) * 0.899 * 1.11 * 0.071 \\ &= 0.0016 \text{ kW}\end{aligned}$$

Second and third year install savings should be calculated using the appropriate ISR and the delta watts and hours from the install year. The appropriate baseline shift adjustment should then be applied to all installs.

NATURAL GAS SAVINGS

Heating penalty if Natural Gas heated home, or if heating fuel is unknown.

$$\Delta \text{Therms} = - (((\text{WattsBase} - \text{WattsEE}) / 1000) * \text{ISR} * \text{Hours} * \text{HF} * 0.03412) / \eta \text{Heat}$$

Where:

HF = Heating factor, or percentage of lighting savings that must be replaced by heating system.

= 49%⁸⁹² for interior or unknown location

= 0% for exterior location

0.03412 = Converts kWh to Therms

ηHeat = Average heating system efficiency.

= 0.70⁸⁹³

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

Bulb replacement costs assumed in the O&M calculations are provided below⁸⁹⁴.

	Std Inc.	EISA Compliant Halogen	CFL	LED-A
2017	\$0.43	\$1.25	N/A	\$3.21

⁸⁹² Average result from REMRate modeling of several different configurations and IL locations of homes

⁸⁹³ This has been estimated assuming that natural gas central furnace heating is typical for Illinois residences (66% of Illinois homes have a Natural Gas Furnace (based on Energy Information Administration, 2009 Residential Energy Consumption Survey) In 2000, 24% of furnaces purchased in Illinois were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 10 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows:

$$(0.24 * 0.92) + (0.76 * 0.8) * (1 - 0.15) = 0.70$$

⁸⁹⁴ Baseline and LED lamp costs are based on field data collected by CLEAResult and provided by ComEd. See ComEd Pricing Projections 06302016.xlsx for analysis.

	Std Inc.	EISA Compliant Halogen	CFL	LED-A
2018	\$0.43	\$1.25	N/A	\$3.21
2019	\$0.43	\$1.25	N/A	\$3.11
2020 & after	\$0.43	N/A	\$2.45	\$2.70

In order to account for the shift in baseline due to the Energy Independence and Security Act of 2007, an equivalent annual levelized baseline replacement cost over the lifetime of the LED bulb is calculated. The key assumptions used in this calculation are documented below:

Installation Location	Omnidirectional LED Measure Hours	Hours of Use per year ⁸⁹⁵	Measure Life in Years (capped at 10)
Residential and in-unit Multi Family	15,000	759	10
Exterior	15,000	2475	6.1
Unknown	15,000	847	10

The NPV for replacement lamps and annual levelized replacement costs using the societal real discount rate of 0.46% are presented below⁸⁹⁶. It is important to note that for cost-effectiveness screening purposes, the O&M cost adjustments should only be applied in cases where the light bulbs are actually in service and so should be multiplied by the appropriate ISR:

Location	Lumen Level	NPV of replacement costs for period			Levelized annual replacement cost savings		
		2018	2019	2020	2018	2019	2020
Residential and in-unit Multi Family	Lumens <310 or >2600 (non-EISA compliant)	\$2.86	\$2.86	\$2.86	\$0.29	\$0.29	\$0.29
	Lumens ≥ 310 and ≤ 2600 (EISA compliant)	\$3.14	\$2.38	\$1.63	\$0.32	\$0.24	\$0.17
Exterior	Lumens <310 or >2600 (non-EISA compliant)	\$5.96	\$5.96	\$5.96	\$0.61	\$0.61	\$0.61
	Lumens ≥ 310 and ≤ 2600 (EISA compliant)	\$9.79	\$7.34	\$4.87	\$1.00	\$0.75	\$0.50
Unknown	Lumens <310 or >2600 (non-EISA compliant)	\$3.19	\$3.19	\$3.19	\$0.33	\$0.33	\$0.33
	Lumens ≥ 310 and ≤ 2600 (EISA compliant)	\$3.50	\$2.66	\$1.82	\$0.36	\$0.27	\$0.19

Note incandescent lamps in lumen range <310 and >2600 are exempt from EISA. For halogen bulbs, we assume the same replacement cycle as incandescent bulbs.⁸⁹⁷ The replacement cycle is based on the location of the lamp and varies based on the hours of use for that location. Both incandescent and halogen lamps are assumed to last for 1,000 hours before needing replacement.

⁸⁹⁵ Based on lighting logger study conducted as part of the PY5/6 ComEd Residential Lighting Program evaluations.

⁸⁹⁶ See "LED TRM Examples_012017.xls" for calculation.

⁸⁹⁷ The manufacturers of the new minimally compliant EISA Halogens are using regular incandescent lamps with halogen fill gas rather than halogen infrared to meet the standard and so the component rated life is equal to the standard incandescent.

MEASURE CODE: RS-LTG-LEDA-V05-180101

REVIEW DEADLINE: 1/1/2020

5.6 Shell End Use

5.6.1 Air Sealing

DESCRIPTION

Thermal shell air leaks are sealed through strategic use and location of air-tight materials. Leaks are detected and leakage rates measured with the assistance of a blower-door. The algorithm for this measure can be used when the program implementation does not allow for more detailed forecasting through the use of residential modeling software.

Prescriptive savings are provided for use only where a blower door test is not possible (for example in large multi family buildings).

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

Air sealing materials and diagnostic testing should meet all eligibility program qualification criteria. The initial and final tested leakage rates should be performed in such a manner that the identified reductions can be properly discerned, particularly in situations wherein multiple building envelope measures may be implemented simultaneously.

DEFINITION OF BASELINE EQUIPMENT

The existing air leakage should be determined through approved and appropriate test methods using a blower door. The baseline condition of a building upon first inspection significantly impacts the opportunity for cost-effective energy savings through air-sealing.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years.⁸⁹⁸

DEEMED MEASURE COST

The actual capital cost for this measure should be used in screening.

LOADSHAPE

Loadshape R08 - Residential Cooling

Loadshape R09 - Residential Electric Space Heat

Loadshape R10 - Residential Electric Heating and Cooling

COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

$$CF_{SSP} = \text{Summer System Peak Coincidence Factor for Central A/C (during utility peak hour)}$$

⁸⁹⁸ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007

$$= 68\%^{899}$$

CF_{SSP} = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)

$$= 72\%^{900}$$

CF_{PJM} = PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)

$$= 46.6\%^{901}$$

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Methodology 1: Blower Door Test

Preferred methodology unless blower door testing is not possible.

$$\Delta kWh = \Delta kWh_{cooling} + \Delta kWh_{heating}$$

Where:

$\Delta kWh_{cooling}$ = If central cooling, reduction in annual cooling requirement due to air sealing

$$= [(((CFM50_{existing} - CFM50_{new})/N_{cool}) * 60 * 24 * CDD * DUA * 0.018) / (1000 * \eta_{Cool})] * LM$$

CFM50_{existing} = Infiltration at 50 Pascals as measured by blower door before air sealing.

= Actual

CFM50_{new} = Infiltration at 50 Pascals as measured by blower door after air sealing.

= Actual

N_{cool} = Conversion factor from leakage at 50 Pascal to leakage at natural conditions

=Dependent on location and number of stories:⁹⁰²

Climate Zone (City based upon)	N _{cool} (by # of stories)			
	1	1.5	2	3
1 (Rockford)	39.5	35.0	32.1	28.4
2 (Chicago)	38.9	34.4	31.6	28.0
3 (Springfield)	41.2	36.5	33.4	29.6
4 (St Louis, MO)	40.4	35.8	32.9	29.1
5 (Paducah, KY)	43.6	38.6	35.4	31.3

⁸⁹⁹ Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

⁹⁰⁰ Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

⁹⁰¹ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

⁹⁰² N-factor is used to convert 50-pascal blower door air flows to natural air flows and is dependent on geographic location and # of stories. These were developed by applying the LBNL infiltration model (see LBNL paper 21040, *Exegisis of Proposed ASHRAE Standard 119: Air Leakage Performance for Detached Single-Family Residential Buildings*; Sherman, 1986; page v-vi, Appendix page 7-9) to the reported wind speeds and outdoor temperatures provided by the NRDC 30 year climate normals. For more information see Bruce Harley, CLEAResult "Infiltration Factor Calculations Methodology.doc".

60 * 24 = Converts Cubic Feet per Minute to Cubic Feet per Day

CDD = Cooling Degree Days
= Dependent on location⁹⁰³:

Climate Zone (City based upon)	CDD 65
1 (Rockford)	820
2 (Chicago)	842
3 (Springfield)	1,108
4 (Belleville)	1,570
5 (Marion)	1,370

DUA = Discretionary Use Adjustment (reflects the fact that people do not always operate their AC when conditions may call for it).

= 0.75⁹⁰⁴

0.018 = Specific Heat Capacity of Air (Btu/ft³*°F)

1000 = Converts Btu to kBtu

η_{Cool} = Efficiency (SEER) of Air Conditioning equipment (kBtu/kWh)

= Actual (where it is possible to measure or reasonably estimate). If unknown assume the following⁹⁰⁵:

Age of Equipment	SEER Estimate
Before 2006	10
2006 - 2014	13
Central AC After 1/1/2015	13
Heat Pump After 1/1/2015	14

LM = Latent multiplier to account for latent cooling demand⁹⁰⁶

Climate Zone (City based upon)	LM
1 (Rockford)	3.3
2 (Chicago)	3.2
3 (Springfield)	3.7
4 (St Louis, MO)	3.6
5 (Paducah, KY)	3.7

$\Delta kWh_{heating}$ = If electric heat (resistance or heat pump), reduction in annual electric heating due to

⁹⁰³ National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 65°F.

⁹⁰⁴ This factor's source is: Energy Center of Wisconsin, May 2008 metering study; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research", p31.

⁹⁰⁵ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

⁹⁰⁶ Derived by calculating the sensible and total loads in each hour. For more information see Bruce Harley, CLEARResult "Infiltration Factor Calculations Methodology.doc".

air sealing

$$= (((CFM50_{existing} - CFM50_{new}) / N_{heat}) * 60 * 24 * HDD * 0.018) / (\eta_{Heat} * 3,412)$$

N_{heat} = Conversion factor from leakage at 50 Pascal to leakage at natural conditions

= Based on climate zone, building height and exposure level:⁹⁰⁷

Climate Zone (City based upon)	N _{heat} (by # of stories)			
	1	1.5	2	3
1 (Rockford)	23.8	21.1	19.3	17.1
2 (Chicago)	23.9	21.1	19.4	17.2
3 (Springfield)	24.2	21.5	19.7	17.4
4 (St Louis, MO)	25.4	22.5	20.7	18.3
5 (Paducah, KY)	27.8	24.6	22.6	20.0

HDD = Heating Degree Days

= Dependent on location:⁹⁰⁸

Climate Zone (City based upon)	HDD 60
1 (Rockford)	5,352
2 (Chicago)	5,113
3 (Springfield)	4,379
4 (Belleville)	3,378
5 (Marion)	3,438

η_{Heat} = Efficiency of heating system

= Actual. If not available refer to default table below⁹⁰⁹:

System Type	Age of Equipment	HSPF Estimate	η_{Heat} (Effective COP Estimate)= (HSPF/3.413)*0.85
Heat Pump	Before 2006	6.8	1.7
	2006 - 2014	7.7	1.92
	2015 on	8.2	2.04
Resistance	N/A	N/A	1

3412 = Converts Btu to kWh

⁹⁰⁷ N-factor is used to convert 50-pascal blower door air flows to natural air flows and is dependent on geographic location and # of stories. These were developed by applying the LBNL infiltration model (see LBNL paper 21040, *Exegisis of Proposed ASHRAE Standard 119: Air Leakage Performance for Detached Single-Family Residential Buildings*; Sherman, 1986; page v-vi, Appendix page 7-9) to the reported wind speeds and outdoor temperatures provided by the NRDC 30 year climate normals. For more information see Bruce Harley, CLEAResult "Infiltration Factor Calculations Methodology.doc".

⁹⁰⁸ National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 60°F.

⁹⁰⁹ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.

For example, a 2 story single family home in Chicago with 10.5 SEER central cooling and a heat pump with COP of 2 (1.92 including distribution losses), has pre and post blower door test results of 3,400 and 2,250:

$$\begin{aligned}
 \Delta kWh &= \Delta kWh_{\text{cooling}} + \Delta kWh_{\text{heating}} \\
 &= [(((3,400 - 2,250) / 31.6) * 60 * 24 * 842 * 0.75 * 0.018) / (1000 * 10.5)) * 3.2] + [((3,400 - 2,250) / 19.4) * 60 * 24 * 5113 * 0.018 / (1.92 * 3,412)] \\
 &= 182 + 1199 \\
 &= 1,381 \text{ kWh}
 \end{aligned}$$

$\Delta kWh_{\text{heating}}$ = If gas furnace heat, kWh savings for reduction in fan run time

$$= \Delta \text{Therms} * F_e * 29.3$$

F_e = Furnace Fan energy consumption as a percentage of annual fuel consumption

$$= 3.14\%^{910}$$

29.3 = kWh per therm

For example, a well shielded, 2 story single family home in Chicago with a gas furnace with system efficiency of 70%, has pre and post blower door test results of 3,400 and 2,250 (see therm calculation in Natural Gas Savings section:

$$\begin{aligned}
 \Delta kWh &= 109.1 * 0.0314 * 29.3 \\
 &= 100 \text{ kWh}
 \end{aligned}$$

Methodology 2: Prescriptive Infiltration Reduction Measures⁹¹¹

Savings shall only be calculated via Methodology 2 if a blower door test is not feasible. Cooling savings are not quantified using Methodology 2.

$$\Delta kWh_{\text{heating}} = (\Delta kWh_{\text{gasket}} * n_{\text{gasket}} + \Delta kWh_{\text{sweep}} * n_{\text{sweep}} + \Delta kWh_{\text{sealing}} * If_{\text{sealing}} + \Delta kWh_{\text{wx}} * If_{\text{wx}}) * \text{ADJ}_{\text{RxAirsealing}}$$

Where:

$\Delta kWh_{\text{gasket}}$ = Annual kWh savings from installation of air sealing gasket on an electric outlet

Climate Zone (City based upon)	$\Delta kWh_{\text{gasket}} / \text{gasket}$	
	Electric Resistance	Heat Pump
1 (Rockford)	10.5	5.3
2 (Chicago)	10.2	5.1
3 (Springfield)	8.8	4.4

⁹¹⁰ F_e is not one of the AHRI certified ratings provided for residential furnaces, but can be reasonably estimated from a calculation based on the certified values for fuel energy (E_f in MMBtu/yr) and E_{ae} (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the Energy Star version 3 criteria for 2% F_e . See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.

⁹¹¹ Prescriptive savings are based upon "Evaluation of the Weatherization Residential Assistance Partnership and Helps Programs (WRAP/Helps)." Middletown, CT: KEMA, 2010. Accessed July 30, 2015, (<http://www.energizect.com/sites/default/files/Final%20WRAP%20%20Helps%20Report.pdf>) and adjusted for relative HDD of Bridgeport/Hartford CT with the IL climate zones. See 'Rx Airsealing HDD adjustment.xls' for more information.

Climate Zone (City based upon)	$\Delta kWh_{\text{gasket}} / \text{gasket}$	
	Electric Resistance	Heat Pump
4 (Belleville)	7.0	3.5
5 (Marion)	7.2	3.6

n_{gasket} = Number of gaskets installed

$\Delta kWh_{\text{sweep}}$ = Annual kWh savings from installation of door sweep

Climate Zone (City based upon)	$\Delta kWh_{\text{sweep}} / \text{sweep}$	
	Electric Resistance	Heat Pump
1 (Rockford)	202.4	101.2
2 (Chicago)	195.3	97.6
3 (Springfield)	169.3	84.7
4 (Belleville)	134.9	67.5
5 (Marion)	137.9	68.9

n_{sweep} = Number of sweeps installed

$\Delta kWh_{\text{sealing}}$ = Annual kWh savings from foot of caulking, sealing, or polyethylene tape

Climate Zone (City based upon)	$\Delta kWh_{\text{sealing}} / \text{ft}$	
	Electric Resistance	Heat Pump
1 (Rockford)	11.6	5.8
2 (Chicago)	11.2	5.6
3 (Springfield)	9.7	4.8
4 (Belleville)	7.7	3.9
5 (Marion)	7.9	3.9

l_{sealing} = linear feet of caulking, sealing, or polyethylene tape

ΔkWh_{wx} = Annual kWh savings from window weatherstripping or door weatherstripping

Climate Zone (City based upon)	$\Delta kWh_{\text{wx}} / \text{ft}$	
	Electric Resistance	Heat Pump
1 (Rockford)	13.5	6.7
2 (Chicago)	13.0	6.5
3 (Springfield)	11.3	5.6
4 (Belleville)	9.0	4.5
5 (Marion)	9.2	4.6

l_{wx} = Linear feet of window weatherstripping or door weatherstripping

$ADJ_{\text{RxAirsealing}}$ = Adjustment for air sealing savings to account for prescriptive estimates overclaiming savings⁹¹².

= 80%

⁹¹² Though we do not have a specific evaluation to point to, modeled savings have often been found to overclaim. Further VEIC reviewed these deemed estimates and consider them to likely be a high estimate. As such an 80% adjustment is applied, and this could be further refined with future evaluations.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = (\Delta kWh_{\text{cooling}} / FLH_{\text{cooling}}) * CF$$

Where:

FLH_{cooling} = Full load hours of air conditioning

= Dependent on location⁹¹³:

Climate Zone (City based upon)	Single Family	Multifamily
1 (Rockford)	512	467
2 (Chicago)	570	506
3 (Springfield)	730	663
4 (Belleville)	1,035	940
5 (Marion)	903	820

CF_{SSP} = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)

= 68%⁹¹⁴

CF_{SSP} = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)

= 72%⁹¹⁵

CF_{PJM} = PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)

= 46.6%⁹¹⁶

Other factors as defined above

For example, a well shielded, 2 story single family home in Chicago with 10.5 SEER central cooling and a heat pump with COP of 2.0, has pre and post blower door test results of 3,400 and 2,250:

$$\Delta kW_{SSP} = 182 / 570 * 0.68$$

$$= 0.22 \text{ kW}$$

$$\Delta kW_{PJM} = 182 / 570 * 0.466$$

$$= 0.15 \text{ kW}$$

NATURAL GAS SAVINGS**Methodology 1: Blower Door Test**

Preferred methodology unless blower door testing is not possible.

If Natural Gas heating:

⁹¹³ Full load hours for Chicago, Moline and Rockford are provided in "Final Evaluation Report: Central Air Conditioning Efficiency Services (CACES), 2010, Navigant Consulting", p.33. An average FLH/Cooling Degree Day (from NCDC) ratio was calculated for these locations and applied to the CDD of the other locations in order to estimate FLH.

⁹¹⁴ Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

⁹¹⁵ Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

⁹¹⁶ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

$$\Delta\text{Therms} = (((\text{CFM50}_{\text{existing}} - \text{CFM50}_{\text{new}}) / \text{N}_{\text{heat}}) * 60 * 24 * \text{HDD} * 0.018) / (\eta_{\text{Heat}} * 100,000)$$

Where:

N_{heat} = Conversion factor from leakage at 50 Pascal to leakage at natural conditions
 = Based on climate zone and building height⁹¹⁷

Climate Zone (City based upon)	N _{heat} (by # of stories)			
	1	1.5	2	3
1 (Rockford)	23.8	21.1	19.3	17.1
2 (Chicago)	23.9	21.1	19.4	17.2
3 (Springfield)	24.2	21.5	19.7	17.4
4 (St Louis, MO)	25.4	22.5	20.7	18.3
5 (Paducah, KY)	27.8	24.6	22.6	20.0

HDD = Heating Degree Days
 = dependent on location⁹¹⁸:

Climate Zone (City based upon)	HDD 60
1 (Rockford)	5,352
2 (Chicago)	5,113
3 (Springfield)	4,379
4 (Belleville)	3,378
5 (Marion)	3,438

η_{Heat} = Efficiency of heating system
 = Equipment efficiency * distribution efficiency
 = Actual⁹¹⁹. If not available use 72%⁹²⁰.

Other factors as defined above

For example, a 2 story single family home in Chicago with a gas furnace with system efficiency of 70%, has pre and post blower door test results of 3,400 and 2,250:

$$\begin{aligned}\Delta\text{Therms} &= ((3,400 - 2,250) / 19.4) * 60 * 24 * 5113 * 0.018 / (0.72 * 100,000) \\ &= 109.1 \text{ therms}\end{aligned}$$

⁹¹⁷ N-factor is used to convert 50-pascal blower door air flows to natural air flows and is dependent on geographic location and # of stories. These were developed by applying the LBNL infiltration model (see LBNL paper 21040, *Exegisis of Proposed ASHRAE Standard 119: Air Leakage Performance for Detached Single-Family Residential Buildings*; Sherman, 1986; page v-vi, Appendix page 7-9) to the reported wind speeds and outdoor temperatures provided by the NRDC 30 year climate normals. For more information see Bruce Harley, CLEAResult "Infiltration Factor Calculations Methodology.doc".

⁹¹⁸ National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 60°F, consistent with the findings of Belzer and Cort, Pacific Northwest National Laboratory in "Statistical Analysis of Historical State-Level Residential Energy Consumption Trends," 2004..

⁹¹⁹ Ideally, the System Efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. The Distribution Efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute: (<http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf>) or by performing duct blaster testing.

⁹²⁰ Based on average Nicor PY4 nameplate efficiencies derated by 15% for distribution losses.

Methodology 2: Prescriptive Infiltration Reduction Measures⁹²¹

Savings shall only be calculated via Methodology 2 if a blower door test is not feasible.

$$\Delta\text{therms} = (\Delta\text{therms}_{\text{gasket}} * n_{\text{gasket}} + \Delta\text{therms}_{\text{sweep}} * n_{\text{sweep}} + \Delta\text{therms}_{\text{sealing}} * l_{\text{sealing}} + \Delta\text{therms}_{\text{SWX}} * l_{\text{WX}}) * \text{ADJ}_{\text{RxAirsealing}}$$

Where:

$\Delta\text{therms}_{\text{gasket}}$ = Annual therm savings from installation of air sealing gasket on an electric outlet

Climate Zone (City based upon)	$\Delta\text{therms}_{\text{gasket}} / \text{gasket}$ Gas Heat
1 (Rockford)	0.49
2 (Chicago)	0.47
3 (Springfield)	0.41
4 (Belleville)	0.33
5 (Marion)	0.33

n_{gasket} = Number of gaskets installed

$\Delta\text{therms}_{\text{sweep}}$ = Annual therm savings from installation of door sweep

Climate Zone (City based upon)	$\Delta\text{therms}_{\text{sweep}} / \text{sweep}$ Gas Heat
1 (Rockford)	9.46
2 (Chicago)	9.13
3 (Springfield)	7.92
4 (Belleville)	6.31
5 (Marion)	6.45

n_{sweep} = Number of sweeps installed

$\Delta\text{therms}_{\text{sealing}}$ = Annual therm savings from foot of caulking, sealing, or polyethylene tape

Climate Zone (City based upon)	$\Delta\text{therms}_{\text{sealing}} / \text{ft}$ Gas Heat
1 (Rockford)	0.54
2 (Chicago)	0.52
3 (Springfield)	0.45
4 (Belleville)	0.36
5 (Marion)	0.37

l_{sealing} = linear feet of caulking, sealing, or polyethylene tape

$\Delta\text{therms}_{\text{SWX}}$ = Annual therm savings from window weatherstripping or door weatherstripping

Climate Zone (City based upon)	$\Delta\text{therms}_{\text{SWX}} / \text{ft}$ Gas Heat
1 (Rockford)	0.63

⁹²¹ Prescriptive savings are based upon "Evaluation of the Weatherization Residential Assistance Partnership and Helps Programs (WRAP/Helps)." Middletown, CT: KEMA, 2010. Accessed July 30, 2015, (<http://www.energizect.com/sites/default/files/Final%20WRAP%20%20Helps%20Report.pdf>) and adjusted for relative HDD of Bridgeport/Hartford CT with the IL climate zones. See 'Rx Airsealing HDD adjustment.xls' for more information.

Climate Zone (City based upon)	Δ therms _{sx} / ft Gas Heat
2 (Chicago)	0.61
3 (Springfield)	0.53
4 (Belleville)	0.42
5 (Marion)	0.43

$l_{f_{wx}}$ = Linear feet of window weatherstripping or door weatherstripping

$ADJ_{RxAirsealing}$ = Adjustment for air sealing savings to account for prescriptive estimates overclaiming savings⁹²².

= 80%

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-SHL-AIRS-V06-180101

REVIEW DEADLINE: 1/1/2020

⁹²² Though we do not have a specific evaluation to point to, modeled savings have often been found to overclaim. Further VEIC reviewed these deemed estimates and consider them to likely be a high estimate. As such an 80% adjustment is applied, and this could be further refined with future evaluations.

5.6.2 Basement Sidewall Insulation

DESCRIPTION

Insulation is added to a basement or crawl space. Insulation added above ground in conditioned space is modeled the same as wall insulation. Below ground insulation is adjusted with an approximation of the thermal resistance of the ground. Insulation in unconditioned spaces is modeled by reducing the degree days to reflect the smaller but non-zero contribution to heating and cooling load. Cooling savings only consider above grade insulation, as below grade has little temperature difference during the cooling season.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

This measure requires a member of the implementation staff or a participating contractor to evaluate the pre and post R-values and measure surface areas. The requirements for participation in the program will be defined by the utilities.

DEFINITION OF BASELINE EQUIPMENT

The existing condition will be evaluated by implementation staff or a participating contractor and is likely to be no basement wall or ceiling insulation.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 25 years.⁹²³

DEEMED MEASURE COST

The actual installed cost for this measure should be used in screening.

DEEMED O&M COST ADJUSTMENTS

N/A

LOADSHAPE

Loadshape R08 - Residential Cooling

Loadshape R09 - Residential Electric Space Heat

Loadshape R10 - Residential Electric Heating and Cooling

COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

$$\begin{aligned} CF_{SSP} &= \text{Summer System Peak Coincidence Factor for Central A/C (during utility peak hour)} \\ &= 68\%^{924} \end{aligned}$$

⁹²³ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007

⁹²⁴ Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

CF_{SSP} = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)
 = 72%⁹²⁵

CF_{PJM} = PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)
 = 46.6%⁹²⁶

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Where available savings from shell insulation measures should be determined through a custom analysis. When that is not feasible for the program the following engineering algorithms can be used with the inclusion of an adjustment factor to de-rate the heating savings.

$$\Delta kWh = (\Delta kWh_{cooling} + \Delta kWh_{heating})$$

Where:

$\Delta kWh_{cooling}$ = If central cooling, reduction in annual cooling requirement due to insulation

$$= \left(\left(\left(\frac{1}{R_{old_AG}} - \frac{1}{R_{added} + R_{old_AG}} \right) * L_{basement_wall_total} * H_{basement_wall_AG} * (1 - Framing_factor) \right) * 24 * CDD * DUA \right) / (1000 * \eta_{Cool})) * ADJ_{BasementCool}$$

R_{added} = R-value of additional spray foam, rigid foam, or cavity insulation.

R_{old_AG} = R-value value of foundation wall above grade.

= Actual, if unknown assume 1.0⁹²⁷

L_{basement_wall_total} = Length of basement wall around the entire insulated perimeter (ft)

H_{basement_wall_AG} = Height of insulated basement wall above grade (ft)

Framing_factor = Adjustment to account for area of framing when cavity insulation is used

= 0% if Spray Foam or External Rigid Foam

= 25% if studs and cavity insulation⁹²⁸

24 = Converts hours to days

CDD = Cooling Degree Days

= Dependent on location and whether basement is conditioned:⁹²⁹

⁹²⁵ Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

⁹²⁶ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

⁹²⁷ ORNL Builders Foundation Handbook, crawl space data from Table 5-5: Initial Effective R-values for Uninsulated Foundation System and Adjacent Soil, 1991, http://www.ornl.gov/sci/roofs+walls/foundation/ORNL_CON-295.pdf

⁹²⁸ ASHRAE, 2001, "Characterization of Framing Factors for New Low-Rise Residential Building Envelopes (904-RP)," Table 7.1

⁹²⁹ National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 65°F. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

Climate Zone (City based upon)	Conditioned CDD 65	Unconditioned CDD 65 ⁹³⁰
1 (Rockford)	820	263
2 (Chicago)	842	281
3 (Springfield)	1,108	436
4 (Belleville)	1,570	538
5 (Marion)	1,370	570
Weighted Average ⁹³¹	947	325

DUA = Discretionary Use Adjustment (reflects the fact that people do not always operate their AC when conditions may call for it).

= 0.75⁹³²

1000 = Converts Btu to kWh

η_{Cool} = Seasonal Energy Efficiency Ratio of cooling system (kBtu/kWh)

= Actual (where it is possible to measure or reasonably estimate). If unknown assume the following:⁹³³

Age of Equipment	η_{Cool} Estimate
Before 2006	10
2006 - 2014	13
Central AC After 1/1/2015	13
Heat Pump After 1/1/2015	14

$ADJ_{BasementCool}$ = Adjustment for cooling savings from basement wall insulation to account for prescriptive engineering algorithms overclaiming savings⁹³⁴.
= 80%

$\Delta kWh_{heating}$ = If electric heat (resistance or heat pump), reduction in annual electric heating due to insulation

$$= (((1/R_{old_AG} - 1/(R_{added} + R_{old_AG})) * L_{basement_wall_total} * H_{basement_wall_AG} * (1 - Framing_factor)) + ((1/(R_{old_BG} - 1/(R_{added} + R_{old_BG})) * L_{basement_wall_total} * (H_{basement_wall_total} - H_{basement_wall_AG}) * (1 - Framing_factor))) * 24 * HDD) / (3,412 * \eta_{Heat})) * ADJ_{BasementHeat}$$

Where

R_{old_BG} = R-value value of foundation wall below grade (including thermal resistance of

⁹³⁰ Five year average cooling degree days with 75F base temp from DegreeDays.net were used in this table because the 30 year climate normals from NCDC used elsewhere are not available at base temps above 72F.

⁹³¹ Weighted based on number of occupied residential housing units in each zone.

⁹³² This factor's source is: Energy Center of Wisconsin, May 2008 metering study; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research", p31.

⁹³³ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

⁹³⁴ As demonstrated in two years of metering evaluation by Opinion Dynamics, see Memo "Results for AIC PY6 HPwES Billing Analysis", dated February 20, 2015. TAC negotiated adjustment factor is 80%.

the earth)⁹³⁵

= dependent on depth of foundation ($H_{\text{basement_wall_total}} - H_{\text{basement_wall_AG}}$):

= Actual R-value of wall plus average earth R-value by depth in table below

Below Grade R-value									
Depth below grade (ft)	0	1	2	3	4	5	6	7	8
Earth R-value (°F-ft ² -h/Btu)	2.44	4.50	6.30	8.40	10.44	12.66	14.49	17.00	20.00
Average Earth R-value (°F-ft ² -h/Btu)	2.44	3.47	4.41	5.41	6.42	7.46	8.46	9.53	10.69
Total BG R-value (earth + R-1.0 foundation) default	3.44	4.47	5.41	6.41	7.42	8.46	9.46	10.53	11.69

$H_{\text{basement_wall_total}}$ = Total height of basement wall (ft)

HDD = Heating Degree Days

= dependent on location and whether basement is conditioned:⁹³⁶

Climate Zone (City based upon)	Conditioned HDD 60	Unconditioned HDD 50
1 (Rockford)	5,352	3,322
2 (Chicago)	5,113	3,079
3 (Springfield)	4,379	2,550
4 (Belleville)	3,378	1,789
5 (Marion)	3,438	1,796
Weighted Average ⁹³⁷	4,860	2,895

η_{Heat} = Efficiency of heating system

= Actual. If not available refer to default table below:⁹³⁸

System Type	Age of Equipment	HSPF Estimate	η_{Heat} (Effective COP Estimate) (HSPF/3.413)*0.85
Heat Pump	Before 2006	6.8	1.7
	After 2006 - 2014	7.7	1.92

⁹³⁵ Adapted from Table 1, page 24.4, of the 1977 ASHRAE Fundamentals Handbook

⁹³⁶ National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 60°F for a conditioned basement and 50°F for an unconditioned basement), consistent with the findings of Belzer and Cort, Pacific Northwest National Laboratory in "Statistical Analysis of Historical State-Level Residential Energy Consumption Trends," 2004. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

⁹³⁷ Weighted based on number of occupied residential housing units in each zone.

⁹³⁸ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.

System Type	Age of Equipment	HSPF Estimate	η_{Heat} (Effective COP Estimate) (HSPF/3.413)*0.85
	2015 on	8.2	2.04
Resistance	N/A	N/A	1

$ADJ_{\text{BasementHeat}}$ = Adjustment for basement wall insulation to account for prescriptive engineering algorithms overclaiming savings⁹³⁹.

= 60%

For example, a single family home in Chicago with a 20 by 25 by 7 foot R-2.25 basement, with 3 feet above grade, insulated with R-13 of interior spray foam, 10.5 SEER Central AC and 2.26 COP Heat Pump:

$$\begin{aligned}\Delta k\text{Wh} &= (\Delta k\text{Wh}_{\text{cooling}} + \Delta k\text{Wh}_{\text{heating}}) \\ &= [((((1/2.25 - 1/(13 + 2.25)) * (20+25+20+25) * 3 * (1 - 0)) * 24 * 281 * 0.75)/(1000 * 10.5)) * 0.8] + [((((1/2.25 - 1/(13 + 2.25)) * (20+25+20+25) * 3 * (1-0)) + ((1 / (2.25 + 6.42) - 1 / (13 + 2.25 + 6.42)) * (20+25+20+25) * 4 * (1-0))) * 24 * 3079) / (3412 * 1.92)) * 0.6] \\ &= (39.4 + 860.9) \\ &= 900.3 \text{ kWh}\end{aligned}$$

$\Delta k\text{Wh}_{\text{heating}}$ = If gas furnace heat, kWh savings for reduction in fan run time

$$= \Delta \text{Therms} * F_e * 29.3$$

F_e = Furnace Fan energy consumption as a percentage of annual fuel consumption

$$= 3.14\%^{940}$$

29.3 = kWh per therm

For example, a single family home in Chicago with a 20 by 25 by 7 foot unconditioned basement, with 3 feet above grade, insulated with R-13 of interior spray foam, and a 70% efficient furnace (for therm calculation see Natural Gas Savings section :

$$= 78.3 * 0.0314 * 29.3$$

$$= 72.0 \text{ kWh}$$

SUMMER COINCIDENT PEAK DEMAND

$$\Delta kW = (\Delta k\text{Wh}_{\text{cooling}} / \text{FLH}_{\text{cooling}}) * CF$$

Where:

$\text{FLH}_{\text{cooling}}$ = Full load hours of air conditioning

⁹³⁹ As demonstrated in two years of metering evaluation by Opinion Dynamics, see Memo “Results for AIC PY6 HPwES Billing Analysis”, dated February 20, 2015. TAC negotiated adjustment factor is 60%.

⁹⁴⁰ F_e is not one of the AHRI certified ratings provided for residential furnaces, but can be reasonably estimated from a calculation based on the certified values for fuel energy (E_f in MMBtu/yr) and E_{ae} (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the Energy Star version 3 criteria for 2% F_e . See “Programmable Thermostats Furnace Fan Analysis.xlsx” for reference.

= dependent on location⁹⁴¹:

Climate Zone (City based upon)	Single Family	Multifamily
1 (Rockford)	512	467
2 (Chicago)	570	506
3 (Springfield)	730	663
4 (Belleville)	1,035	940
5 (Marion)	903	820
Weighted Average ⁹⁴²	629	564

CF_{SSP} = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)
= 68%⁹⁴³

CF_{SSP} = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)
= 72%⁹⁴⁴

CF_{PJM} = PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)
= 46.6%⁹⁴⁵

For example, a single family home in Chicago with a 20 by 25 by 7 foot unconditioned basement, with 3 feet above grade, insulated with R-13 of interior spray foam, 10.5 SEER Central AC and 2.26 COP Heat Pump:

$$\Delta kW_{SSP} = 39.4 / 570 * 0.68$$

$$= 0.047 \text{ kW}$$

$$\Delta kW_{PJM} = 39.4 / 570 * 0.466$$

$$= 0.032 \text{ kW}$$

NATURAL GAS SAVINGS

If Natural Gas heating:

$$\Delta \text{Therms} = [(((1/R_{\text{old_AG}} - 1/(R_{\text{added}} + R_{\text{old_AG}})) * L_{\text{basement_wall_total}} * H_{\text{basement_wall_AG}} * (1 - \text{Framing_factor}) + (1/(R_{\text{old_BG}} - 1/(R_{\text{added}} + R_{\text{old_BG}})) * L_{\text{basement_wall_total}} * (H_{\text{basement_wall_total}} - H_{\text{basement_wall_AG}}) * (1 - \text{Framing_factor})) * 24 * \text{HDD}) / (\eta_{\text{Heat}} * 100,067)] * \text{ADJ}_{\text{BasementHeat}}$$

η_{Heat} = Efficiency of heating system

⁹⁴¹ Full load hours for Chicago, Moline and Rockford are provided in "Final Evaluation Report: Central Air Conditioning Efficiency Services (CACES), 2010, Navigant Consulting", http://ilsagfiles.org/SAG_files/Evaluation_Documents/ComEd/ComEd%20EPY2%20Evaluation%20Reports/ComEd_Central_AC_Efficiency_Services_PY2_Evaluation_Report_Final.pdf, p.33. An average FLH/Cooling Degree Day (from NCDC) ratio was calculated for these locations and applied to the CDD of the other locations in order to estimate FLH. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

⁹⁴² Weighted based on number of occupied residential housing units in each zone.

⁹⁴³ Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

⁹⁴⁴ Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

⁹⁴⁵ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

= Equipment efficiency * distribution efficiency

= Actual. If unknown assume 72%⁹⁴⁶

Other factors as defined above

For example, a single family home in Chicago with a 20 by 25 by 7 foot R-2.25 basement, with 3 feet above grade, insulated with R-13 of interior spray foam, and a 72% efficient furnace:

$$= ((1/2.25 - 1/(13 + 2.25)) * (20+25+20+25) * 3 * (1-0) + (1/8.67 - 1/(13 + 8.67)) * (20+25+20+25) * 4 * (1 - 0)) * 24 * 3079) / (0.72 * 100,067) * 0.60$$

$$= 78.3 \text{ therms}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-SHL-BINS-V08-180101

REVIEW DEADLINE: 1/1/2020

⁹⁴⁶ Based on average Nicor PY4 nameplate efficiencies derated by 15% for distribution losses.

5.6.3 Floor Insulation Above Crawl Space

DESCRIPTION

Insulation is added to the floor above a vented crawl space that does not contain pipes or HVAC equipment. If there are pipes, HVAC, or a basement, it is desirable to keep them within the conditioned space by insulating the crawl space walls and ground. Insulating the floor separates the conditioned space above from the space below the floor, and is only acceptable when there is nothing underneath that could freeze or would operate less efficiently in an environment resembling the outdoors. Even in the case of an empty, unvented crawl space, it is still considered best practice to seal and insulate the crawl space perimeter rather than the floor. Not only is there generally less area to insulate this way, but there are also moisture control benefits. There is a “Basement Insulation” measure for perimeter sealing and insulation. This measure assumes the insulation is installed above an unvented crawl space and should not be used in other situations.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

This measure requires a member of the implementation staff or a participating contractor to evaluate the pre and post R-values and measure surface areas. The requirements for participation in the program will be defined by the utilities.

DEFINITION OF BASELINE EQUIPMENT

The existing condition will be evaluated by implementation staff or a participating contractor and is likely to be no insulation on any surface surrounding a crawl space.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 25 years.⁹⁴⁷

DEEMED MEASURE COST

The actual installed cost for this measure should be used in screening.

DEEMED O&M COST ADJUSTMENTS

N/A

LOADSHAPE

Loadshape R08 - Residential Cooling

Loadshape R09 - Residential Electric Space Heat

Loadshape R10 - Residential Electric Heating and Cooling

COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM’s Forward Capacity Market.

⁹⁴⁷ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007

CF_{SSP}	= Summer System Peak Coincidence Factor for Central A/C (during utility peak hour) = 68% ⁹⁴⁸
CF_{SSP}	= Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour) = 72% ⁹⁴⁹
CF_{PJM}	= PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period) = 46.6% ⁹⁵⁰

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Where available savings from shell insulation measures should be determined through a custom analysis. When that is not feasible for the program the following engineering algorithms can be used with the inclusion of an adjustment factor to de-rate the heating savings.

$$\Delta kWh = (\Delta kWh_{cooling} + \Delta kWh_{heating})$$

Where:

$\Delta kWh_{cooling}$	= If central cooling, reduction in annual cooling requirement due to insulation = $\frac{(((1/R_{old} - 1/(R_{added} + R_{old})) * Area * (1 - Framing_factor)) * 24 * CDD * DUA) / (1000 * \eta_{Cool}))}{ADJ_{FloorCool}}$
R_{old}	= R-value value of floor before insulation, assuming 3/4" plywood subfloor and carpet with pad = Actual. If unknown assume 3.96 ⁹⁵¹
R_{added}	= R-value of additional spray foam, rigid foam, or cavity insulation.
Area	= Total floor area to be insulated
Framing_factor	= Adjustment to account for area of framing = 12% ⁹⁵²
24	= Converts hours to days
CDD	= Cooling Degree Days

⁹⁴⁸ Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

⁹⁴⁹ Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

⁹⁵⁰ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

⁹⁵¹ Based on 2005 ASHRAE Handbook – Fundamentals: assuming 2x8 joists, 16" OC, ¾" subfloor, ½" carpet with rubber pad, and accounting for a still air film above and below: $1 / [(0.85 \text{ cavity share of area} / (0.68 + 0.94 + 1.23 + 0.68)) + (0.15 \text{ framing share} / (0.68 + 7.5" * 1.25 \text{ R/in} + 0.94 + 1.23 + 0.68))] = 3.96$

⁹⁵² ASHRAE, 2001, "Characterization of Framing Factors for New Low-Rise Residential Building Envelopes (904-RP)," Table 7.1

Climate Zone (City based upon)	Unconditioned CDD ⁹⁵³
1 (Rockford)	263
2 (Chicago)	281
3 (Springfield)	436
4 (Belleville)	538
5 (Marion)	570
Weighted Average ⁹⁵⁴	325

DUA = Discretionary Use Adjustment (reflects the fact that people do not always operate their AC when conditions may call for it).

= 0.75⁹⁵⁵

1000 = Converts Btu to kBtu

η_{Cool} = Seasonal Energy Efficiency Ratio of cooling system (kBtu/kWh)

= Actual (where it is possible to measure or reasonably estimate). If unknown assume the following:⁹⁵⁶

Age of Equipment	η_{Cool} Estimate
Before 2006	10
2006 - 2014	13
Central AC After 1/1/2015	13
Heat Pump After 1/1/2015	14

$ADJ_{FloorCool}$ = Adjustment for cooling savings from floor to account for prescriptive engineering algorithms overclaiming savings⁹⁵⁷.

= 80%

$\Delta kWh_{heating}$ = If electric heat (resistance or heat pump), reduction in annual electric heating due to insulation

= $\left(\left(\left(\frac{1}{R_{old}} - \frac{1}{(R_{added} + R_{old})} \right) * Area * (1 - Framing_factor) * 24 * HDD \right) / (3,412 * \eta_{Heat}) \right) * ADJ_{FloorHeat}$

HDD = Heating Degree Days:⁹⁵⁸

⁹⁵³ Five year average cooling degree days with 75F base temp from DegreeDays.net were used in this table because the 30 year climate normals from NCDC used elsewhere are not available at base temps above 72F.

⁹⁵⁴ Weighted based on number of occupied residential housing units in each zone.

⁹⁵⁵ Energy Center of Wisconsin, May 2008 metering study; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research", p31.

⁹⁵⁶ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

⁹⁵⁷ As demonstrated in two years of metering evaluation by Opinion Dynamics, see Memo "Results for AIC PY6 HPwES Billing Analysis", dated February 20, 2015. TAC negotiated adjustment factor is 80%.

⁹⁵⁸ National Climatic Data Center, Heating Degree Days with a base temp of 50°F to account for lower impact of unconditioned space on heating system. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

Climate Zone (City based upon)	Unconditioned HDD
1 (Rockford)	3,322
2 (Chicago)	3,079
3 (Springfield)	2,550
4 (Belleville)	1,789
5 (Marion)	1,796
Weighted Average ⁹⁵⁹	2,895

η_{Heat} = Efficiency of heating system
 = Actual. If not available refer to default table below:⁹⁶⁰

System Type	Age of Equipment	HSPF Estimate	η_{Heat} (Effective COP Estimate) (HSPF/3.413)*0.85
Heat Pump	Before 2006	6.8	1.7
	2006 - 2014	7.7	1.92
	2015 on	8.2	2.04
Resistance	N/A	N/A	1

$ADJ_{\text{FloorHeat}}$ = Adjustment for floor insulation to account for prescriptive engineering algorithms overclaiming savings⁹⁶¹.
 = 60%

Other factors as defined above

For example, a single family home in Chicago with a 20 by 25 footprint, insulated with R-30 spray foam above the crawlspace, a 10.5 SEER Central AC and a newer heat pump:

$$\begin{aligned}
 \Delta \text{kWh} &= (\Delta \text{kWh}_{\text{cooling}} + \Delta \text{kWh}_{\text{heating}}) \\
 &= (((1/3.96 - 1/(30+3.96)) * (20*25) * (1-0.12) * 24 * 281 * 0.75) / (1000 * 10.5)) * 0.8 + (((1/3.96 - 1/(30+3.96)) * (20*25) * (1-0.15) * 24 * 3079) / (3412 * 1.92)) * 0.6) \\
 &= (37.8 + 641.7) \\
 &= 679.5 \text{ kWh}
 \end{aligned}$$

$\Delta \text{kWh}_{\text{heating}}$ = If gas *furnace* heat, kWh savings for reduction in fan run time
 = $\Delta \text{Therms} * F_e * 29.3$

F_e = Furnace Fan energy consumption as a percentage of annual fuel consumption

⁹⁵⁹ Weighted based on number of occupied residential housing units in each zone.

⁹⁶⁰ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.

⁹⁶¹ As demonstrated in two years of metering evaluation by Opinion Dynamics, see Memo "Results for AIC PY6 HPwES Billing Analysis", dated February 20, 2015. TAC negotiated adjustment factor is 60%.

$$= 3.14\%^{962}$$

$$29.3 = \text{kWh per therm}$$

For example, a single family home in Chicago with a 20 by 25 footprint, insulated with R-30 spray foam above the crawlspace, and a 70% efficient furnace (for therm calculation see Natural Gas Savings section):

$$\begin{aligned}\Delta\text{kWh} &= 60.4 * 0.0314 * 29.3 \\ &= 55.6 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta\text{kW} = (\Delta\text{kWh}_{\text{cooling}} / \text{FLH}_{\text{cooling}}) * \text{CF}$$

Where:

$\text{FLH}_{\text{cooling}}$ = Full load hours of air conditioning

= Dependent on location:⁹⁶³

Climate Zone (City based upon)	Single Family	Multifamily
1 (Rockford)	512	467
2 (Chicago)	570	506
3 (Springfield)	730	663
4 (Belleville)	1,035	940
5 (Marion)	903	820
Weighted Average ⁹⁶⁴	629	564

CF_{SSP} = Summer System Peak Coincidence Factor for Central A/C (during system peak hour)

$$= 68\%^{965}$$

CF_{SSP} = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)

$$= 72\%^{966}$$

CF_{PJM} = PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)

⁹⁶² F_e is not one of the AHRI certified ratings provided for residential furnaces, but can be reasonably estimated from a calculation based on the certified values for fuel energy (E_f in MMBtu/yr) and E_{ae} (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the Energy Star version 3 criteria for 2% F_e . See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.

⁹⁶³ Full load hours for Chicago, Moline and Rockford are provided in "Final Evaluation Report: Central Air Conditioning Efficiency Services (CACES), 2010, Navigant Consulting", p.33. An average FLH/Cooling Degree Day (from NCDC) ratio was calculated for these locations and applied to the CDD of the other locations in order to estimate FLH. There is a county mapping table Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

⁹⁶⁴ Weighted based on number of occupied residential housing units in each zone.

⁹⁶⁵ Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

⁹⁶⁶ Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

$$= 46.6\%^{967}$$

For example, a single family home in Chicago with a 20 by 25 footprint, insulated with R-30 spray foam above the crawlspace, a 10.5 SEER Central AC and a newer heat pump:

$$\Delta kW_{SSP} = 37.8 / 570 * 0.68$$

$$= 0.045 \text{ kW}$$

$$\Delta kW_{SSP} = 37.8 / 570 * 0.466$$

$$= 0.031 \text{ kW}$$

NATURAL GAS SAVINGS

If Natural Gas heating:

$$\Delta \text{Therms} = (1/R_{\text{old}} - 1/(R_{\text{added}} + R_{\text{old}})) * \text{Area} * (1 - \text{Framing_factor})) * 24 * \text{HDD} / (100,000 * \eta_{\text{Heat}}) * \text{ADJ}_{\text{FloorHeat}}$$

Where

η_{Heat} = Efficiency of heating system

= Equipment efficiency * distribution efficiency

= Actual. If unknown assume 72%⁹⁶⁸

Other factors as defined above

For example, a single family home in Chicago with a 20 by 25 footprint, insulated with R-30 spray foam above the crawlspace, and a 72% efficient furnace:

$$\Delta \text{Therms} = (1 / 3.96 - 1 / (30 + 3.96)) * (20 * 25) * (1 - 0.12) * 24 * 3079 / (100,000 * 0.72) * 0.60$$

$$= 60.4 \text{ therms}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-SHL-FINS-V08-180101

REVIEW DEADLINE: 1/1/2020

⁹⁶⁷ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

⁹⁶⁸ Based on average Nicor PY4 nameplate efficiencies derated by 15% for distribution losses.

5.6.4 Wall and Ceiling/Attic Insulation

DESCRIPTION

Insulation is added to wall cavities, and/or attic. This measure requires a member of the implementation staff evaluating the pre and post R-values and measure surface areas. The efficiency of the heating and cooling equipment in the home should also be evaluated if possible.

This measure was developed to be applicable to the following program types: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

This measure requires a member of the implementation staff or a participating contractor to evaluate the pre and post R-values and measure surface areas. The requirements for participation in the program will be defined by the utilities.

DEFINITION OF BASELINE EQUIPMENT

The existing condition will be evaluated by implementation staff or a participating contractor and is likely to be empty wall cavities and little or no attic insulation.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 25 years.⁹⁶⁹

DEEMED MEASURE COST

The actual installed cost for this measure should be used in screening.

LOADSHAPE

Loadshape R08 - Residential Cooling

Loadshape R09 - Residential Electric Space Heat

Loadshape R10 - Residential Electric Heating and Cooling

COINCIDENCE FACTOR

The summer peak coincidence factor for cooling is provided in two different ways below. The first is used to estimate peak savings during the utility peak hour and is most indicative of actual peak benefits, and the second represents the *average* savings over the defined summer peak period, and is presented so that savings can be bid into PJM's Forward Capacity Market.

CF_{SSP} = Summer System Peak Coincidence Factor for Central A/C (during utility peak hour)
= 68%⁹⁷⁰

CF_{SSP} = Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)
= 72%⁹⁷¹

CF_{PJM} = PJM Summer Peak Coincidence Factor for Central A/C (average during PJM peak period)

⁹⁶⁹ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007

⁹⁷⁰ Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

⁹⁷¹ Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

$$= 46.6\%^{972}$$

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Where available savings from shell insulation measures should be determined through a custom analysis. When that is not feasible for the program the following engineering algorithms can be used with the inclusion of an adjustment factor to de-rate the heating savings.

$$\Delta kWh = (\Delta kWh_{cooling} + \Delta kWh_{heating})$$

Where

- $\Delta kWh_{cooling}$ = If central cooling, reduction in annual cooling requirement due to insulation

$$= (((1/R_{old} - 1/R_{wall}) * A_{wall} * (1 - \text{Framing_factor_wall}) + (1/R_{old} - 1/R_{attic}) * A_{attic} * (1 - \text{Framing_factor_attic})) * 24 * CDD * DUA) / (1000 * \eta_{Cool})) * ADJ_{WallAtticCool}$$
- R_{wall} = R-value of new wall assembly (including all layers between inside air and outside air).
- R_{attic} = R-value of new attic assembly (including all layers between inside air and outside air).
- R_{old} = R-value value of existing assemble and any existing insulation.
 (Minimum of R-5 for uninsulated assemblies⁹⁷³)
- A_{wall} = Net area of insulated wall (ft²)
- A_{attic} = Total area of insulated ceiling/attic (ft²)
- $\text{Framing_factor_wall}$ = Adjustment to account for area of framing
 $= 25\%^{974}$
- $\text{Framing_factor_attic}$ = Adjustment to account for area of framing
 $= 7\%^{975}$
- 24 = Converts hours to days
- CDD = Cooling Degree Days
 = dependent on location:⁹⁷⁶

Climate Zone (City based upon)	CDD 65
1 (Rockford)	820
2 (Chicago)	842

⁹⁷² Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

⁹⁷³ An estimate based on review of Madison Gas and Electric, Exterior Wall Insulation, R-value for no insulation in walls, and NREL's Building Energy Simulation Test for Existing Homes (BESTEST-EX).

⁹⁷⁴ ASHRAE, 2001, "Characterization of Framing Factors for New Low-Rise Residential Building Envelopes (904-RP)," Table 7.1

⁹⁷⁵ Ibid.

⁹⁷⁶ National Climatic Data Center, Cooling Degree Days are based on a base temp of 65°F. There is a county mapping table Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

Climate Zone (City based upon)	CDD 65
3 (Springfield)	1,108
4 (Belleville)	1,570
5 (Marion)	1,370
Weighted Average ⁹⁷⁷	947

DUA = Discretionary Use Adjustment (reflects the fact that people do not always operate their AC when conditions may call for it).

= 0.75⁹⁷⁸

1000 = Converts Btu to kWh

η_{Cool} = Seasonal Energy Efficiency Ratio of cooling system (kBtu/kWh)

= Actual (where it is possible to measure or reasonably estimate). If unknown assume the following:⁹⁷⁹

Age of Equipment	η_{Cool} Estimate
Before 2006	10
2006 - 2014	13
Central AC After 1/1/2015	13
Heat Pump After 1/1/2015	14

$ADJ_{WallAtticCool}$ = Adjustment for cooling savings from basement wall insulation to account for prescriptive engineering algorithms overclaiming savings⁹⁸⁰.

= 80%

kWh_heating = If electric heat (resistance or heat pump), reduction in annual electric heating due to insulation

= (((1/R_old - 1/R_wall) * A_wall * (1-Framing_factor_wall)) + (1/R_old - 1/R_attic) * A_attic * (1-Framing_factor_attic)) * 24 * HDD / (η_{Heat} * 3412) * $ADJ_{WallAtticHeat}$

HDD = Heating Degree Days

= Dependent on location:⁹⁸¹

⁹⁷⁷ Weighted based on number of occupied residential housing units in each zone.

⁹⁷⁸ This factor's source is: Energy Center of Wisconsin, May 2008 metering study; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research", p31.

⁹⁷⁹ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

⁹⁸⁰ As demonstrated in two years of metering evaluation by Opinion Dynamics, see Memo "Results for AIC PY6 HPwES Billing Analysis", dated February 20, 2015. TAC negotiated adjustment factor is 80%.

⁹⁸¹ National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 60°F, consistent with the findings of Belzer and Cort, Pacific Northwest National Laboratory in "Statistical Analysis of Historical State-Level Residential Energy Consumption Trends," 2004. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

Climate Zone (City based upon)	HDD 60
1 (Rockford)	5,352
2 (Chicago)	5,113
3 (Springfield)	4,379
4 (Belleville)	3,378
5 (Marion)	3,438
Weighted Average ⁹⁸²	4,860

η_{Heat} = Efficiency of heating system

= Actual. If not available refer to default table below:⁹⁸³

System Type	Age of Equipment	HSPF Estimate	η_{Heat} (Effective COP Estimate) (HSPF/3.413)*0.85
Heat Pump	Before 2006	6.8	1.7
	2006 - 2014	7.7	1.92
	2015 on	8.2	2.04
Resistance	N/A	N/A	1

3412 = Converts Btu to kWh

$ADJ_{\text{WallAtticHeat}}$ = Adjustment for wall and attic insulation to account for prescriptive engineering algorithms overclaiming savings⁹⁸⁴.

= 60%

For example, a single family home in Chicago with 990 ft² of R-5 walls insulated to R-11 and 700 ft² of R-5 attic insulated to R-38, 10.5 SEER Central AC and 2.26 (1.92 including distribution losses) COP Heat Pump:

$$\begin{aligned}
 \Delta \text{kWh} &= (\Delta \text{kWh}_{\text{cooling}} + \Delta \text{kWh}_{\text{heating}}) \\
 &= (((((1/5 - 1/11) * 990 * (1-0.25)) + ((1/5 - 1/38) * 700 * (1-0.07))) * 842 * 0.75 * 24) / (1000 * 10.5)) * 0.8) + (((((1/5 - 1/11) * 990 * (1-0.25)) + ((1/5 - 1/38) * 700 * (1-0.07))) * 5113 * 24) / (1.92 * 3412)) * 0.6) \\
 &= 224 + 2181 \\
 &= 2405 \text{ kWh}
 \end{aligned}$$

$\Delta \text{kWh}_{\text{heating}}$ = If gas *furnace* heat, kWh savings for reduction in fan run time

= $\Delta \text{Therms} * F_e * 29.3$

F_e = Furnace Fan energy consumption as a percentage of annual fuel consumption

⁹⁸² Weighted based on number of occupied residential housing units in each zone.

⁹⁸³ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.

⁹⁸⁴ As demonstrated in two years of metering evaluation by Opinion Dynamics, see Memo "Results for AIC PY6 HPwES Billing Analysis", dated February 20, 2015. TAC negotiated adjustment factor is 60%.

$$= 3.14\%^{985}$$

$$29.3 = \text{kWh per therm}$$

For example, a single family home in Chicago with 990 ft² of R-5 walls insulated to R-11 and 700 ft² of R-5 attic insulated to R-38, with a gas furnace with system efficiency of 66% (for therm calculation see Natural Gas Savings section):

$$\begin{aligned}\Delta\text{kWh} &= 216.4 * 0.0314 * 29.3 \\ &= 199.1 \text{ kWh}\end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta\text{kW} = (\Delta\text{kWh}_{\text{cooling}} / \text{FLH}_{\text{cooling}}) * \text{CF}$$

Where:

$$\begin{aligned}\text{FLH}_{\text{cooling}} &= \text{Full load hours of air conditioning} \\ &= \text{Dependent on location as below:}^{986}\end{aligned}$$

Climate Zone (City based upon)	Single Family	Multifamily
1 (Rockford)	512	467
2 (Chicago)	570	506
3 (Springfield)	730	663
4 (Belleville)	1,035	940
5 (Marion)	903	820
Weighted Average ⁹⁸⁷	629	564

$$\begin{aligned}\text{CF}_{\text{SSP}} &= \text{Summer System Peak Coincidence Factor for Central A/C (during system peak hour)} \\ &= 68\%^{988}\end{aligned}$$

$$\begin{aligned}\text{CF}_{\text{SSP}} &= \text{Summer System Peak Coincidence Factor for Heat Pumps (during system peak hour)} \\ &= 72\%^{989}\end{aligned}$$

$$\begin{aligned}\text{CF}_{\text{PJM}} &= \text{PJM Summer Peak Coincidence Factor for Central A/C (average during peak period)} \\ &= 46.6\%^{990}\end{aligned}$$

⁹⁸⁵ F_e is not one of the AHRI certified ratings provided for residential furnaces, but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBtu/yr) and Eae (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the Energy Star version 3 criteria for 2% F_e. See "Programmable Thermostats Furnace Fan Analysis.xlsx" for reference.

⁹⁸⁶ Based on Full Load Hours from ENERGY Star with adjustments made in a Navigant Evaluation, other cities were scaled using those results and CDD. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

⁹⁸⁷ Weighted based on number of occupied residential housing units in each zone.

⁹⁸⁸ Based on metering of 24 homes with central AC during PY4 and PY5 in Ameren Illinois service territory.

⁹⁸⁹ Based on analysis of metering results from 24 heat pumps in Ameren Illinois service territory in PY5 coincident with AIC's 2010 system peak; 'Impact and Process Evaluation of Ameren Illinois Company's Residential HVAC Program (PY5)'.

⁹⁹⁰ Based on analysis of Itron eShape data for Missouri, calibrated to Illinois loads, supplied by Ameren. The average AC load over the PJM peak period (1-5pm, M-F, June through August) is divided by the maximum AC load during the year.

For example, a single family home in Chicago with 990 ft² of R-5 walls insulated to R-11 and 700 ft² of R-5 attic insulated to R-38, 10.5SEER Central AC and 2.26 COP Heat Pump:

$$\Delta kW_{SSP} = 224 / 570 * 0.68$$

$$= 0.27 \text{ kW}$$

$$\Delta kW_{PJM} = 224 / 570 * 0.466$$

$$= 0.18 \text{ kW}$$

NATURAL GAS SAVINGS

If Natural Gas heating:

$$\Delta \text{Therms} = (((1/R_{\text{old}} - 1/R_{\text{wall}}) * A_{\text{wall}} * (1 - \text{Framing_factor_wall})) + ((1/R_{\text{old}} - 1/R_{\text{attic}}) * A_{\text{attic}} * (1 - \text{Framing_factor_attic}))) * 24 * \text{HDD}) / (\eta_{\text{Heat}} * 100,067 \text{ Btu/therm}) * \text{ADJ}_{\text{WallAtticHeat}}$$

Where:

HDD = Heating Degree Days

= Dependent on location:⁹⁹¹

Climate Zone (City based upon)	HDD 60
1 (Rockford)	5,352
2 (Chicago)	5,113
3 (Springfield)	4,379
4 (Belleville)	3,378
5 (Marion)	3,438
Weighted Average ⁹⁹²	4,860

η_{Heat} = Efficiency of heating system

= Equipment efficiency * distribution efficiency

= Actual.⁹⁹³ If unknown assume 72%.⁹⁹⁴

Other factors as defined above

⁹⁹¹ National Climatic Data Center, calculated from 1981-2010 climate normals with a base temp of 60°F, consistent with the findings of Belzer and Cort, Pacific Northwest National Laboratory in "Statistical Analysis of Historical State-Level Residential Energy Consumption Trends," 2004. There is a county mapping table in Volume 1, Section 3.7 providing the appropriate city to use for each county of Illinois.

⁹⁹² Weighted based on number of occupied residential housing units in each zone.

⁹⁹³ Ideally, the System Efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. The Distribution Efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute: (<http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf>) or by performing duct blaster testing.

⁹⁹⁴ Based on average Nicor PY4 nameplate efficiencies derated by 15% for distribution losses.

For example, a single family home in Chicago with 990 ft² of R-5 walls insulated to R-11 and 700 ft² of R-5 attic insulated to R-38, with a gas furnace with system efficiency of 66%:

$$\begin{aligned}\Delta\text{Therms} &= (((1/5 - 1/11) * 990 * (1-0.25)) + ((1/5 - 1/38) * 700 * (1-0.07))) * 24 * 5113 / (0.66 \\ &\quad * 100,067) * 0.60 \\ &= 216.4 \text{ therms}\end{aligned}$$

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-SHL-AINS-V07-180101

REVIEW DEADLINE: 1/1/2020

5.7 Miscellaneous

5.7.1 High Efficiency Pool Pumps

DESCRIPTION

Conventional residential outdoor pool pumps are single speed, often oversized, and run frequently at constant flow regardless of load. Single speed pool pumps require that the motor be sized for the task that requires the highest speed. As such, energy is wasted performing low speed tasks at high speed. Two speed and variable speed pool pumps reduce speed when less flow is required, such as when filtering is needed but not cleaning, and have timers that encourage programming for fewer on-hours. Variable speed pool pumps use advanced motor technologies to achieve efficiency ratings of 90% while the average single speed pump will have efficiency ratings between 30% and 70%⁹⁹⁵. This measure is the characterization of the purchasing and installing of an efficient two speed or variable speed residential pool pump motor in place of a standard single speed motor of equivalent horsepower.

This measure was developed to be applicable to the following program types: TOS, NC, RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The high efficiency equipment is an ENERGY STAR two speed or variable speed residential pool pump for in-ground pools.

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is a single speed residential pool pump.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The estimated useful life for a two speed or variable speed pool pump is 10 years⁹⁹⁶.

DEEMED MEASURE COST

The incremental cost is estimated as \$235 for a two speed motor and \$549 for a variable speed motor⁹⁹⁷.

LOADSHAPE

Loadshape R15 – Residential Pool Pumps

COINCIDENCE FACTOR

The coincidence factor for this measure is assumed to be 0.831⁹⁹⁸.

⁹⁹⁵ U.S. DOE, 2012. Measure Guideline: Replacing Single-Speed Pool Pumps with Variable Speed Pumps for Energy Savings. Report No. DOE/GO-102012-3534.

⁹⁹⁶ The CEE Efficient Residential Swimming Pool Initiative, p18, indicates that the average motor life for pools in use year round is 5-7 years. For pools in use for under a third of a year, you would expect the lifetime to be higher so 10 years is selected as an assumption. This is consistent with DEER, 2014 and the ENERGY STAR Pool Pump Calculator assumptions.

⁹⁹⁷ ENERGY STAR Pool Pump Calculator.

⁹⁹⁸ Based on assumptions of daily load pattern through pool season. Assumption was developed for Efficiency Vermont but is considered a reasonable estimate for Illinois.

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS⁹⁹⁹

$$\Delta \text{kWh two speed} = (((\text{Hrs/Day}_{\text{base}} * \text{GPM}_{\text{base}} * 60) / \text{EF}_{\text{base}}) - (((\text{Hrs/Day}_{2\text{spH}} * \text{GPM}_{2\text{spH}} * 60) / \text{EF}_{2\text{spH}} + ((\text{Hrs/Day}_{2\text{spL}} * \text{GPM}_{2\text{spL}} * 60) / \text{EF}_{2\text{spL}})))) / 1000 * \text{Days}$$

$$\Delta \text{kWh variable speed} = (((\text{Hrs/Day}_{\text{base}} * \text{GPM}_{\text{base}} * 60) / \text{EF}_{\text{base}}) - (((\text{Hrs/Day}_{\text{vsH}} * \text{GPM}_{\text{vsH}} * 60) / \text{EF}_{\text{vsH}} + ((\text{Hrs/Day}_{\text{vsL}} * \text{GPM}_{\text{vsL}} * 60) / \text{EF}_{\text{vsL}})))) / 1000 * \text{Days}$$

Where:

Hrs/Day _{base}	= run hours of single speed pump = 11.4
GPM _{base}	= flow of single speed pump (gal/min) = 64.4
60	= minutes per hour
EF _{base}	= Energy Factor of baseline single speed pump (gal/Wh) = 2.1
Hrs/Day _{2spH}	= run hours of two speed pump at high speed = 2
GPM _{2spH}	= flow of two speed pump at high speed (gal/min) = 56
EF _{2spH}	= Energy Factor of two speed pump at high speed (gal/Wh) = 2.4
Hrs/Day _{2spL}	= run hours of two speed pump at low speed = 15.7
GPM _{2spL}	= flow of two speed pump at low speed (gal/min) = 31
EF _{2spL}	= Energy Factor of two speed pump at high speed (gal/Wh) = 5.4
Hrs/Day _{vsH}	= run hours of variable speed pump at high speed = 2
GPM _{vsH}	= flow of variable speed pump at high speed (gal/min) = 50
EF _{vsH}	= Energy Factor of variable speed pump at high speed (gal/Wh)

⁹⁹⁹ The methodology and all assumptions are sourced from the ENERGY STAR Pool Pump Calculator and assume a nameplate horsepower of 1.5 and a pool size of 22,000 gallons, with 2.0 turnovers per day in the base case and 1.5 turnovers per day in the efficient case.

$$\begin{aligned}
 &= 3.8 \\
 \text{Hrs/Day}_{\text{vsL}} &= \text{run hours of variable speed pump at low speed} \\
 &= 16 \\
 \text{GPM}_{\text{vsL}} &= \text{flow of variable speed pump at low speed (gal/min)} \\
 &= 30.6 \\
 \text{EF}_{\text{vsL}} &= \text{Energy Factor of variable speed pump at high speed (gal/Wh)} \\
 &= 7.3 \\
 \text{Days} &= \text{Number of days per year that the swimming pool is operational} \\
 &= 125^{1000} \\
 \Delta \text{kWh two speed} &= (((11.4 * 64.4 * 60)/2.1) - (((2 * 56 * 60)/2.4 + ((15.7 * 31 * 60)/5.4))))/1000 * 125 \\
 &= 1,596.0 \text{ kWh} \\
 \Delta \text{kWh variable speed} &= (((11.4 * 64.4 * 60)/2.1) - (((2 * 50 * 60)/3.8 + ((16 * 30.6 * 60)/7.3))))/1000 * 125 \\
 &= 1,921.6 \text{ kWh}
 \end{aligned}$$

SUMMER COINCIDENT PEAK DEMAND SAVINGS ¹⁰⁰¹

$$\begin{aligned}
 \Delta \text{kW two speed} &= ((\text{kWh/day}_{\text{base}})/(\text{Hrs/day}_{\text{base}}) - (\text{kWh/day}_{2\text{sp}})/(\text{Hr/day}_{2\text{sp}})) * \text{CF} \\
 \Delta \text{kW variable speed} &= ((\text{kWh/day}_{\text{base}})/(\text{Hrs/day}_{\text{base}}) - (\text{kWh/day}_{\text{var}})/(\text{Hr/day}_{\text{var}})) * \text{CF}
 \end{aligned}$$

Where:

$$\begin{aligned}
 \text{kWh/day}_{\text{base}} &= \text{daily energy consumption of baseline pump, as defined above} \\
 &= 20.98 \\
 \text{Hrs/day}_{\text{base}} &= \text{daily run hours of single speed pump} \\
 &= 11.4 \\
 \text{kWh/day}_{2\text{sp}} &= \text{daily energy consumption of two speed pump, as defined above} \\
 &= 8.21 \\
 \text{Hr/day}_{2\text{sp}} &= \text{run hours of two speed pump} \\
 &= 17.7 \\
 \text{kWh/day}_{\text{var}} &= \text{daily energy consumption of variable speed pump, as defined above} \\
 &= 5.6 \\
 \text{Hr/day}_{\text{var}} &= \text{run hours of variable speed pump} \\
 &= 18
 \end{aligned}$$

¹⁰⁰⁰ Assumes 50% of pools operated from Memorial Day through Labor Day (100 days) and 50% of pools operate for a longer span, typically the 5 month period between May and September (150 days), due to their ability to heat the pool.

¹⁰⁰¹ The methodology and all assumptions are sourced from the ENERGY STAR Pool Pump Calculator and assume a nameplate horsepower of 1.5 and a pool size of 22,000 gallons, with 2.0 turnovers per day in the base case and 1.5 turnovers per day in the efficient case.

CF = Summer Peak Coincidence Factor for measure
= 0.831¹⁰⁰²

ΔkW two speed = $((20.98 / 11.4) - (8.21 / 17.7)) * 0.831$
= 1.144 kW

ΔkW variable speed = $((20.98 / 11.4) - (5.60 / 18)) * 0.831$
= 1.271 kW

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE: RS-MSC-RPLP-V01-180101

REVIEW DEADLINE: 1/1/2022

¹⁰⁰² Based on assumptions of daily load pattern through pool season. Assumption was developed for Efficiency Vermont but is considered a reasonable estimate for Illinois.

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Version 6.0

Volume 4: Cross-Cutting Measures and Attachments

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6 Cross-Cutting Measures

6.1 Behavior

6.1.1 Adjustments to Behavior Savings to Account for Persistence

DESCRIPTION

Energy efficiency program administrators are increasingly including behavior programs as part of their portfolios. These programs are characterized by various kinds of outreach, education, and customer engagement designed to motivate increases in conservation and energy management behaviors, and most commonly include participant-specific energy usage information. Savings impacts are evaluated by ex-post billing analysis comparing consumption before and after (or with and without) program intervention, and require M&V methods that include customer-specific energy usage regression analysis and randomized controlled trial (RCT) experimental designs, among others (see Behavioral protocol set forth in the IL-TRM Attachment A: Illinois Statewide Net-to-Gross Methodologies for more information). As such, initial calculation of savings is treated as a custom protocol¹.

An important issue for many stakeholders is whether energy savings from behavior programs continue over time (i.e., whether they persist beyond the initial program year). Behavior programs have now been delivered for a number of years in many jurisdictions. The weight of evaluation evidence indicates that the energy-saving behaviors influenced through these programs can persist beyond the initial period of program intervention, even without continued program participation². This post-treatment savings persistence has implications for calculations of first-year savings, measure life, and cost-effectiveness testing. Accounting for persistence will yield savings and cost-effectiveness estimates that more accurately reflect the true benefits of these programs. Because annual goals are based on first-year savings, programs should only count savings attributable to first-year spending. The effect of persistence of savings beyond the first year should be included in lifetime savings calculations and cost-effectiveness testing.

The protocol below was developed to outline the adjustments that should be made to account for the persistence of savings beyond the year of program delivery. This protocol is applicable to behavior programs of any type, delivered to residential or C&I customers, that has evaluated evidence of program persistence; however, the persistence values in this version of the protocol are specific to residential home energy reports (HERs)-type programs³. This general protocol should be used for any type of behavior program once supportable assumptions for persistence exist as measured by multi-year, rigorous evaluation studies; persistence factors for those behavioral programs may differ from the specific factors provided in this measure for HERs-type programs.

Currently, evaluations calculate a custom value on an annual basis to estimate yearly savings. Evaluators typically use a regression analysis to estimate program effects. These regression analyses provide what is called an average treatment effect on the treated (ATT) estimate of program savings. The ATT approach takes advantage of the presence of a randomly assigned control group for each cohort that received reports in the service territory. These

¹ The protocol outlined here assumes that adjustments to remove the effects of savings from program lift (participation in other utility programs), including legacy uplift, to account for move-outs and opt-outs, to normalize for effects of weather, and any other appropriate adjustments, have been made as part of the custom calculation of savings – this final savings value is referred to as “Measured Savings” in the calculations below.

² Long-Run Savings and Cost-Effectiveness of Home Energy Reports Programs, Cadmus, October 2014. Also see additional sources in the REFERENCE TABLE below.

³ Residential HERs-type programs: programs that regularly deliver home energy reports to residential customers through direct mail or email channels using a random control trial (RCT) experimental design. At a minimum, the reports include customer-specific usage information used for a comparison to similar households and individualized energy savings tips.

regressions use various methods to account for household-specific usage patterns⁴. Because of the experimental design, we can assume that the treatment and control groups experienced similar historical, political, economic, and other events that had comparable effects on their energy use. Moreover, because these groups experienced generally similar weather conditions, it is not necessary to measure or include weather in the RCT model specification to calculate initial annual savings related to the program.

However, in the case of comparing and summing savings year over year, exogenous factors, such as weather, are likely to make annual estimates non-equivalent. In particular, weather is likely to play an important role in driving behavioral effects, affecting savings magnitude (e.g., a constant percentage change in consumption will result in more cooling savings during a hotter-than-average summer), as well as savings rate (e.g., the percentage change in consumption is likely to be higher during hotter-than-average summers.⁵ As such, for this framework, evaluators will adjust for effects related to weather as part of the custom inputs to this protocol. Each evaluator will choose the most appropriate method for weather normalization. For example, one method would be to provide savings using a model specification that incorporates standard weather year inputs (e.g., HDD and CDD), to be used as the initial input into the calculation of annual savings, as well as inputs for cost effectiveness, as outlined below. This input will approximate average savings for a standard weather year based upon historical data.⁶ Adjusting savings to a standard weather year is consistent with how other weather-sensitive TRM measures are specified, and will remove weather risk from performance goals and cost-effectiveness testing.⁷

The protocol will become effective for residential HERs-type programs as of January 1, 2018. All ongoing programs will undergo a “reset” upon institution of this protocol⁸. Regardless of any previous history of behavior program delivery, the program year ending December 31, 2018 will be assumed to be Year 1 for all HERs-type programs underway at that time for the purpose of the incorporation of multiyear measure life/savings persistence into cost-effectiveness calculations and for the application of the adjustments to annual savings as outlined below. Should any additional new programs (referred to as “waves” in the calculations below) be established in 2018 or in subsequent years, their first year will be assumed to be Year 1 for that wave – that is, each wave is tracked separately and savings are calculated separately using the approach outlined here. Waves that existed prior to the program year ending December 31, 2018 will continue to be tracked separately for each wave. All residential HERs-type programs implemented prior to January 1, 2018 will assume a one-year measure life; the assumptions and protocols outlined below will not be applied retrospectively to any utility programs. Updates to persistence factors from future evaluations, once incorporated into the IL-TRM, will be used when available for calculation of annual savings values for applicable program years but will not be applied retrospectively to previous years’ first-year savings calculations. All other types of behavior programs will continue to use a one-year measure life until

⁴ For example, a linear fixed-effects regression (LFER) model includes a household-specific intercept to account for time-invariant, household-level factors affecting energy use, and a post program regression (PPR) model uses energy use lags to account for household-specific usage in the year prior to the program.

⁵ An analysis to confirm that cross-year effects of weather are material, and therefore should be included as outlined here, is planned.

⁶ In the future, this approach could be empirically tested by comparing actual savings calculated in future program years against standard weather year results, producing a ‘realization rate’ between planned and actual savings results. Standard weather years could potentially be enhanced to better reflect these differences.

⁷ We acknowledge that this approach is a proxy for estimating actual savings to allow for prospective calculation of lifetime savings. However, a substantial limitation to this approach is the issue of unobserved behavioral ramp-up that is likely to occur for future waves of participants.

⁸ It is understood that this approach does not accurately take into account that programs have been in place prior to this date, and the fact that customers at that time will have been receiving reports for variable amounts of time, with varied associated actual savings persistence from these earlier program efforts. The difficulties of trying to “phase in” persistence adjustments to reflect this history have been recognized, and the approach outlined here has been recommended by the Illinois TAC members as a reasonable approximation.

supportable evidence exists for savings persistence, at which time this adjustment protocol can be used with appropriate persistence factors.

DETERMINATION OF EFFICIENT BEHAVIOR

Behavior programs focus primarily on reducing electricity and natural gas consumption through behavioral changes; this reduction is generally measured through ex-post billing analysis after program intervention. Specific energy conservation and management behaviors are not usually directly observable. The specific definition of the efficient case is part of the design of behavioral programs and is included as part of the custom saving protocol, which will include any adjustment necessary to remove effects of program-related investments in efficient equipment.

DETERMINATION OF BASELINE BEHAVIOR

The ideal baseline for behavior programs is the energy usage without the program intervention. Various types of experimental, quasi-experimental, and/or regression-based EM&V approaches are used to present statistically valid approximations to this without-program baseline⁹. The specific definition of the baseline case is part of the design of behavioral programs and is included as part of the custom saving protocol.

DEEMED LIFETIME/PERSISTENCE OF SAVINGS

Evaluations in Illinois have shown that savings from residential HERs-type behavior programs can persist into at least the first and second year following discontinuation of program delivery¹⁰, though on-going savings levels decay in the second year. For other residential RCT programs evaluated to date, savings have been shown to persist for at least 3 years year following program delivery¹¹, and industry expectations are that savings likely persist beyond that. We assume here that savings persist at some level for 5 years¹². On-going savings over those 5 years are not equal, however; it is preferable that actual levels of ongoing savings should be calculated by future year as outlined below (see Application of Persistence for Cost-effectiveness) and used in cost-effectiveness and lifetime savings calculations¹³. For other behavior program types without evaluations that quantify levels of persistence, measure life is assumed = 1 year.

⁹ See the Illinois Behavioral protocol set forth in the IL-TRM Attachment A: IL-NTG Methods for more information concerning randomized control trials and quasi-experimental evaluation methods for non-randomized designs for behavior programs.

¹⁰ ComEd Home Energy Report Opower Program Decay Rate and Persistence Study DRAFT-Navigant, presented to Commonwealth Edison Company, January 29, 2016; ComEd Home Energy Report Program Decay Rate and Persistence Study, Year Two DRAFT - Navigant, Presented to Commonwealth Edison Company, July 20, 2016; Behavioral Energy Savings Programs: Home Energy Reports Persistence Study Part 2 – April 2015 to September 2015 FINAL – Navigant, Prepared for Nicor Gas, September 21, 2016.

¹¹ Long-Run Savings and Cost-Effectiveness of Home Energy Reports Programs, Cadmus, October 2014. Also see additional sources in the REFERENCE TABLE below. Given the limited persistence studies available, we acknowledge that using an average of these studies by fuel type may be the best approximation of persistence rates. However, moving forward, the TAC will incorporate additional study values and develop the most appropriate persistence factors, taking into account participant characteristics, such as the duration of exposure, the frequency of reports, baseline usage, as well as the amount of time that has persisted since receiving their final report, and the shape of the persistence curve.

¹² Determined as a reasonable preliminary assumption by Illinois TAC members. This assumption should be updated as additional research is conducted on these types of programs, and additional evaluation should be undertaken to assess the reasonableness of this assumption for Illinois-specific programs.

¹³ This method of applying calculated values for future year benefits is preferred. Alternatively, an effective measure life can be calculated as Effective Measure Life = Total Discounted Lifetime Savings / First Year Savings.

DEEMED MEASURE COST

It is assumed that most behavior changes in residential settings can be accomplished with homeowner labor only and without investment in new equipment; therefore, without evidence to the contrary, measure costs in such residential programs focused on motivating changes in customer behavior may be defined as \$0¹⁴. Costs for C&I programs may include additional staffing, software purchases, etc. Cost for such programs is therefore program specific and is determined on a custom basis.

LOADSHAPE AND COINCIDENCE FACTOR

While there is evidence from analysis of AMI data that the savings loadshape for residential HERs-type programs mirrors the whole-house electric energy load pattern, there are not yet enough data to develop a behavior-specific loadshape. Indications from several unpublished analyses¹⁵ show that these behavior savings occur in a general pattern most closely approximated by the Residential Electric Heating and Cooling Loadshape (R10) than any other current residential measure loadshape; this is therefore recommended as the most reasonable approximation for use until more-specific data are available. Loadshapes and coincidence factors will need to be determined for other types of behavior programs once sufficient data are in hand.

Algorithm

CALCULATION OF SAVINGS

Throughout these protocols, Year T refers to the current reporting year for which annual savings are being determined¹⁶.

ELECTRIC ENERGY SAVINGS

The algorithm shown below for this measure was developed to calculate the annual persistence-adjusted electric savings in to be reported in year T after adjustment to account for the proportion of the measured savings for that program year that actually reflects any persistent savings from prior years' program activities (Years T-1, T-2, T-3, and T-4)¹⁷.

¹⁴ Future evaluation of costs of behavior change is encouraged to help clarify this assumption. In addition, as noted earlier in this measure characterization, in order to ensure double counting of savings does not occur, the protocol outlined here assumes that adjustments to remove the effects of program lift have been made as part of the custom calculation of savings. In a similar manner, given the savings accounted for by other utility programs are removed from the savings claims and cost-effectiveness for the behavior program, the incremental costs associated with such utility program incentivized measures should also be excluded from the behavior program cost-effectiveness analysis, so as to help ensure double counting of costs does not occur in the utility portfolio cost-effectiveness analysis.

¹⁵ Based on communication from Mathias Bell based on (currently unpublished) studies done by Opower, Cadmus, and LBNL. Also see DTE Energy: Behavior Program Measures for Submission to 2015 MEMD - Year Three Energy Savings - Demand Savings. Energy Optimization, April 15, 2014. http://www.michigan.gov/documents/mpsc/memd_2015_453673_7.pdf

¹⁶ Calculation algorithms account for attrition of customers out of the service territory, as well as persistence decay. It has been noted that there may also be a need to adjust for cross-year effects of large differences in weather conditions or economic impacts. Custom savings inputs therefore are adjusted for standard year weather. Further studies are needed to help determine the magnitude of such effects and if this is the appropriate way to account for them.

¹⁷ This calculation should be carried out separately for each "wave" of behavior programs, where a wave is defined as a newly launched program. For simplicity, any new wave is assumed to start at the beginning of a program year (Year 1) and may include multiple different treatment types such as usage groups, report frequency, etc. For example, any wave added after 2018, will be considered Year 1 in the year they are launched.

$$\Delta kWh_{T \text{ Adjusted}} = \Delta kWh_{T \text{ Measured}} - (\Delta kWh_{T-1 \text{ Adjusted}} * RR_{T-1,T} * PFE_1) - (\Delta kWh_{T-2 \text{ Adjusted}} * RR_{T-2,T} * PFE_2) - (\Delta kWh_{T-3 \text{ Adjusted}} * RR_{T-3,T} * PFE_3) - (\Delta kWh_{T-4 \text{ Adjusted}} * RR_{T-4,T} * PFE_4)$$

Where:

$\Delta kWh_{x \text{ Adjusted}}$ = total program annual savings for year X after adjustments to account for persistence (calculated value)

$\Delta kWh_{x \text{ Measured}}$ = measured kWh savings: total program savings as determined from custom calculation/billing analysis¹⁸ of participants in program during year X (input value)

$RR_{Y,X}$ = Program retention rate in year X from year Y participation

= % of program participants in year Y that are still in program in year X (input value: calculated as # participants still in program in year X / # participants in year Y))

PFE_z = Persistence factor - electric (deemed value)

= % savings that persist Z years after savings were initially measured, where Z is a number from 1 - 4

= use table below to select the appropriate value

Electric Persistence Factors¹⁹

Program Type	Program Year T - record 100% of adjusted savings ($\Delta kWh_{T \text{ Adjusted}}$ above)	Percent adjusted savings from Year T activities that persist 1 year after year T	Percent adjusted savings from Year T activities that persist 2 years after year T	Percent adjusted savings from Year T activities that persist 3 years after year T	Percent adjusted savings from Year T activities that persist 4 years after year T
		PFE_1	PFE_2	PFE_3	PFE_4
Residential HERs-type (RCT)	100%	80%	54%	31%	15%

¹⁸ All appropriate adjustments to remove effects of participation in other utility programs, move-outs, opt-outs, to normalize for effects related to weather, and other adjustments as determined by the program experimental design, are assumed to have been made to result in this value for “measured savings”. This value has been adjusted for standard year weather terms.

¹⁹ See REFERENCE TABLES below for sources.

Example of Adjusted Annual Savings Calculations:

Assume the following information on participation and measured savings for the following program years (all adjustments have been made to remove effects of program lift, weather, etc. within the custom savings calculations). Assume 2018 is the first year of all programs (or is the “reset” year).

	Reporting Year					
	2018	2019	2020	2021	2022	2023
Input data from program information and custom savings analysis						
# Participants (households)	120,000	109,000	103,000	99,000	94,000	90,000
kWh per participant (household)	200	250	245	250	250	265
Measured kWh savings (custom)	24,000,000	27,250,000	25,235,000	24,750,000	23,500,000	23,850,000

Calculation of Retention Rates:For use in 2019:

$$RR_{2018, 2019} = 109,000/120,000 = 0.908$$

For use in 2020:

$$RR_{2018, 2020} = 103,000/120,000 = 0.858$$

$$RR_{2019, 2020} = 103,000/109,000 = 0.945$$

For use in 2021:

$$RR_{2018, 2021} = 99,000/120,000 = 0.825$$

$$RR_{2019, 2021} = 99,000/109,000 = 0.908$$

$$RR_{2020, 2021} = 99,000/103,000 = 0.961$$

For use in 2022:

$$RR_{2018, 2022} = 94,000/120,000 = 0.783$$

$$RR_{2019, 2022} = 94,000/109,000 = 0.862$$

$$RR_{2020, 2022} = 94,000/103,000 = 0.913$$

$$RR_{2021, 2022} = 94,000/99,000 = 0.949$$

For use in 2023:

$$RR_{2019, 2023} = 90,000/109,000 = 0.826$$

$$RR_{2020, 2023} = 90,000/103,000 = 0.874$$

$$RR_{2021, 2023} = 90,000/99,000 = 0.909$$

$$RR_{2022, 2023} = 90,000/94,000 = 0.957$$

Calculation of Adjusted Annual Savings:

$$\Delta kWh_{2018 \text{ Adjusted}} = 24,000,000 \text{ kWh}$$

$$\begin{aligned} \Delta kWh_{2019 \text{ Adjusted}} &= 27,250,000 - (24,000,000 * 0.908 * 0.80) \\ &= 9,816,400 \text{ kWh} \end{aligned}$$

$$\begin{aligned} \Delta kWh_{2020 \text{ Adjusted}} &= 25,235,000 - (9,816,400 * 0.945 * 0.80) - (24,000,000 * 0.858 * 0.54) \\ &= 6,694,122 \text{ kWh} \end{aligned}$$

$$\begin{aligned} \Delta kWh_{2021 \text{ Adjusted}} &= 24,750,000 - (6,694,122 * 0.961 * 0.80) - (9,816,400 * 0.908 * 0.54) - (24,000,000 * 0.825 * 0.31) \\ &= 8,652,382 \text{ kWh} \end{aligned}$$

$$\begin{aligned} \Delta kWh_{2022 \text{ Adjusted}} &= 23,500,000 - (8,652,382 * 0.949 * 0.80) - (6,694,122 * 0.913 * 0.54) - (9,816,400 * 0.862 * 0.31) \\ &\quad - (24,000,000 * 0.783 * 0.15) \\ &= 8,188,837 \text{ kWh} \end{aligned}$$

$$\begin{aligned} \Delta kWh_{2023 \text{ Adjusted}} &= 23,850,000 - (8,188,837 * 0.957 * 0.80) - (8,652,382 * 0.909 * 0.54) - (6,694,122 * 0.874 * 0.31) \\ &\quad - (9,816,400 * 0.826 * 0.15) \\ &= 10,303,561 \text{ kWh} \end{aligned}$$

Apply the same approach to calculate adjusted annual kW and Therms.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

Coincident peak demand savings in year T should also be adjusted to account for persistence from previous years using a similar algorithm²⁰.

²⁰ While there are no current studies that evaluate the persistence of peak savings, without more-specific information on the actual behaviors undertaken by program participants and their corresponding peak savings, it seems reasonable to assume that peak savings will also persist in a similar pattern; both of the approaches given assume persistence in peak savings. Further

If peak demand is measured directly by the custom savings analysis:

$$kW_{T \text{ Adjusted}} = \Delta kW_{T \text{ Measured}} - (\Delta kW_{T-1 \text{ Adjusted}} * RR_{T-1,T} * PFE_1) - (\Delta kW_{T-2 \text{ Adjusted}} * RR_{T-2,T} * PFE_2) \\ - (\Delta kW_{T-3 \text{ Adjusted}} * RR_{T-3,T} * PFE_3) - (\Delta kW_{T-4 \text{ Adjusted}} * RR_{T-4,T} * PFE_4)$$

Where:

$\Delta kW_{X \text{ Adjusted}}$ = total program demand savings for year X after adjustments to account for persistence (calculated value)

$\Delta kW_{X \text{ Measured}}$ = total program demand savings as determined from custom calculation /billing analysis²¹ of participants in program during year X (input value)

Other variables as defined above

If peak demand is not measured directly by the custom savings analysis, peak demand should be calculated as follows:

$$\Delta kW_{T \text{ Adjusted}} = (\Delta kWh_{T \text{ Adjusted Summer}} / \# \text{summer hours}) * \text{peak adjustment factor}$$

Where:

$\Delta kWh_{T \text{ Adjusted Summer}}$ = average adjusted electric energy savings (calculated above) for peak summer months

$$= \Delta kWh_{T \text{ Adjusted}} * 0.42 * (3/5)$$

$$= \Delta kWh_{T \text{ Adjusted}} * 0.25$$

Where:

0.42 = Summer Loadshape % for May – Sept

3/5 = proportion of May-Sept hours that fall in June, July, and Aug

summer hours = # hours in June, July, and Aug

$$= 8760 / 4$$

Where: 8760 = Hours per year

peak adjustment factor = adjustment for peak k/w over average kW for these hours

$$= 1.5^{22}$$

NATURAL GAS ENERGY SAVINGS

The algorithm shown below for this measure was developed to calculate the annual persistence-adjusted Therm savings in to be reported in year T after adjustment to account for the proportion of the measured savings for that program year that actually reflects any persistent savings from prior years' program activities (Years T-1, T-2, T-3,

evaluation should be undertaken to clarify this point and determine appropriate peak-specific persistence values.

²¹ All appropriate adjustments to remove effects of participation in other utility programs, move-outs, opt-outs, to normalize for effects related to weather, and other adjustments as determined by the program experimental design, are assumed to have been made to result in this value for "measured savings". This value has been adjusted for standard year weather terms.

²² Based on an approach used in Michigan that gives resulting values supported by evaluation claims. Also see DTE Energy: Behavior Program Measures for Submission to 2015 MEMD - Year Three Energy Savings - Demand Savings. Energy Optimization, April 15, 2014. http://www.michigan.gov/documents/mpsc/memd_2015_453673_7.pdf

and T-4).²³

$$\Delta\text{Therms}_{T \text{ Adjusted}} = \Delta\text{Therms}_{T \text{ Measured}} - (\Delta\text{Therms}_{T-1 \text{ Adjusted}} * \text{RR}_{T-1,T} * \text{PFG}_1) - (\Delta\text{Therms}_{T-2 \text{ Adjusted}} * \text{RR}_{T-2,T} * \text{PFG}_2) - (\Delta\text{Therms}_{T-3 \text{ Adjusted}} * \text{RR}_{T-3,T} * \text{PFG}_3) - (\Delta\text{Therms}_{T-4 \text{ Adjusted}} * \text{RR}_{T-4,T} * \text{PFG}_4)$$

Where:

$\Delta\text{Therms}_{x \text{ Adjusted}}$ = total program annual savings for year X after adjustments to account for persistence (calculated value)

$\Delta\text{Therms}_{x \text{ Measured}}$ = total program savings as determined from custom calculation/billing analysis²⁴ of participants in program during year X (input value)

PFG_z = Persistence factor - gas (deemed value)

= % savings that persist Z years after savings were initially measured, where Z is a number from 1 - 4

= use table below to select the appropriate value

Other variables as defined above

Gas Persistence Factors²⁵

Program Type	Program Year T - record 100% of calculated savings ($\Delta\text{Therms}_{T \text{ Adjusted}}$ above)	Percent adjusted savings from Year T activities that persist 1 year after year T	Percent adjusted savings from Year T activities that persist 2 years after year T	Percent adjusted savings from Year T activities that persist 3 years after year T	Percent adjusted savings from Year T activities that persist 4 years after year T
		PFG_1	PFG_2	PFG_3	PFG_4
Residential HERs-type (RCT)	100%	45%	20%	9%	4%

APPLICATION OF PERSISTENCE FOR COST-EFFECTIVENESS

For determination of cost effectiveness (or lifetime savings) of programs in year T, future years' savings related to the current year activities should be recorded for this measure as savings for each specific year using the table below²⁶. Because of the potentially confounding effects of differences in weather in future years, the savings inputs used ($\Delta\text{kWh}_{T \text{ Adjusted}}$, $\Delta\text{kW}_{T \text{ Adjusted}}$, $\Delta\text{Therms}_{T \text{ Adjusted}}$) for these future-year savings calculations have been weather

²³ This calculation should be carried out separately for each "wave" of behavior programs, where a wave is defined as a newly launched program. For simplicity, any new wave is assumed to start at the beginning of a program year (Year 1) and may include multiple different treatment types such as usage groups, report frequency, etc.

²⁴ All appropriate adjustments to remove effects of participation in other utility programs, move-outs, opt-outs, to normalize for effects related to weather, and other adjustments as determined by the program experimental design, are assumed to have been made to result in this value for "measured savings". This value has been adjusted for standard year weather terms.

²⁵ See REFERENCE TABLES below for sources.

²⁶ These cost-effectiveness calculations assume a retention rate of 100% after the first program year. Move-out rates and other attrition factors continue to occur and fluctuate year over year, although customers moving within the service territory would continue to produce savings. To be accurate, the value of this persistence for lifetime cost and cost-effectiveness calculations should adjust for attrition through the application of an additional deemed factor. At this time, we do not have sufficient data for such an adjustment and recommend further evaluation to develop appropriate values.

normalized. This input (to be provided by program evaluators) will approximate average savings for a standard weather year based upon historical data.²⁷

Program Year T - record 100% of adjusted annual savings as calculated above	Percent savings from Year T activities that persist 1 year after year T	Percent savings from Year T activities that persist 2 years after year T	Percent savings from Year T activities that persist 3 years after year T	Percent savings from Year T activities that persist 4 years after year T
$\Delta kWh_{TAdjusted}$ $\Delta kW_{TAdjusted}$ $\Delta Therms_{TAdjusted}$	$\Delta kWh_{TAdjusted} * PFE_1$ $\Delta kW_{TAdjusted} * PFE_1$ $\Delta Therms_{TAdjusted} * PFG_1$	$\Delta kWh_{TAdjusted} * PFE_2$ $\Delta kW_{TAdjusted} * PFE_2$ $\Delta Therms_{TAdjusted} * PFG_2$	$\Delta kWh_{TAdjusted} * PFE_3$ $\Delta kW_{TAdjusted} * PFE_3$ $\Delta Therms_{TAdjusted} * PFG_3$	$\Delta kWh_{TAdjusted} * PFE_4$ $\Delta kW_{TAdjusted} * PFE_4$ $\Delta Therms_{TAdjusted} * PFG_4$

²⁷ In the future, this approach could be empirically tested by comparing actual savings calculated in future program years against standard weather year results, producing a ‘realization rate’ between planned and actual savings results. Standard weather years could potentially be enhanced to better reflect these differences.

Example of Calculation of Cost-effectiveness Inputs – for Electric Savings:

Assume the same information as was used in the Example of Adjusted Annual Savings Calculations.

	Reporting Year T					
	2018	2019	2020	2021	2022	2023
Annual Savings = Adj. kWh savings (previously calculated) = $\Delta kWh_{T\text{Adjusted}}$	24,000,000	9,816,400	4,634,922	5,384,166	4,683,858	11,741,354

For calculating cost effectiveness in 2018:

Cost-effectiveness benefit of 2018 savings in 2019 = $\Delta kWh_{2018\text{ Adjusted}} * PFE_1 = 24,000,000 * 0.80 = 19,200,000 \text{ kWh}$

Cost-effectiveness benefit of 2018 savings in 2020 = $\Delta kWh_{2018\text{ Adjusted}} * PFE_2 = 24,000,000 * 0.54 = 12,960,000 \text{ kWh}$

Cost-effectiveness benefit of 2018 savings in 2021 = $\Delta kWh_{2018\text{ Adjusted}} * PFE_3 = 24,000,000 * 0.31 = 7,440,000 \text{ kWh}$

Cost-effectiveness benefit of 2018 savings in 2022 = $\Delta kWh_{2018\text{ Adjusted}} * PFE_4 = 24,000,000 * 0.15 = 3,600,000 \text{ kWh}$

For calculating cost effectiveness in 2019:

Cost-effectiveness benefit of 2019 savings in 2020 = $\Delta kWh_{2019\text{ Adjusted}} * PFE_1 = 9,816,400 * 0.80 = 7,853,120 \text{ kWh}$

Cost-effectiveness benefit of 2019 savings in 2021 = $\Delta kWh_{2019\text{ Adjusted}} * PFE_2 = 9,816,400 * 0.54 = 5,300,856 \text{ kWh}$

Cost-effectiveness benefit of 2019 savings in 2022 = $\Delta kWh_{2019\text{ Adjusted}} * PFE_3 = 9,816,400 * 0.31 = 3,043,084 \text{ kWh}$

Cost-effectiveness benefit of 2019 savings in 2023 = $\Delta kWh_{2019\text{ Adjusted}} * PFE_4 = 9,816,400 * 0.15 = 1,472,460 \text{ kWh}$

For calculating cost effectiveness in 2020:

Cost-effectiveness benefit of 2020 savings in 2021 = $\Delta kWh_{2020\text{ Adjusted}} * PFE_1 = 6,694,122 * 0.80 = 5,355,297 \text{ kWh}$

Cost-effectiveness benefit of 2020 savings in 2022 = $\Delta kWh_{2020\text{ Adjusted}} * PFE_2 = 6,694,122 * 0.54 = 3,614,826 \text{ kWh}$

Cost-effectiveness benefit of 2020 savings in 2023 = $\Delta kWh_{2020\text{ Adjusted}} * PFE_3 = 6,694,122 * 0.31 = 2,075,178 \text{ kWh}$

Cost-effectiveness benefit of 2020 savings in 2024 = $\Delta kWh_{2020\text{ Adjusted}} * PFE_4 = 6,694,122 * 0.15 = 1,004,118 \text{ kWh}$

Etc.

Apply the same approach to calculate cost-effectiveness inputs for kW and for Therms.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

REFERENCE TABLES

Persistence studies done to date for HERs-type programs capture effects only through a limited time frame and only for the specific program characteristics of the programs studied. They may not accurately represent conditions in Illinois or those for all Illinois programs. The Illinois TAC has determined that an average annual persistence rate across the studies done to date (Table 1 below) is the best currently available data to approximate persistence for the first year for the general class of residential HERs-type programs. Additional information about the rate of decay in the following years is limited. Most studies done to date that assess decay after more than one year do not specifically evaluate after each individual year and instead just calculate an average annual decay across the years studied. This is true of persistence studies for gas HERs-type programs. For them, this protocol assumes a linear on-going rate of decay for five years based on the average annual persistence in Table 1.

Navigant has recently undertaken an evaluation of the ComEd electric HERs program specifically designed to determine the first and second year persistence rate separately for each individual year. The results, shown in Table 2 below, indicate an average increase in the year-over-year persistence factor from year 1 to year 2 of 15%. This level of non-linear increase in the persistence factor is assumed to hold for the five years of electric savings persistence for HERs-type programs and is used to calculate persistence factors used in this protocol. The average annual persistence rate from Table 1 is used for the first year.

It is recommended that the persistence values and the shape of the decay function used in this protocol continue to be updated as further longer term and Illinois-specific evaluations are undertaken.

Table 1: Annual Persistence Rate for Residential HERs-type (RCT) Programs: Reference Studies							
Utility/Location	Frequency of Reports when in program	Number of Months in Program Before Terminated	Number of Post-Treatment Savings Analysis Months	Average Annual savings decay	Persistence (= 100% - decay)	Source	Electric or Gas
Upper Midwest	Monthly & quarterly	24-25	26	21%	79%	1	Electric
West Coast	Monthly & quarterly	24	29	18%	82%	1	Electric
West Coast	Monthly & quarterly	25-28	34	15%	85%	1	Electric
SMUD	Monthly & quarterly	27	12	32%	68%	1	Electric
Puget Sound Energy	Monthly & quarterly	24	36	11%	89%	1	Electric
MASS	Monthly & quarterly	26	15	33%	67%	2	Electric
Illinois (ComEd): First Year	Bimonthly	16-52	12	10%	90%	3	Electric
Average Annual Electric Savings Persistence:					80%		
MASS	Monthly & quarterly	15	17	64%	36%	2	Gas
Illinois (Nicor)	Bimonthly	12	12	46%	54%	4	Gas
Average Annual Gas Savings Persistence:					45%		

Sources:

- 1: http://www.cadmusgroup.com/wp-content/uploads/2014/11/Cadmus_Home_Energy_Reports_Winter2014.pdf
- 2: <http://ma-eeac.org/wordpress/wp-content/uploads/Home-Energy-Report-Savings-Decay-Analysis-Final-Report1.pdf>
- 3: http://ilsagfiles.org/SAG_files/Technical_Reference_Manual/Version_5/Sources/ComEd_HER_Opower_Persistence_and_Decay_Study_DRAFT_2016-01-28.pdf
- 4: http://ilsagfiles.org/SAG_files/Technical_Reference_Manual/Version_6/Evaluation_Documents/Nicor_Gas_HER_Persistence_Study_Part_2_Final_2016-09-21.pdf

Table 2: Year-over-Year Persistence Factors for ComEd Residential HERs Programs						
	Annual Persistence Factor				Implied Year-over-Year Persistence	Change in Year-over-Year Persistence
	Wave 1	Wave 3	Wave 5 Non-AMI	Average		
Year 1: 11/2013-10/2014	96%	98%	78%	90%	90%	
Year 2: 11/2014-10/2015	85%	83%	40%	69%	77%	15%

Source:

http://ilsagfiles.org/SAG_files/Evaluation_Documents/Draft%20Reports%20for%20Comment/ComEd_EPY7/ComEd_HER_Year_Two_Persistence_and_Decay_Study_2016-07-20_Draft.pdf This evaluation extends the analysis of the ComEd program waves reviewed in the 2016 study (#3 above) to the second year after reports were terminated. The study shows an increased rate of decay in year two, indicating that a linear decay rate assumption may not be accurate, at least for the first two years. This assessment of a non-linear decay rate will be reviewed, and the rate as it extends beyond the first two years, will be revisited when there have been additional studies designed to explicitly assess the shape of the decay curve across several years.

MEASURE CODE: CC-BEH-BEHP-V02-160601

REVIEW DEADLINE: 1/1/2021

Illinois Statewide Technical Reference Manual for Energy Efficiency Version 6.0

Attachment A

Illinois Statewide Net-to-Gross Methodologies

**Effective for Evaluation:
January 1st, 2018**

All NTG data collection and analysis activities for the program types covered by this document that start after the effective date, January 1, 2018, shall conform to the NTG methods set forth herein.

Attachment A: Illinois Statewide Net-to-Gross Methodologies

Policy Context for this Information

The Illinois Evaluation Teams (ADM Associates, Cadmus Group, Itron, Navigant Consulting, Opinion Dynamics, Ridge & Associates) are working with the Illinois Stakeholder Advisory Group (SAG) to create an Illinois Statewide Net-to-Gross (NTG) Methodologies document (IL-NTG Methods). The IL-NTG Methods document is included as an attachment to the Illinois Statewide Technical Reference Manual for Energy Efficiency (IL-TRM). Through five different dockets, the Illinois Commerce Commission (ICC) has directed the Evaluation Teams to compile and formalize standard NTG methods for use in Illinois energy efficiency (EE) evaluation, measurement, and verification (EM&V) work. The ICC EE dockets are shown in the following table.

Table 1-1. ICC Energy Efficiency Dockets

ICC Order Docket No. and Date	Program Administrator	NTG Discussion – Order Pages	ICC Link
13-0495 (1/28/14)	Commonwealth Edison Company (ComEd)	129-130	http://www.icc.illinois.gov/downloads/public/docket/367591.pdf
13-0498 (1/28/14)	Ameren Illinois Company (Ameren)	167, 171	http://www.icc.illinois.gov/downloads/public/docket/367603.pdf
13-0499 (1/28/14)	Illinois Department of Commerce & Economic Opportunity (Department of Commerce)	20, 23, 49	http://www.icc.illinois.gov/downloads/public/docket/367581.pdf
13-0549 (5/20/14)	Nicor Gas Company (Nicor)	41-42, 78	http://www.icc.illinois.gov/downloads/public/docket/378494.pdf
13-0550 (5/20/14)	North Shore Gas Company (North Shore Gas) and The Peoples Gas Light and Coke Company (Peoples Gas) (collectively, PG&NSG)	54-55, 66	http://www.icc.illinois.gov/downloads/public/docket/378495.pdf

To provide clarity to the ICC directives, the relevant section on IL-NTG Methods is shown in its entirety from the Nicor Gas Order (Docket No. 13-0549). The Nicor Gas Order provides the most detail on the ICC NTG directive in comparison to the other EE orders. The Nicor language is as follows:

The Commission believes that Staff’s recommendations concerning Commission adoption of consistent statewide net-to-gross methodologies (“IL-NTG Methods”) for use by the evaluators are reasonable and will aid in future evaluation of the energy efficiency programs. To help ensure the independence of the evaluators, to improve efficiency in the evaluation process, and to ensure programs across the state as delivered by the various Program Administrators can be meaningfully and consistently evaluated, the Commission hereby adopts Staff’s recommendation that consistent IL-NTG Methods be established for use in the evaluations of comparable energy efficiency programs offered by different Illinois Program Administrators. The Commission notes that Section 8-104(k) of the Act encourages statewide coordination and consistency between the gas and electric energy efficiency programs and Staff’s proposal would help ensure consistency in the evaluation of program performance. The Commission notes that this directive is not to create entirely “new” NTG methodologies for every energy efficiency program, but rather to assess NTG methodologies and survey instruments that have been used to evaluate energy efficiency programs offered in Illinois, and to compile the most justifiable and well-

vetted methodologies (or potentially combine certain components from the existing approaches to better represent the most justifiable and well-vetted method consistent with best practices) in an attachment to the Updated IL-TRM that would get submitted to the Commission for approval. The Commission notes that the IL-NTG Methods will be flexible and adaptable to multiple program designs and budgets and tailored to appropriately assess the specifics of each of the Program Administrators' energy efficiency programs, consistent with standard NTG methodologies adopted in other states that were filed in this proceeding. The Commission agrees with Staff that in the interest of efficiency, the current program evaluators should take the lead in compiling and formalizing standard methodologies for NTG in Illinois taking into consideration SAG input. Because the existing Plan 1 evaluators are under contract with the Company for the evaluation of the program year three energy efficiency programs, it is appropriate for these existing evaluators to work on and complete the compilation of the IL-NTG Methods over the next year. The Commission recognizes that each year considerable time may be spent vetting NTG methodologies for each program evaluation separately for each utility under the existing evaluation plan review practices; adoption of IL-NTG Methods would save on these limited evaluation resources by having a common reference document for the evaluators to use in estimating net savings for Illinois.

The Commission hereby directs the Company to require its evaluators to collaborate with the other Illinois evaluators and the SAG to use best efforts to reach consensus on the approaches used in assessing NTG in particular markets for both residential and non-residential energy efficiency programs in a manner consistent with the direction described herein. (Pages 41-42)

- (16) Northern Illinois Gas Company shall require its evaluators to collaborate with the other Illinois evaluators and the SAG to reach consensus on the most defensible and well-vetted methodologies for assessing net-to-gross ratios in particular markets for both residential and non-residential energy efficiency programs in a manner consistent with the direction provided herein;
- (17) ICC Staff shall file the agreed-upon consensus statewide NTG methodologies with the Commission as an attachment to the Updated IL-TRM, and if consensus is not reached on a certain component of the statewide NTG methodologies, that particular non-consensus component should be submitted in a manner consistent with the approach used for non-consensus IL-TRM Updates; (Page 78)

1.2 Programs Currently Covered in this Document

This document is intended to cover the majority of residential and non-residential programs offered in Illinois.²⁸ Programs covered as of the writing of this document are listed in tables at the beginning of Section 3: Commercial, Industrial, and Public Sector Protocols and Section 4: Residential and Low Income Sector Protocols. If the design of a given program changes significantly, then it may mean that the NTG protocol listed for that program in this document is no longer appropriate. If that happens, the evaluator should follow the procedures outlined below under Section 1.4: Diverging from the IL-NTG Methods.

This document will be updated over time to incorporate new programs and to reflect recommended changes to existing methodologies. All NTG data collection and analysis activities for the program types covered by this document that start after the effective date, January 1, 2018, shall conform to the NTG methods set forth herein.

²⁸ Evaluation reports on those programs can be found at <http://www.ilsag.info/evaluation-documents.html>.

1.3 Updating the IL-NTG Methods

This attachment is part of the IL-TRM and follows the timeline for updating of the IL-TRM, as specified in the Illinois Energy Efficiency Policy Manual. In general, the following will take place:

- Updates will occur annually.
- Any changes to the IL-NTG Methods document will be circulated to the full SAG, and SAG participants will have a ten business day review process.
- Updates will be discussed within the SAG and completed annually.
- The ICC Staff will then submit a Staff Report (with the consensus Updated IL-TRM attached) to the Commission with a request for expedited review and approval.

1.4 Diverging from the IL-NTG Methods

The NTG methods for the programs outlined in this document are partially binding. The criteria for deviating from the IL-NTG Methods document are set forth below. In all cases, the evaluators (or any interested stakeholder) submits the proposed deviation to the full SAG for a ten business day SAG review and comment period. In the event of an objection by a SAG participant, efforts may be made to see if consensus can be reached on the proposed deviation in a subsequent monthly SAG meeting. In this case, a final opportunity for SAG review and comment to the proposed deviation will be provided following the SAG meeting.

Evaluators may modify the approaches described in this document if the following three conditions have been satisfied:

1. Evaluators must explicate within the annual evaluation research plan (or another document) how specific items in the proposed modified NTG method will diverge from what is written in this document. Evaluators must justify why the divergence is appropriate.
2. Prior to the use of the modified NTG method for a particular program, evaluation teams must be in agreement on the use and execution of the modified NTG method.
3. Any objection from SAG participants regarding the proposed modified NTG method is resolved.

Evaluators may test alternative methods of estimating NTG for a particular program in addition to the NTG methods outlined in this document, if the following three conditions have been satisfied:

1. Evaluators must explicate within the annual evaluation research plan (or other document) the proposed alternative NTG method. Evaluators must explain why the proposed alternative NTG method might be superior to the NTG methods outlined in this document for the particular program. Evaluators must discuss the foundation for expecting that the proposed alternative NTG method is likely to produce meaningful results.
2. Prior to the use of the alternative NTG method for a particular program, evaluation teams must be in agreement on the key details of the approach for implementing the alternative NTG method.
3. Any objection from SAG participants regarding the proposed alternative NTG method gets resolved.

When performing alternative NTG methods for a particular program, the choice of methods may vary across the state. For example, if ComEd's evaluator chooses to test Methods 1 and 2 for a particular program, Ameren's and Department of Commerce's evaluators do not also have to perform Methods 1 and 2 for a similar program.

Several sections of this attachment provide example questions that can be used to collect the data required in the NTG algorithms. Adjustments to refine specific question wording, e.g., to better reflect the design of the evaluated program, do not constitute divergence from the IL-NTG Methods.

1.5 Procedure for Non-Consensus Items

Non-consensus items that arise during the development and updating of the IL-NTG Methods document will be handled in substantially the same way as non-consensus IL-TRM Updates are addressed. The approach to be used is as follows.

- Once the Illinois NTG Working Group²⁹ has progressed as far as they can on the methodology, and it has been found that there is non-consensus on a specific Net-to-Gross Methods topic or procedure, the Illinois NTG Working Group shall submit to the ICC Staff and the SAG's Technical Advisory Committee (TAC) a Comparison Exhibit of Non-Consensus Net-to-Gross Methods topics/procedures within two weeks after the Illinois NTG Working Group has failed to reach consensus. The TAC will then deliberate on the issue with a goal of reaching consensus.
- If consensus does not emerge in the TAC regarding a particular Net-to-Gross Methods topic or procedure, the Comparison Exhibit of Non-Consensus NTG Methods topics/procedures is then sent to the full SAG for their deliberations and input. The SAG provides a forum where experts on all sides of the contested issue can present their expert opinions in an effort to inform parties of the contested issue and to also facilitate consensus.
- If the full SAG is unable to reach consensus, the non-consensus item will be referred to the ICC for resolution at the time of the IL-TRM Update proceeding. After receipt of the Comparison Exhibit of Non-Consensus Net-to-Gross Methods topics/procedures, the ICC Staff will submit a Staff Report to the Commission to initiate a proceeding separate from the consensus IL-TRM Update proceeding to resolve the non-consensus Net-to-Gross Methods topics/procedures.

²⁹ The Illinois NTG Working Group consists primarily of the subset of Evaluators deliberating on NTG methodologies; however, any interested party may participate in the Illinois NTG Working Group.

2 Attribution in Energy Efficiency Programs in General

One of the most difficult aspects of evaluation, and not just within evaluation of energy efficiency programs, is attributing results to a program. Attribution provides credible evidence that there is a causal link between the program activities and the outcomes achieved by the program. Attribution research estimates the difference between the outcomes and those that would have occurred absent the program (i.e., the counterfactual). Put in research terms, evaluators must reject the null hypothesis of no causality through probabilistic statements (e.g., “strong evidence”; “high probability”). As such, it is important to realize that the concept of the counterfactual cannot be proven with certainty. So even though the NTG ratio is a single value, conceptually it is a probabilistic statement.³⁰ One of the main academics within evaluation stated that there is a “...total and inevitable absence of certain knowledge [arising] from the methods social scientists use” when assessing the counterfactual. (Shadish, et al., 2002) This statement is not about poor methods, but about the counterfactual itself. Because programs work with people and are usually not a laboratory experiment that can be replicated over and over³¹ to find out what actions people would have taken absent an intervention, one would need a time machine to take people back in time and not provide the program. Since time machines do not exist, evaluators have developed methods that approximate the counterfactual to the best of their ability.

2.1 Definitions

For energy efficiency programs, evaluators differentiate between savings at a “gross” and “net” level as described below in the short set of relevant definitions. These definitions are not all encompassing or meant to restrict evaluation in any way, but to provide context before additional detail is provided in later sections. Research to determine attribution occurs to allow for a better understanding of the net level of savings.

Table 2-1. Definitions

Concept	Term	Definition
Consumers	Nonparticipant	Any consumer who was eligible but did not participate in the subject efficiency program, in a given program year.
	Participant	A consumer who received a service offered through the subject efficiency program, in a given program year; also called program participant. The term “service” is used in this definition to suggest that the service can be a wide variety of inducements, including financial rebates, technical assistance, product installations, training, energy efficiency information, or other services, items, or conditions. Each evaluation plan should define “participant” as it applies to the specific evaluation.
Gross Impacts	Gross Impacts	The change in energy consumption and/or demand that results directly from program-related actions taken by participants in an energy efficiency program, regardless of why they participated.

³⁰ A probabilistic statement is not the same as the confidence and precision information calculated based on sampling theory.

³¹ However, a small number of program designs do lend themselves to experimental or quasi-experimental designs that allow for regression analysis of net impacts.

Concept	Term	Definition
Attribution of Impacts	Net Impacts	The change in energy consumption and/or demand that is attributable to a particular energy efficiency program. This change in energy use and/or demand may include, implicitly or explicitly, consideration of factors such as free ridership, participant and nonparticipant spillover, and induced market effects. These factors may be considered in how a baseline is defined (e.g., common practice) and/or in adjustments to gross savings values.
	Net-to-Gross Ratio	A factor representing net program savings divided by gross program savings that is applied to gross program impacts to convert them into net program impacts. The factor itself may be made up of a variety of factors that create differences between gross and net savings, commonly including free riders and spillover. The factor can be estimated and applied separately to either energy or demand savings. Note that the net-to-gross ratio (NTGR) = ((1-Free Ridership) + Participant Spillover + Nonparticipant Spillover).
	Core NTGR	1-Free Ridership
	Free Rider	A program participant who would have implemented the program's measures or practices in the absence of the program. Free riders can be: (1) total, in which the participant's activity would have completely replicated the program measure; (2) partial, in which the participant's activity would have partially replicated the program measure; or (3) deferred, in which the participant's activity would have partially or completely replicated the program measure, but at a future time.
	Spillover	<p>Reductions in energy consumption and/or demand caused by the presence of an energy efficiency program. There can be participant and/or nonparticipant spillover.</p> <p><i>Participant spillover</i> (PSO) is the additional energy savings that occur as a result of the program's influence when a program participant independently installs incremental energy efficiency measures or applies energy-saving practices after having participated in the energy efficiency program. Evaluated savings associated with Program Administrator Training programs will also be considered Participant spillover. There are several general categories of participant spillover:</p> <ul style="list-style-type: none"> • <i>Inside spillover</i> (ISO): Occurs when program participants implement additional program-induced energy efficiency measures at the program project site. • <i>Outside spillover</i> (OSO): Occurs when program participants implement program-induced efficiency measures at other sites within the Program Administrator's service territory at which program project measures were not implemented. • <i>Like spillover</i>: Occurs when program participants implement program-induced efficiency measures of the same type as those implemented through the program. Like spillover can occur at the program project sites (ISO) or at other sites within the Program Administrator's service territory (OSO). • <i>Unlike spillover</i>: Occurs when program participants implement

Concept	Term	Definition
		<p>program-induced efficiency measures of a different type from those implemented through the program. Unlike spillover can occur at the program project sites (ISO) or at other sites within the Program Administrator's service territory (OSO).</p> <p><i>Nonparticipant spillover (NPSO)</i> refers to energy savings that occur when a program nonparticipant installs energy efficiency measures or applies energy savings practices as a result of a program's influence.</p>
Markets	Market	The commercial activity (e.g., manufacturing, distributing, buying, and selling) associated with products and services that affect energy use.
	Market Effects	A change in the structure of a market or the behavior of participants in a market that is reflective of an increase (or decrease) in the adoption of energy efficient products, services, or practices and is causally related to market interventions (e.g., programs). Examples of market effects include increased levels of awareness of energy-efficient technologies among customers and suppliers, increased availability of energy-efficient technologies through retail channels, reduced prices for energy-efficient models, build-out of energy-efficient model lines, and—the end goal— increased market shares for energy-efficient goods, services, and design practices.
	Market Assessment	An analysis that provides an assessment of how and how well a specific market or market segment is functioning with respect to the definition of well-functioning markets or with respect to other specific policy objectives. A market assessment generally includes a characterization or description of the specific market or market segments, including a description of the types and number of buyers and sellers in the market, the key factors that influence the market, the type and number of transactions that occur on an annual basis, and the extent to which market participants consider energy efficiency an important part of these transactions. This analysis may also include an assessment of whether a market has been sufficiently transformed to justify a reduction or elimination of specific program interventions (or whether continued or even increased intervention is necessary). Market assessment can be blended with strategic planning analysis to produce recommended program designs or budgets. One particular kind of market assessment effort is a baseline study, or the characterization of a market before the commencement of a specific intervention in the market for the purpose of guiding the intervention and/or assessing its effectiveness later.

Sources: State and Local Energy Efficiency Action Network. 2012. Energy Efficiency Program Impact Evaluation Guide. Prepared by Steven R. Schiller, Schiller Consulting, Inc., www.seeaction.energy.gov; Violette and Rathbun 2014. The Uniform Methods Project: Methods for Determining Energy Efficiency Savings for Specific Measures, Chapter 23: Estimating Net Savings: Common Practices, <http://www.nrel.gov/docs/fy14osti/62678.pdf>.

2.2 Spillover-Specific Issues

Some issues related to spillover are applicable for both residential and non-residential programs and are discussed in this section.

2.2.1 Measure Costs

In order to facilitate analysis of program Total Resource Cost (TRC), estimates of the total incremental measure cost (IMC) at the program level must be developed. IMC values are available for most IL-TRM measures and can be summed to the program level. However, the IMC values for spillover measures could also be estimated and added to this total. The problem is that IMC values for spillover measures can be difficult to estimate. When the magnitude of the savings justifies the effort to estimate the total IMC for spillover measures, the following approaches should be used.

- In cases where the evaluator believes the spillover measure incremental costs are not materially different from the rebated measure incremental costs, the evaluator may multiply the IMC for the rebated measure by the spillover rate to derive the IMC for the spillover measure.
- In cases where the evaluator believes the spillover measure incremental costs are materially different from the installed measure incremental costs (e.g., installation of measures that have no efficiency levels), the evaluator should use the estimated incremental project costs as the IMC for the spillover measure.

Normally, the sample-based estimates of IMCs for spillover measures should be extrapolated to the program level using sample weights. Then the total IMCs for rebated measures and the total IMCs for spillover measures should be summed and used in the TRC calculation.

For measures characterized by the IL-TRM, measure effective useful life (EUL) estimates should be based on the IL-TRM. For measures not characterized by the IL-TRM, evaluator can use either the EUL for similar measures or best professional judgment. In either case, the evaluator must provide the rationale for their choices.

3 Commercial, Industrial, and Public Sector Protocols

The table below lists Illinois non-residential programs and the free ridership protocol applicable to each program.³² If the design of a given program changes significantly, then it may mean that the NTG protocol listed for that program in this document is no longer appropriate. If that happens, the evaluator should follow the procedures outlined in Section 1.4: Diverging from the IL-NTG Methods. Note that the Core Non-Residential Spillover protocol described in Section 3.2 is generally applicable to most of these programs.

Table 3-1. Commercial, Industrial, and Public Sector Programs

Program Administrator	Free Ridership Protocol	Program Name
Ameren Illinois	3.1 Core Non-Residential Protocol	C&I Custom
		C&I Standard – Core Program
		C&I Standard – Instant Incentives / Midstream
	3.3 Small Business Protocol	Small Business Direct Install
		Small Business Refrigeration
		C&I Standard – Online Store
		Public HVAC Optimization
		Private HVAC Optimization
		Small Commercial Lit Signage
		LED Linear Lighting
		Demand Based Ventilation Fan Control
	3.5 Study-Based Protocol	C&I Retro-Commissioning
ComEd	3.1 Core Non-Residential Protocol	BILD / Midstream
		Custom Incentive
		Savings through Efficient Products (STEP)
		Standard Incentive
	3.3 Small Business Protocol	Agricultural EE Program (CoAg)
		CLEAResult School DI
		DCV – Matrix Demand- Based Fan Control
		EE Technologies to Address Peak Load in Assisted Living and Senior Housing
		Luminaire Level Lighting Control
		Matrix K through 12 Private Schools DI
		Rural Small Business EE Kits
		Small Business Energy Services
		Small Commercial Lit Signage
		Small Commercial HVAC Tuneup
	3.4 C&I New Construction Protocol	C&I New Construction
		New Construction – Small Buildings
	3.5 Study-Based Protocol	Data Centers
		Enhanced Building Optimization Program
		Industrial Systems Optimization
		Retrocommissioning
		Strategic Energy Management

³² The “Free Ridership Protocol Name” in the second column of the table refers to the numbered sections in this document, e.g., “3.3 Small Business Protocol.”

Program Administrator	Free Ridership Protocol	Program Name
		Root3
	3.5 Study-Based Protocol or 5.1 Behavioral Protocol	Power TakeOff – Small Business MBCx
	5.1 Behavioral Protocol	Agentis C&I Behavioral Program
	NTG = 1	LED Streetlighting
Department of Commerce	3.1 Core Non-Residential Protocol	Public Sector Custom
		Public Sector Custom - CHP Component
		Public Sector Natural Gas Boiler Systems Efficiency
		Public Sector Standard
		Savings through Efficient Products
	3.4 C&I New Construction Protocol	Public Sector New Construction
	3.5 Study-Based Protocol	Public Sector Retro-Commissioning
	3.6 Technical Assistance Protocol	Energy Assessment and New Construction Design Assistance
		Performance Contracting
	5.2 Code Compliance Protocol	Building Energy Code Compliance
	3.1 Core Non-Residential Protocol	Prescriptive Rebates
		Large Business Custom
		Combined Heat-and-Power (CHP)
	3.3 Small Business Protocol	Small Business (Audit/ Direct Install)
		Prescriptive Rebates (Small Business)
		Small Business Custom
	3.4 C&I New Construction Protocol	New Construction
	3.5 Study-Based Protocol	Retro-Commissioning (RCx)
		Strategic Energy Management (SEM)
	4.6 Multifamily Protocol	Multifamily (Audit/ Direct Install) (Common Area)
		Prescriptive Rebates (Multifamily Common Area)
		Multifamily Custom (Common Area)
	5.2 Code Compliance Protocol	Code Compliance
Peoples Gas/ North Shore Gas	3.1 Core Non-Residential Protocol	C&I Custom
		C&I Direct Install
		C&I Prescriptive
	3.3 Small Business Protocol	SB Custom
		SB Direct Install & Assessment
		SB Partner Trade Ally
		SB Prescriptive
	3.4 C&I New Construction Protocol	C&I New Construction (Joint)
	3.5 Study-Based Protocol	C&I Gas Optimization
		MF Gas Optimization
		Retro-Commissioning (Joint)
All	5.2 Code Compliance Protocol	Statewide Codes Collaborative

3.1 Core Non-Residential Protocol

3.1.1 Core Non-Residential Free Ridership Protocol

Key considerations and guidelines for estimation of free ridership under this Core Non-Residential Free Ridership (FR) protocol are listed below:

- **Multiple Questions:** Evaluators will use program participant responses to multiple survey questions as inputs to the free ridership calculation algorithm. Evaluators will not use the response to a single question to establish a survey respondent as either a complete free rider or a complete non-free rider.
- **Program and Non-Program Factors:** Evaluators will administer survey questions to obtain respondent ratings on a numeric scale of the impact, influence, or importance on the decision to implement energy efficiency measures or take energy efficiency actions. A series of questions will focus on factors that the evaluator determines are a function of the program. Such program factors may, for instance, include the availability of the program incentive, technical assistance from program staff, program staff recommendations, Program Administrator marketing materials, and an endorsement or recommendation by a Program Administrator, account manager or program partner staff. Evaluators will also administer a series of questions to obtain respondent ratings, on a numeric scale of the impact, influence, or importance on the decision to implement energy efficiency measures, of different factors that the evaluator determines are not a function of the program. Such non-program factors may include, for example, previous experience with the measure, standard business or industry practice, and organizational policy or guidelines.
- **Vendor Recommendations:** Vendor recommendations may also be a program factor to the extent that such recommendations are a function of the program. Vendors include trade allies, contractors, distributors, suppliers, and other market actors involved in the selection and installation of program-incented equipment on behalf of the participant. The evaluator may administer survey questions to vendors to verify their involvement with participant projects and to obtain their ratings—on a numeric scale—of the impact, influence, or importance of the program on the decision to recommend the energy efficiency measures to the program participant.
- **Consistency Checks:** Evaluators should administer survey questions as checks on the consistency of responses associated with a core free ridership assessment methodology. Evaluators may also reference available quantitative and qualitative data, including consistency check data, to perform documented modifications to individual free ridership estimates resulting from the application of a core free ridership assessment methodology.
- **Quality Control Review:** For programs involving large, complex projects and decision-making, after all the survey data collection has been completed and preliminary NTGRs have been computed using the standard calculation procedures, a quality control review is completed. All quantitative and qualitative data is systematically and independently analyzed by a researcher who is familiar with the program, the individual site and the social science theory that underlies the decision maker survey instrument. They make an independent determination of whether the additional information justifies modifying the previously calculated NTGR score, and present any recommended modifications and their rationale in a well-organized manner, along with specific references to the supporting data. Circumstances that may justify a revision of the previously calculated NTGR score include: (1) significant inconsistencies exist between one of the scores that may lead to elimination of the score that is an outlier; (2) the emerging “story” from the qualitative data is in conflict with the quantitative data, thereby requiring a callback to the customer to resolve the inconsistency and a revision to the original scoring based on the new information; or (3) the entire set of results for an interview are inconsistent, the data are too disparate and would not be helped with a callback. In such cases, a recommendation is made to remove that sample point and replace it with a back-up point.

3.1.1.1 Core Free Ridership Scoring Algorithm

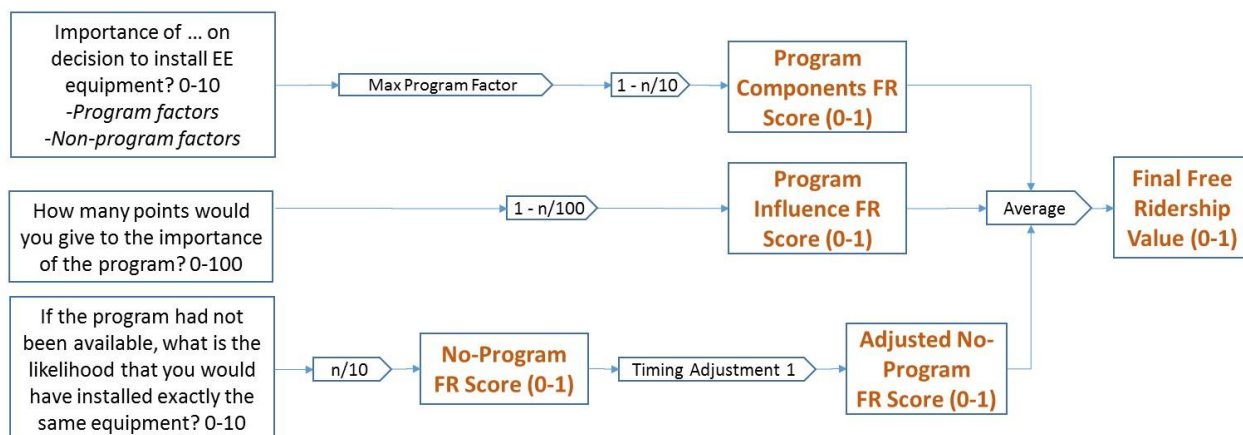
The Core Non-Residential FR protocol combines three scores that test different ways of approaching free ridership: the Program Components FR Score, the Program Influence FR Score, and the No-Program FR Score. The three scores are combined to calculate the final free ridership value.

Two options for combining the three scores are shown graphically in Figure 3-1 and Figure 3-2. These two options use different specifications to account for the impact of the program on project timing (referred to as “deferred free ridership”; see also discussion in Section 3.1.1.1.4). Evaluators will calculate free ridership using both options, and will select one option for purposes of calculating the annual incremental energy savings for comparing to the legislated goal.³³

Evaluators will submit participant survey and net savings analysis data to the Illinois NTG Working Group. The group will analyze these data for the purpose of further refining the protocol and potentially reducing the number of alternative algorithm input specifications.

Figure 3-1. Core Free Ridership Algorithm 1

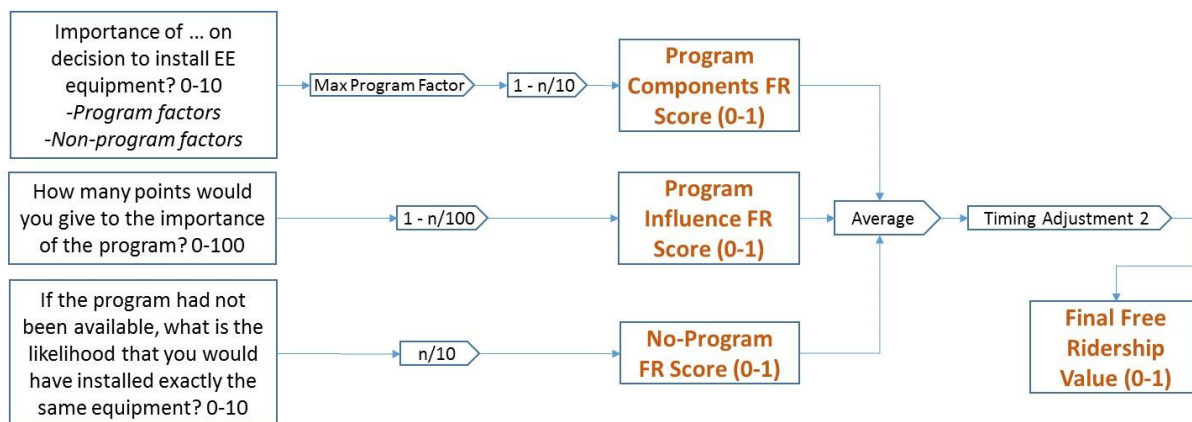
(Program Components FR Score + Program Influence FR Score + (No-Program FR Score * Timing Adjustment 1)) / 3



³³ As defined in 220 ILCS 5/8-103 and 220 ILCS 5/8-104.

Figure 3-2. Core Free Ridership Algorithm 2

$((\text{Program Components FR Score} + \text{Program Influence FR Score} + \text{No-Program FR Score}) / 3) * \text{Timing Adjustment 2}$



3.1.1.1.1 Program Components FR Score

Evaluators will administer survey questions to obtain participants' rating of the importance of various factors on the decision to implement energy efficiency measures. The numeric scales shall range from 0 to 10, where 0 means "not at all important" and 10 means "extremely important". The various factors referenced in the survey will include those that the evaluator determines are program factors and non-program factors that could potentially impact the participant decision making process. A participant rating shall be obtained for each relevant program and non-program factor.

Evaluators will calculate the "Program Components FR Score" for each survey respondent using the following equation:

$$\text{Program Components FR Score} = 1 - ([\text{Maximum Program Factor Rating}]/10).$$

These scores can range from 0 (no free ridership) to 1 (full free rider). Since the algorithm uses the numerical rating for the Program Component receiving the highest score, it is important that such scoring be accurate. To facilitate this, the scores feeding into the Program Components FR Score calculation can be enhanced by adjusting survey wording and adding consistency checks around specific program components to seek clarification on how they influenced decisionmaking. For those program components receiving scores of 8, 9 or 10, additional questions can be included to determine why that specific score was given, and further, how that Program Component specifically influenced the participant's decision to upgrade to energy efficient equipment.

Evaluation reports should list all factors considered program and non-program factors. Evaluators must document why factors were treated as program factors or non-program factors.

3.1.1.1.2 Program Influence FR Score

Evaluators will administer a survey question that asks respondents to quantify the importance of the program on the decision to implement energy efficiency measures relative to the importance or impact of non-program factors. Respondents will be asked to allocate a total of 100 points to the program and to non-program factors. The points allocated to the program by the participants are the "Program Points." Evaluators will calculate the "Program Influence FR Score" as $1 - (\text{Program Points}/100)$. This score can range from 0 (no free ridership) to 1 (full free rider).

3.1.1.1.3 No-Program FR Score

Evaluators will administer a counterfactual likelihood survey question to obtain respondent ratings on a 0 to 10-point numeric scale (where 0 means "not at all likely" and 10 means "extremely likely") of the likelihood of the respondent to implement the exact same energy efficiency measures in the absence of the program. Evaluators will calculate the "No-Program FR Score" as the numeric score of the likelihood of the respondent to implement

specified energy efficiency measures in the absence of the program divided by 10. This score can range from 0 (no free ridership) to 1 (full free rider).

Note that under one of the two deferred free ridership specifications (see next subsection), a timing adjustment is applied to the “No-Program FR Score.” Under this specification, the resulting score is referred to as the “Adjusted No-Program FR Score.”

3.1.1.1.4 Timing and Deferred Free Ridership

Evaluators will ask about the likely timing of measure installation in the absence of the program in two different ways. This is referred to as the counterfactual timing question since the evaluators are asking the respondent to speculate on what might have happened within a particular timeframe.

The first question will present a series of date ranges (e.g., within one year, between 12 months and 2 years, etc.) and ask the respondent to pick one representing their best estimate of when the measure would have been implemented in the absence of the program. The free ridership algorithm uses the midpoint of each date range, referred to as “Number of Months Expedited” below. For respondents that report accelerated adoption due to the program, this variable can take on values from 6 to 48 months.

The second question will prompt the respondent to use a 0 to 10-point numeric scale to report the likelihood, in the absence of the program, of implementing the same measure within 12 months of when it was actually implemented. This is the “Likelihood of Implementing within One Year” in the formulas below.

Evaluators will use the Likelihood of Implementing within One Year and/or the Number of Months Expedited variables to calculate two alternative ways of accounting for deferred free ridership:

- 1) Calculate *Timing Adjustment 1* as equal to:

$$1 - (\text{Number of Months Expedited} - 6)/42$$

Timing Adjustment 1 is multiplied by the No-Program FR Score; it can range from 0 (full deferred free ridership) to 1 (no deferred free ridership). The application of Timing Adjustment 1 is shown in Figure 3-1.

- 2) Calculate *Timing Adjustment 2* as equal to:

$$1 - ((\text{Number of Months Expedited} - 6)/42) * ((10 - \text{Likelihood of Implementing within One Year})/10)$$

Timing Adjustment 2 is multiplied by the average of the Program Components FR Score, the Program Influence FR Score, and the No-Program FR Score; it can range from 0 (full deferred free ridership) to 1 (no deferred free ridership). The application of Timing Adjustment 2 is shown in Figure 3-2.

How these timing adjustments are accounted for in the calculation of the Final FR Value is described below in the subsection “3.1.1.2 Construction of Core Free Ridership Value.”

3.1.1.1.5 Consistency Checks

Respondents may be asked one or more questions to facilitate understanding and potentially reconcile apparently inconsistent responses. Some questions may be asked of all respondents; others may be asked when previous answers appear inconsistent. Evaluators should report on the amount of inconsistency encountered and on the resolution to inform future protocol revisions. Three consistency checks are outlined below.

Program Influence/Program Components Consistency Check

A Program Influence/Program Components consistency check is triggered when the following conditions are met:

- 1) The number of Program Points (supporting calculation of the Program Influence FR Score) is greater than 70; and
- 2) No program factor is rated greater than 2.

A Program Influence/Program Components consistency check is also triggered by the following conditions being met:

- 1) The number of Program Points (supporting calculation of the Program Influence FR Score) is less than 30; and
- 2) At least one program factor is rated greater than 7. In this instance, the highest-rated program factor(s) with a rating of greater than 7 will be referenced in the consistency check question.

Program Components/No-Program Consistency Check

A Program Components/No-Program consistency check is triggered when the following conditions are met:

- 1) The likelihood of installing the exact same equipment without the program (supporting calculation of the No-Program FR Score) is greater than 7; and
- 2) At least one program factor is rated greater than 7.

A Program Components/No-Program consistency check is also triggered when the following conditions are met:

- 1) The likelihood of installing the exact same equipment without the program (supporting calculation of the No-Program FR Score) is less than 3; and
- 2) No program factor is rated greater than 2.

Timing of Installation Decision/Level of Program Attribution Consistency Check

The survey should contain a question to ask whether the respondent learned about the program after finalizing project specifications, including, where applicable, equipment efficiency level and number of units. The Timing of Installation Decision/Level of Program Attribution consistency check is triggered by the following conditions being met:

- 1) A respondent learned about the program after finalizing project specifications; and
- 2) Any of the following occur:
 - a) The number of Program Points (supporting calculation of the Program Influence FR Score) is greater than 70;
 - b) The likelihood of installing the exact same equipment without the program (supporting calculation of the No-Program FR Score) is less than 3; or
 - c) At least one program factor is rated greater than 7.

When the Timing of Installation Decision/Level of Program Attribution consistency check is administered, if the respondent rating of the importance of the vendor on the decision to implement the project is greater than 7, then an open-ended question will be triggered to obtain information regarding the role the vendor played in the participant decision to implement the project.

3.1.1.2 Construction of Core Free Ridership Value

This protocol designates two options of constructing the core free ridership value. Evaluators will calculate free ridership using both options and will select one option for purposes of calculating the annual incremental energy savings for comparing to the legislated goal. Evaluators will present the results of both estimates of free ridership in EM&V reporting.

Evaluators will calculate free ridership values in the following two ways:

- 1) Core FR Algorithm 1 = $\text{AVERAGE}([\text{Program Components FR Score}], [\text{Program Influence FR Score}], [\text{No-Program FR Score} \times \text{Timing Adjustment 1}])$
- 2) Core FR Algorithm 2 = $\text{AVERAGE}([\text{Program Components FR Score}], [\text{Program Influence FR Score}], [\text{No-Program FR Score}]) \times \text{Timing Adjustment 2}$

The two Core FR Algorithms listed above are graphically presented in Figure 3-1 and Figure 3-2, respectively.

3.1.1.3 Vendor Influence in the Free Ridership Calculation

3.1.1.3.1 Treatment of Participant's Rating of Vendor in the Program Components FR Score of the Core FR Algorithm

The Program Components FR Score of the participant Core FR algorithm is based on participant ratings of program and non-program factors. Vendors³⁴ often receive a high rating for their influence on the participant's decision to install the efficient measure. To implement the Core FR algorithm, the evaluator needs to decide whether the vendor rating should be considered a program factor or a non-program factor. This section outlines three scenarios for the treatment of the participant's rating of a vendor in the Program Components FR Score of the Core FR algorithm.

Scenario #1: Vendors are automatically considered a program factor

The vendor is considered a program factor in the calculation of the Program Components FR Score in the FR algorithm if the program meets specific criteria, which could include the following:

1. Trade allies are an integral component of program delivery, as supported by program logic
2. The trade ally network consists of a limited number of Program Administrator-selected, pre-approved trade allies
3. Only trade allies can implement projects and submit applications on behalf of the customer
4. Trade allies complete signed agreements with the Program Administrator
5. Trade allies complete program-sponsored training

In these cases, the vendor is automatically considered a program factor, and no additional input from the vendor is needed regarding the customer's decision-making process related to the project. The participant's influence rating for the vendor goes directly into the Program Components FR Score algorithm as a program factor (if it is the highest rating given to any program factor).

Scenario #2: Vendors are considered a program factor if the program influenced their recommendation to implement the efficient project

For programs that have a trade ally network, but do not meet the conditions under Scenario #1 above, follow-up interviews with vendors may be used to determine if the vendor should be considered a program factor. To qualify for Scenario #2, a program's trade ally network should meet the following conditions:

1. Trade allies are registered with the program
2. Trade allies typically complete signed agreements with the Program Administrator
3. Trade allies complete program-sponsored training
4. Trade allies drive program participation, as supported by program logic

In these cases, if the size of the project warrants a greater level of effort, a follow-up interview with the vendor may be used to determine if the participant's rating of the vendor's influence should be included as a program factor. A follow-up interview is triggered under the following conditions:

1. The participant rated the influence of the vendor as 8, 9, or 10 (on a scale from 0 to 10)
2. The rating the participant gave to vendor influence is higher than any of the program factor ratings

If completed, the interview should include the following questions:

FR1a On a scale of 0 to 10 where 0 is NOT AT ALL IMPORTANT and 10 is EXTREMELY IMPORTANT, how important was the <PROGRAM>, including incentives as well as program services and information, in influencing your decision to recommend that <CUSTOMER> install the energy efficient <MEASURE> at this time?

³⁴ Vendors include trade allies, contractors, distributors, suppliers, and other market actors involved in the selection and installation of program-incented equipment on behalf of the participant.

- FR1b On the same scale, how important was your firm’s past participation in an incentive or study-based program sponsored by <PROGRAM ADMINISTRATOR>?
- FR2 And using a 0 to 10 likelihood scale where 0 is NOT AT ALL LIKELY and 10 is EXTREMELY LIKELY, if the <PROGRAM>, including incentives as well as program services and information, had not been available, what is the likelihood that you would have recommended this specific <MEASURE> to <CUSTOMER>?
- FR3a Approximately, in what percent of projects did you recommend <MEASURE> BEFORE you learned about the <PROGRAM>?
- FR3b And approximately, in what percent of projects do you recommend <MEASURE> now that you have worked with the <PROGRAM>?

The interview will also include consistency checks, if the vendor provides inconsistent responses to these questions.

The vendor is viewed as a program factor and the rating the participant provided for the vendor goes into the Program Components FR Score algorithm as a program factor if, after consideration of any consistency checks:

1. The response to Q. FR1a or FR1b is 8, 9, or 10
- OR
2. The response to Q. FR2 is 0, 1, or 2
- OR
3. The difference between the responses to FR3b and FR3a is 80% or greater

If none of these conditions are met, the rating the participant provided for the vendor does not go into the Program Components FR Score algorithm as a program factor.

In the event that an interview is not completed (e.g., the size of the project did not warrant a vendor interview or the vendor could not be reached), the evaluation reports should explain how the rating the participant provided for the vendor was treated. Guidelines for these situations may be added to this document in the future.

Scenario #3: Vendors are considered a non-program factor

For programs that do NOT have a trade ally network that meets the conditions under Scenario #2, vendors are considered a non-program factor. In these cases, the participant’s rating of the vendor does not go directly into the Program Components FR Score algorithm as a program factor.

3.2 Core Non-Residential Spillover Protocol

Spillover refers to energy savings associated with energy-efficient equipment installed by consumers who were influenced by an energy efficiency program, but without direct intervention (e.g., financial or technical assistance) from the program.

To place the spillover protocols in context, we begin by defining the NTGR as:

$$\text{NTGR} = (1 - \text{Free Ridership Value} + \text{PSO Rate} + \text{NPSO Rate})$$

Where:

PSO Rate =	Participant spillover rate
NPSO Rate =	Nonparticipant spillover rate

The term (1-Free Ridership) is referred to as the Core NTGR for an efficiency program.

3.2.1 Core Participant Spillover Protocol

The Core Participant Spillover protocol is generally applicable to most commercial, industrial, and public sector

programs.

3.2.1.1 Research Methods

Data collection approach. An initial determination of participant spillover may be made based on self-reported findings from surveys of program participants. At a minimum, surveys collecting data pertaining to participant spillover will obtain general information on the specific measures installed and information substantiating their attribution to an energy efficiency program. Research on the specific characteristics of the energy efficient equipment installed and the baseline and operating conditions needed to estimate savings may be done in one of two ways: 1) a detailed battery of measure specific questions may be administered as part of the initial survey; or 2) a separate in-depth follow-up interview may be conducted by the engineer or analyst responsible for the energy savings calculation. In either case, an engineer or analyst will use the collected data to develop an estimate of spillover savings for each project.

Sample Frame. One target for participant spillover research may be the most recent year's program participants who have been sampled for free ridership or process surveys. In the case where a stand-alone spillover study is being conducted, the sample frame may be broader and include those whose participation occurred during the time period of two prior program years.

Because evaluated spillover energy impacts associated with the sample are being extrapolated to the program population, it is important that the sample frame be limited to participating customers for which spillover may potentially be claimed.

Sample frames should be constructed in accordance with the following guidelines:

- Self-directing customers as defined by 220 ILCS 5/8-104(m) should be excluded from the sample frame for natural gas spillover.
- Customers of municipal electric utilities should be excluded from the sample frame for electric spillover.

Timing of Data Collection. Evaluators may either administer the participant spillover module as part of a comprehensive net-to-gross survey, or they may elect to implement it separately. A follow-up in-depth interview may also be conducted by an engineer or analyst to obtain additional details needed to quantify savings. Optimally, the spillover inquiry should be timed in order to allow sufficient time for spillover to occur; at a minimum, three months after the program-incented measure is installed. Projects installed up to two years after program participation occurred may be counted as spillover, provided it can be substantiated.

3.2.1.2 Approach for Identifying and Quantifying Spillover

Attribution Criteria. Program attribution is determined by the responses to the following two survey questions:

1. How important was your experience in the <PROGRAM> in your decision to implement this measure, using a scale of 0 to 10, where 0 is not at all important and 10 is extremely important?
2. If you had not participated in the <PROGRAM>, how likely is it that your organization would still have implemented this measure, using a 0 to 10 scale, where 0 means you definitely WOULD NOT have implemented this measure and 10 means you definitely WOULD have implemented this measure?

The response to the first question cited above is "Measure Attribution Score 1," and the response to the second question cited above is "Measure Attribution Score 2."

There are two methods by which the attribution may be calculated:

1. Program attribution is established if the average of Measure Attribution Score 1 and (10 – Measure Attribution Score 2) exceeds 5.0³⁵; either the Measure Attribution Score 1 or (10 – Measure

³⁵ Note that the threshold value for counting spillover has been lowered from 7.0 to 5.0. The rationale for this lower threshold is: (1) the value of >5 is a strong indicator of program influence on the decision to install non-

Attribution Score 2) could be below 5.0—as long as the average is greater than 5.0, the threshold is met. If the average is greater than 5.0, 100% of the measure energy savings referenced in the question are considered to be attributable to the program. If the average is not greater than 5.0, none of the measure energy savings are considered to be attributable to the program.

2. An attribution rate may be calculated as equal to the sum of Measure Attribution Score 1 and (10 – Measure Attribution Score 2), divided by 20. For instance, if the attribution rate is 0.3, then 30% of the measure energy savings referenced in the question are considered to be attributable to the program.

Program attribution option 2 must be used in cases in which evaluators have performed the data collection and analysis required to attribute energy savings using option 2 identified above.

Calculation of Spillover Measure Energy Savings. Energy savings of spillover measures shall be calculated in one of two ways.

1. Those addressed in the IL-TRM shall be calculated in accordance with the methods and algorithms specified in the IL-TRM, and shall reference the IL-TRM-defined time-of-sale or new construction baseline.
2. For measures not addressed in the IL-TRM, evaluators shall quantify savings using accepted industry-wide savings methods that conform to IPMVP or other industry protocols and documents.

Evaluators will make every effort to ensure that there is no double-counting of participant spillover energy savings across multiple sources of participant and nonparticipant spillover (such as participating customer and trade ally surveys) and will document that effort.

Measure implementation must have occurred within one year of the participant spillover study data collection effort in order to be countable as participant spillover.

For the purposes of accounting for spillover savings attributable to a program, spillover will only be quantified for measures implemented within the Program Administrator’s service territory.

3.2.1.3 Key Participant Spillover Survey Questions

The Participant Spillover question module is designed to be a general inquiry that seeks to: (1) assess whether additional energy efficiency improvements were implemented since the rebated project was completed; (2) confirm that these measures either had not received program incentives, or that there were no plans to submit them for program incentives in the future; (3) gather basic information about the additional energy efficiency measures (e.g., their type, size, quantities, and energy efficiency rating); and (4) establish program attribution.

The basic question structure is shown below. The measure-specific questions can be repeated in order to capture multiple measures. Note that there is considerable flexibility to tailor the questions to specific types of applications and programs.

1. Since your participation in the <PROGRAM>, did you implement any ADDITIONAL energy efficiency improvements at this facility or at your other facilities within <PROGRAM ADMINISTRATOR>’s service territory that did NOT receive incentives through <PROGRAM>?
2. What measures did you implement without an incentive?

MEASURE-SPECIFIC QUESTIONS [repeated for each spillover measure]³⁶

rebated equipment and is currently being used in other states (e.g., California); (2) the previous value of >7 set an unreasonably high standard for demonstrating program influence on the decision to install non-rebated equipment; and (3) past IL evaluation data show that a threshold of >5 will improve spillover estimates as it provides a better approximation of partial spillover (i.e., where a portion of the savings for each measure installed outside the program gets credited as spillover based upon the program influence rating).

³⁶ Example questions to gather engineering information to support the calculation of spillover savings may be

1. How important was your experience in the <PROGRAM> in your decision to implement this <MEASUREX>? Please use a scale of 0 to 10, where 0 is not at all important and 10 is extremely important.
2. Can you explain how your experience with the <PROGRAM> influenced your decision to install this additional high-efficiency measure?
3. If you had not participated in the <PROGRAM>, how likely is it that your organization would still have implemented <MEASURE>? Please use a 0 to 10, scale where 0 means you definitely WOULD NOT have implemented this measure and 10 means you definitely WOULD have implemented this measure.
4. How many of <MEASURE> did you install?
5. Questions to further define the measure (as applicable):
 - a. Type
 - b. Efficiency
 - c. Size
 - d. Other attributes
6. Can you briefly explain why you decided to install this energy efficiency measure on your own, rather than going through the <PROGRAM>?

3.2.1.4 Reporting of Results

Evaluators will report the following information relating to participant spillover data collection and analysis in annual EM&V reporting: 1) the number of participants surveyed; 2) the number of survey respondents reporting spillover; 3) the number of survey respondents who meet the spillover attribution threshold; 4) the number of respondents for which spillover savings were actually quantified; 5) the spillover savings for each project and overall; and 6) the spillover rate. The term (1-Free Ridership) is referred to as the Core NTGR.

The annual EM&V report should also describe the means by which the participant spillover rate is calculated. Two possible approaches are:

- (1) Add the participant spillover rate to each project's Core NTGR. The project-level NTGRs are then weighted by each project's ex ante or ex post (if available) gross savings as a share of the total. This savings-weighted NTGR can then be applied to the ex post gross savings of the participant population. If the sample is stratified, sampling weights must be applied before applying the NTGR to the ex post gross savings of the participant population.
- (2) Estimate program spillover effects by summing overall project-level spillover estimates for the sample and dividing this sum by the total ex ante or ex post (if available) gross savings for the sample to produce the participant spillover rate. This participant spillover rate can be added to the Core NTGR for the sample to yield the NTGR. If the sample is stratified, sampling weights must be applied before applying the NTGR to the ex post gross savings of the participant population.

In both cases, the participant spillover rate must be calculated at the project level for Option 1 or at the program level for Option 2, using the following formula.

$$\text{Participant Spillover Rate} = \frac{\text{ISO} + \text{OSO in sample}}{\text{Ex Post Gross Impacts in sample}}$$

Where:

ISO = Inside participant spillover

accessed here: http://www.ilsag.info/il_ntg_methods.html

OSO = Outside participant spillover

3.2.2 Core Nonparticipant Spillover Protocol

The evaluation may perform research to measure nonparticipant spillover (NPSO). Evaluators will make efforts to ensure that there is no double-counting of energy savings across multiple sources and will document those efforts.

3.2.2.1 Core Nonparticipant Spillover Protocol – Measured from End Users

NPSO for end users is defined as the energy savings that are achieved when a nonparticipant end user—as a result of the influence of a Program Administrator’s programs—implements energy efficiency measures *outside* of the Program Administrator’s programs.

One option for the evaluator would be to survey nonparticipating customers and estimate spillover savings for any efficient measures installed that respondents are able to attribute to specific Program Administrator programs. However, in many cases, nonparticipants might find it difficult, if not impossible, to reliably attribute any of their installations to the influence of a specific Program Administrator program. If an evaluator suspects that nonresidential nonparticipants will not be able to reliably attribute spillover savings to any particular Program Administrator program, a second option would be to survey nonparticipants and estimate spillover savings from the installation of efficient measures that respondents are able to attribute to their general knowledge of the Program Administrator incentives and information, regardless of the particular program source. These protocols are written assuming that the NPSO for end users will be estimated using this second option.

Note that this protocol does not address estimating spillover for upstream and midstream programs where the end user is assumed to be completely ignorant of any Program Administrator influence. Of course, when considered feasible, evaluators are free to estimate spillover and spillover rates at the program-specific level with the suggested questions presented in Section 3.2.2.1.2 modified appropriately.

3.2.2.1.1 Research Methods

Data Collection Approach. An initial determination of spillover may be made based on self-reported findings from surveys of nonparticipants. At a minimum, surveys collecting data pertaining to nonparticipant spillover will obtain general information on the specific measures installed and information substantiating the influence of the Program Administrator on the installation decision. Research on the specific characteristics of the energy efficient equipment installed and the baseline and operating conditions needed to estimate savings may be done in one of two ways: (1) a detailed battery of measure specific questions may be administered as part of the initial survey, or (2) a separate in-depth follow-up interview may be conducted by the engineer or analyst responsible for the energy savings calculation.³⁷ Projects installed within the last two years of the nonparticipant spillover study data collection effort may be counted as spillover, provided program attribution and energy savings can be substantiated. In either case, an engineer or analyst will use the collected data to develop an estimate of spillover savings for each project.

Sample Frame. The sample frame for nonparticipant end user spillover research is composed of customers who have not participated in any programs within the last three years. Because evaluated spillover savings associated with the sample are being extrapolated to the nonparticipant population, it is important that the sample frame be limited to nonparticipants for whom spillover may potentially be claimed.

Sample frames should be constructed in accordance with the following guidelines:

³⁷ See http://www.ilsag.info/il_ntg_methods.html for detailed example questions designed to collect information required to estimate spillover savings for a variety of measures.

- Self-directing customers as defined by 220 ILCS 5/8-104(m) should be excluded from the sample frame for natural gas spillover.
- Customers of municipal electric utilities should be excluded from the sample frame for electric spillover.
- Entities eligible to participate in the Illinois Department of Commerce and Economic Opportunity programs will not be included in sample frames for the study of nonparticipant spillover attributable to utility-administered programs.
- Entities eligible to participate in the utilities' programs will not be included in sample frames for the study of nonparticipant spillover attributable to programs administered by the Department of Commerce and Economic Opportunity.

Timing of Data Collection. Evaluators might administer the nonparticipant end user spillover study in parallel with the program impact evaluation, potential study or saturation study research, or at a different time.

3.2.2.1.2 Approach for Identifying and Quantifying Spillover

Key Nonparticipant Spillover Survey Questions. The nonparticipant end user spillover question module is designed to be a general inquiry that seeks to: (1) assess whether additional energy efficiency improvements were implemented during the study period; (2) confirm that these measures had not received program incentives and that there were no plans to submit them for program incentives in the future; (3) gather basic information about the additional energy efficiency measure(s), e.g., the type, size, quantities, and energy efficiency rating; and (4) establish the Program Administrator importance ratings. Note that while the example questions can be customized to assess the influence of a specific program in the Program Administrator portfolio, they are currently worded to capture influence of the Program Administrator, regardless of program source.

Below are example questions that might be used in a nonparticipant spillover survey. They are grouped by the following topics:

- **Threshold conditions:** Is there some credible evidence that it was at least possible for the Program Administrator to have influenced the decision to install additional energy efficient measures?
- **Measure description:** Enough information needs to be collected for the measure and its operation to support a credible estimate of savings
- **Attribution:** Is there credible evidence that the Program Administrator had substantial influence on the end user's decision to install the efficient measure outside of any of the programs in the Program Administrator portfolio?

Threshold Conditions. Spillover cases are identified using a threshold approach in which certain minimal conditions must be met for a customer's installation to be considered for spillover. The following are example questions that evaluators may use (individually or in combination) to determine that program administrator influence on the installation is possible:

1. Before installing these measures, did you know that <PROGRAM ADMINISTRATOR> offers energy efficiency programs, incentives, and information to help their business customers make energy efficiency improvements at their facilities?
2. <PROGRAM ADMINISTRATOR> offers incentives for energy efficient equipment upgrades and improvements through its <PORTFOLIO NAME> programs. Before installing these measures, had you heard about the <PORTFOLIO NAME> programs?

If the answer to either question is "yes", then the threshold condition is met.

Measure Description. The interview (either the initial interview or a separate in-depth follow-up interview) can be used to determine the following basic attributes (as applicable) required to support a credible estimate of savings:

1. Type
2. Efficiency
3. Size
4. Other attributes

The named measure(s) must represent equipment that is more energy efficient than either: (1) equipment

required by codes or standards; (2) industry-standard practice for certain types of equipment; or (3) for Custom measures, the minimum efficiency equipment available to meet the customer's requirements. For detailed example questions designed to collect engineering information required to estimate spillover savings for a variety of measures, see http://www.ilsag.info/il_ntg_methods.html.

Attribution. The following questions are suggested to assess attribution. These questions should be asked separately for each potential spillover measure:

1. Earlier you mentioned that you knew that <PROGRAM ADMINISTRATOR> offers incentives to customers for installing energy efficient equipment, and also provides information to customers to help them reduce their energy usage. Thinking about all of the reasons you chose to install the energy efficient <MEASURE>, did your knowledge of these incentives and information available through <PROGRAM ADMINISTRATOR> have ANY INFLUENCE on your decision to install <MEASURE>?

ASK IF Q1=YES

2. Using a scale of 0 to 10, where 0 is not at all influential and 10 is extremely influential, how much influence did your knowledge of the incentives and information <PROGRAM ADMINISTRATOR> offers have on your decision to install your energy efficient <MEASURE>?
3. Just to make sure that we understand you correctly, please answer the following hypothetical question. If you had you NOT known about the incentives and information <PROGRAM ADMINISTRATOR> offers, would you still have installed your energy efficient <MEASURE>? Please use a scale of 0 to 10, where 0 means you definitely WOULD NOT have installed your energy efficient <MEASURE> and 10 means you definitely WOULD have done so.

Consistency Checks

Respondents may be asked one or more questions to facilitate understanding and potentially reconcile apparently inconsistent responses. Evaluators should report on the amount of inconsistency encountered and on the resolution to inform future protocol revisions.

ASK IF Q2>7 AND Q3>7 OR Q2<3 AND Q3<3

4. In your own words, can you explain HOW your knowledge of the incentives and information <PROGRAM ADMINISTRATOR> offers influenced your decision to purchase or install your energy efficient <MEASURE>?

The evaluation analyst will assess the response to this open ended question and its consistency with the other questions, and, if warranted based on clear additional information, they will adjust the score based on expert judgment. If an inconsistency exists and the open-ended response does not resolve the inconsistency, the respondent will be removed from the calculation. All instances of this occurring should be documented in the final report. Additional consistency checks, triggered and resolved within the survey with additional questions to participants, remain optional.

Nonparticipant End User Spillover Algorithm. The response to question #2 cited above is "Measure Attribution Score 1," and the response to question #3 cited above is "Measure Attribution Score 2."

There are two methods by which the attribution may be calculated:

1. Provided that the open-ended responses do not contradict influence of the Program Administrator, spillover is considered to be attributable to the Program Administrator if the average of the Measure Attribution Score 1 and $(10 - \text{Measure Attribution Score 2})$ exceeds 5.0³⁸; either the Measure Attribution

³⁸ Note that the same 5.0 threshold value is being used for both Participant and Nonparticipant Spillover.

Score 1 or (10 – Measure Attribution Score 2) could be below 5.0—as long as the average is greater than 5.0, the threshold is met. If the average is greater than 5.0, 100% of the measure energy savings referenced in the question are considered to be attributable to the Program Administrator. If the average is not greater than 5.0, none of the measure energy savings are considered to be attributable to the Program Administrator.

2. Provided that the open-ended responses do not contradict influence of the Program Administrator, the attribution rate is calculated as equal to the sum of Measure Attribution Score 1 and (10 – Measure Attribution Score 2), divided by 20. For instance, if the attribution rate is 0.3, then 30% of the measure energy savings referenced in the question are considered to be attributable to the Program Administrator.

Calculation of Spillover Measure Energy Savings. Energy savings of spillover measures shall be calculated in one of two ways.

1. Those addressed in the IL-TRM shall be calculated in accordance with the methods and algorithms specified in the IL-TRM, and shall reference the IL-TRM-defined time-of-sale or new construction baseline.
2. For measures not addressed in the IL-TRM, evaluators shall quantify savings using accepted industry-wide savings methods that conform to IPMVP and other industry protocols and documents.

Evaluators will make every effort to ensure that there is no double-counting of nonparticipant spillover energy savings across multiple sources of nonparticipant spillover reporting (such as nonparticipating customer and trade ally surveys) and will document that effort.

Measure implementation must have occurred within the last two years of the nonparticipant spillover study data collection effort in order to be countable as nonparticipant spillover.

For the purposes of accounting for spillover savings attributable to the Program Administrator, spillover will only be quantified for measures implemented within the Program Administrator’s service territory.

3.2.2.1.3 Reporting of Results

Evaluators will report the following information relating to nonparticipant spillover data collection and analysis in annual EM&V reporting: 1) how the sample frame was defined; 2) the number of customers surveyed; 3) the number of survey respondents reporting spillover; 4) the number of survey respondents who meet the spillover attribution threshold; 5) the number of respondents for which spillover savings were actually quantified; 6) the spillover savings for each project and overall; 7) the nonparticipant spillover rate, and 8) the calculation of the weights used to extrapolate the spillover to the population of nonparticipants from which the sample was drawn.

The EM&V report should also describe the means by which the nonparticipant spillover (NPSO) rate is calculated. For each sampled site, the verified spillover savings should be summed across measures to derive the total end user NPSO for the sampled sites.³⁹ The estimate of site-level end user NPSO for the entire sample is then extrapolated to the entire nonparticipant population using sampling weights.

There are two options for using the estimated NPSO.

1. Allocate the portfolio-level spillover savings to individual programs in the portfolio based on each program’s share of the ex post gross savings. For each program, the spillover rate could then be calculated for each program using the equation below in which the spillover allocated to each program would be the numerator and the ex post program-specific gross savings would be the denominator.

$$\text{Program – Specific NPSO Rate} = \frac{\text{NPSO}_{\text{Program-Specific}}}{\text{Ex Post Gross Impacts}_{\text{Program-Specific}}}$$

³⁹ This includes all samples sites including those that reported no spillover savings.

The spillover-adjusted NTGR for each program could then be used to adjust the Core NTGR for each program before calculating the TRC. In calculating the Program-Specific NPSO Rate, the numerator and denominator must be consistent in terms of the time period of measure implementation/potential implementation. While this time period must be within the last two years, it may be for a period of less than two years.

2. The NPSO Rate is calculated at the Sector level. The estimated energy savings associated with program-attributable spillover measures implemented during the study period by the entire nonparticipant population is divided by the ex post gross impacts for all the nonresidential programs in the portfolio occurring during the study period. The C&I Sector NPSO Rate is calculated using the following equation

$$\text{Portfolio NPSO Rate} = \frac{NPSO_{\text{portfolio}}}{\text{Ex Post Gross Impacts}_{\text{portfolio}}}$$

The NPSO rate could then be used to adjust the portfolio core NTGR before calculating the portfolio TRC. Again, in calculating the Portfolio NPSO Rate, the numerator and denominator must be consistent in terms of the time period of measure implementation/potential implementation. While this time period must be within the last two years, it may be for a period of less than two years.

3.3 Small Business Protocol

3.3.1 Free Ridership

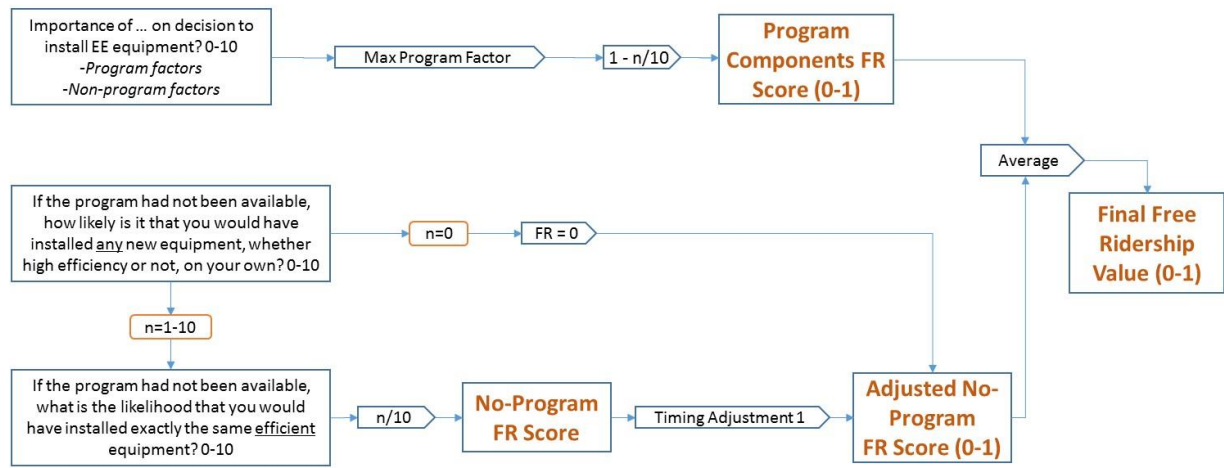
The FR algorithm for non-residential small business programs will follow the Core Non-Residential FR Protocol, with the following exceptions:

1. To reduce respondent burden, the Program Influence FR Score may be dropped from the Small Business FR algorithm. The influence of nonprogram factors will still be captured in the Program Components FR Score.
2. The counterfactual likelihood question (likelihood the participant would have installed the exact same energy efficiency equipment absent the program) may be preceded with a 0-10 scale question about the likelihood the participant would have installed any new equipment—either standard efficiency or high efficiency—on their own.
 - a. If the participant provides a likelihood response of 0, then the No-Program FR Score for that participant is set to 0.
 - b. If the participant provides a likelihood response of 1-10, then the participant is asked the same counterfactual questions (including the first timing question) as in the Core Non-Residential FR protocol.
3. To reduce respondent burden, the second question about timing (likelihood the participant would have installed the exact same energy efficiency equipment within 12 months) may be dropped. In this case, the only Deferred Free Ridership specification would be the one applying Timing Adjustment 1.

The diagram below, Figure 3-3, depicts the Small Business FR approach with the above exceptions implemented.

Figure 3-3. Small Business Free Ridership

$(\text{Program Components FR Score} + (\text{No-Program FR Score} * \text{Timing Adjustment 1})) / 2$



Evaluators will calculate free ridership values for small business projects as follows:

(1) If Program Influence FR Score is dropped:

$$FR = \text{AVERAGE} ([\text{Program Components FR Score}], [\text{No-Program FR Score} * \text{Timing Adjustment 1}])$$

(2) If Program Influence FR Score is included:

$$FR = \text{AVERAGE} ([\text{Program Components FR Score}], [\text{Program Influence FR Score}], [\text{No-Program FR Score} * \text{Timing Adjustment 1}])$$

3.4 C&I New Construction Protocol

3.4.1 Free Ridership

The FR algorithm for non-residential new construction programs will follow the Core Non-Residential FR protocol, with the following exception:

- The concept of project timing and deferred free ridership is not applicable to new construction projects.⁴⁰ As a result, the various deferred free ridership specifications outlined in Figure 3-1 and Figure 3-2 will not be included in the free ridership estimation for new construction projects.

Evaluators will calculate free ridership values for new construction projects as follows:

$$FR = \text{AVERAGE} ([\text{Program Components FR Score}], [\text{Program Influence FR Score}], [\text{No-Program FR Score}])$$

⁴⁰ New Construction programs intervene in the early phases of ongoing construction projects (i.e., after the decision to build has been made). As a result, participation in a New Construction program would not be expected to accelerate the construction of the new building.

3.5 Study-Based Protocol

3.5.1 Free Ridership

The FR algorithm for non-residential study-based programs (See Figure 3-4) will follow the Core Non-Residential FR protocol, with the following exceptions:

- The counterfactual likelihood question (Q.4 in Figure 3-5 and Figure 3-6, below) will be preceded by five questions.⁴¹
- Q.1 A 0-10 scale question about the likelihood that the participant would have conducted the study absent the program will be included.

At the measure-group level, the following should be included:

- Q.2a A yes/no question to determine if the participant performs regular maintenance on the equipment treated through the program
- Q.2b If the response to Q.2a is “yes,” a yes/no question to determine if the maintenance always includes the treatment provided through the program
- Q.3a A yes/no question to determine if the participant had prior awareness of the performance issues identified through the study
- Q.3b A 0-10 scale question about the participant’s level of familiarity with the recommended actions to rectify the performance issue.

The counterfactual likelihood question (Q.4 – likelihood the participant would have taken action absent the program) and the first counterfactual timing question (used to develop Timing Adjustment 1) will be asked at the measure-group level. Measure-group level responses will be aggregated to the project level, using savings-based weights.

There will be two options for developing the No-Program FR Score:

1. The measure-group level Adjusted No-Program FR Score will be developed following Algorithm 1 of the Core Non-Residential FR approach, using responses to the counterfactual likelihood question (Q.4) and Timing Adjustment 1.
2. The measure-group level No-Program FR Scores will be assigned, based on responses to Q.1, Q.2b, Q.3a, and Q.3b, as follows:
 - a. If Q.2b = Yes, then No-Program FR Score = 1. This assumes that if the participant performs regular maintenance on the treated equipment and that maintenance always includes the issue addressed through the program, then the participant is a full free rider for that measure group for purposes of calculating the No-Program FR Score.
 - b. If Q.3a = No and Q1 = 0 and Q.2b ≠ Yes, then No-Program FR Score = 0. This assumes that if the participant was not aware of the performance issue and had a zero likelihood of performing the study absent the program and their maintenance practices do not always include the issue addressed through the program, then the participant is not a free rider for that measure group for purposes of calculating the No-Program FR Score since they would not have found out about the issue absent the program.
 - c. If Q.3b = 0 and Q1 = 0 and Q.2b ≠ Yes, then No-Program FR Score = 0. This assumes that if the participant had no familiarity with how to rectify the performance issue, had a zero likelihood of

⁴¹ It should be noted that the question numbering in Figure 3-5 and Figure 3-6 is for reference purposes only; the additional questions do not have to immediately precede the counterfactual likelihood question.

performing the study absent the program, and their maintenance practices do not always include the issue addressed through the program, then the participant is not a free rider for that measure group for purposes of calculating the No-Program FR Score since they would not have known how to address the issue absent the program.

- d. For all other combinations of responses to Q.1, Q.2b, Q.3a, and Q.3b, the measure-group level Adjusted No-Program FR Scores will be developed following Algorithm 1 of the Core FR approach, using responses to the counterfactual likelihood question (Q.4) and Timing Adjustment 1.

Figure 3-4. Study-Based Free Ridership—Overview

(Program Components FR Score + Program Influence FR Score + (No-Program FR Score * Timing Adjustment 1)) / 3

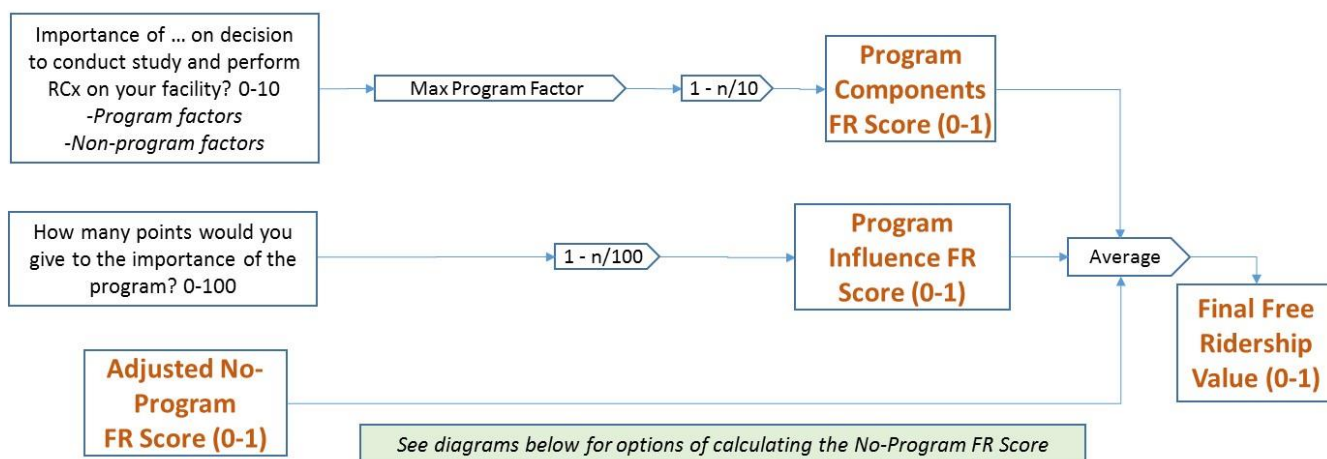


Figure 3-5. Study-Based Free Ridership—No-Program FR Score Option #1

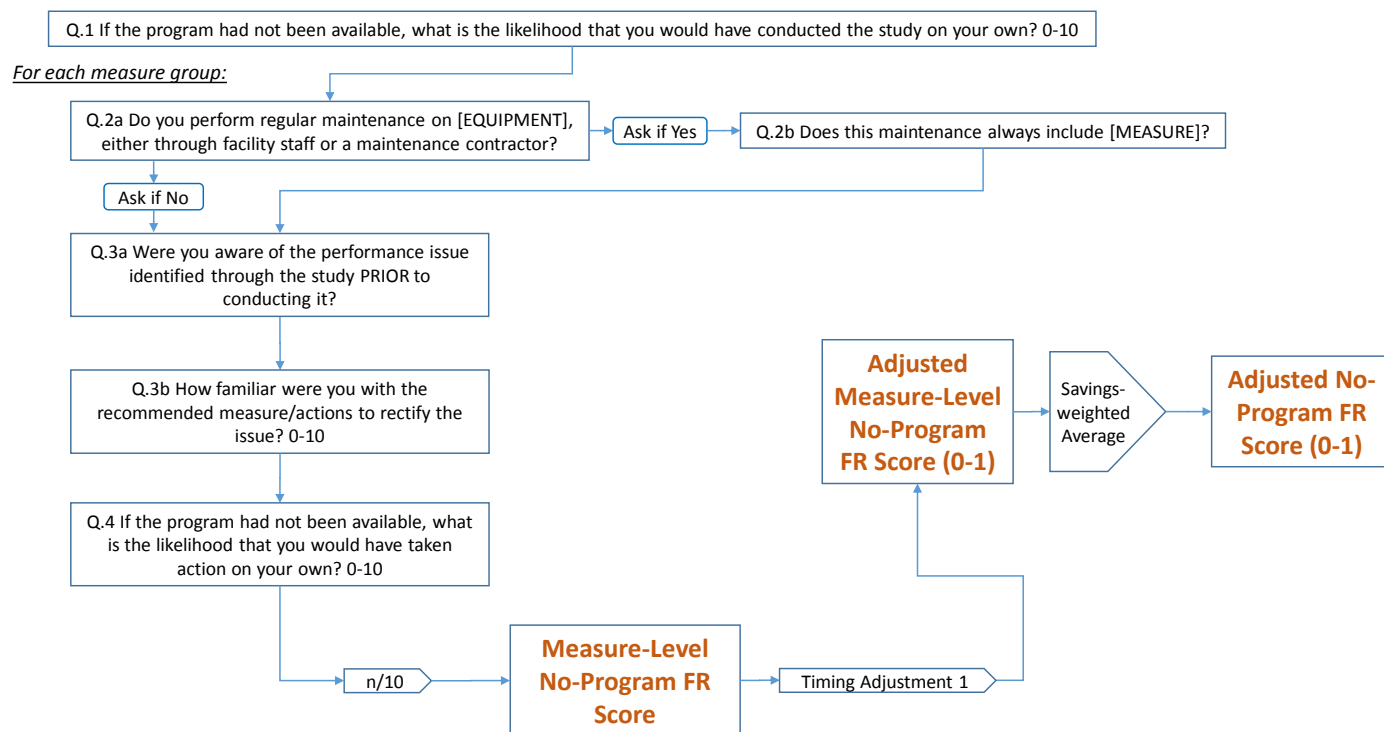
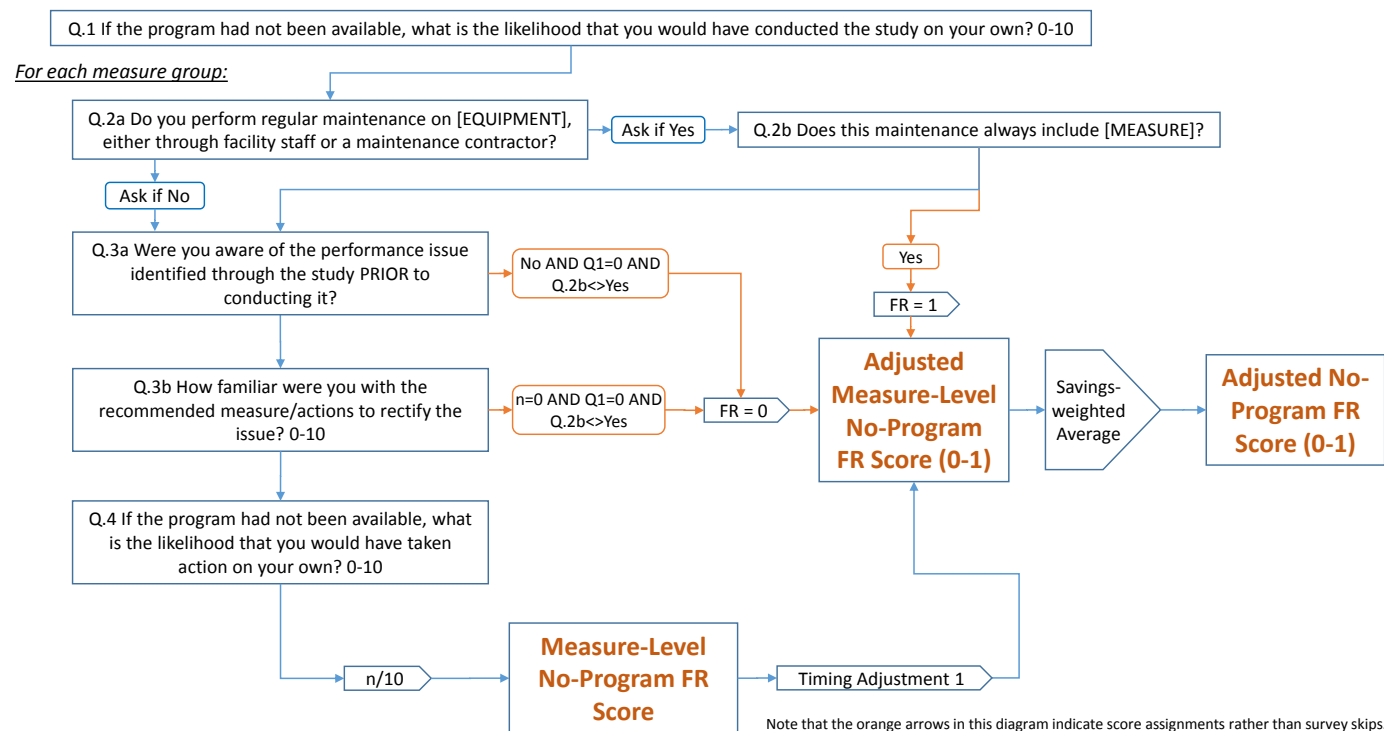


Figure 3-6. Study-Based Free Ridership—No-Program FR Score Option #2



Evaluators will calculate free ridership values for study-based programs as follows:

FR = AVERAGE ([Program Components FR Score], [Program Influence FR Score], [No-Program FR Score * Timing Adjustment 1])

Evaluators will develop estimates of free ridership based on the two No-Program FR Score options outlined above. Evaluators will select one of these for purposes of calculating the annual incremental energy savings for comparing to the legislated goal. Evaluators will present the results of both estimates of free ridership in EM&V reporting.

3.6 Technical Assistance Protocol

This protocol is applicable to programs that provide technical assistance to encourage the adoption of energy efficiency measures in non-residential facilities, but do not provide financial incentives.

Program-attributable savings from Technical assistance programs are achieved when a program participant—as a result of the program’s influence via the training or technical assistance provided—undertakes energy efficiency improvements on their own, without any direct financial assistance from any other Illinois energy efficiency program.

An initial determination of program-attributable savings is made based on self-reported findings from surveys of program participants. At a minimum, surveys collecting data pertaining to participant measure implementation will obtain general information on the specific measures installed and information substantiating their attribution to the program. Research on the specific characteristics of the energy-efficient equipment installed and the baseline and operating conditions needed to estimate savings may be done in one of two ways: 1) a detailed battery of measure specific questions may be administered as part of the initial survey; or 2) a separate in-depth follow-up interview may be conducted by the engineer or analyst responsible for the energy savings calculation. These collected data may be augmented by detailed facility and measure characteristics if provided by program staff.

3.6.1 Free Ridership

- The FR algorithm for Technical Assistance programs is identical to the Core Non-Residential FR protocol, with the following exception:
 - For the Program Components score, the list of program and non-program components differs extensively from conventional programs and therefore, is described in some detail here. As under the Core Protocol, evaluators administer survey questions to obtain participants’ rating of the importance of a comprehensive list of program and non-program factors on the decision to implement energy efficiency measures. Examples of Technical Assistance program factors that may be included are: Documentation in a program-provided technical report of the energy saving opportunities from installing the measure.
- Verbal information or guidance provided by a program representative or energy auditor during a training course or an on-site visit.
- A follow-up communication from the utility regarding implementing the recommendations provided through the audit, training or technical assistance.

Examples of Technical Assistance non-program factors that may be included are:

- Information from trade shows, conferences, or other professional gatherings
- Recommendation from an equipment vendor that sold you the measure and/or installed it
- Previous experience with the measure
- A recommendation from a design or consulting engineer
- Standard practice in your business/industry
- Corporate policy or guidelines
- Payback on the investment

4 Residential and Low Income Sector Protocols

The table below lists Illinois residential programs and the NTG protocol applicable to each program.⁴² If the design of a given program changes significantly, then it may mean that the NTG protocol listed for that program in this document is no longer appropriate. If that happens, the evaluator should follow the procedures outlined in Section 1.4: Diverging from the IL-NTG Methods.

Table 4-1. Residential and Low Income Programs

Program Administrator	Free Ridership Protocol	Program Name
Ameren Illinois	4.2 Appliance Recycling Protocol	Appliance Recycling
	4.3 Residential Upstream Lighting Protocol	Upstream Lighting
	4.4 Prescriptive Rebate (With No Audit) Protocol	Heating and Cooling
	4.5 Single-Family Home Energy Audit Protocol	All Electric Homes (Single Family)
		Home Performance with Energy Star
	4.6 Multifamily Protocol	All Electric Homes (Multifamily-Major Measures)
		Multifamily (In-Unit, Common Area and Major Measures)
	4.7 Energy Saving Kits and Elementary Education Protocol	Direct Mail Kits
		School Kits
	4.8 Residential New Construction Protocol	ENERGY STAR New Homes
ComEd	5.1 Behavioral Protocol	Behavior Modification
	NTG=1†	Moderate Income
	4.2 Appliance Recycling Protocol	Fridge and Freezer Recycling
	4.3 Residential Upstream Lighting Protocol	Lighting Discounts
	4.4 Prescriptive Rebate (With No Audit) Protocol	Appliance Rebates
		Heating and Cooling Rebates
		Weatherization Rebates
	4.5 Single-Family Home Energy Audit Protocol	Home Energy Assessments
	4.6 Multifamily Protocol	Elevate Energy Multifamily All Electric (In-Unit, Common Area and Major Measures)
		Residential Multifamily
	4.7 Energy Saving Kits and Elementary	Community-based CFL Distribution

⁴² The “Free Ridership Protocol Name” in the second column of the table refers to the numbered sections in this document, e.g., “4.6 Multifamily Protocol.”

Program Administrator	Free Ridership Protocol	Program Name
	Education Protocol	Direct to Consumers Kits
		NTC Middle School Take Home Kits
		SuperSavers – Elementary Education Kits
	4.8 Residential New Construction Protocol	Residential New Construction
	5.1 Behavioral Protocol	Cub Energy Saver
		Home Energy Reports (Opower)
		Power Smart Reports
		Shelton Solutions Great Energy Stewards
		Bidgely
		Meter Genius
		Smart Meter Connected Devices
		Weatherbug
		Nest Seasonal Savings
	NTG=1†	Low-Income Kit Energy (LIKE)
	NTG=1†	Low-Income Multi-Family
Department of Commerce	4.7 Energy Saving Kits and Elementary Education Protocol	Energy Smart Schools
	NTG=1†	Affordable Housing Construction Program
	NTG=1†	Public Housing Authority Efficient Living Program
	NTG=1†	Residential Retrofit (Low Income)
Nicor	4.4 Prescriptive Rebate (With No Audit) Protocol	Prescriptive Rebates (Single Family)
	4.5 Single-Family Home Energy Audit Protocol	Single Family (Audit/ Direct Install)
		Weatherization (Wx) Prescriptive
		Weatherization Projects (Moderate Income, Deep Retrofit)
	4.6 Multifamily Protocol	Multifamily (Audit/ Direct Install)
		Prescriptive Rebates (Multifamily In-Unit)
	4.7 Energy Saving Kits and Elementary Education Protocol	Elementary Education
		Energy Saving Kits
	4.8 Residential New Construction Protocol	New Construction
	5.1 Behavioral Protocol	Behavioral Energy Savings
	5.2 Code Compliance Protocol	Code Compliance

Program Administrator	Free Ridership Protocol	Program Name
Peoples Gas/ North Shore Gas	4.4 Prescriptive Rebate (With No Audit) Protocol	Home Energy Rebates
	4.5 Single-Family Home Energy Audit Protocol	Home Energy Jumpstart
	4.6 Multifamily Protocol	MF Custom
		MF Partner Trade Ally
		MF Prescriptive
		Multifamily (Direct Install)
	4.7 Energy Saving Kits and Elementary Education Protocol	Elementary Energy Education
	5.1 Behavioral Protocol	Home Energy Reports
All	5.2 Code Compliance Protocol	Statewide Codes Collaborative

† The Uniform Methods Project notes that “most low-income programs are not subject to NTG analysis (that is, are deemed at 1.0).” In line with that common practice, there is general consensus among Illinois stakeholders that the Illinois low-income programs should not be subject to NTG analysis and thus the NTG ratios for low-income programs are effectively deemed at 1.0. See Violette and Rathbun (2014), Chapter 23: Estimating Net Savings: Common Practices. The Uniform Methods Project: Methods for Determining Energy Efficiency Savings for Specific Measures, available electronically at <http://www.nrel.gov/docs/fy14osti/62678.pdf>, p. 50.

4.1 Residential Cross-Cutting Approaches

The approaches in this section can apply to more than one program type but do not supersede program-specific approaches presented in later sections.

4.1.1 Survey Design Issues

Free ridership questions should be asked near the beginning of a participant survey, before asking satisfaction questions. This should prevent participants from confusing free ridership questions with the satisfaction questions, which could influence free ridership scores.

4.1.2 Participant Spillover

Effective program marketing and outreach generates program participation and increases general energy efficiency awareness among customers. Spillover can be calculated using participant survey questions, which ask participants about energy-savings actions they have taken on their own since participating in the program. Questions should be sufficiently specific to ensure energy savings associated with spillover can be reasonably well-quantified. These may include questions about measure types or measures installed, quantities, and efficiency levels. When program implementers provide recommendations to participants and can provide data on the types of recommendations made to specific participants, evaluations should attempt to determine whether participants took the recommended actions outside of the program at sites within the program administrator’s service territory; if so, savings from those recommended actions should be attributed to the program.

To reduce the respondent’s burden, the survey should first ask participants about the influence the program had on their decision to take additional energy-saving actions on their own. In particular, the evaluation team should ask two close-ended questions to determine program influence on spillover actions. The two required questions, preceded by an optional open-ended warm-up question, are:

- OPTIONAL: Did the program influence you in any way to make these additional improvements?

1. How important was your participation in the <PROGRAM ADMINISTRATOR'S> program on your decision to make additional energy efficiency improvements on your own? [Scale from 0-10 where 0 is "not at all important" and 10 is "extremely important"]
2. If you had not participated in the <PROGRAM ADMINISTRATOR'S> program, how likely is it that you would still have implemented this measure, using a 0 to 10, scale where 0 means you definitely WOULD NOT have implemented this measure and 10 means you definitely WOULD have implemented this measure?

The response to the first required question cited above is "Measure Attribution Score 1," and the response to the second required question cited above is "Measure Attribution Score 2." The specific measures referenced in the question are considered to be attributable to the program if the "Spillover Score" is greater than 5.0:

$$\text{Spillover Score} = (\text{Measure Attribution Score 1} + (10 - \text{Measure Attribution Score 2}))/2 > 5.0$$

If these conditions are met, the evaluator determines that the specific measures referenced in the question are attributable to the program; otherwise, the evaluator determines that the specific measures referenced in the question are not attributable to the program. The attribution criterion represents a threshold approach, in which energy impacts associated with measures implemented by program participants outside the program are either 100% program-attributable or 0% program-attributable.

For each measure mentioned, customers will be asked how they know the measure is more efficient than other models. If the respondent can identify the measure as ENERGY STAR or name an efficiency level that the evaluator confirms as being above the minimum federal standard, or if they identify a technology that the evaluator can confirm is above the minimum federal standard, it will count towards Participant Spillover.

Finally, depending on the measure type cited by the customer, follow-up questions should ask customers to provide reasonable information to allow the evaluator to estimate the amount of savings using IL-TRM protocols, such as quantity of appliances or the location and amount of insulation.

To calculate the spillover energy and demand savings for these actions, the appropriate version of the IL-TRM should be used. To develop the spillover rate, the total energy and demand impacts from the sampled participants who installed additional measures due to participation in the program are summed, and then this sum is divided by the total ex post sample energy and demand impacts:

$$\text{Participant Spillover Rate (PSO)} = \frac{\text{Sum of Energy or Demand from Additional Measures Installed}}{\text{Sample Ex Post Gross Energy or Demand Impacts}}$$

The equation used to adjust the Core NTGR based on participant spillover is as follows:

$$\text{NTGR} = (1 - \text{FR} + \text{PSO})$$

4.1.2.1 Data Collection

Respondents should be drawn from a random sample of current or up to one year of previous program participants. Regardless of the participation year, spillover should be measured within the last 12 months (from the survey date), but after previous participation; the tracking database should supply this information.

4.1.2.2 Data Analysis

The following four steps calculate spillover:

1. Calculate total spillover savings for each participant installing an efficient measure not rebated through the program where the Spillover Score is greater than 5.0:

$$\text{Measure Spillover} = \text{Measure Savings} * \text{Number of Units}$$

2. Total savings associated with each program participant to calculate overall participant spillover savings.

3.
$$\text{Spillover Percentage Estimate} = \frac{\sum \text{Sample Spillover kWh Savings}}{\text{Sample Evaluated Program kWh Savings}}$$

4.1.3 Nonparticipant Spillover Measured Through Trade Allies

In addition to participant free ridership and spillover, residential programs may create nonparticipant spillover (NPSO) through trade allies exposed to the program but not actually facilitating program participation. Rather, they promote and stock higher-efficiency equipment due to the program.⁴³ NPSO caused by trade allies can be determined by surveying three groups of trade allies:

- Participating trade allies that do not submit rebates or otherwise act as program agents on behalf of their customers. For this group, care should be taken to ensure spillover is not double-counted with program sales.
- “Drop out” trade allies, who participated in the program previously but have not participated in the past 12 months.
- True nonparticipating trade allies that report they were aware of the program but had never participated.

Surveys ask nonparticipating trade allies if the program influenced their sales of high-efficiency equipment to nonparticipating customers and to quantify the program’s impact on their high-efficiency sales. The general questions take the following form:

- Q.1: How many <measures> did you sell in <period>?
- Q.2: How many of them were <efficiency level> or higher?
- Q.3: Had the <program> not existed, how many <measures> do you think you would have sold?

Evaluators should ensure that trade allies receive sufficient time to collect specific data and not rely on “guesses” to respond. Additional questions should be included to document how the program influenced sales of additional measures. Responses should also clarify whether sales counts are specific to the utility service territory in question.

The following steps calculate the program’s nonparticipant trade ally spillover percentage:

1. Compute the difference between the total reported number of high-efficiency units sold and the total that would have been sold in the program’s absence to obtain the total number of spillover units for that trade ally.
2. Multiply the total net number of spillover units of each measure sold by each surveyed trade ally by the average gross unit savings for each measure type.
3. Sum the result for each contractor from the previous step, and weight the results by the ratio of the population of non-active trade allies to the sample to compute the total spillover energy over the program period.
4. Divide the spillover energy savings by program gross savings.

⁴³ NPSO also can arise from nonparticipating customers as a direct result of general energy efficiency education and promotion efforts. A separate protocol addresses such NPSO. Care should be taken to ensure the different approaches do not double-count NPSO.

4.1.4 Nonparticipant Spillover Measured from Customers

The evaluation may perform research to measure nonparticipant spillover (NPSO). If so, care should be taken to ensure spillover is not double-counted with a trade-ally approach. The basic method uses a two-step process: (1) conduct a nonparticipant survey to identify potential spillover measures and (2) if needed, conduct a follow-up call or on-site visit by technical staff to confirm attribution and obtain information needed to estimate energy savings.

4.1.4.1 Basic Method

4.1.4.1.1 Sampling

As spillover may be rare in the nonparticipating population, determining spillover will likely require a large sample of customers who have not participated in any energy efficiency programs, including a behavioral program, within the past three years. Customers will be removed from the sample frame if their account numbers can be cross-referenced against a list of program participants from the previous three years. The survey should target household members responsible for paying utility bills. Survey respondents will be asked a screening question (whether they have participated in a program in the past three years) to confirm their household qualifies as a true nonparticipant.

4.1.4.1.2 Measure-Specific Questions

Depending on the spillover measure type reported by the customer, follow-up questions should be included to gather sufficient information to reasonably assess the saving amount by applying the IL-TRM, understanding that assumptions must be made if IL-TRM inputs cannot be easily supplied by the participant. Such assumptions should be conservative, or, if not conservative, reasons for deviating from the conservative application should be documented. Measures that cannot be reasonably quantified within available evaluation budgets should be excluded from spillover calculations.

For measures included in the IL-TRM, savings will be assessed using the IL-TRM algorithms. Baselines for measures not in the IL-TRM will be assessed based on appliance standards and building codes, if applicable, and, if not, through engineering judgements of existing or market conditions. Engineering assumptions and analysis by the evaluator will be applied for measures not included in the IL-TRM. Key assumptions should be documented in the report.

4.1.4.2 Attribution Approach

To receive credit for energy savings, the nonparticipant must fit the following criteria: (1) be familiar with the Program Administrator's energy efficiency campaign (e.g., ActOnEnergy for Ameren); and (2) indicate that some aspect of the Program Administrator's energy efficiency programs motivated their purchasing decisions. Influence will be measured on a scale of 0 to 10, where 10 is extremely influential and 0 is not at all influential. Savings attribution requires a Spillover Score of greater than 5.0.

Survey respondents will be asked a series of questions following the logic shown in Figure 4-1. First, the customer will indicate whether they know about their Program Administrator's energy efficiency programs and/or marketing messages. If customer is aware, the survey will ask if they or anyone in their household made an energy efficiency improvement within the last year, and if so, what improvements they made. Responses to these questions will generate a list of potential spillover measures (shown at point "[A]" in Figure 4-1). Customers will be asked how they know the measure is more efficient than other models. If the respondent can identify the measure as ENERGY STAR or name an efficiency level that the evaluator confirms as being above the minimum federal standard, or if they identify a technology that the evaluator can confirm is above the minimum federal standard, it will count towards NPSO. At this point in the NPSO process, the customer could be referred for a follow-up call with a technical interviewer.⁴⁴

⁴⁴ Customers who installed efficient lighting (CFL/LED) will not be eligible for NPSO if those savings are already

To assess attribution for each spillover measure mentioned, the customer will be asked questions to be scored in two areas. Spillover may be program-attributable for those measures for which self-report data meet the following threshold condition:

$$\text{Spillover Score} = (\text{Attribution Score 1} + (10 - \text{Attribution Score 2}))/2 > 5.0$$

4.1.4.2.1 Attribution Score 1

The first score, “Attribution Score 1,” measures the influence level (on a scale of 0 to 10, where 10 is extremely influential and 0 is not at all influential) their Program Administrator had on the decision to purchase the measure.

Influence can derive from the following:

1. General information about energy efficiency provided by the Program Administrator (e.g. through a bill insert)
2. Information from a contractor or retailer related to the Program Administrator’s programs.
3. Word-of-mouth from people installing energy-efficient equipment and receiving a rebate from the Program Administrator.

Attribution Score 1 is the maximum score (or Yes response) assigned to any source of influence from the Program Administrator.

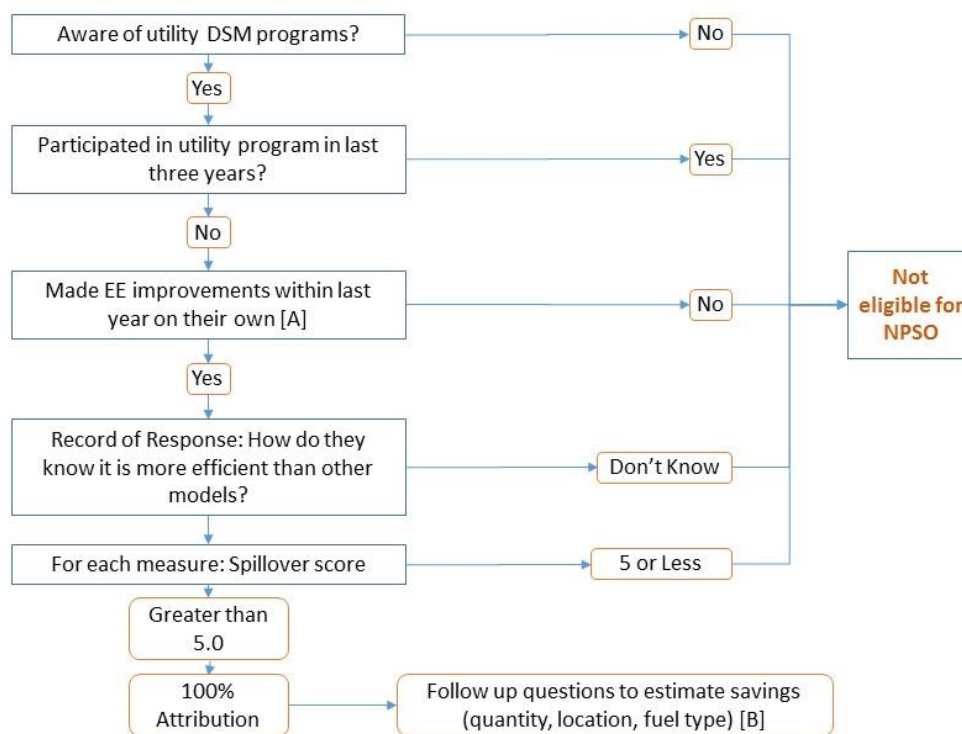
4.1.4.2.2 Attribution Score 2

The second score, “Attribution Score 2,” comes from the customer’s response to a single question to assess the counterfactual, asking about the likelihood (on a scale of 0 to 10, where 10 is extremely likely and 0 is not at all likely) that the customer would have installed the measure had they not been influenced by the program.

The Spillover Score is then the average of the Attribution Score 1 and (10 – Attribution Score 2). If that Spillover Score is greater than 5.0, 100% of the savings are attributed to the Program Administrator for that measure.

Finally, depending on the measure type cited by the customer, follow-up questions will gather information to enable an estimate of savings (shown in the figure as [B]), such as quantity of appliances or the location of insulation.

claimed by an upstream lighting program. A separate NPSO protocol is provided specifically for upstream lighting programs.

Figure 4-1. NPSO Question Logic

4.1.4.3 Scoring

Survey respondents' answers to the NPSO questions will determine total energy and demand savings attributed to the program. Table 4-2 lists NPSO measures under column A, the Spillover Score under column B, the estimated measure savings under column C, the percentage of allocated savings under column D, and the total allocated savings under column E. Column F shows the calculated average energy savings per spillover measure, determined by dividing the total allocated savings (the sum of column E) by the number of surveyed nonparticipating customers. The table shows how kWh NPSO savings would be calculated; calculations of therm or demand savings would be accomplished in the same manner.

Table 4-2. Estimation of Respondents' NPSO Savings

A	B	C	D	E	F
Spillover Measure	Spillover Score	Measure Savings (kWh)	Allocated Savings	Total kWh Savings	Average kWh Per Surveyed Customer
Measure1	Scale of 0 to 10	Savings1	100% if [B] > 5.0 0% if [B] ≤ 5.0	[C] x [D]	N/A
Measure2	Scale of 0 to 10	Savings2		[C] x [D]	
MeasureN	Scale of 0 to 10	SavingsN		[C] x [D]	
				Sum of column E = Total kWh Savings	Total kWh Savings ÷ Number of Completed Surveys

Table 4-3 shows the process for estimating total NPSO generated by the Program Administrator during the

program year (for electric savings). The savings attributed from the survey population will be extrapolated to the nonparticipating residential customer population to determine the overall NPSO savings. Then NPSO energy savings will be converted into a percentage using the total evaluated electric savings for the program year. A similar process would apply for calculating therm or demand NPSO.

Table 4-3. Calculation of Total NPSO Generated

Variable	Description	Source/Calculation
F	Average kWh Energy Savings per Surveyed Customer	Survey data and Savings Calculation
J	Total Nonparticipating Residential Population	Customer database
K	NPSO MWh Energy Savings Extrapolated to Nonparticipating Population	$[F \times J] \div 1,000 \text{ kWh/MWh}$
S	Total Evaluated MWh Savings	Residential Portfolio Savings
G	NPSO Spillover Rate	$K \div S$

4.2 Appliance Recycling Protocol

Appliance recycling programs (ARPs) typically offer some mix of incentives and free pickups for the removal of old but operable refrigerators, freezers, or room air conditioners. These programs encourage consumers to undertake the following:

- Discontinue use of secondary or inefficient appliances;
- Relinquish appliances previously used as primary units upon their replacement (rather than keeping the old appliance as a secondary unit); and
- Prevent the continued use of old appliances in other households through direct transfers (i.e., giving it away or selling it) or indirect transfers (resale in the used appliance market).

As the program theory and logic for appliance recycling differ significantly from standard “downstream” incentive programs (which typically offer rebates for purchases of efficient products), the free ridership estimation approach also significantly differs.

The basic and enhanced methods are described next.

4.2.1 Basic Method

4.2.1.1 Free Ridership

Free ridership is based on participants’ anticipated plans had the program not been available, thus classifying a free rider as a participant who would have removed the unit from service regardless of the program.

Estimating net savings for ARPs should adopt a multistep process to segment participants into different groups, each with specific attributable savings.

In general, independent of program intervention, participating appliances would have been subject to one of the following options:

1. The appliance would have been kept by the participating household.
2. The appliance would have been discarded in a way that transfers the unit to another customer for continued use.
3. The appliance would have been discarded in a way that would have permanently removed the unit from service.

Only Option 3 constitutes free ridership (the proportion of units that would have been taken off the grid absent the program). Options 1 and 2 both indicate non-free riders. However, these respondents need to be further classified to account for potential induced replacement and secondary market impacts, both described below.

4.2.1.1.1 Data Collection

A participant survey—drawn from a random sample of participants—will serve as the primary source of data collected for estimating NTG for the ARP. To determine the percentage of participants in each of the three options, evaluators will begin by asking surveyed participants about the likely fate of their recycled appliance had it not been decommissioned through the program. Responses provided by participants generally can be categorized as follows:

1. Kept the appliance.
2. Sold the appliance to a private party (either an acquaintance or through a posted advertisement).
3. Sold or gave the appliance to a used-appliance dealer.
4. Gave the appliance to a private party, such as a friend or neighbor.
5. Gave the appliance to a charity organization, such as Goodwill Industries or a church.
6. Had the appliance removed by the dealer from whom the new or replacement appliance was obtained.
7. Hauled the appliance to a landfill or recycling center.
8. Hired someone else to haul the appliance away for junking, dumping, or recycling.

Additional, follow-up questions will be included to validate the viability of all responses.

Next, evaluators will assess whether each participant’s final response indicates free ridership:

- Some final responses clearly indicate free ridership, such as: “I would have taken it to the landfill or recycling center myself.”
- Other responses clearly indicate no free ridership, as when the appliance would have remained active within the participating home (“I would have kept it and continued to use it”) or used elsewhere within the Program Administrator’s service territory (“I would have given it to a family member, neighbor, or friend to use”).

If the respondent planned to have the unit picked up by the retailer and the retailer would likely resell the unit in the secondary market, they are not a free rider. Absent retailer survey primary research described in the Enhanced Options below, the evaluators will utilize data from the most recent research conducted of the ComEd program to determine the proportion of free riders unless another metric is mutually agreed upon by the evaluators.⁴⁵

Secondary Market Impacts

In the event that the unit would have been transferred to another household (Option 2 above), the question then becomes what purchasing decisions are made by the would-be acquirers of participating units now that these units are unavailable. Such would-be acquirers could:

1. Not purchase/acquire another unit.
2. Purchase/acquire another used unit.

Adjustments to savings based on these factors are referred to as the program’s secondary market impacts.

⁴⁵ Note that such retailer interviews are being conducted annually for the ComEd ARP evaluation, and answers are used directly in the calculation of the NTG ratio in cases where: (1) the respondent planned to have the unit picked up by the retailer; and (2) the retailer was interviewed.

If it is determined that the participant would have directly or indirectly (through a market actor) transferred the unit to another customer on the grid, the next question addresses what that potential acquirer did because that unit was unavailable. There are three possibilities:

- A. **None of the would-be acquirers would find another unit.** That is, program participation would result in a one-for-one reduction in the total number of appliances operating on the grid. In this case, the total energy consumption of avoided transfers (participating appliances that otherwise would have been used by another customer) should be credited as savings to the program. This position is consistent with the theory that participating appliances are essentially convenience goods for would-be acquirers. (That is, the potential acquirer would have accepted the appliance had it been readily available, but because the appliance was not a necessity, the potential acquirer would not seek out an alternate unit.)
- B. **All of the would-be acquirers would find another unit.** Thus, program participation has no effect on the total number of appliances operating on the grid. This position is consistent with the notion that participating appliances are necessities and that customers will always seek alternative units when participating appliances are unavailable.
- C. **Some of the would-be acquirers would find another unit, while others would not.** This possibility reflects the awareness that some acquirers were in the market for an appliance and would acquire another unit, while others were not (and would only have taken the unit opportunistically).

The evaluators will assume Possibility C unless primary research within a Program Administrator's service territory to assess the secondary appliance market is undertaken as described in the Enhanced Options below. Specifically, evaluators will assume that half (0.5, the midpoint of Possibilities A and B) of the would-be acquirers of avoided transfers found an alternate unit.

Once the proportion of would-be acquirers who are assumed to find alternate units is determined, the next question is whether the alternate unit was likely to be another used appliance (similar to those recycled through the program) or, with fewer used appliances presumably available in the market due to program activity, would the customer acquire a new standard-efficiency unit instead.

4.2.1.2 Induced Replacement

If, however, the unit would have been kept by the participating household, the next question is whether the appliance was replaced and, if so, whether the household would have replaced the appliance regardless of the program.

The purchase of a refrigerator in conjunction with program participation does not necessarily indicate induced replacement. (The refrigerator market is continuously replacing older refrigerators with new units, independent of any programmatic effects.) However, if a customer would have not purchased the replacement unit (put another appliance on the grid) in the absence of the program, the net program savings should reflect this fact. This is, in effect, akin to negative spillover and will be used to adjust net program savings downward.

Estimating the proportion of households induced to replace their appliance should be done through participant surveys. As an example, participants could be asked, "Would you have purchased your replacement refrigerator if the recycling program had not been offered?"

Because an incentive ranging from \$35 to \$50 is unlikely to be sufficient motivation for purchasing an otherwise-unplanned replacement unit (which can cost \$500 to \$2,000), it is critical that evaluators include a follow-up question. That question should confirm the participants' assertions that the program alone caused them to replace their refrigerator. For example, participants could be asked, "Let me be sure I understand correctly. Are you saying that you chose to purchase a new appliance because of the appliance recycling program, or are you saying that you would have purchased the new appliance regardless of the program?"

When assessing participant survey responses to calculate induced replacement, evaluators will consider the appliance recycled through the program as well as the participant's stated intentions in the absence of the program. For example, if customers indicate they would have discarded their primary refrigerator independent of the program, it is not possible that the replacement was induced (because it is extremely unlikely the participant

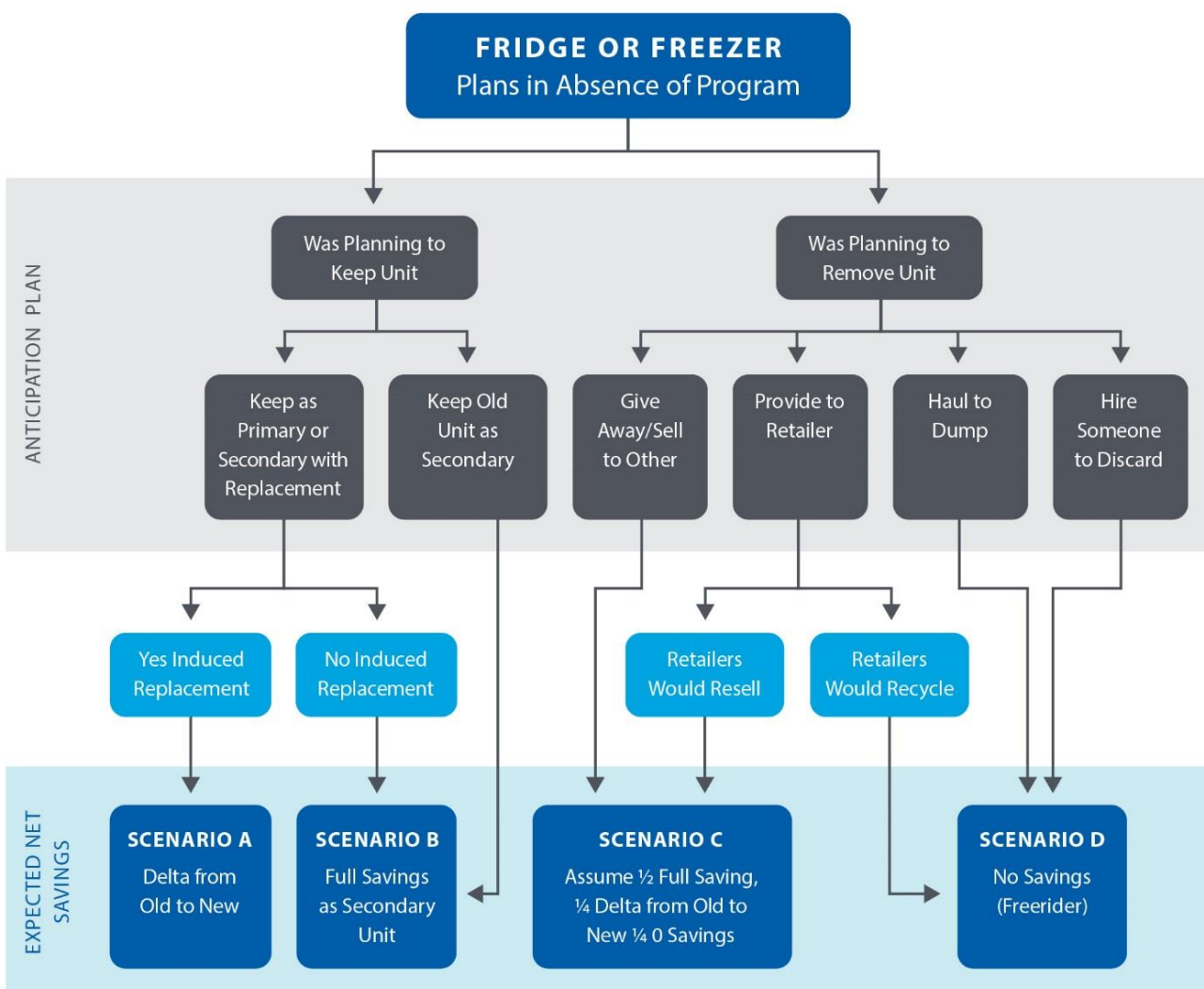
would live without a primary refrigerator). Induced replacement is a viable response for all other usage types and stated intention combinations.

As one might expect, previous evaluations have shown the number of induced replacements to be considerably smaller than the number of naturally occurring replacements unrelated to the program. Once the number of induced replacements is determined, this information is combined with the energy consumption replacement appliance to determine the total energy consumption induced by the program (on a per-unit basis).

4.2.1.3 Integrating Free Ridership, Secondary Market Impacts, and Induced Replacement

The flow chart shown in Figure 4-2 illustrates how net savings will be derived for an ARP. As shown, below, expected savings fall into four different scenarios.

Figure 4-2. Appliance Retirement Scenarios



Source: Adapted from the Pennsylvania Statewide Evaluator Common Approach for Measuring Net Savings for Appliance Retirement Programs, Guidance Memo-026, March 14, 2014.

4.2.1.4 Scoring Algorithm

Net savings will be assigned individually to each respondent, based on responses provided to the questions discussed above. Net savings will be averaged across all respondents to calculate program-level net savings. The following equation will be used:

$$FR = (\text{free ridership and secondary market impacts \%} - \text{induced replacement \%})$$

Table 4-4 demonstrates the proportion of a sample population classified into each of the eight potential (Tertiary Classification) categories and the resulting weighted net savings.

Table 4-4. Net Savings Example for a Sample Population*

Table 1: kWh Savings Example for a Sample Population						
Primary Classification	Secondary Classification	Tertiary Classification	Population (%)	UEC (kWh) w/out Program	UEC (kWh) w/ Program	kWh Savings
Would have kept unit	Scenario A: Kept but Induced Replacement	Non-ES unit	3%	1,026	520	506
		ES unit	2%	1,026	404	622
	Scenario B: Kept but NO Induced Replacement	N/A	25%	1,026	0	1,026
Would have removed unit	Scenario C1: Transferred No Induced Replacement	N/A	30%	1,026	520	506
	Scenario C2: Transferred With Induced Replacement	Non-ES unit	3.5%	1,026	520	506
		ES unit	3.5%	1,026	404	622
	Scenario D: Removed from Service	Recycled/ Destroyed	20%	0	0	0
		Retailer would Recycle	13%	0	0	0
	Net Savings (kWh)					

*The percent values presented in this table serve only as examples; actual research should be conducted to determine the percentage of units falling into each of these categories. Note that UEC (Unit Energy Consumption) values presented in the table represent example values, factoring in part-use.

4.2.2 Enhanced Method

Results can be enhanced by including three additional research efforts. The basic method has defaults where primary research on enhanced approaches cannot be performed:

1. A retailer survey, to determine the quantity and/or proportion of units returned to a retailer and that the retailer would deconstruct or recycle. Through this survey, one would determine a retailer's criteria for reselling used units vs. deconstructing them, based on unit age and condition. Results from the survey and analysis would be used to determine the proportion of those who would have returned an old appliance

to the retailer that should be included in Scenario D (free riders). This research was conducted for ComEd in EPY6 evaluation and those results were applied to Ameren.

2. An appliance market assessment study to determine the size of the secondary appliance market and whether removal of participating units from the market would cause an otherwise would-be receiver to purchase an alternative used or new unit. Savings attributable to these participants are the most difficult to estimate, as the scenario attempts to estimate what the prospective buyer of a used appliance would do in the absence of finding a program-recycled unit in the marketplace (i.e., the program took the unit off the grid, so the prospective purchaser faced, in theory, a smaller supply of used appliances). It is difficult to answer this question with certainty, absent Program Administrator-specific information regarding the change in the total number of appliances (overall and used appliances specifically) that were active before and after program implementation. In some cases outside of Illinois, evaluators have conducted in-depth market research to estimate both the program's impact on the secondary market and the appropriate attribution of savings for this scenario. Although these studies are imperfect, they can provide Program Administrator-specific information related to the program's net energy impact. Where feasible, evaluators and utilities should design and implement such an approach. Unfortunately, this type of research tends to be cost-prohibitive, or the necessary data may simply be unavailable.
3. However, it is possible to estimate through nonparticipant surveys which of the disposal responses given by nonparticipants were most likely to have been to an opportunistic would-be-acquirer. Transfers that would most likely have been opportunistic are determined primarily based on the cost to the recipient. If the appliance was sold or transferred to a retailer, there would have been a cost to the recipient of that appliance. If the recipient was willing to pay for the appliance or was willing to exert the effort to visit a retail location, this suggests the recipient was actively seeking an appliance. However, if the unit were given away for free, there was little cost to the recipient and it is a reasonable proxy for the proportion of opportunistic acquirers. This proportion would replace the 50% default assumption (scenario C in Figure 4-2) of would-be-acquirers that would or would not find an alternate unit.
4. A nonparticipant survey can be used to assess how nonparticipants acquire and dispose of used units. As nonparticipants do not have the same perceived response bias as participants, they can help offset some of this potential bias in estimating the true proportion of the population that would have recycled their units in program's absence. The evaluators will average the results of the nonparticipant survey with the participant survey if the nonparticipant survey is of sufficient sample size. Otherwise, results may be used for a qualitative characterization of potential bias. Though recommended, use of a nonparticipant survey need not be required, given budget and time considerations. A nonparticipant survey was completed as part of ComEd's EPY6 evaluation and used qualitatively to validate participant results.

4.3 Residential Upstream Lighting Protocol

The Illinois Residential Upstream Lighting programs to date have provided discounts on efficient lighting through retailers at the point of purchase. Such programs often remain transparent to customers purchasing incentivized lighting. Program administrators also do not know the identity of most customers purchasing the program-discounted lighting; so these customers cannot easily be contacted once they leave the store for a traditional self-report NTG evaluation survey (i.e., an after-the-fact, direct solicitation of customers regarding what they would have done in the program's absence). Similar surveys can be conducted with customers within program retailers after they have made their lighting purchasing decision but before they leave the store. For programs such as this, in store customer surveys are preferable to the traditional self-report telephone surveys that ask customers to recall their past light bulb purchases. Light bulbs are a small and relatively insignificant purchase for most people, thus the recall bias could be substantial.

Further, as upstream programs work with multiple market actors and can include wide-reaching marketing campaigns promoting energy efficiency to the general public, they tend to stimulate spillover and "market effects." As a result, estimating NTG for upstream residential lighting programs can be challenging. Multiple methods exist, each with their own strengths and weaknesses.

Ameren and ComEd implement their residential lighting programs comparably, and the evaluation teams have used a consistent primary NTG evaluation method. This section details the consensus NTG methodology, which has been used multiple times for both ComEd and Ameren and is considered the most well-vetted and defensible NTG method that has been successfully used in Illinois.

For EPY5 and EPY6, Ameren and ComEd used a customer self-report methodology to estimate NTG for their upstream residential lighting programs.⁴⁶ Customer self-report data in this method are collected during surveys conducted within program retailers with customers purchasing program bulbs (i.e., in-store intercept surveys). This method separately estimates free ridership, participant spillover, and nonparticipant spillover. Details follow on the primary data collection and scoring algorithms.

4.3.1 Basic Method

4.3.1.1 Free Ridership

Free ridership for this program is calculated as the proportion of program bulbs that would have been purchased if the program did not exist. Three alternative scenarios could occur:

1. Full Free Rider: The customer would have purchased the same quantity of efficient bulbs (CFLs or LEDs) in the program's absence.
2. Partial Free Rider: The customer would have purchased fewer efficient bulbs (CFLs or LEDs) in the program's absence.
3. Non-Free Rider: The customer would have not purchased any efficient bulbs (CFLs or LEDs) in the program's absence.

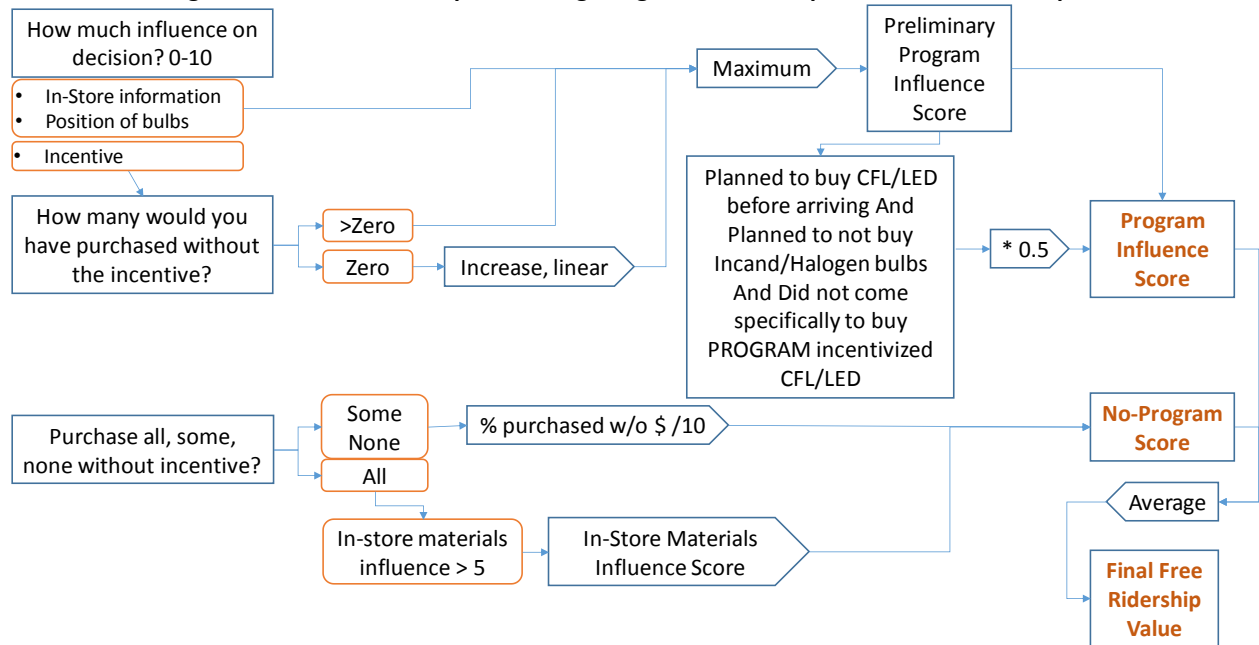
Free ridership is calculated as the average of two distinct scores: a Program Influence Score and a No-Program score. These scores are defined as follows:

1. The Program Influence Score captures the maximum level of program influence, reported by a survey respondent, of the residential lighting program on their decisions to purchase program bulbs on the day of the survey. This program influence can take a number of forms, such as: the monetary incentive provided to decrease the cost of high-efficiency bulbs; program-sponsored educational materials that explain the benefits of efficient lighting; in-store product placement of efficient bulbs; and program bulb recommendations provided by retail store personnel.
2. The No-Program Score is used to estimate how many program bulbs a survey respondent would have purchased in the absence of the residential lighting program.

Figure 4-3 illustrates the scoring algorithm for Residential Upstream Lighting Free Ridership via In-Store Intercepts.

⁴⁶ ComEd has used this method since EPY2. Ameren began using it in EPY5.

Figure 4-3. Residential Upstream Lighting Free Ridership via In-Store Intercept



4.3.1.2 Data Collection

To estimate free ridership, the evaluation teams will conduct in-store intercept surveys with customers purchasing program-discounted lighting at participating retailers. Customers are asked questions that are used to estimate a Program Influence Score and a No-Program Score for each customer and efficient bulb type purchased.

Primary Program Influence Score Questions

1. Light bulb purchasing plans for current shopping trip (Yes/No)
2. If planning to purchase bulbs:
 - a. Bulb type (CFL, LED, Incandescent, Halogen)
 - b. Program administrator-incentivized bulbs (Yes/No)
3. Influence of various program factors:
 - a. Program incentive
 - b. In-store information (printed materials or information from Program Administrator representatives or retail personnel)
 - c. Positioning of discounted bulbs within the store

Primary No-Program Score Questions

1. Stated preference of light bulb purchases had the Program Administrator incentive not been available (purchase all, some, or none of efficient bulbs)
2. Quantity of light bulbs purchased absent the incentive

4.3.1.3 Scoring Algorithms

Using the data collected from program participants during the in-store intercept surveys, Program Influence and No-Program Scores are calculated for each survey respondent and then combined to estimate a respondent-specific Free Ridership Score.

4.3.1.3.1 Calculation of the Program Influence Score

Survey respondents purchasing one or more program-discounted bulbs are assigned a Preliminary Program

Influence Score based on the maximum program influence level (on a 0 to 10 scale) they assigned to one or more program factors (e.g., monetary incentive/informational materials [printed or from store personnel]/product positioning). The influence level assigned to the monetary incentive should be increased for survey respondents (using a linear decreasing function)⁴⁷ who indicated that, absent the incentive, they would not have purchased any of the program bulbs they were purchasing that day.

After the Preliminary Program Influence Score is assigned, a secondary algorithm is run that adjusts the preliminary program influence based on survey data regarding the customers purchasing plans when they entered the store. Survey respondents who indicated they planned to purchase high-efficiency bulbs prior to entering the store and who had not come to the store specifically to buy Program Administrator-incentivized program bulbs, should have their Program Influence Score cut in half. This adjustment makes the final Program Influence Score reflective of their stated planned intention to purchase efficient bulbs in the program's absence.

4.3.1.3.2 Calculation of the No-Program Score

The No-Program Score is based on whether a respondent states they would have purchased all, some, or none of the program-discounted bulbs in the absence of Program Administrator incentives. Respondents reporting they would have purchased all of the efficient bulbs without the incentive should be considered free riders and receive a No-Program Score of zero. Those reporting they would have purchased none of the efficient bulbs without the incentives should be classified as non-free riders and receive a No-Program Score of 10, the maximum. Respondents reporting they would have purchased some of the efficient bulbs without the incentive should be assigned a No-Program Score between 0 and 10, reflective of the percentage of efficient bulbs they would not have purchased absent the program.

Respondents reporting they would have purchased all of the program-discounted bulbs in the program's absence, but in-store materials provided by the Program Administrator had a moderate to high influence on their decision, should have their No-Program Scores adjusted to equal the level of influence they attributed to these program-sponsored informational materials.

4.3.1.4 Calculation of Free Ridership

The Free Ridership rate is calculated as follows:

$$\text{Free Ridership} = 1 - (\text{Program Influence Score} + \text{No-Program Score})/20$$

Using the calculated Program Influence and No-Program Scores, Free Ridership is calculated as one minus the sum of the two scores (Program Influence Score plus No-Program score), divided by 20. Dividing the sum of scores by 20 results in a ratio (between 0 and 1) that is representative of the average of the two zero to 10 scores. Subtracting this ratio from one reverses the score, thus representing the free ridership level. If either the No-Program or Program Influence Scores are missing, Free Ridership can be calculated using the single available score divided by 10. Evaluators may also reference available data to perform documented modifications to individual free ridership estimates resulting from the application of this free ridership assessment methodology.

4.3.2 Participant Spillover

For this program, participant spillover results from purchases of non-discounted efficient bulbs by program bulb purchasers who are influenced by their participation in the residential lighting program to purchase additional non-

⁴⁷ The function, adjusted monetary score = (monetary score + 10)/2, increases the monetary score using a decreasing linear function. This function results in an increase in the monetary influence score of between 0 and 5 points depending on their original monetary score (i.e., an original score of 0 would become a 5, a 5 would become a 7.5, and a 10 would remain a 10). In past Illinois evaluations, this adjustment has typically changed less than 10% of all monetary scores.

discounted efficient bulbs.

4.3.2.1 Data Collection

Data collected during in-store intercept surveys with customers purchasing program bulbs should be used to estimate participant spillover. During these surveys, customers purchasing program-discounted and non-discounted efficient bulbs (CFLs or LEDs) should be asked questions to determine whether the residential lighting program influenced their purchases of non-discounted efficient bulbs.

Primary Program Influence Score Question

1. Influence of the lighting program or in-store information on the customer's decision to purchase non-discounted CFLs or LEDs. (0 to 10 scale where 0 is not at all influential and 10 is extremely influential)

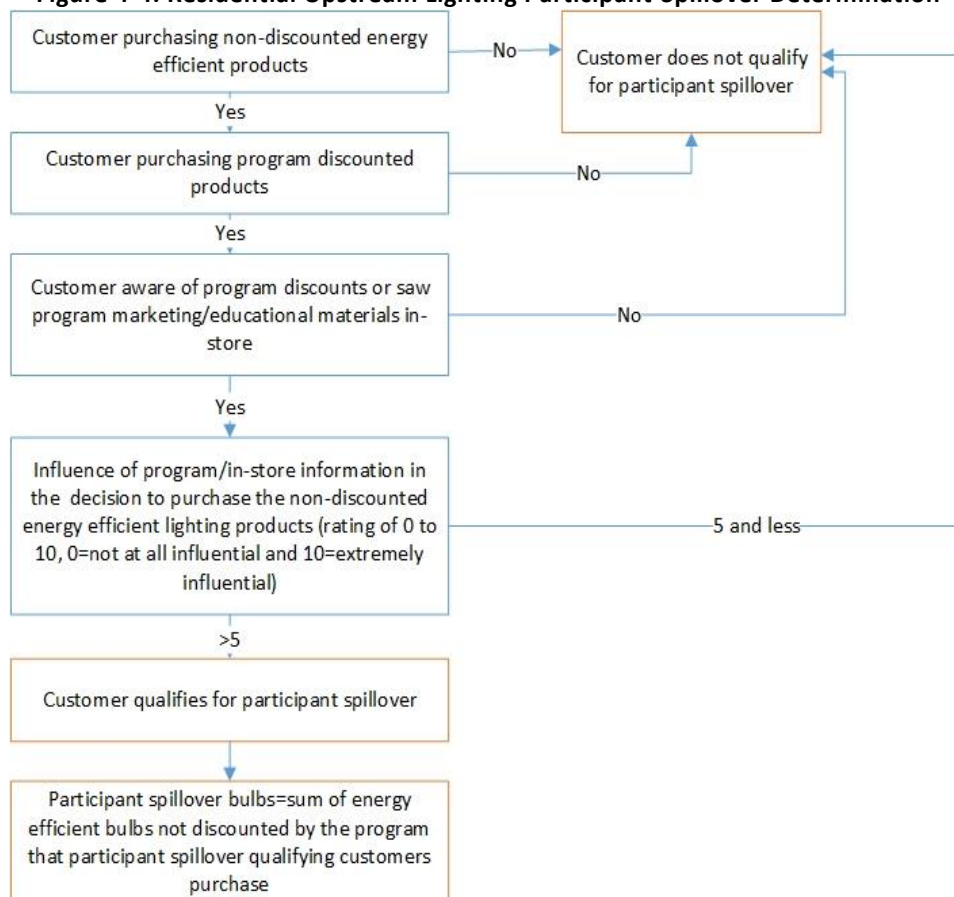
4.3.2.2 Scoring Algorithm

To estimate participant spillover, the number of program-influenced, non-discounted efficient bulbs (CFLs or LEDs) purchased by program participants is divided by the total number of program bulbs purchased by these program participants. This results in the Participant Spillover Rate.

Step 1: Estimate the total number of non-discounted energy efficient bulbs purchased by respondents that had also purchased program-discounted bulbs and were influenced by the program. Respondents who gave a rating of greater than 5 on the program influence question are considered to be influenced by the program.

Figure 4-4 below provides a visual depiction of the process of qualifying non-discounted bulbs as participant spillover bulbs.

Figure 4-4. Residential Upstream Lighting Participant Spillover Determination



Step 2: Calculate the total number of program-discounted bulbs purchased by summing the number discounted

bulbs purchased by all respondents.

Program Bulb Purchases = sum(Number of Discounted CFLs or LEDs purchased)

Step 3: Calculate the spillover rate by dividing the total number of spillover bulbs purchased by the total number of program-discounted bulbs purchased.

Spillover Rate = Spillover Purchases/Program Purchases

4.3.3 Nonparticipant Spillover

Nonparticipant spillover results from purchases of non-discounted efficient bulbs by customers who are not purchasing program-discounted bulbs, but report that the residential lighting program influenced their decision to purchase non-discounted efficient bulbs.

4.3.3.1 Data Collection

Data collected during in-store intercept surveys with customers purchasing efficient bulbs not discounted by the program should be used to estimate nonparticipant spillover. During these surveys, customers purchasing non-discounted efficient bulbs (CFLs or LEDs) and not purchasing any program-discounted bulbs should be asked questions about awareness of the program discounts and point-of-purchase program marketing and educational materials. These questions are used to determine whether the residential lighting program influenced their purchases of non-discounted efficient bulbs.

Primary Program Influence Score Question

1. Influence of the lighting program or in-store information on the customer's decision to purchase non-discounted CFLs or LEDs. (0 to 10 scale where 0 is not at all influential and 10 is extremely influential)

4.3.3.2 Scoring Algorithm

The non-participant spillover scoring algorithm involves estimating the total number of non-participants, the incidence of non-participants in the sample, the total number of non-participant spillover bulbs, and the average number of non-participant spillover bulbs per customer in the sample, and then extrapolating the sample estimates to the population of the utility customers. Below are the steps used to calculate the non-participant spillover rate.

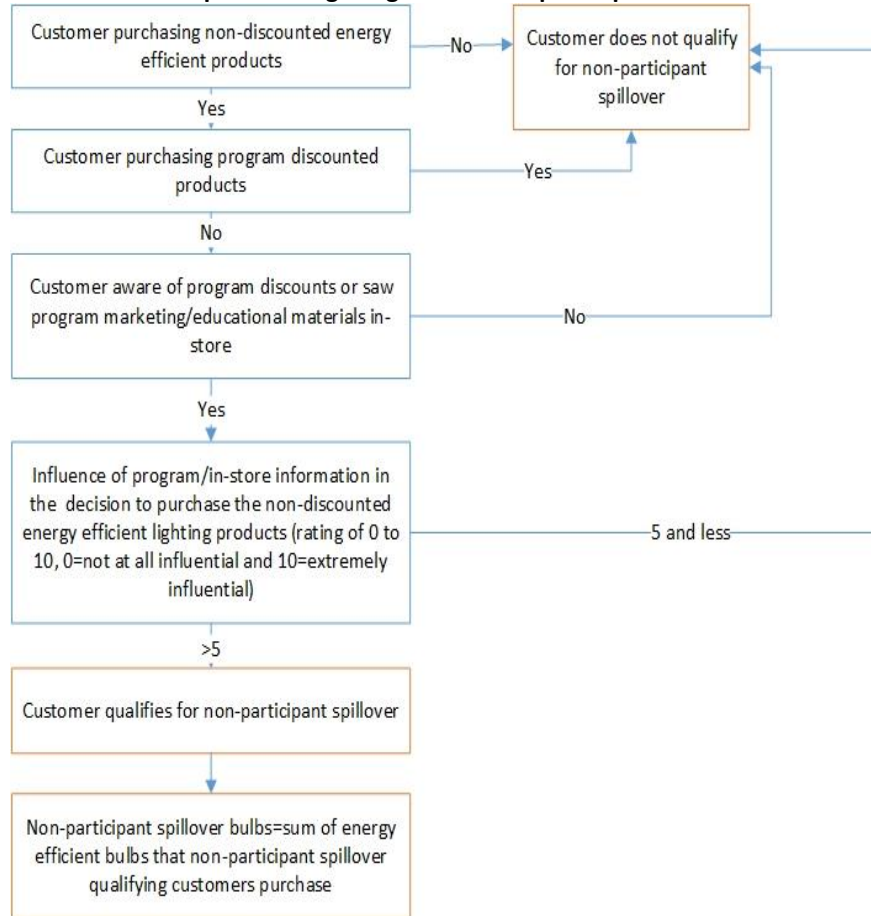
Step 1. Determine non-participant spillover in the sample by following the steps outlined below.

- A. Determine the total number of non-participating customers in the survey sample:
Non-participating customers (survey) = customers who did not purchase any program-discounted energy efficient lighting products. These customers may have purchased non-discounted energy efficient lighting products, less efficient lighting products or both.
- B. Determine the incidence of non-participating customers in the survey sample by dividing non-participating customers by total customers in the sample:
Incidence of non-participating customers (survey) = Non-participating customers (survey) / total customers (survey)
- C. Determine total number of non-participant spillover bulbs by summing CFLs and LEDs not discounted by the program that were purchased by non-participating customers who were aware of the program discounts or marketing promoting energy efficient lighting and were influenced by it. Spillover qualifying bulbs are those purchased by customers who rate the program's influence as greater than 5. The graphic below provides a visual depiction of the process of qualifying non-discounted products as spillover products.

Figure 4-5 below provides a visual depiction of the process of qualifying non-discounted bulbs as non-participant

spillover bulbs.

Figure 4-5. Residential Upstream Lighting Non-Participant Spillover Determination



- D. Determine the average number of non-participating spillover bulbs per non-participating customer by dividing the total number of non-participating spillover bulbs in the survey by the total number of non-participating customers in the survey.

Average number of non-participating spillover bulbs (survey)=total number of non-participant spillover bulbs (survey)/non-participating customers (survey)

Step 2. Extrapolate non-participant spillover to the population

- A. Determine the total number of non-participating customers in the population by applying the non-participant incidence rate from the sample to the population

Total number of non-participating customers (population)=Utility residential customer count incidence of non-participating customers (survey)*

- B. Determine the total number of spillover bulbs by multiplying the average number of spillover bulbs per non-participating customer in the survey by the total estimate of non-participating customers

*Total number of non-participant spillover bulbs=Average number of non-participant spillover bulbs (survey)*total number of non-participating customers (population)*

Step 3. Calculate non-participant spillover rate by dividing the total number of non-participant spillover bulbs in the population by the total number of program-discounted bulbs:

Non-participant spillover rate=total number of non-participant spillover bulbs/total number of program discounted bulbs

4.3.3.3 Method Advantages and Disadvantages

The in-store intercept method described above has certain advantages and disadvantages.

Advantages: This approach catches customers at their point of purchase, before they leave the store and can no longer be contacted directly. Given the interview's timing, customers can more easily recall price factors leading to their purchase choices. Also, as customers are intercepted at the store rather than surveyed by telephone, a higher cooperation rate results.

Disadvantages: Customers may not fully connect the impact that in-store education, product placement, and advertising have on their decision making. While many consumers believe they are not influenced by advertising, retailers know advertising and product placement work. Further, store intercepts typically must be coordinated with education events, and many retailers do not allow interviews to take place in their stores. Consequently, results are not based on random samples of customers purchasing program-discounted lighting throughout the year and across all participating retailers, which could bias the results.

4.4 Prescriptive Rebate (With No Audit) Protocol

Prescriptive Rebate programs typically offer predetermined rebates to residential customers for purchasing measures such as high-efficiency furnaces, clothes washers, brushless/electronically commutated motors (ECMs), boilers, boiler reset controls, water heaters, air-source heat pumps (ASHPs), ground-source heat pumps (GSHPs), central air conditioners (CACs), programmable thermostats, smart thermostats, insulation, air sealing, duct sealing, and desktop power management software. The program may require installation by a registered program ally, but it does not require a home audit (although purchases may be made in response to an audit).

These programs encourage consumers to undertake the following:

- Purchase higher-efficiency equipment than they otherwise would have, had they shopped for such equipment at the same time (replace on burnout); and
- Replace operating but inefficient equipment with higher-efficiency equipment (early replacement).

The basic method for estimating free ridership and participant spillover (See Section 4.1.2) for these programs uses a participant self-report, based on a standard battery of questions. An enhanced method may utilize trade ally surveys to provide another quantitative assessment, which may be triangulated with the basic method approach. As discussed further in Section 4.4.2, trade ally surveys may also be used to assess nonparticipant spillover.

4.4.1 Basic Method

4.4.1.1 Free Ridership

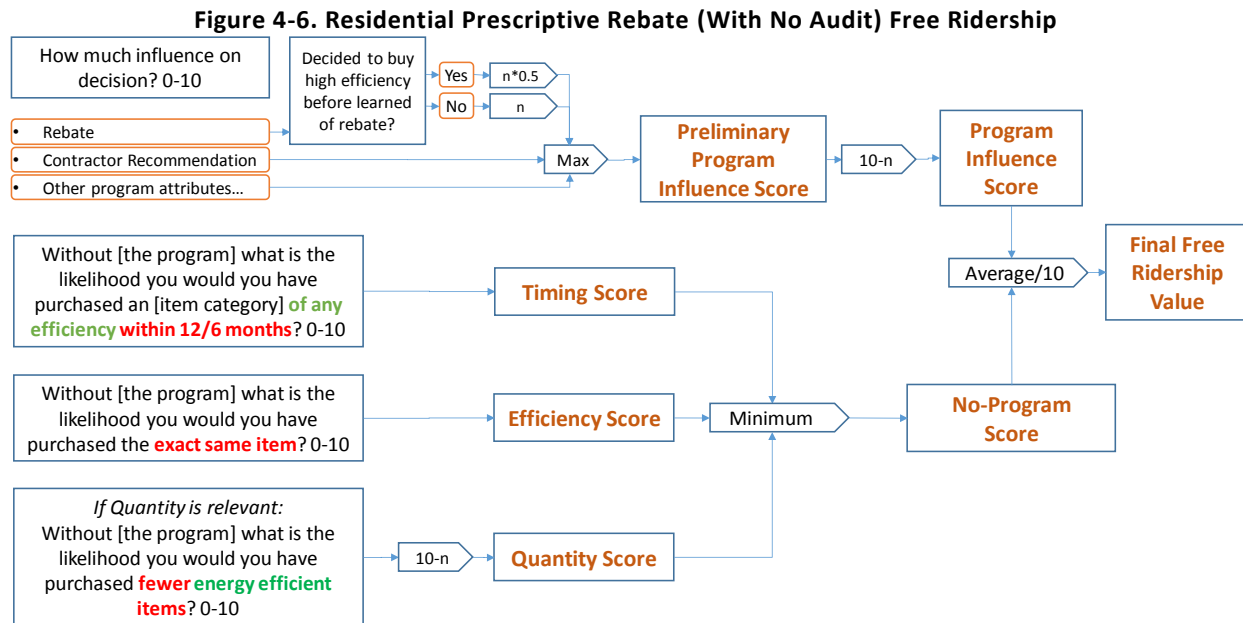
The free ridership assessment battery is brief to avoid applying an undue survey burden, yet it seeks to reduce self-report biases by including two main free ridership components:

- A Program Influence component, based on the participant's perception of the program's influence on the decision to carry out the energy-efficient project; and
- A No-Program component, based on the participant's intention to carry out the energy-efficient project without program funds.

When scored, each component assesses the likelihood of free ridership on a scale of 0 to 10, with the two scores averaged and for a combined total free ridership score. As different and opposing biases potentially affect the two main components, the No-Program component typically indicates higher free ridership than the Program Influence

component. Therefore, combining these decreases the biases.

Figure 4-6 illustrates the scoring algorithm.



4.4.1.1.1 Calculation of the Program Influence Score

Program influence is assessed by asking respondents, on a scale from 0 (not at all important) to 10 (extremely important), how important they found various program elements were on their decision to undertake the project the way they did. The number of elements included will vary, depending on the program's design. Logic models, program theory, and staff interviews typically inform the list of elements. Programs typically use the following elements to influence customer decision making: information; incentives or rebates; interaction with program staff (i.e., technical assistance); interaction with program proxies, such as members of a trade ally network; building audits or assessments; and financing.

In addition to asking about specific program influences, surveys ask respondents whether they planned to purchase a high-efficiency version of the product before learning of the rebate program. The respondent's rating of the rebate's influence is adjusted by 0.5 for those answering the question "yes."⁴⁸ Evaluators should conduct a sensitivity analysis around the use of this adjustment and present it in the report.

The Preliminary Program Influence Score equals the maximum influence rating for any program element rather than, for example, the mean influence rating. This is based on the rationale that if any given program element had a great influence on the respondent's decision, then the program itself had a great influence, even if other elements had less influence.

⁴⁸ The Illinois NTG Working Group discussed using this question to check for consistencies rather than adjusting the score. The NTG working group agreed that it is preferable not to directly ask about conflicting language with residential customers and to utilize an open ended question instead to assess possible reasons for conflicting statements. It is the experience of the NTG working group members that residential customers tend to be more impatient with these types of questions and can typically respond easier to an open-ended question about their motivations.

An inverse relationship occurs between high program influence and free ridership: the greater the program influence, the lower the free ridership. The Program Influence (PI) Score = 10 - Preliminary Program Influence Score.

4.4.1.1.2 Calculation of the No-Program Score

The No-Program (NP) Score is based on three measures of the likelihood of a participant purchasing the exact same item(s) at the same time in the absence of the program. Each of these likelihood measures are assessed on a 0-10 scale in which 0 means not at all likely and 10 means very likely.

First, the participant should be asked their likelihood of purchasing an item of *any efficiency* within 12 or 6 months (12 months for a single or big ticket item and 6 months for less expensive items) for the Timing (T) Score. Participants who were influenced by the program to replace still-functioning equipment will likely give a low score to this question, while participants who needed to replace burned out equipment will give a high score. This measure enables the analysis to use a single algorithm for both early replacement and replace-on-burnout scenarios.

Next, the participant should be asked a key question that asks the respondent to gauge their likelihood of purchasing the *exact same item* (e.g., make, model, efficiency) had the program not existed. This measure forms the Efficiency (E) Score. A respondent stating the likelihood of purchasing the same exact item as a 5 on a scale of 0 to 10 is assigned an Efficiency Score of 5.

If multiple quantities of an item are purchased, the respondent should be asked about the likelihood of purchasing fewer energy-efficient items. The response to this question is subtracted from 10 to compute the Quantity (Q) Score.

The No-Program Score is the minimum of the Timing, Efficiency, and (if applicable) Quantity Scores. Finally, the No-Program Score is averaged with the Program Influence Score to calculate the Final Free Ridership Value.

$$\text{No Program Score (NP)} = \text{Min}(T, E, Q)$$

$$\text{Free Ridership (FR)} = \text{Mean}(PI, NP)$$

4.4.1.1.3 Consistency Checks

To address the possibility of conflicting responses (i.e., low intention score and high influence score), the survey should include consistency checks that, at a minimum, ask participants an open-ended question to address the program's influence. For example:

- In your own words, please tell me the influence the program had on your decision to purchase the <insert measure name>.

In this case, the evaluation analyst will assess the response to this open ended question and its consistency with the other questions, and, if warranted based on clear additional information, they will adjust the score based on expert judgement. If an inconsistency exists and the open-ended response does not resolve the inconsistency, the respondent will be removed from the calculation. All instances of this occurring should be documented in the final report. Additional consistency checks, triggered and resolved within the survey with additional questions to participants, remain optional.

Missing responses to specific questions should be treated as “missing” for that particular question, but the observation or case will be retained in the analysis. Evaluation reports should note if this affects more than 5% of the responses.

4.4.2 Enhanced Method

4.4.2.1 Free Ridership

Free ridership results may be enhanced by including additional research efforts. A trade ally survey can be conducted to assess the percentage change in sales of high-efficiency equipment resulting from the program and the percentage of efficient equipment sales rebated through the program. Though these questions avoid directly asking for total sales before and after the program, the “with program” sales volume can be calculated by dividing

program tracking database counts of rebated products by the percentage of efficient products rebated through the program. The “without program” sales volume would then equal the “with program” sales volume, adjusted by the reported percentage change in equipment sales resulting from the program. Evaluators should ensure that trade allies receive sufficient time to collect specific data and not rely on “guesses” to respond. These results may be triangulated with participant survey results.

4.4.2.2 Triangulation

When multiple methods are used, evaluators may triangulate results by rating the analysis methodology and data collected using responses (rated on a scale of 0 to 10) to the following three questions:

1. All things being equal, on a scale of 0 to 10, with 0 being not at all likely and 10 being extremely likely, how likely is the approach to provide a more accurate estimate of free ridership?
2. Similarly, how valid is the data collected and the analysis performed (i.e., consider missing data, whether data collected was based on recollection or record keeping, is the analysis technique able to properly utilize the data collected)?
3. How representative is the sample (accounting for confidence and precision, and non-response or any sample frame bias)?

The weight for each method is the average score for that method divided by the sum of the average scores for all methods.

Table 4-5 illustrates example scoring for two different methods, illustrating the calculated weights.

Table 4-5. Example Triangulation Weighting Approach

NTG Triangulation Data and Analysis	Method 1	Method 2
1. How likely is this approach to provide an accurate view of free ridership?	6	8
2. How valid is the data collected/analysis?	3	5
3. How representative is the sample?	8	10
Average Score	5.7	9
Sum of Averages	14.7	14.7
Weight	39%	61%

4.5 Single-Family Home Energy Audit Protocol

Single-Family Home Energy Audit programs (or energy assessment programs) seek to secure energy savings for residential customers by providing audits, direct-install measures, and incentives for additional energy efficiency opportunities. The participation process generally begins with an energy audit, performed by a program-affiliated companies or individuals; this involves an auditor assessing the customer’s home to identify energy-saving opportunities. At that time, the auditor may install free instant-savings measures, such as CFLs, low-flow showerheads, and faucet aerators. Auditors also may educate customers about incentives available through the audit program (e.g., air sealing, insulation) or other Program Administrator-sponsored energy efficiency programs.

For these programs, free ridership and participant spillover (See Section 4.1.2) estimates rely on participant self-reports, gathered through surveys.

4.5.1 Basic Method

Given the multiple components of some audit programs, net impacts should be estimated using survey batteries tailored to a customer’s experience (e.g., receipt of free direct-install measures and discounted or rebated measures). The following sections outline the approach for two program components, one dealing with the direct

installation of free low-cost measures and a second dealing with envelope measures, such as air sealing and insulation.

4.5.1.1 No-Cost, Direct Install Measures

For free measures directly installed by program staff due to the audit, free ridership calculations should include the following components: Timing, Efficiency, and Quantity.

This approach provides several important benefits, such as deriving a partial free ridership score based on the likelihood that the participant would take similar actions in the absence of the audit. For example, partial scores can be assigned to customers who planned to install the measure, but the program influenced that decision, particularly in terms of timing (e.g., the program might have accelerated the installation) or quantity (e.g., the program might have led to installation of additional program-qualified measures).

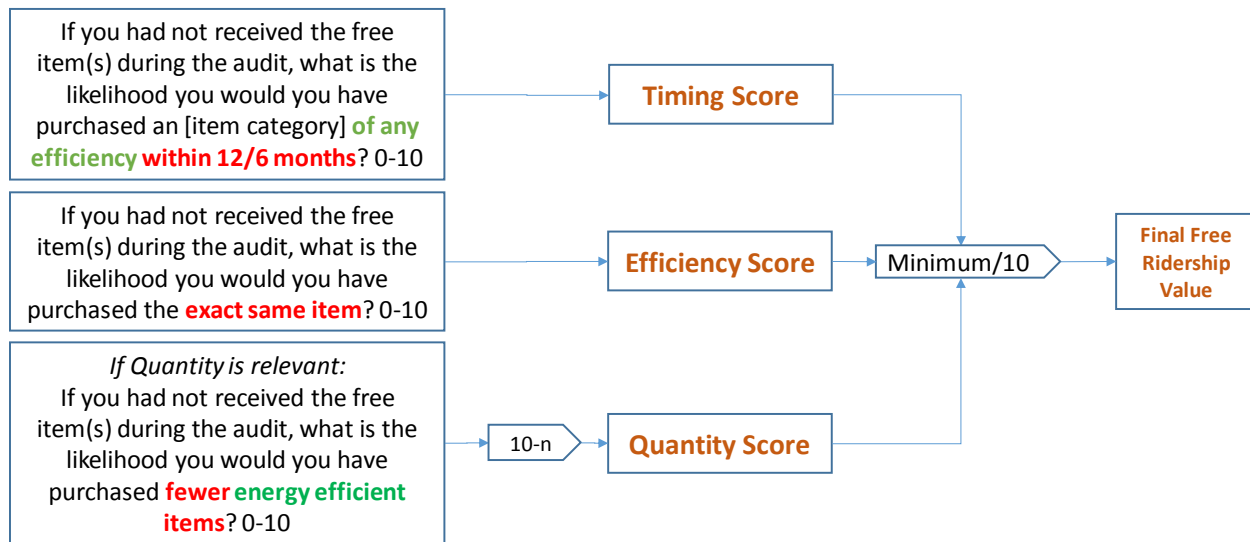
Outlines of components and their associated survey questions follow:

- Timing (T).** The first question is compute the Timing (T) Score accounts for earlier installation of measures due to the program by asking respondents about their likelihood (0-10 scale) to have installed an item *of any efficiency* within 6 or 12 months, had they not received it through the program (12 months for a single or big ticket item and 6 months for less expensive items).
- **Efficiency (E).** This score reflects the likelihood that customers would have installed the exact same energy-efficient measures, had the program not existed. For free measures, this is based on a question asking respondents to rate the likelihood that they would have installed the exact same measures had they not received them for free through the audit (on a 0 to 10 scale, where 0 is not at all likely and 10 is extremely likely). A higher likelihood value means a higher level of free ridership (i.e., a lower attribution level for the program).
- **Quantity (Q).** The question to compute the Quantity (Q) Score asks respondents about the likelihood that they would have installed fewer measures or performed less weatherization without the program. The response to this question is subtracted from 10 to compute the Quantity Score, as a lower score means a greater likelihood the respondent would have installed the same or a greater number of measures.

Given the low cost of the measures provided through the direct-install component of most audit programs and the number of measures received per participant, efforts have been made to streamline the free ridership battery to reduce the respondent's burden. As such, the overall Final Free Ridership Value per measure can be calculated by taking the minimum of the Timing, Efficiency, and Quantity Scores, as shown in the following equation:

$$\text{Free Ridership (FR)} = \text{Min}(T, E, Q)$$

Figure 4-7 illustrates the algorithm for no-cost measures.

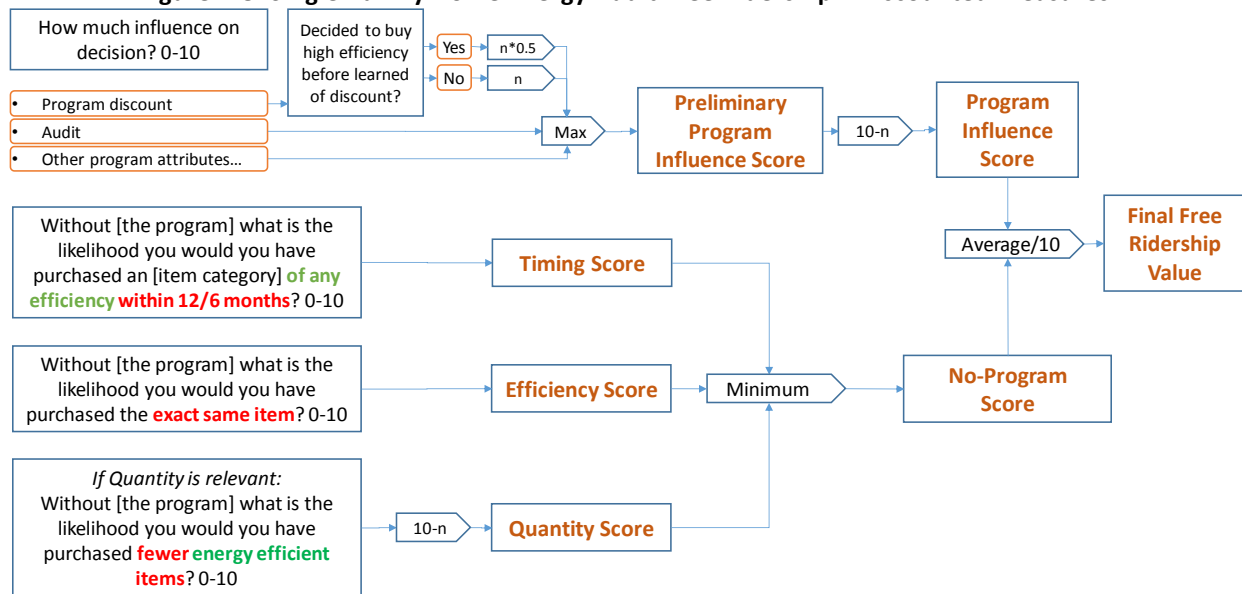
Figure 4-7. Single-Family Home Energy Audit Free Ridership—No Cost Measures

4.5.1.2 Rebated/Discounted Measures

Estimating NTG for rebated measures (typically for building shells) requires a more rigorous process than estimating NTG for free direct-install measures. In particular, the approach integrates an assessment of various program components that may have influenced the participant's decision to install the measures. For discounted envelope measures, the basic free ridership factor consists of the following two components:

- A Program Influence component, based on the participant's perception of the influence of various program elements—including the discount and the audit itself—on the decision to carry out the energy-efficient project; and
- A No-Program component, based on the participant's likelihood of purchasing the exact same items at the same time in the absence of the program.

The free ridership method for discounted measures is identical to that used in the Prescriptive Rebate (With No Audit) protocol, with the one exception that the questions about program influence should be sure to include the audit itself as one of the program attributes. Evaluators should refer to Section 4.4.1.1 for details of the method. Figure 4-8 illustrates the algorithm for discounted measures.

Figure 4-8. Single-Family Home Energy Audit Free Ridership—Discounted Measures

4.5.1.3 Consistency Checks

To address the possibility of conflicting responses (e.g., the high likelihood to install the same measure in the program's absence and the high importance of program factors), the survey should include consistency checks that, at a minimum, ask participants an open-ended question to address a program's influence, such as the following:

- In your own words, please tell me the influence the program had on your decision to purchase the <insert measure name>.

For low or no-cost, direct-install measures, surveys should include two questions to assess a program's influence on the respondent. The first should be asked at the beginning of the NTG battery, and the second should be asked at its conclusion. Questions include the following:

- Prior to the audit, had you purchased any <measures>? Y/N
- IF YES AND LIKELIHOOD TO INSTALL WITHOUT THE PROGRAM IS <7: Given that you had purchased <measures> before receiving the audit, why didn't you purchase additional <measures> on your own without the program? [OPEN END]
- IF NO AND LIKELIHOOD TO INSTALL WITHOUT THE PROGRAM IS >6: Given that you have not purchased <measures> before, why were you likely to purchase <measures> on your own without the program? [OPEN END]

In both cases, the evaluation analyst will assess responses to open ended questions and their consistency with the other questions; if warranted, based on clear additional information, the evaluator will adjust the original question score if required. If inconsistency occurs and the open-ended response does not resolve it, the original question response will be removed from the calculation. Final reports should document all instances of such adjustments. Optionally, additional participant questions can be included to trigger and resolve additional consistency checks.

Missing responses to specific questions (e.g., don't know or refused) should be treated as "missing" for those particular questions, but the analysis retains the observation or case. The evaluation reports should note if this affects more than 5% of responses.

4.6 Multifamily Protocol

Multifamily energy efficiency programs typically offer direct installation of low-cost, energy-efficient measures in multifamily dwelling units, in addition to rebates for common area lighting retrofits, air sealing, insulation, and improvements to HVAC systems and controls. These programs have various target audiences from owners, managers, or developers of market rate multifamily housing to those operating lower income or assisted living housing. Across these groups, properties must generally have a minimum of between three and five units to qualify for the programs.

Most multifamily program savings are typically achieved by encouraging customers to install higher-efficiency equipment than they would have installed on their own. However, programs may also encourage early replacement of still functioning equipment that is less efficient, thus impacting the timing of the installation, so that savings is realized earlier. The incentive may also make it more affordable for customers to install a greater number of high-efficiency measures.

The basic method for estimation of free ridership and participant spillover (See Section 4.1.2) for these types of programs is based on participant self-report gathered through surveys. For common area and building shell components of the program, participants are property managers and owners responsible for building maintenance and renovation. However, depending on the program design for the in-unit component of the program and specifically the installation of efficient lighting, the decision to participate in the program (i.e., install program measures) may arise from either property managers/owners or tenants or, potentially, both. This distinction is due to the fact that in some market-rate apartments, the tenant is responsible for decisions related to the installation of program measures, including light bulbs, while this is not common practice in income-qualified or assisted-living settings. For other in-unit measures, such as faucet aerators and low-flow showerheads, evaluators interview property managers/owners regarding program influence, as these measures are typically direct installed by program staff, and there is a limited likelihood of tenants making changes to these features.

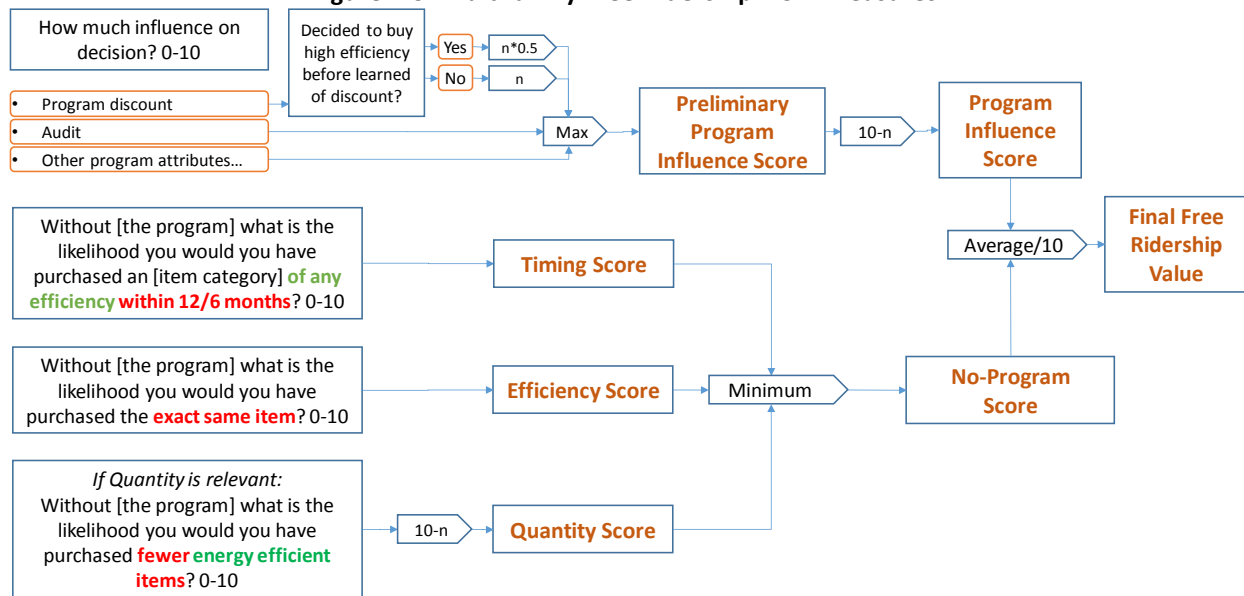
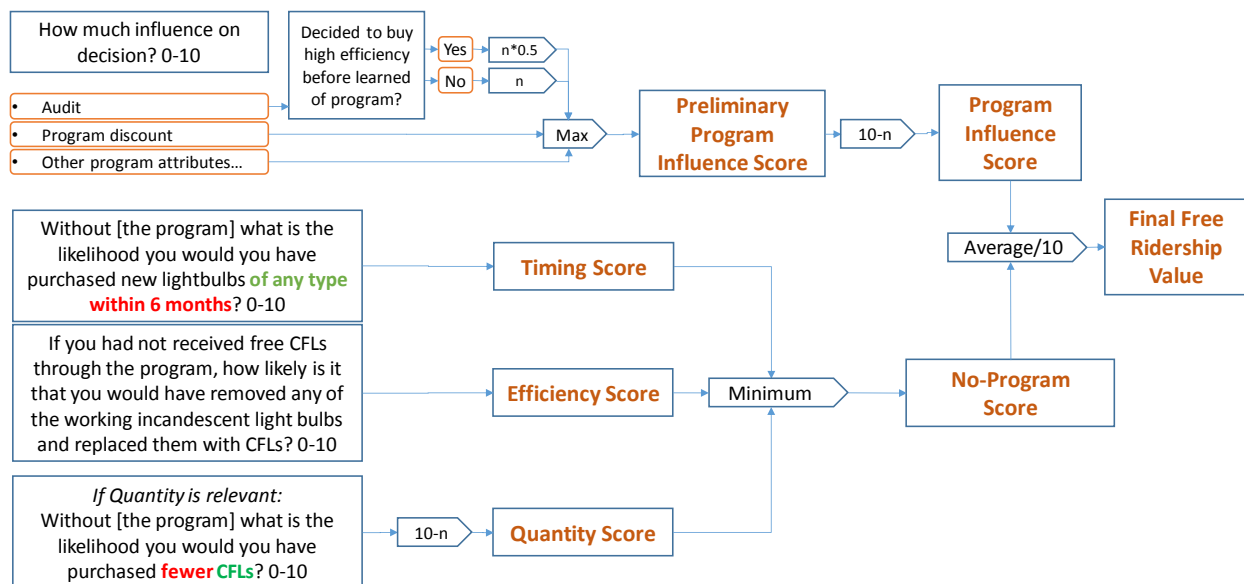
To date, most programs have included CFLs as one of their measures; so the text in this section refers to CFLs. The protocol can also be applied when the program installs LEDs.

4.6.1 Basic Method

Estimating NTG for rebated measures requires a more rigorous process than estimating NTG for free direct-install measures. In particular, the approach integrates an assessment of various program components that may have influenced the participant's decision to install the measures. For discounted measures, the basic free ridership factor consists of the following two components:

- A Program Influence component, based on the participant's perception of the influence of various program elements—including the discount and the audit itself—on the decision to carry out the energy-efficient project; and
- A No-Program component, based on the participant's likelihood of purchasing the exact same items at the same time in the absence of the program.

The free ridership method for discounted measures is identical to that used in the Prescriptive Rebate (With No Audit) protocol, with the one exception that the questions about program influence should be sure to include the audit itself as one of the program attributes. Evaluators should refer to Section 4.4.1.1.1 and 4.4.1.1.2 for details of the method. Figure 4-9 and Figure 4-10 also illustrate the algorithms for CFL and non-CFL measures.

Figure 4-9. Multifamily Free Ridership—CFL Measures**Figure 4-10. Multifamily Free Ridership for Property Managers—Non-CFL Measures**

4.6.1.1 Consistency Checks

To address the possibility of conflicting responses (e.g., high likelihood to install the same measure without the program, high importance to program factors), the survey should include consistency checks that, at a minimum, ask participants an open-ended question to address the program's influence. For example:

- In your own words, please tell me the influence the program had on your decision to purchase the <insert measure name>.

The evaluation analyst will assess the responses to the open ended questions and their consistency with the other

survey questions, and, if warranted based on clear additional information, will adjust the original question score. If the open-ended response does not resolve the inconsistency, responses to the original question should be removed from the calculation. The survey may include additional consistency check triggers and resolutions through additional participant questions. The final report should document how often the consistency check rules were triggered, how often adjustments were made to scores, and how often inconsistencies could not be resolved.

Missing responses to specific questions (including don't know or refused) should be treated as missing for that particular question, but the analysis should retain that observation or case. Evaluation reports should note if this affects more than 5% of the responses.

4.6.1.2 Data Collection

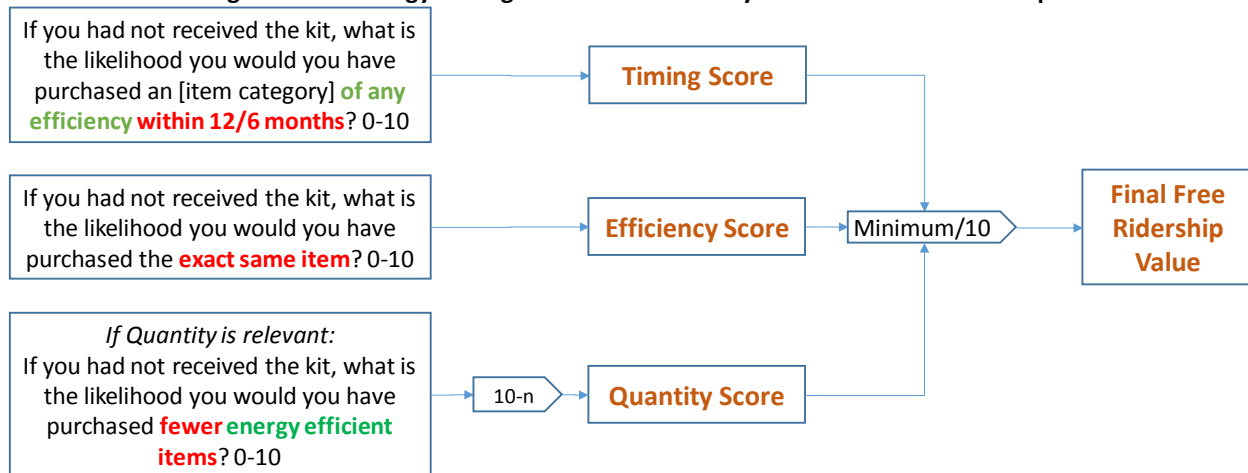
A participant survey should be used as the primary source of data collected for estimating free ridership in residential multifamily programs. As discussed, evaluators may field surveys with owners, property managers, or tenants, depending on a program's design and theory. Determining the appropriate audience from which to gather information for estimating free ridership depends on the program's design, and, ultimately, the party responsible for deciding to install specific program measures.

4.7 Energy Saving Kits and Elementary Education Protocol

Energy Saving Kits and Elementary Education Programs aim to secure energy savings through the distribution of kits containing various energy-saving measures, including (but not limited to): high-efficiency lighting (CFLs or LED lamps); bathroom and kitchen faucet aerators; and low-flow showerheads. Energy Saving Kits operate as an opt-in program; customers can request a kit by completing an Internet or phone application. Elementary Education Program participants do not request a kit as kits are distributed to all students in a classroom.

Free ridership and participant spillover (See Section 4.1.2) estimations for both programs rely upon participant self-report information gathered through surveys, despite the differences in distribution models. This methodology can be used for other energy-saving kit programs, including kits with alternative distribution methods (e.g., kits dropped off at a participant's home).

The following section contains a description of the basic NTG method used. Figure 4-11 illustrates the method.

Figure 4-11. Energy Saving Kits and Elementary Education Free Ridership

4.7.1 Basic Method

Free ridership calculations should include the following components: No-Program, Timing, and Quantity.

This approach provides several important benefits, such as the ability to derive a partial free ridership score based on the likelihood that similar actions would have taken place, even if the participant had not received a kit. For instance, partial scores can be assigned to customers with plans to install the measure, but the program at least influenced that decision, particularly in terms of timing (e.g., the program might have accelerated the installation) or quantity (e.g., the program might have led to the installation of additional measures).

Outlines of components and their associated survey questions follow:

- Timing (T).** The first question is compute the Timing (T) Score accounts for earlier installation of measures due to the program by asking respondents about their likelihood (0-10 scale) to have installed an item of *any efficiency* within 6 or 12 months, had they not received it through the program (12 months for a single or big ticket item and 6 months for less expensive items).
- Efficiency (E).** This score reflects the likelihood that customers would have installed the exact same energy-efficient measures, had the program not existed. This is based on a question asking respondents to rate the likelihood that they would have installed the exact same measures had they not received them for free through the kit (on a 0 to 10 scale, where 0 is not at all likely and 10 is extremely likely). A higher likelihood value means a higher level of free ridership (i.e., a lower attribution level for the program).
- Quantity (Q).** The question to compute the Quantity (Q) Score asks respondents about the likelihood that they would have installed fewer measures without the program. The response to this question is subtracted from 10 to compute the Quantity Score, as a lower score means a greater likelihood the respondent would have installed the same or a greater number of measures.

Given the low cost of measures provided in the energy-saving kits as well as the number of measures included in each kit, efforts have been made to streamline the free ridership battery to reduce the respondent's burden. As such, the overall Final Free Ridership Value per measure can be calculated by taking the minimum of the Timing, Efficiency, and Quantity Scores, as shown in the following equation:

$$\text{Free Ridership (FR)} = \text{Min}(T, E, Q)$$

Missing responses to specific questions (e.g., don't know or refused) should be treated as "missing" for that particular question. Despite missing responses, the case will be retained in the analysis (pairwise deletion). The evaluation reports should present the percent missing for each of the three questions.

4.7.1.1 Data Collection

Evaluators should use a participant survey as the primary data collection source for estimating free ridership in Energy Saving Kits and Elementary Education Programs. As a general rule, a free ridership rate should be calculated for each separate kit component, and then be weighted by savings to determine the program-level results.

4.8 Residential New Construction Protocol

Residential New Construction programs typically offer builder training, technical information, marketing materials, and incentives to builders for the construction of eligible homes. Eligible homes must meet specific standards, designed to achieve energy efficiency levels above local building codes. Programs may use different tiers of standards to meet correspondingly different incentives.

The basic method for estimating free ridership and participant spillover for these programs is based on builder participant self-reporting, gathered through surveys.

The following section describes the basic method used.

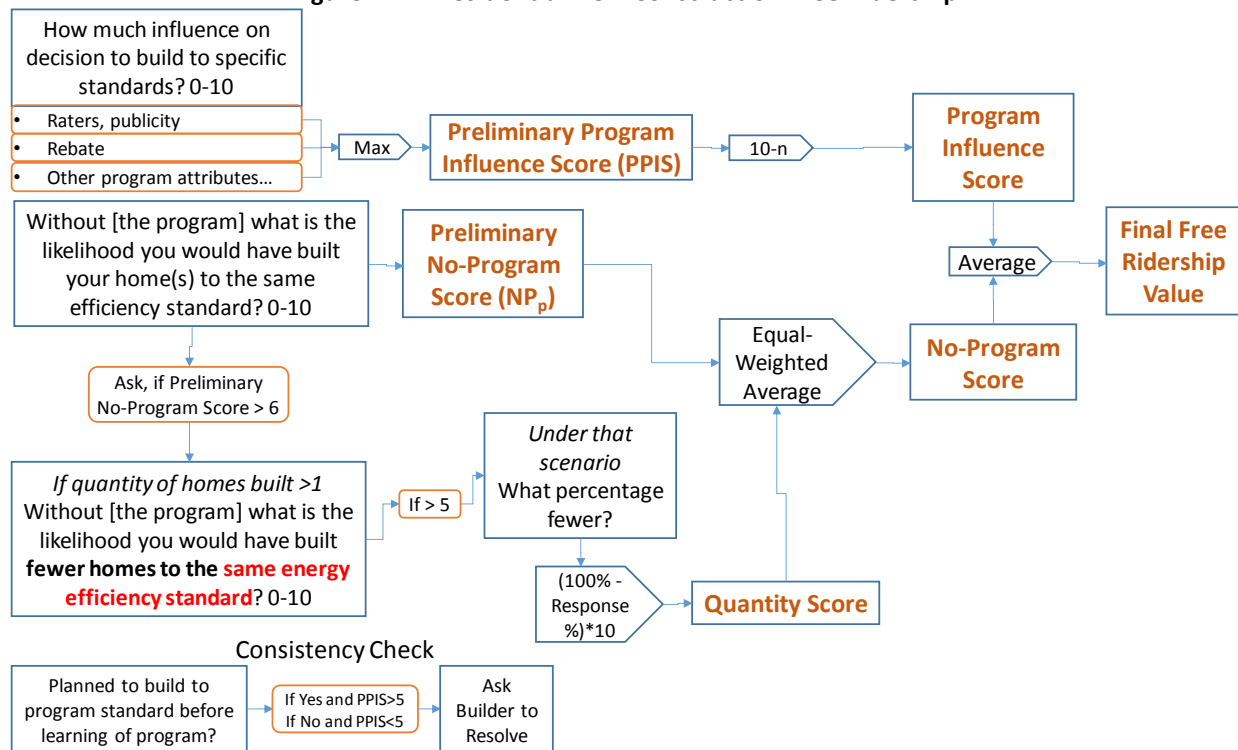
4.8.1 Basic Method

For this program, a free rider is a builder who would have constructed a home at the program's efficiency level in the program's absence. Given the multiple methods available to achieve desired home energy efficiency levels, survey questions consider the builder's likelihood of meeting the same energy efficiency standard, rather than whether or not the builder would have installed certain energy efficiency measures. Figure 4-12 (below) illustrates the method in more detail.

Evaluators assess Program Influence by asking respondents, on a scale from 0 (not at all important) to 10 (extremely important), how important they found various program elements in deciding to build to specific energy efficiency standards. The number of elements included vary, depending on the program's design. Logic models, program theory, and staff interviews typically inform the list of program elements included. Programs typically use the following elements to influence builder decision making: marketing materials; incentives or rebates; contacts with HERS Raters; and technical assistance.

In addition to asking about specific program influences, surveys should ask builders whether they planned to build homes to the same standard before learning of the program.

Figure 4-12. Residential New Construction Free Ridership



4.8.1.1.1 Calculation of the Program Influence Score

The Program Influence Score (PI) equals 10 minus the maximum influence rating for any program element rather than, for example, the mean influence rating. This is based on the rationale that if any given program element had a great influence on the respondent's decision, the program itself had a great influence, even if other elements had less influence.

4.8.1.1.2 Calculation of the No-Program Score

Evaluators calculate the No-Program score using a set of questions that ask respondents to gauge their likelihood of building homes to the same standards and in the same quantities had the program not existed. Three separate responses are considered in calculating the No-Program Score:

- The likelihood, on a scale of 0 to 10, that the builder would have built their homes to the same efficiency standard (Preliminary No-Program Score (NP_p))
- If that likelihood is greater than 6, the likelihood of fewer homes being built to the same efficiency standard.
- If that likelihood is greater than 6, the response to the question "for that scenario, what percentage of fewer homes would be built to the standard?" (Quantity Score = $(100\% - \% \text{ answer}) * 10$, which will be a number between 0 and 10)

The resulting No-Program (NP) Score is calculated as follows:

$$NP = \text{Mean}(NP_p, Q)$$

The overall Free Ridership Value derives from the average of the PI and NP scores, as shown in the following formula:

$$FR = Mean(PI, NP)$$

4.8.1.2 Consistency Checks

To address the possibility of conflicting responses (e.g., the high likelihood to build to the same efficiency standards without the program, the high importance of program factors), the survey should include, at a minimum, consistency checks that ask participants an open-ended question to address the program's influence. For example:

- In your own words, please tell me the influence the program had on your building practices.

If a high (>6) Preliminary Program Influence Score (PPIS) results, yet the builder planned to meet the same efficiency standard prior to learning of the program; or if the Preliminary Program Influence Score is lower (<7), and the builder did not plan to build to the standards prior to learning of the program, the survey should include a question to determine why this occurred, using wording that gets at the following inconsistencies:

- IF Preliminary Program Influence Score is >6 and Builder planned to meet the same efficiency standard prior to learning OF THE PROGRAM: Given that you had plans to meet the standard prior to learning about the program, why do you think the <program elements> were influential in your decision to meet the standard? [OPEN END]
- IF Preliminary Program Influence Score is <7 and Builder had no plans to meet the same efficiency standard prior to learning of the program: Given that you had no plans to meet the standard prior to learning about the program, why do you think the <program elements> were not more influential in your decision to meet the standard? [OPEN END]

The evaluation analyst will assess the responses to the open ended questions and their consistency with the other survey questions, and, if warranted based on clear additional information, will adjust the original question score. If the open-ended response does not resolve the inconsistency, responses to the original question should be removed from the calculation. The survey may include additional consistency check triggers and resolutions through additional participant questions. The final report should document how often the consistency check rules were triggered, how often adjustments were made to scores, and how often inconsistencies could not be resolved.

Missing responses to specific questions (including don't know or refused) should be treated as missing for that particular question, but the analysis should retain that observation or case. Evaluation reports should note if this affects more than 5% of the responses.

4.8.2 Participant Spillover

Participant spillover occurs when, due to program participation, a builder increases the energy efficiency of homes built outside the program (but inside a utility's service territory) by adopting certain building practices used in participating homes. Participant spillover can be calculated based on participant builder survey questions that ask builders about homes built within the utility service territory but outside the program. Survey questions ask whether the builder increased the energy efficiency standards of non-program homes after participating in the program, and the number of homes they applied these increased standards to, within the utility's service territory. Depending on the program characteristics, spillover should be measured as changes in specific building practices or as installation of specific measures. The text below assumes the program has been targeted at modifying building practices.

Spillover may be recorded depending on responses to the following questions:

1. How important was your experience in the <PROGRAM ADMINISTRATOR'S> program in your decision to incorporate this building practice your other homes, using a scale of 0 to 10, where 0 is not at all important and 10 is extremely important?
2. If you had not participated in the <PROGRAM ADMINISTRATOR'S> program, how likely is it that you would still have incorporated this building practice using a 0 to 10, scale where 0 means you definitely WOULD NOT have implemented this practice and 10 means you definitely WOULD have implemented this practice?

Responses to the first question establish the Practice Attribution Score 1, and responses to the second question establish the Practice Attribution Score 2. Spillover may be program-attributable for building practices with self-report data meeting the following condition:

$$\text{Spillover Score} = (\text{Practice Attribution Score 1} + (10 - \text{Practice Attribution Score 2}))/2 > 5.0$$

For responses meeting these conditions, an evaluator determines that specific building practices referenced in the question are attributable to the program; otherwise, the evaluator determines that specific building practices referenced in the question are not attributable to the program. The attribution criteria represent a threshold approach, in which energy impacts associated with building practices program participants implement outside the program are either 100% program-attributable or 0% program-attributable.

For each building practice discussed, builders will be asked how they know the building practice is more efficient than other options. If the respondent can identify the building practice as ENERGY STAR or name an efficiency level that the evaluator confirms as above the minimum federal standard, or if they identify a technology that the evaluator can confirm is above the minimum federal standard, this counts towards participant spillover.

Finally, depending on the building practice cited by the builder, follow-up questions should ask customers to provide reasonable information to allow the evaluator to estimate the amount of savings using IL-TRM protocols, such as quantity of appliances or the location and amount of insulation.

To calculate the spillover energy and demand savings for these actions, further questions should be asked to assess the gross savings of the building practice, through the appropriate version of the IL-TRM, if available, and the number of homes to which it applied. To develop the Spillover Rate, the total energy and demand impacts from the sampled participants who implemented efficient building practices in other homes due to participation in the program is summed, and then this sum is divided by the total ex post sample energy and demand impacts:

$$\text{Participant Spillover Rate (PSO)} = \frac{\text{Sum of Energy or Demand from Additional EE Practices}}{\text{Sample Ex Post Gross Energy or Demand Impacts}}$$

The equation used to adjust the Core NTGR based on participant spillover is as follows:

$$\text{NTGR} = (1 - \text{FR} + \text{PSO})$$

4.8.2.1 Sample

The sample for a spillover survey should be a random sample of current and up to one year previous program participants. Regardless of the year of participation, spillover should be measured within the set of homes that were completed within 12 months of the survey date.

4.8.3 Builder Nonparticipant Spillover

In addition to participant free ridership and spillover, new construction programs may create NPSO through builders exposed to the program but not actually participating. Rather, they implement some or all of the efficiency measures incorporated through the program in order to compete with builders that are participating.⁴⁹ NPSO caused by builders can be determined by surveying two groups of builders:

- “Drop out” builders, who participated in the program previously but have not participated in the past 12 months.

⁴⁹ NPSO also can arise from nonparticipating customers as a direct result of general energy efficiency education and promotion efforts. A separate protocol addresses such NPSO. Care should be taken to ensure the different approaches do not double-count NPSO.

- True nonparticipating builders that report they were aware of the program or that other builders were taking steps to improve new home efficiency, but had never participated.

Surveys ask nonparticipating builders if their knowledge of other builders' increased focus on energy efficiency influenced their building practices and in what manner, to quantify the program's impact on nonparticipating homes. The survey questions will first identify specific building practices that go beyond the implemented energy code for the specific jurisdiction in which the builder is active. Table 4-6 lists the latest building energy code in place for most areas of Illinois. Evaluators should make efforts to ensure the building code under enforcement for each jurisdiction is used as the baseline when evaluating spillover savings.

Table 4-6. IECC 2012 Building Energy Code

Component	IECC 2012
Thermostat	Heating 72F Cooling 75F Programmable Thermostat
Ceiling	U-0.026
Walls	U-0.057
Floors	U-0.033
Slab	R-10, 2ft
Windows	U-0.32
Infiltration	5ACH50
Duct Leakage	4CFM/100CFA
Duct Insulation	R-8 Attic Supply, R-6 Otherwise
Heat Pump	7.7 HSPF
Furnace	80 AFUE
Component	IECC 2012
Boiler	82 AFUE
AC	13 SEER
Lighting	75% CFL
Appliances	RESNET Default
Gas Water Heat	0.59 EF
Electric Water Heat	0.91 EF

For each component that is more efficient than code, the following additional questions are asked:

1. How many homes did you sell in <period> that incorporated this upgrade?
2. Of these homes, how many would have incorporated this upgrade, had the <program> not existed?

Evaluators should ensure that nonparticipant builders receive sufficient time to collect specific data and not rely on "guesses" to respond. Responses should also clarify whether sales counts are specific to the utility service territory in question.

The following steps calculate the program's nonparticipant builder spillover percentage:

1. Compute the difference between the total reported number of efficiency upgrades sold and the total that would have been sold in the program's absence to obtain the total number of upgrades by type of upgrade for that builder.
2. Multiply the total net number of upgrades of each type sold by each surveyed builder by the average gross unit savings for each upgrade type.
3. Sum the result for each builder from the previous step, and weight the results by the ratio of the population of non-active builders to the sample to compute the total spillover energy over the program period.
4. Divide the spillover energy savings by program gross savings.

Should a general population survey be implemented for nonparticipant spillover, care should be taken to ensure spillover is not double-counted.

5 Cross-Sector Protocols

The following sections include protocols that may be applicable to programs in the residential as well as in the commercial, industrial, and public sectors. Table 3-1 Commercial, Industrial, and Public Sector Programs and Table 4-1 Residential and Low Income Programs present information regarding the applicability of these protocols to specific programs.

5.1 Behavioral Protocol

5.1.1 Randomized Controlled Trials

The SEE Action Network's recent monograph on evaluating residential behavioral energy efficiency programs⁵⁰ indicates most of these programs are designed as randomized controlled trials (RCTs).⁵¹ In this design, evaluators (and sometimes implementation contractors) randomly assign sampled members of a population of interest to treatment group or a control group. Among the benefits offered by an RCT—when properly applied—is that it eliminates most selection bias, including free ridership and participant spillover effects. Hence, producing net savings estimates. For some programs, evaluators must take a second step to calculate net savings to ensure savings are not being double-counted, either counting savings being claimed by other programs or savings already credited to earlier program efforts (often called “legacy uplift”). Only increases in participation in other programs should be considered in this uplift adjustment; changes to total savings do not need to be made based on decreases in participation in other programs.

Free ridership refers to participants in an energy efficiency program that would have saved energy even without the program's stimulus. As these program participants would have engaged in energy-saving actions in the program's absence, counting their savings exaggerates the program's impact. RCTs eliminate free ridership bias because the random assignment of customers to treatment and control groups equally distributes such participants between the two.⁵² Upon comparing the two groups' energy consumption, free ridership energy savings in the control group cancel out those in the treatment group, eliminating free ridership bias.

Participant spillover refers to the tendency of participants in an energy efficiency program to engage in additional energy-saving actions. Though these actions occur outside of the program's scope, they also occur as a direct or indirect result of the program. The extent that these additional savings are not measured and attributed to the program by the evaluator understates the program's impact.

Consideration of participant spillover effects begins by considering what participant spillover means in the context of behavior-based energy efficiency programs. As behavioral programs prescribe neither the installation of any specific measures or sets of measures nor the adoption of any particular behaviors, they likely would not cause

⁵⁰ State and Local Energy Efficiency Action Network. *Evaluation, Measurement and Verification (EM&V) of Residential Behavior-Based Energy Efficiency Programs: Issues and Recommendations*. Prepared by A. Todd, E. Stuart, S. Schiller, and C. Goldman. Lawrence Berkeley National Laboratory, 2012. (<http://emp.lbl.gov/publications/evaluation-measurement-and-verification-emv-residential-behavior-based-energy-efficiency>)

⁵¹ For example, most residential, behavior-based energy efficiency programs administered by Opower on behalf of energy utilities are designed as RCTs, as are some commercial and industrial behavioral programs: for example, the EnergyCheck program that Pulse Energy implements for Commonwealth Edison.

⁵² Small differences may occur between the distributions of free ridership's propensity in the two groups for any given sample. Their expected values, however, will be identical, and in any case the size of any such discrepancies shrinks as sample size increases. Thus, this is only a potential concern for programs with unusually small numbers of participants.

participant spillover effects: energy savings resulting from a behavioral program's influence would, by definition, be "in scope," eliminating nearly all possibilities for participant spillover.

The only exceptions would be participant spillover effects not reflected on customers' bills or meter data. These can arise if spillover savings occur in another venue—outside of the home (e.g., a workplace) for a residential behavioral program—or if the program's design reduces energy consumption in one form (e.g., electricity) but results in spillover savings in another form (e.g., natural gas). To the extent that either situation occurs, an evaluation relying on an RCT would understate program savings.

In general, RCTs do not address nonparticipant spillover, which reflects a program's influence on nonparticipants. Such spillover may arise from a behavioral energy efficiency program if, for example, the program indirectly influences customers in the control group or affects the availability of energy efficiency products and services to those served by the relevant market, regardless of whether they participate in the program or belong to the control group. Where significant nonparticipant spillover occurs, an evaluation relying on RCT would understate program savings.

In an RCT, energy consumption of the treatment and control groups can be appropriately compared through a regression analysis, using time-series observations on the usage of individual customers in the treatment and control groups during the pre- and post-treatment periods. Such data most commonly derive from customers' monthly bill records, hence the frequent use of "billing analysis" to describe this approach (although higher-frequency usage data from customer AMI meters also can be used and provide some additional benefits).⁵³ Due to the combined time-series/cross-section structure of such data sets, the NTG Working group recommends that panel regression techniques be used.⁵⁴

5.1.2 Non-Randomized Designs

Where randomized assignments prove infeasible, quasi-experimental evaluation methods can be substituted. These methods select a control group using nonrandom methods and are less reliable than RCTs, but, with appropriate care, they can produce valid results. Non-randomized designs can still produce net savings as their primary output, just as RCTs do.

Three quasi-experimental approaches are commonly used to evaluate behavior-based energy efficiency programs that cannot be construed as RCTs:

- Regression discontinuity (RD)
- Variation-in-adoption (VIA)
- Matched controls (MC).

All three create a nonrandom control group to replace a random control group used in the RCT approach.

Regression Discontinuity. RD requires basing a program's eligibility on a continuous variable (e.g., customers' adjusted gross income falling below a cutoff value for them to qualify for the program). When this is true, the RD

⁵³ These benefits include: having more observations per customer, which improves model precision; obviating concerns over billing periods with differing numbers of days; and providing the ability to observe intraday load shifting in addition to energy savings.

⁵⁴ "Panel" refers to the data set consisting of time-series observations on energy consumption of a cross-section of treatment and control customers. Panel estimation techniques refer to the model's inclusion of terms that control for individual customer heterogeneity (e.g., customer fixed effects or a lagged dependent variable), and cluster-robust standard errors, which can accommodate differing error variances across customers and an intracustomer correlation of errors.

method assumes customers just beyond the cutoff likely will be very similar, on average, to those just inside of it. The method compares changes in energy usage for a group just outside of the eligible range to that of a group of participants just on the other side of the eligibility cutoff. The RD approach, however, is susceptible to an important weakness: misspecification of the regression functional form.⁵⁵

Variation-in-Adoption. The VIA model applies only to program participants.⁵⁶ For this method, customers must sign up for the program on a rolling basis. VIA takes advantage of its enrollment's differential timing to compare energy usage of customers opting in to that of customers not yet opting in (but doing so later). The method relies on an assumption that, in any given month, customers have already opted in; those that soon opt in have similar characteristics to those who have enrolled, both in observable and unobservable characteristics. For this assumption to prove valid, customers must decide to opt into the program at different times for essentially random reasons (e.g., influenced only by marketing exposure and program awareness).⁵⁷ In particular, the decision to opt in should not relate to observable or unobservable household characteristics.⁵⁸

Matched Controls. MC creates a control group by matching each treatment customer to the most similar nonparticipant customer available on the basis of exogenous covariates from the pre-enrollment period known to highly correlate with post-enrollment usage.⁵⁹ The covariate most likely to correlate with post-enrollment energy usage in a given time period is customer energy usage during the same period of the preceding year, but other observable factors may be used when available. Implementing MC requires customer usage data for the year preceding all opt-in customers' decisions to participate in the program, along with a large group of nonparticipants who can be assumed to be similar to opt-in customers, aside from their program participation status. The pool of potential matches should be drawn from the same customer class and rate category.

The MC method involves identifying a nonparticipant customer whose energy usage closely matches that of a program participant in the months preceding the participant's enrollment in the program. The logic inherent in this approach is: if the analyst finds a set of nonparticipants who, on average, are the same as participants regarding energy consumption before program enrollment, these matches will provide a good counterfactual estimate of how much energy participants would have used in the program's absence.

The MC approach does present a main weakness: it can only identify matches based on observable customer characteristics, which leaves open the exclusion of the possible influence of relevant unobservables. While factors other than pre-enrollment energy usage plausibly could be used (e.g., household income, demographics,

⁵⁵ The most common misspecifications are: mistaking a nonlinear relationship for a discontinuity; and failing to recognize potential interactions between assignments and the treatment studied. See W.R. Shadish, T.D. Cook and D.T. Campbell, *Experimental and Quasi-Experimental Designs for Generalized Causal Inference*, adsworth 2002, pp. 229-238.

⁵⁶ M. Harding and A. Hsiaw, "Goal Setting and Energy Conservation," July 2013. Available at: http://people.duke.edu/~mch55/resources/Harding_Goals.pdf.

⁵⁷ This differs from an RCT with a recruit-and-delay design, in which customers do not choose when to opt in, but instead are randomly assigned different times to opt in, or from an RCT with a recruit-and-deny design, where customers are randomly denied access to the program.

⁵⁸ As the validity of the VIA method depends on this assumption, it should be empirically tested to the extent possible. If program marketing is punctuated and dates of marketing exposure are known, it is possible to test whether household enrollment in any particular month is driven by marketing activity, as opposed to observed household characteristics or unobserved heterogeneity. A test of whether the energy usage of households before they opt in differs from households that opt in during any particular month as opposed to another month is built into the VIA regression model's functional form. See Harding and Hsiaw, op. cit., for details.

⁵⁹ See Daniel E. Ho, Kosuke Imai, Gary King, and Elizabeth Stuart, 2007, "Matching as Nonparametric Preprocessing for Reducing Model Dependence in Parametric Causal Inference." *Political Analysis* 15(3): 199-236.

geographic location) in the matching process to address relevant unobservable characteristics (e.g., attitudes toward energy conservation and environmental concerns), this assumption cannot be directly tested.⁶⁰

There is a special case of MC called propensity-score matching. This develops a binary choice model to predict the probability that a customer will opt into the program, and then, for a control group, chooses customers with a high propensity for opting in but choosing not to do so. This functions well if observable variables used to calculate the propensity score sufficiently correlate with relevant unobservables to explain differences between treatment and control customers that cannot be explained by matching observables. With most evaluations of energy efficiency programs, however, little (if any) data are available on customers other than their energy usage; so the distinction usually becomes irrelevant.

5.2 Code Compliance Protocol

The protocol represents a basic framework for estimating the NTGR that may be refined based on impact evaluation results. The NTGR is used to convert an estimate of gross savings into an estimate of net savings. Two general methods can be used to estimate gross energy impacts: (1) utility billing data analysis; and (2) building energy modeling.⁶¹ The specific method used depends on the availability of necessary data.

5.2.1 Data Collection

5.2.1.1 Program Documentation

To inform the NTGR estimate, the evaluator documents program delivery. Information collected includes the following: the number, location, and dates of training workshops; the topics covered; materials disseminated; the number of trainees in each workshop and the type of trainee; and the hours of instruction.

5.2.1.2 Stakeholder Interviews

To inform the NTGR estimate, the evaluator conducts interviews with key stakeholders involved in the program. Interviews should include training program managers, instructors, and trainees. Trainees typically include contractors, builders, consultants, code officials, and others involved in building design and construction. The interviews seek to gather information on how training affected building design, construction, new code compliance, and enforcement.

5.2.2 Attribution Assessment

The NTGR estimation method stays the same, regardless of the method used to estimate gross energy savings.

A Delphi panel⁶² produces an NTGR estimate that reflects the share of gross energy savings resulting from increased code compliance attributable to the program. Formed by selecting four to six knowledgeable professionals not associated with the program in any way,⁶³ the panel receives estimates of gross energy savings,

⁶⁰ Such secondary, observable characteristics are rarely available to evaluators of energy efficiency programs, except for geographic location (e.g., postal zone of customer premise).

⁶¹ The modeled energy savings approach is similar to the approach described by Department of Commerce in Exhibits 6.1 and 6.2 from excerpts of Docket 13-0499 through estimation of potential energy savings.

⁶² The Delphi panel should be conducted according to best practices. For example, see: Day J and Bobeva M (2005) "A Generic Toolkit for the Successful Management of Delphi Studies" *The Electronic Journal of Business Research Methodology* Volume 3 Issue 2, pp. 103-116, available online at www.ejbrm.com.

⁶³ Delphi panelists should have no biases that would affect their assessment of the program's effectiveness.

building construction data, and evidence of attribution—including the results of stakeholder interviews and program documentation. Panel members individually review the information and provide feedback regarding their NTGR estimates and rationales. Responses are compiled, with combined, anonymous responses circulated to all panel members. Panelists review this information, revise their initial estimates and rationales, as they deem appropriate, and provide new estimates and rationales. Evaluators review the second set of estimates and rationales to develop a final attribution estimate, accompanied with a summary of supporting rationales. This NTGR estimate, used in combination with the gross energy savings estimate and building construction data, produces a final estimate of net energy savings attributable to the program.

Selected individuals should be knowledgeable about building codes and all factors that could conceivably affect code compliance.

6 Appendix A: Overview of NTG Methods

The evaluation teams present information in this appendix to provide a relatively quick overview of NTG methods for readers unaccustomed to the possible methods that evaluators may deploy. It is not meant to be a complete or deep discussion about each of the methods presented. However, the evaluators in Illinois considered the inclusion of this appendix to be very important in acknowledging the current suite of methods deployed by evaluators throughout the U.S. and giving a framework for work within Illinois.

Much of the information shown below is taken directly from a single source—the national Uniform Methods Project, Chapter 23: Estimating Net Savings: Common Practices. (Violette and Rathbun, 2014) This document has done a nice job of summarizing the eight most common attribution methods currently in use across the U.S. The evaluation teams recommend that readers go first to this reference for further information. Additionally, while there are slightly over 100 references within the Violette and Rathbun document, other non-duplicative references are included where reasonable as additional resources for those interested in further research into any specific method.

6.1 Survey-Based Approaches

Virtually all Illinois based evaluations use a survey-based approach for programs where primary data is used to determine net savings. (The main exception is for behavioral programs which use statistical analysis based on a randomized control trial program design.) Survey-based approaches obtain data from program participants and nonparticipants using a structured data collection instrument implemented via phone, in person, or online. At times, evaluators create and use an unstructured depth-interview guide to collect information about attribution, and this provides both contextual data and quantitative data about a given project.

6.1.1 Self-Report Approach

The self-report approach relies on the abilities of customers to discuss the program influence as well as the somewhat abstract ideas of the counterfactual (i.e., what would have occurred absent the program) after making a choice to purchase an energy efficient item or take an energy efficient action unrelated to a purchase. For program participants, this could include doing nothing (i.e., leaving the existing equipment as-is), installing the same energy efficient equipment as they did through the program, or an intermediate step of installing equipment that is more efficient than what they had in place previously, but less efficient than what they installed through the program. Evaluators also use this approach when collecting information from trade allies or distributors. This self-report approach is not new, nor is it exclusively used by the energy efficiency industry. An important attribute of this approach is its reliance on well-designed and fielded survey questions; so that the data underlying subsequent analyses are accurate and complete.

The output of this approach is a NTG ratio which can be considered an index of the program's influence on the decision to install energy-efficient equipment. The NTG ratio is applied to gross savings in order to obtain an estimate of net savings. The NTG ratio may include free ridership, spillover, or market effects, depending on the survey and analytical design. NTG ratios may be calculated at the measure, suite of measures, or program level and are typically average values weighted by savings. If sufficient information is available, analysis of NTG ratios among certain customer segments may be done to further inform changes to program design.

References

- Sudman, 1996
- Stone, et al., 2000
- Bradburn, et al., 2004

6.1.2 Econometric/Revealed Preference Approach

The econometric/revealed preference approach, while still considered a survey approach due to how data is collected, moves beyond asking people about the counterfactual and instead uses the observations of the evaluator to collect information for analysis of a NTG ratio. Within this approach, evaluators typically deploy similar sampling designs as for the self-report approach to collect data, but actively gather what a person is doing (i.e., what is being purchased in a store) to determine attribution.

6.2 Randomized Control Trials and Quasi-Experimental Designs

As mentioned earlier, evaluators deploy randomized control trials (RCT) for estimating savings from the behavioral programs within Illinois. Additionally, quasi-experimental designs (QED) have been used in the past in Illinois to estimate net savings from the upstream CFL program, and CFL, insulation, and air-sealing measures within the Home Performance with ENERGY STAR program.

RCT and QED use statistical analysis to determine regularities within the data that reveal net savings due to a program intervention.⁶⁴ The analytical design attempts to control for factors that can confound net analysis.⁶⁵ When estimating net savings within both an RCT and QED, two groups are included within the analysis: 1) a group that has been exposed to (i.e., treated by) a program; and 2) a group that has not been exposed to the program. Evaluators must carefully consider the choice of the non-exposed group (called a control group for RCTs or comparison group for QEDs).

RCT: This design must be integral to a program's implementation. Without the ability to randomly assign customers to one group or another (or at least randomly encourage customers to participate in a program), the ability of the design to yield unambiguous estimates of net impacts is compromised. Evaluators often help design how a program is implemented and, if not involved at the outset, carefully review choices made by the implementation team.

QED: A QED may be designed after a program has been implemented. It relies on determination of an equivalent comparison group, which is often chosen based on energy use. QED is difficult to perform well within the commercial sector due to the heterogeneity of end uses within the sector.

The output of an RCT or QED equals the average net savings for the population within the statistical model. Evaluators may also analyze the data to help understand the savings within specific known segments if sufficient information and data points are available.

References

- Mohr, 1995
- Shadish, Cook, Campbell, 2002
- Scriven, 2008
- Donaldson, 2009

⁶⁴ Net savings are calculated when a comparison or control group of non-treated customers are part of the design. Statistical analyses can also obtain gross savings.

⁶⁵ Economists strongly support this approach, but among program evaluators, the idea that an RCT is a "gold standard" for attribution research has been hotly debated for decades.

6.3 Deemed or Stipulated NTG Ratios

A deemed (or stipulated) NTG ratio is a value known prior to implementing a program and applied to estimate net savings for that program in a certain year.

Deemed or stipulated NTG ratios may be based on previous primary data collection, a review of secondary data, or agreed to among stakeholders. In Illinois, deemed or stipulated NTG ratios should reflect best estimates of likely future actual NTG ratios for the relevant program year, taking into consideration stakeholder input, the evaluator's expertise, and the best and most up-to-date information.

6.4 Common Practice Baseline Approaches

For this method, the evaluation team estimates what a typical consumer would have done at the time of the project implementation. Essentially, what is "commonly done" becomes the basis for baseline energy consumption and calculation of net savings. No gross impacts are calculated in this approach. This baseline is defined as the counterfactual "i.e., what would have occurred absent the program" and has been referred to as current practice, common practice, or industry standard practice. Evaluators determine these practices through multiple methods, but often can be from self-report or on-site audits. The difference between the energy use of measures installed in the program and the energy use associated with current practice is considered by some to be sufficiently close to the net savings.

This approach is not in use in Illinois, but it is used elsewhere in the country, such as the Pacific Northwest and Delaware.

6.5 Market Analyses

Market analyses can be done in several ways. Market analyses are often used in theory-driven evaluations of market transformation programs.

Other non-sales data market analyses can be postulated on changes specified in program logic such as: 1) changes in the number of energy-efficient units manufactured; 2) changes in market actor behavior around promotion or stocking of energy-efficient items; or 3) reductions in prices. The analyses involving non-sales data must make a clear link between the program intervention and the changes found in the market. Additionally, outside of Illinois, while evaluators have extrapolated the market changes to specific energy or demand reductions, this activity may be viewed as tenuous due to assumptions that evaluators must make within the analysis.

Illinois is in a position to begin to discuss market analyses and how specific research may be able to interpret changes that have occurred (or may occur in the future) because of the program interventions over the past eight years. Market analyses can be backward looking through historical tracing, but it is best used when the logic of an intervention is described and specific market metrics are tracked over time.

6.6 Structured Expert Judgment Approaches

Closely tied to market analysis, this approach is a way for evaluators to gather credible evidence of changes that arise due to the intervention of a program. When deployed, it is often used as a cost-effective approach to estimate market effects or reach agreement on a NTG value when several different types of evidence are available. The key premise of this approach is the use of a select group of known experts that all stakeholders agree can provide unbiased information as well as having sufficient knowledge to judge what may have occurred absent a program intervention.

A Delphi Panel is an example of this approach where data are collected from two or more rounds of data collection (which can occur via e-mail, Internet, or in person). A round is when experts make their thoughts known about a specific subject; the evaluation team synthesizes the data and provides this collated data back to the group to discuss again. Allowing the full experts to see how their peers think about a topic helps to move the group towards consensus.

References

- Mosenthal, et al., 2000
- Powell, 2002

6.7 Program Theory-Driven Approach

This approach is not included in the Violette and Rathbun (2014) document as a high-level method, but it is discussed by the authors under the historical tracing method. The Illinois evaluators believe that it deserves at least a short discussion within this framework.

A program theory is the written narrative about why the activities of a program are expected to bring about change. Typically associated with this approach is the direct graphical explication of the linkages between activities, outputs, and outcomes through an impact logic model.⁶⁶

A theory-driven evaluation denotes “[A]ny evaluation strategy or approach that explicitly integrates and uses stakeholder, social science, some combination of, or other types of theories in conceptualizing, designing, conducting, interpreting, and applying an evaluation.” (Coryn 2011) Within this approach, the ultimate conclusions regarding the efficacy of a program are based on the preponderance of the evidence and not on the results of any single analysis. Coryn and colleagues systematically examined 45 cases of theory-driven evaluations published over a 20-year period to ascertain how closely theory-driven evaluation practices comport with the key tenants of theory-driven evaluation as described and prescribed by prominent theoretical writers. One output from this analysis was the identification of the core principles and sub-principles of theory-driven evaluation. If interested, please review the reference under Coryn 2011.

As an approach, it is best used for complex programs and/or causal mechanisms that extend far into the future. Evaluators collect evidence that supports or rejects hypotheses that are explicit in the logic model. The case for program attribution is strengthened based on the extent to which an evaluation shows that the expected changes occur. Additionally, the evaluation team may be able to collect data that will answer questions about the longer-term outcomes of a program. This type of data collection may be very similar to market tracking activities described briefly above under Market Analyses.

This approach does not specifically estimate a NTG value, but Program Administrators can choose to keep, drop, or change a program based on intermediary data. Regulators must be convinced that the logic of a program is sound and that the intermediary outcomes are causally linked to expected savings.

References

- Weiss, 1997
- Chen, 2000
- Coryn, 2011

6.8 Case Studies Design

Case studies are used extensively in social sciences as well as many other disciplines or practice-oriented areas, such as political science, economics, education, and public policy. Case studies help to understand the how and why of a situation and typically retain a holistic aspect of real-life events. As such, they may be a useful approach to determine attribution. As with program theory design, though, the data collected and analyzed within a case

⁶⁶ Evaluators may use logic models to show program processes as well, but this is a program flow chart, not an impact model.

study approach will not typically yield a specific NTG value, but can provide credible evidence and insight that supports or refutes the changes brought about by program intervention.

To be used to assess attribution, evaluators must carefully design case studies to assure they account for the threats to causality (i.e., internal validity) that arise in any design. While not typically thought of in this manner, case study design can address multiple types of validity such as construct, internal, and external validity as well as assuring reliability. When establishing construct validity and reliability, evaluators must use multiple sources of evidence, create and maintain a study database, and maintain a “chain of evidence” within the analysis. Internal validity is shown through analytic tactics such as pattern matching, explanation building, addressing rival explanations, or using logic models. External validity centers on the ability to generalize the analytical findings to other similar situations. External validity may be shown through the replication of findings.

References

- Yin, 2003
- Stake, 2006

7 Appendix B: References

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